



UNIVERSITAT POLITÈCNICA DE CATALUNYA  
BARCELONATECH

Escola Superior d'Enginyeries Industrial,  
Aeroespacial i Audiovisual de Terrassa

ESEIAAT - UPC

# **Study for the computational resolution of conservation equations of mass, momentum and energy. Possible application to different aeronautical and industrial engineering problems: Case 1B**

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Attachment A - Results

**Author:** Laura Pla Olea

**Director:** Carlos David Perez Segarra

**Co-Director:** Asensio Oliva Llena

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# 1 | Smith-Hutton problem

In the following sections there are the results of the Smith-Hutton problem for all the resolution schemes that have been coded. The purpose of this chapter is to demonstrate that all the schemes produce a similar result.

## 1.1 $\rho/\Gamma = 10$

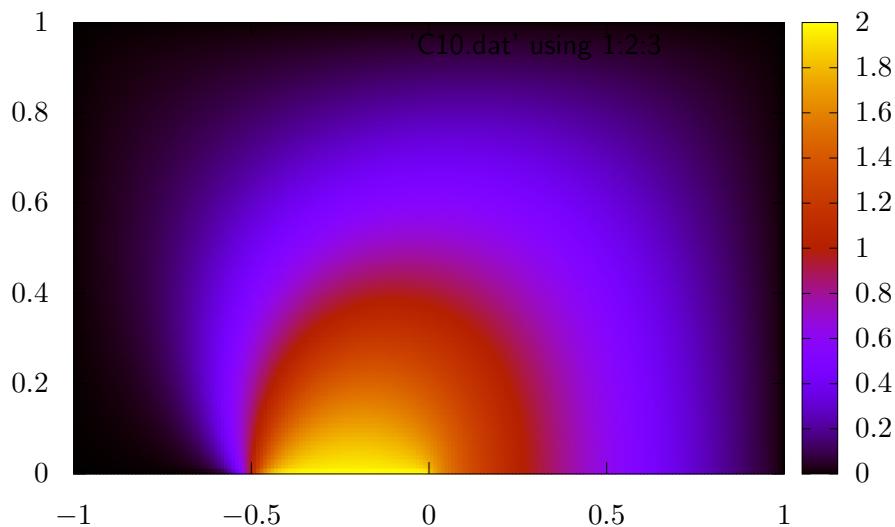


Figure 1.1: Representation of the whole domain for  $\rho/\Gamma = 10$  (CDS)

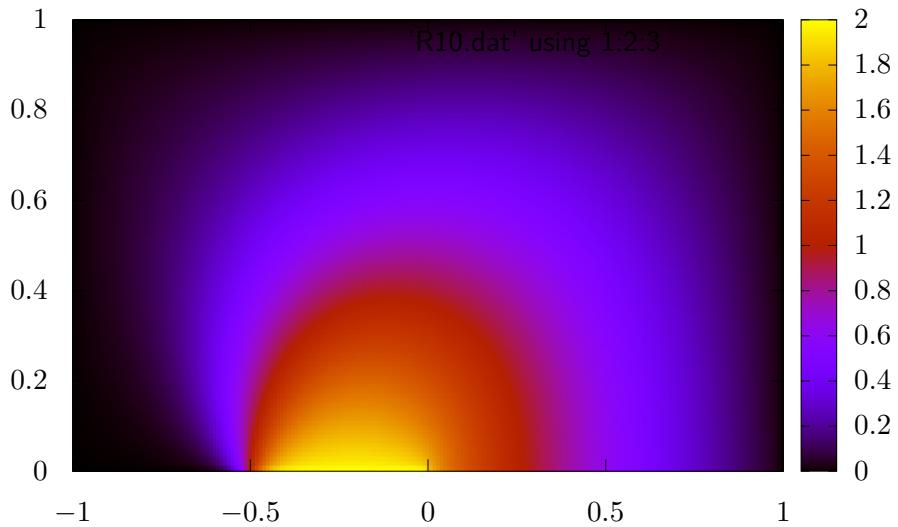


Figure 1.2: Representation of the whole domain for  $\rho/\Gamma = 10$  (UDS)

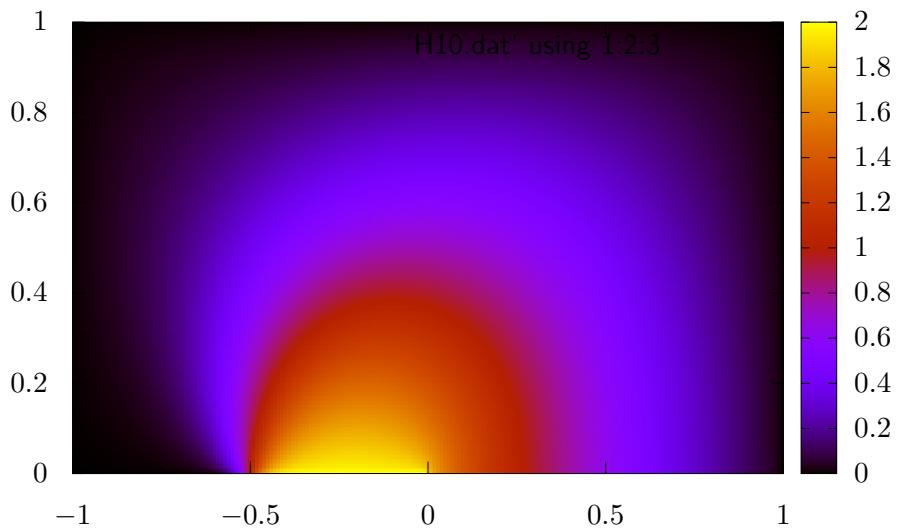


Figure 1.3: Representation of the whole domain for  $\rho/\Gamma = 10$  (HDS)

$$\rho/\Gamma = 10^3$$

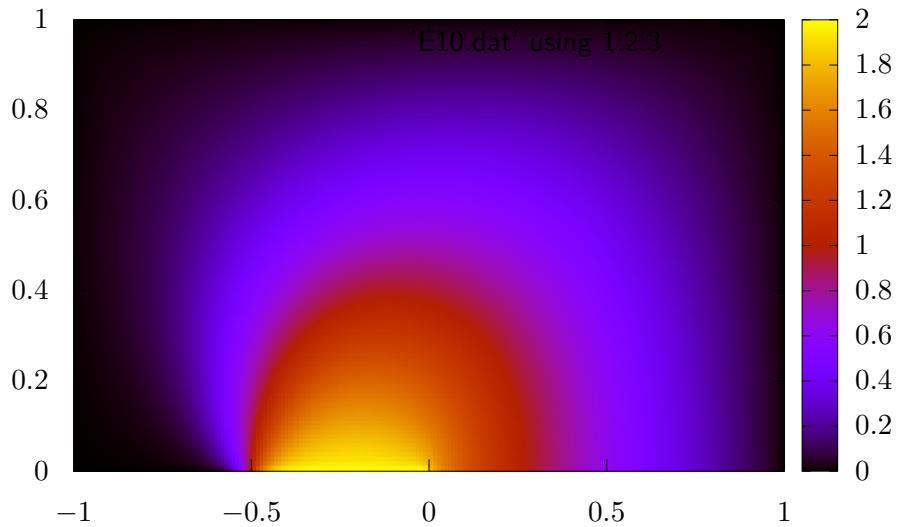


Figure 1.4: Representation of the whole domain for  $\rho/\Gamma = 10$  (EDS)

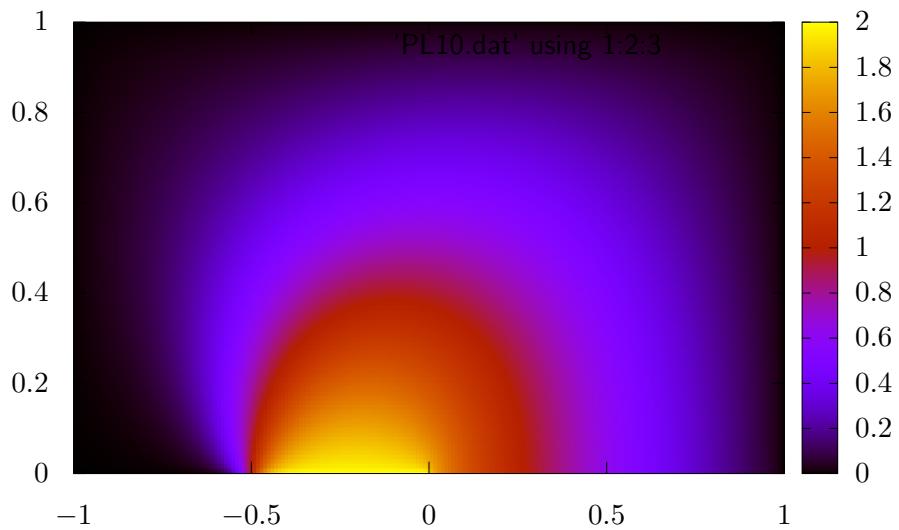


Figure 1.5: Representation of the whole domain for  $\rho/\Gamma = 10$  (PLDS)

## 1.2 $\rho/\Gamma = 10^3$

In this section there is not a solution for the central differencing scheme because it diverges.

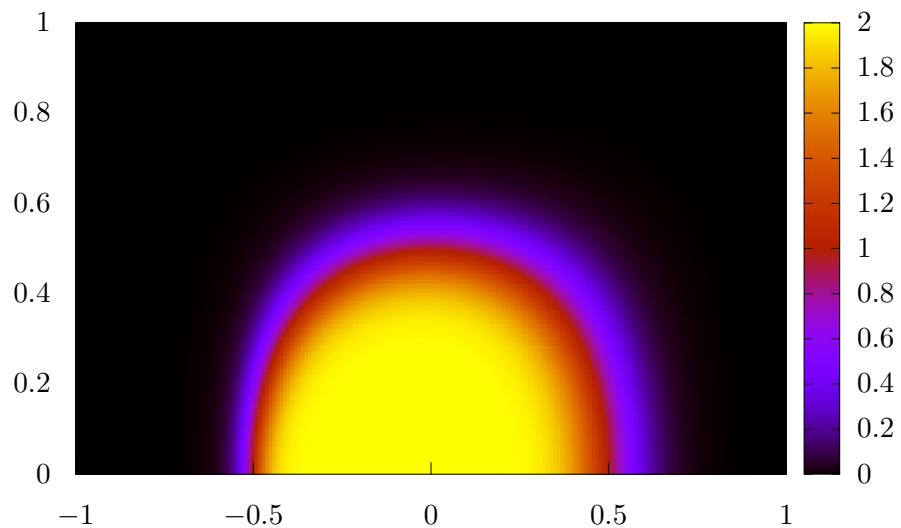


Figure 1.6: Representation of the whole domain for  $\rho/\Gamma = 10^3$  (UDS)

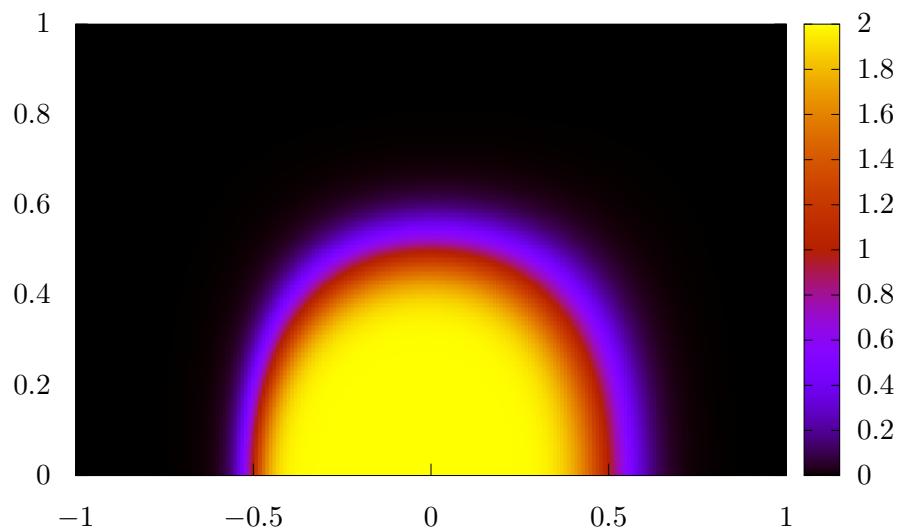


Figure 1.7: Representation of the whole domain for  $\rho/\Gamma = 10^3$  (HDS)

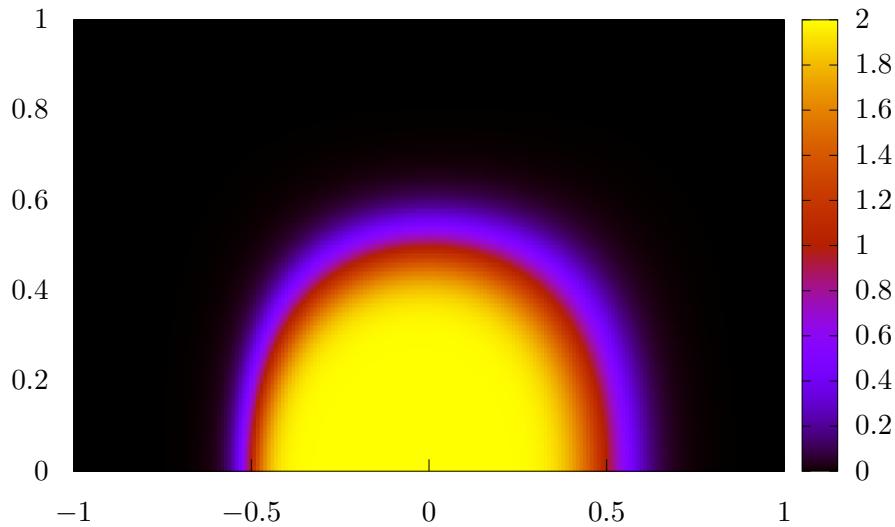


Figure 1.8: Representation of the whole domain for  $\rho/\Gamma = 10^3$  (EDS)

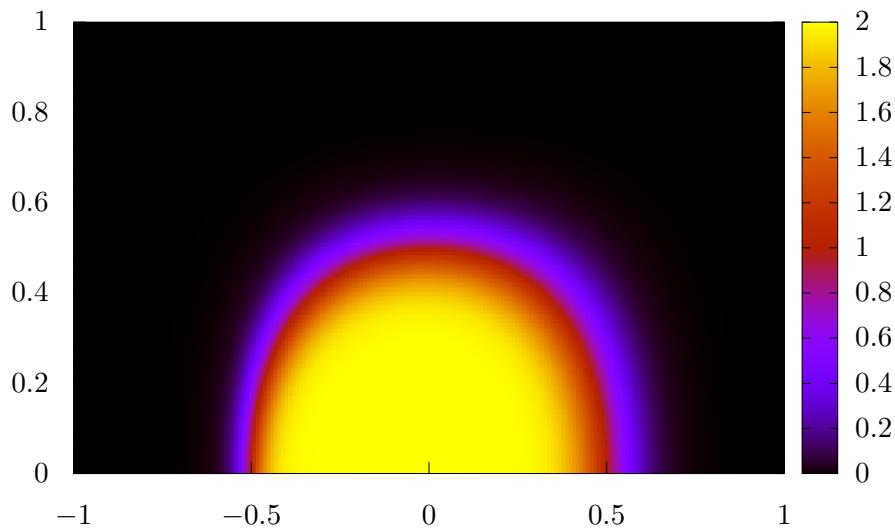


Figure 1.9: Representation of the whole domain for  $\rho/\Gamma = 10^3$  (PLDS)

$$\rho/\Gamma = 10^6$$

### 1.3 $\rho/\Gamma = 10^6$

Like in the previous section, in this case there is no results for the central differencing scheme because of its divergence.

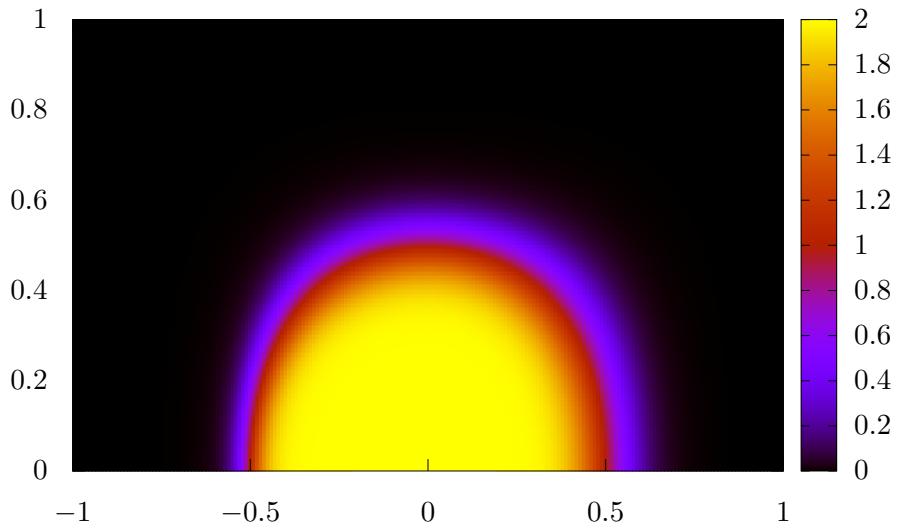


Figure 1.10: Representation of the whole domain for  $\rho/\Gamma = 10^6$  (UDS)

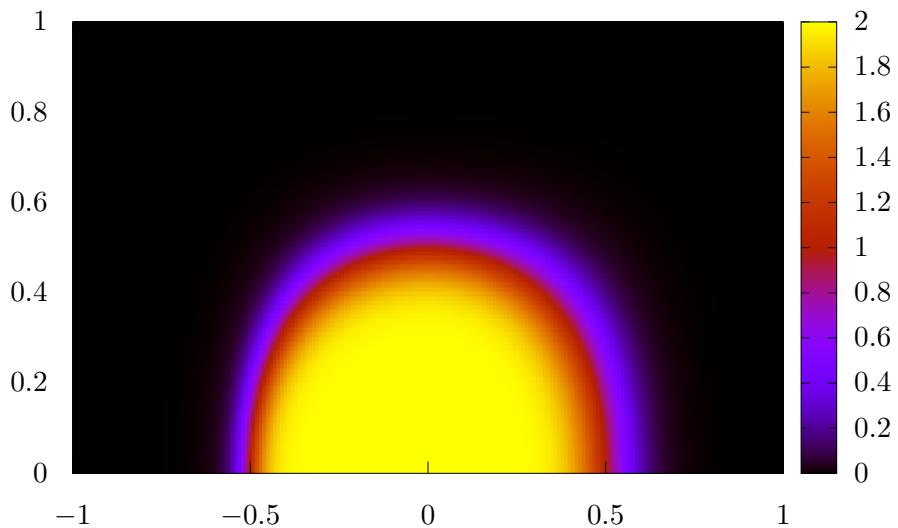


Figure 1.11: Representation of the whole domain for  $\rho/\Gamma = 10^6$  (HDS)

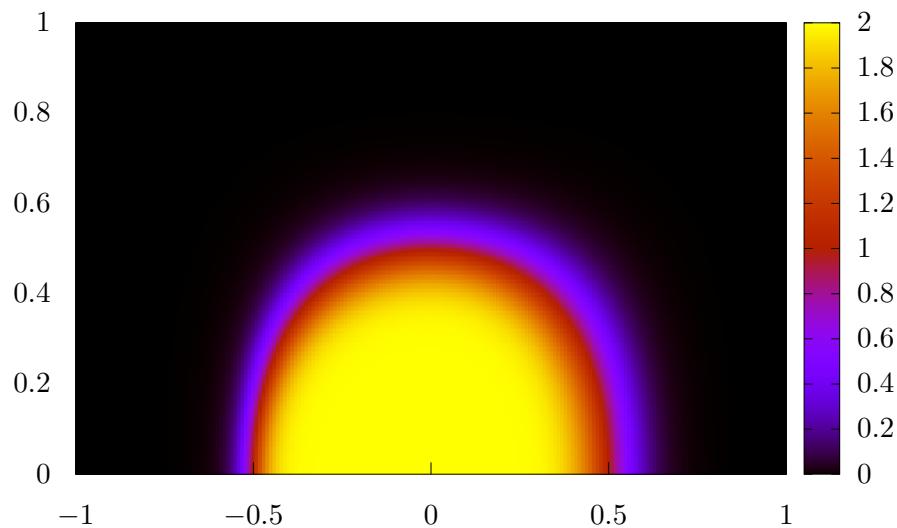


Figure 1.12: Representation of the whole domain for  $\rho/\Gamma = 10^6$  (EDS)

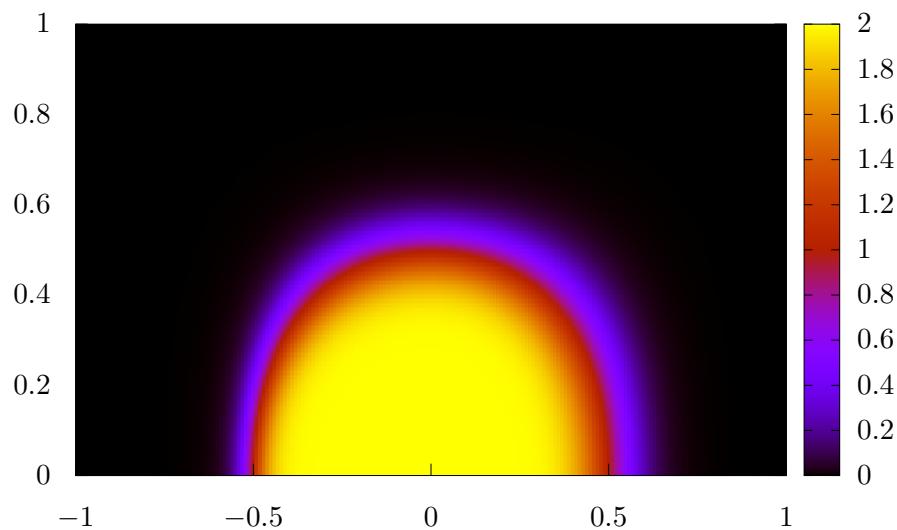
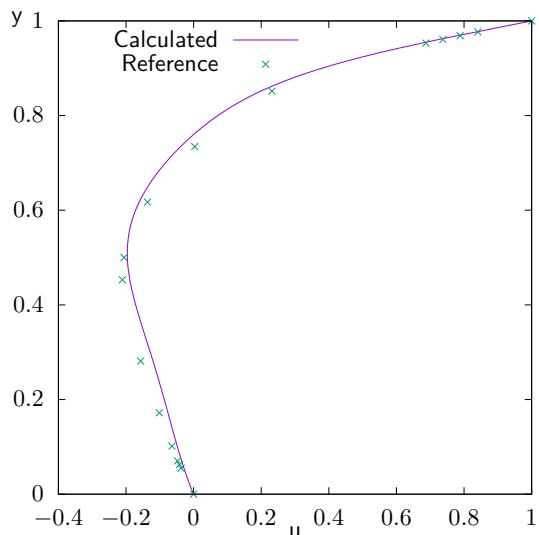
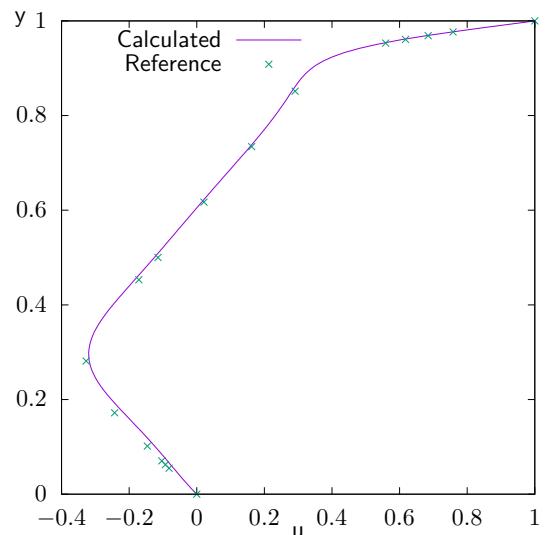


Figure 1.13: Representation of the whole domain for  $\rho/\Gamma = 10^6$  (PLDS)

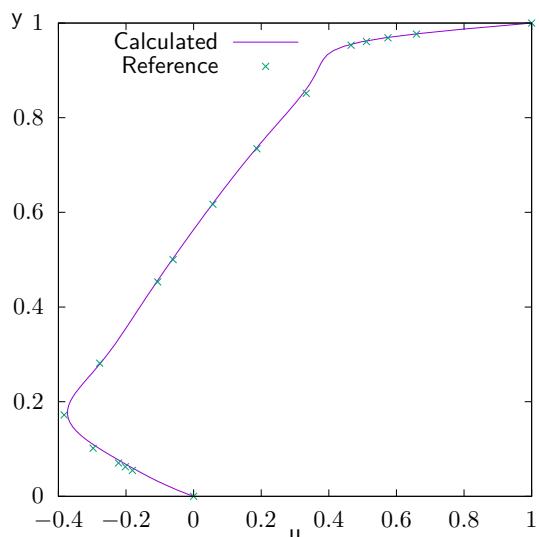
## 2 | Driven cavity problem



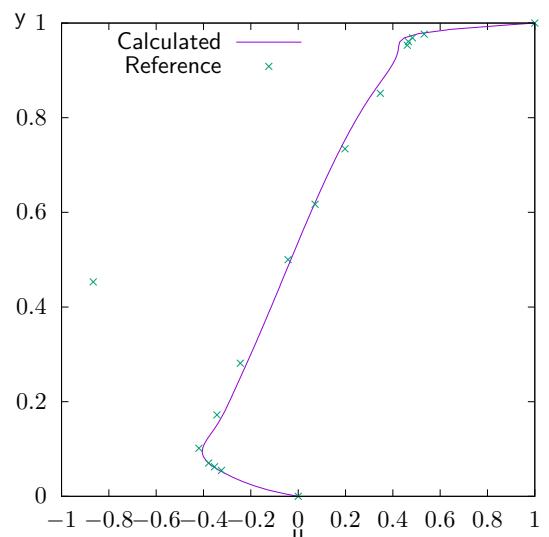
(a)  $Re = 100$



(b)  $Re = 400$



(c)  $Re = 1000$



(d)  $Re = 3200$

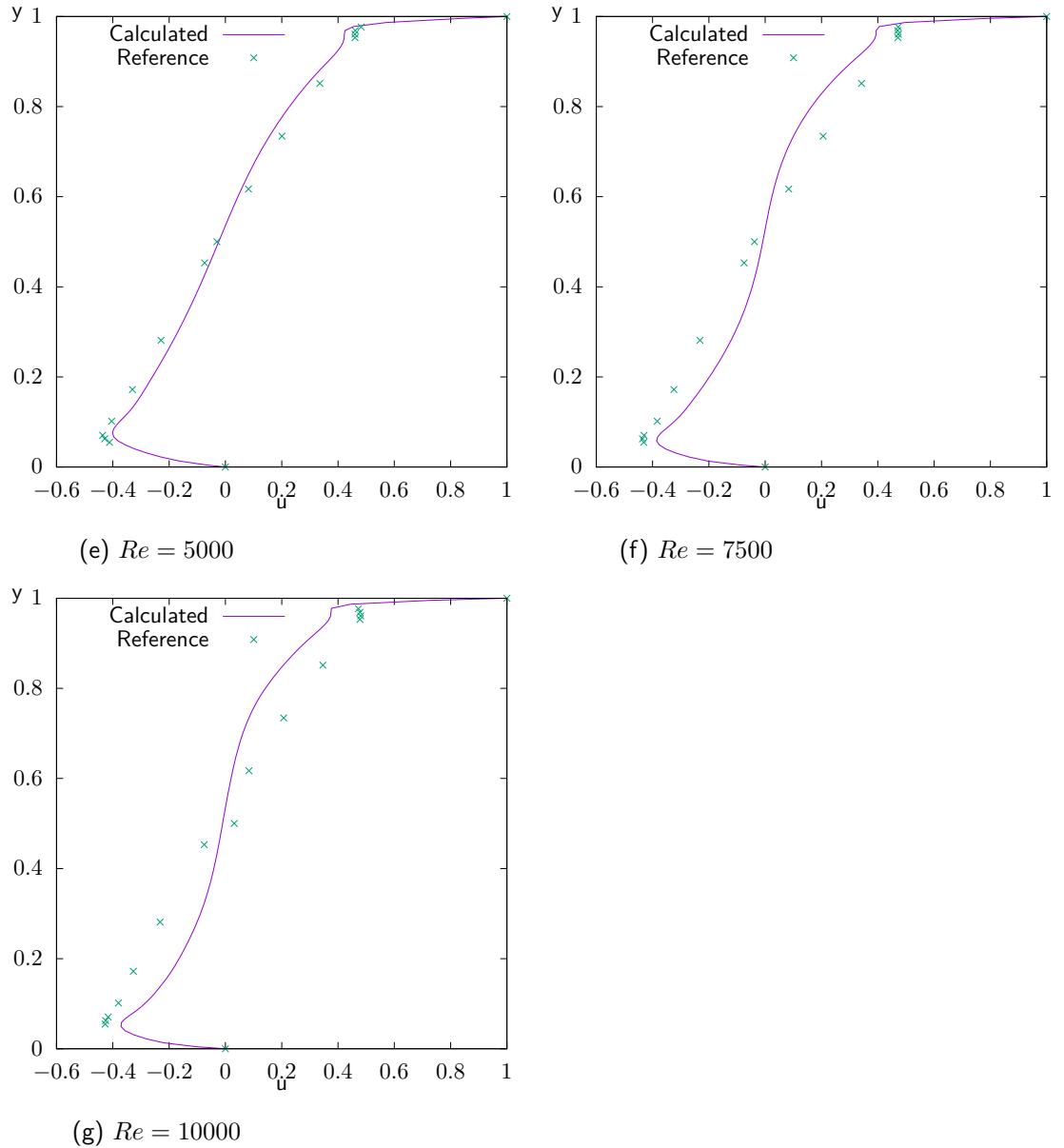
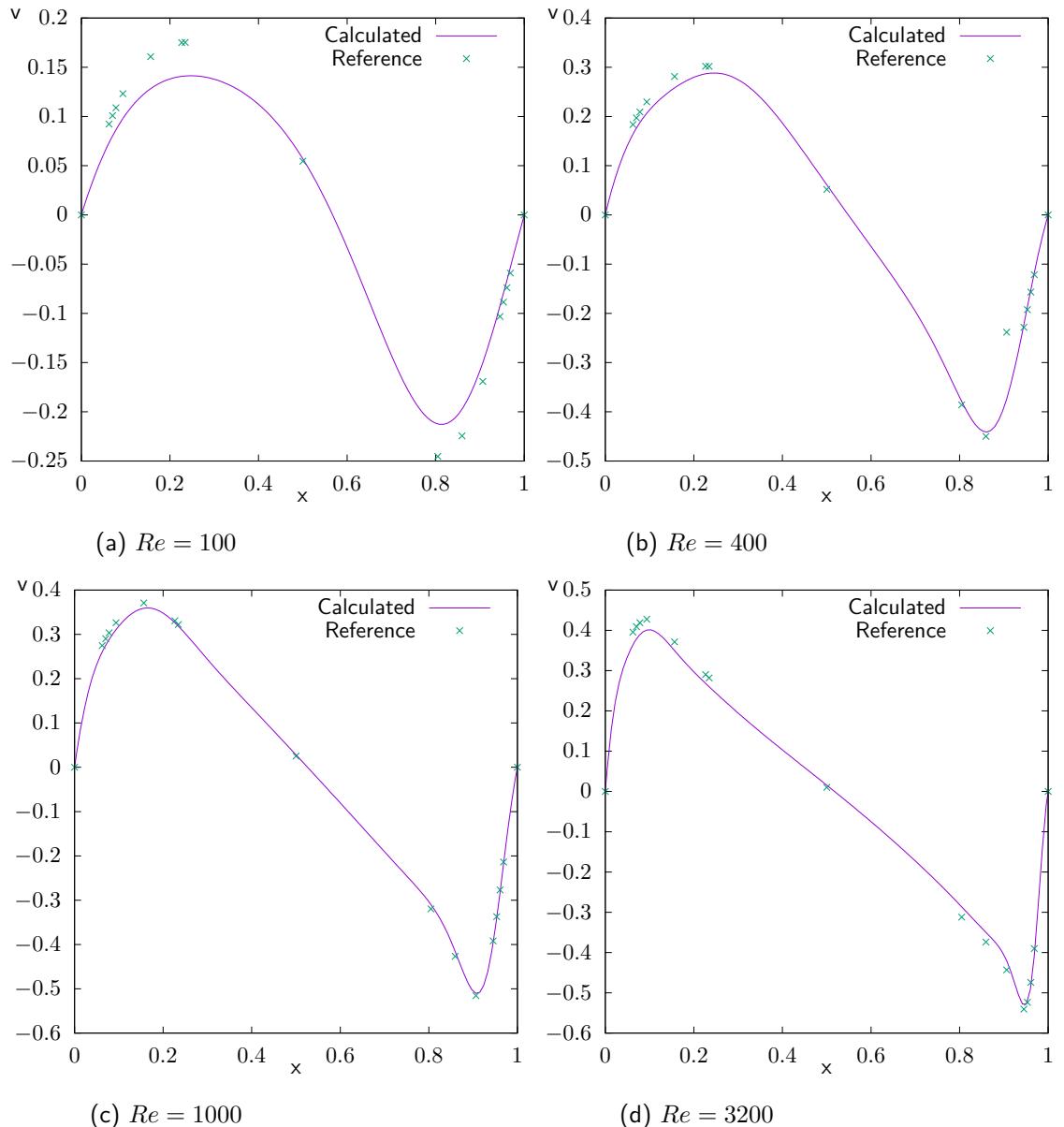


Figure 2.1: Comparison between the reference solution and the calculated one of the horizontal velocity along the vertical line in the geometric centre of the cavity [1]



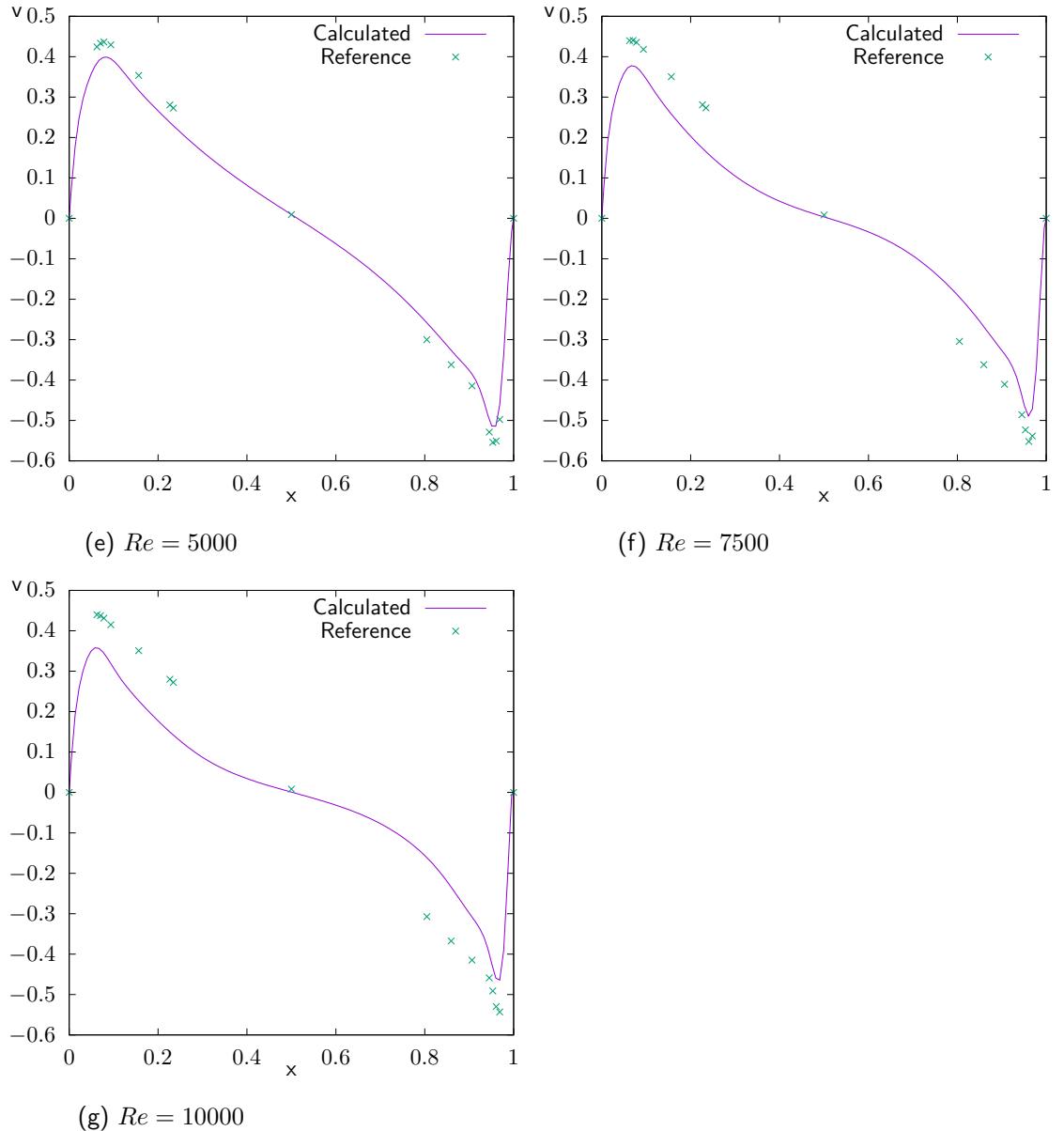


Figure 2.2: Comparison between the reference solution and the calculated one of the vertical velocity along the horizontal line in the geometric centre of the cavity [1]

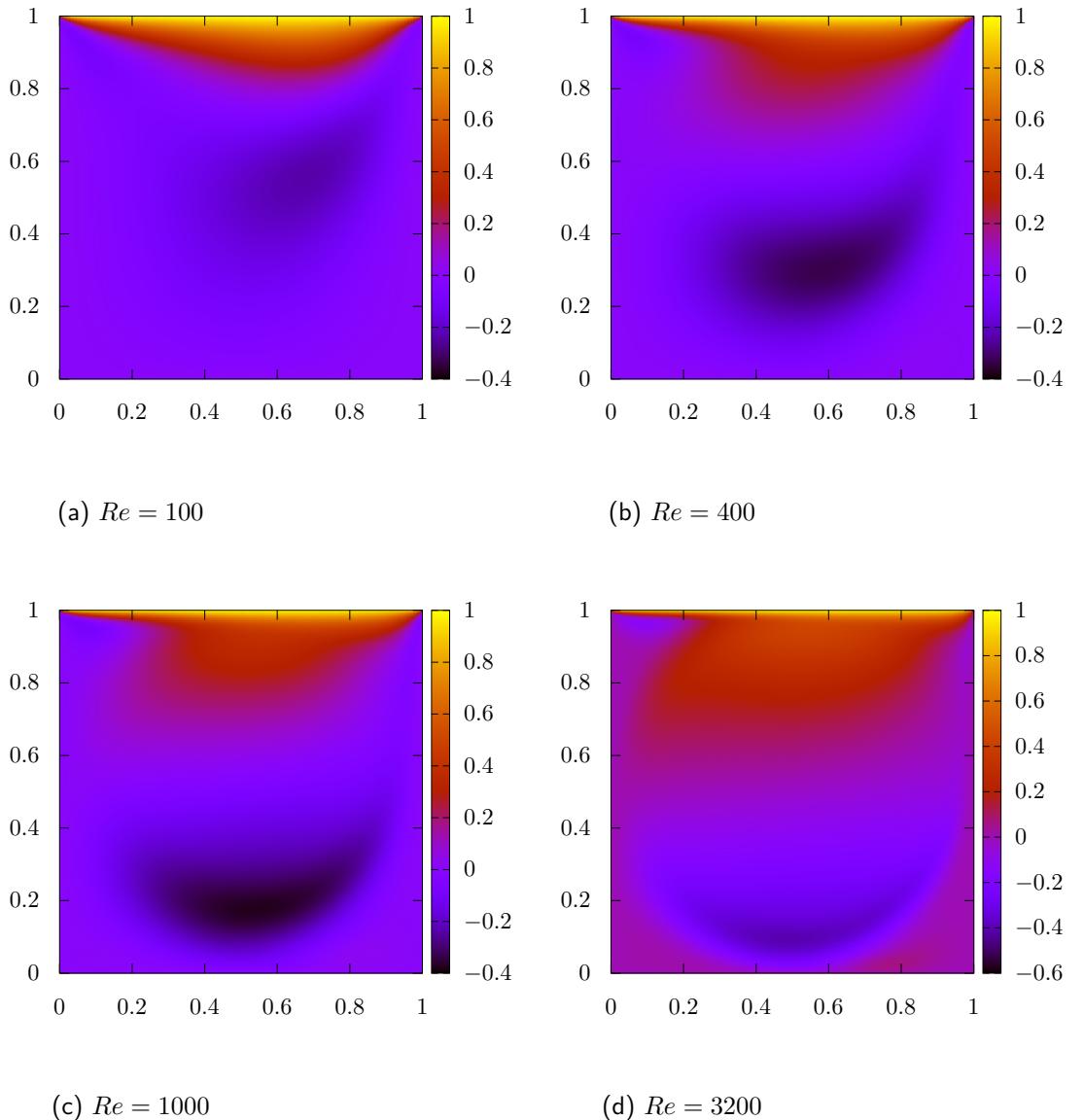
x	Re=100	Re=400	Re=1000	Re=3200	Re=5000	Re=7500	Re=10000
1	0.00%	0.00%	0.00%	0.00%	0.00%	0.00%	0.00%
0.9766	0.53%	0.41%	1.07%	5.91%	7.13%	15.08%	28.11%
0.9688	0.72%	0.49%	1.14%	6.57%	8.95%	17.61%	32.03%
0.9609	1.04%	0.77%	1.69%	7.74%	10.25%	18.46%	33.60%
0.9531	1.32%	1.06%	2.40%	8.22%	10.50%	18.68%	34.58%
0.8516	11.04%	2.76%	3.10%	8.88%	15.28%	32.24%	52.58%
0.7344	1020.07%	2.66%	2.01%	10.68%	24.81%	50.19%	73.70%
0.6172	-15.41%	25.44%	3.47%	10.85%	31.55%	61.89%	73.63%
0.5	3.19%	-7.42%	-4.01%	18.68%	30.72%	74.06%	34.89%
0.4531	7.98%	-5.56%	-1.91%	91.66%	30.59%	64.80%	112.56%
0.2813	19.18%	2.53%	1.24%	12.55%	21.26%	46.49%	55.93%
0.1719	21.37%	9.01%	3.10%	8.44%	14.73%	29.52%	49.33%
0.1016	20.90%	12.06%	5.98%	5.22%	8.95%	17.16%	36.85%
0.0703	22.33%	13.08%	7.10%	1.78%	10.75%	14.22%	25.13%
0.0625	20.17%	13.27%	7.29%	0.43%	11.22%	14.20%	22.76%
0.0547	19.93%	13.41%	7.39%	-1.15%	11.26%	13.84%	20.37%
0	0.00%	0.00%	0.00%	0.00%	0.00%	0.00%	0.00%

Table 2.1: Error of the horizontal velocities in the vertical central plane

y	Re=100	Re=400	Re=1000	Re=3200	Re=5000	Re=7500	Re=10000
1	0.00%	0.00%	0.00%	0.00%	0.00%	0.00%	0.00%
0.9688	7.43%	2.41%	-1.00%	-1.16%	11.69%	17.15%	20.24%
0.9609	7.29%	2.02%	-0.95%	0.95%	10.81%	14.42%	17.89%
0.9531	7.43%	1.98%	-0.68%	2.17%	9.08%	11.53%	16.78%
0.9453	7.52%	1.87%	-0.57%	3.11%	7.55%	10.48%	18.96%
0.9063	8.21%	-57.88%	1.57%	5.68%	8.89%	18.94%	37.44%
0.8594	9.09%	1.84%	2.81%	7.39%	12.11%	26.30%	47.77%
0.8047	10.32%	1.77%	3.34%	9.04%	16.06%	35.75%	53.28%
0.5	-6.69%	-14.99%	-10.95%	-40.59%	3.41%	61.67%	331.52%
0.2344	15.90%	3.89%	1.56%	8.54%	18.07%	39.01%	46.30%
0.2266	16.04%	4.24%	1.74%	8.44%	17.50%	37.78%	45.79%
0.1563	16.84%	6.79%	3.61%	6.87%	12.28%	26.49%	44.39%
0.0938	17.15%	8.22%	4.96%	7.54%	9.99%	16.36%	32.74%
0.0781	17.29%	8.58%	5.25%	8.26%	10.81%	15.88%	29.86%
0.0703	17.31%	8.73%	5.40%	8.62%	11.32%	16.21%	29.30%
0.0625	17.29%	8.82%	5.51%	8.93%	11.70%	16.66%	29.23%
0	0.00%	0.00%	0.00%	0.00%	0.00%	0.00%	0.00%

Table 2.2: Error of the vertical velocities in the horizontal central plane

## 2.1 Horizontal velocities



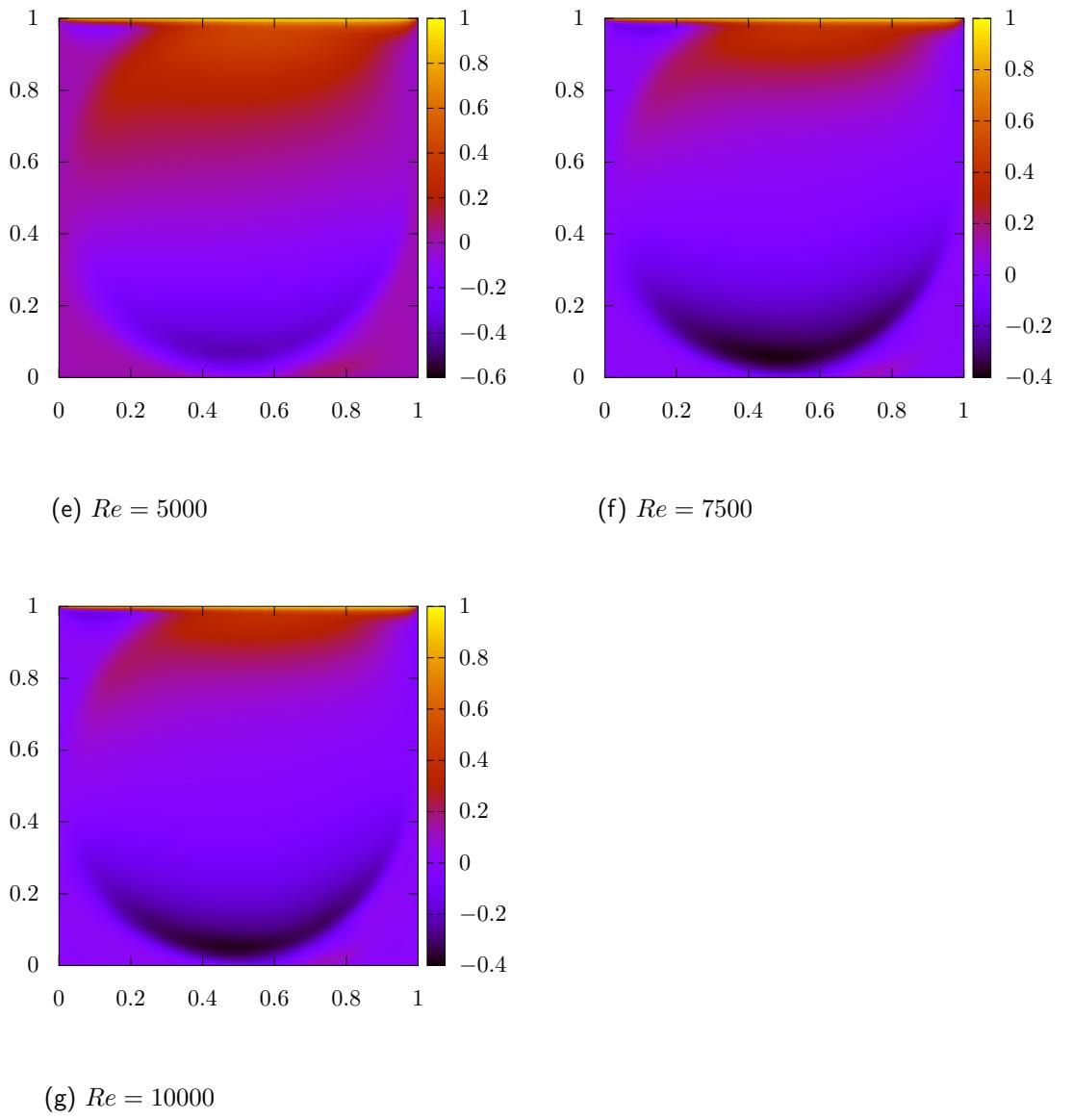
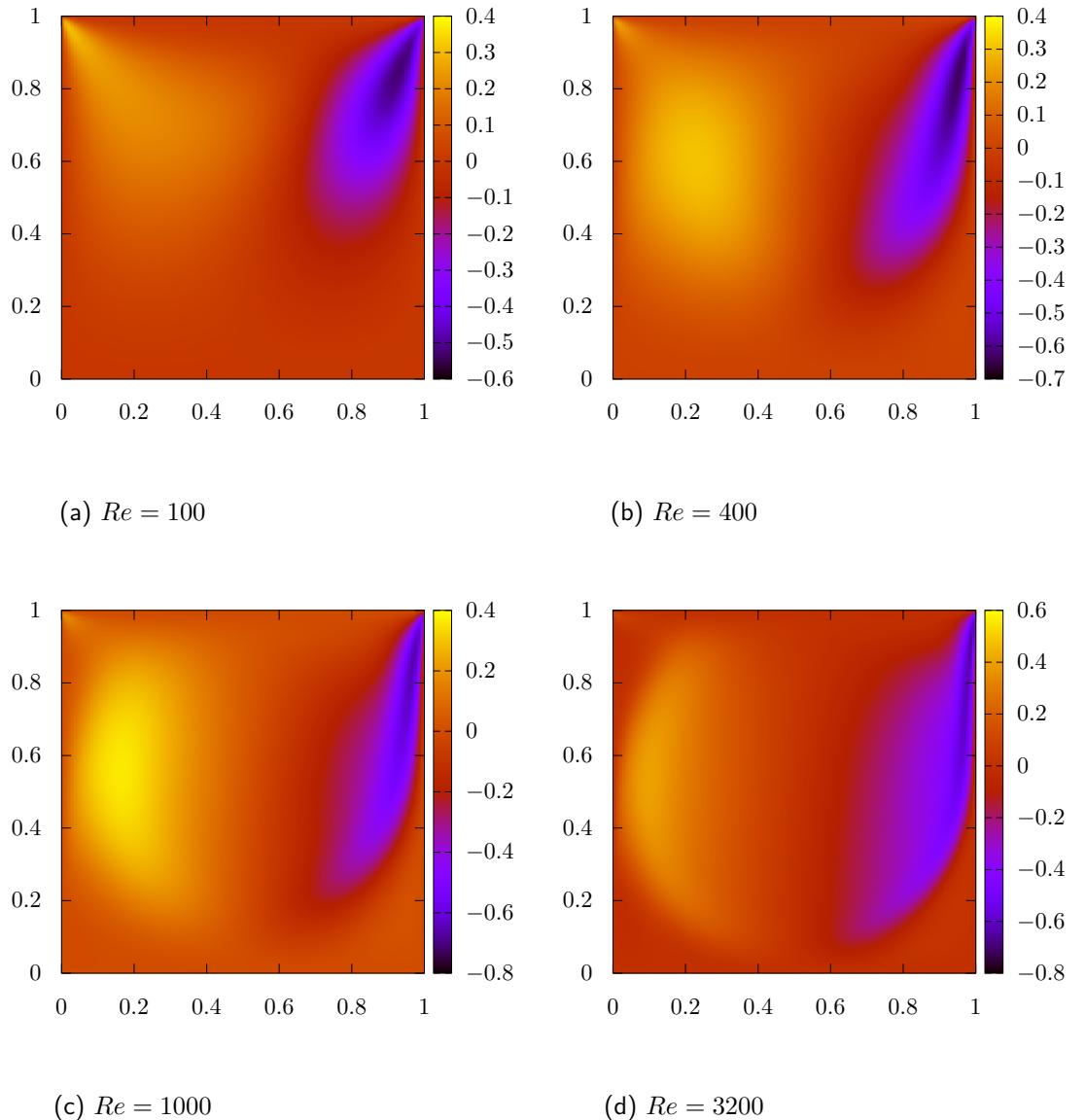


Figure 2.3: Horizontal velocity inside the cavity

## 2.2 Vertical velocities



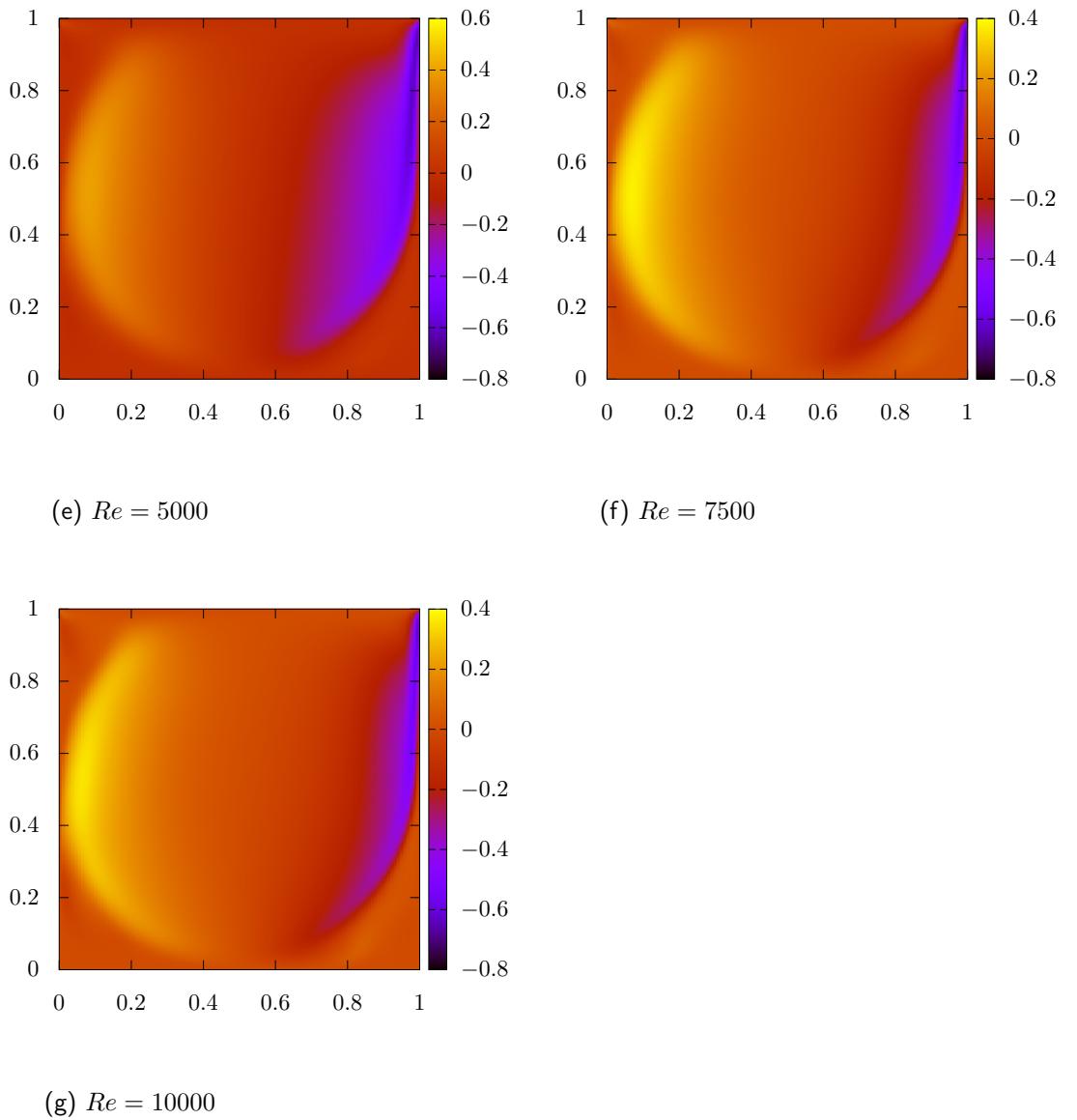
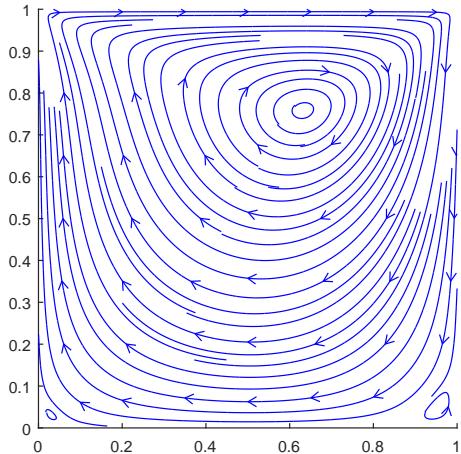
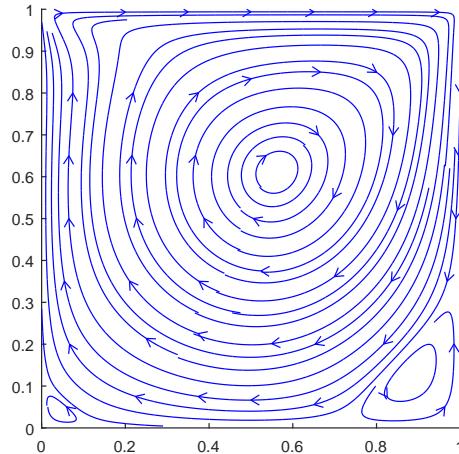
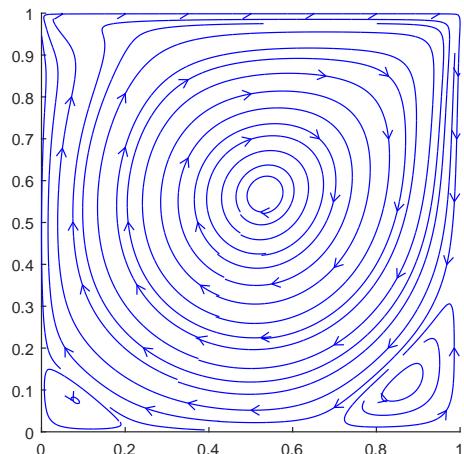
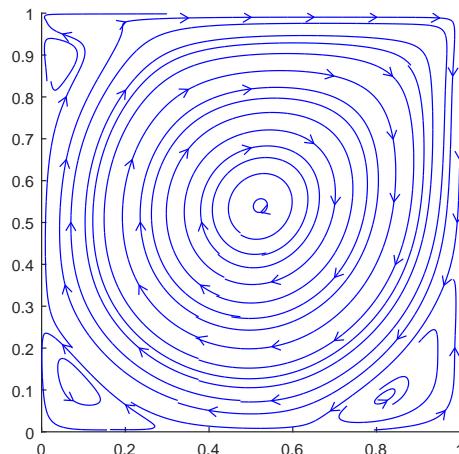


Figure 2.4: Vertical velocity inside the cavity

## 2.3 Streamlines

(a)  $Re = 100$ (b)  $Re = 400$ (c)  $Re = 1000$ (d)  $Re = 3200$

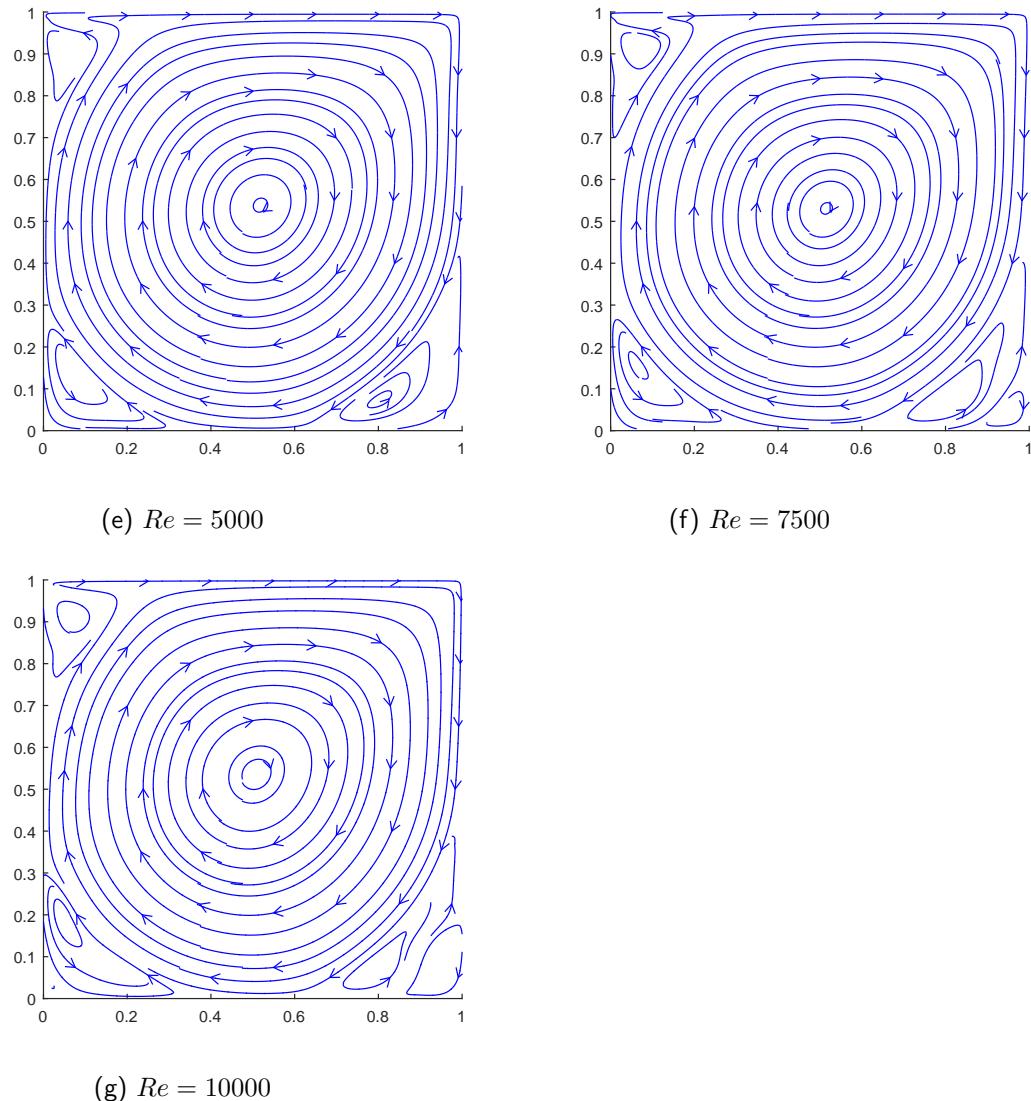


Figure 2.5: Streamlines of the flow inside the cavity

# 3 | Buoyancy

	$Ra = 10^3$		$Ra = 10^4$		$Ra = 10^5$		$Ra = 10^6$	
	Calculated	Bench	Calculated	Bench	Calculated	Bench	Calculated	Bench
$u_{max}$	3.708	3.649	16.198	16.178	34.508	34.73	66.192	64.63
$y$	0.81	0.813	0.83	0.823	0.85	0.855	0.85	0.85
$v_{max}$	3.908	3.697	19.648	19.617	68.663	68.59	220.23	219.36
$x$	0.17	0.178	0.11	0.119	0.07	0.066	0.03	0.038
$\bar{N}u$	1.122	1.118	2.244	2.243	4.535	4.519	9.031	8.8
$Nu_{1/2}$	1.095	1.118	2.228	2.243	4.536	4.519	9.110	8.799
$Nu_0$	1.367	1.117	2.331	2.238	4.669	4.509	9.419	8.817
$Nu_{max}$	1.795	1.505	3.626	3.528	8.045	7.717	20.019	17.925
$y$	0.09	0.092	0.15	0.143	0.07	0.081	0.03	0.038
$Nu_{min}$	0.873	0.692	0.616	0.586	0.764	0.729	1.062	0.989
$y$	1	1	1	1	1	1	1	1

Table 3.1: Comparison between the benchmark solution and the calculated one [2]

	$Ra = 10^3$	$Ra = 10^4$	$Ra = 10^5$	$Ra = 10^6$
$u_{max}$	-1.627%	-0.126%	0.640%	-2.417%
$y$	0.369%	-0.851%	0.585%	0.000%
$v_{max}$	-5.715%	-0.159%	-0.106%	-0.397%
$x$	4.494%	7.563%	-6.061%	20.844%
$\bar{N}u$	-0.358%	-0.046%	-0.355%	-2.620%
$Nu_{1/2}$	2.049%	0.654%	-0.375%	-3.540%
$Nu_0$	-22.343%	-4.158%	-3.547%	-6.830%
$Nu_{max}$	-19.282%	-2.787%	-4.251%	-11.681%
$y$	2.174%	-4.895%	13.580%	20.635%
$Nu_{min}$	-26.116%	-5.128%	-4.742%	-7.386%
$y$	0.000%	0.000%	0.000%	0.000%

Table 3.2: Percentage of error between the benchmark solution and the calculated one

# 4 | Square cylinder

## 4.1 Recirculation length

Re	Reference	Calculated
1	-0.0096	0.1067
3	0.1012	0.1865
5	0.212	0.2650
10	0.489	0.4474
30	1.597	1.4971
50	2.705	2.7702

Table 4.1: Comparison of the recirculation length (Reference value vs. Calculated value) [3]

## 4.2 Horizontal velocity

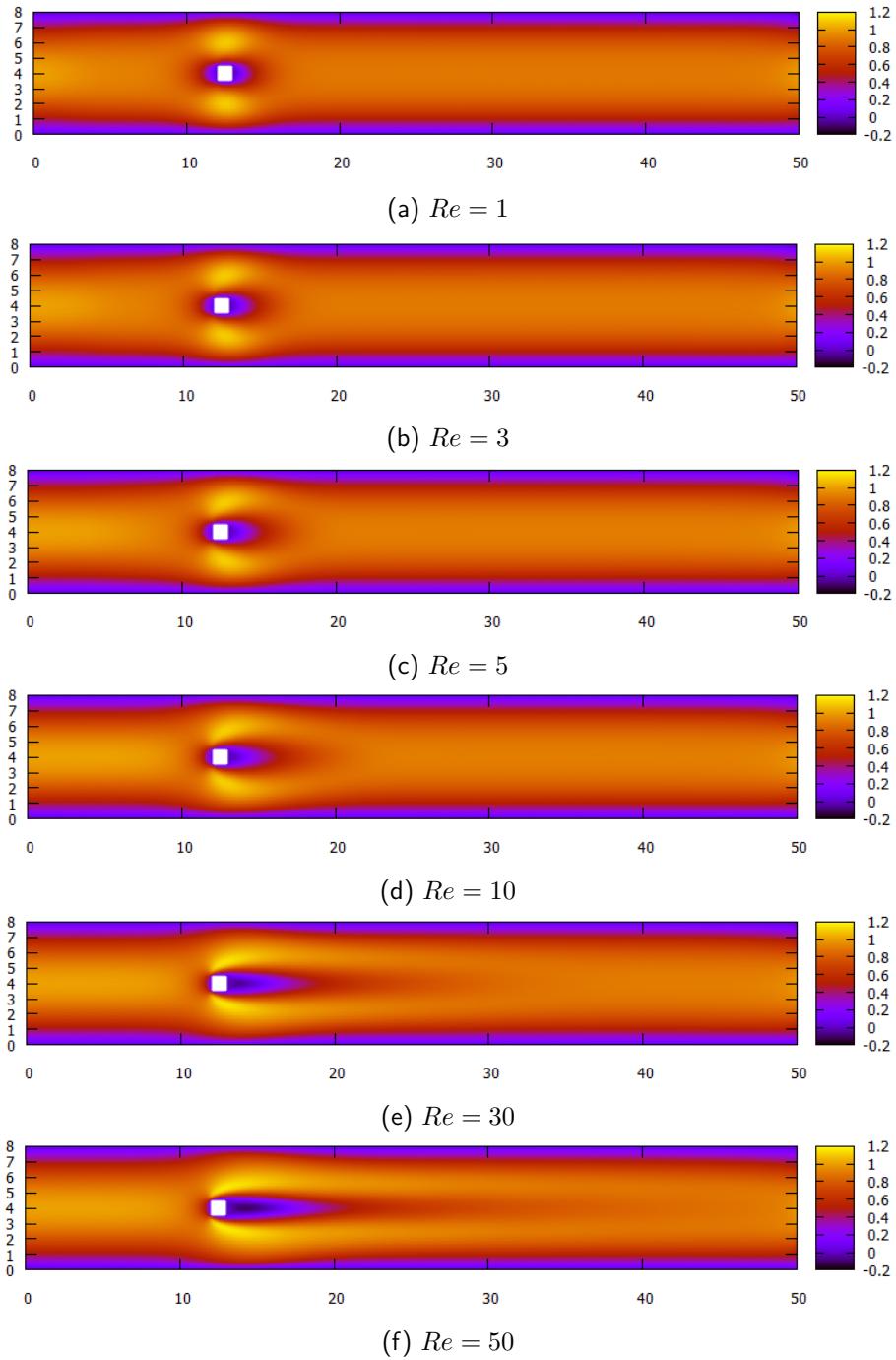


Figure 4.1: Horizontal velocity in the channel

### 4.3 Vertical velocity

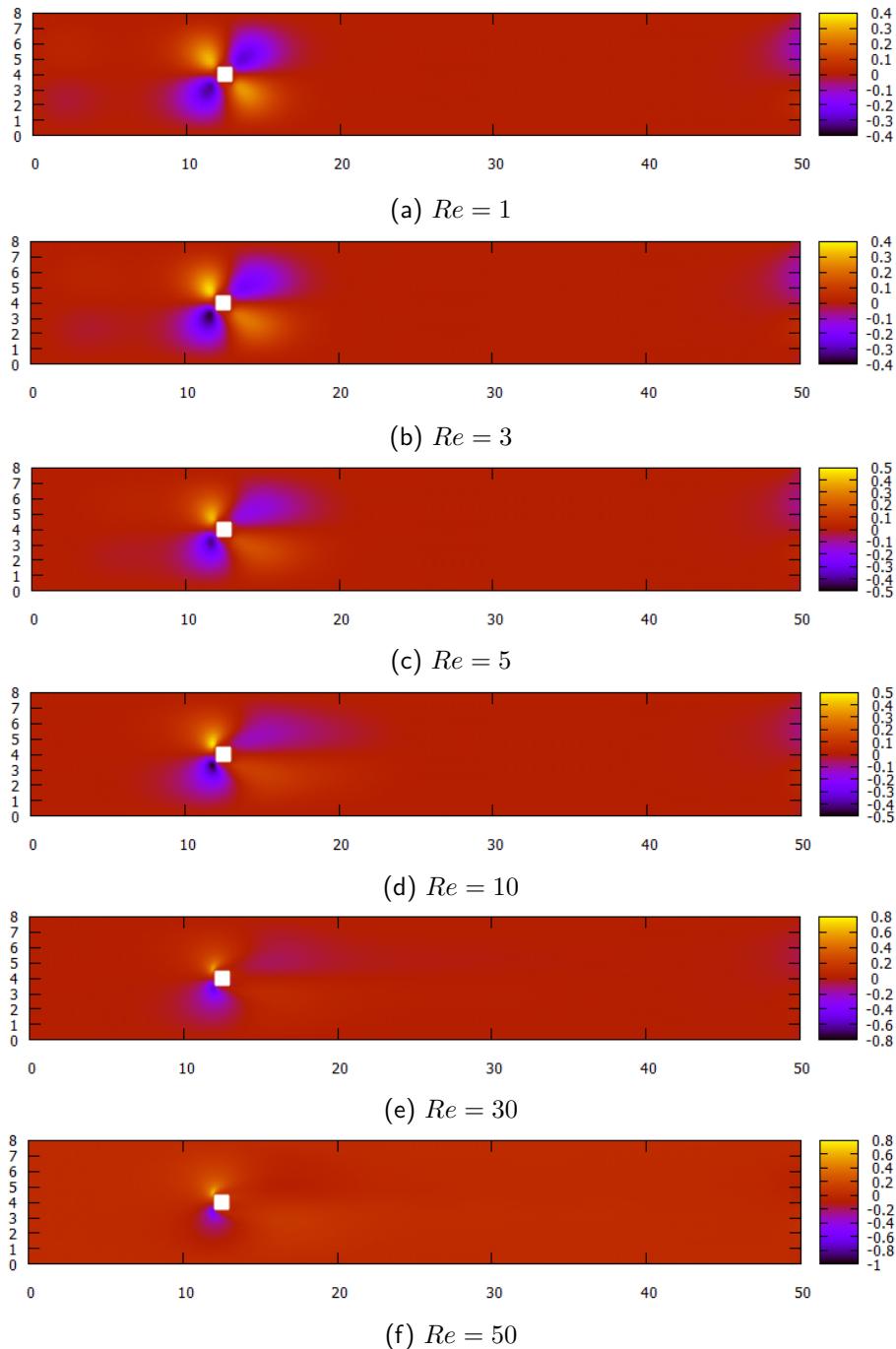


Figure 4.2: Vertical velocity in the channel

# 5 | Bibliography

- [1] U. Ghia, K. N. Ghia, and C. T. Shin. High-Re solutions for incompressible flow using the Navier-Stokes equations and a multigrid method. *Journal of Computational Physics*, 48:387–411, 1982.
- [2] G. de Vahl Davis. Natural convection of air in square cavity : A benchmark numerical solution. *International Journal for Numerical methods in Fluids*, 3(July 1982):249–264, 1983.
- [3] M. Breuer, J. Bernsdorf, T. Zeiser, and F. Durst. Accurate computations of the laminar flow past a square cylinder based on two different methods: Lattice-Boltzmann and finite-volume. *International Journal of Heat and Fluid Flow*, 21(2):186–196, 2000.