几何与物理动画

张淦淦

目	录		1 几何动画
1	几何动画	1	1.1 蒙皮骨骼动画
	 1.1 蒙皮骨骼动画	1 1	1.2 刚体运动学(Kinematics)
2	弹簧质点系统	1	2 弹簧质点系统
	2.1 显示欧拉与隐式欧拉	1	胡克定律
	2.2 Verlet方法	2	$f_{ij} = -k(x_i - x_j _2 - l_{ij}) \frac{x_i - x_j}{ x_i - x_j _2}$
3	PBD方法	2	11 · J11-
	3.1 XPBD	3	每个点的受力 $f_{i} = \sum_{j} f_{j}$
	3.1.1 基本流程	3	$f_i = \sum_{j,j eq i} f_{ij}$
	3.1.2 原理	3	牛顿第二定理得到点i的运动信息
4	刚体仿真	4	$rac{\partial v_i}{\partial t} = rac{1}{m_i} f_i$
	4.1 刚体运动参数化表达	4	
	4.2 刚体速度的表示	5	$rac{\partial x_i}{\partial t} = v_i$
	4.3 作用于刚体的力的表示	5	2.1 显示欧拉与隐式欧拉
	4.4 Forward Kinematics	5	
	4.5 Inverse Kinematics	5 c	显示欧拉 (explicit)
	4.6 刚体动力学	6 6	$v_{t+1} = v_t + \triangle t M^{-1} f(x_t)$
5	弹性体仿真	6	$x_{t+1} = x_t + \triangle t v_t$
J	5.1 弹性体模型	6	半隐式欧拉 (Semi-implicit Euler,symplectic Eu-
	5.2 FEM方法	8	ler, explicit)
	5.3 MPM方法	9	$v_{t+1} = v_t + \triangle t M^{-1} f(x_t)$
	5.3.1 APIC仿真框架	9	$x_{t+1} = x_t + \triangle t v_{t+1}$
	5.3.2 MLSMPM仿真框架	10	
6	流体仿真	10	隐式欧拉 (Backward Euler)
U	6.1 原理	10 10	$x_{t+1} = x_t + \triangle t v_{t+1}$
	6.2 实现	11	$v_{t+1} = v_t + \triangle t M^{-1} f(x_{t+1})$
7	附录	12	$=v_t + \Delta t M^{-1} f(x_t + \Delta t v_{t+1})$
•	7.1 Schur Complement	12	$\approx v_t + \triangle t M^{-1} \left[f(x_t) + \frac{\partial f}{\partial x}(x_t) \triangle t v_{t+1} \right]$

$$\left[I - \Delta t^2 M^{-1} \frac{\partial f}{\partial x}(x_t)\right] v_{t+1} = v_t + \Delta t M^{-1} f(x_t)$$

可以用Jacobi/Gauss-Seidel iterations或者Conjugate gradients求解.

融合explicit与implicit积分器

$$\left[I - \beta \triangle t^2 M^{-1} \frac{\partial f}{\partial x}(x_t)\right] v_{t+1} = v_t + \triangle t M^{-1} f(x_t)$$

- $\beta = 0$ 时是forward/semi-implicit(explicit)
- $\beta = 1/2$ 时是middle-point(implicit)
- $\beta = 1$ 时是backward Euler(implicit)

对于Explicit方法(forward,symplectic,RK,...),容易实现,但 $\triangle t$ 选取过大也容易爆炸,不适用于stiff材料.

$$\triangle t \le c\sqrt{\frac{m}{k}}, \ c \sim 1$$

Implicit方法(backward Euler,middle-point),一般比较难实现,每一步变得更加昂贵但是 $\triangle t$ 可以调得大一点.

2.2 Verlet方法

基本Verlet 假设例子 $t + \triangle t$ 时刻的位置是 $r(t + \triangle t)$,将其泰勒展开

$$r(t + \triangle t) = r(t) + v(t)\triangle t + \frac{f(t)}{2m}\triangle t^2 + \frac{\partial r(t)}{3!\partial t}\triangle t^3 + O(\triangle t^4)$$

也将 $r(t + \triangle t)$ 泰勒展开

$$r(t - \Delta t) = r(t) - v(t)\Delta t + \frac{f(t)}{2m}\Delta t^2 - \frac{\partial r(t)}{\partial t}\Delta t^3 + O(\Delta t^4)$$

故

$$r(t + \Delta t) + r(t - \Delta t) = 2r(t) + \frac{f(t)}{m} \Delta t^2 + O(\Delta t^4)$$
$$r(t + \Delta t) = 2r(t) - r(t - \Delta t) + \frac{f(t)}{m} \Delta t^2 + O(\Delta t^4)$$

速度只有二阶精度

$$v(t) = \frac{r(t + \triangle t) - r(t - \triangle t)}{2\triangle t} + O(\triangle t^2)$$

LeapFrog 速度和位置在时间上不同步

$$v(t - \triangle t/2) = \frac{r(t) - r(t - \triangle t)}{\triangle t}$$
$$v(t + \triangle t/2) = \frac{r(t + \triangle t) - r(t)}{\triangle t}$$
$$r(t + \triangle t) = r(t) + \triangle t v(t + \triangle t/2)$$
$$v(t + \triangle t) = v(t - \triangle t/2) + \triangle t \frac{f(t)}{m}$$

Velocity Verlet

$$v(t + \frac{1}{2}\triangle t) = v(t) + \frac{1}{2}\frac{f(t)}{m}\triangle t$$

$$r(t + \triangle t) = r(t) + v(t + \frac{1}{2}\triangle t)\triangle t$$

$$v(t + \triangle t) = v(t + \frac{\triangle t}{2}) + \frac{1}{2} \frac{f(t + \triangle t)}{m} \triangle t$$

$$r(t + \triangle t) = r(t) + v(t)\triangle t + \frac{1}{2} \frac{f(t)}{m} \triangle t^2$$

$$v(t + \triangle t) = v(t) + \frac{1}{2} \left(\frac{f(t)}{m} + \frac{f(t + \triangle t)}{m}\right) \triangle t$$

3 PBD方法

下述算法是对牛顿第二运动定律的忠实描述.

$$\label{eq:Algorithm 1} \begin{split} & \text{Algorithm 1 Cloth simulation} \\ \hline & 1: \ \mathbf{v}_0 \leftarrow \mathbf{0} \\ & 2: \ \text{for } t = 1 \ \text{to } n \ \text{do} \\ & 3: \qquad \mathbf{M}, \mathbf{f} \leftarrow \text{compute_forces}(\mathbf{x}, \mathbf{v}) \\ & 4: \quad \mathbf{a}_t \leftarrow \mathbf{M}^{-1} \mathbf{f} \\ & 5: \quad \mathbf{v}_t \leftarrow \mathbf{v}_{t-1} + \mathbf{a}_t \Delta t \\ & 6: \quad \mathbf{x}_t \leftarrow \mathbf{x}_{t-1} + \mathbf{v}_t \Delta t \\ & 7: \quad \mathbf{x}_t \leftarrow \mathbf{x}_t + \text{collision_response}(\mathbf{x}_t, \mathbf{v}_t, \mathbf{x}_t^{obs}, \mathbf{v}_t^{obs}) \\ & \mathbf{8:} \quad \mathbf{v}_t \leftarrow (\mathbf{x}_t - \mathbf{x}_{t-1}) / \Delta t \\ & 9: \ \text{end for} \end{split}$$

如果 Δt 过大,某时刻 $\mathbf{x}_t = \mathbf{x}_{t-1} + \mathbf{v}_t \Delta t$ 可能出现在真实世界中不可能出现的位置,从而让 $\mathbf{f}_t(\mathbf{x}_t)$ 很大, \mathbf{f}_t 又反过来影响 \mathbf{x}_{t+1} ,造成正反馈从而系统不稳定.

从以上分析可以发现,仿真不稳定的主要原因是,x出现在了某些不该出现的位置,如果为系统加入针对位置的约束 $C(x_{i_1},\ldots,x_{i_{n_j}})$ (如,针对衣物,限制衣物质点间距离不得小于 l_0 ,不得大于 l_1),则可以保证系统的稳定性.但是如果只是限制距离,无法保证系统动量和角动量的守恒,引入额外的ghost force.

PBD系列算法解决上文提到加入质点位置约束后,保证 动量角动量守恒的问题(个人理解).具体做法如下

We represent a dynamic object by a set of N vertices and M constraints. A vertex $i \in [1, ..., N]$ has a mass m_i , a position \mathbf{x}_i and a velocity \mathbf{v}_i .

A constraint $j \in [1, ..., M]$ consists of

- a cardinality n_j,
- a function $C_j: \mathbb{R}^{3n_j} \to \mathbb{R}$,
- a set of indices $\{i_1, \dots i_{n_j}\}, i_k \in [1, \dots N],$
- a stiffness parameter $k_j \in [0...1]$ and
- a type of either equality or inequality.

(1) forall vertices
$$i$$

(2) initialize $\mathbf{x}_i = \mathbf{x}_i^0, \mathbf{v}_i = \mathbf{v}_i^0, w_i = 1/m_i$
(3) endfor
(4) loop
(5) forall vertices i do $\mathbf{v}_i \leftarrow \mathbf{v}_i + \Delta t w_i \mathbf{f}_{ext}(\mathbf{x}_i)$
(6) dampVelocities $(\mathbf{v}_1, \dots, \mathbf{v}_N)$
(7) forall vertices i do $\mathbf{p}_i \leftarrow \mathbf{x}_i + \Delta t \mathbf{v}_i$
(8) forall vertices i do generateCollisionConstraints $(\mathbf{x}_i \rightarrow \mathbf{p}_i)$
(9) loop solverIterations times
(10) projectConstraints $(C_1, \dots, C_{M+M_{coll}}, \mathbf{p}_1, \dots, \mathbf{p}_N)$
(11) endloop
(12) forall vertices i
(13) $\mathbf{v}_i \leftarrow (\mathbf{p}_i - \mathbf{x}_i)/\Delta t$
(14) $\mathbf{x}_i \leftarrow \mathbf{p}_i$
(15) endfor
(16) velocityUpdate $(\mathbf{v}_1, \dots, \mathbf{v}_N)$
(17) endloop

解约束,已知p,求 Δp 使 $C(p + \Delta p) = 0$,当质点质量大小一致时,当 Δp 在 ∇C_p 方向上,下降最快.

$$\Delta \boldsymbol{p} = \lambda \nabla C_{\boldsymbol{p}}$$

又由

$$C(\boldsymbol{p} + \Delta \boldsymbol{p}) \approx C(\boldsymbol{p}) + \nabla_{\boldsymbol{p}} C(\boldsymbol{p}) \Delta \boldsymbol{p} = 0$$

得

$$\begin{split} \Delta \boldsymbol{p} &= -\frac{C(\boldsymbol{p})}{|\nabla_{\boldsymbol{p}} C(\boldsymbol{p})|^2} \nabla_{\boldsymbol{p}} C(\boldsymbol{p}) \\ \Delta \boldsymbol{p}_i &= -\frac{C(\boldsymbol{p})}{\sum_i |\nabla_{\boldsymbol{p}_i} C(\boldsymbol{p})|^2} \nabla_{\boldsymbol{p}_i} C(\boldsymbol{p}) \end{split}$$

具体求解策略流程上类似于Gauss-Seidel,对多个约束 C_i 逐个更新得 $\Delta p_i^{(k)}$ 后,求 $p=\Delta p_i^{(k)}+p$,直到满足 $\sum_i |C_i(p)|=0$,可以通过设置最大迭代次数保证实时性

$$egin{aligned} oldsymbol{p} &= oldsymbol{p}_0 \ & ext{while } \sum_i |C_i(oldsymbol{p})|! = 0 ext{ do} \ & ext{for } i = 0 \dots n ext{ do} \ & \Delta oldsymbol{p}_i &= -\frac{C_i(oldsymbol{p})}{|
abla_{oldsymbol{p}} C_i(oldsymbol{p})|^2}
abla_{oldsymbol{p}_i} C_i(oldsymbol{p}) \ & oldsymbol{p} &= oldsymbol{p} + \Delta oldsymbol{p}_i \ & ext{end for} \ & ext{end while} \end{aligned}$$

当质点大小不一样时,为了保重动量守恒, $\sum_i \boldsymbol{p}_i m_i = 0$,设 $w_i = 1/m_i$.

$$\Delta oldsymbol{p}_i = -rac{w_i C(oldsymbol{p})}{\sum_i w_j |
abla_{oldsymbol{p}_i} C(oldsymbol{p})|^2}
abla_{oldsymbol{p}_i} C(oldsymbol{p})$$

3.1 XPBD

PBD中两个关键参数对模拟的效果会产生直接影响,一个是时间步长 time step,另一个是解算迭代次数(solverIterations times). 具体直观表现是time step越小,迭代次数越大其模拟的直观表现为越硬,反之表现的越软绵绵的效果,这样的话,这样的话刚度系数意义不大,理想状态下刚性系数对魔力物体强度有直接最大影响,XPBD解决这个问题.

3.1.1 基本流程

Algorithm 1 XPBD simulation loop

1: predict position
$$\tilde{\mathbf{x}} \Leftarrow \mathbf{x}^n + \Delta t \mathbf{v}^n + \Delta t^2 \mathbf{M}^{-1} \mathbf{f}_{ext}(\mathbf{x}^n)$$
2:
3: initialize solve $\mathbf{x}_0 \Leftarrow \tilde{\mathbf{x}}$
4: initialize multipliers $\boldsymbol{\lambda}_0 \Leftarrow \mathbf{0}$
5: **while** $i < solver Iterations$ **do**
6: **for all** constraints **do**
7: compute $\Delta \lambda$ using Eq (18)
8: compute Δx using Eq (17)
9: update $\lambda_{i+1} \Leftarrow \lambda_i + \Delta \lambda$
10: update $\lambda_{i+1} \Leftarrow x_i + \Delta x$
11: **end for**
12: $i \Leftarrow i+1$
13: **end while**
14:
15: update positions $\mathbf{x}^{n+1} \Leftarrow \mathbf{x}_i$
16: update velocities $\mathbf{v}^{n+1} \Leftarrow \frac{1}{\Delta t} \left(\mathbf{x}^{n+1} - \mathbf{x}^n \right)$

$$\Delta \boldsymbol{x} = \boldsymbol{M}^{-1} \nabla \boldsymbol{C}(\boldsymbol{x}_i)^T \Delta \boldsymbol{\lambda} \quad (17)$$
$$\Delta \lambda_j = \frac{-C_j(\boldsymbol{x}_i) - \widehat{\alpha}_j \lambda_{ij}}{\nabla C_i \boldsymbol{M}^{-1} \nabla C_i^T + \widehat{\alpha}_j} \quad (18)$$

3.1.2 原理

$$egin{aligned} oldsymbol{M}rac{\partial^2 oldsymbol{x}}{\partial t^2} &= - riangle oldsymbol{U}^T(oldsymbol{x}) \ oldsymbol{U}(oldsymbol{x}) &= rac{1}{2} oldsymbol{C}(oldsymbol{x})^T oldsymbol{lpha}^{-1} oldsymbol{C}(oldsymbol{x}) \ oldsymbol{f}_{elastic} &= - riangle oldsymbol{U}^T(oldsymbol{x}) &= - riangle oldsymbol{C}^T oldsymbol{lpha}^{-1} oldsymbol{C} \end{aligned}$$

将运动方程对时间离散化

$$\boldsymbol{M}(\frac{\boldsymbol{x}^{i+1} - 2\boldsymbol{x}^i + \boldsymbol{x}^{i-1}}{\Delta t^2}) = -\nabla \boldsymbol{U}^T(\boldsymbol{x}^{i+1})$$

设

$$oldsymbol{\lambda}_{elastic} = -\widehat{oldsymbol{lpha}}^{-1} oldsymbol{C}, \quad \widehat{oldsymbol{lpha}} = rac{oldsymbol{lpha}}{\Delta t^2}$$

则运动方程可以写为方程组

$$egin{aligned} oldsymbol{M}(oldsymbol{x}^{i+1}-2oldsymbol{x}^i+oldsymbol{x}^{i-1}) - oldsymbol{
abla} oldsymbol{C}(oldsymbol{x}^{i+1})^Toldsymbol{\lambda}^{i+1} = oldsymbol{0} \ & oldsymbol{C}(oldsymbol{x}^{i+1}) + \widehat{oldsymbol{lpha}}oldsymbol{\lambda}^{i+1} = oldsymbol{0} \end{aligned}$$

记为

$$g(x, \lambda) = 0$$

$$h(x, \lambda) = 0$$

目标是求解,对于某一时刻

$$g(x_i, \lambda_i) \neq 0$$

$$h(x_i, \lambda_i) \neq 0$$

找到 Δx 和 $\Delta \lambda$, 令

$$q(x_i + \Delta x, \lambda_i + \Delta \lambda) = 0$$

$$h(x_i + \Delta x, \lambda_i + \Delta \lambda) = 0$$

然后更新得

$$\boldsymbol{x}_{i+1} = \boldsymbol{x}_i + \Delta \boldsymbol{x}$$

$$\lambda_{i+1} = \lambda_i + \Delta \lambda$$

线性化方程组

$$egin{pmatrix} rac{\partial oldsymbol{g}(oldsymbol{x}_i, oldsymbol{\lambda}_i)}{\partial oldsymbol{x}} & -oldsymbol{
abla} oldsymbol{C}^T(oldsymbol{x}_i) \ oldsymbol{
abla} egin{pmatrix} \Delta oldsymbol{x} \ \Delta oldsymbol{\lambda} \end{pmatrix} = -egin{pmatrix} oldsymbol{g}(oldsymbol{x}_i, oldsymbol{\lambda}_i) \ oldsymbol{h}(oldsymbol{x}_i, oldsymbol{\lambda}_i) \end{pmatrix}$$

为了简化运算,提出两个假设

• 为了避免求约束的Hessian矩阵, 假设

$$rac{\partial oldsymbol{g}(oldsymbol{x}_i,oldsymbol{\lambda}_i)}{\partial oldsymbol{x}}pprox oldsymbol{M}$$

省去了stiffness 和约束Hessian,这个优化会改变收敛rate,但不会改变最终的解

• 如果约束梯度变化不快, g会一直在一个比较小的值

$$g(x_i, \lambda_i) \approx 0$$

$$egin{pmatrix} m{M} & - egin{pmatrix} m{C}^T(m{x}_i) \ egin{pmatrix} \Delta m{x} \ \Delta m{\lambda} \end{pmatrix} = - egin{pmatrix} m{0} \ m{h}(m{x}_i, m{\lambda}_i) \end{pmatrix}$$

根据Schur Complement方法求解可以得

$$[\Delta C(x_i)M^{-1}\Delta C(x_i)^T + \widehat{\alpha}]\Delta \lambda = -C(x_i) - \widehat{\alpha}\lambda_i$$

$$\Delta \boldsymbol{x} = \boldsymbol{M}^{-1} \nabla \boldsymbol{C} (\boldsymbol{x}_i)^T \Delta \boldsymbol{\lambda} \quad (17)$$

对于 $\Delta\lambda$ 用Gauss-Seidel更新

$$\Delta \lambda_j = \frac{-C_j(\boldsymbol{x}_i) - \widehat{\alpha}_j \lambda_{ij}}{\nabla C_j \boldsymbol{M}^{-1} \nabla C_j^T + \widehat{\alpha}_j} \quad (18)$$

4 刚体仿真

4.1 刚体运动参数化表达

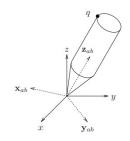
旋转矩阵 定义SO(n)为 \mathcal{R}^n 上的特殊正交群,SO(n)是以I为单位元素.矩阵乘法为群操作的群.

$$SO(n) = \{ \mathbf{R} \in \mathcal{R}^{n \times n} : \mathbf{R}\mathbf{R}^T = \mathbf{I}, det(\mathbf{R}) = \pm 1 \}$$

性质:

$$oldsymbol{R}(oldsymbol{v} imesoldsymbol{u}) = (oldsymbol{R}oldsymbol{v}) imes(oldsymbol{R}oldsymbol{w})$$

可以证明旋转是刚体运动



$$oldsymbol{R}_{ab} = egin{pmatrix} oldsymbol{x}_{ab} & oldsymbol{x}_{ab} \end{pmatrix}$$

se(3)**群的Exponential coordinates表达** 是旋转矩阵的一种参数化方法,参数空间是 ω , θ , 3维空间中,给定旋转轴 ω ,和旋转角度 θ , 求满足同样旋转的旋转矩阵 \mathbf{R} .

设旋转体上有点q,此时若以 ω 为单位旋转向量,则线速度

$$\frac{\partial \boldsymbol{q}(t)}{\partial t} = \boldsymbol{\omega} \times \boldsymbol{q}(t) = \widehat{\boldsymbol{\omega}} \boldsymbol{q}(t)$$

其中

$$\widehat{\boldsymbol{\omega}} = \begin{pmatrix} 0 & -a_3 & a_2 \\ a_3 & 0 & -a_1 \\ -a_2 & a_1 & 0 \end{pmatrix}$$

定义 $so(n) = \{ \mathbf{S} \in \mathcal{R}^{n \times n} : \mathbf{S}^T = -\mathbf{S}, \widehat{\boldsymbol{\omega}} \in so(3)$ 则微分方程解为

$$\boldsymbol{q}(t) = \boldsymbol{e}^{\widehat{\boldsymbol{\omega}}t}\boldsymbol{q}(0)$$

如果以 ω 为轴以单位角速度旋转 θ 时间,则

$$\widehat{m{\omega}}^2 = m{\omega} m{\omega}^T - ||m{\omega}||^2 m{I}$$

$$\widehat{oldsymbol{\omega}}^3 = -||oldsymbol{\omega}||^2 \widehat{oldsymbol{\omega}}$$

$$R(\omega, \theta) = e^{\widehat{\omega}\theta} = I + \theta \widehat{\omega} + \frac{\theta^2}{2!} \widehat{\omega}^2 + \frac{\theta^3}{3!} \widehat{\omega}^3 \dots$$

$$= I + (\theta - \frac{\theta^3}{3!} + \frac{\theta^5}{5!} + \dots) \widehat{\omega} + (\frac{\theta^2}{2!} - \frac{\theta^4}{4!} + \frac{\theta^6}{6!} + \dots) \omega^2$$

$$= I + \widehat{\omega} \sin \theta + \widehat{\omega}^2 (1 - \cos \theta)$$

己知旋转矩阵,求其exponential coordinates表达

$$\theta = \cos^{-1}(\frac{trace(\mathbf{R}) - 1}{2})$$

$$\boldsymbol{\omega} = \frac{1}{\sin \theta} \begin{pmatrix} r_{32} - r_{23} \\ r_{13} - r_{31} \\ r_{21} - r_{12} \end{pmatrix}$$

刚体 是点的集合,无论何种力或运动作用,每个点之间的距离都是固定的.

刚体运动 一个映射 $g: \mathcal{R}^3 \to \mathcal{R}^3$ 当满足如下两个条件时是 **4.2** 刚体运动:

• 对于所有 $p, q \in \mathcal{R}^3$

$$||g(p) - g(q)|| = ||p - q||$$

• 设 $\mathbf{v} = \mathbf{q} - \mathbf{p}, g_*(\mathbf{v}) = g(\mathbf{q}) - g(\mathbf{p}),$ 则对于所有 $\mathbf{v}, \mathbf{w} \in \mathbb{R}^3$.

$$g_*(\boldsymbol{v} \times \boldsymbol{w}) = g_*(\boldsymbol{v}) \times g_*(\boldsymbol{w})$$

定义特殊欧拉群

$$SE(n) = \{(t, \mathbf{R}) : t \in \mathcal{R}^n, \mathbf{R} \in SO(n)\} = \mathcal{R}^n \times SO(3)$$

设 $g_{ab}=(\boldsymbol{t}_{ab},\boldsymbol{R}_{ab}),$ 则

$$\mathbf{q}_a = g_{ab}(\mathbf{q}_b) = \mathbf{t}_{ab} + \mathbf{R}_{ab}\mathbf{q}_b$$

可以证明 $g \in SE(n)$ 是刚体运动.

$$\mathbf{q}_b = g_{ab}^{-1}(\mathbf{q}_a) = \mathbf{R}_{ab}^T \mathbf{q}_a - \mathbf{R}_{ab}^T \mathbf{t}_{ab}$$

SE(3)群的齐次坐标系表达 如果 $g = (t, \mathbf{R}) \in SE(3)$

$$g = \begin{pmatrix} \mathbf{R} & \mathbf{t} \\ 0 & 1 \end{pmatrix}$$

SE(3)群的exponential coordinates表达 设p是刚体上的某个点,绕轴 ω 以单位角速度旋转,q是旋转轴上的某点.

$$\frac{\partial \boldsymbol{p}(t)}{\partial t} = \boldsymbol{\omega} \times (\boldsymbol{p}(t) - \boldsymbol{q}) \Longrightarrow \frac{\partial \overline{\boldsymbol{p}}(t)}{\partial t} = \begin{pmatrix} \widehat{\boldsymbol{\omega}} & -\boldsymbol{w} \times \boldsymbol{q} \\ 0 & 1 \end{pmatrix} \overline{\boldsymbol{p}} = \widehat{\boldsymbol{\varepsilon}} \overline{\boldsymbol{p}}$$

$$\overline{\boldsymbol{p}}(t) = \boldsymbol{e}^{\widehat{\boldsymbol{\varepsilon}}t}\overline{\boldsymbol{p}}(0)$$

设 $v=-w\times q$, 定义, $se(n)=\{(v,\widehat{\omega}),v\in\mathcal{R}^n,\widehat{\omega}\in so(n)\}$,在齐次空间 $\widehat{\epsilon}\in so(3)$.

$$\widehat{\boldsymbol{\varepsilon}} = \begin{pmatrix} \widehat{\boldsymbol{\omega}} & \boldsymbol{v} \\ 0 & 0 \end{pmatrix}$$

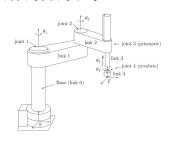
定义 $\varepsilon = (v, \omega) \in \mathcal{R}^6$ 是twist $\widehat{\varepsilon}$ 的twist 坐标. 对于 $\widehat{\epsilon} \in se(3), \theta \in \mathcal{R}$ 有

$$e^{\widehat{\boldsymbol{\varepsilon}}\boldsymbol{\theta}} = \begin{pmatrix} e^{\widehat{\boldsymbol{\omega}}\boldsymbol{\theta}} & (\boldsymbol{I} - e^{\widehat{\boldsymbol{\omega}}\boldsymbol{\theta}})(\boldsymbol{\omega} \times \boldsymbol{v}) + \boldsymbol{\omega}\boldsymbol{\omega}^T\boldsymbol{v}\boldsymbol{\theta} \\ 0 & 1 \end{pmatrix} \in SE(3)$$

SE(3)群的screw表达

4.2 刚体速度的表示

4.3 作用于刚体的力的表示



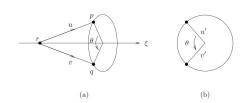
4.4 Forward Kinematics

Forward Kinematic解决的问题是, 给定各个关节之间相对configuration,求end effector在base frame S下的相应位置.

4.5 Inverse Kinematics

给定tool frame *T*期望的位置, 求各个关节之间相对的configuration.可以通过Paden Kahan子问题方法解决.

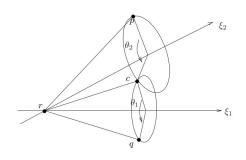
子问题1



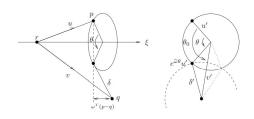
假设 ϵ 不产生平移,求 θ 令

$$e^{\widehat{oldsymbol{arepsilon}} heta}p=q$$

子问题2



子问题3



使用子问题解决总问题 若p在 ϵ 的轴上,则有

$$e^{\widehat{oldsymbol{arepsilon}} heta}oldsymbol{p}=oldsymbol{p}$$

假设已知 $g, \varepsilon_1, \varepsilon_2, \varepsilon_3$ 求 $\theta_1, \theta_2, \theta_3$

$$e^{\widehat{oldsymbol{arepsilon}}_1 heta_1}e^{\widehat{oldsymbol{arepsilon}}_2 heta_2}e^{\widehat{oldsymbol{arepsilon}}_3 heta_3}=oldsymbol{q}$$

设p在 ϵ 轴上则, 在两边同时乘上p

$$e^{\widehat{oldsymbol{arepsilon}}_1 heta_1}e^{\widehat{oldsymbol{arepsilon}}_2 heta_2}e^{\widehat{oldsymbol{arepsilon}}_3 heta_3}p=gp$$

$$e^{\widehat{oldsymbol{arepsilon}}_1 heta_1}e^{\widehat{oldsymbol{arepsilon}}_2 heta_2}p=gp$$

此时变成子问题2,可以求解出 θ_1 , θ_2 ,对于 θ_3 ,可以假设 ϵ_1 , ϵ_2 交于p.

$$egin{align} \delta = &||gm{p}-m{q}|| = ||e^{\widehat{m{arepsilon}}_1 heta_1}e^{\widehat{m{arepsilon}}_2 heta_2}e^{\widehat{m{arepsilon}}_3 heta_3}m{p}-m{q}|| \ = &||e^{\widehat{m{arepsilon}}_1 heta_1}e^{\widehat{m{arepsilon}}_2 heta_2}(e^{\widehat{m{arepsilon}}_3 heta_3}m{p}-m{q})|| \ = &||e^{\widehat{m{arepsilon}}_3 heta_3}m{p}-m{q}|| \ \end{aligned}$$

4.6 刚体动力学

刚体系统的运动方程可以描述为

$$oldsymbol{ au} = oldsymbol{H}(q) rac{\partial^2 oldsymbol{q}}{\partial t^2} + oldsymbol{C}(oldsymbol{q}, rac{\partial oldsymbol{q}}{\partial t})$$

其中, τ 是施加力,H是描述inertia term的矩阵,C是外力?.

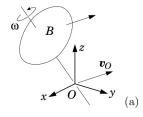
对于Forward Dynamic 和 Inverse Dynamic问题分别可以描述为

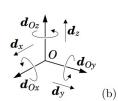
$$\frac{\partial^2 \boldsymbol{q}}{\partial t^2} = FD(model, \boldsymbol{q}, \frac{\partial \boldsymbol{q}}{\partial t}, \boldsymbol{\tau})$$

$$\boldsymbol{\tau} = ID(model, \boldsymbol{q}, \frac{\partial \boldsymbol{q}}{\partial t}, \frac{\partial^2 \boldsymbol{q}}{\partial t^2})$$

4.6.1 Spatial Vector Algebra

Plucker Basis on \mathcal{M}^6





在欧式空间中表达速度,

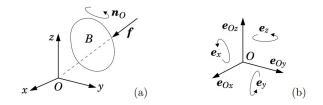
$$v_n = v_o + \omega \times \overrightarrow{OP}$$

用Plucker Basis表达

$$\boldsymbol{v} = \omega_x \boldsymbol{d}_{Ox} + \omega_y \boldsymbol{d}_{Oy} + \omega_z \boldsymbol{d}_{Oz} + v_{Ox} \boldsymbol{d}_x + v_{Oy} \boldsymbol{d}_y + v_{Oz} \boldsymbol{d}_z$$

$$\widehat{\underline{\boldsymbol{v}}}_{O} = \begin{pmatrix} \omega_{x} \\ \omega_{y} \\ \omega_{z} \\ v_{Ox} \\ v_{Oy} \\ v_{Oz} \end{pmatrix} = \begin{pmatrix} \underline{\boldsymbol{\omega}} \\ \underline{\boldsymbol{v}}_{O} \end{pmatrix}$$

Plucker Basis on \mathcal{F}^6



在欧式空间中表达动量

$$n_P = n_O + f \times \overrightarrow{OP}$$

在Plucker Basis中表达

$$\widehat{\mathbf{f}} = n_{Ox}\mathbf{e}_x + n_{Oy}\mathbf{e}_y + n_{Oz}\mathbf{e}_z + f_x\mathbf{e}_{Ox}f_y\mathbf{e}_{Oy}f_z\mathbf{e}_{Oz}$$

$$\widehat{\underline{v}}_O = egin{pmatrix} n_{Ox} \\ n_{Oy} \\ n_{Oz} \\ f_x \\ f_y \\ f_z \end{pmatrix} = egin{pmatrix} \underline{n}_O \\ \underline{\underline{f}} \end{pmatrix}$$

5 弹性体仿真

5.1 弹性体模型

设X,x分别是弹性体形变前和形变后的位置. 定义deformation function $\phi: \mathcal{R}^3 \to \mathcal{R}^3$, 则

$$\boldsymbol{x} = \phi(\boldsymbol{X})$$

定义deformation gradient tensor, $F \in \mathbb{R}^{3\times3}$

$$\boldsymbol{F} = \frac{\partial(\phi_1, \phi_2, \phi_3)}{\partial(\boldsymbol{X}_1, \boldsymbol{X}_2, \boldsymbol{X}_3)} = \begin{pmatrix} \frac{\partial \phi_1}{\partial \boldsymbol{X}_1} & \frac{\partial \phi_1}{\partial \boldsymbol{X}_2} & \frac{\partial \phi_1}{\partial \boldsymbol{X}_3} \\ \frac{\partial \phi_2}{\partial \boldsymbol{X}_1} & \frac{\partial \phi_2}{\partial \boldsymbol{X}_2} & \frac{\partial \phi_2}{\partial \boldsymbol{X}_3} \\ \frac{\partial \phi_3}{\partial \boldsymbol{X}_1} & \frac{\partial \phi_3}{\partial \boldsymbol{X}_2} & \frac{\partial \phi_3}{\partial \boldsymbol{X}_3} \end{pmatrix}$$

Strain energy 定义为 $\Phi(\mathbf{F})$,

定义force density为 $f(\boldsymbol{X})$ 描述force per unit undeformed volume, 定义 traction $\tau(\boldsymbol{X})$ 为force density function 测量force per unit undeformed area. 若 $A \in \Omega, B \in \partial \Omega$

$$f_{aggregate(A)} = \int_{A} f(\boldsymbol{X}) d\boldsymbol{X}$$

$$f_{aggregate(B)} = \int_{B} \tau(\mathbf{X}) dS$$

1st Piola-Kirchhoff stress tensor, $P \in \mathbb{R}^{3\times 3}$

$$\tau(\boldsymbol{X}) = -\boldsymbol{P}N$$

N是在reference configuration 下, 边界上朝外的法向量.

$$f(X) = div_X P$$

$$f_i = \sum_{j=1}^{3} P_{ij,j} = \frac{\partial P_{i1}}{\partial X_1} + \frac{\partial P_{i2}}{\partial X_2} + \frac{\partial P_{i3}}{\partial X_3}$$

对于超弹性材料

$$m{P}(m{F}) = rac{\partial \Psi(m{F})}{\partial m{F}}$$

Strain Measure Green strain tensor, $E \in \mathbb{R}^{3\times 3}$.

$$\boldsymbol{E} = \frac{1}{2} (\boldsymbol{F}^T \boldsymbol{F} - \boldsymbol{I})$$

$$m{E}(m{F}) = \underbrace{m{E}(m{I})}_{-0} + \frac{\partial m{E}}{\partial m{F}}|_{m{F}=m{I}} : (m{F} - m{I})$$

由

$$\frac{\partial \boldsymbol{E}}{\partial \boldsymbol{F}} : \delta \boldsymbol{F} = \delta \boldsymbol{E} = \frac{1}{2} (\delta \boldsymbol{F}^T \boldsymbol{F} + \boldsymbol{F}^T \delta \boldsymbol{F})$$

得

$$\frac{\partial \boldsymbol{E}}{\partial \boldsymbol{F}}: (\boldsymbol{F} - \boldsymbol{I}) = \frac{1}{2}[(\boldsymbol{F} - \boldsymbol{I})^T \boldsymbol{I} + \boldsymbol{I}^T (\boldsymbol{F} - \boldsymbol{I})] = \frac{1}{2}(\boldsymbol{F} + \boldsymbol{F}^T) - \boldsymbol{I}$$

记 $\epsilon = E(F)$, ϵ 称为infinitesimal strain tensor 或 small strain tensor. 化非线性为线性, 在计算上更为快速.

$$oldsymbol{\epsilon} = rac{1}{2}(oldsymbol{F} + oldsymbol{F}^T) - oldsymbol{I}$$

Linear elasticity

$$\Psi(\mathbf{F}) = \mu \mathbf{\epsilon} : \mathbf{\epsilon} + \frac{\lambda}{2} tr^2(\mathbf{\epsilon})$$

$$\mu = \frac{k}{2(1+\nu)}, \quad \lambda = \frac{k\nu}{(1+\nu)(1-2\nu)}$$

其中 μ , λ 是Lame coefficient, k是Young's modulus(用于measure stretch of resistance), ν 是 Poisson's ratio(a measure of incompressibility).

$$\begin{split} \delta\Psi &= 2\mu\boldsymbol{\epsilon}: \delta\boldsymbol{\epsilon} + \lambda tr(\boldsymbol{\epsilon})tr(\delta\boldsymbol{\epsilon}) = \underbrace{\left[2\mu\boldsymbol{\epsilon} + \lambda tr(\boldsymbol{\epsilon})\boldsymbol{I}\right]}_{=\partial\Psi/\partial\boldsymbol{F}}: \delta\boldsymbol{F} \\ \boldsymbol{\epsilon}: \delta\boldsymbol{\epsilon} &= \boldsymbol{\epsilon}: Sym\{\delta\boldsymbol{F}\} = \boldsymbol{\epsilon}: \delta\boldsymbol{F} \end{split}$$

$$tr(\delta \epsilon) = \boldsymbol{I} : Sym\{\delta \boldsymbol{F}\} = \boldsymbol{I} : \delta \boldsymbol{F}$$

$$\delta \boldsymbol{\epsilon} = \frac{1}{2} (\delta \boldsymbol{F} + \delta \boldsymbol{F}^T) = Sym\{\delta \boldsymbol{F}\}$$

所以

$$\begin{split} \boldsymbol{P} = & 2\mu\boldsymbol{\epsilon} + \lambda tr(\boldsymbol{\epsilon})\boldsymbol{I} \\ = & \mu(\boldsymbol{F} + \boldsymbol{F}^T - 2\boldsymbol{I}) + \lambda tr(\boldsymbol{F} - \boldsymbol{I})\boldsymbol{I} \end{split}$$

P是关于**F**的线性方程,在实现时计算速度可以很快, 此外small strain tensor只适用于小的形变场景,不能保证rotational invariance.

StVK模型 性质: 保证rotational invariance, 但是poor resisitance to forceful compression(StVK elastic body 是compressed的).

$$\begin{split} \Psi(\boldsymbol{F}) &= \mu \boldsymbol{E} : \boldsymbol{E} + \frac{\lambda}{t} r^2(\boldsymbol{E}) \\ \delta \boldsymbol{E} &= \frac{1}{2} (\delta \boldsymbol{F}^T \boldsymbol{F} + \boldsymbol{F}^T \delta \boldsymbol{F}) = Sym \{ \boldsymbol{F}^T \delta \boldsymbol{F} \} \\ \boldsymbol{E} : \delta \boldsymbol{E} &= \boldsymbol{E} : \{ \boldsymbol{F}^T \delta \boldsymbol{F} \} = \{ \boldsymbol{F} \boldsymbol{E} \} : \delta \boldsymbol{F} \\ tr(\delta \boldsymbol{E}) &= \boldsymbol{I} : \{ \boldsymbol{F}^T \delta \boldsymbol{F} \} = \boldsymbol{F} : \delta \boldsymbol{F} \\ \delta \Psi &= 2\mu \boldsymbol{E} : \delta \boldsymbol{E} + \lambda tr(\boldsymbol{E}) tr(\delta \boldsymbol{E}) = \boldsymbol{F} [2\mu \boldsymbol{E} + \lambda tr(\boldsymbol{E}) \boldsymbol{I}] : \delta \boldsymbol{F} \\ \boldsymbol{P}(\boldsymbol{F}) &= \boldsymbol{F} [2\mu \boldsymbol{E} + \lambda tr(\boldsymbol{E}) \boldsymbol{I}] \end{split}$$

Corotated linear elasticity 结合linear material 和一些 非线性特性保证rotational invariance

$$\begin{aligned} \boldsymbol{F} &= \boldsymbol{R}\boldsymbol{S} \quad \boldsymbol{\epsilon} = \boldsymbol{S} - \boldsymbol{I} \\ \Psi(\boldsymbol{F}) &= \mu \boldsymbol{\epsilon}_C : \boldsymbol{\epsilon}_C + \frac{\lambda}{2}tr^2(\boldsymbol{\epsilon}_C) = \mu||\boldsymbol{S} - \boldsymbol{I}||_F^2 + \frac{\lambda}{2}tr^2(\boldsymbol{S} - \boldsymbol{I}) \\ \boldsymbol{P}(\boldsymbol{F}) &= \boldsymbol{R}[2\mu\boldsymbol{\epsilon}_C + \lambda tr(\boldsymbol{\epsilon}_C)\boldsymbol{I}] \\ &= \boldsymbol{R}[2\mu(\boldsymbol{S} - \boldsymbol{I}) + \lambda tr(\boldsymbol{S} - \boldsymbol{I})\boldsymbol{I}] \\ &= 2\mu(\boldsymbol{F} - \boldsymbol{R}) + \lambda tr(\boldsymbol{R}^T\boldsymbol{F} - \boldsymbol{I})\boldsymbol{R} \end{aligned}$$

Isotropic material and invariants

Neohookean模型 Neo-Hookean是最常用的非线性超弹性 模型

$$\psi(\mathbf{F}) = \frac{\mu}{2} (tr(\mathbf{F}^T \mathbf{F}) - d) - \mu \log(\mathbf{J}) + \frac{\lambda}{2} \log^2(\mathbf{J})$$

d=2,3取决于物体维度, μ,λ 与Young's modulusE和Poisson ratio的关系是

$$\begin{split} \mu &= \frac{E}{2(1+v)}, \lambda = \frac{Ev}{(1+v)(1-2v)} \\ \boldsymbol{P} &= \frac{\partial \psi}{\partial \boldsymbol{F}} = \mu(\boldsymbol{F})(\boldsymbol{F} - \boldsymbol{F}^{-T}) + \lambda(\boldsymbol{F})\log(\boldsymbol{J}\boldsymbol{F}^{-T}) \end{split}$$

Fixed Corotated Constitutive模型 简单且广泛使用,taichi语言中很多例子使用这种模型,假设polar SVD

$$F = U\Sigma V^{T}$$

$$\psi(F) = \hat{\psi}(\Sigma) = \mu \sum_{i=1}^{d} (\sigma_{i} - 1)^{2} + \frac{\lambda}{2}(J - 1)$$

$$J = \prod_{i=1}^{d} \sigma_{i}$$

$$m{P}(m{F}) = rac{\partial \psi}{\partial m{F}}(m{F}) = 2\mu(m{F})(m{F} - m{R}) + \lambda(m{F})(J-1)Jm{F}^{-T}$$
 综上、计算 $m{P}$ 的流程

$$m{F} = m{U}m{\Sigma}m{V}^T$$
 $\psi(m{F}) = \hat{\psi}(m{\Sigma})$ $m{R} = m{R}m{V}^T$ $m{P} = m{U}\hat{m{P}}m{V}^T = m{U}diag(rac{\hat{\psi}}{\partial \sigma_0}, \dots, rac{\hat{\psi}}{\partial \sigma_i}, \dots)m{V}^T$

5.2 FEM方法

在有限的顶点 X_1, \ldots, X_N 上储存deformation map, $\Phi(X), x_i = \phi(X_i), \ \exists x = (x_1, \ldots, x_N), \$ 对于任意给定的deformation $\phi(x)$ 的starin energy

$$E(\boldsymbol{\phi}) = \int_{\Omega} \Psi(\boldsymbol{F}) d\boldsymbol{X}$$

对于离散情况,如利用四面体离散出来的弹性体,通过插值得出X对应的x,进而求出 $\widehat{\Phi}(X,x)$

$$E(\boldsymbol{x}) = E[\widehat{\Phi}(\boldsymbol{X}, \boldsymbol{x})] = \int_{\Omega} \Psi(\widehat{\boldsymbol{F}}(\boldsymbol{X}, \boldsymbol{x})) d\boldsymbol{X}$$

其中 $\widehat{F}(X, x) = \partial \widehat{\phi}(X, x) / \partial X$

Mesh上每个点的受力为

$$f_i(x_0) = \frac{\partial E(x)}{\partial x_i}$$

实际计算中, 对于每个elements, Ω_e

$$E(\boldsymbol{x}) = \sum_{e} E^{e}(\boldsymbol{x}) = \sum_{e} \int_{\Omega_{e}} \Psi(\widehat{\boldsymbol{F}}(\boldsymbol{X}, \boldsymbol{x}) d\boldsymbol{X}$$

每个点上的力 f_i 可以通过all elements in neighborhood \mathcal{N}_i 计算.

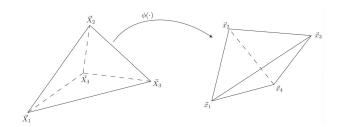
$$m{f}_i(m{x}_0) = \sum_{m{x} \in \mathcal{N}_i} m{f}_i^e(m{x}), \quad m{f}_i^e(m{x}) = -rac{\partial E^e(m{x})}{\partial m{x}_i}$$

对于任意一个四面体 T_i

$$\widehat{\phi}(X) = A_i X + b, X \in \mathcal{T}_i$$

其中 $\mathbf{A}_i \in \mathcal{R}^{3\times 3}$ 且 $\mathbf{b} \in \mathcal{R}^3$,由 $\mathbf{F} = \partial \widehat{\phi} / \partial \mathbf{X} = \mathbf{A}_i$.

$$\widehat{\phi}(X) = FX + b, X \in \mathcal{T}_i$$



$$egin{cases} x_1 = FX_1 + b \ x_2 = FX_2 + b \ x_3 = FX_3 + b \ x_4 = FX_4 + b \end{cases} \Rightarrow egin{cases} x_1 - x_4 = F(X_1 - X_4) \ x_2 - x_4 = F(X_2 - X_4) \ x_3 - x_4 = F(X_3 - X_4) \end{cases}$$

$$=oldsymbol{F}oldsymbol{X}_4+oldsymbol{b}$$
 $oldsymbol{D}_s=oldsymbol{F}oldsymbol{D}_m$ $oldsymbol{D}_s=egin{pmatrix}oldsymbol{x}_1-oldsymbol{x}_4 & oldsymbol{x}_2-oldsymbol{x}_4 & oldsymbol{x}_3-oldsymbol{x}_4\end{pmatrix}$ $oldsymbol{D}_m=egin{pmatrix}oldsymbol{X}_1-oldsymbol{X}_4 & oldsymbol{X}_2-oldsymbol{X}_4 & oldsymbol{X}_3-oldsymbol{X}_4\end{pmatrix}$

四面体的undeformed volume等于 $W = \frac{1}{6} |det \mathbf{D}_m|$

$$m{E}_i = \int_{\mathcal{T}_i} \Psi(m{F}) dm{X} = \Psi(m{F}_i) \int_{\mathcal{T}_i} dm{X} = W \cdot \Psi(m{F}_i)$$

对于四个点上的力 $\mathbf{f}_{ik} = -\partial E_i(\mathbf{x})/\partial \mathbf{x}_k$.

$$oldsymbol{H} = egin{pmatrix} oldsymbol{f}_1 & oldsymbol{f}_2 & oldsymbol{f}_3 \end{pmatrix} = -oldsymbol{W}oldsymbol{P}(oldsymbol{F})oldsymbol{D}_m^{-T}, \quad oldsymbol{f}_4 = -oldsymbol{f}_1 - oldsymbol{f}_2 - oldsymbol{f}_3 \end{pmatrix}$$

计算tetrahedral mesh 上所有顶点的力

Algorithm 1 Batch computation of elastic forces on a tetrahedral mesh

1: procedure PRECOMPUTATION($\mathbf{x}, \mathbf{B}_m[1 \dots M], W[1 \dots M]$)

2: for each $\mathcal{T}_e = (i, j, k, l) \in \mathcal{M}$ do $\triangleright \mathcal{M}$ is the number of tetrahedra

3: $\mathbf{D}_m \leftarrow \begin{bmatrix} X_i - X_l & X_j - X_l & X_k - X_l \\ Y_i - Y_l & Y_j - Y_l & Y_k - Y_l \\ Z_i - Z_l & Z_j - Z_l & Z_k - Z_l \end{bmatrix}$ 4: $\mathbf{B}_m[e] \leftarrow \mathbf{D}_m^{-1}$ 5: $W[e] \leftarrow \frac{1}{6} \det(\mathbf{D}_m)$ $\triangleright W$ is the undeformed volume of \mathcal{T}_e 6: end for

7: end procedure

8: procedure COMPUTEELASTICFORCES($\mathbf{x}, \mathbf{f}, \mathcal{M}, \mathbf{B}_m[], W[]$)

9: $\mathbf{f} \leftarrow \mathbf{0}$ $\triangleright \mathcal{M}$ is a tetrahedral mesh

10: for each $\mathcal{T}_e = (i, j, k, l) \in \mathcal{M}$ do

11: $\mathbf{D}_s \leftarrow \begin{bmatrix} x_i - x_l & x_j - x_l & x_k - x_l \\ y_i - y_l & y_j - y_l & y_k - y_l \\ z_i - z_l & z_j - z_l & z_k - z_l \end{bmatrix}$ 12: $\mathbf{F} \leftarrow \mathbf{D}_s \mathbf{B}_m[e]$ 13: $\mathbf{P} \leftarrow \mathbf{P}(\mathbf{F})$ \triangleright From the constitutive law

14: $\mathbf{H} \leftarrow -W[e]\mathbf{P}(\mathbf{B}_m[e])^T$ 15: $\vec{f}_i + = \vec{h}_1, \vec{f}_j + = \vec{h}_2, \vec{f}_k + = \vec{h}_3$ $\triangleright \mathbf{H} = \begin{bmatrix} \vec{h}_1 \ \vec{h}_2 \ \vec{h}_3 \end{bmatrix}$ 16: $\vec{f}_i + = (-\vec{h}_1 - \vec{h}_2 - \vec{h}_3)$ 17: end for

18: end procedure

显式欧拉算法即可进行仿真过程,定义 $f_e(x^*)$ 是 x^* 上的 弹力, $K(x^*) = -\frac{\partial f_e}{\partial x}|_{x^*}$ 是elasticity stiffness matrix, $f_d(x^*, v^*) = -\gamma K(x^*)v^*$ Damping forces, $f(x^*, v^*) = f_e(x^*) + f_d(x^*, v^*)$ 是 aggregate forces. M是mass matrix, x^{n+1}, v^{n+1} 是 t^{n+1} 时刻的位置和速度.

$$\boldsymbol{x}^{n+1} = \boldsymbol{x}^n + \triangle t \boldsymbol{v}^{n+1}$$

$$egin{aligned} m{v}^{n+1} = & m{v}^n + \triangle t m{M}^{-1} m{f}(m{x}^{n+1}, m{v}^{n+1}) \ = & m{v}^n + \triangle t m{M}^{-1} (m{f}_e(m{x}^{n+1}) + m{f}_d(m{x}^{n+1}, m{v}^{n+1})) \end{aligned}$$

由于上述Backward Euler System 是非线性的, 所以用一个 序列 $x_{(k)}^{n+1}, v_{(k)}^{n+1}$ 趋近 x^{n+1}, v^{n+1} .

5.3 MPM方法

MPM(Material Point Method)方法最开始是针对各向 同性体弹性物体(如:流体,果冻,雪等)的仿真方法.

针对各向同性体弹性物体仿真最朴素的观点就是将物 体视作很多个弹性"粒子"构成.粒子间相互作用力决定了物 体运动(拉格朗日观点). 描述成粒子可以方便的表达...但存 在.. 的问题.

5.3.1APIC仿真框架

插值函数定义及其性质

$$w_{ip}^n = N_i(\boldsymbol{x}_p^n)$$

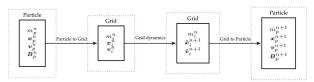
$$N_i(\boldsymbol{x}_p) = N(\frac{1}{h}(x_p - x_i))N(\frac{1}{h}(y_p - y_i))N(\frac{1}{h}(z_p - z_i))$$

常用的插值函数有quadratic B splines或cubic B splines(分别如下),后者比前者计算开销大但是数值更稳定.

$$\begin{split} \mathsf{N}(\mathbf{x}) = \begin{cases} \frac{1}{2}|\mathbf{x}|^3 - |\mathbf{x}|^2 + \frac{2}{3} & 0 \leqslant |\mathbf{x}| < 1 \\ \frac{1}{6}(2 - |\mathbf{x}|)^3 & 1 \leqslant |\mathbf{x}| < 2 & \mathsf{N}(\mathbf{x}) = \begin{cases} \frac{3}{4} - |\mathbf{x}|^2 & 0 \leqslant |\mathbf{x}| < \frac{1}{2} \\ \frac{1}{2}(\frac{3}{2} - |\mathbf{x}|)^2 & \frac{1}{2} \leqslant |\mathbf{x}| < \frac{3}{2} \\ 0 & \frac{3}{2} \leqslant |\mathbf{x}| \end{cases} \\ & \sum_{i} w_{ip}^n = 1 \\ & \sum_{i} w_{ip}^n \boldsymbol{x}_i^n = \boldsymbol{x}_p^n \end{split}$$

$$\sum_{i} w_{ip}^{n} (\boldsymbol{x}_{i}^{n} - \boldsymbol{x}_{p}^{n}) = 0$$

APIC的算法更新迭代流程,可证明这个迭代过程中可以 保证G2P和P2G的动量以及角动量守恒.



从particles到grid

$$\begin{split} m_i^n &= \sum_p w_{ip}^n m_p \\ \boldsymbol{D}_p^n &= \sum_i w_{ip}^n (\boldsymbol{x}_i^n - \boldsymbol{x}_p^n) (\boldsymbol{x}_i^n - \boldsymbol{x}_p^n)^T = \sum_i w_{ip}^n \boldsymbol{x}_i^n (\boldsymbol{x}_i^n)^T - \boldsymbol{x}_p^n (\boldsymbol{x}_p^n)^T \\ m_i^n \boldsymbol{v}_i^n &= \sum_p w_{ip}^n m_p (\boldsymbol{v}_p^n + \boldsymbol{B}_p^n (\boldsymbol{D}_p^n)^{-1} (\boldsymbol{x}_i^n - \boldsymbol{x}_p^n)) \end{split}$$

从grid到particle

$$\begin{split} \boldsymbol{v}_p^{n+1} &= \sum_i w_{ip}^n \tilde{\boldsymbol{v}}_i^{n+1} \\ \boldsymbol{B}_p^{n+1} &= \sum_i w_{ip}^n \tilde{\boldsymbol{v}}_i^{n+1} (\boldsymbol{x}_i^n - \boldsymbol{x}_p^n)^T. \end{split}$$

在quadratic 情况下

$$\boldsymbol{D}_p^n = \frac{1}{4} \triangle x^2 \boldsymbol{I}$$

在cubic 情况下

$$\boldsymbol{D}_p^n = \frac{1}{3} \triangle x^2 \boldsymbol{I}$$

 $\triangle x$ 是网格间距距离(从代码Demo中习得,需要考证3维下是 否一致)

故APIC流程伪代码

- 1 Particle to grid (P2G)
 - $(m\mathbf{v})_i^{n+1} = \sum_p w_{ip}[m_p\mathbf{v}_p^n + m_p\mathbf{C}_p^n(\mathbf{x}_i \mathbf{x}_p^n)]$ (Grid momentum) • $m_i^{n+1} = \sum_p m_p w_{ip}$ (Grid mass)
- - $\hat{\mathbf{v}}_i^{n+1} = (m\mathbf{v})_i^{n+1}/m_i^{n+1}$ (Grid velocity)
 - Apply Chorin-style pressure projection: $\mathbf{v}^{n+1} = \mathbf{Project}(\hat{\mathbf{v}}^{n+1})$
- 3 Grid to particle (G2P)
 - $\mathbf{v}_p^{n+1} = \sum_i w_{ip} \mathbf{v}_i^{n+1}$ (Particle velocity)
 - $\mathbf{C}_p^{n+1} = \frac{4}{\Delta x^2} \sum_i w_{ip} \mathbf{v}_i^{n+1} (\mathbf{x}_i \mathbf{x}_p^n)^T$ (Particle velocity gradient) $\mathbf{x}_p^{n+1} = \mathbf{x}_p^n + \Delta t \mathbf{v}_p^{n+1}$ (Particle position)

上图写漏了Deformation Gradient的更新

$$oldsymbol{F}_p^{n+1} = (oldsymbol{I} + riangle t \sum_i oldsymbol{v}_i^{n+1} (riangle w_{ip}^n)^T) oldsymbol{F}_p^n$$

在Grid Operation中,要通过力来更新速度

$$egin{aligned} oldsymbol{v}_i^{n+1} &= \hat{oldsymbol{v}}_i^{n+1} - rac{oldsymbol{f}_i}{m_i} \ oldsymbol{f}_i &= -rac{\partial e}{\partial oldsymbol{x}_i}(oldsymbol{x}) = rac{-\partial \sum_p V_p^0 \psi_p(oldsymbol{F}_p)}{\partial oldsymbol{x}} \end{aligned}$$

除此之外需要额外做碰撞检测(如:布料与人体的碰 撞),添加重力和摩擦力等,排除额外处理,物体的运动规律完 全由 $\psi_p(\mathbf{F_p})$ 控制.

5.3.2MLSMPM仿真框架

MLSMPM可以视作是对APIC的优化,Particle 中要储 存的信息,位置 x_p ,速度 v_p ,Deformation gradient F_p ,APIC中 的Affine momentum C_n

Grid 中只要储存速度和质量 \mathbf{v}_i, m_i

- Particle to grid (P2G)
 - $\mathbf{F}_{v}^{n+1} = (\mathbf{I} + \Delta t \mathbf{C}_{v}^{n}) \mathbf{F}_{v}^{n}, \dots$ (Deformation update)
 - $(m\mathbf{v})_i^{n+1} = \sum_p w_{ip} \{ m_p \mathbf{v}_p^n + [m_p \mathbf{C}_p^n \frac{4\Delta t}{\Delta x^2} \sum_p V_p^0 \mathbf{P}(\mathbf{F}_p^{n+1}) (\mathbf{F}_p^{n+1})^T] (\mathbf{x}_i \mathbf{x}_p^n) \}$ (Grid momentum)
 - $m_i^{n+1} = \sum_p m_p w_{ip}$ (Grid mass)
- Grid operations

 - $\hat{\mathbf{v}}_i^{n+1} = (m\mathbf{v})_i^{n+1}/m_i^{n+1}$ (Grid velocity) $\hat{\mathbf{v}}_i^{n+1} = \mathrm{BC}(\hat{\mathbf{v}}_i^{n+1})$ (Grid boundary condition. BC is the boundary condition
- Grid to particle (G2P)

 - $\begin{array}{l} \quad \mathbf{v}_{p}^{n+1} = \sum_{i} w_{ip} \mathbf{v}_{i}^{n+1} \text{ (Particle velocity)} \\ \quad \mathbf{C}_{p}^{n+1} = \frac{4}{\Delta c^{2}} \sum_{i} w_{ip} \mathbf{v}_{i}^{n+1} (\mathbf{x}_{i} \mathbf{x}_{p}^{n})^{T} \text{ (Particle velocity gradient)} \\ \quad \mathbf{x}_{p}^{n+1} = \mathbf{x}_{p}^{n} + \Delta t \mathbf{v}_{p}^{n+1} \text{ (Particle position)} \end{array}$

BC表示网格Boundray Condition,比如,衣物(particle)如 果和人体(some grid node)发生碰撞,将发生碰撞地方速度归 零 $\mathbf{v}_i = 0$

物体的运动规律完全由 $P(F_p) = \frac{\partial \psi_p}{\partial F_p}$ 控制.

具体物理对象的仿真 从上文的讨论中可以发现, μ , λ 系数决 定了弹性物体的运动规律.以冰雪仿真为例(mls-mpm88.cpp)

$$m{F} = m{F}_E m{F}_P$$
 $\mu(m{F}_p) = \mu_0 e^{\epsilon(1-J_p)}, \lambda(m{F}_p) = \lambda_0 e^{\epsilon(1-J_p)}$ $J_p = det(m{F}_p)$

 F_P 计算流程

$$\hat{m{F}}_E^{n+1} = m{F}^{n+1}(m{F}_P^n)^{-1}$$
 $\hat{m{F}}_E^{n+1} = m{U}_E^{n+1}\hat{m{\Sigma}}_E^{n+1}(m{V}_E^{n+1})^T$
 $\sigma_{Ei}^{n+1} = clamp(\hat{\sigma}_{Ei}^{n+1}, 1 - heta_c, 1 + heta_s)$
 $m{F}_E^{n+1} = m{U}_E^{n+1}m{\Sigma}_E^{n+1}(m{V}_E^{n+1})^T$
 $m{F}_n^{n+1} = (m{F}_P^{n+1})^{-1}m{F}^{n+1}$

子.这 取 $10.0, \mu, \lambda$ 是Lame系 ϵ 是hardening因 里 数,由E,v表达,E,v分别取1e4,0.2

$$\mu_0 = \frac{E}{2(1+v)}, \lambda_0 = \frac{Ev}{(1+v)(1-2v)}$$

6 流体仿真

原理 6.1

对于一个温度和密度各处均匀的流体, 其状态可以用向 量场u和压力场p来描述,u,p受边界条件影响随时间和空间 变化,假设边界条件和和 $u(t_0,x),p(t_0,x)$ 已知,则之后任意 时刻流体状态u(t, x), p(t, x)可由 Navier-Stokes 方程推出

其中 ρ 为流体密度, v为流体动力粘度(kinematic viscosity), $\nabla = (\partial/\partial x, \partial/\partial y, \partial/\partial z), \nabla^2 = \nabla \cdot \nabla$. 假设在流体被限 制在领域D内.

根据Helmholtz-Hodge Decomposition定理,对于任意一 个向量场 \boldsymbol{w} ,可以分解唯一分解为一个mass conserving field 和一个 gradient field之和, 设

$$\mathbf{w} = \mathbf{u} + \nabla p, \quad \nabla \cdot \mathbf{u} = 0$$

$$N(\mathbf{x}) = \begin{cases} \frac{3}{4} - |\mathbf{x}|^2 & 0 \le |\mathbf{x}| < \frac{1}{2} \\ \frac{1}{2} (\frac{3}{2} - |\mathbf{x}|)^2 & \frac{1}{2} \le |\mathbf{x}| < \frac{3}{2} \\ 0 & \frac{3}{2} \le |\mathbf{x}| \end{cases}$$

已知w, 等式两边同乘 \triangledown

$$\nabla^2 p = \nabla \boldsymbol{w}$$

则对于p来说是一个给定Neumann边界条件($\partial p/\partial n = 0$ on ∂D)泊松方程, p可求. 定义操作符**P**

$$u = Pw = w - \nabla p$$

改写Navier-Stokes方程为

$$\nabla \cdot \boldsymbol{u} = 0$$
$$\frac{\partial \boldsymbol{u}}{\partial t} = \boldsymbol{P}(-(\boldsymbol{u} \cdot \nabla)\boldsymbol{u} + v\nabla^2\boldsymbol{u} + \boldsymbol{f})$$

以下讨论Navier-Stokes方程求解,使用[?]提供的方法, 一定程度上牺牲物理真实性, 但是速度快且无条件稳定. 首 先讨论时间上离散化

$$egin{align*} oldsymbol{u}(t,oldsymbol{x}) &= oldsymbol{w}_0(oldsymbol{x}) & \stackrel{add\ force}{\longrightarrow} oldsymbol{w}_1(oldsymbol{x}) & \stackrel{advect}{\longrightarrow} oldsymbol{w}_2(oldsymbol{x}) & & \\ & \stackrel{diffuse}{\longrightarrow} oldsymbol{w}_3(oldsymbol{x}) & \stackrel{project}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) &= oldsymbol{u}(t+\Delta t,oldsymbol{x}) & & \\ & \stackrel{particle}{\longrightarrow} oldsymbol{w}_3(oldsymbol{x}) & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) &= oldsymbol{u}(t+\Delta t,oldsymbol{x}) & & \\ & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & & \\ & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & & \\ & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & & \\ & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & & \\ & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & & \\ & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & & \\ & \stackrel{particle}{\longrightarrow} oldsymbol{w}_4(oldsymbol{x}) & \stackrel{particle}$$

求解采用一种propagation的方式,

$$\frac{\partial \boldsymbol{u}}{\partial t} = f, \ \boldsymbol{u}(0) = \boldsymbol{w}_0 \overset{\boldsymbol{w}_1 = \boldsymbol{u}(\triangle t)}{\frown}$$

$$\frac{\partial \boldsymbol{u}}{\partial t} = -\boldsymbol{u} \cdot \nabla \boldsymbol{u}, \ \boldsymbol{u}(0) = \boldsymbol{w}_1 \overset{\boldsymbol{w}_2 = \boldsymbol{u}(\triangle t)}{\frown}$$

$$\frac{\partial \boldsymbol{u}}{\partial t} = v \nabla^2 \boldsymbol{u}, \ \boldsymbol{u}(0) = \boldsymbol{w}_2 \overset{\boldsymbol{w}_3 = \boldsymbol{u}(\triangle t)}{\frown}$$

$$\boldsymbol{w}_4 = \boldsymbol{P}(\boldsymbol{w}_3)$$

add force

$$\frac{\partial \boldsymbol{u}}{\partial t} = f, \ \boldsymbol{u}(0) = \boldsymbol{w}_0$$
$$\boldsymbol{w}_1(\boldsymbol{x}) = \boldsymbol{w}_0(\boldsymbol{x}) + \Delta t \boldsymbol{f}(\boldsymbol{x}, t)$$

advect

$$\frac{\partial \boldsymbol{u}}{\partial t} = -\boldsymbol{u} \cdot \nabla \boldsymbol{u}, \ \boldsymbol{u}(0) = \boldsymbol{w}_1$$

此处做一个近似, 时间步长越大误差越大

$$rac{\partial oldsymbol{u}}{\partial t} = -oldsymbol{w}_1 \cdot oldsymbol{ riangle} oldsymbol{u}$$

利用Method of Characteristics 求解此问题,假设欲求

$$\frac{\partial \boldsymbol{a}(\boldsymbol{x},t)}{\partial t} = -\boldsymbol{v}(\boldsymbol{x}) \cdot \nabla \boldsymbol{a}(\boldsymbol{x},t), \ \boldsymbol{a}(\boldsymbol{x},0) = \boldsymbol{a}_0(\boldsymbol{x})$$

对于某个 x_0 ,假设 $p(x_0,t)$,有

$$\boldsymbol{p}(\boldsymbol{x}_0, 0) = \boldsymbol{x}_0, \ \frac{\mathrm{d} \boldsymbol{p}(\boldsymbol{x}_0, t)}{\mathrm{d} t} = \boldsymbol{v}(\boldsymbol{x}_0)$$

则

$$\frac{\mathrm{d}\boldsymbol{a}(\boldsymbol{p}(\boldsymbol{x}_0,t),t)}{\mathrm{d}t} = \nabla \boldsymbol{a} \cdot \frac{\mathrm{d}\boldsymbol{p}}{\mathrm{d}t} + \frac{\partial \boldsymbol{a}}{\partial t}$$
$$= \nabla \boldsymbol{a} \cdot \boldsymbol{v}(\boldsymbol{x}_0) - \boldsymbol{v}(\boldsymbol{x}_0) \cdot \nabla \boldsymbol{a} = 0$$

即 $\mathbf{a}(\mathbf{p}(\mathbf{x}_0,t),t)$ 是常数.

$$egin{aligned} m{w}_2(m{x}) &= m{w}_1(m{p}(m{x}, -\triangle t)) \ m{p}(m{x}, -\triangle t) &= m{x} - \triangle t m{w}_1(m{x}) \ \end{pmatrix} \ m{N}(m{x}) &= egin{cases} rac{1}{2} |\mathbf{x}|^3 - |\mathbf{x}|^2 + rac{2}{3} & 0 \leqslant |\mathbf{x}| < 1 \ rac{1}{6} (2 - |\mathbf{x}|)^3 & 1 \leqslant |\mathbf{x}| < 2 \ 0 & 2 \leqslant |\mathbf{x}| \end{cases}$$

diffuse 这种方式牺牲物理上的精确性(?), 但稳定且便于实现.

考虑粘度影响

$$\frac{\partial \boldsymbol{u}}{\partial t} = v \nabla^2 \boldsymbol{u}$$

若采用显示欧拉解原微分方程

$$\frac{\boldsymbol{u}(t+\triangle t)-\boldsymbol{u}(t)}{\triangle t}=v\nabla^2\boldsymbol{u}(t)$$

$$\boldsymbol{u}(t+\triangle t) = (\boldsymbol{I} + \triangle t v \nabla^2) \boldsymbol{u}(t)$$

则diffuse过程可以写为

$$\boldsymbol{w}_3(\boldsymbol{x}) = (\boldsymbol{I} + v \triangle t \nabla^2) \boldsymbol{w}_2(\boldsymbol{x})$$

显示对步长有要求, 为得到稳定的仿真方法, 使用隐式欧拉

$$\frac{\boldsymbol{u}(t+\Delta t)-\boldsymbol{u}(t)}{\Delta t}=v\nabla^2\boldsymbol{u}(t+\Delta t)$$
$$\boldsymbol{w}_3(\boldsymbol{x})=(\boldsymbol{I}-v\Delta t\nabla^2)^{-1}\boldsymbol{w}_2(\boldsymbol{x})$$

project 此步需要解possion方程,可以使用差分方法,最后变为求解一个线性方程.

$$\nabla^2 p = \nabla \cdot \boldsymbol{w}_3, \quad \boldsymbol{w}_4 = \boldsymbol{w}_3 - \nabla p$$

6.2 实现

[?]给出了一种实时的实现方法. 考虑2维情况, 密度和速度都定义在网格中心, 空间中实际网格为 $N \times N$, 边缘多加一层用于考虑boundary conditions.

$$\begin{split} & m_i^n = \sum_p w_{ip}^n m_p \\ & D_p^n = \sum_i w_{ip}^n (x_i^n - x_p^n) (x_i^n - x_p^n)^T = \sum_i w_{ip}^n x_i^n (x_i^n)^T - x_p^n (x_p^n)^T \\ & m_i^n v_i^n = \sum_i w_{ip}^n m_p (v_p^n + B_p^n (D_p^n)^{-1} (x_i^n - x_p^n)) \end{split}$$

```
#define IX(i,j) ((i)+(N+2)*(j))
#define SWAP(x0,x) {float * tmp=x0;x0=x;x=tmp;}
#define FOR_EACH_CELL for ( i=1 ; i<=N ; i++ ) { for ( j=1 ; j<=N ; j++ ) {
#define END FOR }}
static float * u, * v, * u_prev, * v_prev;
static float * dens, * dens_prev;
      仿真主程序
void sim main(void) {
 get_force_source( dens_prev, u_prev, v_prev, w_prev );
 vel_step ( N, u, v, w, u_prev, v_prev, w_prev, visc, dt );
 // 更新密度场
 dens_step ( N, dens, dens_prev, u, v, w, diff, dt );
void dens_step ( int N, float * x, float * x0,
                float * u, float * v, float diff, float dt ) {
 add_source ( N, x, x0, dt );
 SWAP ( x0, x ); diffuse ( N, 0, x, x0, diff, dt );
 SWAP ( x0, x ); advect ( N, 0, x, x0, u, v, dt );
void vel_step ( int N, float * u, float * v, float * u0,
               float * v0, float visc, float dt ) {
 add_source ( N, u, u0, dt ); add_source ( N, v, v0, dt );
 SWAP ( u0, u ); diffuse ( N, 1, u, u0, visc, dt );
 SWAP ( v0, v ): diffuse ( N. 2, v, v0, visc, dt ):
 project ( N, u, v, u0, v0 );
 SWAP ( u0, u ); SWAP ( v0, v );
 advect ( N, 1, u, u0, u0, v0, dt ); advect ( N, 2, v, v0, u0, v0, dt );
 project ( N, u, v, u0, v0 );
```

流体有从密度高的地方流向密度低地方的趋势, diffuse过程用于描述这一现象.

$$x(i,j) = x_0(i,j) + a(x_0(i-1,j) + x_0(i+1,j) + x_0(i,j-1) + x_0(i,j+1) - 4x_0(i,j))$$

然而这样写会在系数a较大时不稳定, 所以改为隐式写法

$$x_0(i,j) = x(i,j) - a(x(i-1,j) + x(i+1,j) + x(i,j-1) + x(i,j+1) - 4x(i,j))$$

即解线性系统, 矩阵系数, 利用Gauss-Seidel relaxation方式 迭代求解.

advect步骤,假设待求网格(i,j)中心处的粒子,前一时刻处于(i',j')中,则(i,j)处的速度等于上一时刻(i',j')处的速度.

$$egin{aligned} oldsymbol{v}_p^{n+1} &= \sum_i w_{ip}^n ilde{oldsymbol{v}}_i^{n+1} \ oldsymbol{B}_p^{n+1} &= \sum_i w_{ip}^n ilde{oldsymbol{v}}_i^{n+1} (oldsymbol{x}_i^n - oldsymbol{x}_p^n)^T. \end{aligned}$$

法

Projection过程要求泊松方程, 二维情况使用五点差分

$$\nabla^2 \phi = f$$

$$\frac{\partial^2 \phi}{\partial x^2} + \frac{\partial^2 \phi}{\partial y^2} = \mathbf{f}$$

$$\frac{\phi_{i+1,j} - 2\phi_{i,j} + \phi_{i-1,j}}{\Delta x^2} + \frac{\phi_{i,j+1} - 2\phi_{i,j} + \phi_{i,j-1}}{\Delta y^2} = f_{i,j}$$

边界上 $\phi_{i,j}$ 为0?. 可以总结出一个线性方程 $\mathbf{A}\phi = \mathbf{f}$, 同样通过Gauss-Seidel relaxation求解.

```
lin_solve ( N, 0, p, div, 1, 4 );
FOR_EACH_CELL
  u[IX(i,j)] -= 0.5f*N*(p[IX(i+1,j)]-p[IX(i-1,j)]);
  v[IX(i,j)] -= 0.5f*N*(p[IX(i,j+1)]-p[IX(i,j-1)]);
END_FOR
  set_bnd ( N, 1, u ); set_bnd ( N, 2, v );
```

此外有两个辅助函数,一个用于直接叠加场,另外一个 用于边界条件设置.

```
void add_source ( int N, float * x, float * s, float dt ) {
   int i, size=(N+2)*(N+2);
   for ( i=0 ; i<size ; i++ ) x[i] += dt*s[i];
}
void set_bnd ( int N, int b, float * x ) {
   int i;
   for ( i=1 ; i<=N ; i++ ) {
        x[IX(0 ,i)] = b==1 ? -x[IX(1,i)] : x[IX(1,i)];
        x[IX(N+1,i)] = b==1 ? -x[IX(N,i)] : x[IX(N,i)];
        x[IX(i,0)] = b==2 ? -x[IX(i,1)] : x[IX(i,1)];
        x[IX(i,0)] = b==2 ? -x[IX(i,0)] : x[IX(i,0)];
}
x[IX(0 ,0 )] = 0.5f*(x[IX(1,0 )]+x[IX(0 ,1)]);
x[IX(0 ,N+1)] = 0.5f*(x[IX(1,N+1)]+x[IX(0 ,N)]);
x[IX(N+1,0 )] = 0.5f*(x[IX(N,0 )]+x[IX(N+1,1)]);
}</pre>
```

7 附录

7.1 Schur Complement

定义矩阵

$$oldsymbol{M} = egin{pmatrix} oldsymbol{A} & oldsymbol{B} \ oldsymbol{C} & oldsymbol{D} \end{pmatrix}$$

若D可逆,D在M 中的舒尔补为:

$$A - BD^{-1}C$$

$$egin{aligned} oldsymbol{L} = egin{pmatrix} oldsymbol{I} & oldsymbol{0} \ -oldsymbol{D}^{-1}oldsymbol{C} & oldsymbol{D}^{-1} \end{pmatrix} \ oldsymbol{M}oldsymbol{L} = egin{pmatrix} oldsymbol{A} - oldsymbol{B}oldsymbol{D}^{-1}oldsymbol{C} & oldsymbol{B}oldsymbol{D}^{-1} \ oldsymbol{0} & oldsymbol{I} \end{pmatrix} \end{aligned}$$

若A可逆,A在M 中的舒尔补为:

$$D-CA^{-1}B$$
 $T=egin{pmatrix} A^{-1} & 0 \ -CA^{-1} & I \end{pmatrix}$ $TM=egin{pmatrix} I & A^{-1}B \ 0 & D-CA^{-1}B \end{pmatrix}$ $Ax+By=a$ $Cx+Dy=b$

对第二个式子左乘 BD^{-1} 可以得到

$$BD^{-1}Cx + By = BD^{-1}b$$

综上可推出

$$(A - BD^{-1}C)x = a - BD^{-1}b$$

$$x = (A - BD^{-1}C)^{-1}(a - BD^{-1}b)$$

同理, 对第一个式子左乘 CA^{-1}

$$Cx + CA^{-1}By = CA^{-1}a$$

综上可以推出

$$(D - CA^{-1}B)y = b - CA^{-1}a$$

$$y = (D - CA^{-1}B)^{-1}(b - CA^{-1}a)$$