

Recap:

$$\varphi = \bar{\varphi}(x, t)$$

$$\frac{D}{Dt} \int_{P_t} \varphi dx = \int_{P_t} \left[\frac{\partial \varphi}{\partial t} + \frac{\partial (\varphi v)}{\partial x} \right] dx$$

RTT

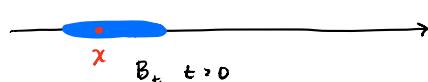
$$P_t \subseteq B_t$$

$$\rho = \bar{\rho}(x, t) \Rightarrow M(P_t) = \text{const.} = \int_{P_t} \rho dx$$

$$\frac{\partial \rho}{\partial t} + \frac{\partial (\rho v)}{\partial x} = 0$$

Eulerian continuity equation, holds for every little x in P_t $\forall x \in B_t$

$$\frac{Dm(P_t)}{Dt} = 0 \quad \text{global balance of mass}$$



$$M(P_t) = \text{const.}$$

$$M(P_{t_0}) = M(P_{t_0}) = M(P_0)$$

$$M(P_0) = m(P_0)$$

$$\int_{P_0} \rho(\bar{x}, 0) d\bar{x} = \int_{P_t} \bar{\rho}(x, t) dx \quad \bar{\rho}(\bar{x}, 0) \equiv \rho_0(\bar{x})$$

\uparrow \uparrow
 Lagrangian Eulerian

$$\int_{P_0} \rho_0(\bar{x}) d\bar{x} = \int_{P_t} \bar{\rho}(\bar{x}, t) F d\bar{x}$$

$$\int_{P_0} (\rho_0 - \rho F) d\bar{x} = \Delta F \cdot P_0 = B_0$$

$$\therefore \rho_0 = \rho F \quad \forall \bar{x} \in B_0$$

local Lagrangian balance of mass + continuity equation

... will need to know how to start from this (Lagrangian) to Eulerian

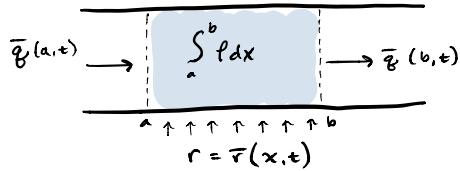
Another case: thermodynamic properties always defined as specific

$$\begin{aligned} \psi = \frac{\varphi}{\rho} \Rightarrow \frac{D}{Dt} \int_{P_t} \rho \psi dx &= \int_{P_t} \left[\frac{\partial(\rho \psi)}{\partial t} + \frac{\partial(\rho \psi v)}{\partial x} \right] dx \\ &= \int_{P_t} \left[\frac{\partial \rho}{\partial t} \psi + \rho \frac{\partial \psi}{\partial t} + \frac{\partial(\rho v)}{\partial x} \psi + \rho v \frac{\partial \psi}{\partial x} \right] dx \\ &= \underbrace{\int_{P_t} \left[\frac{\partial \rho}{\partial t} + \frac{\partial(\rho v)}{\partial x} \right] \psi dx}_{= 0 \text{ because of mass conservation}} + \underbrace{\int_{P_t} \left[\frac{\partial \psi}{\partial t} + v \frac{\partial \psi}{\partial x} \right] \rho dx}_{\frac{D\psi}{Dt}} \end{aligned}$$

$$\frac{D}{Dt} \int_{P_t} \rho \psi dx = \int_{P_t} \rho \frac{D\psi}{Dt} dx$$

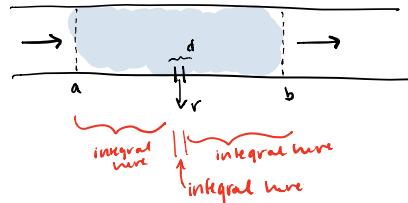
RTT 2

Example #5: jet engine - mass not conserved, but mass added



$$\begin{aligned}
 \frac{\partial}{\partial t} \int_a^b \bar{p}(x,t) dx &= \int_a^b \frac{\partial \bar{p}(x,t)}{\partial t} dx = \bar{p}(a,t) \bar{v}(a,t) - \bar{p}(b,t) \bar{v}(b,t) + \int_a^b \bar{r}(x,t) dx = \\
 &= - \int_a^b \bar{d}(ev) + \int_a^b r dx = - \int_a^b \frac{\partial(ev)}{\partial x} dx + \int_a^b r dx \\
 &= \int_a^b \left[\frac{\partial \bar{p}}{\partial t} + \frac{\partial(ev)}{\partial x} - r \right] dx = 0 \quad \forall a,b \\
 \therefore \frac{\partial \bar{p}}{\partial t} + \frac{\partial(ev)}{\partial x} - r &= 0 \Rightarrow \boxed{\frac{\partial \bar{p}}{\partial t} + \frac{\partial(ev)}{\partial x} = r}
 \end{aligned}$$

now imagine there's single hole



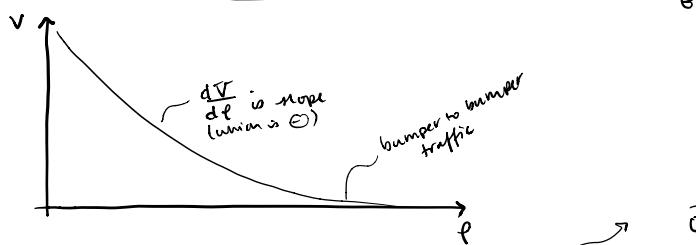
Application: Car Traffic

$$p = \text{car density} = \frac{\# \text{ cars}}{\text{length}}$$

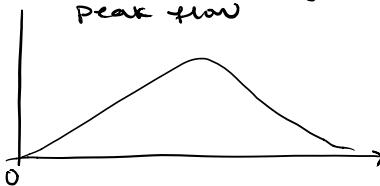
assume same amount of cars in/out: conservation

$$\boxed{\frac{\partial p}{\partial t} + \frac{\partial(ev)}{\partial x} = 0}$$

$$p=? \\ v=?$$



$q = pV$, at some point between 0 and max density, will have peak flow



$$Q(p) = pV(p) \dots \text{experimental data} \\ \text{"state equation"}$$

$$\therefore \frac{\partial p}{\partial t} + \frac{\partial Q}{\partial x} = 0$$

$$\frac{\partial p}{\partial t} + \frac{\partial Q}{\partial x} = \frac{\partial p}{\partial t} + \frac{\partial Q}{\partial p} \frac{\partial p}{\partial x} = 0$$

$$\frac{\partial Q}{\partial p} = c(p) \quad \therefore \quad \boxed{\frac{\partial p}{\partial t} + c(p) \frac{\partial p}{\partial x} = 0} \quad \text{we call this the PDE}$$

$$\rho(x, t) = \rho^* = \text{const.}$$

$$\rho = \rho^* + \hat{\rho}(x, t) \quad \dots \quad \hat{\rho} = \text{small disturbance}$$

$$c(\rho) = c(\rho^* + \hat{\rho}) = c(\rho^*) + \frac{dc(\rho^*)}{d\rho} \hat{\rho} + \cancel{H.O.T.}^0$$

$$\boxed{c(\rho)} = c(\rho^*) + \frac{dc(\rho^*)}{d\rho} \hat{\rho} = \boxed{c^* + \frac{dc(\rho^*)}{d\rho} \hat{\rho}}$$

$$\boxed{\frac{\partial \hat{\rho}}{\partial t} + c^* \frac{\partial \hat{\rho}}{\partial x} = 0}$$

unidirectional wave
equation, c^* wave propagation good

describes wave propagation
in only one direction

D'Alembert's Solution

$$x, t \rightarrow \xi, \eta \quad \xi = x + c^* t \quad \eta = x - c^* t$$

$$\frac{\partial \hat{\rho}}{\partial t} = \frac{\partial \hat{\rho}}{\partial \xi} \frac{\partial \xi}{\partial t} + \frac{\partial \hat{\rho}}{\partial \eta} \frac{\partial \eta}{\partial t} = \frac{\partial \hat{\rho}}{\partial \xi} c^* - \frac{\partial \hat{\rho}}{\partial \eta} c^*$$

$$\frac{\partial \hat{\rho}}{\partial x} = \frac{\partial \hat{\rho}}{\partial \xi} \frac{\partial \xi}{\partial x} + \frac{\partial \hat{\rho}}{\partial \eta} \frac{\partial \eta}{\partial x} = \frac{\partial \hat{\rho}}{\partial \xi} + \frac{\partial \hat{\rho}}{\partial \eta}$$

$$c^* \left(\frac{\partial \hat{\rho}}{\partial \xi} - \frac{\partial \hat{\rho}}{\partial \eta} \right) + c^* \left(\frac{\partial \hat{\rho}}{\partial \xi} + \frac{\partial \hat{\rho}}{\partial \eta} \right) = 0 \Rightarrow 2 c^* \frac{\partial \hat{\rho}}{\partial \xi} = 0$$

$$\therefore \frac{\partial \hat{\rho}}{\partial \xi} = 0 \Rightarrow \hat{\rho} = f(\eta) = \underline{f(\eta)} = f(x - c^* t)$$

$f(\cdot) = ?$ Determine it from IC

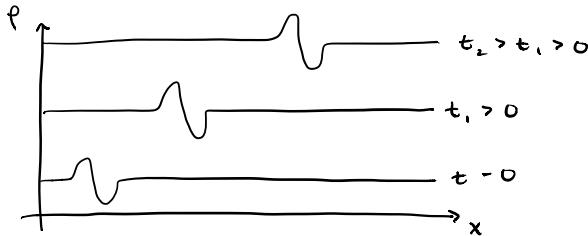
$$\text{suppose at } t=0 \Rightarrow \rho(\underline{x}, 0) = \rho_0(\underline{x}) = \rho^* + f(\underline{x})$$

$\therefore f(\underline{x}) = \rho_0(\underline{x}) - \rho^*$ true for any \underline{x} ($\forall \underline{x}$)

$$\hat{\rho} = f(x - c^* t) = \rho_0(x - c^* t) - \rho^*$$

$$\hat{\rho} = \underbrace{\rho^* + \rho_0(x - c^* t) - \rho^*}_{\hat{\rho}} = \rho_0(x - c^* t)$$

$$\boxed{\rho(x, t) = \rho_0(x - c^* t)}$$



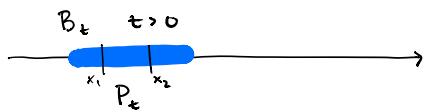
$$c(\rho) \neq V(\rho)$$

$$Q(\rho) = \rho V'(\rho) \Rightarrow \underline{c(\rho)} = \frac{dQ}{d\rho} = V(\rho) + \rho \frac{dV}{d\rho}$$

$$\frac{dV}{d\rho} < 0 \quad c(\rho) < V(\rho)$$

for bumper-to-bumper $V(\rho) \approx 0 \Rightarrow c(\rho) < 0 \dots$ disturbance will be in opposite direction

Balance of Momentum



body forces and contact forces act on piece of material P_t
 like gravity, centripetal
 ↑
 t between bodies

$$\sigma(x,t) \xleftarrow{\text{body forces}} \xrightarrow{\text{contact forces}} \sigma(x,t)$$

$$F(P_t) = \sigma(x_2, t) - \sigma(x_1, t) + \int_{P_t} \rho b(x, t) dx =$$

$$= \int_{P_t} d\sigma(x, t) + \int_{P_t} \rho b dx =$$

$$= \int_{P_t} \frac{\partial \sigma}{\partial x} dx + \int_{P_t} \rho b dx = \boxed{\int_{P_t} \left(\frac{\partial \sigma}{\partial x} + \rho b \right) dx}$$

resultant force
 $\forall P_t \subseteq B_t$

$$P(\ell_t) = \int_{P_t} \rho v dx \quad \forall P_t \subseteq B_t$$

$$1^{\text{st}} \text{ Euler's Axiom: } \frac{D P(P_t)}{Dt} = F(P_t)$$

$$\frac{D}{Dt} \int_{P_t} \rho v dx = \int_{P_t} \left(\frac{\partial \sigma}{\partial x} + \rho b \right) dx$$

$$\downarrow (\text{by ETT2})$$

$$\int_{P_t} \rho \frac{DV}{Dt} dx = \int_{P_t} \left(\frac{\partial \sigma}{\partial x} + \rho b \right) dx$$

$$\int_{P_t} \left(\rho \frac{DV}{Dt} - \frac{\partial \sigma}{\partial x} - \rho b \right) dx = 0 \quad \forall P_t \subseteq B_t$$

$$\boxed{\rho \frac{DV}{Dt} = \frac{\partial \sigma}{\partial x} + \rho b}$$

Local Eulerian balance of momentum
 $\forall x \in B_t, t$

$$\boxed{\rho \hat{v} = \frac{\partial \sigma}{\partial x} + \rho b}$$

Static conditions ($v=0$)

$$\boxed{\frac{\partial \sigma}{\partial x} + \rho b = 0}$$

Local Equilibrium

$$\rho \frac{DV}{Dt} = \frac{\partial \sigma}{\partial x} + \rho b / F \Rightarrow \underbrace{\rho F}_{F_0} \frac{DV}{Dt} = \underbrace{\frac{\partial \sigma}{\partial x} F}_{\frac{\partial \sigma}{\partial x} \frac{\partial x}{\partial X}} + \underbrace{\rho F b}_{\rho_0}$$

$$\boxed{\rho_0 \frac{DV}{Dt} = \frac{\partial \sigma}{\partial X} + \rho_0 b}$$

Local Lagrangian balance of momentum
 $\forall X \in B_0$

Lagrangian Balance of Mass can be written as

$$\rho_0 = \rho F \\ \rho_0 \equiv \rho(\mathbf{x}, 0) \quad \text{true for every } \mathbf{x} \text{ in } B_0 \quad \text{*local balance of mass}$$

(aka $\forall \mathbf{x} \in B_0$)

$$\frac{\partial \rho}{\partial t} + \frac{\partial (\rho v)}{\partial x} = 0 \quad \left[\begin{array}{l} \text{Eulerian local balance of} \\ \text{mass } \nabla \times B_x \end{array} \right]$$

Lagrangian continuity equation is algebraic while Eulerian is a first order PDE

9) from Lagrangian to Eulerian

$$\rho_0 = \rho F \rightarrow \frac{D\rho_0}{Dt} = \frac{D(\rho F)}{Dt} = \frac{D(\rho)}{Dt} F + \rho \frac{DF}{Dt} = \left(\frac{\partial \rho}{\partial t} + \frac{\partial \rho}{\partial x} v \right) F + \rho \frac{\partial v}{\partial x} F$$

\nearrow Lagrangian \searrow Eulerian

$$\frac{\partial \rho}{\partial t} + \frac{\partial \rho}{\partial x} v + \rho \frac{\partial v}{\partial x} = 0 \Rightarrow \underbrace{\frac{\partial \rho}{\partial t} + \frac{\partial (\rho v)}{\partial x}}_{\text{Eulerian}} = 0$$

Material time

derivative of momentum : $\frac{D\mathcal{P}(P_t)}{Dt} = F(P_t) \quad \forall P_t \subseteq B_t \quad F(P_t) = \int_{P_t} \rho b dx + \sigma(x_2, t) - \sigma(x_1, t)$

\mathcal{P} =linear momentum

$$\mathcal{P}(P_t) \equiv \int_{P_t} \rho v dx$$

$$\boxed{\frac{\partial \mathcal{P}}{\partial x} + \rho b = \rho \frac{Dv}{Dt}} \quad \forall x \in B_t, t$$

$\boxed{\frac{\partial \mathcal{P}}{\partial x}}$ body forces
(forces that act at distances)

$\boxed{\frac{\partial \mathcal{P}}{\partial x}}$ contact forces
(represent physical interactions between two bodies)

local eulerian balance of momentum

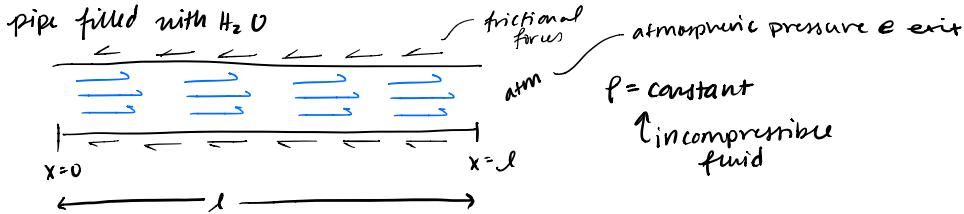
$$\text{For } v=0 \text{ or } v=\text{const.} \Rightarrow \boxed{\frac{\partial \mathcal{P}}{\partial x} + \rho b = 0} \quad \text{local eulerian equilibrium}$$

$$\left. \begin{aligned} & \underbrace{\frac{\partial \mathcal{P}}{\partial x} F}_{\frac{\partial \mathcal{P}}{\partial x} \frac{\partial x}{\partial \mathbf{x}}} + \underbrace{\rho F b}_{\rho_0} = \underbrace{\rho F \frac{Dv}{Dt}}_{\rho_0} \\ & \left. \Rightarrow \boxed{\frac{\partial \mathcal{P}}{\partial x} + \rho_0 b = \rho_0 \frac{Dv}{Dt}} \quad \forall \mathbf{x} \in B_0, t \right. \end{aligned} \right\} \quad \begin{aligned} & \text{local lagrangian momentum} \\ & \text{Lagrangian time deriv.} \\ & \frac{Dv}{Dt} = \frac{\partial v}{\partial t} \quad v=v(\mathbf{x}, t) \end{aligned}$$

Ex. 6: Simple Fluid Flow in a Pipe

For Fluids, $\sigma(x, t) = -p(x, t)$

\uparrow pressure
 by convention bc \uparrow press means \downarrow volume of body



- a. Use continuity equation to show velocity just function of time (as it doesn't change w/x)

$$v = v(t)$$

$$\frac{dp}{dt} + \frac{d(\rho v)}{dx} = 0 \Rightarrow \rho \frac{dv}{dx} = 0 \Rightarrow \frac{dv}{dx} = 0 \Rightarrow \underbrace{v = v(t)}$$

0 because incompressible fluid

- b. $\rho b = -Cv$, effective body force from frictional resistance
 $\uparrow C = \text{const.}$

momentum eq: $\frac{d\sigma}{dx} + pb = -\rho \frac{dv}{dt}$ $\sigma = \text{pressure} = -p$

\Downarrow

$$-\frac{dp}{dx} - Cv = \rho \frac{dv}{dt}$$

why go from material to ordinary derivative?
 bc v just function of time

$$\therefore \frac{dp}{dx} = -\underbrace{(cv + \rho \frac{dv}{dt})}_{\text{function of time}} \Rightarrow \text{if integrate} \Rightarrow P = -\left(cv + \rho \frac{dv}{dt}\right)x + f(t)$$

- c. suppose at $x=0$ then $p(0, t) = P_0(t)$ "manis pressure" — pressure of faucet
 $x=l p(l, t) \equiv P_1(t)$ atmospheric pressure

to have good flow, always need $P_0 > P_1$ manis must be greater than atmospheric

momentum equation becomes
 $\rho \frac{dv}{dt} + Cv = \frac{P_0 - P_1}{l}$

show how this is \rightarrow .

in B.C. at $x=0$ and $x=l$

$$P_0(t) = f(t)$$

$$p(x, t) = -\left(cv + \rho \frac{dv}{dt}\right)x + P_0(t)$$

$$P_1(t) = -\left(cv + \rho \frac{dv}{dt}\right)l + P_0(t) \Rightarrow \rho \frac{dv}{dt} + Cv = \frac{P_0 - P_1}{l}$$

$$\text{and then at } t \rightarrow \infty \quad v = \frac{P_0 - P_1}{cl}$$

- d. obtain velocity distribution, $v(0)=0$ find $v(t)$

÷ by ρ since constant and nonzero

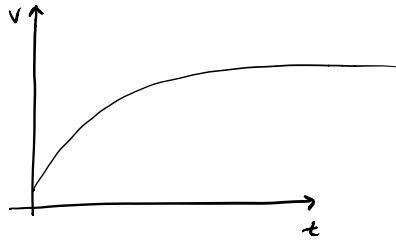
$$\frac{dv}{dt} + \frac{C}{\rho} v = \frac{P_0 - P_1}{\rho l} \Rightarrow v(t) = K e^{-\frac{C}{\rho} t} + \frac{P_0 - P_1}{cl}$$

constant of integration

$$0 = v(0) = K + \frac{P_0 - P_1}{cl} \Rightarrow K = \frac{P_1 - P_0}{cl}$$

$$\left. \begin{array}{l} \text{general solution of the DE, now} \\ \text{need to find } K \end{array} \right\}$$

as $t \uparrow$, exponential smaller, $v \uparrow$



Balance of Energy

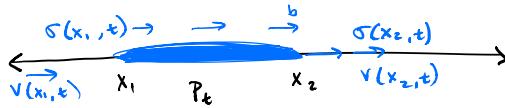
→ we'll start with mechanical

Power Th. - rate of mechanical working that is done on part or whole of material body is equal to change in KE on body

$W(P_t)$ mechanical working, rate of external forces acting

$$W(P_t) = \frac{D K(P_t)}{D t} + \int_{P_t} \rho \frac{d v}{d x} dx \quad \forall P_t \leq B_t \quad \text{Power Th.}$$

kinetic energy $K(P_t) = \int_{P_t} \frac{1}{2} \rho v^2 dx$ kinetic energy of $P_t \leq B_t$



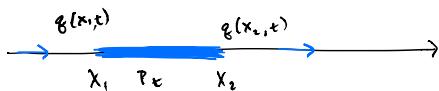
$$\begin{aligned} W(P_t) &= \int_{P_t} \rho b v dx + \underbrace{\sigma(x_2, t)v(x_2, t) - \sigma(x_1, t)v(x_1, t)}_{\int_{P_t} d(\sigma v) = \int_{P_t} \frac{\partial(\sigma v)}{\partial x} dx} = \int_{P_t} \left[\rho b v + \frac{\partial(\sigma v)}{\partial x} \right] dx = \\ &\quad \int_{P_t} \left(\rho b v + \frac{\partial \sigma}{\partial x} v + \sigma \frac{\partial v}{\partial x} \right) dx \\ &\quad \uparrow \text{force} \cdot \text{velocity} = \text{rate of work} = \text{power} \\ &= \int_{P_t} \rho \frac{D v}{D t} v dx + \int_{P_t} \sigma \frac{\partial v}{\partial x} dx - \int_{P_t} \frac{1}{2} \rho \frac{D v^2}{D t} dx + \int_{P_t} \sigma \frac{\partial v}{\partial x} dx \\ &\quad \underbrace{\int_{P_t} \frac{D}{D t} \left(\frac{1}{2} \rho v^2 \right) dx}_{\text{from RTT2}} \\ &= \frac{D K(P_t)}{D t} + \int_{P_t} \sigma \frac{\partial v}{\partial x} dx \end{aligned}$$

↑
by
momentum
equation

1st Law of Thermodynamics \downarrow internal

$$W(P_t) = \frac{D K(P_t)}{D t} + \frac{D U}{D t} - Q(P_t)$$

$$U(P_t) = \int_{P_t} \rho u dx \quad u = \text{specific internal energy (energy/mass)}$$



$$Q(P_t) = \int_{P_t} \rho r dx + q(x_2, t) - q(x_1, t) \quad \forall P_t \leq B_t$$

$r = \text{specific internal heat (heat/mass)}$

$$\frac{D K(P_t)}{Dt} + \int_{P_t} \sigma \frac{\partial v}{\partial x} = \frac{D K(P_t)}{Dt} + \underbrace{\int_{P_t} \rho u dx - \int_{P_t} \rho r dx}_{\text{mL ETT2}} + g(x_+, t) - g(x_-, t)$$

$$\int_{P_t} \sigma \frac{\partial v}{\partial x} = \int_{P_t} \rho \frac{Du}{Dt} dx - \int_{P_t} \rho r dx + \int_{P_t} \frac{\partial g}{\partial x} dx$$

$$\int_{P_t} \left(\rho \frac{Du}{Dt} - \sigma \frac{\partial v}{\partial x} - \rho r + \frac{\partial g}{\partial x} \right) dx = 0 \quad \forall P_t \subseteq B_t$$

\therefore the integrand must be zero ... $\rho \frac{Du}{Dt} - \sigma \frac{\partial v}{\partial x} - \rho r + \frac{\partial g}{\partial x} = 0$

$$\therefore \rho \frac{Du}{Dt} = \sigma \frac{\partial v}{\partial x} + \rho r + \frac{\partial g}{\partial x} \quad \begin{array}{l} \text{local Eulerian} \\ \text{1st law } \forall x \in B_t, t \end{array}$$

Multiply everything by F to get Lagrangian

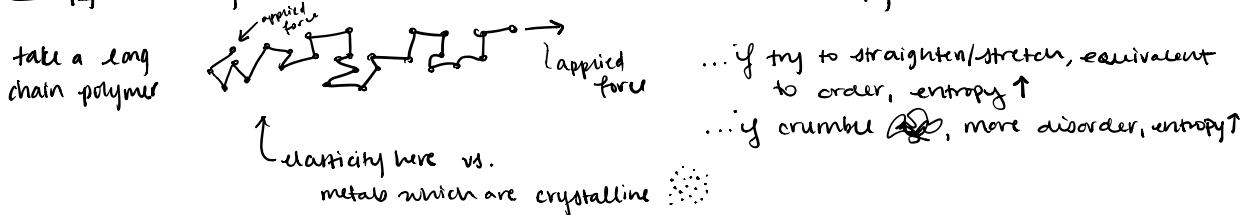
$$\underbrace{\rho F \frac{Du}{Dt}}_{\rho_0} = \sigma \frac{\partial v}{\partial x} F + \underbrace{\rho F r}_{\rho_0 r} - \frac{\partial g}{\partial x} F$$

$$\rho_0 \frac{Du}{Dt} = \sigma \frac{\partial v}{\partial x} \frac{\partial x}{\partial X} + \rho_0 r - \frac{\partial g}{\partial x} \frac{\partial x}{\partial X}$$

$$\boxed{\rho_0 \frac{Du}{Dt} = \sigma \frac{\partial v}{\partial X} + \rho_0 r - \frac{\partial g}{\partial X} \quad \begin{array}{l} \text{Local Lagrangian} \\ \text{1st law } \forall X \in B_0, t \end{array}}$$

What we don't know from 1st law is the direction of heat transfer

Entropy - quantity of disorder/randomness ... \uparrow disorder, \uparrow entropy



Recap:

$$W(P_t) = \int_{P_t} \rho b v dx + \sigma(x_+, t) v(x_+, t) - \sigma(x_-, t) v(\sigma x_-, t) =$$

$$\underbrace{\frac{D K(P_t)}{Dt}}_{\substack{\text{due to motion} \\ \downarrow \text{work power} \\ \downarrow \text{heat}}} + \underbrace{\int_{P_t} \sigma \frac{\partial v}{\partial x} dx}_{\substack{\text{change of KE} \\ \downarrow \text{stress power}}} \quad \begin{array}{l} \text{due to deformation} \\ \text{kinetic energy} \\ \downarrow \text{potential energy} \\ \downarrow \text{thermal energy} \\ \downarrow \text{internal energy} \end{array}$$

$$W(P_t) + Q(P_t) = \frac{DE_{tot}(P_t)}{Dt} \quad \text{where } E_{tot}(P_t) = K(P_t) + \Phi(P_t) + U(P_t) \text{ in classical thermodynamics}$$

Φ in continuum mechanics is part of $W(P_t)$ even though not classically true

So in continuum mechanics we can write the 1st law as

$$W(P_t) + Q(P_t) = \frac{DE_{tot}(P_t)}{Dt} = \frac{DK(P_t)}{Dt} + \frac{DU(P_t)}{Dt}$$

$$W(P_t) = \frac{DK(P_t)}{Dt} + \frac{DU(P_t)}{Dt} - Q(P_t) \quad \forall P_t \subseteq B_t$$

something DS will ask:

- where is the potential

energy?

- can you quantify?

* know concepts + math

$$\rho \frac{Du}{Dt} = \sigma \frac{\partial v}{\partial x} + pr - \frac{\partial \sigma}{\partial x} \quad \forall x \in B_t, t \quad \text{local Eulerian}$$

$$\rho_o \frac{Du}{Dt} = \sigma \frac{\partial v}{\partial X} + \rho_o r - \frac{\partial \sigma}{\partial X} \quad \forall X \in B_o, t \quad \text{local Langrangian}$$

r = specific heat (internal heat source)

2nd Law of Thermodynamics

$$\rho \frac{Du}{Dt} = \sigma \frac{\partial v}{\partial x} + pr - \frac{\partial \sigma}{\partial x}$$

$S(P_t)$ = internal entropy of the body

$$S(P_t) = \int_{P_t} \rho \eta dx \quad \forall P_t \subseteq B_t$$

entropy generated by the system

and η = specific entropy

$\dot{H}(P_t)$ = rate of entropy input into body

$$\dot{H}(P_t) = \int_{P_t} \rho \frac{r}{T} dx + \frac{q(x_1, t)}{T} - \frac{q(x_2, t)}{T}$$

T = absolute temperature

$\Gamma(P_t)$ = total entropy production of the body

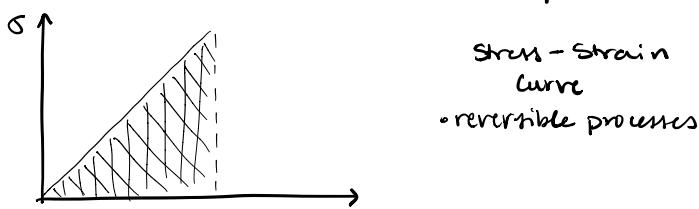
$$\Gamma(P_t) = \frac{D S(P_t)}{Dt} - \dot{H}(P_t) \quad \forall P_t \subseteq B_t$$

$$\Gamma(P_t) \geq 0 \quad \forall P_t \subseteq B_t$$

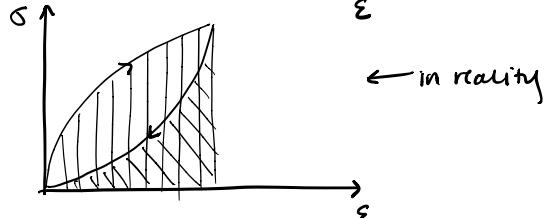
2nd Law of Thermodynamics

If $\Gamma(P_t) = 0$, then we have reversible processes

If $\Gamma(P_t) > 0$, then we have irreversible processes



In reality, nothing is ^{absolutely} reversible, unless quasi-static, because there is internal friction



$$\dot{H}(P_t) = \int_{P_t} \rho \frac{r}{T} dx + \frac{\dot{S}(x_1, t)}{T} - \frac{\dot{S}(x_2, t)}{T} = \int_{P_t} \rho \frac{r}{T} dx - \int_{\partial P_t} \frac{\partial}{\partial x} \left(\frac{\dot{S}}{T} \right) dx$$

using RTT2:

$$\frac{\partial}{\partial t} \int_{P_t} \rho \eta dx - \int_{P_t} \rho \frac{r}{T} dx + \int_{P_t} \frac{\partial}{\partial x} \left(\frac{\dot{S}}{T} \right) dx \geq 0$$

$$\int_{P_t} \left[\rho \frac{\partial \eta}{\partial t} + \frac{\partial}{\partial x} \left(\frac{\dot{S}}{T} - \rho \frac{r}{T} \right) \right] dx \geq 0 \quad \forall P_t \subseteq B_{t_0, t}$$

$$\rho \frac{\partial \eta}{\partial t} + \frac{\partial}{\partial x} \left(\frac{\dot{S}}{T} \right) - \rho \frac{r}{T} \geq 0$$

*Local Eulerian form
of the 2nd Law of TD*

$$\boxed{\rho \frac{\partial \eta}{\partial t} - \rho \frac{r}{T} + \frac{1}{T} \frac{\partial \dot{S}}{\partial x} - \frac{\dot{S}}{T} \frac{\partial T}{\partial x} \geq 0} \quad \forall x \in B_{t_0, t}$$

↑ "Clausius-Duhem Inequality"

sometimes useful to replace $\frac{\partial \dot{S}}{\partial x}$ term with what was found earlier

$$\rho \frac{\partial u}{\partial t} = \sigma \frac{\partial v}{\partial x} + \rho r - \frac{\partial \dot{S}}{\partial x} \Rightarrow \frac{\partial \dot{S}}{\partial x} = \sigma \frac{\partial v}{\partial x} + \rho r - \rho \frac{\partial u}{\partial t}$$

$\boxed{\rho T \frac{\partial \eta}{\partial t} + \sigma \frac{\partial v}{\partial x} - \rho \frac{\partial u}{\partial t} - \frac{\dot{S}}{T} \frac{\partial T}{\partial x} \geq 0}$ ← an alternative form of the "Clausius-Duhem Inequality"

$\underbrace{\hspace{1cm}}$ internal (mechanical) dissipation $\underbrace{\hspace{1cm}}$ thermal (conductive) dissipation

$$\frac{\partial T}{\partial x} \leq 0 \quad \text{because } x \text{ entering heat from warm to cold body} \Rightarrow \therefore \text{this must always be } \geq 0$$

$$\boxed{\rho T \frac{\partial \eta}{\partial t} + \sigma \frac{\partial v}{\partial x} - \rho \frac{\partial u}{\partial t} \geq 0}$$

↑ also Eulerian form

"Clausius-Planck Inequality"
A more stringent form of 2nd Law of TD postulates that mechanical dissipation is non-negative, so entropy production of system is non-negative

Lagrangian Forms:

$$\rho_0 \frac{\partial \eta}{\partial t} - \rho_0 \frac{r}{T} + \frac{1}{T} \frac{\partial \dot{S}}{\partial X} - \frac{\dot{S}}{T} \frac{\partial T}{\partial X} \geq 0 \quad \forall X \in B_{0, t}$$

$$\rho_0 T \frac{\partial \eta}{\partial t} + \sigma \frac{\partial v}{\partial X} - \rho_0 \frac{\partial u}{\partial t} - \frac{\dot{S}}{T} \frac{\partial T}{\partial X} \geq 0 \quad \forall X \in B_{0, t}$$

$$\rho_0 T \frac{\partial \eta}{\partial t} + \sigma \frac{\partial v}{\partial X} - \rho_0 \frac{\partial u}{\partial t} \geq 0 \quad \forall X \in B_{0, t} \quad \left. \right\} \text{Clausius-Planck}$$

Clausius-Duhem

Idea of Introducing 1D-CM:

- + introduce Lagrangian + Eulerian methods of finding solutions
- + introduce Field Equations