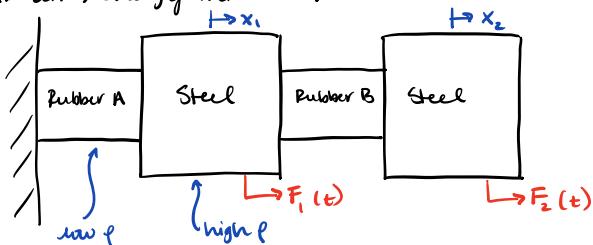


Multidegree-of-freedom Systems will give way to lots of linear equations

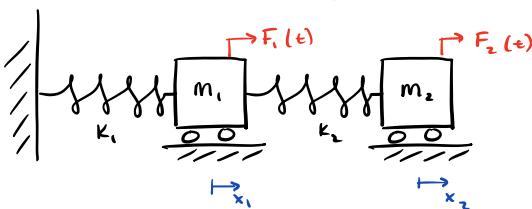
Vibrations - Lec 2

2021-01-28

Dashpots are fictitious systems that we'll frequently use, which turns vibrational (kinetic) energy into heat



- here, we take this system and make an assumption that rubbers stretch (storing potential energy), steel displaces (K.E.)
- we model things w/potential energy as springs and those with KE as rigid bodies



This is called a lumped-parameter approximation

the displacements of  $m_1$  and  $m_2$  are responsible for giving insight into spring forces thereby  $k_1$  and  $k_2$ , which in turn cause an acceleration on  $m_1$  and  $m_2$

Rules for Springs (something more flexible than massive)

$$\begin{array}{l} \text{Spring force } F = kx \\ \text{at tensile strain via} \\ \text{make cut here} \end{array}$$

$$\begin{aligned} \sum F &= ma^0 \\ F - G &= 0 \\ F &= G \end{aligned}$$

ignoring the direction the force is in

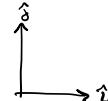
$\rightarrow \oplus$

$$F \leftarrow \text{---} \rightarrow F$$

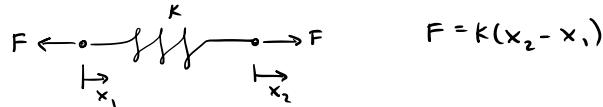
Applied force to right meaning wall pulls back in other direction  
 $\rightarrow$  if  $F \oplus$  then tension,  $\ominus$  compression

so far we've assumed that left end is always fixed, but what if it gets to move independently like the other... this leads to the "real" Hooke's Law

$$\begin{array}{l} F \leftarrow \text{---} \rightarrow F \\ \text{tension} \end{array} \quad F = k\delta \quad \delta = x_2 - x_1$$

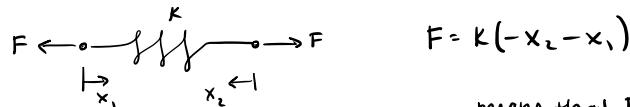


Let's play a game:  $F = Kx$



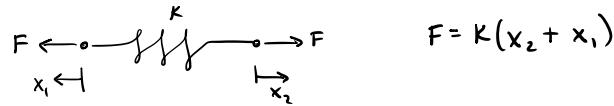
arrows pointing to inside of spring  $\ominus$   
arrows pointing to outside of spring  $\oplus$

$$F = K(x_2 - x_1)$$

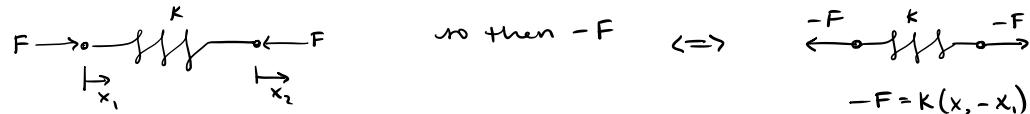


$$F = K(-x_2 - x_1)$$

means that  $F$  is a compressive force



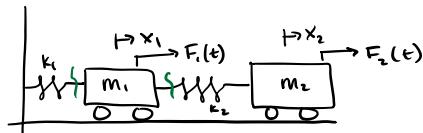
$$F = K(x_2 + x_1)$$



so then  $-F \Leftrightarrow -F = K(x_2 - x_1)$

$$-F = K(x_2 - x_1)$$

Derive Equations of Motion for this System



make a cut:



(make assumption  $x_2 > x_1$ )

$$\sum F_x = m_1 \ddot{x}_1 \quad \rightarrow \hat{\uparrow}$$

$$-k_1 x_1 \hat{i} + F_1(t) \hat{i} + k_2(x_2 - x_1) \hat{i} = m_1 \ddot{x}_1$$

$$(1) \quad m_1 \ddot{x}_1 + (k_1 + k_2)x_1 - k_2 x_2 = F_1(t)$$

$$-k_2(x_2 - x_1) \hat{i} + F_2(t) \hat{i} = m_2 \ddot{x}_2$$

$$(2) \quad m_2 \ddot{x}_2 - k_2 x_1 + k_2 x_2 = F_2(t)$$

The displacement  $x_2$  will influence the elongation of  $k_2$  which in turn affects the acceleration of  $m_1$ , etc.

In order to solve, need to know initial conditions ( $x_1(0)$ ,  $\dot{x}_1(0)$ ,  $x_2(0)$ ,  $\dot{x}_2(0)$ ) and equations of motion ( $F_1(t) + F_2(t)$ ) ... find  $x_1(t)$  and  $x_2(t)$

Putting these two equations of motion into matrix form to solve.

$$\begin{Bmatrix} \dot{x} \end{Bmatrix} = \begin{Bmatrix} x_1(t) \\ x_2(t) \end{Bmatrix} = \text{displacement vector}$$

$\frac{1}{x \text{ vector}}$

$$\begin{Bmatrix} \ddot{x} \end{Bmatrix} = \begin{Bmatrix} \ddot{x}_1(t) \\ \ddot{x}_2(t) \end{Bmatrix}$$

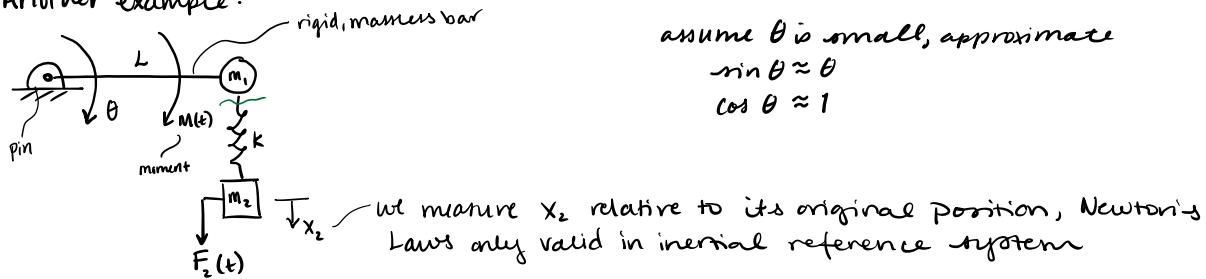
$$\underbrace{\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix}}_{\substack{2 \times 2 \\ \text{Mass Matrix}}} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \underbrace{\begin{bmatrix} (k_1+k_2) & -k_2 \\ -k_2 & k_1 \end{bmatrix}}_{\substack{2 \times 2 \\ \text{Stiffness Matrix}}} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} F_1(t) \\ F_2(t) \end{Bmatrix}$$

① first row is eq. (1)  
② second row is eq. (2)

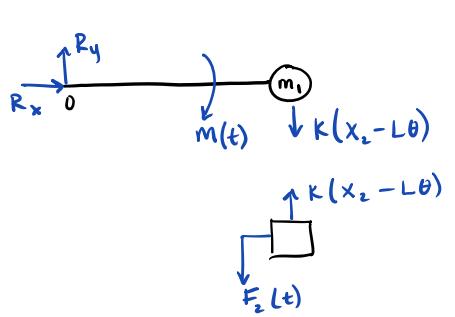
$\underbrace{\text{Force Vector}}$

2 equations of motion, 2 Degree of Freedom system

Another example:



FBD: can assume vertical displacement of  $m_1$  to be  $L\theta$  because small  $\theta$  and that  $x_2 > L\theta$



$$\sum M_o = \overbrace{J_o \ddot{\theta}}^{\text{max moment about } 0}$$

take  $\theta$  clockwise  $\oplus$ ,  $\therefore m(t) \oplus$

$$m(t) + KL(x_2 - L\theta) = J_o \ddot{\theta}$$

$J_o = m_1 L^2$  since  $m_1$  pt. mass

$$(1) \quad m(t) + KL(x_2 - L\theta) = m_1 L^2 \ddot{\theta}$$

$$(2) \quad F_2(t) - K(x_2 - L\theta) = m_2 \ddot{x}_2$$

$$\begin{bmatrix} J_o & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} \ddot{\theta} \\ \ddot{x}_2 \end{Bmatrix} + \begin{bmatrix} KL^2 & -KL \\ -KL & K \end{bmatrix} \begin{Bmatrix} \theta \\ x_2 \end{Bmatrix} = \begin{Bmatrix} m(t) \\ F_2(t) \end{Bmatrix}$$

### Supplemental Book Notes

Procedure to derive equations of motion of multidegree-of-freedom system using Newton's 2nd Law of Motion

1. set up coordinate system, assume suitable  $\oplus$  directions
2. determine static equilibrium configuration and measure displacements of masses + rigid bodies from respective static equilibrium positions
3. draw FBD
4. apply Newton's 2nd to each mass or rigid body shown by FBD as

$$m_i \ddot{x}_i = \sum_j F_{ij} \quad (\text{for mass } m_i)$$

-or-

$$J_i \ddot{\theta}_i = \sum_j M_{ij} \quad (\text{for rigid body of inertia } J_i)$$

where  $\sum F_{ij}$  denotes the sum of all forces acting on mass  $m_i$  and  $\sum M_{ij}$  indicates sum of moments of all forces about suitable axis acting on the rigid body of mass moment of inertia  $J_i$

$$\begin{matrix} \downarrow \text{mass} & \downarrow \text{damping} & \downarrow \text{stiffness} \\ [m] \ddot{x} + [c] \dot{x} + [k] x = \vec{F} \end{matrix}$$

in general form, mass, damping, and stiffness matrices given by :

$\begin{matrix} \text{mass} \\ \text{matrix} \end{matrix}$	$[m] = \begin{bmatrix} m_{11} & m_{12} & m_{13} & \dots & m_{1n} \\ m_{21} & m_{22} & m_{23} & \dots & m_{2n} \\ \vdots & & & & \\ m_{n1} & m_{n2} & m_{n3} & \dots & m_{nn} \end{bmatrix}$	displacement: $\vec{x} = \begin{Bmatrix} x_1(t) \\ x_2(t) \\ \vdots \\ x_n(t) \end{Bmatrix}$
$\begin{matrix} \text{damping} \\ \text{matrix} \end{matrix}$	$[c] = \begin{bmatrix} c_{11} & c_{12} & c_{13} & \dots & c_{1n} \\ c_{21} & c_{22} & c_{23} & \dots & c_{2n} \\ \vdots & & & & \\ c_{n1} & c_{n2} & c_{n3} & \dots & c_{nn} \end{bmatrix}$	velocity: $\dot{\vec{x}} = \begin{Bmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \\ \vdots \\ \dot{x}_n(t) \end{Bmatrix}$
$\begin{matrix} \text{stiffness} \\ \text{matrix} \end{matrix}$	$[k] = \begin{bmatrix} k_{11} & k_{12} & k_{13} & \dots & k_{1n} \\ k_{21} & k_{22} & k_{23} & \dots & k_{2n} \\ \vdots & & & & \\ k_{n1} & k_{n2} & k_{n3} & \dots & k_{nn} \end{bmatrix}$	acceleration: $\ddot{\vec{x}} = \begin{Bmatrix} \ddot{x}_1(t) \\ \ddot{x}_2(t) \\ \vdots \\ \ddot{x}_n(t) \end{Bmatrix}$
		force: $\vec{F} = \begin{Bmatrix} F_1(t) \\ F_2(t) \\ \vdots \\ F_n(t) \end{Bmatrix}$

for an undamped system, the equations of motion reduce to

$$[m] \ddot{\vec{x}} + [k] \vec{x} = \vec{F}$$