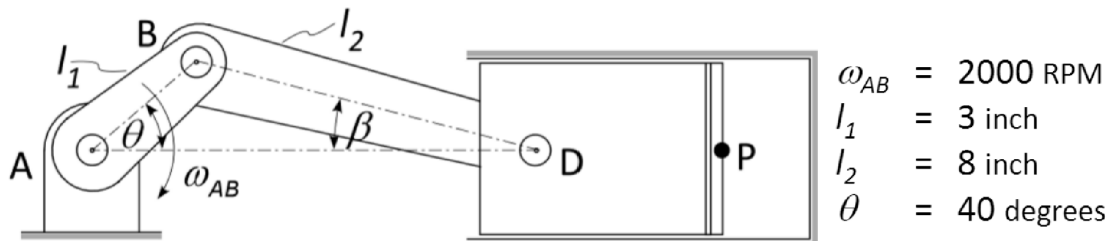


Problem 1

General Motion - Velocity

Example: In the piston system shown, the crank AB has a constant clockwise angular velocity of 2000 RPM.



Determine the velocity of point P on the piston for the configuration parameters given above

Review

The last lecture is about **Instantaneous center** (IC) of a rigid body. The key feature of instantaneous center is:

1. It is not constant in time, and not necessarily on the object.
2. Its velocity is zero at this instant. All points on the body can be described as:

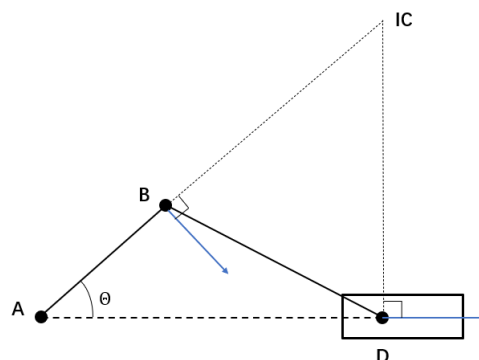
$$\vec{v}_A = \vec{\omega} \times \vec{r}_{A/IC}$$

Where ω is the angular velocity around instantaneous center. Therefore, for every point on the rigid body, its velocity is perpendicular to the position vector from instantaneous center to itself.

We can find the instantaneous center by using physical features of the rigid body (rotating around a fixed point), or we can determine instantaneous center by know velocities, positions and angular velocities.

Analysis

For this problem, we have already done analysis and found $\vec{v}_D = \vec{v}_P$. To find \vec{v}_D , we can apply instantaneous center method on rod BD:



We used the feature "all points on the rigid body have perpendicular velocity to IC".

Solution

With the graph we have from analysis, we can first write down B and D's velocities described with IC:

$$\vec{v}_B = \vec{\omega} \times \vec{r}_{B/IC} \quad (1)$$

$$\vec{v}_D = \vec{\omega} \times \vec{r}_{D/IC} \quad (2)$$

And we can use geometric features of the system to solve $\vec{r}_{B/IC}$ and $\vec{r}_{D/IC}$

$$\begin{aligned}\vec{r}_{B/IC} &= -(\bar{AD} - \frac{l_1}{\cos\theta})\hat{i} - (\frac{\bar{AD}}{\tan\theta} - l_1 \sin\theta)\hat{j} \\ &= -7.76\hat{i} - 6.51\hat{j} \\ \vec{r}_{D/IC} &= -\frac{\bar{AD}}{\tan\theta} \cdot \hat{j} \\ &= -8.44\hat{j}\end{aligned}$$

Also, B is rotating around A, we have:

$$\vec{v}_B = \vec{v}_A + \vec{\omega}_{AB} \times \vec{r}_{B/A} \quad (3)$$

Where $\vec{v}_A = 0$ and $\vec{r}_{B/A} = l_1 \cos\theta \hat{i} + l_1 \sin\theta \hat{j}$. and we can solve that (just like last Friday)
 $\vec{v}_B = 403.9\hat{i} - 481.3\hat{j} (ft/s)$

Plugging into equation (1), we found that the remaining unknowns are: ω, \vec{v}_D , with 2 equations and 2 unknowns, we can plug in to solve.

First, $\omega = 403.9/6.51 = 62.03 rad/s$

Then plug in (2), $\vec{v}_D = 62.03\hat{k} \times (-8.44\hat{j}) = 523.4 in/s \hat{i}$

Which is the same with our solution from last Friday.

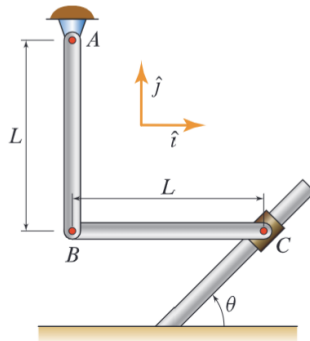
Problem 2

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Solutions Manual

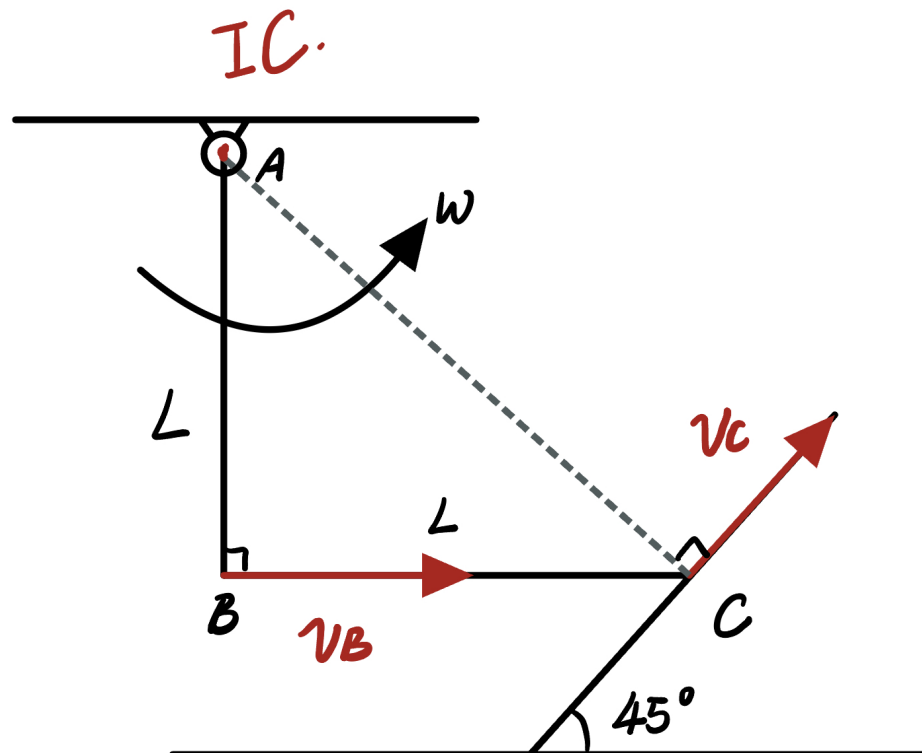
Problem 6.41

At the instant shown bars AB and BC are perpendicular to each other, and bar BC is rotating counter-clockwise at 20 rad/s . Letting $L = 2.5 \text{ ft}$ and $\theta = 45^\circ$, determine the angular velocity of bar AB as well as the velocity of the slider C .



Note: AB and BC are fixed together so they are always perpendicular to each other.

Analysis



Repeat what we have done for the first problem: find IC. In this case, the problem set is easier since we have been told BC is rotating around A . so the instantaneous center is A .

Solution

The angular velocity of C w.r.t. A is $\omega_{BC} = 20\text{rad/s} \cdot \hat{k}$

$$\begin{aligned}\vec{v}_C &= \vec{v}_A + \vec{\omega} \times \vec{r}_{C/A} \\ &= 0 + 20\hat{k} \times (2.5\hat{i} - 2.5\hat{j}) \\ &= 50\hat{i} + 50\hat{j}\end{aligned}$$

For B, we have

$$\begin{aligned}\vec{v}_B &= \vec{v}_{IC} + \vec{\omega} \times \vec{r}_{B/IC} \\ \vec{v}_B &= \vec{v}_A + \vec{\omega}_{AB} \times \vec{r}_{B/A}\end{aligned}$$

It is obvious that $\vec{\omega}_{AB} = \vec{\omega} = 20\text{rad/s} \cdot \hat{k}$