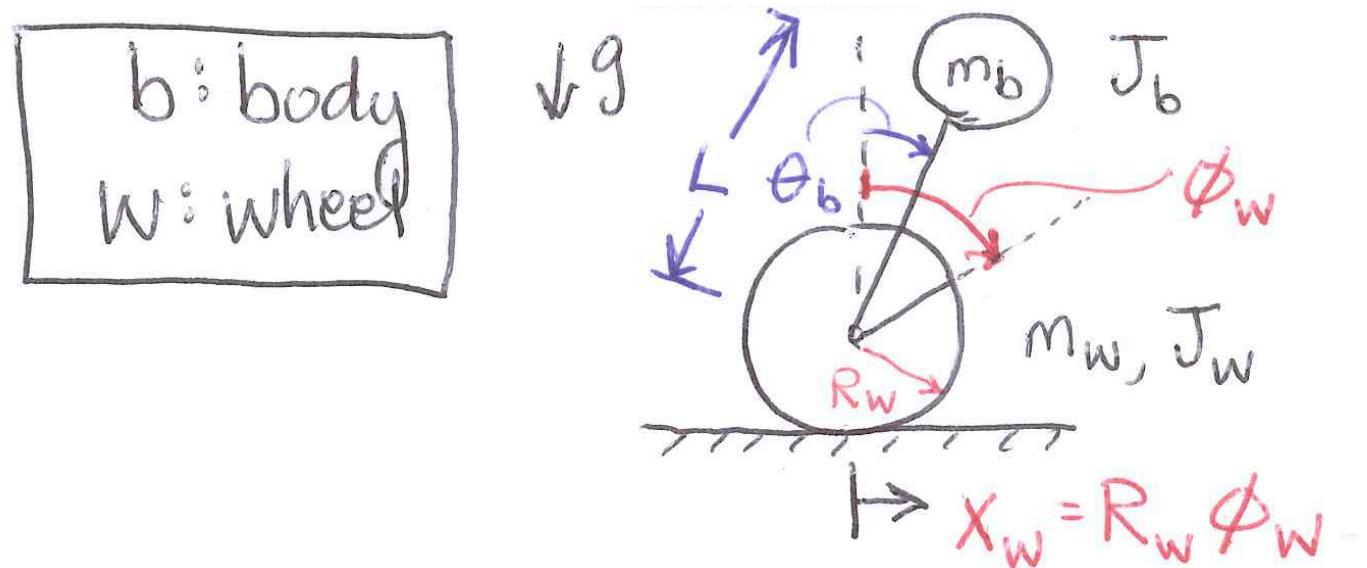


MATLAB's “Symbolic Toolbox”

Symbolic differentiation, to get Lagrangian EOMs

Use this document as a guide for HW 4

Balancing Inverted Pendulum Robot :



(...aka the “Segway” style system)

Here's an approach:

Note: I used "q" instead of "little xi" for G.C.'s here... Don't get confused.
Various references may use either one. Roboticists like to use "q".

1. Define needed variables and constraints:

- a) Define the Generalized Coordinates (G.C.'s), q_1 through q_n .
- b) Define geometry of masses and inertia elements.
- c) Differentiate, to define velocities (for kinetic energy).

2. Define kinetic energy, T

3. Define potential energy, V

4. Then the Lagrangian is: $L = T - V$

5. The lefthand of the i^{th} EOM is:

6. Righthand side of each EOM is "Big Xi".

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = \Xi_i$$

Here is annotated matlab code to generate EOM's...

- Requires an additional m-file called:
fulldiff.m

Fulldiff.m performs the “chain rule” for differentiation of variables with time derivatives.

0. First, define symbolic variables you will need...

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```
% segway_eom.m  
%  
% "Segway-Style" Inverted Pendulum: Equations of Motion.  
% (See slides from Lecture 11 for a visual description of the system.)  
% Katie Byl, Nov. 2, 2017. ECE/ME 179D, UCSB.  
  
clear all; format compact % compact produces single-spaced output  
  
% Define symbolic variables in matlab:  
syms phiw thetab L mb Jb mw Jw Rw g b tau
```

Modify “segway_eom.m” for HW 4 part a

1. Define needed variables and constraints:

- a) Define the Generalized Coordinates (G.C.'s), q_1 through q_n .
- b) Define geometry of masses and inertia elements.
- c) Differentiate, to define velocities (for kinetic energy).

Roboticians like to use “ q ” (for joints), instead of “ x_i ”.

```

% 1a. GC's (generalized coordinates), and their derivatives:
GC = [{phiw},{thetab}];% Using ABSOLUTE angles here
dphiw = fulldiff(phiw,GC);% time derivative. GC are variables (over time)
dthetab = fulldiff(thetab,GC);

% 1b. Geometry of the masses/inertias, given GC's are freely changing...
xw = R*w*phiw;
xb = xw+L*sin(thetab);
yw = 0;
yb = L*cos(thetab);

% 1c. Define any required velocity terms (for masses):
dxw = fulldiff(xw,GC);
dxb = fulldiff(xb,GC);
dyb = fulldiff(yb,GC);

```

1. Define needed variables and constraints:

- a) Define the Generalized Coordinates (G.C.'s), q_1 through q_n .
- b) Define geometry of masses and inertia elements.
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- 2. Define kinetic energy, T**
- 3. Define potential energy, V**
- 4. Then the Lagrangian is: $L = T - V$**

% 2. Kinetic Energy:
 $T = (1/2)*(mw*dxw^2 + Jw*dphiw^2 + mb*(dxb^2 + dyb^2) + Jb*dthetab^2)$

% 3. Potential Energy:
 $V = mb*g*yb$

% 4. Lagrangian:
 $L = T - V$

2. Define kinetic energy, T
3. Define potential energy, V
4. Then the Lagrangian is: $L = T - V$

5. The **lefthand** of the i^{th} EOM is:
6. Righthand side of each EOM is “**Big Xi**”.

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = \Xi_i$$

% 5. EOMs:

```
eq1 = fulldiff(diff(L,dphiw),GC) - diff(L,phiw)
eq2 = fulldiff(diff(L,dthetab),GC) - diff(L,thetab);
eq2 = simplify(eq2)
```

% 6. \dot{X}_i : non-conservative terms

```
Xi1 = tau - b*(dphiw-dthetab)% Motor torque tau, and back emf damping b
Xi2 = -tau + b*(dphiw-dthetab)% (equal and opposite to above)
```

Equations of Motion (EOM's) are then:

eq1 = \dot{X}_1

eq2 = \dot{X}_2

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = \Xi_i$$

1. The **lefthand** of the i^{th} EOM is:

2. Righthand side of each EOM is “**Big \dot{X}_i** ”.

Output from segway_eom.m :
 (Katie Byl, 2012)

```

T =
(Jb*dthetab^2)/2 + (Jw*dphiw^2)/2 + (mb*((Rw*dphiw + L*dthetab*cos(theta)) ^2 + L^2*dthetab^2*sin(theta)^2))/2 +
(Rw^2*dphiw^2*mw)/2

V =
L*g*mb*cos(theta)

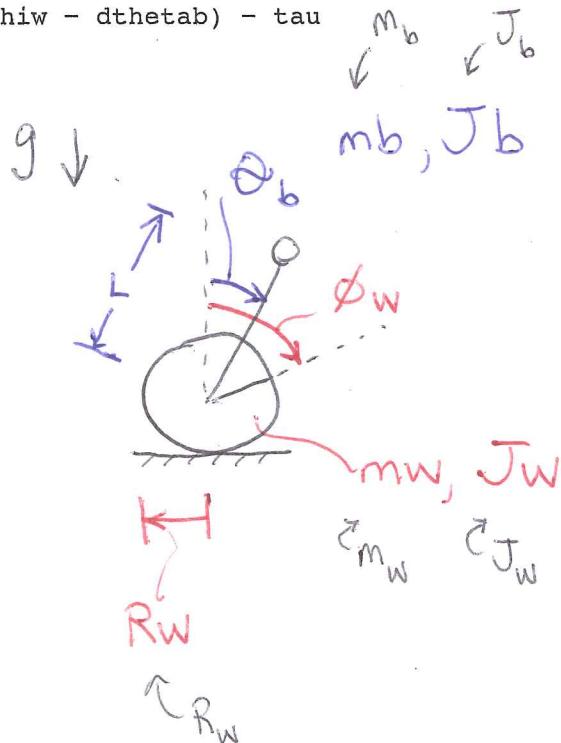
L =
(Jb*dthetab^2)/2 + (Jw*dphiw^2)/2 + (mb*((Rw*dphiw + L*dthetab*cos(theta)) ^2 + L^2*dthetab^2*sin(theta)^2))/2 +
(Rw^2*dphiw^2*mw)/2 - L*g*mb*cos(theta)

eq1 =
- L*Rw*mb*sin(theta)*dthetab^2 + d2phiw*(Jw + Rw^2*mb + Rw^2*mw) + L*Rw*d2thetab*mb*cos(theta)

eq2 =
Jb*d2thetab + L^2*d2thetab*mb - L*g*mb*sin(theta) + L*Rw*d2phiw*mb*cos(theta)

Xi1 =
tau - b*(dphiw - dthetab)

Xi2 =
b*(dphiw - dthetab) - tau
    
```



θ_b \rightarrow theta	$\phi_w \rightarrow \phi_i w$
$\dot{\theta}_b$ \rightarrow dtheta	$\dot{\phi}_w \rightarrow d\phi_i w$
$\ddot{\theta}_b$ \rightarrow d2theta	$\ddot{\phi}_w \rightarrow d2\phi_i w$