

“Implementation of the Dynamic Time-History Linear Seismic Analysis in MatLab for 3D frames”

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Overview

This software is the implementation of the Dynamic Time-History Seismic Analysis for 3D frames fully in MatLab. The functions involve the modal analysis and the method for the numerical solution of Second Order differential equations in time steps. Two structural examples are presented, both of which are analysed with damping effects and without them. An artificial acceleration history vector is given as entry data to assess the equivalent inertial forces in the different DOF of the structures at each time step from the modes and frequencies primarily obtained by solving the system $[K - \omega^2 M]\phi = 0$. The numerical solution is entirely linear, that is without considering non-linearities of the elements' materials.

Keywords: Dynamic Time-History, Seismic Analysis, Linear solution, 3D Frames

1 The Linear Time-History Seismic Analysis

This analysis consists of analysing time-step wise the structural response of SDOF or MDOF systems before an acceleration excitation at the base of the structures (1). The equivalent inertial forces at any of the system DOF's are obtained by solving the classical eigenvalue and eigenvector problem without considering the damping matrix (2), even when damping effects are considered. As the term *linear* in the method's name implies, the displacement response before such obtained equivalent inertial forces is carried out through a linear static analysis at each time-step.

$$[M]\{\ddot{U}\} + [C]\{\dot{U}\} + [K]\{U\} = \{F(t)\} = \frac{\phi_i^T M[1]}{M_*} a(M\phi_i) \quad (1)$$

$$([K] - \omega^2[M])[\phi] = \{0\} \quad (2)$$

1.1 The consistent mass method

The construction of the mass matrix is done through a Finite Element method approach in which an element mass matrix Me as the one shown below is built for all elements and assembled into a global one as for the stiffness matrix.

1.2 Structural Rayleigh damping

For MDOF the damping matrix C can be computed as a function of the mass matrix and the stiffness matrix by using the Rayleigh's coefficients α and β as in (3), where α and β are determined by solving

$$\frac{\rho AL}{420} \begin{bmatrix} 140 & 0 & 0 & 0 & 0 & 0 & 70 & 0 & 0 & 0 & 0 & 0 \\ & 156 & 0 & 0 & 0 & 22L & 0 & 54 & 0 & 0 & 0 & -13L \\ & & 156 & 0 & -22L & 0 & 0 & 0 & 54 & 0 & 13L & 0 \\ & & & 140r_x^2 & 0 & 0 & 0 & 0 & 0 & 70r_x^2 & 0 & 0 \\ & & & & 4L^2 & 0 & 0 & 0 & -13L & 0 & -3L^2 & 0 \\ & & & & & 4L^2 & 0 & 13L & 0 & 0 & 0 & -3L^2 \\ & & & & & & 140 & 0 & 0 & 0 & 0 & 0 \\ & & & & & & & 156 & 0 & 0 & 0 & -22L \\ & & & & & & & & 156 & 0 & 22L & 0 \\ & & & & & & & & & 140r_x^2 & 0 & 0 \\ & & & & & & & & & & 4L^2 & 0 \\ & & & & & & & & & & & 4L^2 \end{bmatrix}$$

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the linear system of equations (4) from the selection of a pair of chosen modal damping factors ξ_1, ξ_2 and a corresponding pair of frequencies ω_1, ω_2 :

$$C_e = \alpha \cdot M_e + \beta \cdot K_e \quad (3)$$

$$\begin{aligned} \xi_1 &= \frac{1}{2\omega_1} \alpha + \frac{\omega_1}{2} \beta \\ \xi_2 &= \frac{1}{2\omega_2} \alpha + \frac{\omega_2}{2} \beta \end{aligned} \quad (4)$$

Even though Rayleigh damping has the clear disadvantage of not allowing realistic damping for all modes of interest (in case there are far more than two in question) it provides a great solution approach when only a few modes are sought to be analysed.

2 Dynamic acceleration

When aiming at solving a dynamic system subject to an earthquake excitation the accelerations in time are required, for which purpose, a real or artificial accelerogram as the one shown in **Fig. 1** should be given as entry data.

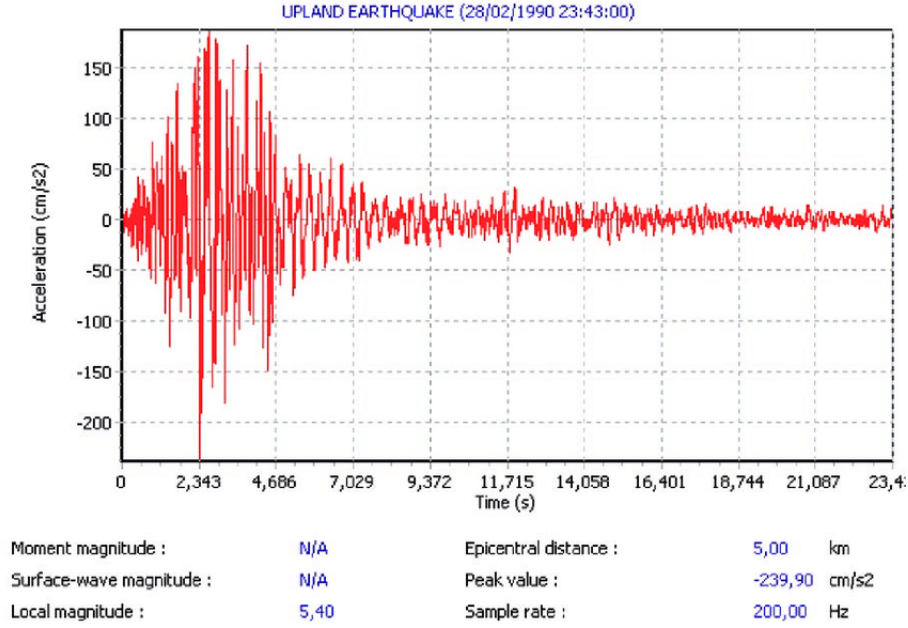


Figure 1. Example of an accelerogram for an earthquake.

From such accelerations in time the vector $f(t)$ in (1) corresponding to the DOF forces in time could be obtained by solving the eigenvalue/eigenvector problem for the modes of vibration in function of such accelerations.

Note that the direction of the acceleration at any time will be given by the chosen mode of vibration.

3 Numerical evaluation of dynamic response

When considering coupled equations of motion in which damping is integrated then one must opt to apply other procedures different than *mode-superposition* methods to solve them, such as **direct integration** methods that aim mainly at the solution of second-order differential equations (see [2]).

3.1 Beta-Newmark method

The Newmark- β method initially introduced as the *Constant average acceleration method* is presented in a form appropriate for direct integration of linear second-order systems, such as those whose damping matrix cannot be diagonalized by the system of normal modes. This method in contrast with the *Central difference method* - which is probably the most known and fundamental algorithm for the approximate numerical solution second-order differential ordinary equations - provides **unconditionally stable** solutions, that is, that it is not entirely required to have qualitative knowledge a priori of the frequencies presented in the system's response to select a proper time step [2].

The basis of the Newmark- β is the assumption that the system displacement vector u_{n+i} and its derivative \dot{u}_{n+1} can be written in the form (5), which can be viewed as a type of generalized Taylor series approximation. Thus, the Newmark- β method is said to be an implicit method with the discrete

equation of motion being enforced at the time step t_{n+1} as (6):

$$\begin{aligned}\dot{u}_{n+1} &= \dot{u}_n + ((1 - \gamma)\ddot{u}_n + \gamma\ddot{u}_{n+1})h \\ u_{n+1} &= u_n + \dot{u}_nh + (1 - 2\beta)\frac{h^2}{2}\ddot{u}_n + \beta h^2\ddot{u}_{n+1}\end{aligned}\tag{5}$$

$$M\ddot{u}_{n+1} + C\dot{u}_{n+1} + Ku_{n+1} = p(t_{n+1})\tag{6}$$

3.1.1 Algorithm

Algorithm 3.1: Pseudo-code for the Newmark- β method for 2nd Order Ordinary Differential Equations.

1. Input M,K and C (if any)
 2. Input initial conditions u_0, v_0 and compute the initial acceleration a_0 as:

$$a_0 = M^{-1}(p_0 - Cv_0 - Ku_0)$$
 3. Set the time step length h
 4. Initialize loop iteration:

For $n = 1$ **To** $n = n - steps$

Compute the right hand side term RHS as:

$$RHS_n = -Ku_n - (C + hK)\dot{u}_n - [h(1 - \gamma)C + \frac{h^2}{2}(1 - 2\beta)K]\ddot{u}_n$$

Solve for the second derivatives at the next time step:

$$[M + \gamma C + \beta h^2 K]\ddot{u}_{n+1} = RHS_n$$

Evaluate the set of displacements and velocities for the next time step:

$$\dot{u}_{n+1} = \dot{u}_n + ((1 - \gamma)\ddot{u}_n + \gamma\ddot{u}_{n+1})h$$

$$u_{n+1} = u_n + \dot{u}_nh + (1 - 2\beta)\frac{h^2}{2}\ddot{u}_n + \beta h^2\ddot{u}_{n+1}$$

End For
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References

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- [2] Fundamentals of Structural Dynamics, Roy R. Craig Jr., Andrew J. Kurdila, 2nd Ed., John Wiley & Sons, Inc., 2006