

# EE2101 - Control Systems

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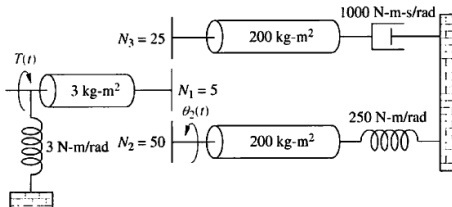
## 1 Problem Statement

## 2 Solution

- Rotational Mechanical Systems
  - Gears
- Problem solving

# Problem

Find the transfer function  $G(s) = \frac{\theta_2(s)}{T(s)}$ ,  
for rotational mechanical system shown in the figure.



# Rotational Mechanical Systems

**Rotational mechanical systems** move about a fixed axis. These systems mainly consist of three basic elements. They are moment of inertia, torsional spring and a dashpot.

- If a torque is applied to a rotational mechanical system, then it is opposed by opposing torques due to moment of inertia, elasticity and friction of the system.
- Since the applied torque and the opposing torques are in opposite directions, **the algebraic sum of torques acting on the system is zero.**

We know that, for:

•**Spring:**

$$\text{Torque, } T(t) = K\theta(t)$$

$$\text{Impedance, } T(s)/\theta(s) = K \quad (2.1)$$

•**Moment of Inertia:**

$$\text{Torque, } T(t) = J\left(\frac{d\omega}{dt}\right) = J\left(\frac{d^2\theta}{dt^2}\right)$$

$$\text{Impedance, } T(s)/\theta(s) = Js^2 \quad (2.2)$$

**•Dashpot:**

a) One free end (i.e. if one end is fixed to a reference):

$$T(t) = D\omega(t) = D\left(\frac{d\theta}{dt}\right)$$

b) For two free ends,

$$T(t) = D\left(\frac{d(\theta_1 - \theta_2)}{dt}\right)$$

And,

$$\text{Impedance, } T(s)/\theta(s) = Ds \quad (2.3)$$

# Gears

We will assume that connected gears fit perfectly together.

Consider input gear has radius  $r_1$  and is rotated by  $\theta_1$ ; output gear has radius  $r_2$ , then  $\theta_2 = (r_1/r_2)\theta_1 = (N_1/N_2)\theta_1$ .

where,  $N_1$  and  $N_2$  are no. of teeth of gears respectively and radius is proportional to no. of teeth.

No. of teeth is inversely proportional to torque.

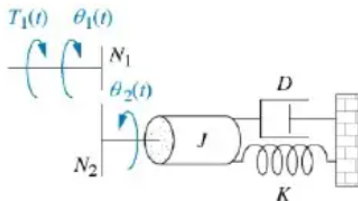
Assuming there is no energy loss, we have  $T\theta = \text{constant}$ ..i.e.

$$T_1\theta_1 = T_2\theta_2.$$

$$\text{And, } T_1N_2 = T_2N_1$$

## Solving the problem

To find the relation between  $T_1$  and  $\theta_1$ :



Consider the torque due to the damper on shaft2:  $T_{D2} = Ds\theta_2(s)$

$$T_{D2} = Ds \frac{N_1}{N_2} \theta_1(s)$$

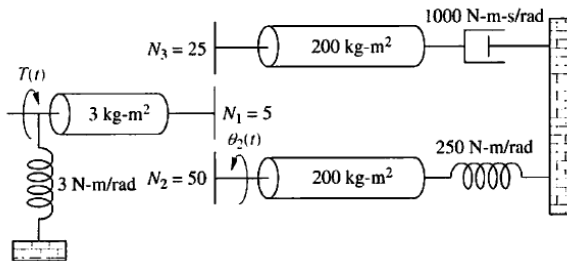
And,

$$T_{D1} = (N_1/N_2) T_{D2}$$

$$T_{D1} = \frac{N_1}{N_2} \left[ Ds \frac{N_1}{N_2} \theta_1(s) \right] = \left( \frac{N_1}{N_2} \right)^2 Ds \theta_1(s)$$



## Reflecting Impedances to $\theta_2$ on the system:



Due to springs  $\Rightarrow 250 + 3(50/5)^2$ .

Due to dashpot  $\Rightarrow [1000(\frac{(5)(50)}{(25)(5)})^2]s$ .

Due to moment of inertia

$$\Rightarrow [200 + 3(50/5)^2 + 200(\frac{(5)(50)}{(25)(5)})^2]s^2.$$

Adding up all the effects,

$$[[250 + 3(50/5)^2] + [1000[(\frac{(5)(50)}{(25)(5)})^2]s + [200 + 3(50/5)^2 + 200(\frac{(5)(50)}{(25)(5)})^2s^2] \theta_2(s) = (50/5)[T(s)] \quad (2.4)$$

$$\Rightarrow [(550) + (4000)s + (500 + 800)s^2](\theta_2(s)) = 10[T(s)]$$

$$\Rightarrow \frac{(\theta_2(s))}{T(s)} = \frac{10}{1300s^2 + 4000s + 550}$$

Therefore, the required transfer function  $G(s)$  is

$$\frac{\theta_2(s)}{T(s)} = \frac{1}{130s^2 + 400s + 55}.$$