

CG2023 Selected Tutorial Questions

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Tutorial 5

Q2

Q.2 Solve the following linear second order differential equation using Laplace Transform:

$$\frac{d^2 y(t)}{dt^2} + 4 \frac{dy(t)}{dt} + 3y(t) = 2u(t)$$

where $y(0^-) = 1$ and $y'(0^-) = 0$.

Solution to Q.2

Given: $\frac{d^2 y(t)}{dt^2} + 4\frac{dy(t)}{dt} + 3y(t) = 2u(t)$, $y(0^-) = 1$ and $y'(0^-) = 0$ $\mathcal{L}\left\{\frac{df(t)}{dt}\right\} = s\tilde{F}(s) - f(0^-)$

Apply Laplace transform: $\mathcal{L}\left\{\frac{d^2 f(t)}{dt^2}\right\} = s^2\tilde{F}(s) - sf(0^-) - f'(0^-)$

$$s^2 Y(s) - \underbrace{y(0^-)}_1 s - \underbrace{y'(0^-)}_0 + 4sY(s) - 4\underbrace{y(0^-)}_1 + 3Y(s) = \frac{2}{s}$$

$$(s^2 + 4s + 3)Y(s) = \frac{2}{s} + s + 4$$

$$Y(s) = \frac{s^2 + 4s + 2}{s(s^2 + 4s + 3)} = \frac{s^2 + 4s + 2}{s(s+1)(s+3)} = \frac{2}{3s} + \frac{1}{2(s+1)} - \frac{1}{6(s+3)}$$

Apply inverse Laplace transform:

$$y(t) = \frac{2}{3}\mathcal{L}^{-1}\left\{\frac{1}{s}\right\} + \frac{1}{2}\mathcal{L}^{-1}\left\{\frac{1}{s+1}\right\} - \frac{1}{6}\mathcal{L}^{-1}\left\{\frac{1}{s+3}\right\} = \left[\frac{2}{3} + \frac{1}{2}e^{-t} - \frac{1}{6}e^{-3t}\right]u(t)$$



Tutorial 6

Q3

At steady-state, the input to a 1st-order system $\tilde{H}(s) = \frac{K}{sT + 1}$ is $x(t) = 0.4 + 2\sin(0.25t)$, and $y(t)$ is the corresponding system output.

- (a) Find an expression for $y(t)$ in terms K and T .
- (b) If $y(t) = 2 + 8\sin(0.25(t - 2.574))$, find the values of K and T .

(a)

Sinusoidal Response at Steady-State

$$x(t) = Ae^{j(\omega_0 t + \psi)} \rightarrow \tilde{H}(j\omega) \rightarrow y(t) = A|\tilde{H}(j\omega_0)|e^{j(\omega_0 t + \psi + \angle\tilde{H}(j\omega_0))}$$

Frequency response of the system $\tilde{H}(s) = \frac{K}{sT + 1}$ is given by

$$\tilde{H}(j\omega) = \tilde{H}(s)\Big|_{s=j\omega} = \frac{K}{j\omega T + 1} \rightarrow \begin{cases} |\tilde{H}(j\omega)| = \frac{K}{[\omega^2 T^2 + 1]^{1/2}} & \dots\dots\dots \text{Magnitude response} \\ \angle\tilde{H}(j\omega) = -\tan^{-1}\left(\frac{\omega T}{1}\right) & \dots\dots\dots \text{Phase response} \end{cases}$$

The system input $x(t) = 0.4 + 2\sin(0.25t)$ consists of a DC component which has a frequency of 0 rad/s and a sine wave of frequency 0.25 rad/s. The sinusoidal responses of the system at these frequencies are:

$$@ 0 \text{ rad/s} \rightarrow \begin{cases} |\tilde{H}(j0)| = K \\ \angle \tilde{H}(j\omega) = 0 \end{cases} \quad \text{and} \quad @ 0.25 \text{ rad/s} \rightarrow \begin{cases} |\tilde{H}(j0.25)| = \frac{K}{[0.25^2 T^2 + 1]^{1/2}} \\ \angle \tilde{H}(j0.25) = -\tan^{-1}\left(\frac{0.25T}{1}\right) \end{cases}$$

Hence,

$$\begin{aligned} y(t) &= 0.4|\tilde{H}(j0)| + 2|\tilde{H}(j0.25)|\sin(0.25t + \angle \tilde{H}(j0.25)) \\ &= 0.4K + \frac{2K}{(0.25^2 T^2 + 1)^{1/2}} \sin\left(0.25t - \tan^{-1}\left(\frac{0.25T}{1}\right)\right) \quad \dots\dots\dots (\clubsuit) \end{aligned}$$

(b)

$$\begin{aligned} y(t) &= 0.4 \left| \tilde{H}(j0) \right| + 2 \left| \tilde{H}(j0.25) \right| \sin(0.25t + \angle \tilde{H}(j0.25)) \\ &= 0.4K + \frac{2K}{(0.25^2 T^2 + 1)^{1/2}} \sin\left(0.25t - \tan^{-1}\left(\frac{0.25T}{1}\right)\right) \quad \dots\dots\dots (\clubsuit) \end{aligned}$$

Comparing (\clubsuit) with the given output $y(t) = 2 + 8\sin(0.25(t - 2.574))$, we get

$$0.4K = 2 \quad \rightarrow \quad K = 5$$

$$\left. \begin{array}{l} \text{Using } \frac{2K}{\sqrt{0.25^2 T^2 + 1}} = 8 \text{ or} \\ \tan^{-1}(0.25T) = 0.25 \times 2.574 \end{array} \right\} \rightarrow T = 3$$

Q4

At steady-state, $y(t) = 5 + 4\cos(3t + 0.3142)$ is the output of a 1st-order system $\tilde{H}(s) = \frac{20}{s + 4}$. Derive the corresponding system input $x(t)$.

$$\left. \begin{aligned}
 x(t) = Ae^{j(\omega_o t + \psi)} &\longrightarrow \boxed{\tilde{H}(j\omega)} \longrightarrow y(t) = A|\tilde{H}(j\omega_o)|e^{j(\omega_o t + \psi + \angle\tilde{H}(j\omega_o))} \\
 x(t) = A\cos(\omega_o t + \psi) &\longrightarrow \boxed{\tilde{H}(j\omega)} \longrightarrow y(t) = A|\tilde{H}(j\omega_o)|\cos(\omega_o t + \psi + \angle\tilde{H}(j\omega_o)) \\
 x(t) = A\sin(\omega_o t + \psi) &\longrightarrow \boxed{\tilde{H}(j\omega)} \longrightarrow y(t) = A|\tilde{H}(j\omega_o)|\sin(\omega_o t + \psi + \angle\tilde{H}(j\omega_o))
 \end{aligned} \right\} \quad (7.20)$$

Steady-state Sinusoidal Response of a LTI System in ω -domain

Frequency response of the system $\tilde{H}(s) = \frac{20}{s+4}$ is given by

$$\tilde{H}(j\omega) = \tilde{H}(s)\big|_{s=j\omega} = \frac{20}{j\omega + 4} \rightarrow \begin{cases} |\tilde{H}(j\omega)| = \frac{20}{[\omega^2 + 16]^{1/2}} & \dots\dots \text{Magnitude response} \\ \angle\tilde{H}(j\omega) = -\tan^{-1}\left(\frac{\omega}{4}\right) & \dots\dots\dots \text{Phase response} \end{cases}$$

The system output $y(t) = 5 + 4\cos(3t + 0.3142)$ consists of a DC component which has a frequency of 0 rad/s and a cosine wave of frequency 3 rad/s. The sinusoidal responses of the system at these frequencies are:

$$@ 0 \text{ rad/s} \rightarrow \begin{cases} |\tilde{H}(j0)| = 5 \\ \angle \tilde{H}(j\omega) = 0 \end{cases} \quad \text{and} \quad @ 3 \text{ rad/s} \rightarrow \begin{cases} |\tilde{H}(j3)| = \frac{20}{[9+16]^{1/2}} = 4 \\ \angle \tilde{H}(j3) = -\tan^{-1}\left(\frac{3}{4}\right) = -0.6435 \end{cases}$$

Hence, the input at steady-state is

$$\begin{aligned} x(t) &= \frac{5}{|\tilde{H}(j0)|} + \frac{4}{|\tilde{H}(j3)|} \cos(3t + 0.3142 - \angle \tilde{H}(j3)) \\ &= \frac{5}{5} + \frac{4}{4} \cos\left(3t + 0.3142 - \left[-\tan^{-1}\frac{3}{4}\right]\right) \\ &= 1 + \cos(3t + 0.9577) \end{aligned}$$

Q5

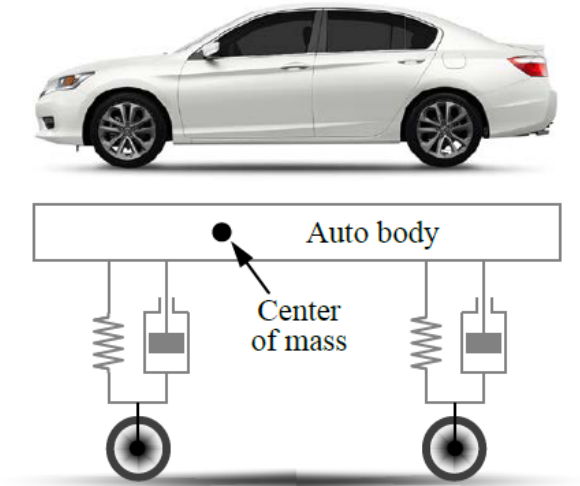
A car suspension system and a very simplified version of the system are shown in Figure 5(a) and 5(b) respectively.

The transfer function of the simplified car suspension system is

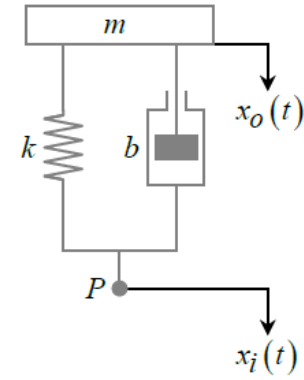
$$\frac{X_o(s)}{X_i(s)} = \frac{bs + k}{ms^2 + bs + k}$$

Suppose a car ($m = 4 \text{ kg}$, $k = 1 \text{ Nm}^{-1}$, $b = 1 \text{ Nm}^{-1}\text{s}$) is travelling on a road that has speed reducing stripes and the input to the simplified car suspension system, $x_i(t)$, may be modelled by the periodic square wave, of frequency $\omega = 1 \text{ rad/s}$, shown in Figure 5(c).

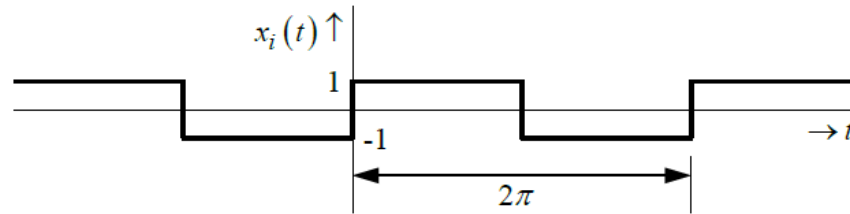
Q5



(a) Automobile suspension system



(b) Simplified suspension system



(c) Input waveform, $x_i(t)$

Figure 5

Q5

Determine the displacement of the car body, $x_o(t)$.

Hint : The Fourier Series representation of the periodic square wave shown in Figure 5(c) is

$$x_i(t) = \frac{4}{\pi} \left[\sin(t) + \frac{1}{3} \sin(3t) + \frac{1}{5} \sin(5t) + \dots \right]$$

Transfer function of suspension system:

$$\tilde{H}(s) = \frac{\tilde{X}_o(s)}{\tilde{X}_i(s)} = \frac{bs + k}{ms^2 + bs + k}$$

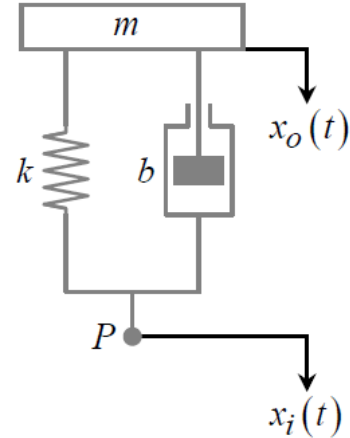
Substituting $m = 4 \text{ kg}$, $k = 1 \text{ Nm}^{-1}$, $b = 1 \text{ Nm}^{-1}\text{s}$, we get

$$\tilde{H}(s) = \frac{s + 1}{4s^2 + s + 1}$$

Frequency response of suspension system:

$$\tilde{H}(j\omega) = \tilde{H}(s) \Big|_{s=j\omega} = \frac{s+1}{4s^2+s+1} \Big|_{s=j\omega} = \frac{1+j\omega}{1-4\omega^2+j\omega}$$

$$\rightarrow \begin{cases} \text{Magnitude Response: } |\tilde{H}(j\omega)| = \left(\frac{1+\omega^2}{(1-4\omega^2)^2 + \omega^2} \right)^{1/2} \\ \text{Phase Response: } \angle \tilde{H}(j\omega) = \tan^{-1}\left(\frac{\omega}{1}\right) - \tan^{-1}\left(\frac{\omega}{1-4\omega^2}\right) \end{cases}$$



$$\left. \begin{aligned}
 x(t) &= Ae^{j(\omega_o t + \psi)} \longrightarrow \boxed{\tilde{H}(j\omega)} \longrightarrow y(t) = A|\tilde{H}(j\omega_o)|e^{j(\omega_o t + \psi + \angle\tilde{H}(j\omega_o))} \\
 x(t) &= A\cos(\omega_o t + \psi) \longrightarrow \boxed{\tilde{H}(j\omega)} \longrightarrow y(t) = A|\tilde{H}(j\omega_o)|\cos(\omega_o t + \psi + \angle\tilde{H}(j\omega_o)) \\
 x(t) &= A\sin(\omega_o t + \psi) \longrightarrow \boxed{\tilde{H}(j\omega)} \longrightarrow y(t) = A|\tilde{H}(j\omega_o)|\sin(\omega_o t + \psi + \angle\tilde{H}(j\omega_o))
 \end{aligned} \right\} \quad (7.20)$$

Steady-state Sinusoidal Response of a LTI System in ω -domain

Fourier series expansion of input: $x_i(t) = \frac{4}{\pi} \left[\sin(t) + \frac{1}{3} \sin(3t) + \frac{1}{5} \sin(5t) + \dots \right]$

- Response of system due to sinusoidal input $\sin(t)$ is given by

$$|\tilde{H}(j1)| \cdot \sin(t + \angle\tilde{H}(j1)) = 0.4472 \sin(t - 2.0344)$$

- Response of system due to sinusoidal input $\sin(3t)$ is given by

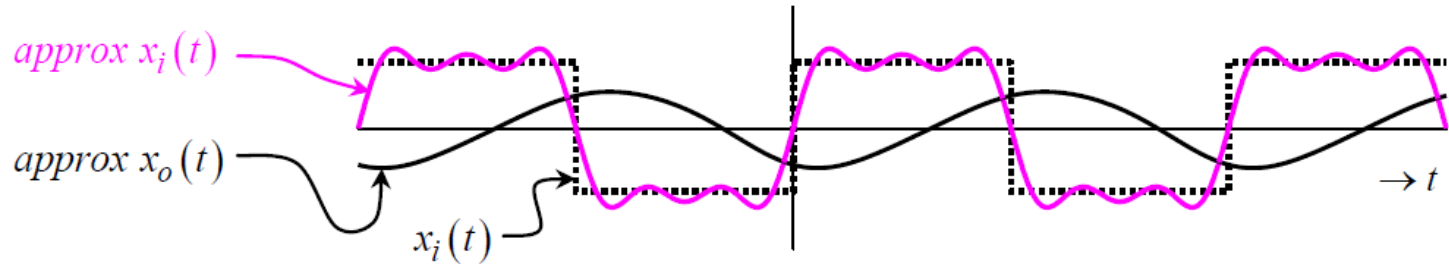
$$|\tilde{H}(j3)| \cdot \sin(3t + \angle\tilde{H}(j3)) = 0.09 \sin(3t - 1.807)$$

- Response of system due to sinusoidal input $\sin(5t)$ is given by

$$|\tilde{H}(j5)| \cdot \sin(5t + \angle\tilde{H}(j5)) = 0.0514 \sin(5t - 1.7177)$$

Since system is linear, the output of the system can be obtained by superposition. Hence,

$$\begin{aligned}x_o(t) &= \frac{4}{\pi} \left[0.4472 \sin(t - 2.0344) + \frac{1}{3} \times 0.09 \sin(3t - 1.807) + \frac{1}{5} \times 0.0514 \sin(5t - 1.7177) + \dots \right] \\&= 0.5694 \sin(t - 2.0344) + 0.0382 \sin(3t - 1.807) + 0.0131 \sin(5t - 1.7177) + \dots\end{aligned}$$





Tutorial 7

Q3

The transfer function of a 2nd-order system is given by $\tilde{H}(s) = \frac{s + 50}{0.2s^2 + 1.2s + 5}$.

- (a) Determine the DC gain (in dB), the damping ratio, ζ , and the natural frequency, ω_n , of the system.
- (b) Is the system overdamped, critically damped, underdamped or undamped, and why?
- (c) Draw the Bode magnitude and phase plots for the system using the semilog-x grid provided below.

(a)

$$\begin{aligned}\text{DC gain} &= \tilde{H}(s) \Big|_{s=0} = \frac{s+50}{0.2s^2+1.2s+5} \Big|_{s=0} = 10 \\ &\equiv 20\log_{10}(10) = 20 \text{ dB}\end{aligned}$$

To determine ζ and ω_n , first rewrite the denominator of $\tilde{H}(s)$ as $0.2(s^2 + 6s + 25)$ so that the coefficient of s^2 is equal to 1. Then compare the bracketted term $s^2 + 6s + 25$ with $s^2 + 2\zeta\omega_n s + \omega_n^2$. This results in

$$\left. \begin{aligned} 2\zeta\omega_n &= 6 \\ \omega_n^2 &= 25 \end{aligned} \right\} \text{ solving } \rightarrow \omega_n = 5 \text{ and } \zeta = 0.6$$

 (b)

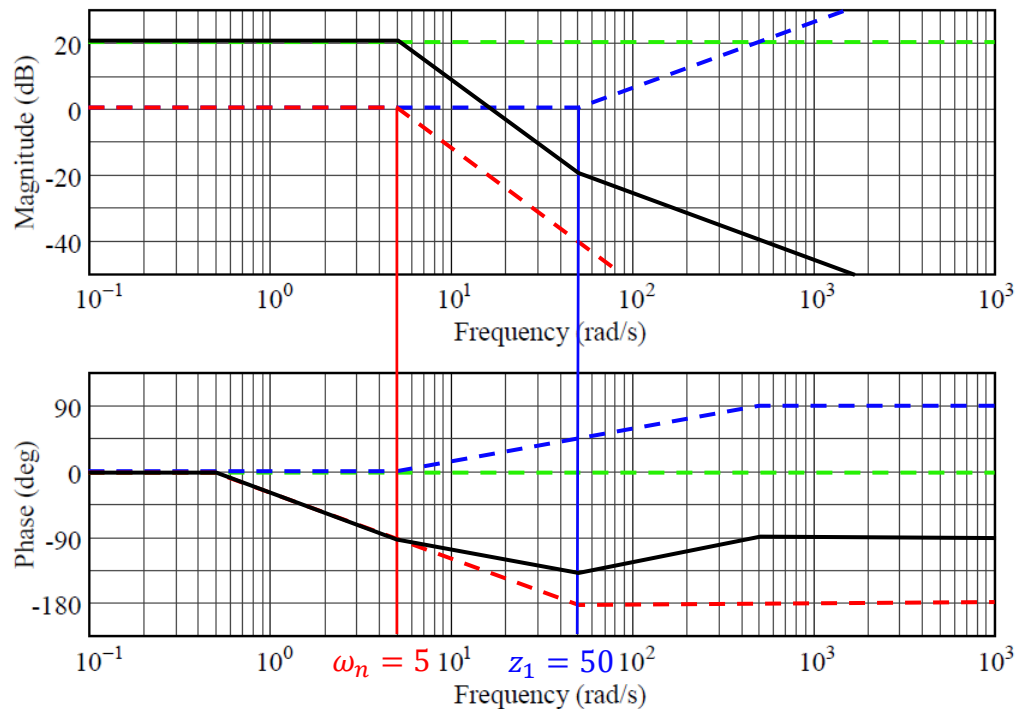
$$\left. \begin{array}{l} 2\zeta\omega_n = 6 \\ \omega_n^2 = 25 \end{array} \right\} \text{ solving } \rightarrow \omega_n = 5 \text{ and } \zeta = 0.6$$

The system UNDERDAMPED because $0 < \zeta < 1$.

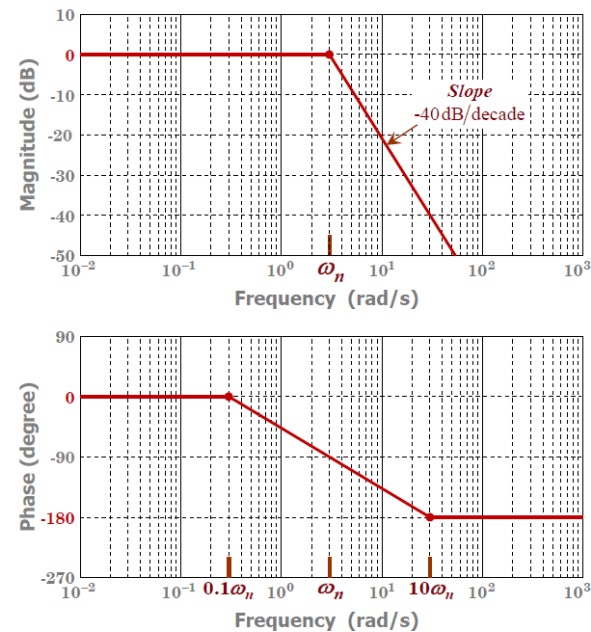
Rewrite $\tilde{H}(s) = \frac{s+50}{0.2s^2+1.2s+5}$ as

(c)

$$\tilde{H}(s) = \frac{50}{0.2} \cdot \frac{\frac{s}{50} + 1}{s^2 + 6s + 25} = 250 \cdot \left(\frac{s}{50} + 1 \right) \cdot \left(\frac{1}{s^2 + 6s + 25} \right) = \underbrace{10}_{\text{dc}} \cdot \underbrace{\left(\frac{s}{50} + 1 \right)}_{\text{zero factor}} \cdot \underbrace{\left(\frac{25}{s^2 + 6s + 25} \right)}_{\text{2nd-order factor}}$$

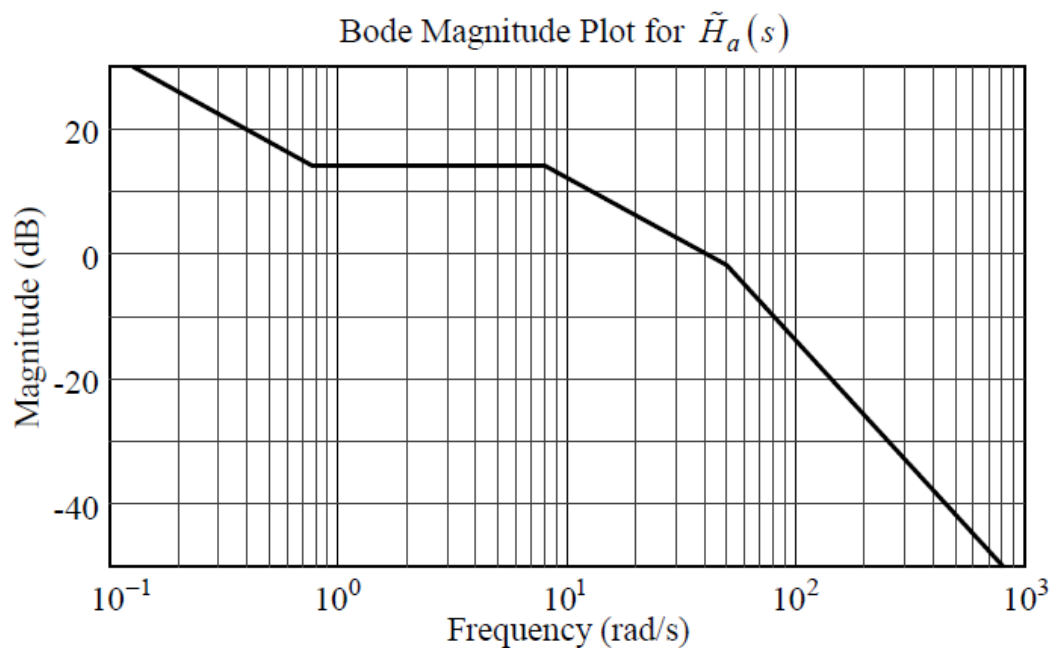


2nd-order Factor

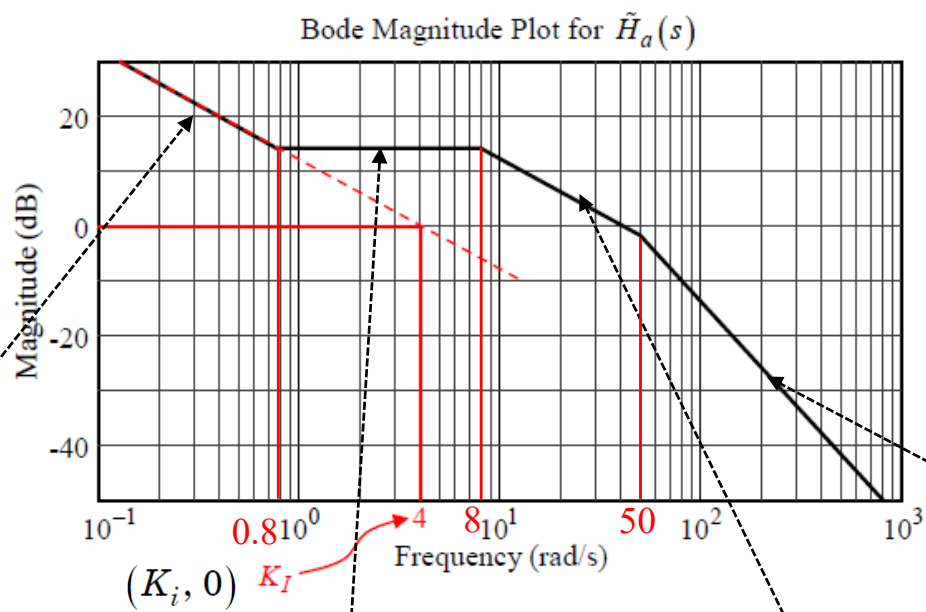


Q5

- (a) Identify the system $\tilde{H}_a(s)$ from its Bode magnitude plot shown below. Describe the behavior of the system at very low frequency ($\omega \ll 0.8$ rad/s) and at very high frequency ($\omega \gg 50$ rad/s).



Q5(a)



The low frequency asymptote is a line with slope -20 dB/decade, which indicates the presence of an integrator

$$\frac{K_I}{s} = \frac{4}{s}$$

A slope change of +20 dB/decade occurs at 0.8 rad/s, which indicates the presence of a zero factor

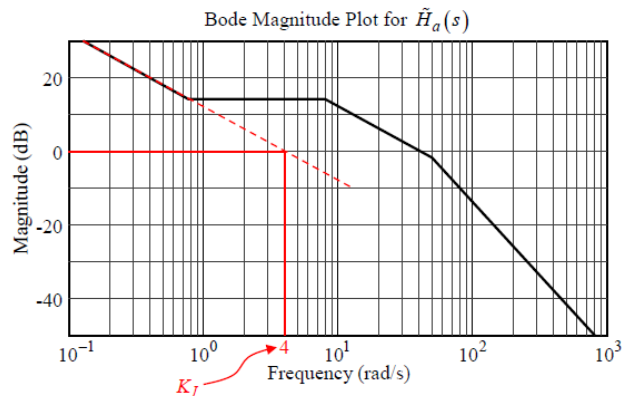
$$\frac{s}{0.8} + 1$$

A slope change of -20 dB/decade occurs at 8 rad/s, which indicates the presence of a pole factor

$$\frac{1}{\frac{s}{8} + 1}$$

A slope change of -20 dB/decade occurs at 50 rad/s, which indicates the presence of a pole factor

$$\frac{1}{\frac{s}{50} + 1}$$



Therefore,

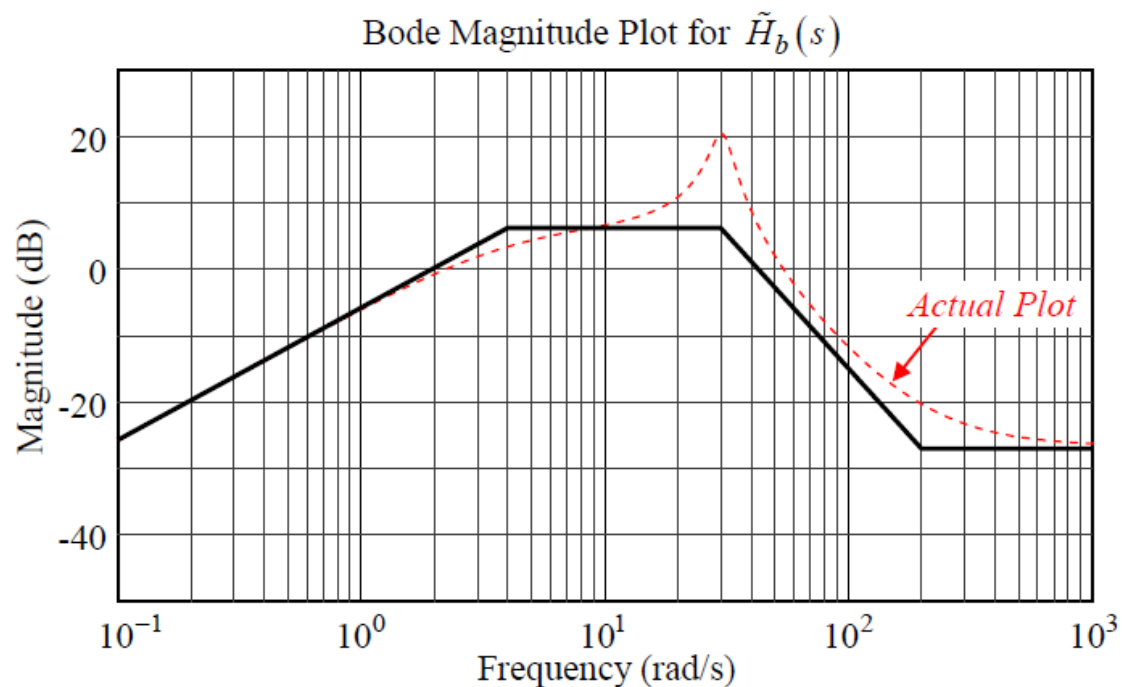
$$\begin{aligned}\tilde{H}_a(s) &= \frac{4}{s} \cdot \left(\frac{s}{0.8} + 1 \right) \cdot \left(\frac{1}{\frac{s}{8} + 1} \right) \cdot \left(\frac{1}{\frac{s}{50} + 1} \right) = \frac{4}{s} \cdot \frac{1}{0.8} (s + 0.8) \cdot 8 \left(\frac{1}{s + 8} \right) \cdot 50 \left(\frac{1}{s + 50} \right) \\ &= \frac{2000}{s} \cdot \frac{s + 0.8}{(s + 8)(s + 50)}\end{aligned}$$

$$\text{At } \omega \ll 0.8 \text{ rad/s: } \tilde{H}_a(s) \cong \frac{2000}{s} \cdot \frac{\cancel{s} + 0.8}{(\cancel{s} + 8)(\cancel{s} + 50)} = \frac{4}{s} \quad \leftarrow \text{ (Integrator of gain 4)}.$$

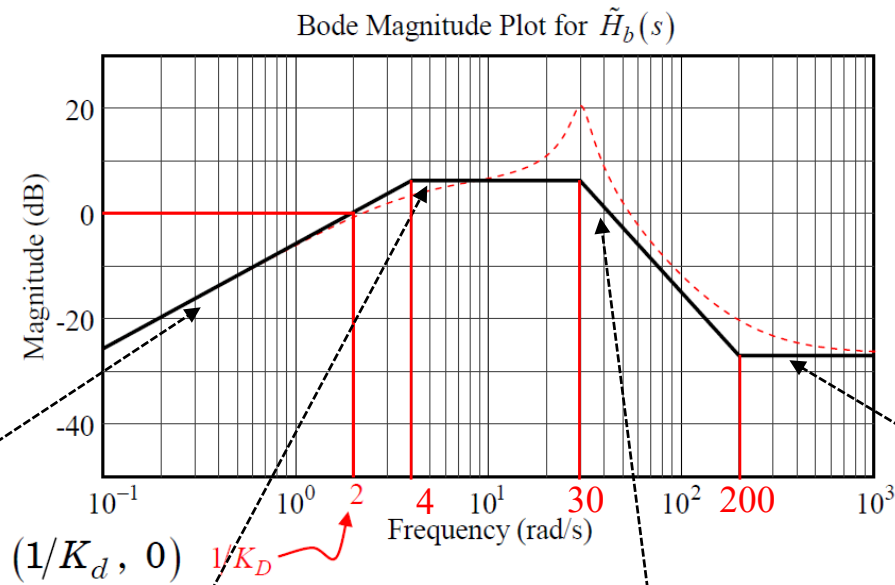
$$\text{At } \omega \gg 50 \text{ rad/s: } \tilde{H}_a(s) \cong \frac{2000}{s} \cdot \frac{s + \cancel{0.8}}{(\cancel{s} + 8)(\cancel{s} + 50)} = \frac{2000}{s^2} \quad \leftarrow \text{ (double integrator of combined gain 2000) }.$$

- (b) The Bode magnitude plot for the system $\tilde{H}_b(s) = K \frac{(s+a)(s+b)(s+c)}{(s+d)(s^2 + 2\zeta\omega_n s + \omega_n^2)}$ is shown below.

Find the values of K , a , b , c , d and ω_n .



Q5(b)



The low frequency asymptote is a line with slope +20 dB/decade, which indicates the presence of a differentiator

$$K_D s = 0.5s$$

A slope change of -20 dB/decade occurs at 4 rad/s, which indicates the presence of a pole factor

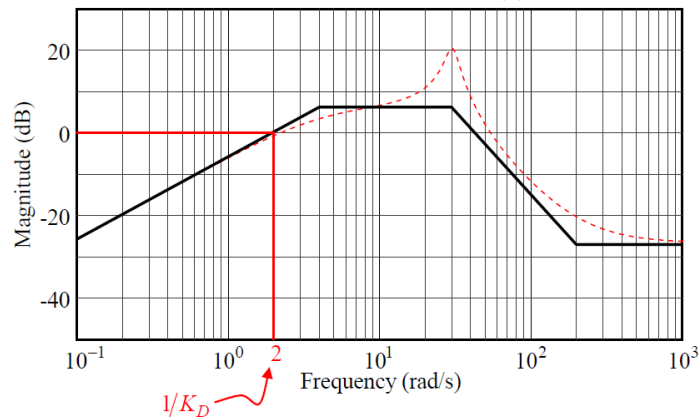
$$\frac{1}{\frac{s}{4} + 1}$$

A slope change of -40 dB/decade occurs at 30 rad/s, which indicates the presence of a 2nd-order factor due to the existence of a resonant peak

$$\frac{30^2}{s^2 + 60\zeta s + 30^2}$$

A slope change of +40 dB/decade occurs at 200 rad/s, which indicates the presence of a double zero factor

$$\left(\frac{s}{200} + 1\right)^2$$

Bode Magnitude Plot for $\tilde{H}_b(s)$ 

Therefore,

$$\begin{aligned}
 \tilde{H}_b(s) &= 0.5s \cdot \left(\frac{1}{\frac{s}{4} + 1} \right) \cdot \left(\frac{900}{s^2 + 60\zeta s + 900} \right) \cdot \left(\frac{s}{200} + 1 \right)^2 \\
 &= 0.5s \cdot 4 \left(\frac{1}{s + 4} \right) \cdot 900 \left(\frac{1}{s^2 + 60\zeta s + 900} \right) \left(\frac{1}{s + 30} \right)^2 \cdot \frac{1}{200^2} (s + 200)^2 \\
 &= 0.045 \frac{s(s + 200)^2}{(s + 4)(s^2 + 60\zeta s + 900)} = K \frac{(s + a)(s + b)(s + c)}{(s + d)(s^2 + 2\zeta\omega_n s + \omega_n^2)}
 \end{aligned}$$

which results in $K = 0.045$, $a = 0$, $b = c = 200$, $d = 4$ and $\omega_n = 30$.



Tutorial 8

Q.3 The spectrums of two real bandpass signals, $x(t)$ and $y(t)$, are shown in Figure Q3.

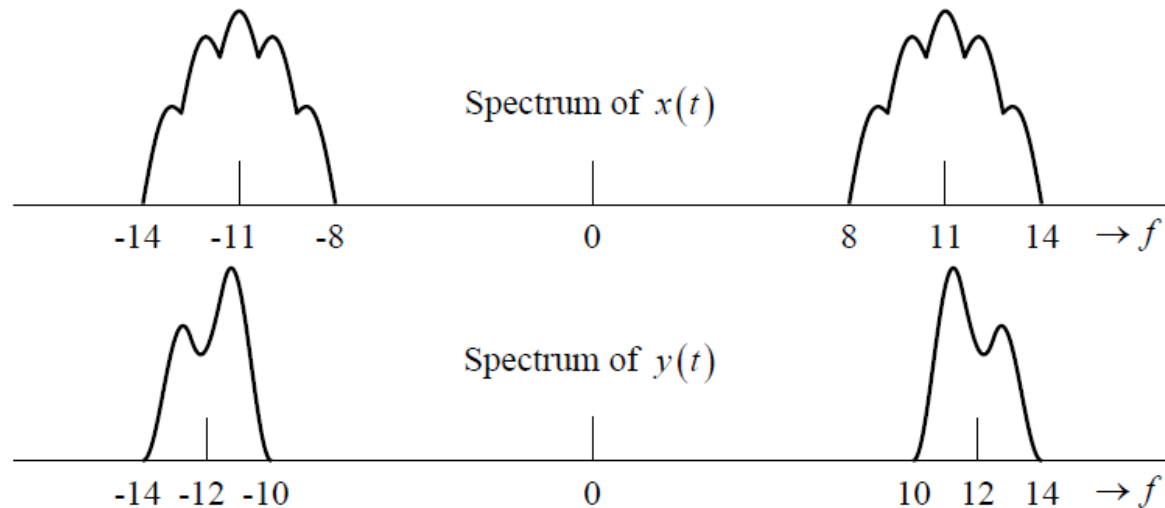


Figure Q3

For each of $x(t)$ and $y(t)$:

- What is the Nyquist sampling frequency?
- Find the smallest sampling frequency that allows for exact reconstruction of the signal from its sampled version and determine the frequency response, $H(f)$, of the corresponding ideal reconstruction filter.

Q.3 The spectrums of two real bandpass signals, $x(t)$ and $y(t)$, are shown in Figure Q3.

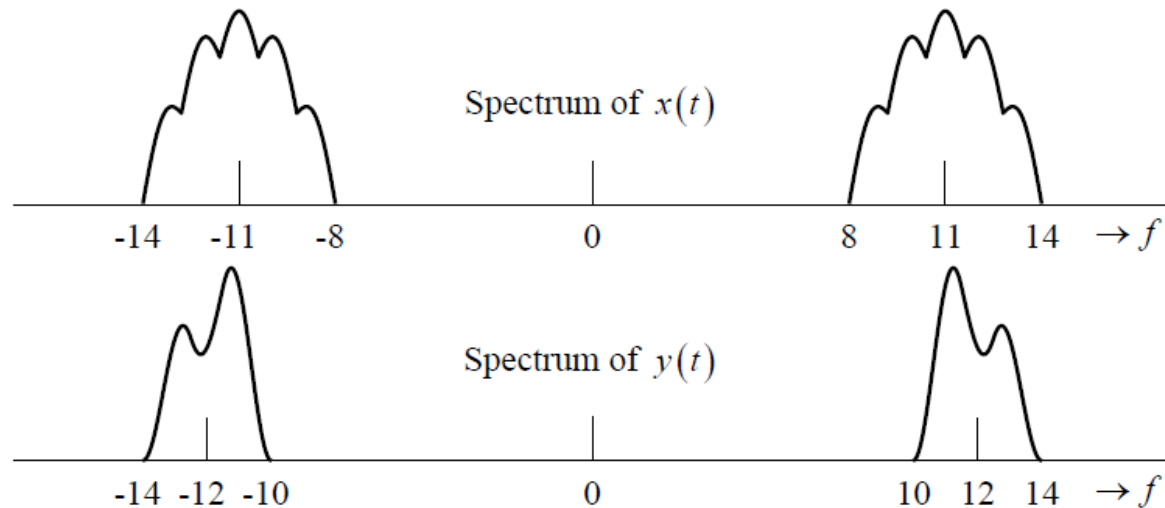


Figure Q3

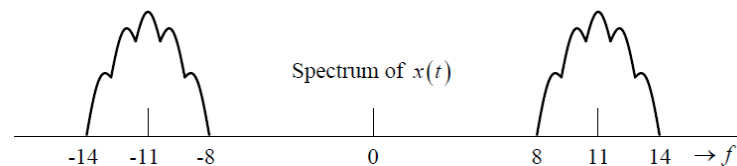
For each of $x(t)$ and $y(t)$:

- (a) What is the Nyquist sampling frequency?

Solution to Q.3

- (a) Nyquist sampling frequency for both $x(t)$ and $y(t)$ is $2 \times 14 = 28$ Hz.

(b) Find the smallest sampling frequency that allows for exact reconstruction of the signal from its sampled version and determine the frequency response, $H(f)$, of the corresponding ideal reconstruction filter.



SIGNAL $x(t)$:

Due to symmetry of the positive-frequency and negative-frequency spectral images about the center frequencies 11 Hz and -11 Hz, respectively, both the Overlapping Spectral Images formula and the Un-aliased Spectral Images formula are applicable. Of these, the Overlapping Spectral Images formula always yield the lowest sampling frequency.

Center frequency : $f_c = 11$ Hz

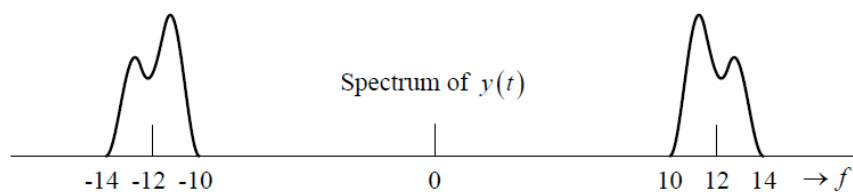
Bandwidth : $B = 6$ Hz

$$\text{Overlapping spectral images formula : } \begin{cases} f_s = \frac{2f_c}{k}; \quad k = 1, 2, \dots, \left\lfloor \frac{2f_c}{B} \right\rfloor \\ = \frac{22}{k}; \quad k = 1, 2, 3 \\ = 22, 11, \frac{22}{3} \text{ Hz} \end{cases}$$

(Possible if $f_c > 0.5B$)

- Therefore, the smallest possible sampling frequency is $f_{s,\min} = \frac{22}{3}$ Hz
- For exact reconstruction of $x(t)$, the frequency response of the ideal reconstruction filter should be:

$$H(f) = \frac{1}{2f_{s,\min}} \left[\text{rect}\left(\frac{f+f_c}{B}\right) + \text{rect}\left(\frac{f-f_c}{B}\right) \right] = \frac{3}{44} \left[\text{rect}\left(\frac{f+11}{6}\right) + \text{rect}\left(\frac{f-11}{6}\right) \right]$$



SIGNAL $y(t)$:

Due to asymmetry of the positive-frequency and negative-frequency spectral images about the center frequencies 12 Hz and -12 Hz, respectively, only the Un-aliased Spectral Images formula is applicable.

Center frequency : $f_c = 12$ Hz

Bandwidth : $B = 4$ Hz

$$\text{Un-aliased spectral images formula : } \begin{cases} \frac{2f_c + B}{k+1} \leq f_s \leq \frac{2f_c - B}{k}; & k=1, 2, \dots, \left\lfloor \frac{2f_c - B}{2B} \right\rfloor \\ \rightarrow \frac{28}{k+1} \leq f_s \leq \frac{20}{k}; & k=1, 2 \\ \rightarrow \underbrace{14 \leq f_s \leq 20}_{k=1} \text{ and } \underbrace{\frac{28}{3} \leq f_s \leq 10}_{k=2} \end{cases}$$

(Possible if $f_c > 1.5B$)

- Therefore, the smallest possible sampling frequency is $f_{s,\min} = \frac{28}{3}$ Hz
- For exact reconstruction of $y(t)$, the frequency response of the ideal reconstruction filter should be:

$$H(f) = \frac{1}{f_{s,\min}} \left[\text{rect}\left(\frac{f + f_c}{B}\right) + \text{rect}\left(\frac{f - f_c}{B}\right) \right] = \frac{3}{28} \left[\text{rect}\left(\frac{f + 12}{4}\right) + \text{rect}\left(\frac{f - 12}{4}\right) \right]$$

Q.5 Speech studies have shown that perceptual cues pertaining to speech intelligibility and speaker identity are mainly found within the first 4 kHz of the speech spectrum. We shall refer to this frequency band as the *TQ-Band*, where *TQ* stands for ‘*Telephone Quality*’.

The spectrum, $X(f)$, of a speech signal $x(t)$ is shown in Figure Q5 where the *TQ-Band* is indicated.

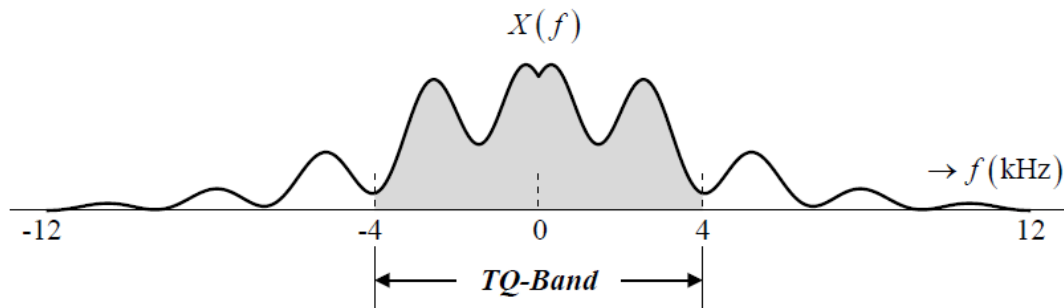


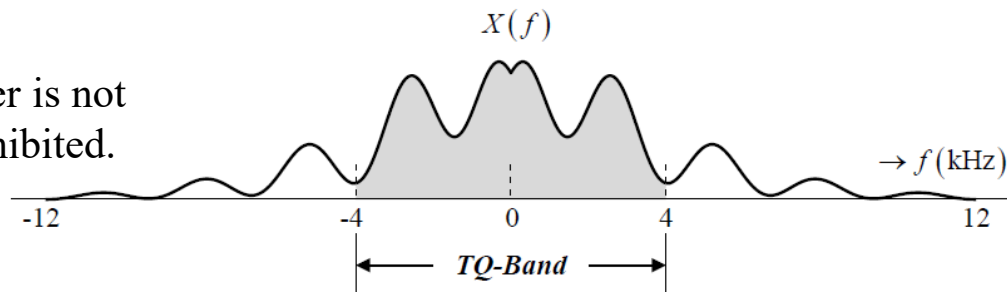
Figure Q5

In order to send $x(t)$ over a digital telephone network, the first step is to sample $x(t)$ without corrupting its *TQ-Band*. Suggest a method for sampling $x(t)$ in each of the following situations.

- Situation A: Anti-aliasing lowpass filter is not available and frequency aliasing is prohibited.
- Situation B: Lowest sampling frequency must be used at all cost.
- Situation C: Anti-aliasing lowpass filter is not available and frequency aliasing is permitted.

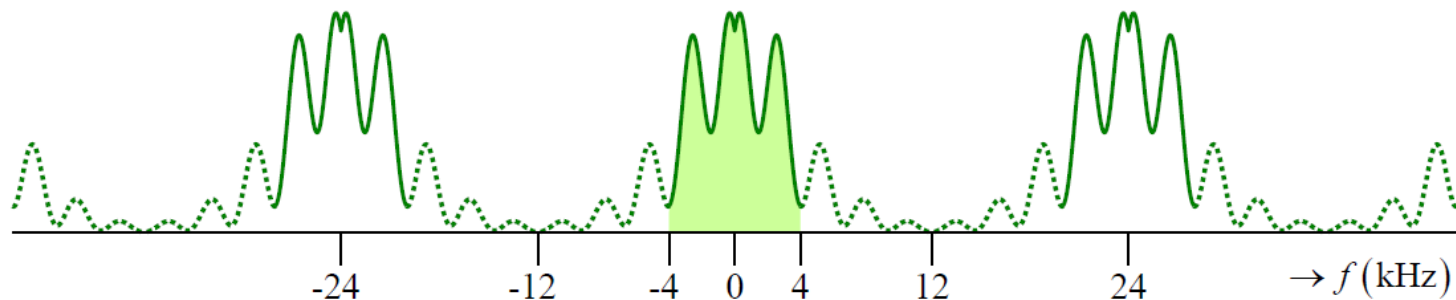
Comment on the advantage and disadvantage of each method.

Situation A: Anti-aliasing lowpass filter is not available and frequency aliasing is prohibited.



Solution to Q.5

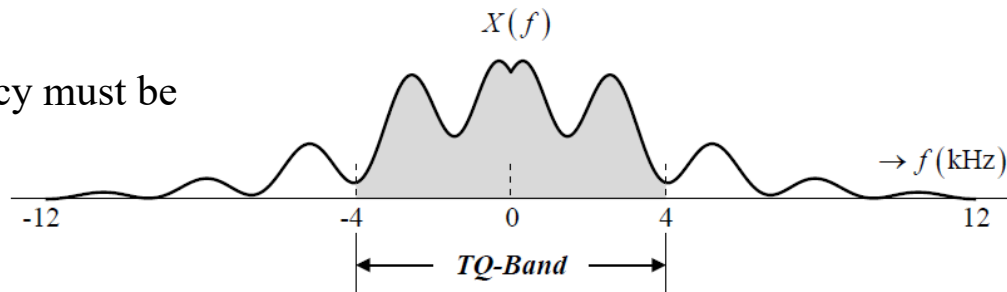
Situation A: (No anti-aliasing lowpass filter
Frequency aliasing prohibited): Sampling frequency = $2 \times 12 = 24$ kHz



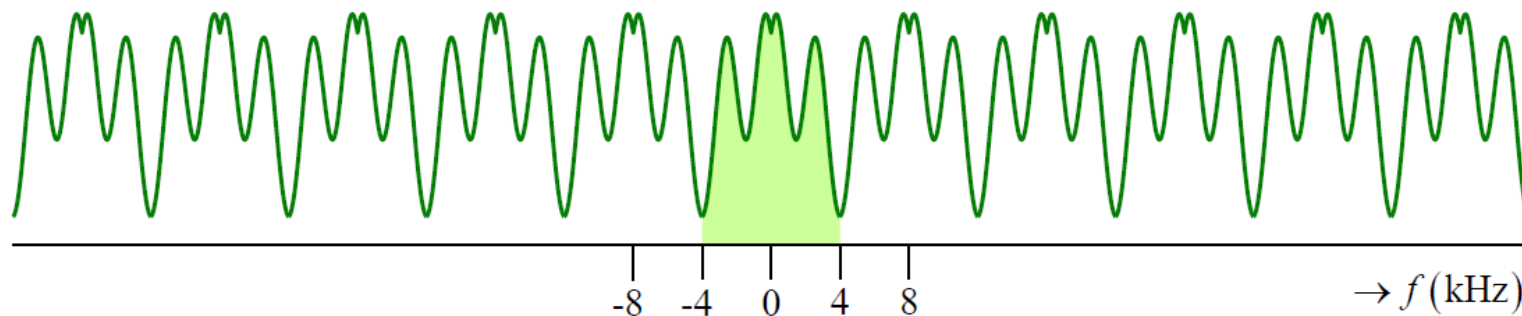
Advantage: No anti-aliasing LPF needed.

Disadvantage: Sampling frequency is higher than necessary to preserve the *TQ-Band*.

Situation B: Lowest sampling frequency must be used at all cost.



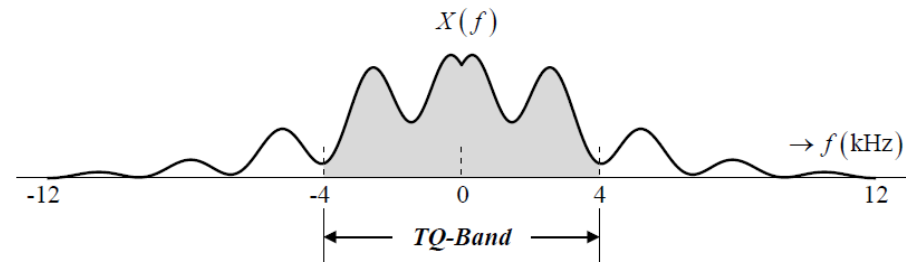
Situation B: Lowest sampling frequency: $\left\{ \begin{array}{l} \text{Ideal anti-aliasing filter of bandwidth 4 kHz} \\ \text{Sampling frequency} = 2 \times 4 = 8 \text{ kHz} \end{array} \right.$



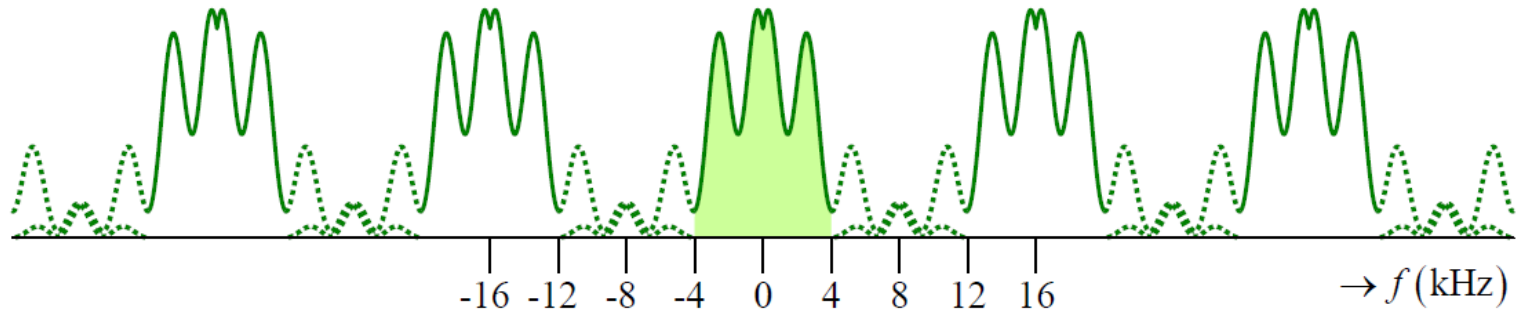
Advantage: Lowest possible sampling frequency to preserve the *TQ-Band*.

Disadvantage: Require anti-aliasing LPF with sharp cutoff.

Situation C: Anti-aliasing lowpass filter is not available and frequency aliasing is permitted.



Situation C: (No anti-aliasing lowpass filter): Sampling frequency = $24 - 8 = 16$ kHz
Frequency aliasing permitted



Advantage: Lowest possible sampling frequency to preserve the *TQ-Band* without requiring anti-aliasing LPF with sharp cutoff.

Disadvantage: Sampling frequency is still higher than the minimum needed to preserve the *TQ-Band*.

This problem illustrates the trade-off between oversampling and the requirement of expensive anti-aliasing LPF with sharp cutoff frequency.



Thanks!