

[4]

OR

A multi-plate disc clutch transmits 55 kW of power at 1800 rpm. Coefficient of friction for the friction surfaces is 0.1. Axial intensity of pressure is not to exceed  $160 \text{ kN/m}^2$ . The external radius is 80 mm and is 0.7 times the external radius. Find the number of plates needed to transmit the required torque.

5. a) What is the effect of centrifugal tension on power transmission?
- b) Differentiate between a self-locking and self-energizing brake.
- c) Explain jump and cross-over shock for a cam.
- d) Design a set of stepped pulleys to drive a machine from a countershaft that runs at 220 rpm. The distance between centres of the two sets of pulleys is 2 m. The diameter of the smallest step on the countershaft is 160 mm. The machine is to run at 80, 100 and 130 rpm and should be able to rotate in either direction.

OR

A differential band brake has a drum with diameter of 800 mm. The two ends of the band are fixed to the pins on the opposite sides of the fulcrum of the lever at distances of 40 mm and 200 mm from the fulcrum. The angle of contact is  $270^\circ$  and the coefficient of friction is 0.2. Determine the brake torque when a force of 600 N is applied to the lever at a distance of 800 mm from the fulcrum.

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Total No. of Questions :5]

Roll No .....

**ME - 505****B.E. V Semester**

Examination, June 2015

**Dynamics of Machines****Time : Three Hours****Maximum Marks : 70**

*Note:* i) Answer five questions. In each question part A, B, C is compulsory and D part has internal choice.

ii) All parts of each questions are to be attempted at one place.

iii) All questions carry equal marks, out of which part A and B (Max.50 words) carry 2 marks, part C (Max.100 words) carry 3 marks, part D (Max.400 words) carry 7 marks.

iv) Except numericals, Derivation, Design and Drawing etc.

1. a) Explain piston effort and crank effort.
- b) What is main function of a flywheel?
- c) Write the procedure for determining the turning moment diagram.
- d) A horizontal gas engine running at 210 rpm has a bore of 220 mm and a stroke of 440 mm. The connecting rod is 924 mm long and the reciprocating parts weigh 20 kg. When the crank has turned through an angle of  $30^\circ$  from the inner dead centre, the gas pressure on the cover and the crank sides are  $500 \text{ kN/m}^2$  and  $60 \text{ kN/m}^2$  respectively. Diameter of the piston rod is 40 mm. determine:
  - i) Turning moment on the crank shaft
  - ii) Thrust on the bearing
  - iii) Acceleration of the flywheel which has a mass of 8 kg and radius of gyration of 600 mm while the power of the engine is 22kW.

OR

The turning moment diagram for a petrol engine is drawn to a vertical scale of  $1 \text{ mm} = 500 \text{ Nm}$  and a horizontal scale of  $1 \text{ mm} = 3^\circ$ . The turning moment diagram repeats itself after every half revolution of the crank shaft. The areas above and below the mean torque line are  $260, -580, 80, -380, 870$  and  $-250 \text{ mm}^2$ . The rotating parts have a mass of  $55 \text{ kg}$  and radius of gyration of  $2.1 \text{ m}$ . If the engine speed is  $1600 \text{ rpm}$ , determine the coefficient of fluctuation of speed.

2. a) What is the main function of a governor?
- b) Define effort and power of a governor.
- c) Explain the meaning of sensitiveness, hunting, stability and controlling force of a governor.
- d) Each arm of a Porter governor is  $250 \text{ mm}$  long. The upper and lower arms are pivoted to links of  $40 \text{ mm}$  and  $50 \text{ mm}$  respectively from the axis of rotation, each ball has a mass of  $5 \text{ kg}$  and the sleeve mass is  $50 \text{ kg}$ . The force of friction on the sleeve of the mechanism is  $40 \text{ N}$ . Determine the range of speed of the governor for extreme radii of rotation of  $125 \text{ mm}$  and  $150 \text{ mm}$ .

OR

In a Hartnell governor, the radius of rotation of the balls is  $60 \text{ mm}$  at the minimum speed of  $240 \text{ rpm}$ . The length of the ball arm is  $130 \text{ mm}$  and the sleeve arm is  $80 \text{ mm}$ . The mass of each ball is  $3 \text{ kg}$  and the sleeve is  $4 \text{ kg}$ . The stiffness of the spring is  $20 \text{ N/mm}$ . Determine the

- i) Speed when the sleeve is lifted by  $50 \text{ mm}$
- ii) Initial compression of the spring
- iii) Governor effort

3. a) What do you mean by static and dynamic balance of machinery?
- b) Define hammer blow, tractive effort and swaying couple.
- c) What do you mean by primary and secondary unbalance in reciprocating engine?
- d) Four masses A, B, C and D carried by a rotating shaft at radii  $80 \text{ mm}$ ,  $100 \text{ mm}$ ,  $160 \text{ mm}$  and  $120 \text{ mm}$  respectively are completely balanced. Masses B, C and D are  $8 \text{ kg}$ ,  $4 \text{ kg}$ , and  $3 \text{ kg}$  respectively. Determine the mass A and the relative angular positions of the four masses if the planes are spaced  $500 \text{ mm}$  apart.

OR

Each crank and the connecting rod of a four-crank in line engine are  $200 \text{ mm}$  and  $800 \text{ mm}$  respectively. The outer cranks are set at  $120^\circ$  to each other and each has a reciprocating mass of  $200 \text{ kg}$ . The spacing between adjacent planes of cranks is  $400 \text{ mm}$ ,  $600 \text{ mm}$  and  $500 \text{ mm}$ . If the engine is in complete primary balance, determine the reciprocating masses of the inner cranks and their relative angular positions. Also find the secondary unbalanced force if the engine speed is  $210 \text{ rpm}$ .

4. a) What is friction? Is it necessary evil or blessing?
- b) Explain friction circle.
- c) Explain uniform pressure and uniform wear theories.
- d) In a thrust bearing, the external and internal diameters of the contacting surfaces are  $320 \text{ mm}$  and  $200 \text{ mm}$  respectively. The total axial load is  $80 \text{ kN}$  and the intensity of pressure is  $350 \text{ kN/m}^2$ . The shaft rotates at  $400 \text{ rpm}$ . Taking the coefficient of friction as  $0.06$ , calculate the power lost in overcoming the friction. Also, find the number of collars required for the bearing.