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## **MMMD/MMPD-302(A)** M.E./M.Tech., III Semester

Examination, June 2016

## Fluid Film Lubrication (Elective-II)

Time: Three Hours

Maximum Marks: 70

Note: Attempt any five questions. All questions carry equal marks. Use of design data hand book is permitted. Assume suitable data or dimensions, if necessary, clearly mentioned it.

- Give the classification of Bearings with their illustrations and applications.
  - Derive the Reynolds equations for bearing lubrication from continuity equation.
- Write and explain the boundary conditions for infinitely long full journal bearing.
  - What is hydrostatic lubrication? What are its advantages and disadvantages over hydrodynamic lubrications?
- Derive the Petroff's equation for coefficient of friction for a vertical shaft rotating in a guide bearing.
  - b) Discuss the geometry of different types of non-circular bearings.
- Derive the governing equation for gas bearings.
  - Discuss the thermal analysis for the finite length fluid film lubrication bearings.

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- What are rolling contact bearings? Discuss their advantages over sliding contact bearings.
  - List the Numerical methods for solution of fluid film equations for bearing and explain any one of them.

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Design a journal bearing for a centrifugal pump from the following data:

Load on the journal

Speed of the journal 1000 RPM

Lubricant SAE 10

Absolute viscosity at 55°C 0.017kg/m-s

Ambient temperature of oil 15.5°C

Maximum bearing pressure 105 N/mm<sup>2</sup>

Also calculate the mass of the lubricating oil required artificial cooling, if the temperature of oil be limited to 10°C. Take heat dissipation coefficient is 1230 W/m<sup>2</sup>/°C.

- 7. A SKF bearing 6200 is used to support a radial load of 1000 N. Determine Rated life and life at 99% survival rate.
- Write short notes on following (Any four)
  - Hydrostatic lubrication
  - Rayleigh-step bearings
  - Sommerfeld Conditions
  - Finite element method
  - Boundary lubrications
  - Performance characteristics of bearing

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