AU/ME-803

B. E. (Eighth Semester) EXAMINATION, June, 2012

(Common for AU & ME Engg. Branch)

REFRIGERATION AND AIRCONDITIONING

Time : Three Hours Maximum Marks: 100 Minimum Pass Marks : 35

Note: Attempt five questions in all selecting one question from each Unit. Assume suitable missing/misprint data (if any). Use of standard properties table and charts of refrigerants and psychrometric chart—is permissible in the examination.

Unit-I

- 1. (a) What is refrigeration? Define 1 ton of refrigeration. 6
- (b) What is the principle of operation of vortex tube ? 6
- (c)The relative C. O. P. of a refrigeration system operating between temperature limits 13°C and 39°C is 0-5. Obtain the amount of ice that can be obtained from the machine if it takes 5 kW. The ice is formed at 10°C from water available at 33°C. 8
- 2- (a) What is COP? What value of COP is desired large or small and why? 5
- j(b) The following data refer to a simple aircfaft '

 $=27^{\circ}C$

refrigeration system: 15

Temp, at the end of compression = 517° C

Room air temp.

Cabin pressure= 1-01 bar

Pressure at inlet to cooling turbine = 4-85bar

Heat rejection to heat exchanger = 105 kW

and G = 0-9

Obtain the following:

- (i) mass flow rate of air for refrigeration
- (ii)tonnage
- (iii)power and C. O. P. of the system

Unit-II

- 3.(a) Explain the working of the saturated reversed Carnot refrigeration on T-S diagram. 8
- (b) A vapour compression refrigeration system operates between the evaporating and condensing temperatures of 258 K and 313 K respectively. Calculate: 12
- (i) Tonnage (il) Volume handled by compressor
- (iii)CO. Heat transfer to condenser

The compressor power input is 10 kW. The refrigerant used in R-22 and enthalpy at the end of isentropic compression is $h_2 = 287*07$ kJ/kg.

Or

- 4.(a) Which of two factors (increase in upper pressure limit,
- and decreasing lower pressure limit) has more adverse effect on C. O. P. of a vapour compression refrigeration system? Explain. 6
- (b) A single stage ammonia ice plant operates on simple saturation cycle at the condensing temp, of 40°C and evaporating temp, of 15^DC. It produces 10 ton of ice per day at 5°C from water at 30°C. 1 Determine: f 14
- (i) Capacity of refrigeration plant
- (ii) Mass flow rate of refrigerant '.
- (iii)Isentropic diacharge temp.
- (iv)Compressor dimensions if its volumetric efficiency is assumed at 65%. The compressor is to run at 1400 r. p. m. Take L/D ratio as 1-2.
- (v)Theoretical and Actual C. O. P.

Unit-III

- 5.(a) Discuss the advantages and disadvantages of vapour absorption system. 10
- (b) Explain the principle and working of steam jet refrigeration system. 10, Or
- 6.(a) Why is ammonia-water vapour absorption system so popular? Explain its working in detail. 10

- (b) What is a refrigerant ? Discuss desired thermodynamic -properties of refrigerant. 10 Unit-IV
- 7.(a) Define specific humidity, relative humidity, psychrometry and wet and dry bulb temperature. 10
- (b) A mixture of dry air and water vapour is at a temp, of 23°C under a total pressure of 736 mm of Hg. The dew point temp, is 16°C.

Find: 10 (i) Partial pressure of water vapour, -(ii) Relative humidity

- (iii)Specific humidity
- (iv)Specific enthalpy of water
- (v)Enthalpy of air per kg of dry air

Or

- 8.(a) What is bypass factor? Explain its usefulness. 5
- (b) Given for the air-conditioning of a room:

Room conditions = $26-5^{\circ}$ DBT and 50% RH

Room sensible heat gain = 26 = 3 kW

Room sensible heat factor =0-82 Find: 15

- (i) The room latent heat gain.
- (ii) The apparatus dew point.
- (iii) The volume flow rate of air if it is supplied to the room at apparatus dew point.
- (iv)Volume flow rate of air if it is supplied to the room at 17°C,

Unit-V

9.A retail shop located in a city at 30°N has the following loads: \20

Room sensible heat = 58-15 kW

Room latent heat = 14 - 54 kW

The summer outside and inside design conditions are:

Outside - 42° DBT, 27° WBT

Inside -25 DBT, 40% RH

70 cmm of ventilation air is used.

Determine:

- (i) Ventilation load
- (ii) Grand total heat
- (iii)Effective sensible heat factor
- (iv)Apparatus dew point
- (v)Dehumidified air quantity
- (vi)Condition of air entering and leaving apparatus. Assume B. F. =0-15

Or

10. The following data refers to an air-conditioning system for a restaurant for capacity of 50 persons :

Heat transfer through structure = 22000 kJ/hr.,

Solar heat gain= 7000 kJ/hr.,

Sensible heat gain due to various

experiments = 10000 kJ/hr.

Latent heat gain by equipments = 5000 kJ/hr.,

Ventilation and infiltration of air = 2000 mVhr.

Ambient condition $= T_{db} = 308 \text{ K},$

 $T^{\wedge} = 299 \text{ K}.$

Inside temperature $T_{db} = 300 \text{ K}$

and .

Air supply temperature = 290 K

and there are five service men.

Calculate: 20

- (i) Air supply to room
- (ii)Percentage of make up air
- (iii)Tonnage of cooling coil
- (iv)ADP and Bypass factor