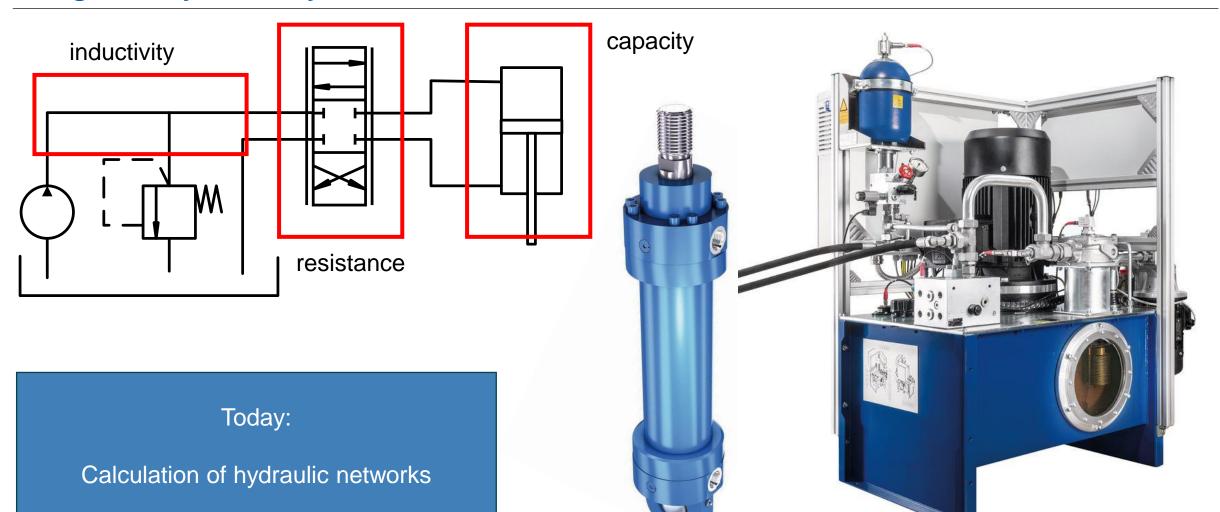






Design of a hydraulic system







Source: Bosch Rexroth

Outline of todays lecture

1	Components	in a	hvdraulic	network
			,	

- 1.1 Hydraulic resistance
- 1.2 Hydraulic capacity
- 1.3 Hydraulic inductivity
- 2 Calculation of a hydraulic network
- 3 Liquid collumn as homogeneouse resonator
- 4 Summary





Hydraulic networks

resistance:

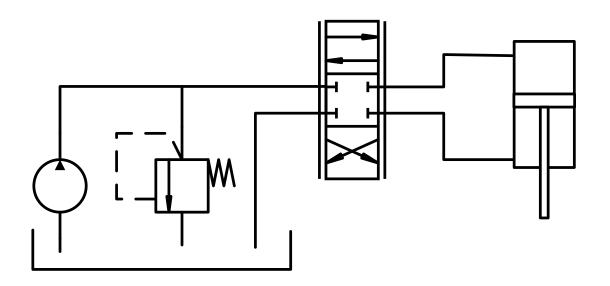
$$R_{\rm H} = \frac{\Delta p}{Q}$$

capacity:

$$C_{\rm H} = \frac{Q}{\dot{p}}$$

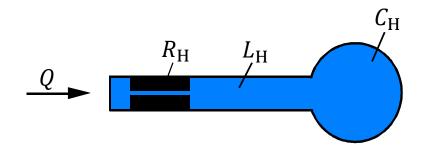
inductivity:

$$L_{\rm H} = \frac{\Delta p}{\dot{Q}}$$

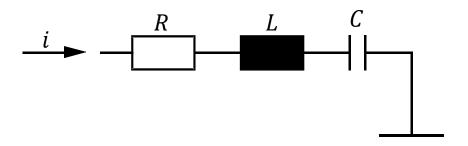




Serial connection of complex resistors



analogy with electrical engineering



complex resistance:

$$p = R_{\rm H} \cdot Q + L_{\rm H} \cdot \frac{\mathrm{d}Q}{\mathrm{d}t} + \frac{1}{C_{\rm H}} \cdot \int Q \mathrm{d}t$$

analogy with electrical engineering:

$$u = R \cdot i + L \cdot \frac{\mathrm{d}i}{\mathrm{d}t} + \frac{1}{C} \cdot \int i \mathrm{d}t$$

$$p = u$$

$$Q = i$$





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Basic resistances: throttle / gap

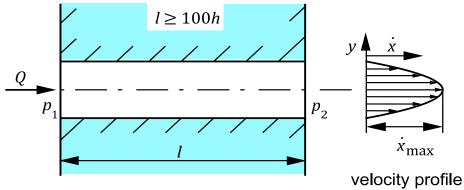
throttle: Hagen-Poisouille equation:

$$Q = \frac{\pi \cdot r^4}{8 \cdot \eta \cdot l} \cdot (p_1 - p_2)$$

gap: Hagen-Poisouille equation:

$$Q = \frac{b \cdot h^3}{12 \cdot \eta \cdot l} \cdot (p_1 - p_2)$$

length of gap *l*



Hydraulic resistance:

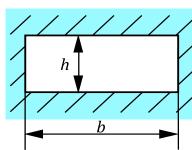
$$\Delta p = Q \cdot R_{\rm H}$$

$$R_{\rm H} = \frac{8 \cdot \eta \cdot l}{\pi \cdot r^4}$$

Hydraulic resistance:

$$R_{\rm H} = \frac{12 \cdot \eta \cdot l}{b \cdot h^3}$$

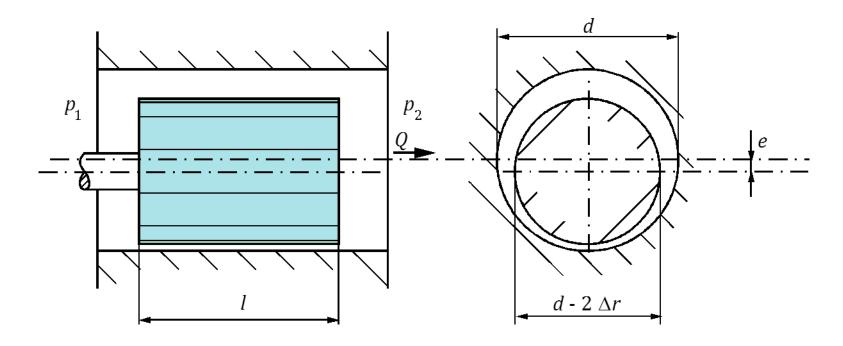
gap section $b \cdot h$







Flow through an Excentric Annular Gap

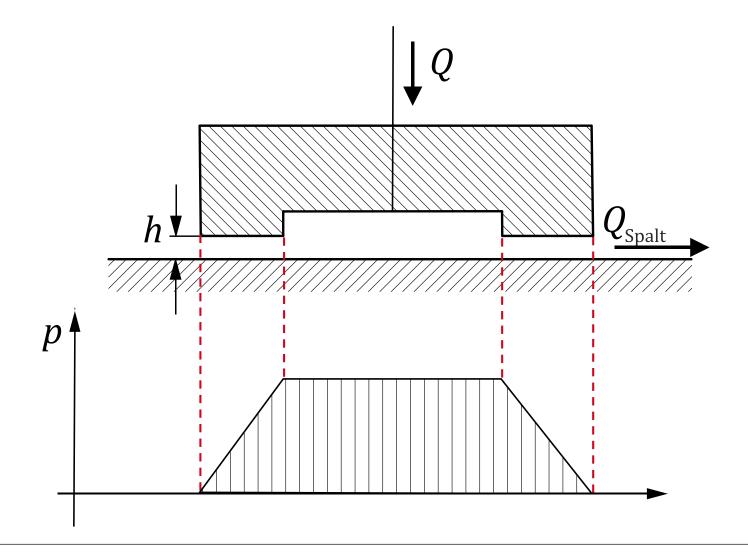


$$Q = \frac{d \cdot \pi \cdot \Delta r^3}{12 \cdot \eta \cdot l} \cdot \left[1 + 1.5 \cdot \left(\frac{e}{\Delta r} \right)^2 \right] \cdot (p_1 - p_2)$$



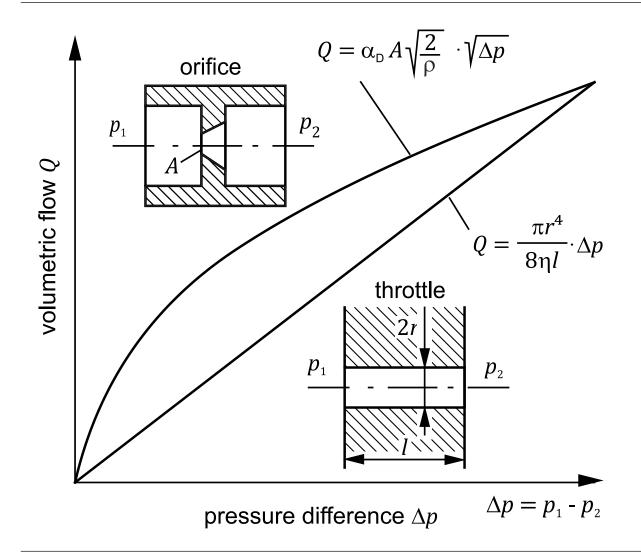


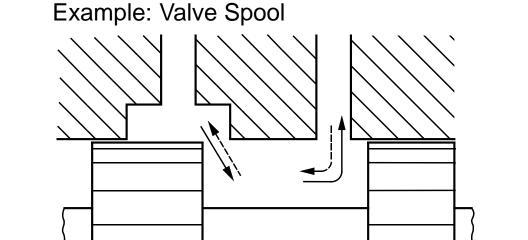
Hydrostatic Bearing





Basic Resistances: throttle - orficie





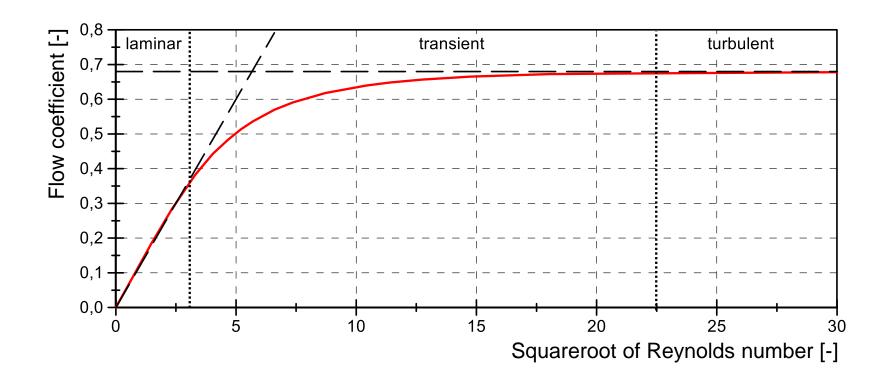
Orifice: turbulent resistance temperature indepentent

Throttle: laminar resistance temperature depentent





Flow coefficient as a function of the Reynolds number





Pressure losses in hydraulic circuits

1. Pressure loss on pipes (Blasius)

$$\Delta p_{\rm R} = \sum \lambda \cdot \frac{l}{d} \cdot \frac{\rho}{2} \cdot v^2$$
 λ – pipe loss factor

2. Pressure loss in fittings (bends, intersections, etc.)

$$\Delta p_{\rm F} = \sum_{i} \xi_i \cdot \frac{\rho}{2} \cdot v_i^2 \qquad \qquad \xi - \text{resistance coefficient}$$



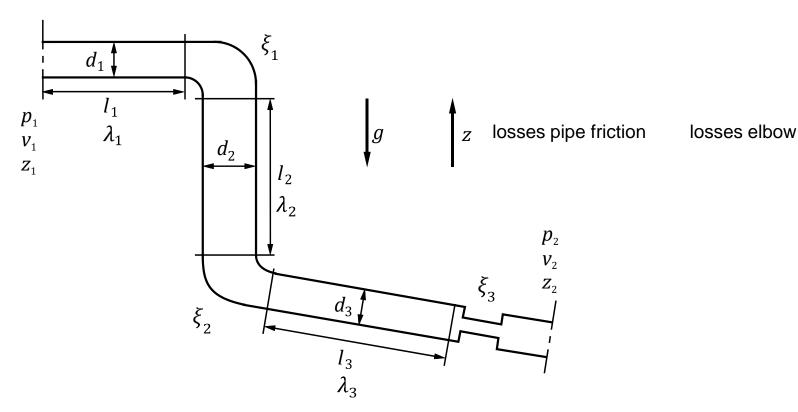
Pressure drop in a pipe assembly

Bernoulli with losses

condition 1

condition 2

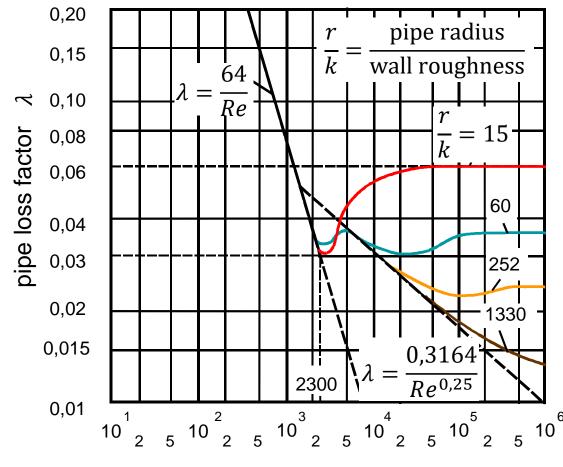
$$\left(p_1 + \frac{\rho}{2} \cdot v_1^2 + \rho \cdot g \cdot z_1\right) -$$







Loss factor for smooth pipes



laminar flow:

$$\lambda = \frac{64}{Re}$$

turbulent flow:

$$\lambda = \frac{0,3164}{Re^{0,25}}$$



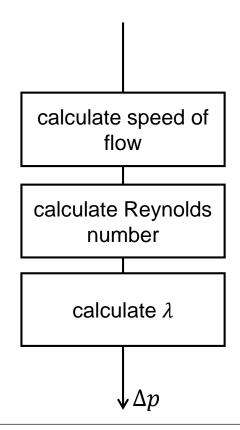




Calculation of pipe resistance according blasius

given: Δp wanted: Q estimate initial value for λ calculate speed of repeat if necessary flow with new λ calculate Reynolds number re-calculate λ

given: Q wanted: Δp







Comparison Flow Coefficient vs. Loss Factor

orifice equation:

$$Q = \alpha_{\rm D} \cdot A \cdot \sqrt{\frac{2 \cdot \Delta p}{\rho}}$$

$$\Rightarrow v = \alpha_{\rm D} \cdot \sqrt{\frac{2 \cdot \Delta p}{\rho}}$$

pressure drop:

$$\Delta p = \xi \cdot \frac{\rho}{2} \cdot v^2$$

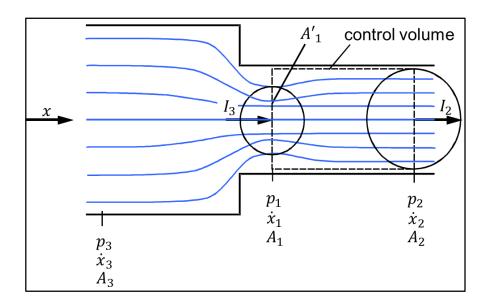
$$v = \sqrt{\frac{1}{\xi}} \cdot \sqrt{\frac{2 \cdot \Delta p}{\rho}}$$

comparison yields:

$$\alpha_{\rm D} = \sqrt{\frac{1}{\xi}}$$



Pressure Loss at Cross Section Changes



$$\Delta p_{V} = \frac{\rho}{2} \cdot \dot{x}_{2}^{2} \cdot \left(\frac{1}{\alpha_{K}} - 1\right)^{2}$$

$$\xi = \frac{\Delta p_{V}}{\frac{\rho}{2} \cdot \dot{x}_{2}^{2}} = \left(\frac{1}{\alpha_{K}} - 1\right)^{2}$$

$$\xi = \frac{\Delta p_{\rm V}}{\frac{\rho}{2} \cdot \dot{\chi}_2^2} = \left(\frac{1}{\alpha_{\rm K}} - 1\right)^2$$

shape of reduction	$lpha_{ m K}$	ξ
A_0 A_1	0,5	1
A_0 A_1	0,61 0,65	0,4 0,3
A_0 A_1	0,99	0





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What is a hydraulic capacity?



In a sealed volume:

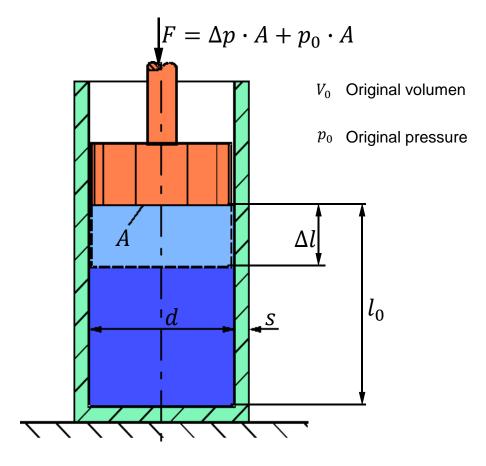
pressure build-up only because of change in volume or change in temperature

→ Compressibility of fluid influences pressure build-up and energy stored in fluid





Hydraulic Capacity



compressibility of a liquid

compressibility of the liquid

$$\Delta V_{\rm Fl} = A \cdot \Delta l = V_0 \cdot \frac{\Delta p}{E_{\rm Fl}}$$

bulk modulus

$$E_{\rm Fl} = \frac{1}{\beta} \left[\frac{\rm N}{\rm m^2} \right]$$

Bulk modulus is not const., better use differential writing:

$$\frac{\mathrm{d}V}{V_0} = -\frac{\mathrm{d}p}{E_{\mathrm{Fl}}}$$

Hydraulic capacity relates volumen change due to change in pressure

$$C_{\rm H} = -\frac{\mathrm{d}V}{\mathrm{d}p} = \frac{V_0}{E_{\rm Fl}} \qquad \left[\frac{\mathrm{m}^5}{\mathrm{N}}\right]$$

with
$$\mathrm{d}V = -Q \cdot \mathrm{d}t$$

with
$$dV = -Q \cdot dt$$
 $\Rightarrow p = \frac{1}{C_H} \cdot \int Q dt$

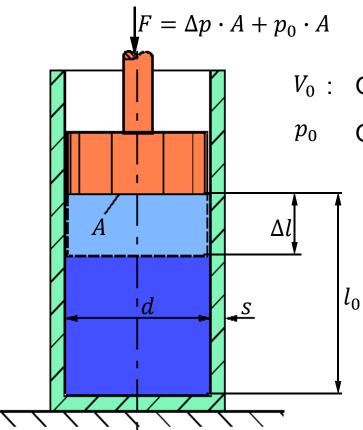
Electrical engineering:

$$u = \frac{1}{C} \cdot \int i dt$$





Hydraulic Capacity – Influences of compressibility



 V_0 : Original volumen

p₀ Original pressure

Bulk modulus

$$E_{\rm Fl} = \frac{1}{\beta} \left[\frac{\rm N}{\rm m^2} \right]$$

Not only compression of liquid!

1. elasticity of the pipe wall

$$\Delta V_{\rm R} = V_0 \cdot \frac{\Delta p}{E_{\rm R}} \cdot \frac{d}{s}$$

2. compressibility of air bubbles

$$\Delta V_{\rm L} = \frac{V_{\rm L,0}}{p_0} \cdot \frac{1}{\kappa} \cdot \Delta p$$

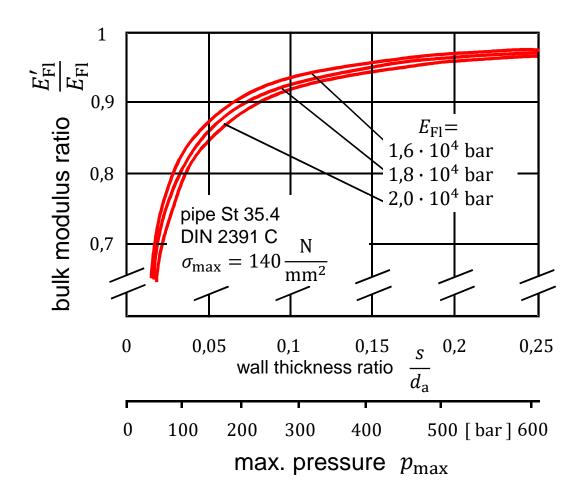
→ equivalent bulk modulus

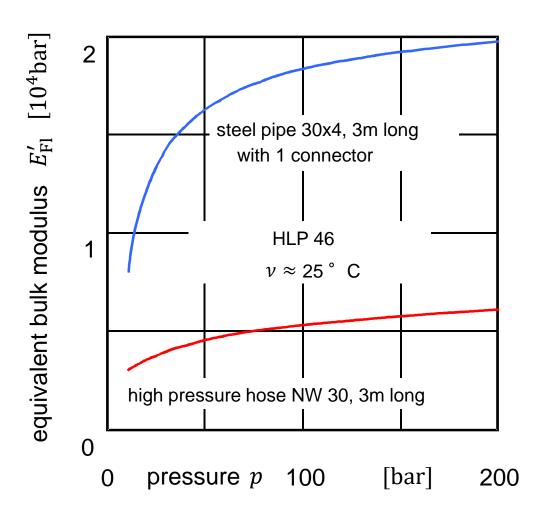
$$E'_{\mathrm{Fl}} = \frac{V_0 \cdot \Delta p}{\Delta V_{\mathrm{Fl}} + \Delta V_{\mathrm{R}} + \Delta V_{\mathrm{L}}} = \frac{1}{\frac{1}{E_{\mathrm{Fl}}} \cdot \left(1 + \frac{E_{\mathrm{Fl}}}{E_{\mathrm{R}}} \cdot \frac{d}{s}\right) + \frac{\Delta V_{\mathrm{L}}}{V_0 \cdot \Delta p}}$$





Influence of Pipe Elasticity on (Equivalent) Bulk Modulus

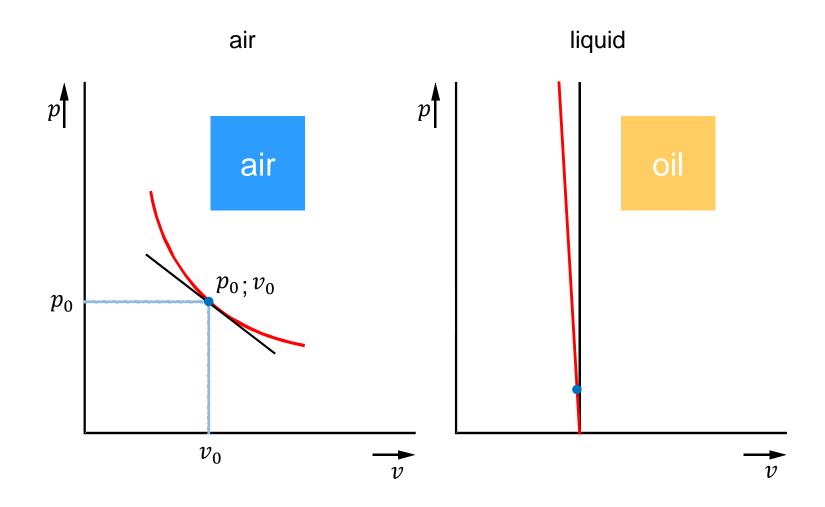








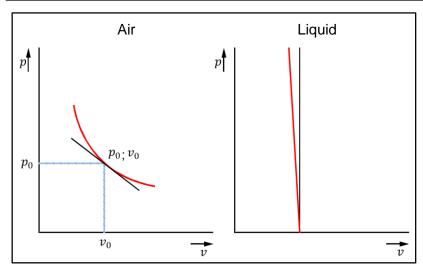
Comparison of Stiffness of Air and Liquid Volumes







Comparison of Stiffness of Air and Liquid Volumes



compression of an air volume:

$$\frac{\partial V}{\partial p} = -\frac{V_0}{p_0 \cdot \kappa}$$

compression of a liquid volume:

$$\frac{\partial V}{\partial p} = -\frac{V_0}{E_{\rm F}}$$

same stiffness if

$$p_{
m air}=rac{E_{
m Fl}}{\kappa}pprox 10^4{
m bar}$$



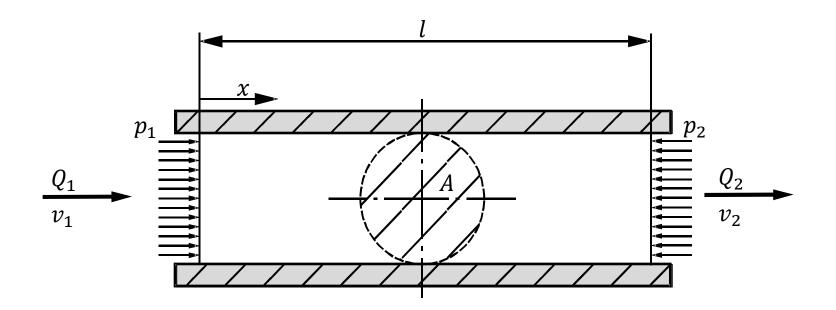
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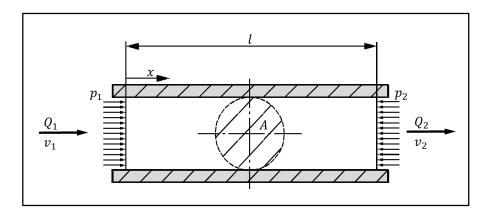
Hydraulic Inductivity of a Short Section of Pipe







Hydraulic Inductivity of a Short Section of Pipe



liquid with mass \Rightarrow hydraulic inductivity $L_{\rm H}$

$$F = m \cdot a = A \cdot l \cdot \rho \cdot a$$

$$a = \frac{\mathrm{d}v}{\mathrm{d}t} = \frac{1}{A} \cdot \frac{\mathrm{d}Q}{\mathrm{d}t}$$

$$\Rightarrow \Delta p = \underbrace{\frac{l \cdot \rho}{A}} \cdot \frac{\mathrm{d}Q}{\mathrm{d}t}$$

hydraulic inductivity $L_{
m H}$

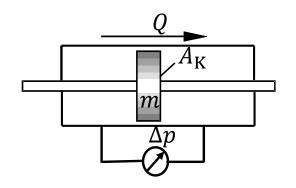
compare with electrical engineering:

$$u = L \cdot \frac{\mathrm{d}i}{\mathrm{d}t}$$





Hydraulic Inductivity of a Symmetric Cylinder



inductivity:

$$L_{\rm H} = \frac{\Delta p}{\dot{Q}}$$

(1)

flow "through" the cylinder:

$$Q = A_{K} \cdot \dot{x}$$

$$\Rightarrow \dot{Q} = A_{\rm K} \cdot \ddot{x}$$

(2)

(3)

Newton:

$$F = m \cdot \ddot{x} = \Delta p \cdot A_{K}$$

$$\Rightarrow \ddot{x} = \frac{\Delta p \cdot A_{K}}{m}$$

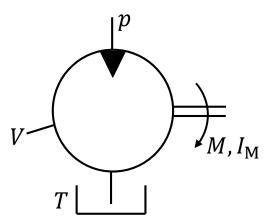
$$L_{\rm H} = \frac{\Delta p}{\frac{A_{\rm K}^2}{m} \cdot \Delta p} = \frac{m}{A_{\rm K}^2}$$





Hydraulic Inductivity of a Rotating Motor

hydro motor



$$M = I_{\rm M} \cdot \dot{\omega} = \frac{V}{2\pi} \cdot \Delta p \bigcirc$$

$$Q = \frac{V}{2\pi} \cdot \omega$$

$$\frac{V}{2\pi} \cdot \Delta p = I_{\mathbf{M}} \cdot \frac{2\pi \cdot \dot{Q}}{V}$$

$$L_{
m H} = rac{\Delta p}{\dot{Q}} \quad ext{ yields } \quad L_{
m M} = rac{I_{
m M}}{\left(rac{V}{2\pi}
ight)^2}$$



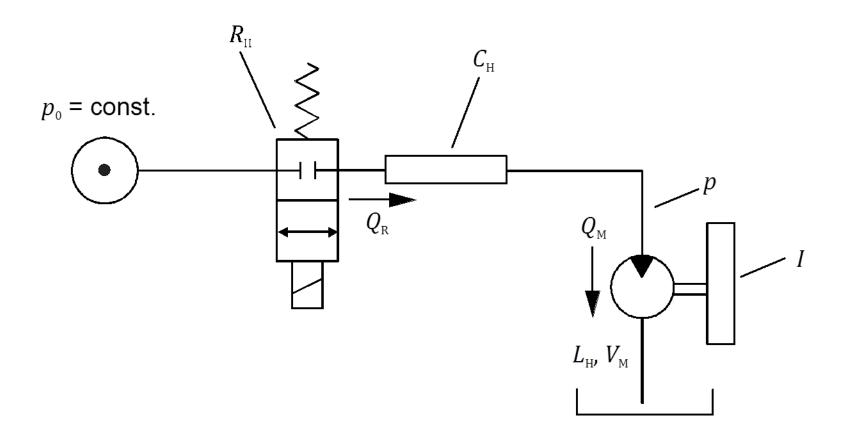
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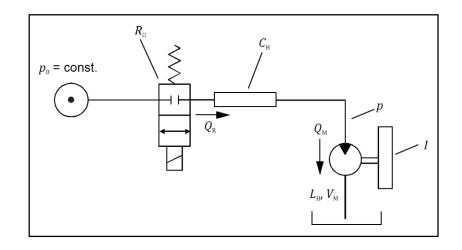
Calculation Example for a Hydraulic Network







Hydraulic Network



pressure rise in the capacity

$$\dot{p} = \frac{1}{C_{\rm H}} \cdot \sum_{i} Q_{i} = \frac{1}{C_{\rm H}} \cdot (Q_{\rm R} - Q_{\rm M})$$

$$\Rightarrow \dot{p} = \frac{1}{C_{\rm H}} \cdot (Q_{\rm R} - Q_{\rm M})$$

inflow over the valve

$$Q_{\rm R} = \frac{1}{R_{\rm H}} \cdot (p_0 - p)$$

$$\Rightarrow Q_{\rm R} = -\frac{1}{R_{\rm H}} \cdot p$$

acceleration of the motor

$$\dot{Q}_{\rm M} = \frac{p}{L_{\rm H}}$$

Insertion yields pressure rise equation:

$$\ddot{p} + \frac{1}{R_{\rm H} \cdot C_{\rm H}} \cdot \dot{p} + \frac{1}{C_{\rm H} \cdot L_{\rm H}} \cdot p = 0$$

eigen angular frequency:

$$\omega_0 = \sqrt{\frac{1}{L_{\rm H} \cdot C_{\rm H}}}$$

damping:

$$D_{\rm H} = \frac{1}{2R_{\rm H}} \cdot \sqrt{\frac{L_{\rm H}}{C_{\rm H}}}$$





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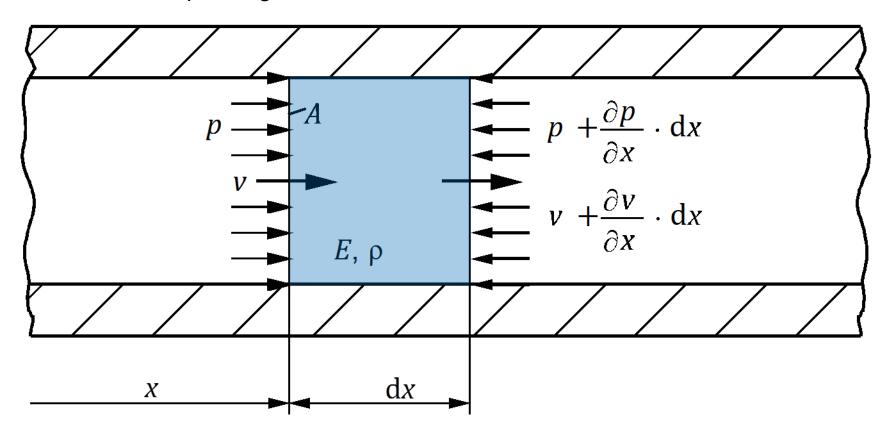




Transition from Lumped to Distributed Parameters

Distributed parameters: pressure & flow rate depending on time & location

Due to compression /
expansion
Due to accelleration /
decelleration
→ resonances in the system







Basics for liquid collumn

Velocity of pressure transfer in a pipe → speed of sound in oil

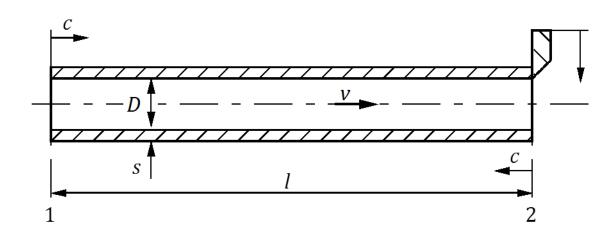
$$c = \sqrt{\frac{E'_{\rm Fl}}{\rho}}$$

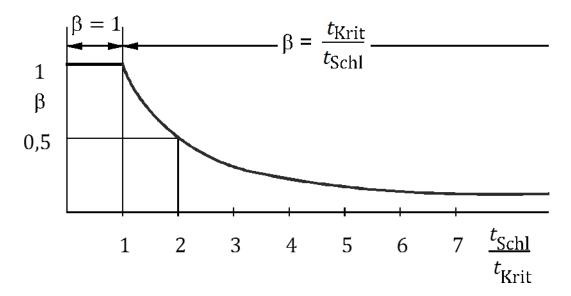
Pressure waves travel with speed of sound through pipes and systems





Pressure Surge in a Pipe when closing valve





Fast closing of a valve results in pressure surge (Joukowsky-surge)

$$t_{\rm Schl} \le t_{\rm Krit}$$
 $t_{\rm Krit} = \frac{2 \cdot l}{c}$ (2 x pipe passage)

$$\Delta p = \rho \cdot c \cdot \Delta v$$

Slower closing results in a reduced reflected wave reducing pressure surge

$$\Delta p = \frac{t_{\text{Krit}}}{t_{\text{Schl}}} \cdot \rho \cdot c \cdot \Delta v$$





Example for pressure surge

- Given.: steel pipe 25x3, 4m length
 Liquid: E'_{FI} = 1,6 ·10⁴ bar, ρ = 850 kg/m³, c = 1370 m/s
- Critical closing time of the valve is:

$$t_{\text{Krit}} = \frac{2 \cdot l}{c} = 5.8 \text{ ms}$$

• When closing the valve in faster than 5,8 ms, the pressure surge with a change in velocity of $\Delta v = 3$ m/s has a value

$$\Delta p = \rho \cdot c \cdot \Delta v = 35 \ bar$$

• When closing the valve slower, e.g. with a closing time of 20 ms, the pressure surge reduces

$$\Delta p = \frac{5.8 \, ms}{20 \, ms} \cdot 35 \, bar = 10 \, bar$$



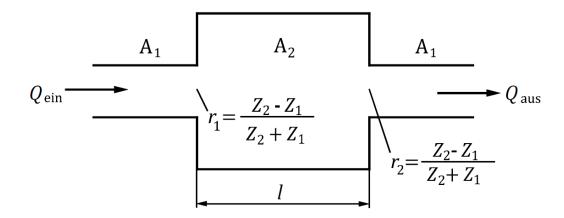


Dynamic consideration often necessary

- Wave propagation in systems:
 - impair the control behavior
 - Lead to noise
 - Impair the life of the components
 - Lead to "shaking loose" of fittings

Remedies:

- Meaningful dimensioning of piping / system (natural frequency of pipes / liquid column and connected components must not be too close to each other)
- Reflection Muffler (Silincer)



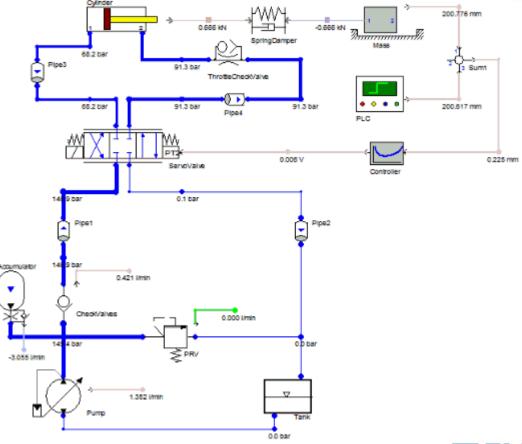




Simulation of hydraulic systems

- System simulation consists of
 - capacities
 - resistances
 - inductances
- Visualisation of dynamic effects in system











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Zusammenfassung

- What is a hydraulic resistance?
 - Throtteling of the flow that leads to pressure loss

$$\Delta p = Q \cdot R_{\rm H}$$

- Which basic resistances exist and what are the differences?
 - Orifice: turbulent resistance, temperature independent; Throttle: laminar resistance, temperature dependent
- What is a hydraulic capacity?
 - Ratio of volumen change with corresponding pressure change

$$C_{\rm H} = -\frac{\mathrm{d}V}{\mathrm{d}p} = \frac{V_0}{E_{\rm Fl}} \qquad \left[\frac{\mathrm{m}^5}{\mathrm{N}}\right] \qquad \Rightarrow p = \frac{1}{C_{\rm H}} \cdot \int Q \mathrm{d}t$$

- What is a hydraulic inductivity?
 - Resistance against acceleration

$$L_H = \frac{l \cdot \rho}{A} \qquad \Rightarrow \Delta p = L_H \cdot \frac{\mathrm{d}Q}{\mathrm{d}t}$$

• What is the Joukowski surge? $\Delta p = \rho \cdot c \cdot \Delta v$





Thank you for your attention.



