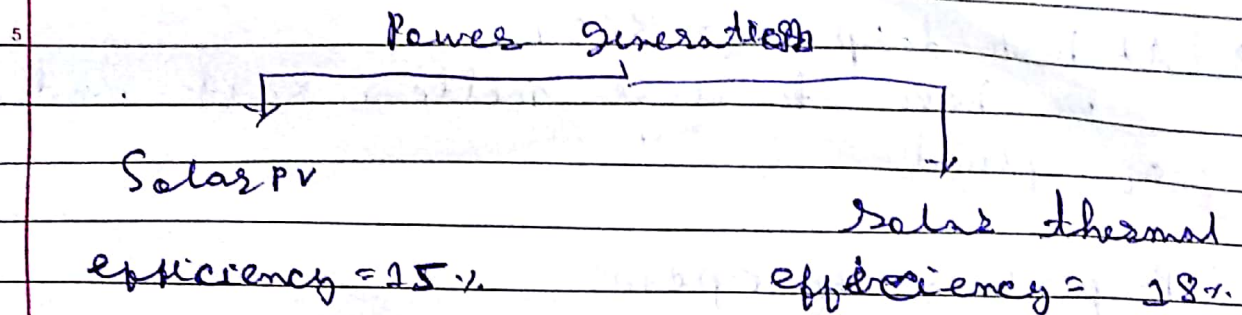


★ Dynamic Behavior of Medium temp. ( $200^{\circ}\text{C} - 250^{\circ}\text{C}$ ) Solar thermal power plant integrated with latent heat thermal energy storage.



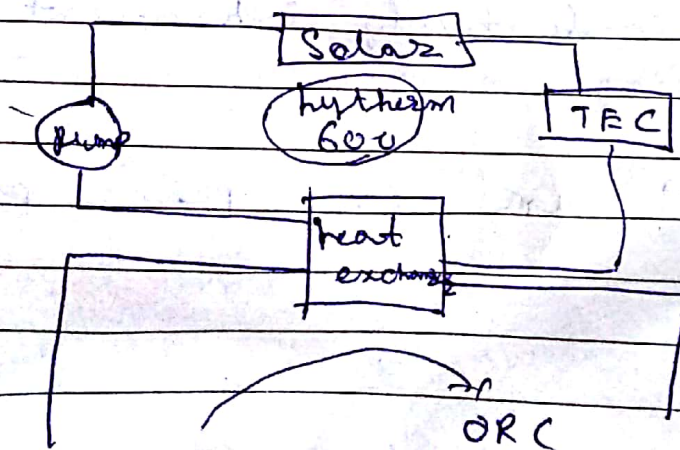
Target:

Temp range =  $900^{\circ}\text{C}$   
challenging power 100 kW - 1 MW

ORC  $\rightarrow$  organic Rankine cycle  
Advantage over Rankine

- (i) Long service life
- (ii) Low maintenance
- (iii) Full automatic
- (iv) Can produce shaft work from medium heat source upto  $370^{\circ}\text{C}$  efficiency

Why thermal storage?



- Ceramic ( $Al_2O_3$ ) → used to store heat upto  $1500^\circ C - 2000^\circ C$
- Thermal Conductivity  $Cu = 390 W/m.k$

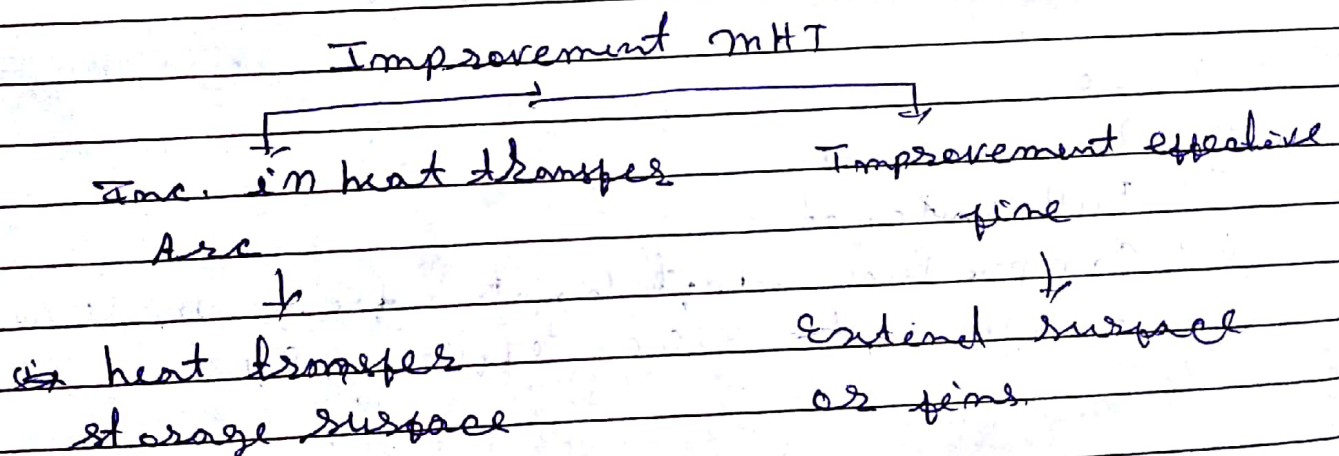
→ At high temp  $T > 360^\circ C$  we have to use molten salt instead of fluid

\* Properties of PCM:

- Most organic PCM with low melting point ( $\sim 0.2$ )

Why we use organic?

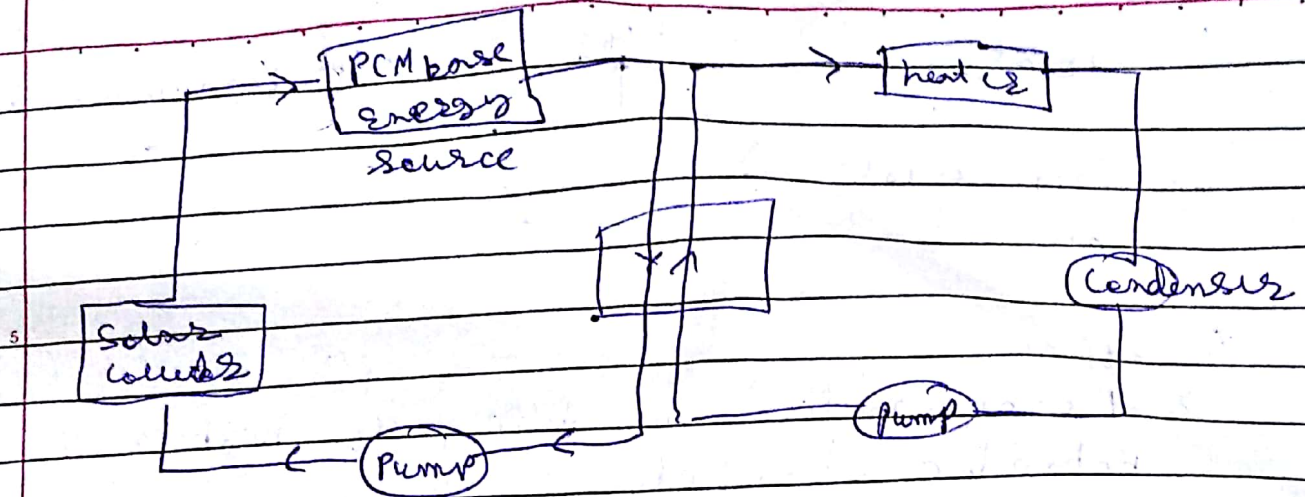
- Freeze without much supercooling



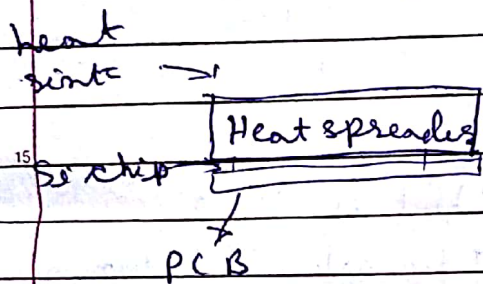
- Thermal conductivity Enhancement by encapsulating in 35316

\* Enconil → used in power plant

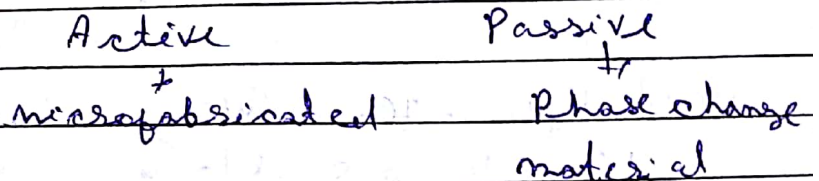




## Thermal management



## Cooling technology



\* Micro-channel: has more area/volume ratio  
Result in high heat transfer  
Dimple shaped collectors

\* Fabrication of microchannel (Silicon)

Cleaning  $\rightarrow$  acid  $\rightarrow$  photolithography  $\rightarrow$  etching  
(thickness 75 - 100  $\mu$ m) Waterbending  $\leftarrow$  heater deposition  $\leftarrow$

\* ~~Effect~~

most effective shape for micro-channel

trapezoidal channel



\* Effect

- Pressure medium modelling of high scale
- Rhombic channel

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~~Reynolds~~ Reynolds no  $Re = \frac{\mu u \tau}{\mu_p}$

heat transfer co-efficient =  $\frac{P_m}{T_{max} - T_{bulk}}$

\* <sup>15</sup> Two phase flow pattern:

- (i) Heat flux 10.70 W/cm<sup>2</sup> Flow rate 3 ml/min
- (ii) Heat flux = 13.18 W/cm<sup>2</sup> Flow rate = 3 ml/min

- (iii) Heat flux = 16.39 W/cm<sup>2</sup>
- <sup>20</sup> Flow rate = 3.5 ml/min

\* ~~Experimental CHF~~

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