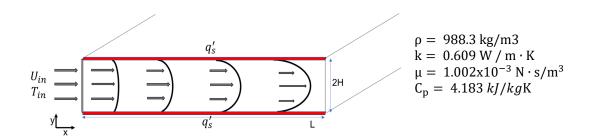
MEEN 644- Numerical Heat Transfer and Fluid Flow Spring 2018

Homework 5

Name		
	Instructor: N. K.	Anand

Due Date: April 15, 2019

Maximum Points: 100



Consider an incompressible laminar flow between two infinite parallel plates. Flow enters at constant velocity U_{in} and constant heat flux q's of 500 W/m² is applied to each wall. Height of channel 2H and length of channel L are 0.02 m and 2.0 m, respectively. Constant velocity U_{in} and temperature T_{in} of 25 C is set as inlet condition. Reynold number is defined as $Re = \rho U_{in}(2H)/\mu$.

- (1) Specify your boundary condition for U, V, P, and T (10 Points).
- (2) Write a finite volume method code to predict velocity, pressure and temperature profiles for Re = 100. Employ Power Law scheme to represent a solution to 1-D convection diffusion equation. Use the <u>SIMPLE algorithm</u> to link velocity and pressure fields. Use 10 uniformly sized CVs in X-direction and 5 uniformly sized CVs in Y- direction (10 X 5). Declare convergence at R_u, R_v R_p and R_T< 10⁻⁶ (50 Points).
- (3) Run your program with 10x5 grid and tabulate U, (P-P₀) and T fields to check for symmetry. P₀ is pressure at outlet. (10 Points).
- (4) Use 180 X 54 grid to solve for U, V, P, and T (30 Points).
 - (a) Plot U/U_{in} along the centerline of the channel (as function of x).
 - (b) Plot U, V, and T profile at x = 0.8 m as function of y.
 - (c) Plot Nusselt number as function of stream-wise distance X from the channel entrance.

Definition of Residuals:

$$\begin{split} \mathbf{R}_{\mathbf{u}} &= \frac{\sum_{node} \left| a_{p} u_{p} - \sum a_{nb} u_{nb} - b_{u} - A_{U} (P_{i,j} - P_{i+1,j}) \right|}{\sum_{node} \left| a_{p} u_{p} \right|} \\ \mathbf{R}_{\mathbf{v}} &= \frac{\sum_{node} \left| a_{p} v_{p} - \sum a_{nb} v_{nb} - b_{v} - A_{V} (P_{i,j} - P_{i,j+1}) \right|}{\sum_{node} \left| a_{p} V_{p} \right|} \\ \mathbf{R}_{\mathbf{p}} &= \frac{\sum_{node} \left| (\rho_{w} u_{w} - \rho_{e} u_{e}) dy + (\rho_{s} v_{s} - \rho_{n} v_{n}) dx \right|}{\rho u_{ref} L_{ref}} \\ \mathbf{R}_{T} &= \frac{\sum_{node} \left| a_{p} T_{p} - \sum a_{nb} T_{nb} - b_{T} \right|}{\sum_{node} \left| a_{p} T_{p} \right|} \end{split}$$

Reference

- 1. Patankar, S. V., and D. Brian Spalding. "A calculation procedure for heat, mass and momentum transfer in three-dimensional parabolic flows. "International Journal of Heat and Mass Transfer 15.10 (1972): 1787-1806.
- 2. Shah, R. K., and A. L. London. "Laminar Flow Forced Convection in Ducts: a Source Book for Compact Heat Exchanger Analytical Data, Supplement. 1." (1978).