ENGT 320 Applied Thermal Systems
Quiz 3 Formula Sheet

$$m = \rho \forall$$
$$\dot{\forall} = \frac{\forall}{t}$$
$$q = m c_p \Delta T$$
$$q = -kA\frac{dT}{dt}$$
$$q = hA\left(T_s - T_\infty\right)$$
$$q = \varepsilon \sigma A\left(T_s^4 - T_{sur}^4\right)$$
$$\sigma = 5.67 \times 10^{-8}\frac{W}{m^2K^4}$$
$$\Delta E_{sys} = \Sigma E_{in} - \Sigma E_{out}$$
$$\frac{dE_{sys}}{dt} = \Sigma \dot{E}_{in} - \Sigma \dot{E}_{out}$$
$$q'' = \frac{q}{A}$$
$$\hat{q}'' = -k_x\frac{\partial T}{\partial x}\hat{i} - k_y\frac{\partial T}{\partial y}\hat{j} - k_z\frac{\partial T}{\partial z}\hat{k}$$
$$k = \frac{9\gamma-5}{4}\frac{c_v}{\pi d^2}\sqrt{\frac{M_w k_B T}{N_A \pi}}$$
$$k_B = 1.381 \times 10^{-23}\frac{J}{K}$$
$$N_A = 6.022 \times 10^{23}$$
$$\alpha = \frac{k}{\rho c_p}$$
$$q_{x+dx} = q_x + \frac{\partial q_x}{\partial x}dx$$
$$q_{y+dy} = q_y + \frac{\partial q_y}{\partial y}dy$$
$$q_{z+dz} = q_z + \frac{\partial q_z}{\partial z}dz$$

$$\frac{\partial}{\partial x}\left(k\frac{\partial T}{\partial x}\right) + \frac{\partial}{\partial y}\left(k\frac{\partial T}{\partial y}\right) + \frac{\partial}{\partial z}\left(k\frac{\partial T}{\partial z}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$

$$\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} + \frac{\dot{q}}{k} = \frac{1}{\alpha}\frac{\partial T}{\partial t}$$

$$\frac{\partial}{\partial x}\left(k\frac{\partial T}{\partial x}\right) + \frac{\partial}{\partial y}\left(k\frac{\partial T}{\partial y}\right) + \frac{\partial}{\partial z}\left(k\frac{\partial T}{\partial z}\right) + \dot{q} = 0$$

$$\frac{\partial}{\partial x}\left(k\frac{\partial T}{\partial x}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$
$$\frac{\partial}{\partial y}\left(k\frac{\partial T}{\partial y}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$
$$\frac{\partial}{\partial z}\left(k\frac{\partial T}{\partial z}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$

$$\frac{1}{r}\frac{\partial}{\partial r}\left(kr\frac{\partial T}{\partial r}\right) + \frac{1}{r^2}\frac{\partial}{\partial \phi}\left(k\frac{\partial T}{\partial \phi}\right) + \frac{\partial}{\partial z}\left(k\frac{\partial T}{\partial z}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$

$$\frac{1}{r}\frac{\partial}{\partial r}\left(r\frac{\partial T}{\partial r}\right) + \frac{1}{r^2}\frac{\partial^2 T}{\partial \phi^2} + \frac{\partial^2 T}{\partial z^2} + \frac{\dot{q}}{k} = \frac{1}{\alpha}\frac{\partial T}{\partial t}$$

$$\frac{1}{r}\frac{\partial}{\partial r}\left(kr\frac{\partial T}{\partial r}\right) + \frac{1}{r^2}\frac{\partial}{\partial \phi}\left(k\frac{\partial T}{\partial \phi}\right) + \frac{\partial}{\partial z}\left(k\frac{\partial T}{\partial z}\right) + \dot{q} = 0$$

$$\frac{1}{r}\frac{\partial}{\partial r}\left(kr\frac{\partial T}{\partial r}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$
$$\frac{1}{r^2}\frac{\partial}{\partial \phi}\left(k\frac{\partial T}{\partial \phi}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$
$$\frac{\partial}{\partial z}\left(k\frac{\partial T}{\partial z}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$

$$\frac{1}{r^2}\frac{\partial}{\partial r}\left(kr^2\frac{\partial T}{\partial r}\right) + \frac{1}{r^2\sin^2\theta}\frac{\partial}{\partial \phi}\left(k\frac{\partial T}{\partial \phi}\right) + \frac{1}{r^2\sin\theta}\frac{\partial}{\partial \theta}\left(k\sin\theta\frac{\partial T}{\partial \theta}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$

$$\frac{1}{r^2}\frac{\partial}{\partial r}\left(r^2\frac{\partial T}{\partial r}\right) + \frac{1}{r^2\sin^2\theta}\frac{\partial^2 T}{\partial \phi^2} + \frac{1}{r^2\sin\theta}\frac{\partial}{\partial \theta}\left(\sin\theta\frac{\partial T}{\partial \theta}\right) + \frac{\dot{q}}{k} = \frac{1}{\alpha}\frac{\partial T}{\partial t}$$

$$\frac{1}{r^2}\frac{\partial}{\partial r}\left(kr^2\frac{\partial T}{\partial r}\right) + \frac{1}{r^2\sin^2\theta}\frac{\partial}{\partial \phi}\left(k\frac{\partial T}{\partial \phi}\right) + \frac{1}{r^2\sin\theta}\frac{\partial}{\partial \theta}\left(k\sin\theta\frac{\partial T}{\partial \theta}\right) + \dot{q} = 0$$

$$\frac{1}{r^2}\frac{\partial}{\partial r}\left(kr^2\frac{\partial T}{\partial r}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$
$$\frac{1}{r^2\sin^2\theta}\frac{\partial}{\partial \phi}\left(k\frac{\partial T}{\partial \phi}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$
$$\frac{1}{r^2\sin\theta}\frac{\partial}{\partial \theta}\left(k\sin\theta\frac{\partial T}{\partial \theta}\right) + \dot{q} = \rho c_p\frac{\partial T}{\partial t}$$
$$T\left(0,t\right) = T_s$$
$$-k\left.\frac{\partial T}{\partial x}\right|_{x=0} = q_s''$$
$$\left.\frac{\partial T}{\partial x}\right|_{x=0} = 0$$
$$-k\left.\frac{\partial T}{\partial x}\right|_{x=0} = h\left(T_\infty - T\left(0,t\right)\right)$$

$$\Delta U = N\left(V^- - V^+\right)$$
$$q = \frac{\Delta U}{\Delta t} = \frac{N}{\Delta t}\Delta V$$
$$q = -IV = \frac{-V^2}{R} = -I^2R$$
$$R = \frac{\rho L}{A} = \frac{L}{\sigma A}$$
$$S_{AB} = S_B - S_A = \frac{-\Delta V}{\Delta T}$$
$$C_1 = \frac{T_{s,2} - T_{s,1}}{L}$$
$$C_2 = T_{s,1}$$
$$T\left(x\right) = \left(T_{s,2} - T_{s,1}\right)\frac{x}{L} + T_{s,1}$$
$$q = \frac{kA}{L}\left(T_{s,1} - T_{s,2}\right)$$
$$R_{t,cond} = \frac{T_{s,1} - T_{s,2}}{q} = \frac{L}{kA}$$
$$R = \frac{V}{I}$$
$$R_{t,conv} = \frac{T_s - T_\infty}{q} = \frac{1}{hA}$$

$$R_{t,rad} = \frac{T_s - T_{sur}}{q} = \frac{1}{h_r A}$$
$$h_r = \varepsilon\sigma\left(T_s + T_{sur}\right)\left(T_s^2 + T_{sur}^2\right)$$
$$q = UA\Delta T$$
$$U = \frac{1}{R_{tot}A}$$
$$R_{tot} = \Sigma R_t = \frac{1}{UA}$$
$$R_{t,c}'' = \frac{T_A - T_B}{q''}$$
$$q = \frac{k_{eff}A}{L}\left(T_1 - T_2\right)$$
$$k_{eff,min} = \frac{1}{\frac{1-\varepsilon}{k_s} + \frac{\varepsilon}{k_f}}$$
$$k_{eff,max} = \varepsilon k_f + \left(1-\varepsilon\right)k_s$$
$$R_{t,cond} = \frac{\ln\frac{r_2}{r_1}}{2\pi kL}$$
$$R_{t,conv} = \frac{1}{h2\pi rL}$$

$$\frac{d^2T}{dx^2} + \left(\frac{1}{A_c}\frac{dA_c}{dx}\right)\frac{dT}{dx} - \left(\frac{1}{A_c}\frac{h}{k}\frac{dA_s}{dx}\right)\left(T - T_\infty\right) = 0$$

$$A_s = Px$$
$$\frac{d^2T}{dx^2} - \left(\frac{hP}{kA_c}\right)\left(T - T_\infty\right)$$
$$\frac{d^2\theta}{dx^2} - m^2\theta = 0$$
$$\theta = T\left(x\right) - T_\infty$$
$$m^2 = \frac{hP}{kA_c}$$
$$P = 2w + 2t$$
$$A_c = wt$$
$$P = \pi D$$
$$A_c = \frac{\pi}{4}D^2$$
$$\theta\left(x\right) = C_1e^{mx} + C_2e^{-mx}$$
$$\theta_b = T_b - T_\infty$$
$$M = \sqrt{hPKA_c}\theta_b$$
$$\varepsilon_f = \frac{q_f}{hA_{c,b}\theta_b}$$
$$\varepsilon_f = \sqrt{\frac{kP}{hA_c}}$$
$$R_{t,f} = \frac{\theta_b}{q_f}$$
$$R_{t,b} = \frac{1}{hA_{c,b}}$$
$$\varepsilon_f = \frac{R_{t,b}}{R_{t,f}}$$
$$\eta_f = \frac{q_f}{q_{max}} = \frac{q_f}{hA_f\theta_b}$$
$$\eta_o = \frac{q_t}{q_{max}} = \frac{q_t}{hA_t\theta_b}$$
$$A_t = NA_f + A_b$$
$$q_t = N\eta_f hA_f\theta_b + hA_b\theta_b$$

$$\eta_o = 1 - \frac{NA_f}{A_t}\left(1-\eta_f\right)$$
$$R_{t,o} = \frac{\theta_b}{q_t} = \frac{1}{\eta_o h A_t}$$

$$\frac{u(y)}{u_\infty} = 0.99$$
$$C_f = \frac{\tau_s}{\rho u_\infty^2/2}$$
$$\tau_s = \mu \frac{\partial u}{\partial y}\Big|_{y=0}$$
$$\frac{T_s - T(y)}{T_s - T_\infty} = 0.99$$
$$q_s^{''} = -k\frac{\partial T}{\partial y}\Big|_{y=0}$$
$$q_s^{''} = h\left(T_s - T_\infty\right)$$
$$h = \frac{-k\frac{\partial T}{\partial y}\Big|_{y=0}}{T_s - T_\infty}$$
$$q = \int_{A_s} q^{''} dA_s = \left(T_s - T_\infty\right)\int_{A_s} h dA_s$$
$$\overline{q}^{''} = \overline{h}\left(T_s - T_\infty\right)$$
$$\overline{q} = \overline{q}^{''} A_s = \overline{h} A_s\left(T_s - T_\infty\right)$$
$$\overline{h} = \frac{1}{A_s}\int_{A_s} h dA_s$$
$$\overline{h} = \frac{1}{L}\int_0^L h dx$$
$$Re_x = \frac{\rho u_\infty x}{\mu}$$
$$x^* = \frac{x}{L}$$
$$y^* = \frac{y}{L}$$
$$u^* = \frac{u}{V}$$
$$\mathrm{v}^* = \frac{\mathrm{V}}{V}$$
$$T^* = \frac{T - T_s}{T_\infty - T_s}$$
$$p^* = \frac{p_\infty}{\rho V^2}$$
$$u^*\frac{\partial u^*}{\partial x^*} + \mathrm{v}^*\frac{\partial u^*}{\partial y^*} = -\frac{dp^*}{dx^*} + \frac{1}{Re_L}\frac{\partial^2 u^*}{\partial y^{*2}}$$
$$Re_L = \frac{VL}{\nu}$$
$$\nu = \frac{\mu}{\rho}$$
$$u^*\frac{\partial T^*}{\partial x^*} + \mathrm{v}^*\frac{\partial T^*}{\partial y^*} = \frac{1}{Re_L Pr}\frac{\partial^2 T^*}{\partial y^{*2}}$$
$$Pr = \frac{\nu}{\alpha}$$
$$Nu = \frac{hL}{k} = \frac{\partial T^*}{\partial y^*}\Big|_{y^*=0}$$
$$\overline{Nu} = \frac{\overline{h}L}{k}$$
$$T_f = \frac{T_s + T_\infty}{2}$$

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Isothermal Flat Plate

$$\frac{\delta}{\delta_t} \approx Pr^{1/3}$$

Laminar:

$$\delta = \frac{5}{\sqrt{\frac{u_\infty}{\nu x}}} = \frac{5x}{\sqrt{Re_x}}$$
$$C_{f,x} = \frac{\tau_{s,x}}{\frac{\rho u_\infty^2}{2}} = 0.664 Re_x^{-1/2}$$
$$Nu_x = \frac{h_x x}{k} = 0.332 Re_x^{1/2} Pr^{1/3}$$
$$(Pr \gtrsim 0.6)$$
$$\overline{C}_{f,x} = 1.328 Re_x^{-1/2}$$
$$\overline{Nu}_x = \frac{\overline{h}_x x}{k} = 0.664 Re_x^{1/2} Pr^{1/3}$$
$$(Pr \gtrsim 0.6)$$

Turbulent:

$$\delta = 0.37 x Re_x^{-1/2}$$
$$C_{f,x} = 0.0592 Re_x^{-1/5}$$
$$\left(Re_{x,c} \lesssim Re_x \lesssim 10^8\right)$$

$$Nu_x = 0.0296 Re_x^{4/5} Pr^{1/3}$$
$$(0.6 \lesssim Pr \lesssim 60)$$

$$A = 0.037 Re_{x,c}^{4/5} - 0.664 Re_{x,c}^{1/2}$$
$$\overline{C}_{f,L} = 0.074 Re_L^{-1/5} - \frac{2A}{Re_L}$$
$$\left(Re_{x,c} \lesssim Re_L \lesssim 10^8\right)$$

$$\overline{Nu}_L = \left(0.037 Re_L^{4/5} - A\right) Pr^{1/3}$$
$$(0.6 \lesssim Pr \lesssim 60)$$
$$\left(Re_{x,c} \lesssim Re_L \lesssim 10^8\right)$$

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$$Re_D = \frac{\rho V D}{\mu} = \frac{VD}{\nu}$$
$$C_D = \frac{F_D}{A_f\left(\frac{\rho V^2}{2}\right)}$$

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Cylinder

$$\overline{Nu}_D = \frac{\overline{h}D}{k} = C Re_D^m Pr^{1/3}$$
$$(Pr \gtrsim 0.7)$$

See Table 4

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Various Geometries

$$\overline{Nu}_D = \frac{\overline{h}D}{k} = C Re_D^m Pr^{1/3}$$
$$(Pr \gtrsim 0.7)$$

See Table 5

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Sphere

$$A_s = \pi D^2$$
$$C_D = \frac{25}{Re_D}$$
$$(Re_D \lesssim 0.5)$$

$$\overline{Nu}_D = 2 + \left(0.4 Re_D^{1/2} + 0.06 Re_D^{2/3}\right) Pr^{0.4}\left(\frac{\mu}{\mu_s}\right)^{1/4}$$
$$(0.71 \lesssim Pr \lesssim 380)$$
$$\left(3.5 \lesssim Re_D \lesssim 7.6 \times 10^4\right)$$
$$\left(1.0 \lesssim \frac{\mu}{\mu_s} \lesssim 3.2\right)$$

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$$m = \rho u_m A_c$$
$$Re_D = \frac{\rho u_m D}{\mu} = \frac{u_m D}{\nu}$$
$$\left(\frac{x_{fd,t}}{D}\right) \approx 0.05 Re_D Pr$$
$$f = \frac{-\frac{dP}{dx}D}{\rho\frac{u_m^2}{2}}$$
$$f = \frac{64}{Re_D}$$
$$\frac{1}{\sqrt{f}} = -2.0\log\left[\frac{e/D}{3.7} + \frac{2.51}{Re_D\sqrt{f}}\right]$$

$$dq_{conv} = mc_p\left[\left(T_m + dT_m\right) - T_m\right] = mc_p dT_m$$

$$dq_{conv} = q_s^{''} P dx$$
$$\frac{dT_m}{dx} = \frac{q_s^{''} P}{mc_p} = \frac{P}{mc_p} h\left(T_s - T_m\right)$$
$$T_m\left(x\right) = T_{m,i} + \frac{q_s^{''} P}{mc_p} x$$
$$\Delta T = T_s - T_m$$
$$\frac{dT_m}{dx} = -\frac{d(\Delta T)}{dx} = \frac{P}{mc_p} h \Delta T$$
$$\frac{T_s - T_m(x)}{T_s - T_{m,i}} = e^{-\frac{Px}{mc_p}\overline{h}}$$
$$\Delta T_{lm} = \frac{\Delta T_o - \Delta T_i}{\ln\left(\frac{\Delta T_o}{\Delta T_i}\right)}$$
$$\Delta T_o = T_s - T_{m,o}$$

$$\Delta T_i = T_s - T_{m,i}$$

$$q_{conv} = \overline{h} A_s \Delta T_{lm}$$

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Circular Tubes

Fully dev. lam. w/ $q'' = \mathbb{C}$:

$$Nu_D = 4.36$$

Fully dev. lam. w/ $T_s = \mathbb{C}$:

$$Nu_D = 3.66$$

Fully dev. turb. w/ small ΔT:

$$Nu_D = 0.023 Re_D^{4/5} Pr^n$$

$$n = 0.4, \quad T_s > T_m$$

$$n = 0.3, \quad T_s < T_m$$

$$(0.6 \lesssim Pr \lesssim 160)$$

$$(Re_D \gtrsim 10,000)$$

$$(L/D \gtrsim 10)$$

Fully dev. turb. w/ large ΔT:

$$Nu_D = 0.027 Re_D^{4/5} Pr^{1/3} \left(\frac{\mu}{\mu_s}\right)^{0.14}$$

$$(0.7 \lesssim Pr \lesssim 16,700)$$

$$(Re_D \gtrsim 10,000)$$

$$(L/D \gtrsim 10)$$

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Noncircular Tubes

$$D_h = \frac{4A_c}{P}$$

$$Nu_D = \frac{hD_h}{k}$$

See Table 6

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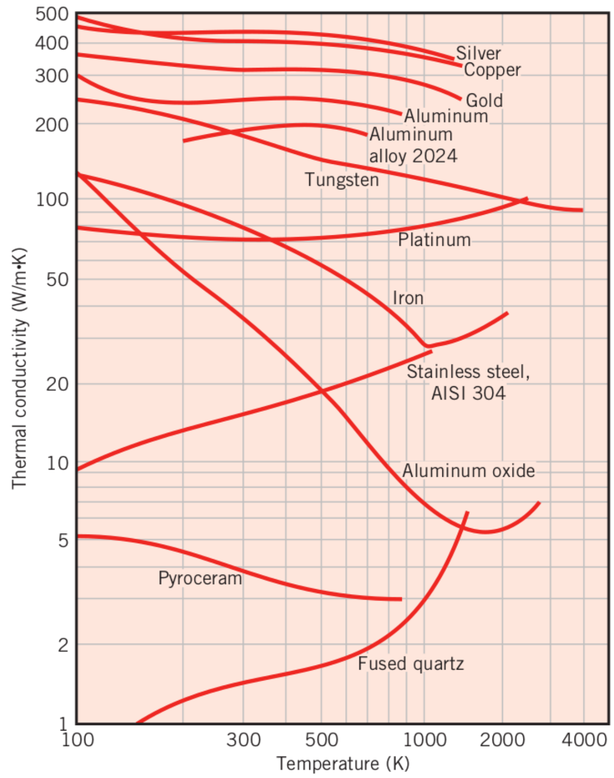
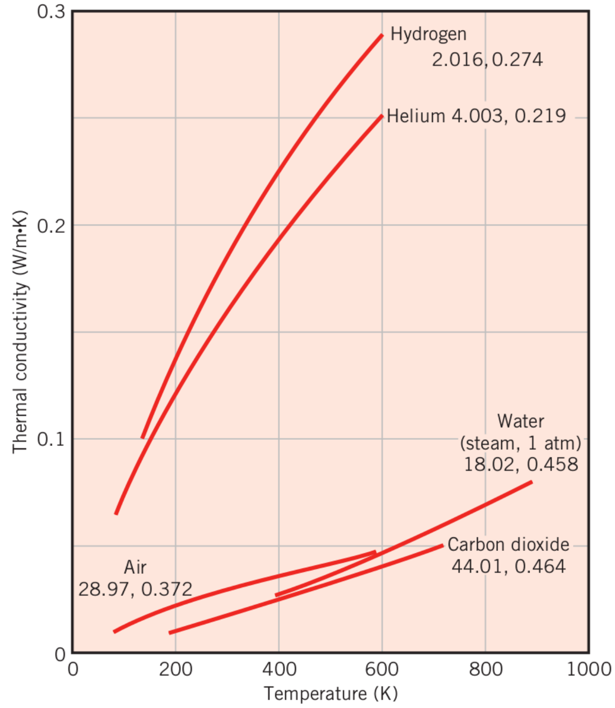
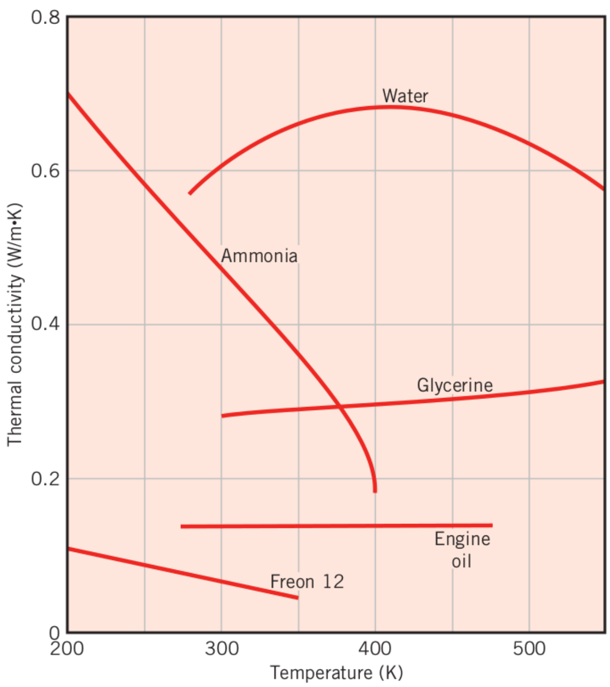
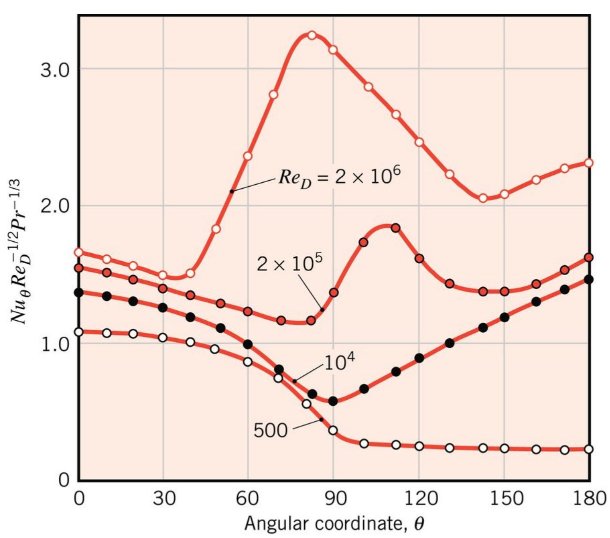
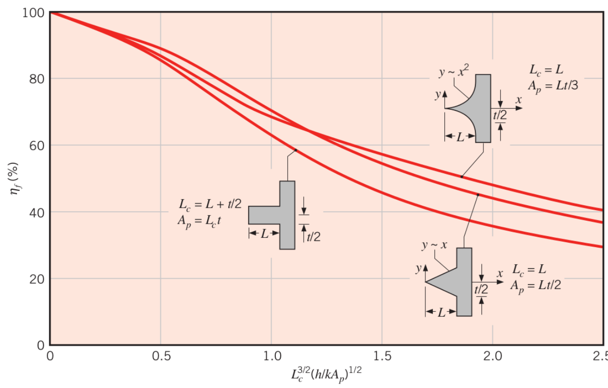
ENGT 320 Applied Thermal Systems
Quiz 3 Formula Sheet

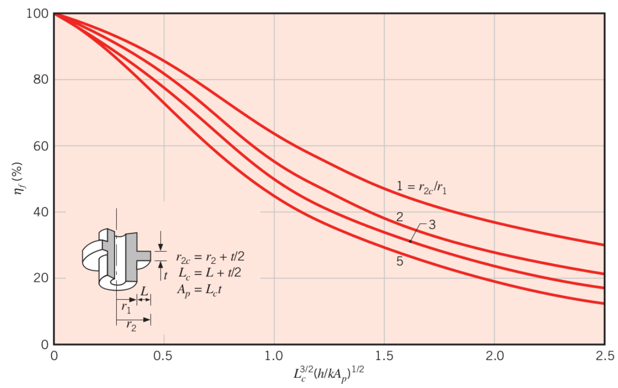












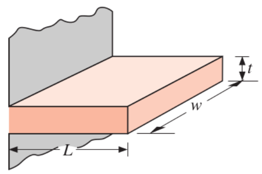
Straight Fins

Rectangular

$A_f = 2wL_c$

$L_c = L + (t/2)$

$A_p = tL$

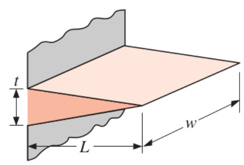


$$\eta_f = \frac{\tanh mL_c}{mL_c}$$

Triangular

$A_f = 2w[L^2 + (t/2)^2]^{1/2}$

$A_p = (t/2)L$



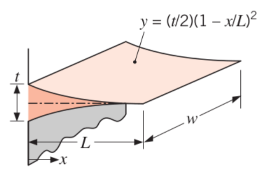
$$\eta_f = \frac{1}{mL}\frac{I_1(2mL)}{I_0(2mL)}$$

Parabolic

$A_f = w[C_1L + (L^2/t)\ln(t/L + C_1)]$

$C_1 = [1 + (t/L)^2]^{1/2}$

$A_p = (t/3)L$



$$\eta_f = \frac{2}{[4(mL)^2 + 1]^{1/2} + 1}$$

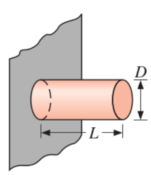
Pin Fins

Rectangular

$A_f = \pi DL_c$

$L_c = L + (D/4)$

$V = (\pi D^2/4)L$

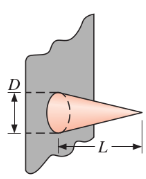


$$\eta_f = \frac{\tanh mL_c}{mL_c}$$

Triangular

$A_f = \dfrac{\pi D}{2}[L^2 + (D/2)^2]^{1/2}$

$V = (\pi/12)D^2L$



$$\eta_f = \frac{2}{mL}\frac{I_2(2mL)}{I_1(2mL)}$$

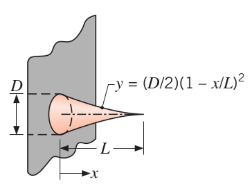
Parabolic

$$A_f = \frac{\pi L^3}{8D}\left\{C_3C_4 - \frac{L}{2D}\ln[(2DC_4/L) + C_3]\right\}$$

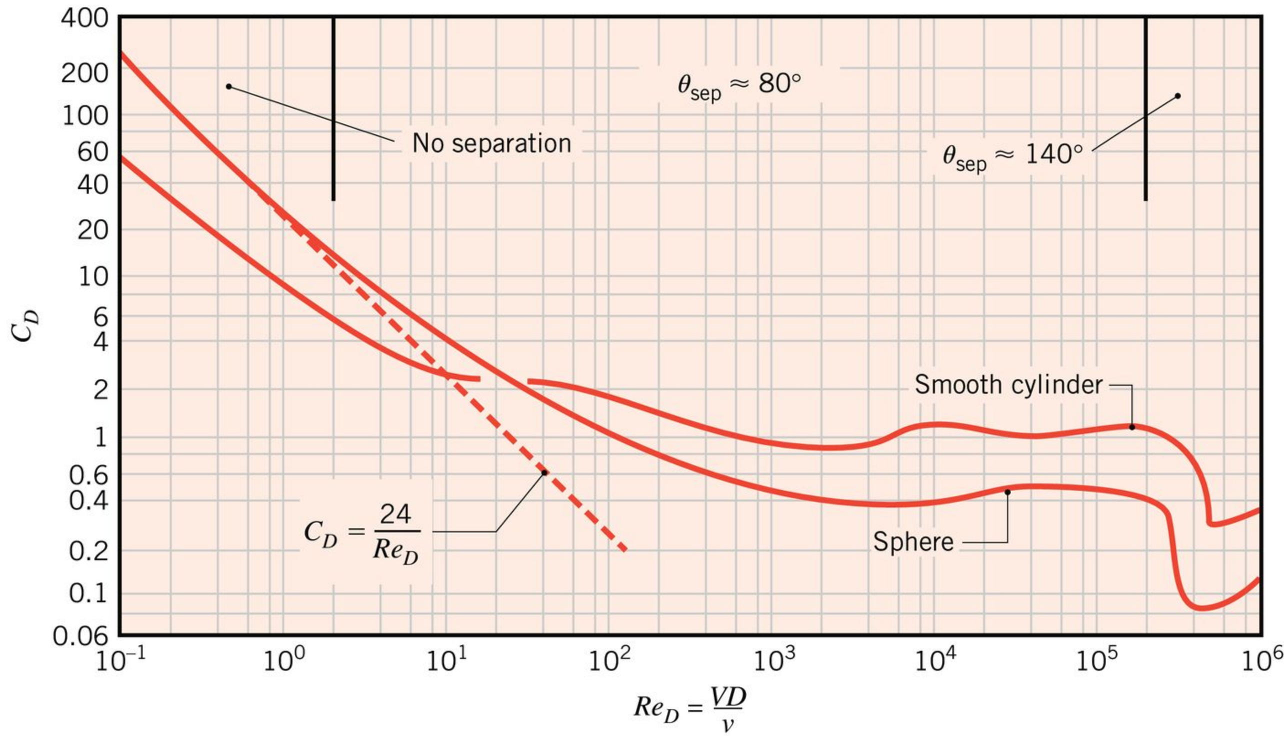
$C_3 = 1 + 2(D/L)^2$

$C_4 = [1 + (D/L)^2]^{1/2}$

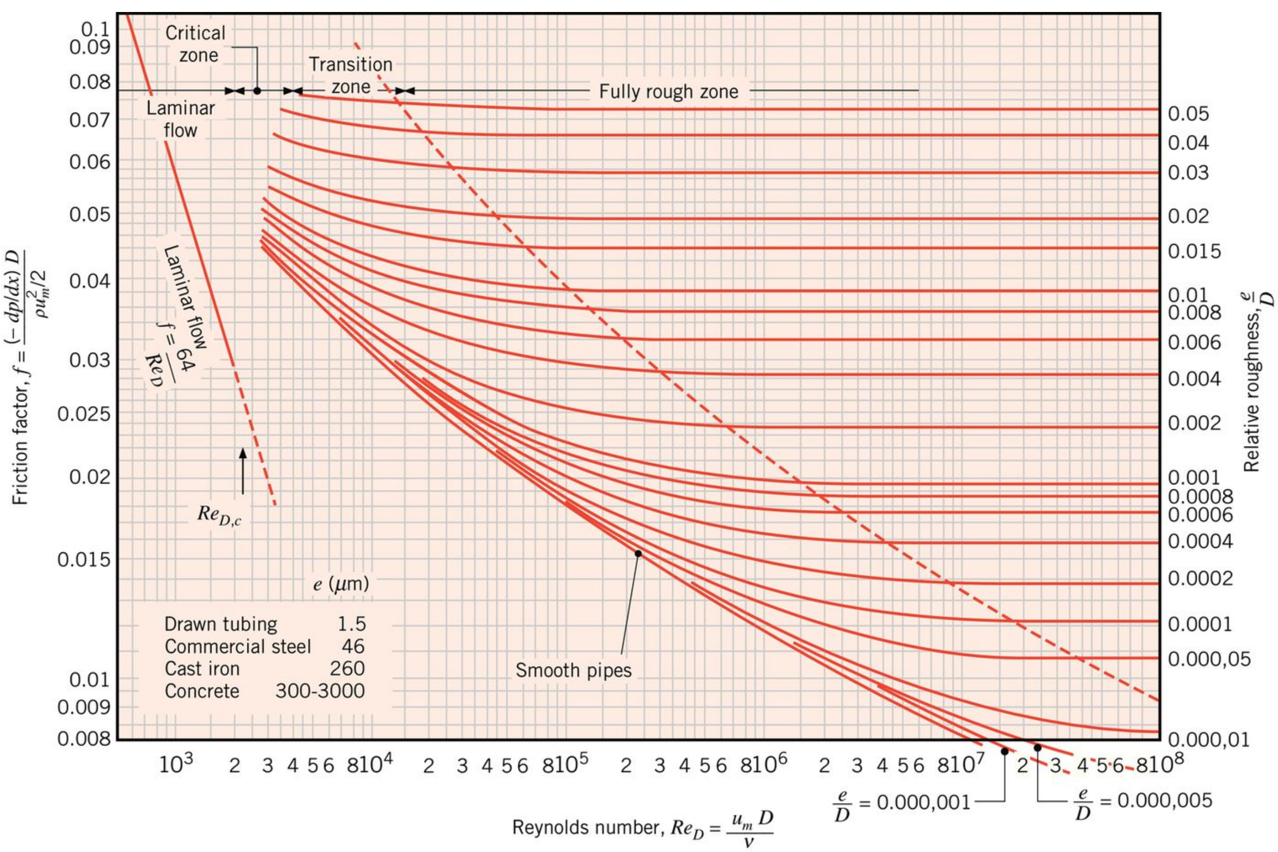
$V = (\pi/20)D^2L$



$$\eta_f = \frac{2}{[4/9(mL)^2 + 1]^{1/2} + 1}$$



C_D
400
200
100
60
40
20
10
6
4
2
1
0.6
0.4
0.2
0.1
0.06
No separation
$\theta_{sep} \approx 80°$
$\theta_{sep} \approx 140°$
Smooth cylinder
Sphere
$C_D = \frac{24}{Re_D}$
10^{-1} 10^0 10^1 10^2 10^3 10^4 10^5 10^6
$Re_D = \frac{VD}{\nu}$



Friction factor, $f = \frac{(-dp/dx)\,D}{\rho u_m^2/2}$
0.1
0.09
0.08
0.07
0.06
0.05
0.04
0.03
0.025
0.02
0.015
0.01
0.009
0.008
Critical zone
Transition zone
Fully rough zone
Laminar flow
Laminar flow $f = \frac{64}{Re_D}$
$Re_{D,c}$
e (μm)
Drawn tubing 1.5
Commercial steel 46
Cast iron 260
Concrete 300-3000
Smooth pipes
Relative roughness, $\frac{e}{D}$
0.05
0.04
0.03
0.02
0.015
0.01
0.008
0.006
0.004
0.002
0.001
0.0008
0.0006
0.0004
0.0002
0.0001
0.000,05
0.000,01
10^3 2 3 4 5 6 8 10^4 2 3 4 5 6 8 10^5 2 3 4 5 6 8 10^6 2 3 4 5 6 8 10^7 2 3 4 5 6 8 10^8
$\frac{e}{D}$ = 0.000,001
$\frac{e}{D}$ = 0.000,005
Reynolds number, $Re_D = \frac{u_m D}{\nu}$

Table 1: Contact Resistance for vacuum interfaces, $R''_{t,c} \times 10^4 \left(\frac{m^2K}{W}\right)$

Material	$100\,(kPa)$	$10,000\,(kPa)$
Stainless Steel	6-25	0.7-4.0
Copper	1-10	0.1-0.5
Magnesium	1.5-3.5	0.2-0.4
Aluminum	1.5-5.0	0.2-0.4

Table 2: Contact Resistance for aluminum ($10\mu m$ surface roughness, $10kPa$ contact pressure)

Interfacial Fluid	$R''_{t,c} \times 10^4 \left(\frac{m^2K}{W}\right)$
Air	2.75
Helium	1.05
Hydrogen	0.720
Silicone Oil	0.525
Glycerine	0.265

Table 3: Fin Tip Conditions

Case	Tip Condition	Temperature Distribution $\frac{\theta}{\theta_b}$	Fin Heat Transfer Rate q_f
A	Convection	$\frac{\cosh m(L-x)+\left(\frac{h}{mk}\right)\sinh m(L-x)}{\cosh mL+\left(\frac{h}{mK}\right)\sinh mL}$	$M\frac{\sinh mL+\left(\frac{h}{mk}\right)\cosh mL}{\cosh mL+\left(\frac{h}{mk}\right)\sinh mL}$
B	Adiabatic	$\frac{\cosh m(L-x)}{\cosh mL}$	$M\tanh mL$
C	Constant Temperature	$\frac{\left(\frac{\theta_L}{\theta_b}\right)\sinh mx+\sinh m(L-x)}{\sinh mL}$	$M\frac{\cosh mL-\frac{\theta_L}{\theta_b}}{\sinh mL}$
D	Infinite Fin ($L \to \infty$)	e^{-mx}	M

Table 4: Cylinder In Cross Flow

Re_D	C	m
$0.4 - 4$	0.989	0.330
$4 - 40$	0.911	0.385
$40 - 4000$	0.683	0.466
$4000 - 40,000$	0.193	0.618
$40,000 - 400,000$	0.027	0.805

Table 5: Various Geometries In Cross Flow

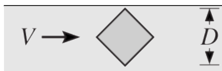
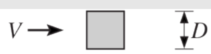
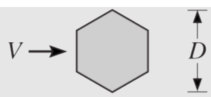
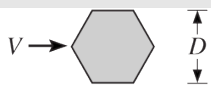
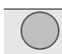
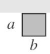
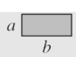
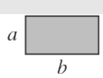
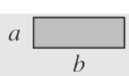
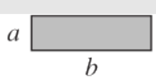
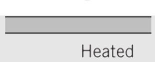
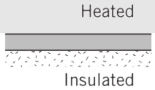
Geometry		Re_D	C	m
		$6000 - 60,000$	0.304	0.59
		$5000 - 60,000$	0.158	0.66
		$5200 - 20,400$ $20,400 - 105,000$	0.164 0.039	0.638 0.78
		$4500 - 90,700$	0.150	0.638
	Front Back	$10,000 - 50,000$ $7,000 - 80,000$	0.667 0.191	0.500 0.667

Table 6: Noncircular Tubes

Cross Section	$\frac{b}{a}$	Uniform q_s''	Uniform T_s	fRe_{D_h}
		4.36	3.66	64
	1.0	3.61	2.98	57
	1.43	3.73	3.08	59
	2.0	4.12	3.39	62
	3.0	4.79	3.96	69
	4.0	5.33	4.44	73
	∞	8.23	7.54	96
	∞	5.39	4.86	96
		3.11	2.49	53