

## Problem n.1

(15 points)

Given the discrete-time system  $S(A,B,C,D)$  described below:

$$\begin{cases} x_1(k+1) = -x_1(k) + x_2(k) \\ x_2(k+1) = x_1(k) + u(k) \end{cases} \quad y(k) = \begin{bmatrix} x_1(k) \\ x_2(k) \end{bmatrix}$$

Where:

$$r(k) = \begin{bmatrix} 1(k) \\ 1(k) \end{bmatrix} \quad 1(k) = 1 \quad \forall k \geq 0$$

Cost function:

$$V(k) = \sum_{i=H_w}^{H_p} [r(k+i) - x(k+i)]^T \begin{bmatrix} 4 & 0 \\ 0 & -3 \end{bmatrix} [r(k+i) - y(k+i)]$$

$$H_p = 2 \quad H_w = 1 \quad H_c = 2$$

## Questions:

- Compute the cost function  $V(k)$
- Compute  $u(k)$
- Compute  $u(k+1)$
- Considering the computed  $u(k)$ , what can be said about the stability of the closed-loop system?
- By considering the cost function defined below, compute  $V_1(k), u(k), u(k+1)$

$$V_1(k) = V(k) + \sum_{i=1}^{H_c-1} 2[u(k+i)]^2$$

Table n.1 (see Problem n.2)		
Name	Description	Value [unit]
$x_1$	Car position on the x axis (x)	[m]
$x_2$	Car position on the y axis (y)	[m]
$x_3$	Orientation of the car with respect to the x axis. ( $\theta$ )	[rad]
$x_4$	The angle of the front wheel with respect to the longitudinal symmetry axis of the car. ( $\varphi$ )	[rad]
$u_1$	Longitudinal car speed (v)	[m/s]
$u_2$	The angle of the front wheel with respect to the longitudinal symmetry axis of the car. ( $\varphi$ )	[m/s]
$y_1$	Car position on the x axis (x)	[m]
$y_2$	Car position on the y axis (y)	[m]

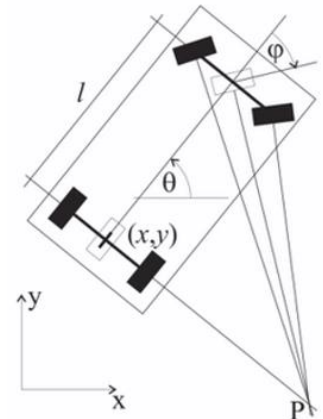
## Problem n.2

(15 points)

The follow image depicts the kinematic car in the horizontal plane. Let us suppose that the Ackermann steering assumptions hold true, hence all wheels turn around the same point (denoted by P) which lies on the line of the rear axle. It follows that the kinematics of the car can be fully described by the kinematics of a bicycle fitted in the middle of the car.

All lengths involved in the kinematic calculations, and in particular  $l$ , equal to one.

$$\begin{cases} x_1(k+1) = x_1(k) + u_1(k) * \cos(x_3(k)) \\ x_2(k+1) = x_2(k) + u_1(k) * \sin(x_3(k)) \\ x_3(k+1) = x_3(k) + u_1(k) * \tan(x_4(k)) \\ x_4(k+1) = u_2(k) \end{cases} \quad y(k) = \begin{bmatrix} x_1(k) \\ x_2(k) \end{bmatrix}$$



The vehicle parameters are written in Table n.1 in page 1.

### Questions:

- Consider  $x_0=[0 \ 0 \ 0 \ 0]'$ ,  $y_{ref}=[1 \ 1]'$ . The settling time required is 1s. Calculate  $T_s, H_p, H_c$ .
- The vehicle considered is a golf cart. It can reach maximum 3.6km / h. In reverse it can reach a maximum of 1.8km / h. The steering can orient the wheels in a range of  $+ -60^\circ$ . Forward from 0 to maximum speed takes 5 seconds. In reverse the vehicle has the same acceleration. Deceleration has the same magnitude.
  - Initialize the MPC controller with the found parameters.
  - Constrain the system accordingly.
  - Run a simulation lasting 5s.
  - Observe and comment the simulation results.
- Specify an analytical Jacobian for the output function and the state function.
- Run a simulation considering  $x_0=[0 \ 0 \ 0 \ 0]'$ ,  $y_{ref}=[5 \ 1]'$  and Set the RateMin of  $u_1 = -0.025$ .  
 As you can see, the car does not reach its goal smoothly.  
 What parameters can be tuned to improve the simulation?
- Tune the identified parameters and perform a simulation in Matlab.

**SUGGESTION:** Analyze the data using 4 figures:

- fig. 1 with the input subplots
- fig. 2 with state subplots
- fig. 3 with the subplots of the outputs
- fig. 4 representation on the xy plane of the path traveled by the car.

Add titles to charts and anything else that makes them easier to understand.

**IMPORTANT:** remember that your system is not continuous but it is discrete!

**IMPORTANT:** To get a full score it is necessary to comment on the design choices.