



# **Vehicle Lateral Dynamics**

## **Path follower driver model**

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Simulate vehicle lateral dynamics through single track vehicle model plus path follower.

- Path follower model equations
  - Inputs, outputs, state vector
  - Axle forces description
  - Slip angles
- Assignments

Vehicle in plane motion. One rigid body

3 dofs:

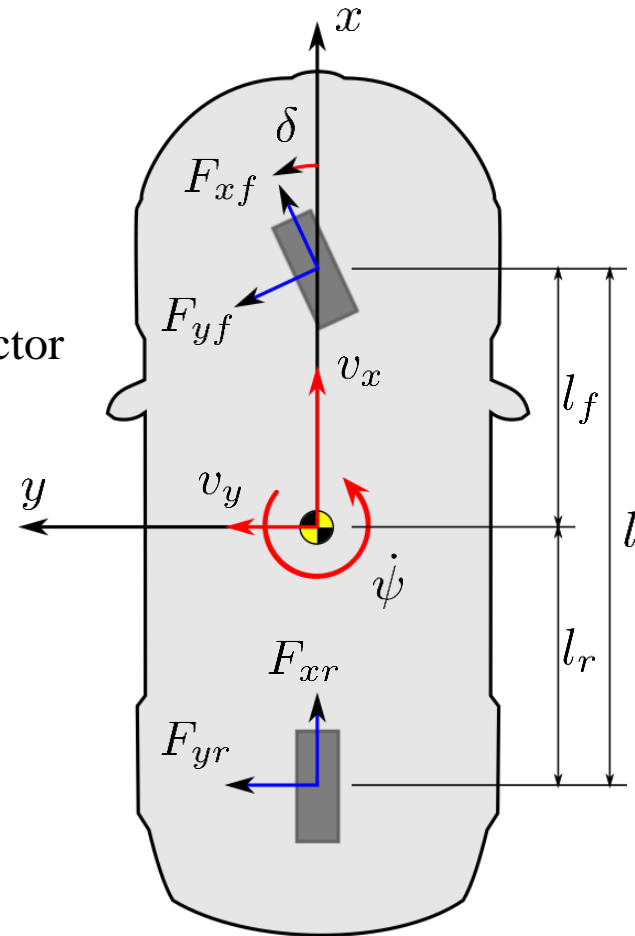
- Longitudinal speed
- Lateral speed
- Yaw rate

Imposing the longitudinal speed, the state vector becomes

$$\mathbf{x} = \begin{Bmatrix} v_y \\ \dot{\psi} \end{Bmatrix}$$

Remember the kinematic relationship

$$a_y = \dot{v}_y + v_x \dot{\psi}$$



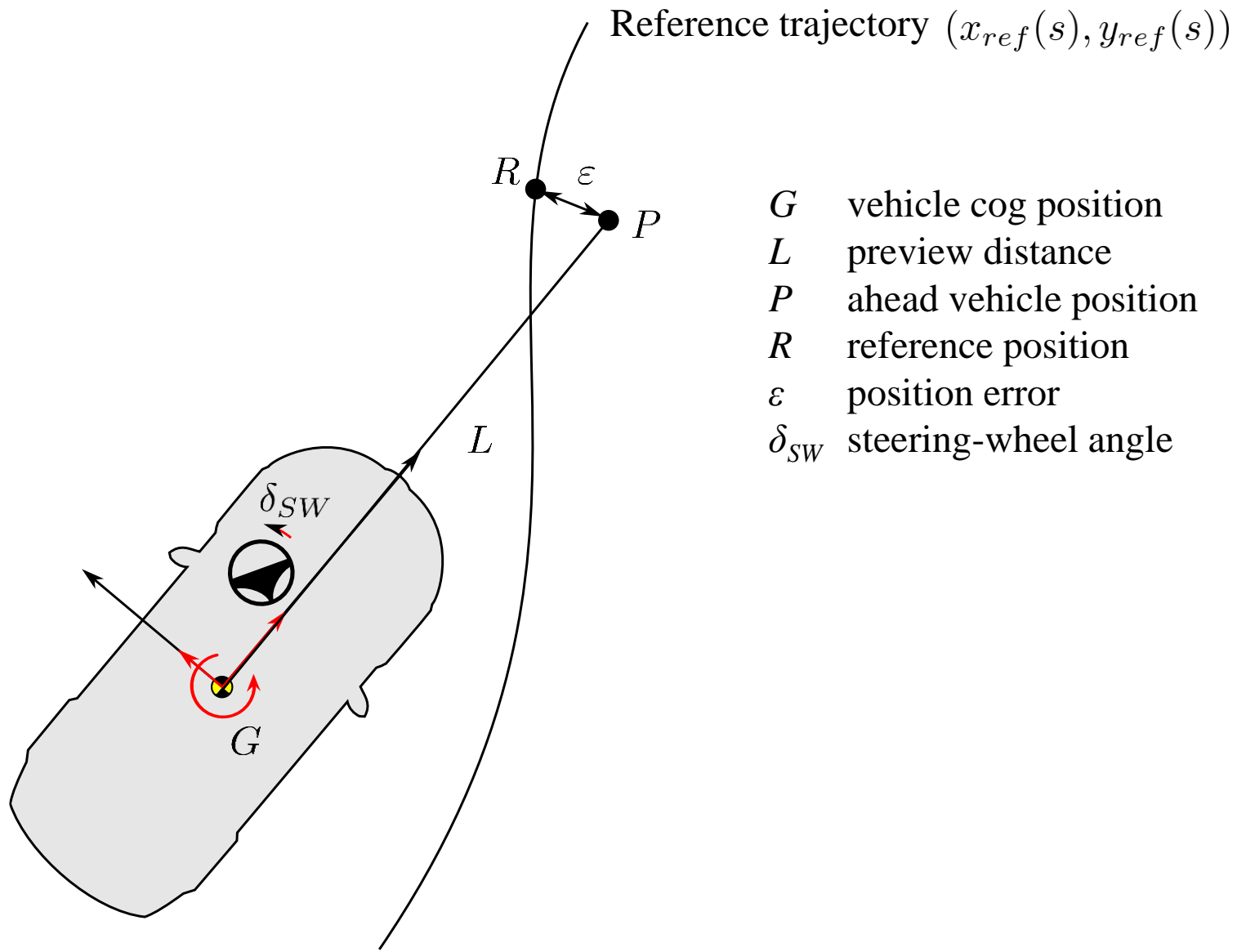
How to model a driver that follows a reference trajectory?

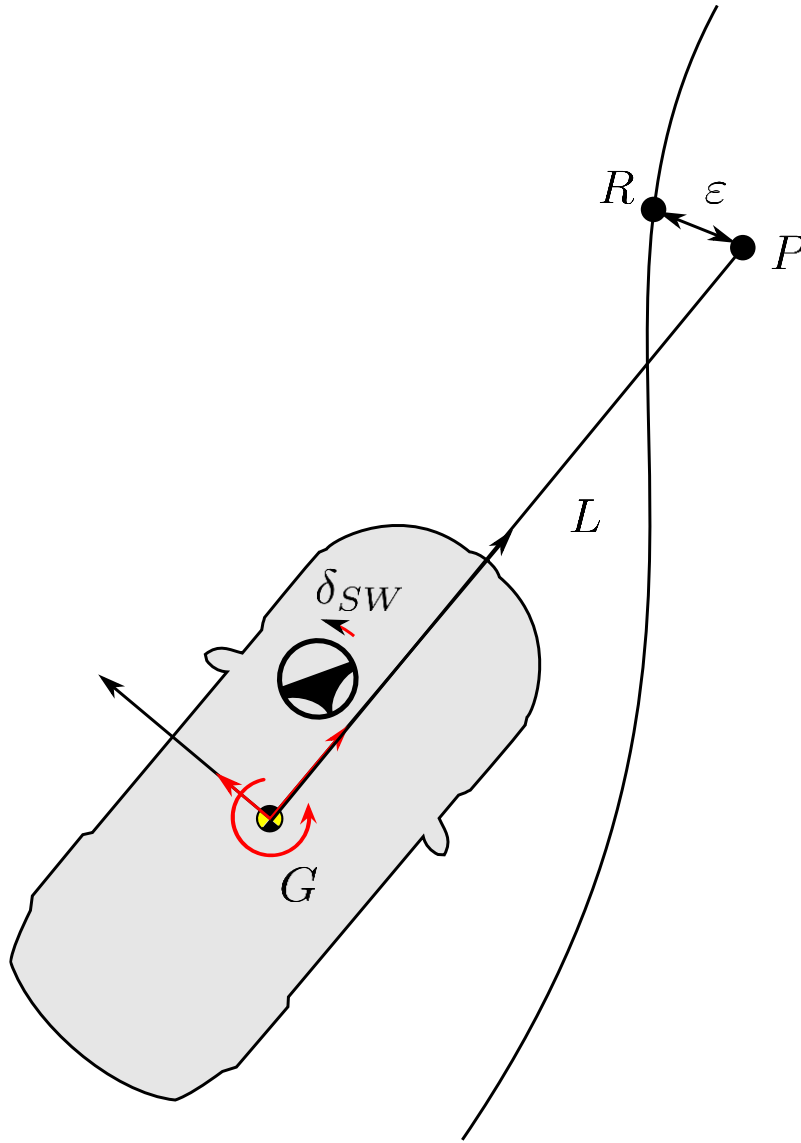
Path follower driver

- reference trajectory
- actual position of the vehicle
- Look ahead capability
- control law
- Time lag modeling



Lateral displacement error





Reference trajectory is the collection of points describing the track shape as a function of curvilinear abscissa  $s$

$$(x_{ref}(s), y_{ref}(s))$$

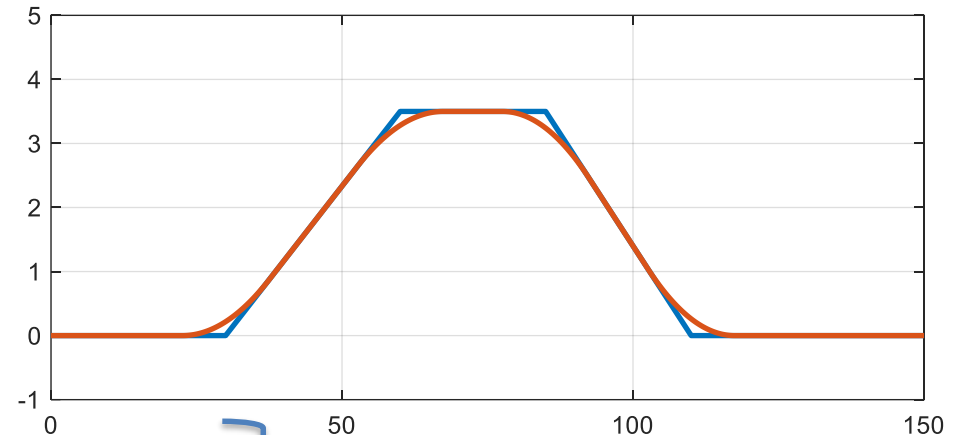
Curvilinear abscissa  $s$  represents the travelled distance thus

$$s = \int_0^t v dt$$

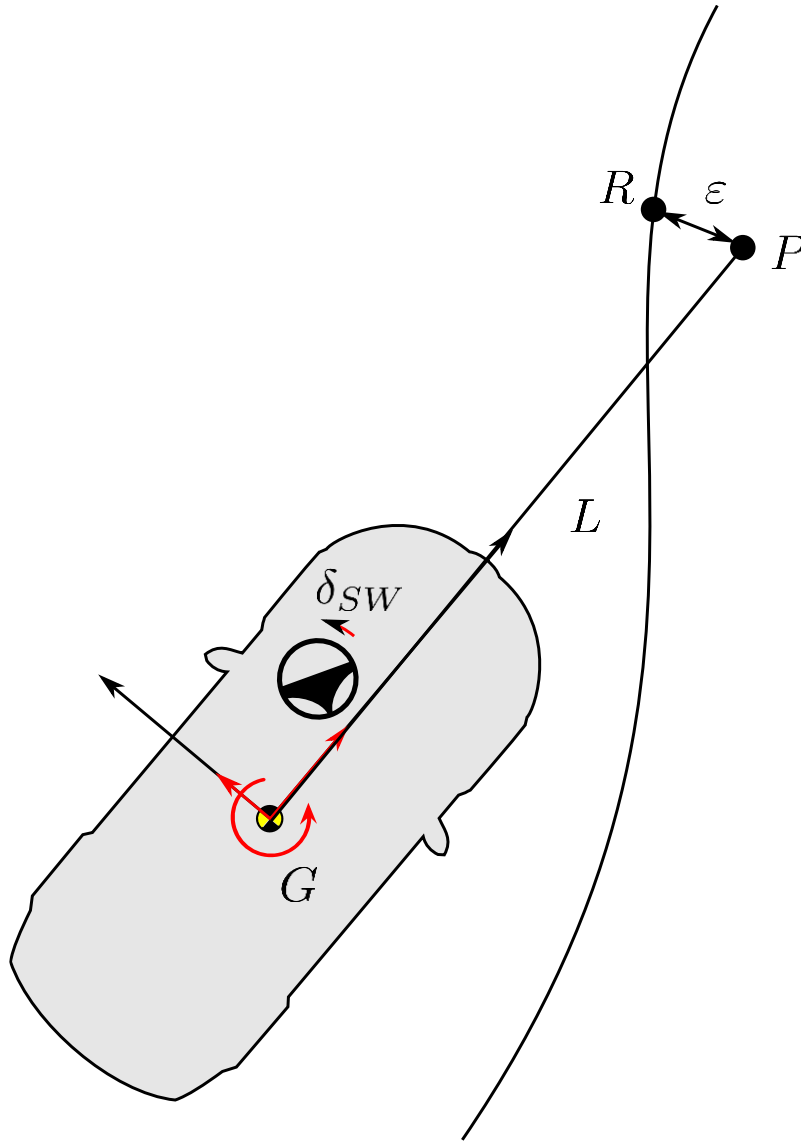
## Reference trajectory of a double lane change

```
x = [0 20 30 60 85 110 140 1000]';  
y = [0 0 0 3.5 3.5 0 0 0]';  
xx = [0:.1:x(end)]';  
yy = interp1(x,y,xx);  
yy = smooth(yy,150);  
x = xx;  
y = yy;
```

```
dx = diff(x);  
dy = diff(y);  
dx = [0; dx];  
dy = [0; dy];  
for ii=1:length(x)  
    s(ii,1) = sum(sqrt(dx(1:ii).^2+dy(1:ii).^2));  
end  
ss = [0:.1:s(end)]';  
xx = interp1(s,x,ss);  
yy = interp1(s,y,ss);  
traiettoria.s = ss;  
traiettoria.x = xx;  
traiettoria.y = yy;
```



Compute curvilinear abscissa  
of the reference path



Vehicle position is obtained by integrating vehicle speed in absolute reference frame

$$X_G(t) = \int_0^t \dot{X}_G dt + X_{G0}$$

$$Y_G(t) = \int_0^t \dot{Y}_G dt + Y_{G0}$$

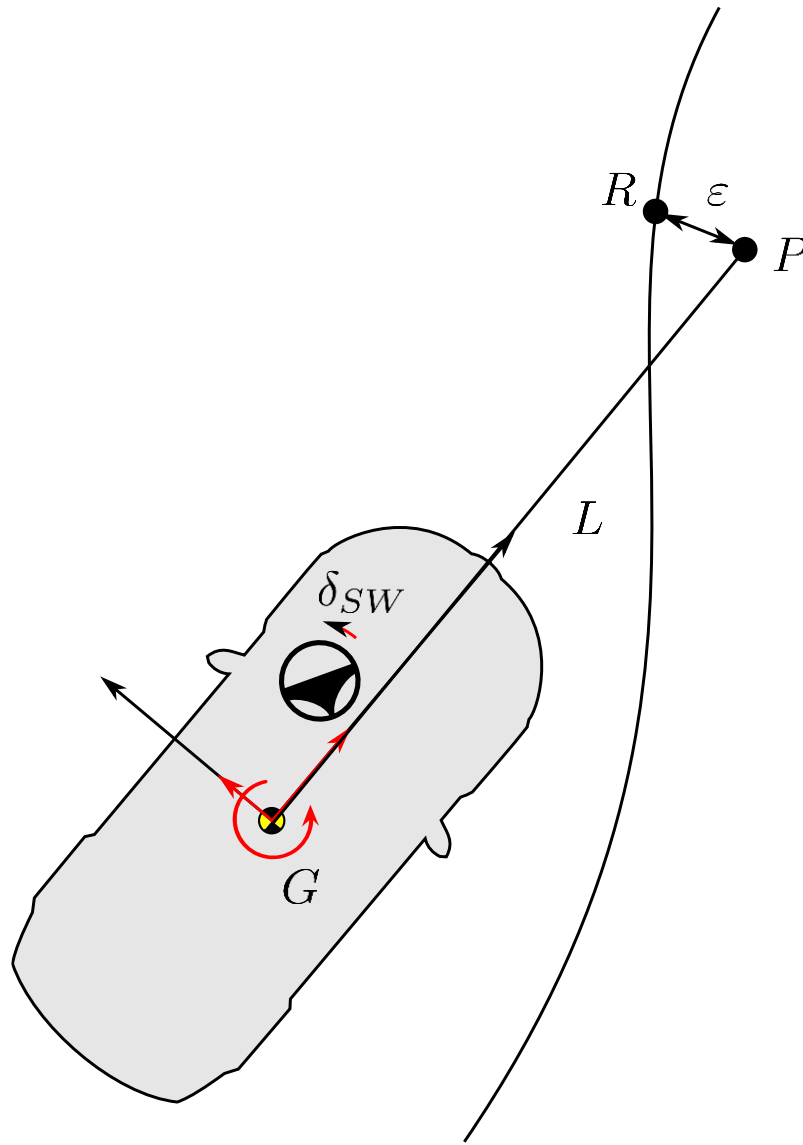
$$\psi(t) = \int_0^t \dot{\psi} dt + \psi_0$$

$$\dot{X}_G = v_x \cos \psi - v_y \sin \psi$$

$$\dot{Y}_G = v_x \sin \psi + v_y \cos \psi$$

$$s = \int_0^t v dt$$





The driver is usually capable of anticipating the necessary maneuver. The position error can thus be computed according to a position in front of the vehicle ( $P$ ) of a given distance  $L$

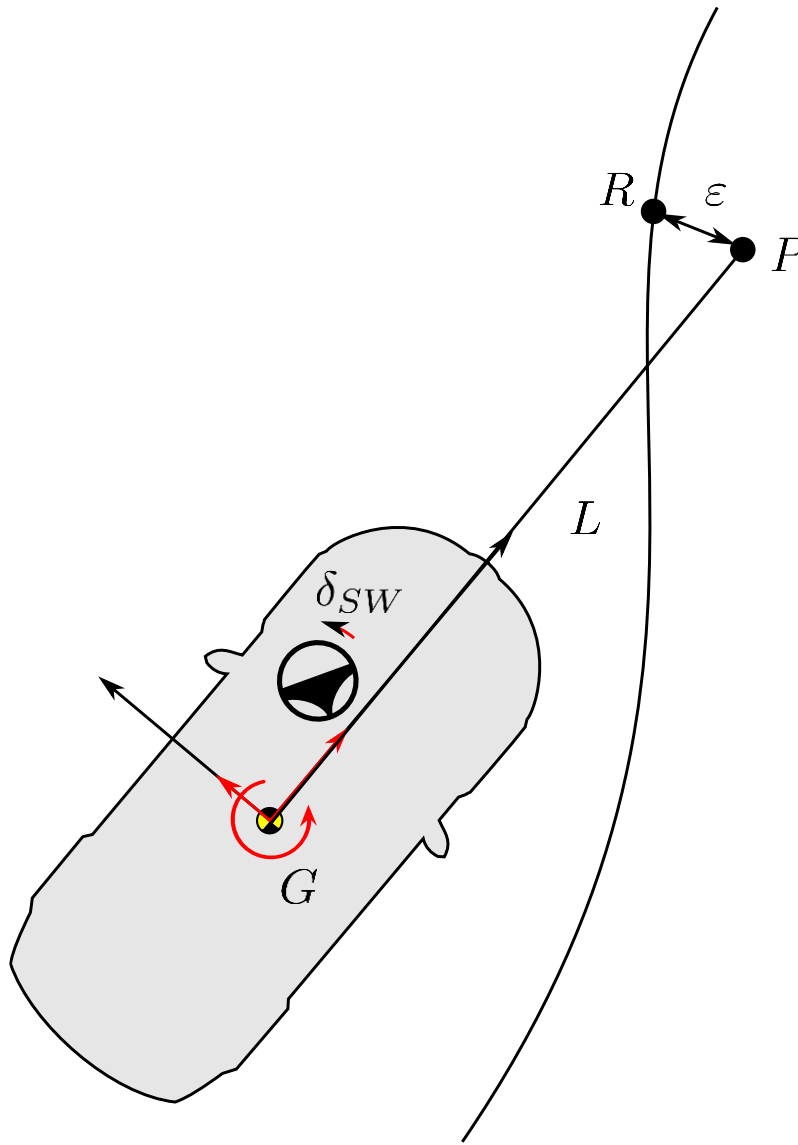
$$X_P = X_G + L \cos \psi$$

$$Y_P = Y_G + L \sin \psi$$

The preview distance  $L$  is not constant but changes with vehicle speed

$$L = v_x t_{driver} + L_0$$

$t_{driver}$  is the driver characteristic time.  
**When going faster, the preview distance increases**



We need now to compute the R-P vector

$$(R - P) = (X_R - X_P)\mathbf{i} + (Y_R - Y_P)\mathbf{j}$$

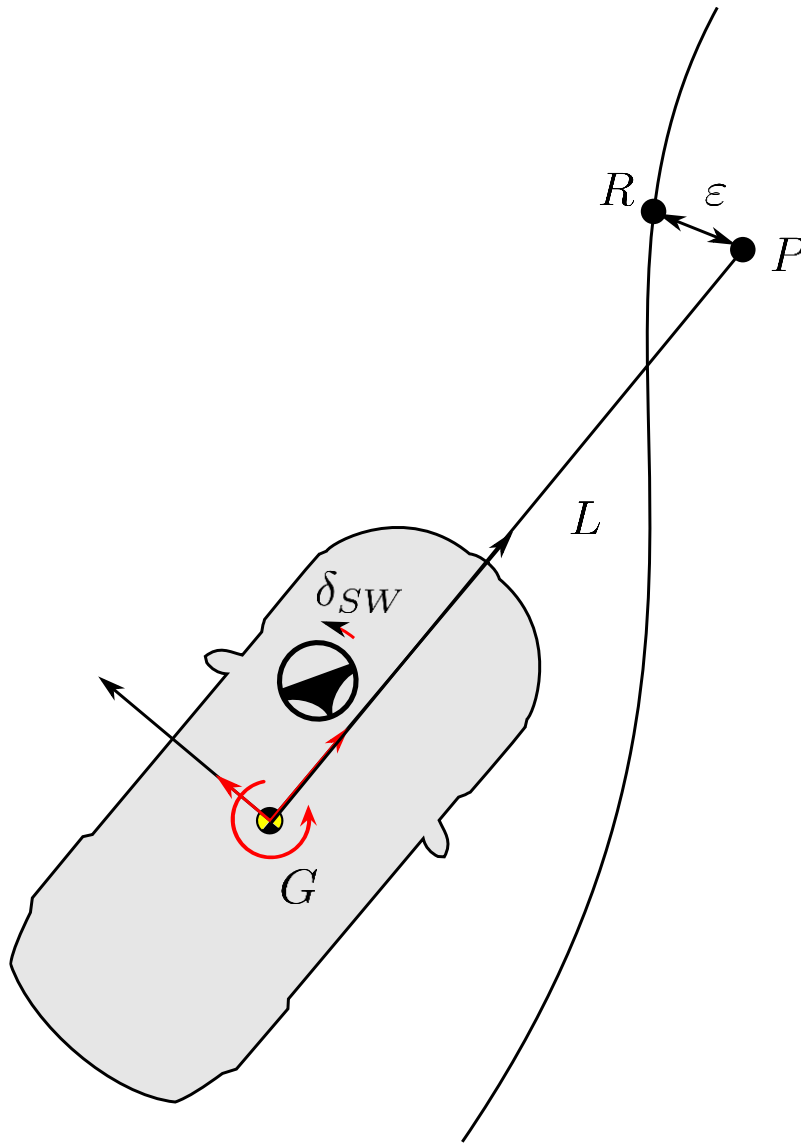
Transformed into vehicle reference frame

$$x_R = (X_R - X_P) \cos \psi + (Y_R - Y_P) \sin \psi$$

$$y_R = -(X_R - X_P) \sin \psi + (Y_R - Y_P) \cos \psi$$

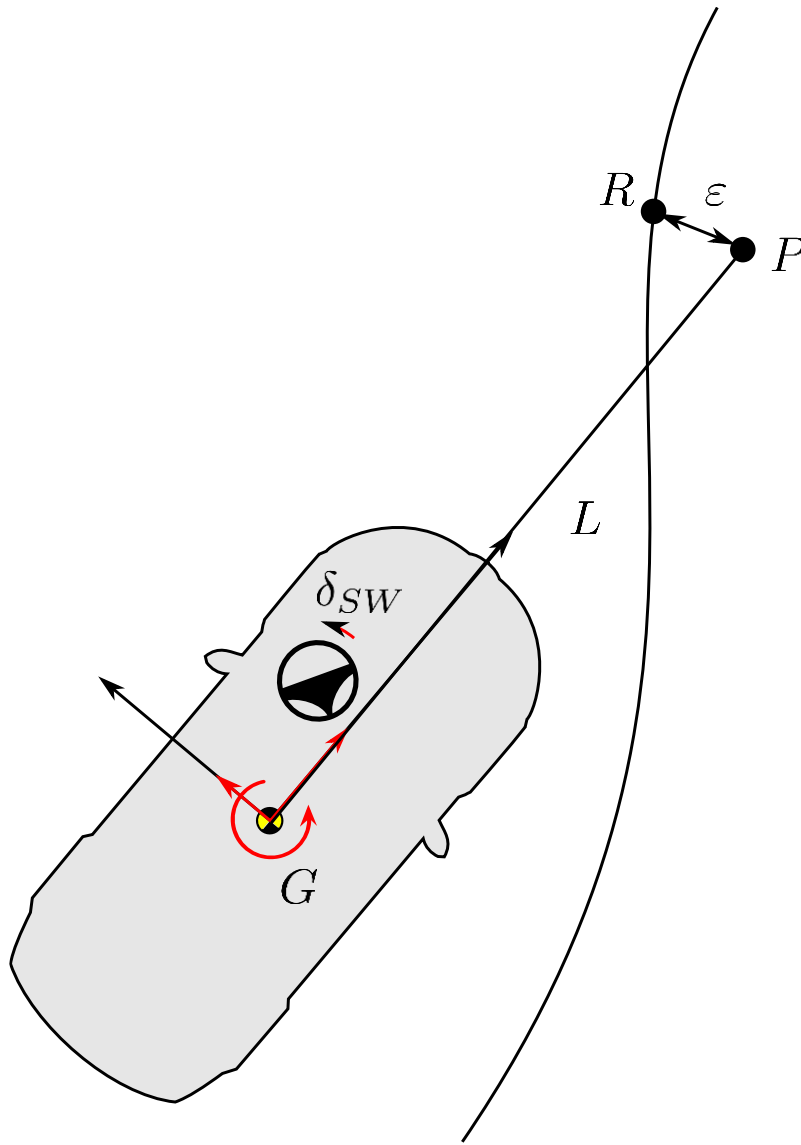
The position error is then

$$\varepsilon = y_R$$



Steering wheel angle is thus proportional to the positioning error. A PD control law can be used

$$\hat{\delta}_{SW} = k_P \epsilon + k_D \dot{\epsilon}$$

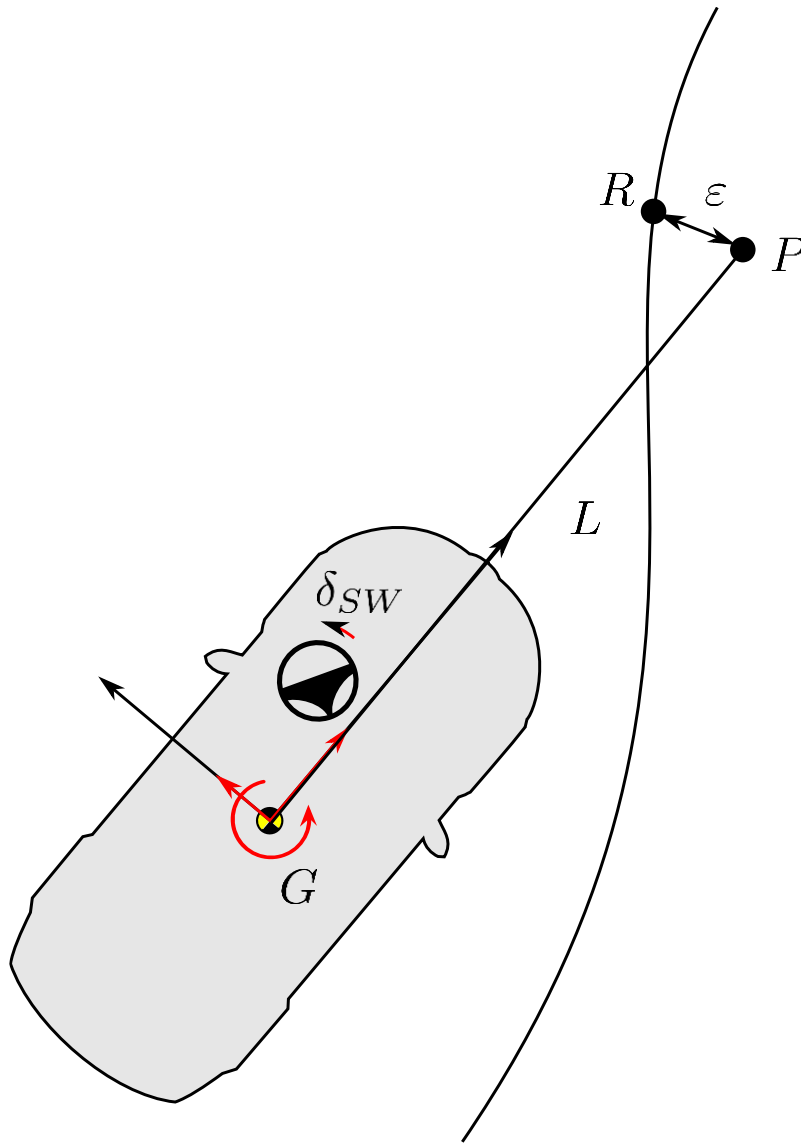


To reproduce the neuro-muscular time lag of the driver, a simple first order time lag transfer function is used

$$TF = \frac{1}{\tau s + 1}$$

thus

$$\dot{\delta}_{SW} = \frac{1}{\tau_D} (\hat{\delta}_{SW} - \delta_{SW})$$



Consider the steering system ratio

$$\delta = \frac{1}{\tau_{SW}} \delta_{SW}$$

Simulate following maneuvers:

- Double lane change maneuver

For each maneuver report the following graph

- $V_y$  vs time
- Yaw rate vs time
- Sideslip angle vs time
- Lateral acceleration vs time
- Steering angle

Change vehicle speed and driver gains, analyse the different results obtained

```
% Steering wheel ratio
T_SW = 18;

% path follower data
t_driver = 1.5; % [s]
L0      = 0.5;  % [m]

kP      = 10;
kD      = 1;

TauD = 0.1;
```