# 5.1.

# Teoría de vibraciones II

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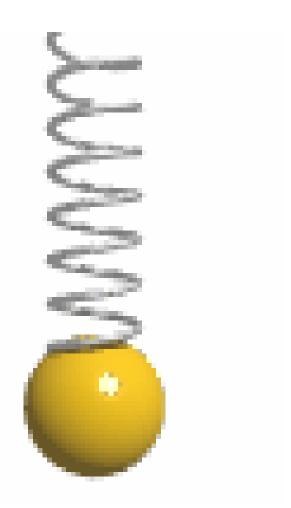
Todos los cuerpos que poseen masa y elasticidad son capaces de vibrar

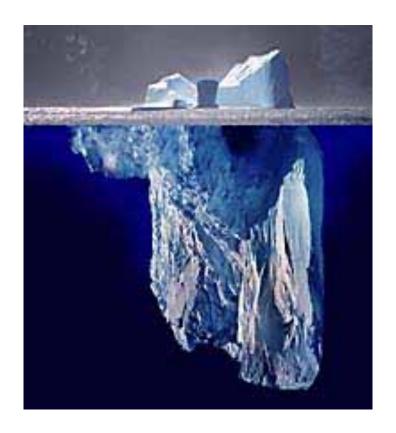
#### INERCIA + ELASTICIDAD

Las 3 condiciones básicas para que un cuerpo vibre

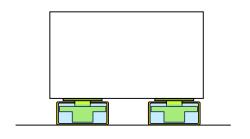
+ fuerzas externas

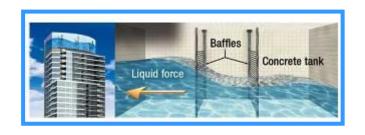


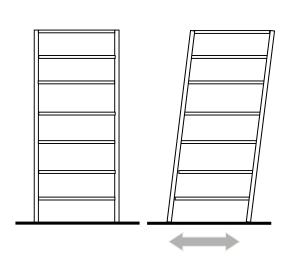


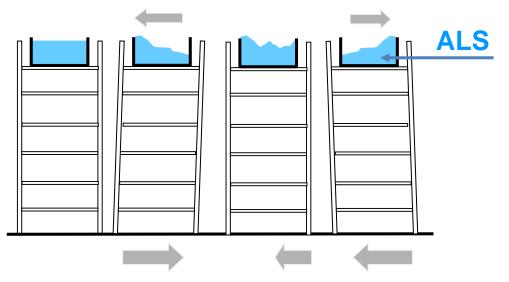






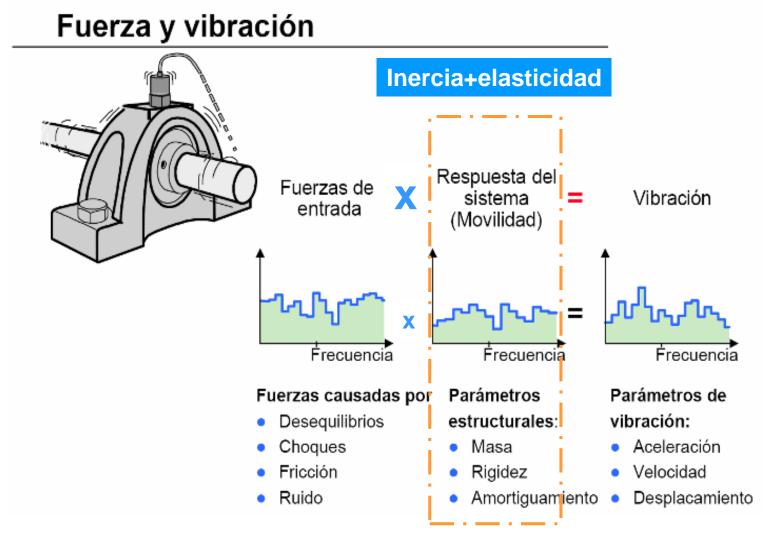






Edificios antisísmicos







## ....en la forma que son producidas: visión energética

#### **Libres**



Cris: 3 años

#### **Forzadas**



#### **Autoexcitadas**



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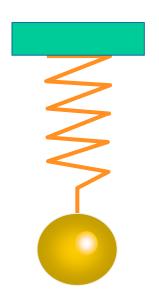
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### ...en f. de su comportamiento durante la vibración.

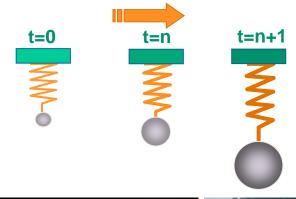
#### Lineales

#### Sistema clásico de masa-muelle





#### **No Lineales**

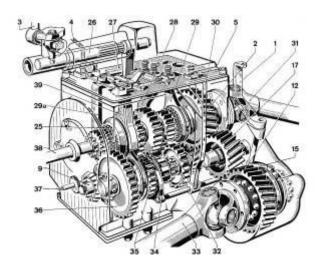






Ref: Ángel Franco García Física. U.Pais Vasco





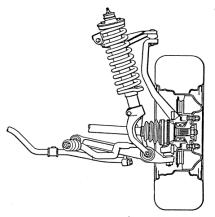


Figura 3.a – Suspensión. Sistema real

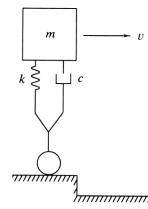


Figura 3.b – Modelo matemático discretizado

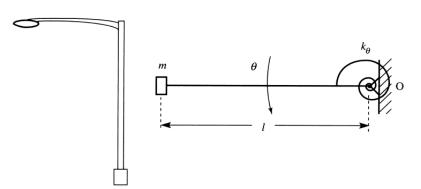


Figura 6.a – Farola modelizada como un sistema de 1 gdl

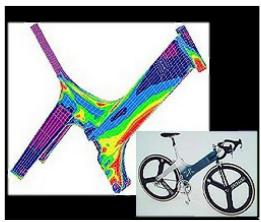
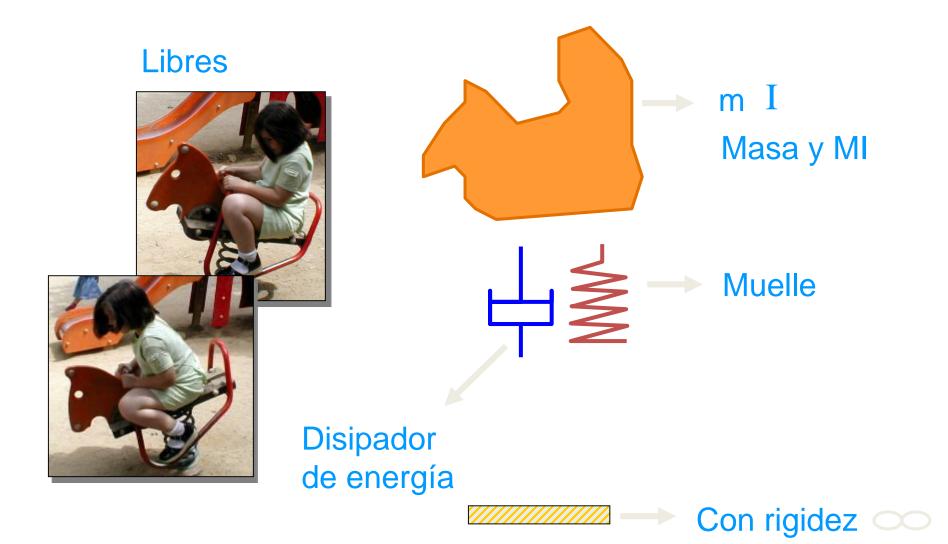
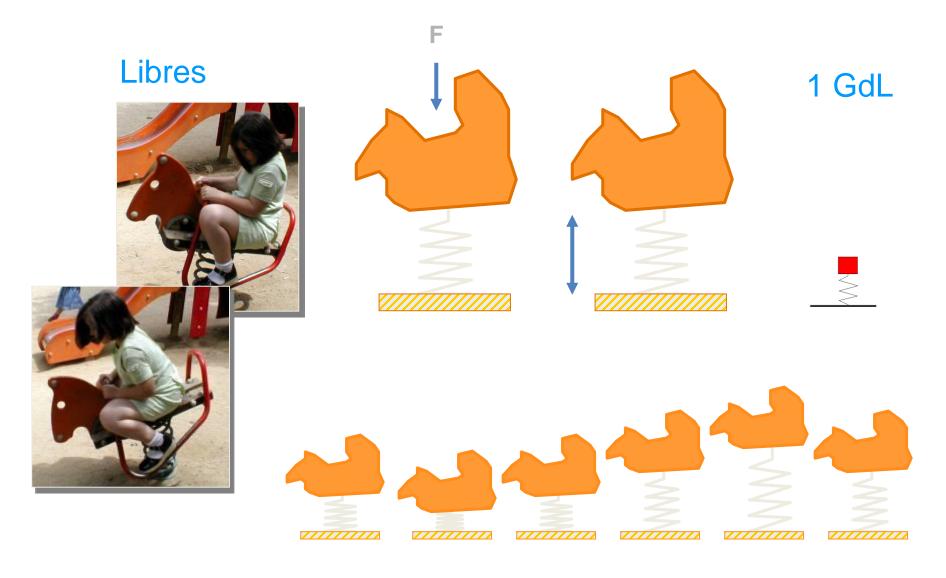


Figura 5 – Modelización por Elementos Finitos



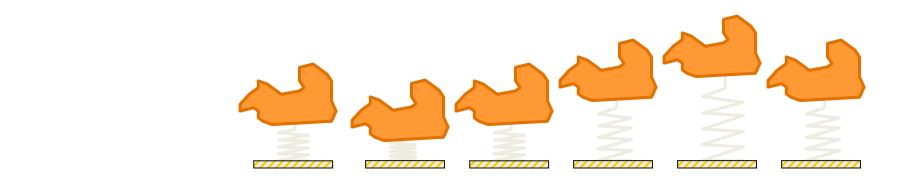






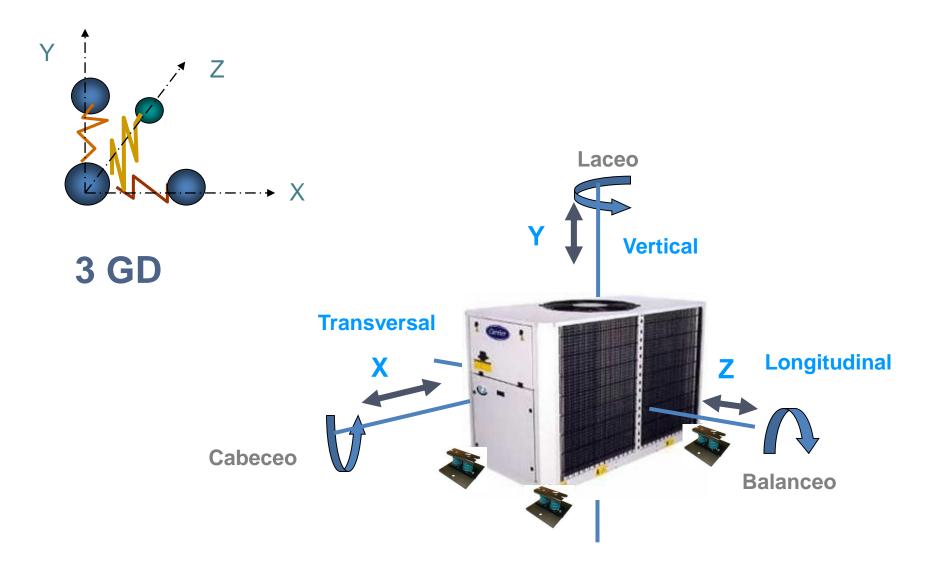


#### Parámetro independiente 1: Amplitud Y









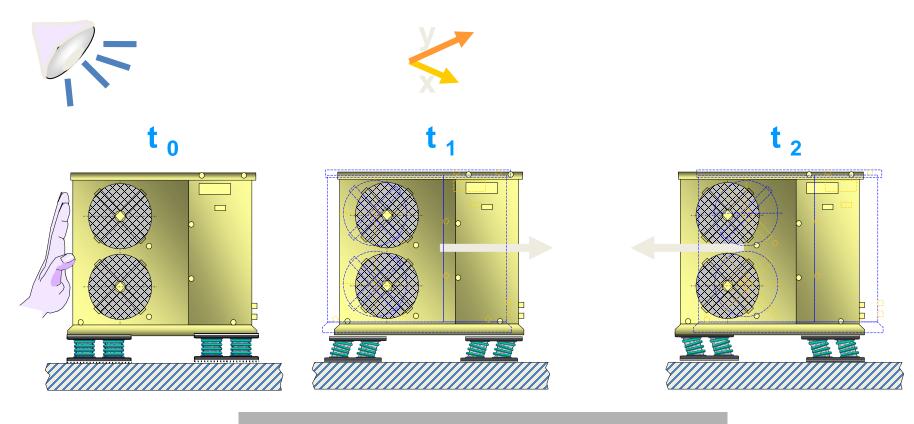
Fuente: Apuntes Rafa VIB. 5/95

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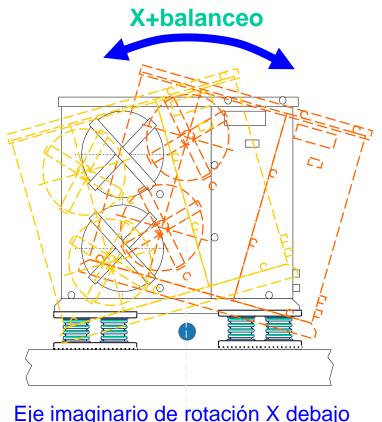
6 GD

#### Observación estática

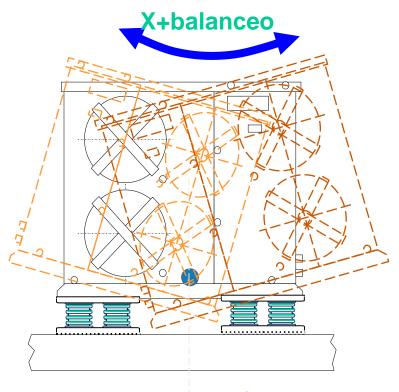


El sistema está DESACOPLADO





Eje imaginario de rotación X debajo

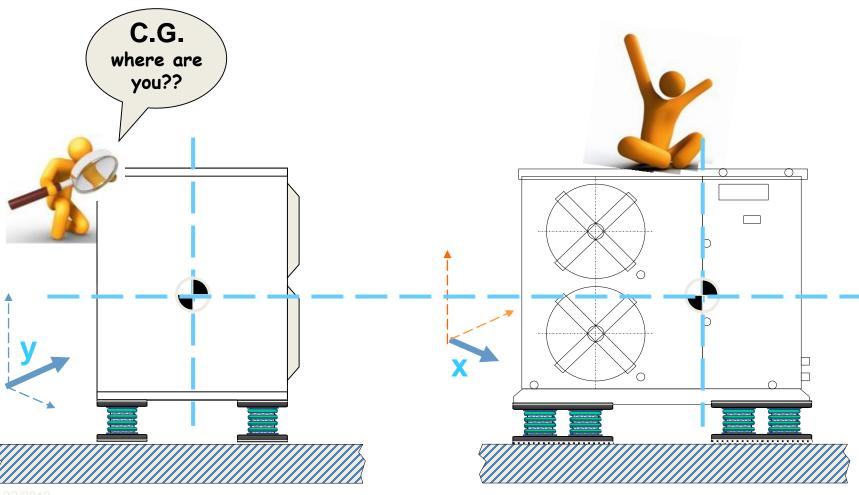


Eje imaginario de rotación X encima

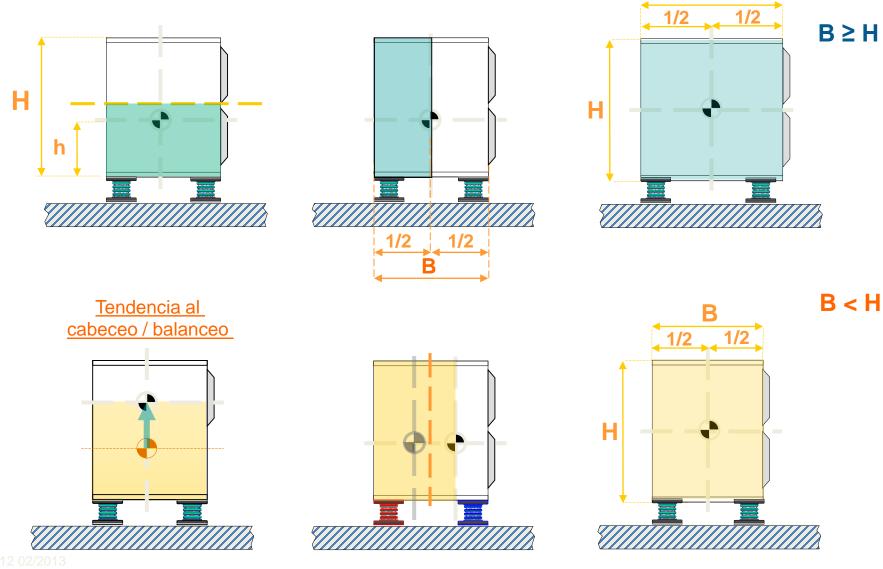
#### **ACOPLAMIENTO**



# Asimetría mecánica









# Critero esbeltez :H/B

H/B>5

En maquinaria y edificios; carga de viento, lluvia y nieve

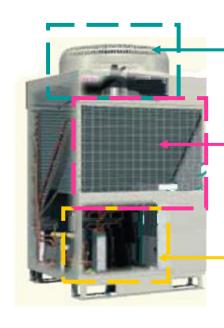


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#### Unidades VRV de volumen variable de frio/calor





ventilador

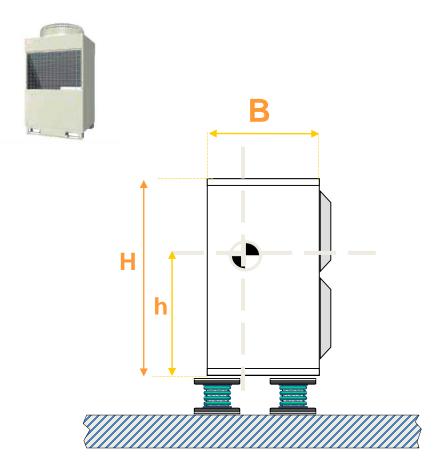
baterías

compresor

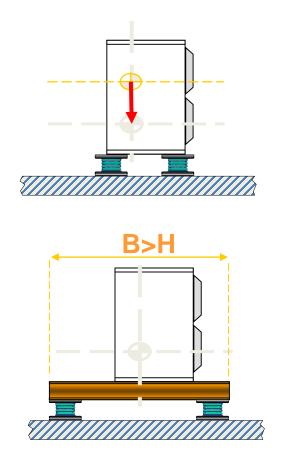








medidas PREVENTIVAS
"Control de Vibraciones"

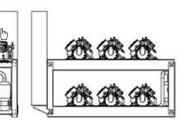


Estrategia adoptada: Rediseño del montaje antivibratorio.





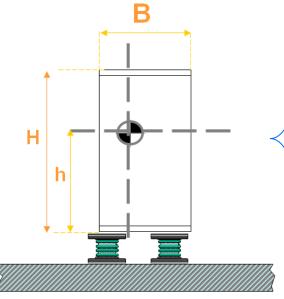




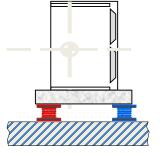
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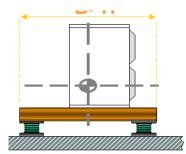
H/B>5



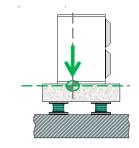
Estrategias adoptadas



Colocar aisladores de rigideces diferentes.

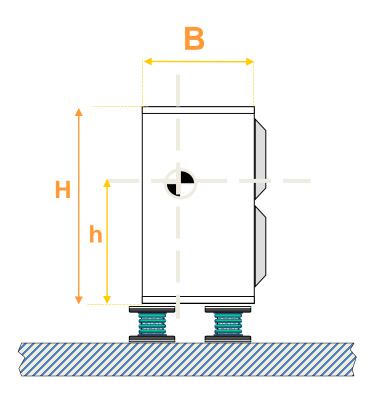


Realizar una bancada excéntrica con perfiles metálicos.

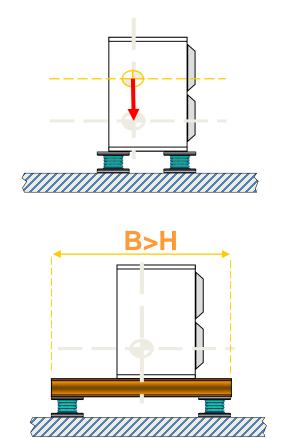


Realizar una bancada de hormigón suspendida con aisladores





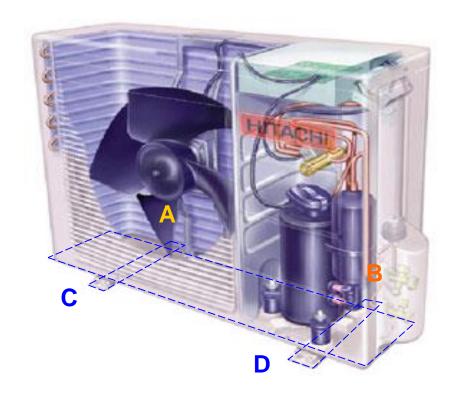
medidas PREVENTIVAS "Control de Vibraciones"



Estrategia adoptada: Rediseño del montaje antivibratorio.



### Unidades exteriores splits



Apoyo	Reacciones carga en %	
Α	25%	
В	35%	
С	18%	
D	22%	

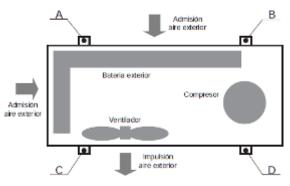


#### Enfriadoras agua pequeñas







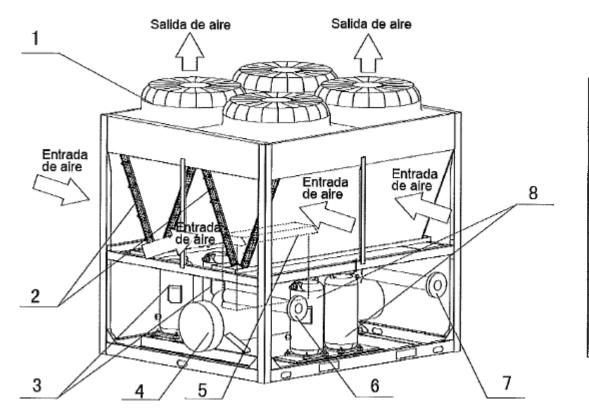


Apoyo	Reacciones carga en %
Α	26%
В	25%
С	25%
D	24%

Ref: Elecnor 10/02/2012



#### Enfriadoras agua grandes



Nº.	Nombre	
1	Cubierta superior	
2	Condensador	
3	Compresor	
4	Evaporador	
5	Caja de control eléctrica de la entrada de aire	
6	Salida de agua	
7	Entrada de agua	
8	Compresor	

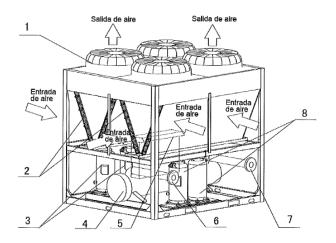
1.150Kg

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Ref: enviado por Gasifred 3/07/2012



#### Enfriadoras agua grandes



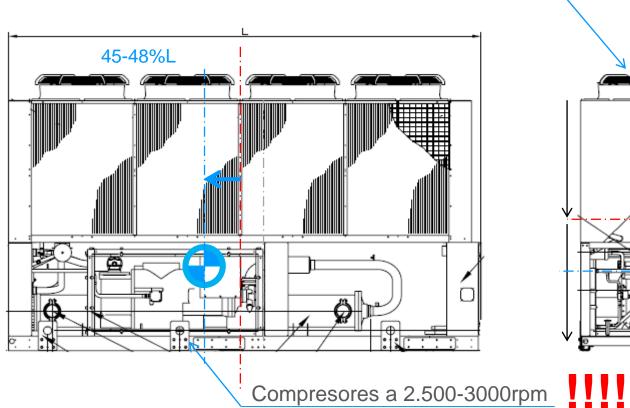
MCAC-ATSM-2011-08 Unidad enfriadora modular refrigerada por aire 50 Hz Módulo 130 O Vista frontal Vista lateral Instalando pernos de sujeción Agujero de instalación Φ 15mmx Vista desde arriba Orificio de anclaje

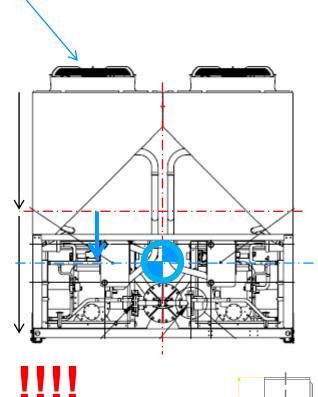
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Ref: enviado por Gasifred 3/07/2012



#### Ventiladores a 900rpm



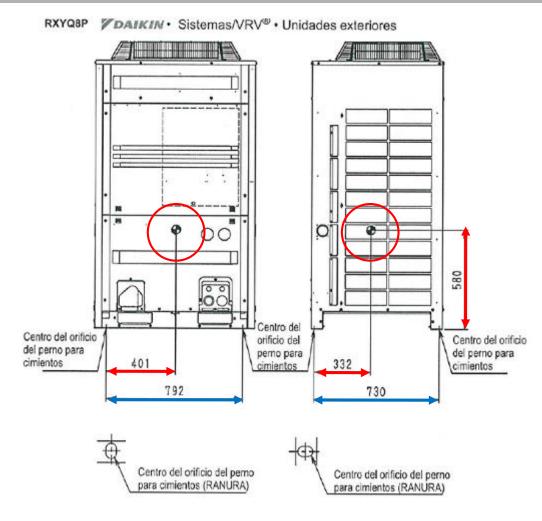


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Fuente: Consulta de Dasil MAAM 2010:30/08/2012



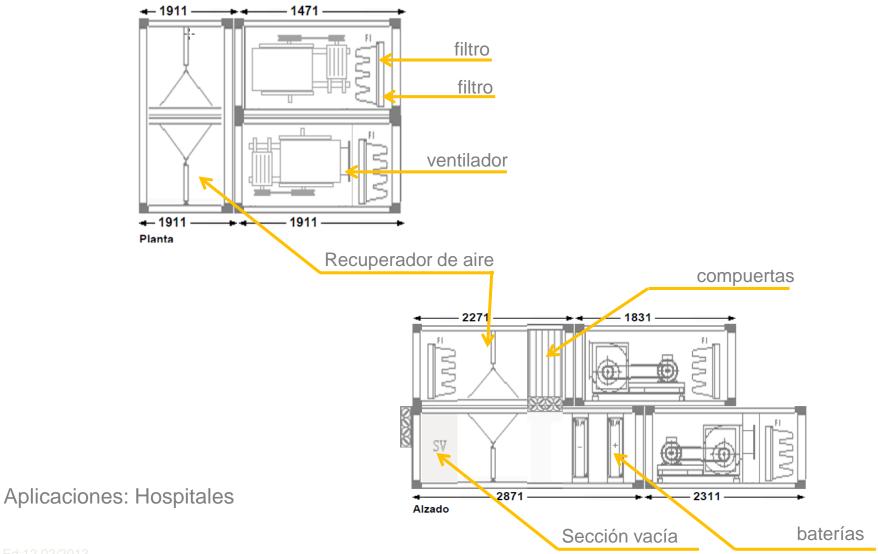
#### No siempre es frecuente que el fabricante facilite la distribución de cargas



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Rafa 10/08/2011





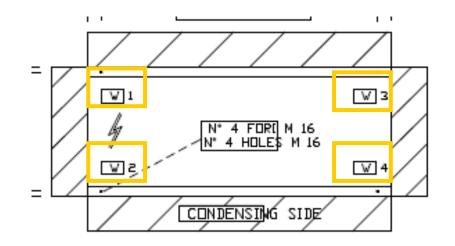


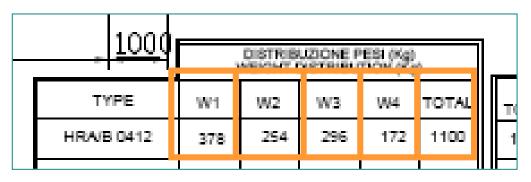
#### No siempre es frecuente que el fabricante facilite la distribución de cargas



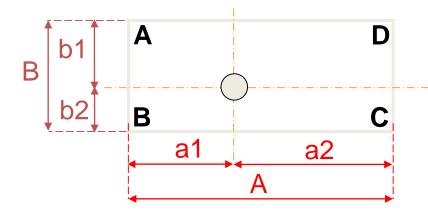


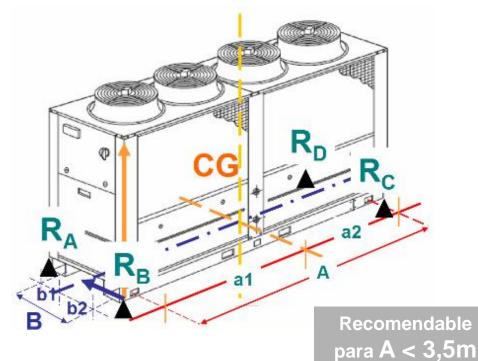
Enfriadoras refrigeradas por agua condensadas por aire



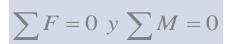








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$$M_{x} = R_{X} \cdot d_{x}$$

$$R_{\scriptscriptstyle A} + R_{\scriptscriptstyle B} + R_{\scriptscriptstyle C} + R_{\scriptscriptstyle D} + P = 0$$

$$R_{A} = \frac{b2}{B} \cdot \frac{a2}{A} P$$

$$R_{B} = \frac{b1}{B} \cdot \frac{a2}{A} P$$

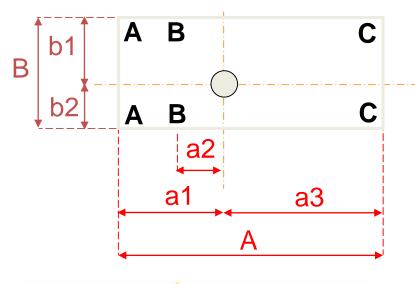
$$R_{D} = \frac{b1}{B} \cdot \frac{a1}{A} P$$

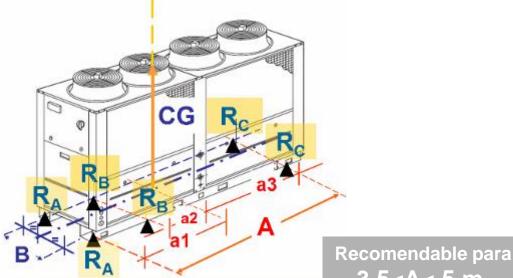
$$R_{C} = \frac{b2}{B} \cdot \frac{a1}{A} P$$



(ref: Criterio Rafa Torres)

#### 6 Apoyos es + ESTABLE





$$2R_A + 2R_B + 2R_D = P$$

$$2R_A + 2R_B + 2R_D = P$$

$$R_A = R_B = R_C = \frac{1}{6}P$$

$$MR_{A} + MR_{B} - MR_{C} = 0$$

$$MR_A + MR_B = MR_C$$

$$(R_A \cdot a1) + (R_A \cdot a2) = R_A \cdot a3$$

$$R_{A}(a1+a2) = R_{A}a3$$

$$a1 + a2 = a3$$

$$R_A = \frac{1}{6}P$$



3,5<A< 5 m. (ref: Criterio Rafa Torres)







HCT-40C

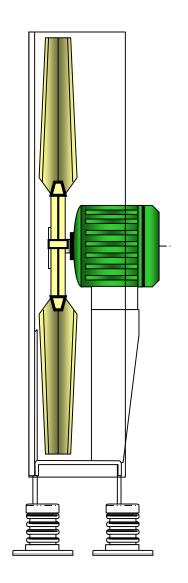


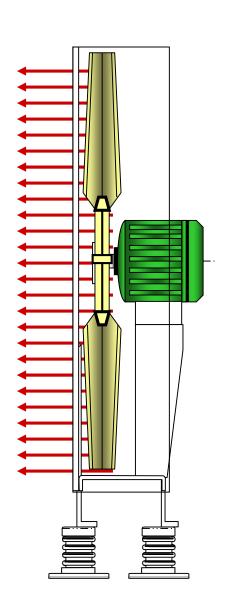


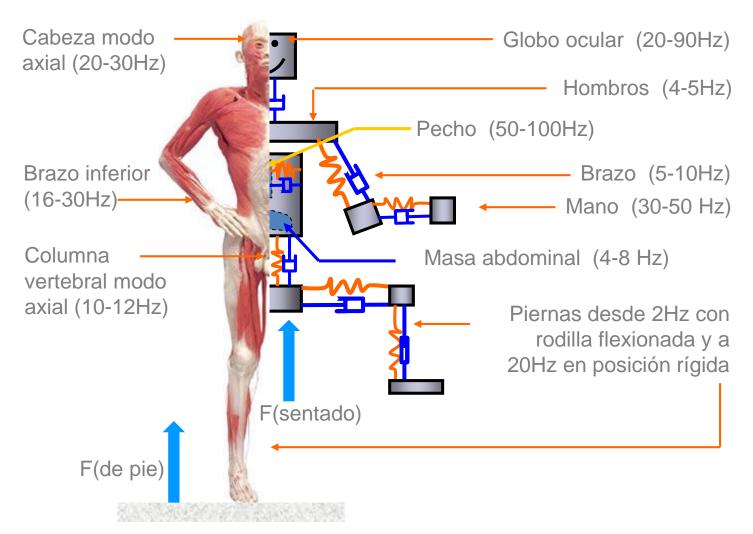




#### Caso 3: Aislamiento ventiladores axiales









Actualizado18/12/2012.Ref: Curso B&K-2008

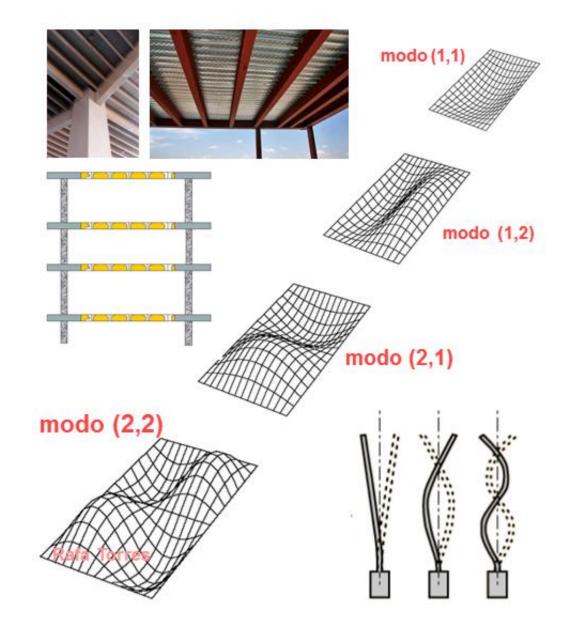
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Ref: Jesús Uriol Acusticsambient 30/04/2012 en Sort (Llelida)



# Vigas y forjados en edificios

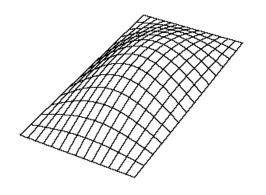
Ref: Acústica Aparejadores BCN 2013

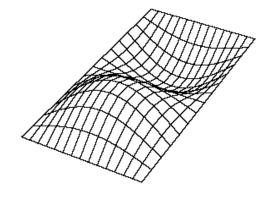




Modo (1,1)

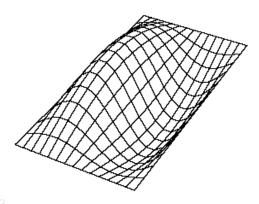
Modo (1,2)

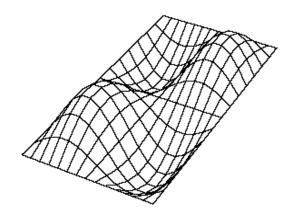




Modo (2,1)

Modo (2,2)

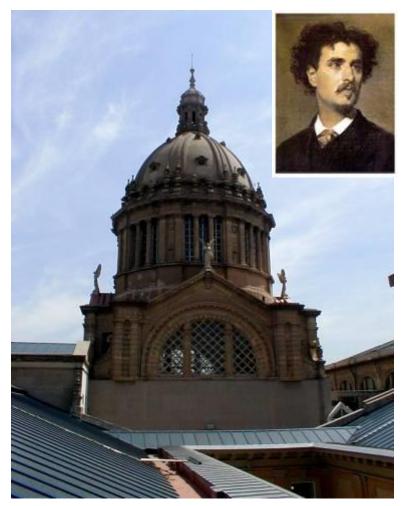




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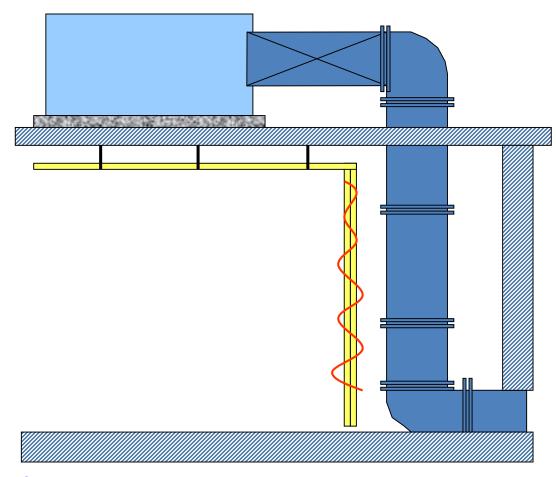






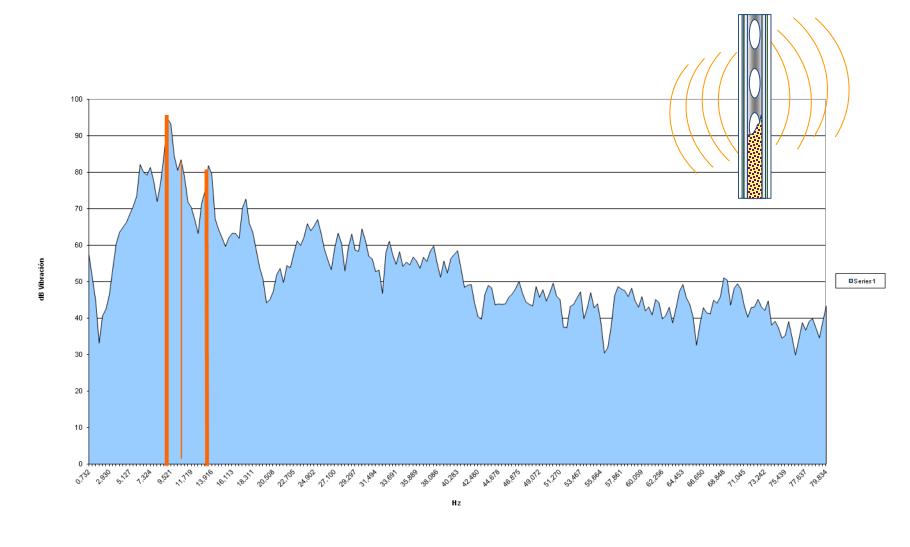


#### 750 rpm (12,5 hz.)



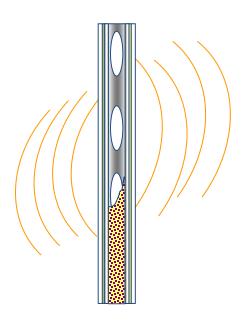
Fuente: MNAC SALA 2 FORTUNY 12/2004







$$f_0 = \frac{600}{\sqrt{md}}$$



- m=masa superficial de la membrana Kg/m²
- •d= distancia de la cámara de aire en cm.

m	d	fO
28	70	14 Hz



# 1. Estrategias de actuación



2.Grado de libertad-1



3. Montaje antivibratorio



4. Vibraciones forzadas



5. Tipos de MA



