

## Psychrometrics:

①

1) Specific humidity / Humidity Ratio:

$$\omega \triangleq \frac{m_v}{m_a} = 0.622 \frac{p_v}{p - p_v}$$

2) Relative humidity:  $\phi \triangleq \frac{m_v}{m_{v, \text{sat}}} = \frac{p_v}{p_{\text{sat}}}$

(3) Specific Enthalpy

$$h = c_{p,a} T + \omega h_v(T)$$

Assume  $h_v(T) \approx h_g(T)$  (from the tables)

$$[h] = \frac{\text{kJ}}{\text{kg d.a}}$$

$$h = c_{p,a} T + \omega h_g(T) \leftarrow \text{If you're given } T$$

$$\text{Another approximation: } h = c_{p,a} T + \omega (2500 + 1.9 T)$$

$\nwarrow$  if you need to find  $T$  from  $h$ .

(4) Specific volume

$$v \triangleq \frac{V}{m_a} = \frac{T(1+\omega)}{\frac{p_a}{R_a} + \frac{p_v}{R_v}}$$

$$R_a = \frac{\bar{R}}{MW_{\text{air}}}$$

$$R_v = \frac{\bar{R}}{MW_{H_2O}}$$

$$[v] = \text{m}^3 / \text{kg d.a}$$



Divide by  $\dot{m}_a$   $\frac{\dot{m}_f}{\dot{m}_a} = \frac{\dot{m}_{v2}}{\dot{m}_a} - \frac{\dot{m}_{v1}}{\dot{m}_a}$   
 $= \omega_2 - \omega_1$

## Energy Balance

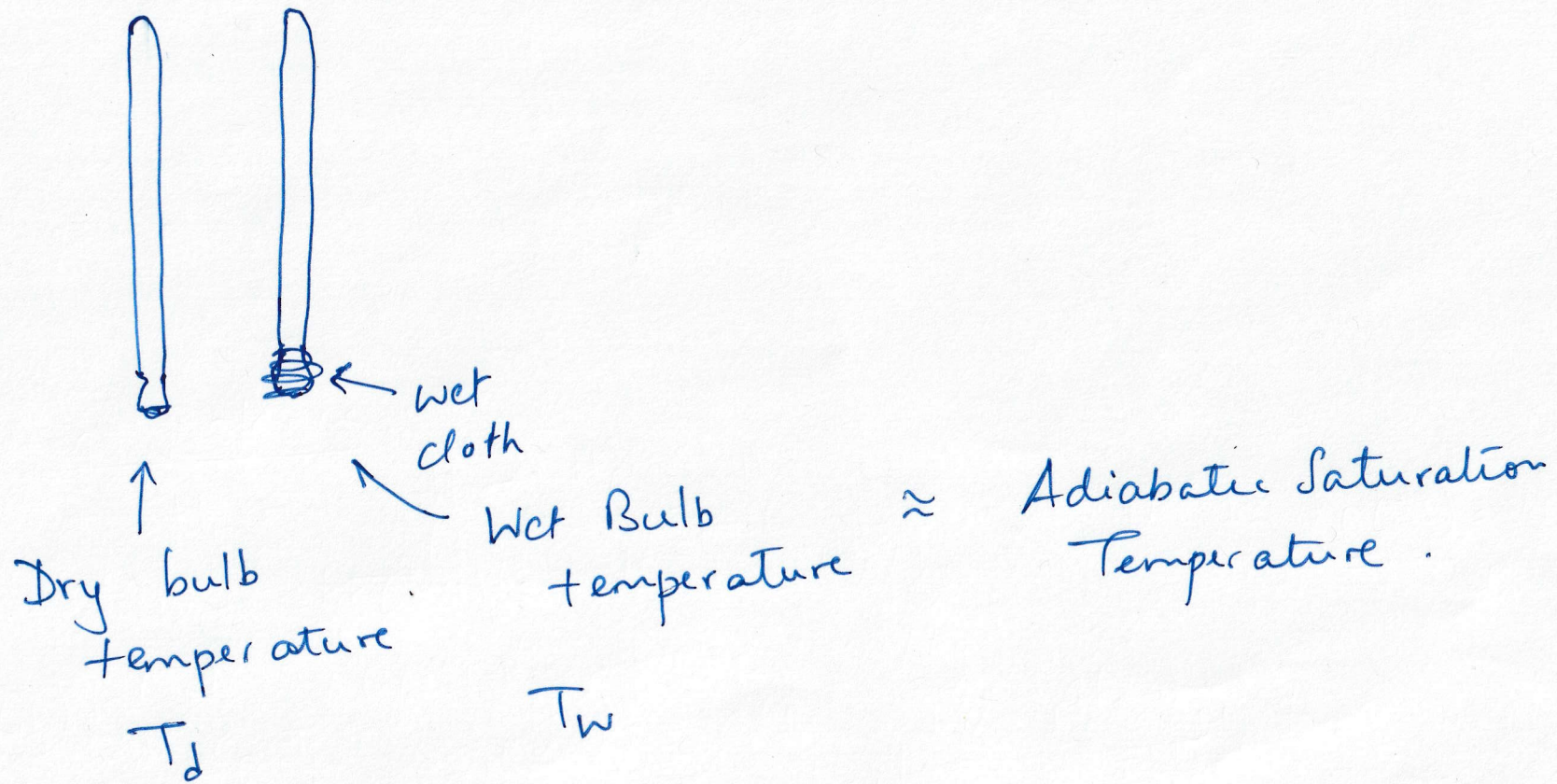
$$\dot{m}_a c_p T_1 + \dot{m}_a \omega_1 h_g(T_1) + \dot{m}_a (\omega_2 - \omega_1) h_f(T_2) = \dot{m}_a c_p T_2 + \dot{m}_a \omega_2 h_g(T_2)$$

$$\omega_1 (h_g(T_1) - h_f(T_2)) = c_p (T_2 - T_1) + \omega_2 [h_g(T_2) - h_f(T_2)]$$

→ Adiabatic Saturation

$T_2 \rightarrow$  Adiabatic Saturation Temperature .

(3)





$$T_d = 40^\circ\text{C} \quad T_w = 25^\circ\text{C} \quad RH = ? \quad \omega = ?$$

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$$\omega_1 = \frac{C_p a (T_2 - T_1) + \omega_2 h_{fg}(T_2)}{h_g(T_1) - h_f(T_2)}$$

$$T_1 = T_d$$

$$T_2 = T_w$$

$$h_g(T_1) = h_g(40) = 2574.3 \text{ kJ/kg}$$

$$h_{fg}(T_2) = h_{fg}(25) = 2547.2 \text{ kJ/kg}; \quad h_f(T_2) = h_f(25) = 104.89 \text{ kJ/kg}$$

$$\omega_2 = 0.622 \frac{p_{v2}}{P - p_{v2}} \quad \text{At 2, } \phi = 1 \Rightarrow p_v = p_{\text{sat}}(25)$$

$$= 0.03169 \text{ bar}$$

$$= (0.622) \frac{0.03169}{1.013025 - 0.03169}$$

$$= 0.0201 \text{ kg/kg d.a.}$$

$$\omega_1 = \frac{(1.005)(25 - 40) + (0.0201)(2547.2)}{2574.3 - 104.89} = 0.01463 \text{ kg/kg d.a.}$$

$$\omega_1 = 0.622 \frac{p_{v1}}{P - p_{v1}} \Rightarrow \omega_1 P - \omega_1 p_{v1} = 0.622 p_{v1}$$

$$\Rightarrow p_{v1} = \frac{\omega_1 P}{0.622 + \omega_1} = \frac{(0.01463)(1.013025)}{0.622 + 0.01463}$$

$$= 0.02328 \text{ bar}$$

$$\phi_1 = \frac{p_{v1}}{p_{\text{sat}}}$$

$$p_{\text{sat}}(40) = 0.07384 \text{ bar}$$

$$\phi = \frac{0.02328}{0.07384} = 31.5\%$$

$$h = C_{pa} T_d + \omega h_g(T_d)$$

$$= (1.005) 40 + (0.01463) (2574.3)$$

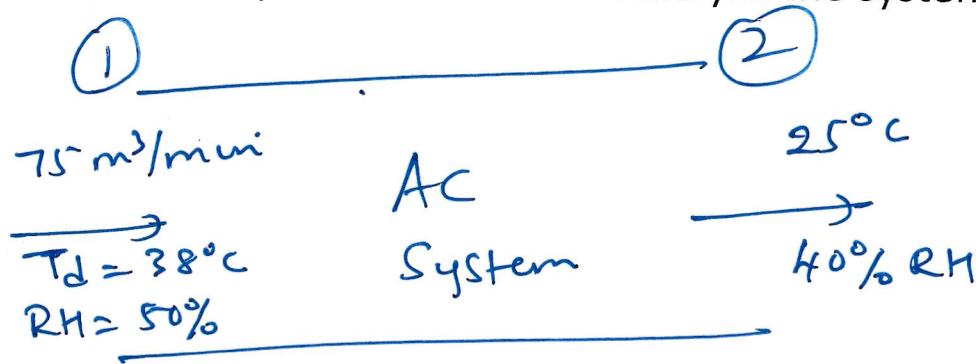
$$h = 77.86 \text{ kJ/kg d.a.}$$

$$v = \frac{T(1+\omega)}{\frac{p_a}{R_a} + \frac{p_v}{R_v}} = \frac{(1+0.01463)(273+40)}{\frac{1.010325-0.01463}{29} + \frac{0.01463}{18}}$$

$$= \frac{(1+0.01463)(273+40)}{\left[ \left( \frac{1.010325-0.0238}{29} \right) + \left( \frac{0.0238}{18} \right) \right]} 10^3 \leftarrow \text{bar to kPa conversion}$$

(6)

75 m<sup>3</sup>/min of air at 38°C and 50% RH is to be conditioned and delivered to a room at 25°C and 40% RH. The coefficient of performance of 3. Analyze the system



$$h_1 = c_p T_1 + \omega_1 h_g(T_1)$$

At  $T_d = 38^\circ\text{C}$ ,  $p_{\text{sat}} = 0.06632$ , since  $\phi = 50\%$ ,  $p_v = 0.03316$  bar

$$\omega_1 = (0.622) \frac{0.03316}{1.013025 - 0.03316} = 0.02105 \text{ kg/kg d.a.}$$

At  $T_d = 25^\circ\text{C}$ ,  $p_{\text{sat}} = 0.03169$  bar,  $\phi = 40\% \Rightarrow p_v = 0.012676$  bar

$$\omega_2 = (0.622) \frac{0.012676}{1.013025 - 0.012676} = 7.882 \times 10^{-3} \text{ kg/kg d.a.}$$

$p_v$  corresponding to  $\omega_2 = \frac{\omega_2 p}{0.622 + \omega_2} = \frac{7.882 \times 10^{-3} \times 1.013025}{0.622 + 7.882 \times 10^{-3}} = 0.01268$  bar

Cool it down to 10.5°C (corresponding to  $p_{\text{sat}} = 0.01268$  bar)