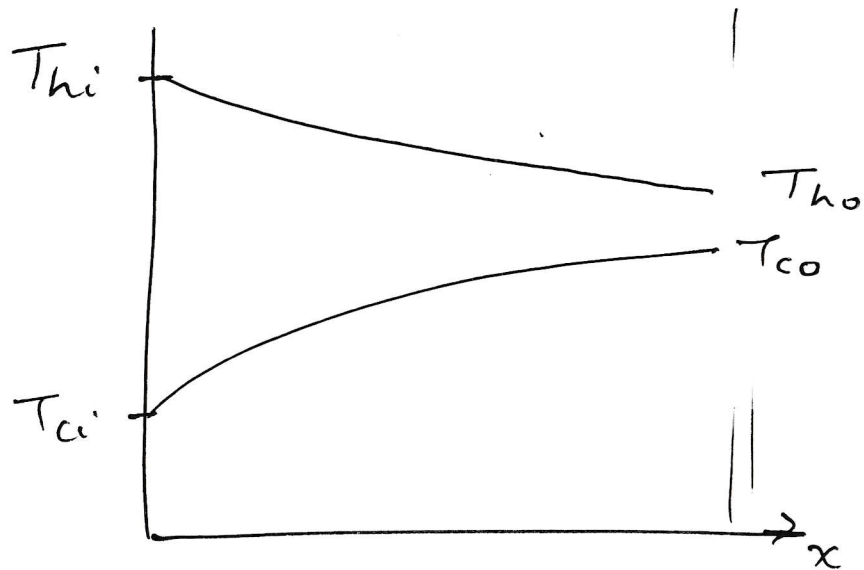
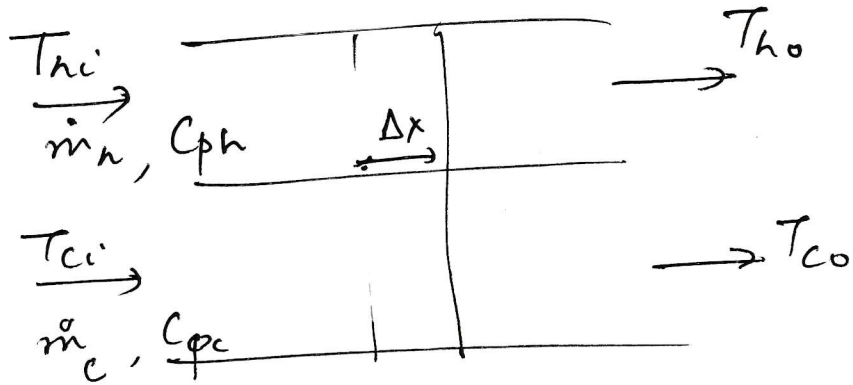


# Parallel flow heat exchangers

①



$$\dot{Q} = \underbrace{\dot{m}_h C_{ph}}_{\parallel \Delta} (T_{hi} - T_{ho}) = \underbrace{\dot{m}_c C_{pc}}_{\parallel \Delta} (T_{co} - T_{ci})$$

$C_h \qquad C_c$

$$\dot{Q} = C_h (T_{hi} - T_{ho}) = C_c (T_{co} - T_{ci})$$

$$\Rightarrow T_{hi} - T_{ho} = \frac{\dot{Q}}{C_h} ; T_{co} - T_{ci} = \frac{\dot{Q}}{C_c}$$

$$\Rightarrow (T_{hi} - T_{ho}) + (T_{co} - T_{ci}) = \dot{Q} \left( \frac{1}{C_h} + \frac{1}{C_c} \right)$$

$$\Rightarrow (T_{hi} - T_{ci}) - (T_{ho} - T_{co}) = \dot{Q} \left( \frac{1}{C_h} + \frac{1}{C_c} \right)$$

$$\Delta T_i - \Delta T_o = \dot{Q} \left( \frac{1}{C_h} + \frac{1}{C_c} \right)$$

$$\Rightarrow \left( \frac{1}{C_h} + \frac{1}{C_c} \right) = \frac{\Delta T_i - \Delta T_o}{\dot{Q}}$$

From Page ⑨ for previous lecture -

②

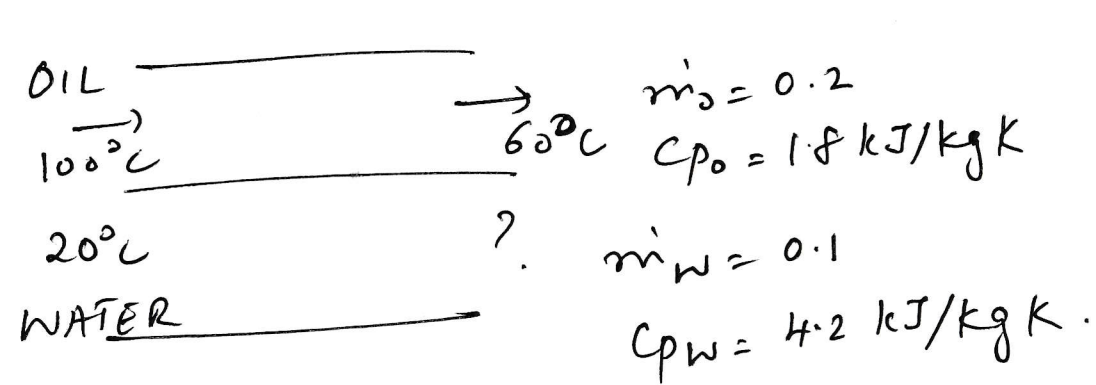
$$\boxed{\ln \frac{T_{ho} - T_{co}}{T_{hi} - T_{ci}} = - \left( \frac{1}{C_h} + \frac{1}{C_c} \right) U A}$$
$$= - \frac{\Delta T_i - \Delta T_o}{\dot{Q}} U A$$

$$\dot{Q} = U A \frac{\Delta T_i - \Delta T_o}{-\ln \left( \frac{\Delta T_o}{\Delta T_i} \right)}$$

$$= U A \left( \frac{\Delta T_i - \Delta T_o}{\ln \frac{\Delta T_i}{\Delta T_o}} \right)$$

log mean temperature difference  
(LMTD)

Oil at 100°C is to be cooled to 60°C in a parallel-flow heat exchanger by means of water that is available at 20°C. The flow rates of the oil and the water are respectively 0.2 kg/s and 0.1 kg/s. The overall heat transfer coefficient is  $40 \text{ Wm}^2\text{K}^{-1}$ . The specific heat of oil is  $1.8 \text{ kJ/kg K}$ . What is the surface area of the heat exchanger?



$$\dot{Q} = \dot{m}_h (T_{h,i} - T_{h,o}) = \dot{m}_c (T_{c,o} - T_{c,i})$$

$$\dot{Q} = (0.2)(1.8)(100 - 60) = 14.4 \text{ kW}$$

$$= (0.1)(4.2)(T_{c,o} - 20)$$

$$\Rightarrow T_{c,o} = 54.3^\circ\text{C}$$

$$\dot{Q} = \dot{V} A (LMTD)$$

$$LMTD = \frac{\Delta T_i - \Delta T_o}{\ln \frac{\Delta T_i}{\Delta T_o}}$$

$$= \frac{80 - 5.7}{\ln \left( \frac{80}{5.7} \right)} = 28.13^\circ\text{C}$$

$$\Delta T_i = T_{h,i} - T_{c,i} = 100 - 20 = 80^\circ\text{C}$$

$$\Delta T_o = T_{h,o} - T_{c,o} = 60 - 54.3 = 5.7^\circ\text{C}$$

$$(14.4)(10^3) = (40) A (28.13) \Rightarrow \boxed{A = 12.8 \text{ m}^2}$$

## Effectiveness - NTU Method

NTU = net transfer units. (4)

$$\text{Effectiveness} = \frac{\text{Actual heat transfer}}{\text{Maximum possible heat transfer}} = \frac{\dot{q}}{\dot{q}_{\max}}$$

$$\dot{q}_{\max} = C_h (T_{hi} - T_{ci})$$

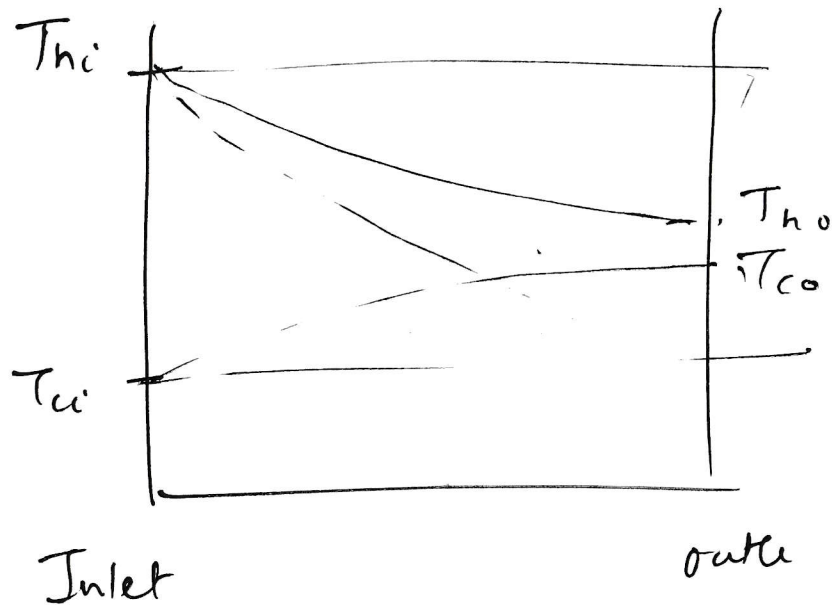
In general

$$\dot{q}_{\max} = C_{\min} (T_{hi} - T_{ci})$$

where  $C_{\min} = \min(C_h, C_c)$

$$C_{\max} = \max(C_h, C_c)$$

$$\text{Define } Cr \triangleq \frac{C_{\min}}{C_{\max}}$$



$$\text{Effectiveness } \epsilon = \frac{C_h (T_{hi} - T_{ho})}{C_{\min} (T_{hi} - T_{ci})} = \frac{C_c (T_{co} - T_{ci})}{C_{\min} (T_{hi} - T_{ci})}$$

Let  $C_{\min} = C_c$ ;  $C_{\max} = C_h$

$$\epsilon = \frac{C_{\max} (T_{hi} - T_{ho})}{C_{\min} (T_{hi} - T_{ci})} = \frac{C_{\min} (T_{co} - T_{ci})}{C_{\min} (T_{hi} - T_{ci})} \Rightarrow$$

$$\epsilon = \frac{1}{C_r} \cdot \frac{T_{hi} - T_{ho}}{T_{hi} - T_{ci}} = \frac{T_{co} - T_{ci}}{T_{hi} - T_{ci}} \quad (5)$$

From Page 2,

$$\begin{aligned} \ln \frac{T_{ho} - T_{co}}{T_{hi} - T_{ci}} &= - \left( \frac{1}{C_h} + \frac{1}{C_c} \right) UA \\ &= - \left( \frac{1}{C_{max}} + \frac{1}{C_{min}} \right) UA \\ &= - \frac{1}{C_{min}} \left( \frac{C_{min}}{C_{max}} + 1 \right) UA \\ &= - \left( \frac{UA}{C_{min}} \right) (1 + Cr) \end{aligned}$$

Let  $NTU \stackrel{\Delta}{=} \frac{UA}{C_{min}}$   
 net transfer unit

$$\begin{aligned} &= - NTU (1 + Cr) \\ &= e^{-NTU (1 + Cr)} \\ \Rightarrow \frac{T_{ho} - T_{co}}{T_{hi} - T_{ci}} &= e^{-NTU (1 + Cr)} \end{aligned}$$

$$LHS = \frac{T_{ho} - T_{co}}{T_{hi} - T_{ci}}$$

(6)

$$= \frac{T_{ho} - T_{hi} + T_{hi} - T_{ci} + T_{ci} - T_{co}}{T_{hi} - T_{ci}}$$

$$= - \left( \frac{T_{hi} - T_{ho}}{T_{hi} - T_{ci}} \right) + 1 - \left( \frac{T_{co} - T_{ci}}{T_{hi} - T_{ci}} \right)$$

From page (5),  $\frac{T_{hi} - T_{ho}}{T_{hi} - T_{ci}} = \epsilon \cancel{Cr}$ ;  $\frac{T_{co} - T_{ci}}{T_{hi} - T_{ci}} = \epsilon$   
 $T_{hi} - T_{ci} = \epsilon Cr$

$$LHS = 1 - \epsilon Cr - \epsilon$$

$$= 1 - \epsilon (Cr + 1)$$

$$1 - \epsilon (Cr + 1) = e^{-NTU (1 + Cr)} \Rightarrow$$

$$\epsilon = \frac{1 - e^{-NTU (1 + Cr)}}{1 + Cr}$$

$$NTU = \frac{-\ln(1 - \epsilon (Cr + 1))}{1 + Cr}$$

Oil at  $100^\circ\text{C}$  is to be cooled in a parallel heat exchanger with area  $9\text{ m}^2$  by means of water that is available at  $20^\circ\text{C}$ . The flow rates of the oil and the water are respectively  $0.2\text{ kg/s}$  and  $0.1\text{ kg/s}$ . The overall heat transfer coefficient is  $40\text{ Wm}^{-2}\text{K}^{-1}$ . The specific heat of oil is  $1.8\text{ kJ/kg K}$ . What are the temperatures of the oil and water at their exits?

$$\begin{array}{l} \text{OIL} \xrightarrow{\dot{m}_o = 0.2; C_{ph} = 1.8} \\ 100^\circ\text{C} \\ \text{WATER} \xrightarrow{\dot{m}_w = 0.1; C_{pw} = 4.2} \\ 20^\circ\text{C} \end{array} \quad A = 9\text{ m}^2$$

$$\begin{aligned} C_h &= \dot{m}_h C_{ph} = (0.2)(1.8) = 0.36 \\ C_c &= \dot{m}_c C_{pc} = (0.1)(4.2) = 0.42 \\ C_{min} &= 0.36, \quad C_{max} = 0.42 \\ C_r &= \frac{C_{min}}{C_{max}} = \frac{0.36}{0.42} = 0.857 \end{aligned}$$

$$NTU = \frac{UA}{C_{min}} = \frac{(40)(9)}{0.36} = 1000$$

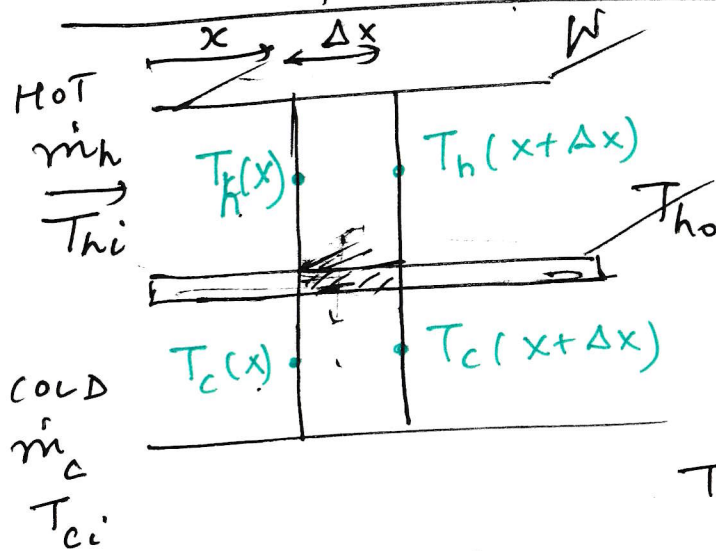
$$\epsilon = \frac{1 - e^{-NTU(1+C_r)}}{1+C_r} = \frac{1 - e^{-1000(1+0.857)}}{1+0.857} = 0.539$$

$$\epsilon = \frac{T_{hi} - T_{ho}}{T_{hw} - T_{ci}} \Rightarrow 0.539 = \frac{100 - T_{ho}}{100 - 20} \Rightarrow T_{ho} = 56.9^\circ\text{C}$$

$$= \frac{(T_{co} - T_{ci})}{C_r(T_{hi} - T_{ci})} \Rightarrow 0.539 = \frac{T_{co} - 20}{0.857(100 - 20)} \Rightarrow T_{co} = 56.9^\circ\text{C}$$



# Parallel flow heat exchanger



$$\dot{Q} = \underbrace{\dot{m}_h C_{ph}}_{C_h} (T_{hi} - T_{ho}) = \underbrace{\dot{m}_c C_{pc}}_{C_c} (T_{co} - T_{ci})$$

Define  $C_h \triangleq \dot{m}_h C_{ph}$ ;  $C_c \triangleq \dot{m}_c C_{pc}$

Balance for hot fluid

$$\dot{m}_h C_h T_h(x) - \dot{m}_h C_h T_h(x+\Delta x) - \delta \dot{q} = 0$$

$$\begin{aligned} -\delta \dot{q} &= C_h (T_h(x+\Delta x) - T_h(x)) \\ &= C_h \left[ T_h(x) + \frac{dT_h}{dx} \Delta x - T_h(x) \right] \\ &= C_h \frac{dT_h}{dx} \Delta x \rightarrow (1) \end{aligned}$$

Cold fluid:  $C_c T_c(x) - C_c T_c(x+\Delta x) + \delta \dot{q} = 0$

$$\begin{aligned} \delta \dot{q} &= C_c (T_c(x+\Delta x) - T_c(x)) \\ &= C_c \frac{dT_c}{dx} \Delta x \rightarrow (2) \end{aligned}$$

$$\begin{aligned} (1) &\Rightarrow \frac{dT_h}{dx} \Delta x = \frac{-\delta \dot{q}}{C_h} \\ (2) &\Rightarrow \frac{dT_c}{dx} \Delta x = \frac{\delta \dot{q}}{C_c} \end{aligned}$$

Subtract (2) from (1)

$$\left( \frac{dT_h}{dx} - \frac{dT_c}{dx} \right) \Delta x = -\delta \dot{q} \left( \frac{1}{C_h} + \frac{1}{C_c} \right)$$



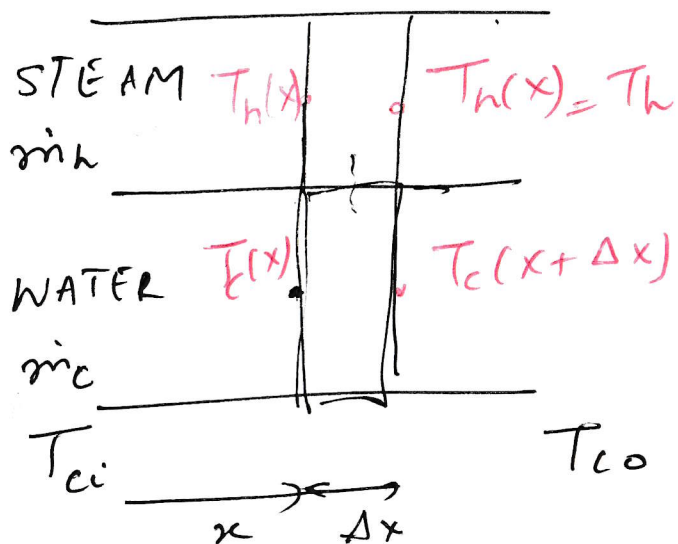
$$\delta \dot{q} = U (W \Delta x) (T_h(x) - T_c(x)) \quad (9)$$

$$\frac{d(T_h - T_c)}{dx} \Delta x = - \left( \frac{1}{C_h} + \frac{1}{C_c} \right) U W \Delta x (T_h(x) - T_c(x))$$

$$\int \frac{d(T_h - T_c)}{(T_h - T_c)} = \int_0^L \left( \frac{1}{C_h} + \frac{1}{C_c} \right) U W dx$$

$$\ln(T_h - T_c) \Big|_{\text{inlet}}^{\text{outlet}} = - \left( \frac{1}{C_h} + \frac{1}{C_c} \right) U \underbrace{W L}$$

$$\ln \frac{T_{ho} - T_{co}}{T_{hi} - T_{ci}} = \left( \frac{1}{C_h} + \frac{1}{C_c} \right) U A$$



$$\dot{Q} = \dot{m}_h h_{fg} = \dot{m}_c C_{p,c} (T_{c,o} - T_{c,i})$$

$$\dot{Q} = \dot{m}_h h_{fg} = C_c (T_{c,o} - T_{c,i})$$

Balance on cold fluid

$$C_c T_c(x) - C_c (T_c(x + \Delta x)) + \delta \dot{q} = 0$$

$$\delta \dot{q} = C_c (T_c(x + \Delta x) - T_c(x))$$

$$= C_c \frac{dT_c}{dx} \Delta x$$

But  $\delta \dot{q} = U(\Delta x W)(T_h - T_c(x))$

$$U \cancel{\Delta x} W (T_h - T_{c, \text{output}}(x)) = C_c \frac{dT_c}{dx} \cancel{\Delta x}$$

$$\int_{x=0}^{x=L} \frac{UW}{C_c} dx = \int_{\text{inlet}}^{\text{outlet}} \frac{dT_c}{T_h - T_c(x)}$$

$$\Rightarrow \frac{U(WL)}{C_c} = \ln T_h - T_c(x) \Big|_{\text{in}}^{\text{out}}$$

$$\Rightarrow \ln \left( \frac{T_h - T_{c,o}}{T_h - T_{c,i}} \right) = -\frac{UA}{C_c}$$

$$\Rightarrow \frac{T_h - T_{c,o}}{T_h - T_{c,i}} = e^{-NTU}$$

$$= -\frac{UA}{C_{min}} = -NTU$$

$$\frac{T_h - T_{ci} + T_{ci} - T_{co}}{T_h - T_{ci}} = e^{-NTU}$$

$$1 - \frac{T_{co} - T_{ci}}{T_h - T_{ci}} = e^{-NTU}$$

$$1 - \epsilon = e^{-NTU}$$

$$\Rightarrow \boxed{\epsilon = 1 - e^{-NTU}}$$

**TABLE 11.3** Heat Exchanger Effectiveness Relations [5]

| Flow Arrangement                                | Relation   |                        |
|---|--|------------------------|
| Parallel flo                                    | $\varepsilon = \frac{1 - \exp[-NTU(1 + C_r)]}{1 + C_r}$  | (11.28a)               |
| Counterflo                                      | $\varepsilon = \frac{1 - \exp[-NTU(1 - C_r)]}{1 - C_r \exp[-NTU(1 - C_r)]}$  | ( $C_r < 1$ )          |
|   | $\varepsilon = \frac{NTU}{1 + NTU}$  | ( $C_r = 1$ ) (11.29a) |
| Shell-and-tube                                  |  |                        |
| One shell pass (2, 4, . . . tube passes)        | $\varepsilon_1 = 2 \left\{ 1 + C_r + (1 + C_r^2)^{1/2} \times \frac{1 + \exp[-(NTU)_1(1 + C_r^2)^{1/2}]}{1 - \exp[-(NTU)_1(1 + C_r^2)^{1/2}]} \right\}^{-1}$                                   | (11.30a)               |
| $n$ shell passes ( $2n, 4n, . . .$ tube passes) | $\varepsilon = \left[ \left( \frac{1 - \varepsilon_1 C_r}{1 - \varepsilon_1} \right)^n - 1 \right] \left[ \left( \frac{1 - \varepsilon_1 C_r}{1 - \varepsilon_1} \right)^n - C_r \right]^{-1}$ | (11.31a)               |
| Cross-flow (single pass                         |  |                        |
| Both fluids unmixed                             | $\varepsilon = 1 - \exp \left[ \left( \frac{1}{C_r} \right) (NTU)^{0.22} \{ \exp[-C_r(NTU)^{0.78}] - 1 \} \right]$   | (11.32)                |
| $C_{\max}$ (mixed), $C_{\min}$ (unmixed)        | $\varepsilon = \left( \frac{1}{C_r} \right) (1 - \exp \{ -C_r [1 - \exp(-NTU)] \})$  | (11.33a)               |
| $C_{\min}$ (mixed), $C_{\max}$ (unmixed)        | $\varepsilon = 1 - \exp(-C_r^{-1} \{ 1 - \exp[-C_r(NTU)] \})$  | (11.34a)               |
| All exchangers ( $C_r = 0$ )                    | $\varepsilon = 1 - \exp(-NTU)$   | (11.35a)               |



**TABLE 11.4** Heat Exchanger NTU Relations

| Flow Arrangement                                   | Relation  |                |
|--|---|----------------|
| Parallel flo                                       | $\text{NTU} = -\frac{\ln[1 - \varepsilon(1 + C_r)]}{1 + C_r}$   | (11.28b)       |
| Counterflo   | $\text{NTU} = \frac{1}{C_r - 1} \ln\left(\frac{\varepsilon - 1}{\varepsilon C_r - 1}\right) \quad (C_r < 1)$  |                |
|  | $\text{NTU} = \frac{\varepsilon}{1 - \varepsilon} \quad (C_r = 1)$  | (11.29b)       |
| Shell-and-tube                                     |   |                |
| One shell pass<br>(2, 4, . . . tube passes)        | $(\text{NTU})_1 = -(1 + C_r^2)^{-1/2} \ln\left(\frac{E - 1}{E + 1}\right)$  | (11.30b)       |
|  | $E = \frac{2/\varepsilon_1 - (1 + C_r)}{(1 + C_r^2)^{1/2}}$   | (11.30c)       |
| $n$ shell passes<br>( $2n, 4n, \dots$ tube passes) | Use Equations 11.30b and 11.30c with<br>$\varepsilon_1 = \frac{F - 1}{F - C_r} \quad F = \left(\frac{\varepsilon C_r - 1}{\varepsilon - 1}\right)^{1/n} \quad \text{NTU} = n(\text{NTU})_1$ | (11.31b, c, d) |
| Cross-flow (single pass)                           |   |                |
| $C_{\max}$ (mixed), $C_{\min}$ (unmixed)           | $\text{NTU} = -\ln\left[1 + \left(\frac{1}{C_r}\right) \ln(1 - \varepsilon C_r)\right]$   | (11.33b)       |
| $C_{\min}$ (mixed), $C_{\max}$ (unmixed)           | $\text{NTU} = -\left(\frac{1}{C_r}\right) \ln[C_r \ln(1 - \varepsilon) + 1]$  | (11.34b)       |
| All exchangers ( $C_r = 0$ )                       | $\text{NTU} = -\ln(1 - \varepsilon)$  | (11.35b)       |