

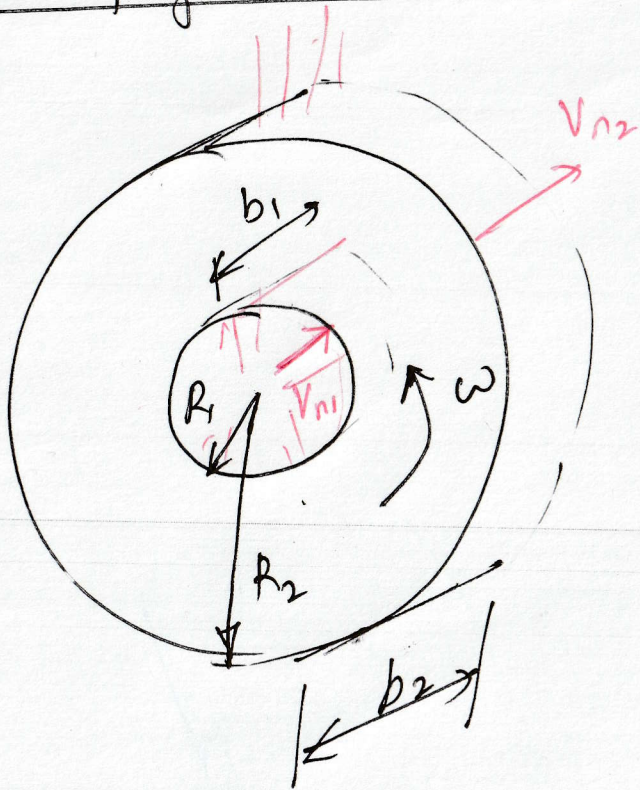
# Turbomachines

1) Centrifugal

2) Axial

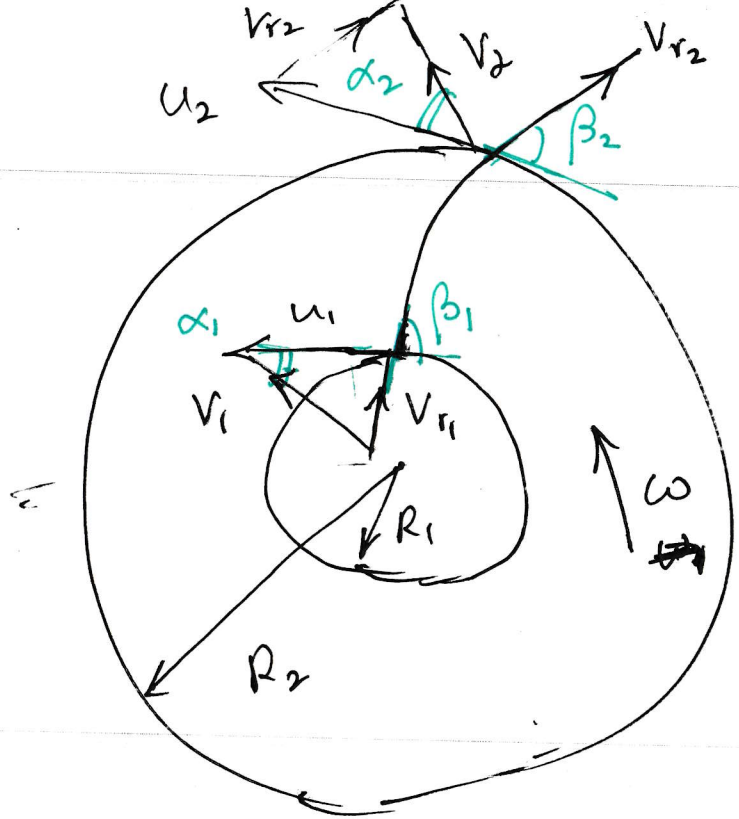
①

## 1) Centrifugal machines



$$\begin{aligned}\dot{V} &= A V_{n1} \\ &= 2\pi R_1 b_1 V_{n1} \\ &= 2\pi R_2 b_2 V_{n2}\end{aligned}$$

$$\dot{m} = \rho \int \dot{V}$$



$$u_1 = \omega R_1$$

$$u_2 = \omega R_2$$

$\beta_1, \beta_2$  are the blade angles at inlet and outlet relative to the tangent of the circle at the blade

$V_{r1}, V_{r2}$  are the relative velocity vectors at the inlet and outlet

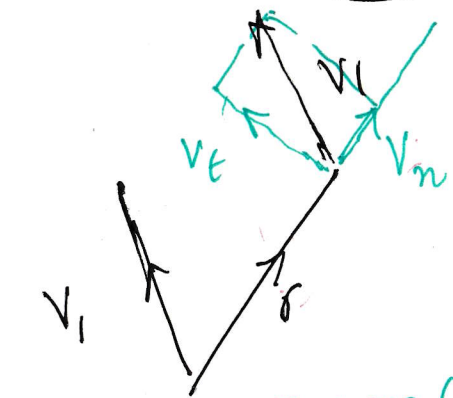
$V_1 =$  ~~vel~~ absolute velocity at inlet

$$= V_{r1} + u_1$$

$V_2 =$  absolute velocity at exit

$$= V_{r2} + u_2$$

$\alpha_1, \alpha_2 =$  angle made by inlet and outlet velocity vectors with the tangent to the circle at the inlet and outlet respectively.



$$\begin{aligned} r \times m v &= r \times m (V_t + V_r) \\ &= m(r \times V_t) + (r \times V_r) \\ &= m r V_t \end{aligned}$$

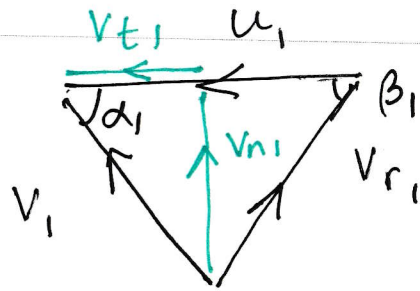
Conservation of angular momentum (steady state)

$$\dot{H}_1 - \dot{H}_0 + \sum T = 0 \Rightarrow T = \dot{H}_0 - \dot{H}_1 = \dot{m} r_2 V_{t2} - \dot{m} r_1 V_{t1}$$

$$H_i = r \times m v = r m V_t$$



Inlet



$$V_{n1} = V_1 \sin \alpha_1 = V_{r1} \sin \beta_1 \Rightarrow V_{r1} = \frac{V_{n1}}{\sin \beta_1}$$

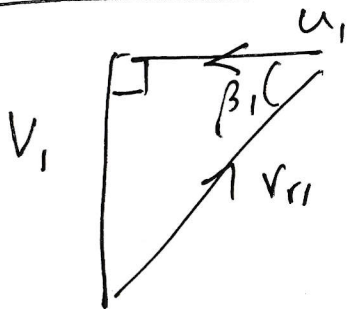
$$V_{t1} = u_1 - V_{r1} \cos \beta_1$$

$$= V_1 \cos \alpha_1$$

$$u_1 - \left( \frac{V_{n1}}{\sin \beta_1} \right) \cos \beta_1$$

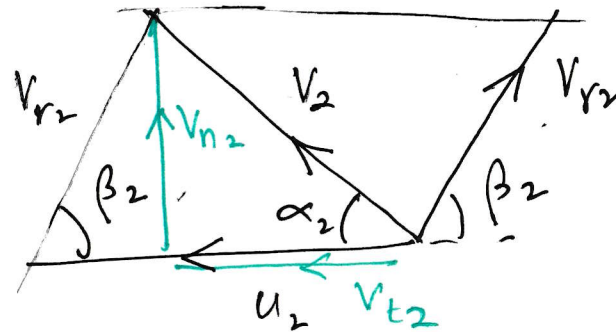
$$V_{t1} = u_1 - V_{n1} \cot \beta_1$$

Special Case:  $V_{t1} = 0$  (Radial entry)



$$\boxed{\alpha_1 = 90^\circ}$$

Outlet



$$V_{n2} = V_2 \sin \alpha_2 = V_{r2} \sin \beta_2 \Rightarrow V_{r2} = \frac{V_{n2}}{\sin \beta_2}$$

$$V_{t2} = V_2 \cos \alpha_2 = u_2 - V_{r2} \cos \beta_2$$

$$= u_2 - \frac{V_{n2}}{\sin \beta_2} \cos \beta_2$$

$$= u_2 - V_{n2} \cot \beta_2$$

(3)

$$T = \dot{m} r_2 V_{t2} - \dot{m} r_1 V_{t1}$$

$$= \dot{m} [r_2 V_{t2} - r_1 V_{t1}]$$

$$P = T \omega$$

$$= \dot{m} [\omega r_2 V_{t2} - \omega r_1 V_{t1}]$$

$$= \dot{m} [u_2 V_{t2} - u_1 V_{t1}]$$

$$w = \frac{P}{\dot{m}} = u_2 V_{t2} - u_1 V_{t1}$$

work  
done per  
unit mass

Special Case:  $V_{t1} = 0$

$$w = u_2 V_{t2}$$

$$= u_2 (u_2 - V_{n2} \cot \beta_2)$$

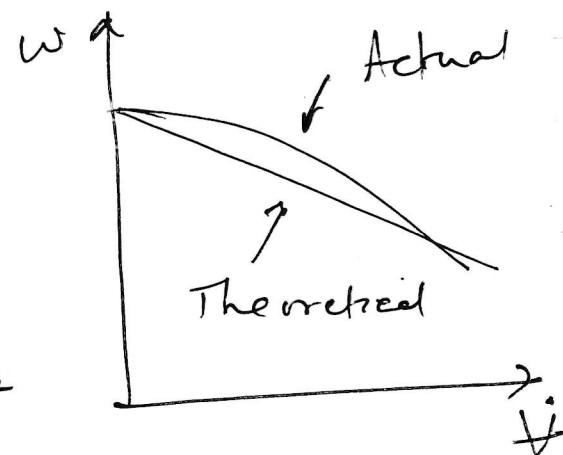
$$= u_2^2 - u_2 V_{n2} \cot \beta_2$$

Since  $\dot{V} = 2\pi R_2 b_2 V_{n2} \Rightarrow$

$$V_{n2} = \frac{\dot{V}}{2\pi R_2 b_2}$$

$$w = u_2^2 - u_2 \frac{\cot \beta_2}{2\pi R_2 b_2} \dot{V}$$

← Euler's Turbomachines  
equation.



# Thermodynamics

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Energy in + work done = Energy out

$$h_1 + \frac{V_1^2}{2} + w = h_2 + \frac{V_2^2}{2}$$

$h_{01}$   
Stagnation  
enthalpy at  
inlet

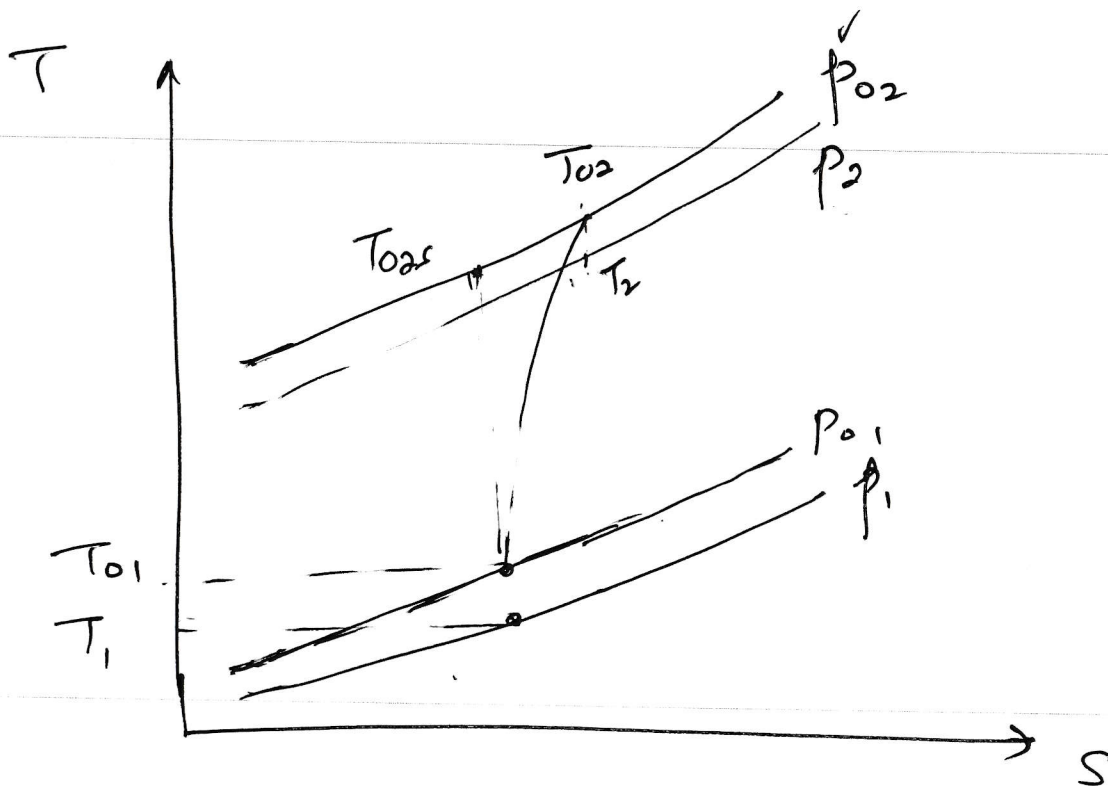
↑  
work done from  
Euler's equation.

$h_{02}$   
Stagnation enthalpy  
at exit

$$\begin{aligned} h_{01} + w &= h_{02} \Rightarrow w = h_{02} - h_{01} \\ &= c_p(T_{02} - T_{01}) \\ \Rightarrow T_{02} &= T_{01} + \frac{w}{c_p} \end{aligned}$$



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stagnation temp at inlet

$$h_{01} = h_1 + \frac{V_1^2}{2} \Rightarrow C_p T_{01} = C_p T_1 + \frac{V_1^2}{2} \Rightarrow T_{01} = T_1 + \frac{V_1^2}{2C_p}$$

must be J/kg K

$$\frac{P_{01}}{P_1} = \left( \frac{T_{01}}{T_1} \right)^{\frac{\gamma}{\gamma-1}}$$

$$T_{02} = T_{01} + \frac{w}{C_p}$$

$$\eta = \frac{T_{02s} - T_{01}}{T_{02} - T_{01}} \leftarrow \text{find } T_{02s}$$

$$\frac{P_{02}}{P_{01}} = \left( \frac{T_{02s}}{T_{01}} \right)^{\frac{\gamma}{\gamma-1}}$$

$$T_2 = T_{02} - \frac{V_2^2}{2C_p}$$

$$\frac{P_2}{P_{02}} = \left( \frac{T_2}{T_{02}} \right)^{\frac{\gamma}{\gamma-1}}$$