

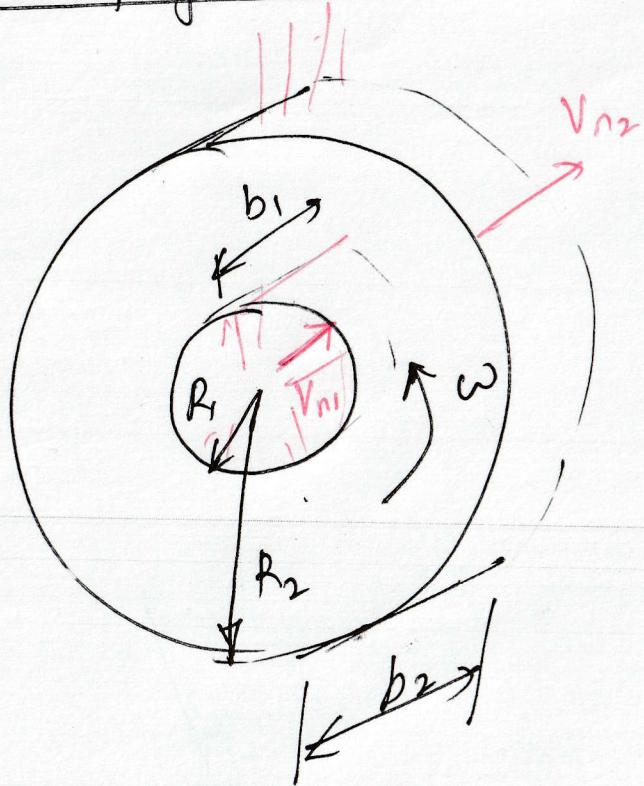
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# Turbomachines

1) Centrifugal

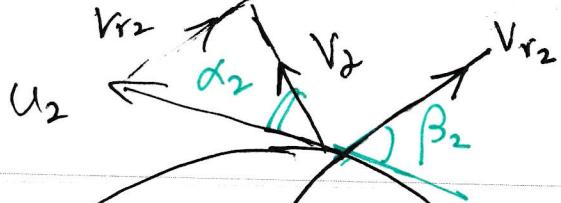
2) Axial

## 1) Centrifugal machines



$$\begin{aligned}\dot{V} &= AV_n, \\ &= 2\pi R_1 b_1 V_{n1}, \\ &= 2\pi R_2 b_2 V_{n2}\end{aligned}$$

$$\begin{aligned}\dot{m} &= \rho \dot{V} \\ &= \frac{\dot{V}}{U}\end{aligned}$$



$$\dot{H} = \omega R_1 \quad u_2 = \omega R_2$$

(2)

$\beta_1, \beta_2$  are the blade angles at inlet and outlet relative to the tangent of the circle at the blade

$V_{r1}, V_{r2}$  are the relative velocity vectors at the inlet and outlet

$V_i = \text{absolute velocity at inlet}$

$$= V_{r1} + u_1$$

$V_2 = \text{absolute velocity at exit}$

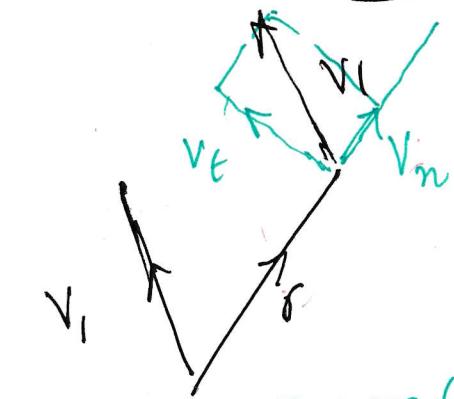
$$= V_{r2} + u_2$$

$\alpha_1, \alpha_2$  = angle made by inlet and outlet velocity vectors with the tangent to the circle at the inlet and outlet respectively.

Conservation of angular momentum (steady state)

$$\ddot{H}_i - \ddot{H}_o + \sum T = 0 \Rightarrow T = \ddot{H}_o - \ddot{H}_i = \dot{m} r_2 v_{t2} - \dot{m} r_1 v_{t1}$$

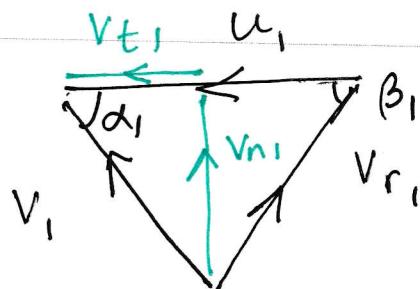
$$H_i = r \times m v = r m v_t$$



$$\begin{aligned} r \times m v &= r \times m (\underline{v_t} + \underline{v_n}) \\ &= m(r \times \underline{v_t}) + (r \times \underline{v_n})n \\ &= mrV_t \end{aligned}$$

(3)

Inlet



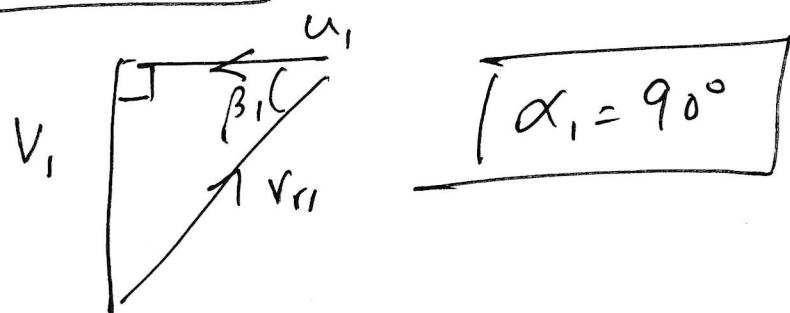
$$V_{n1} = V_1 \sin \alpha_1 = V_{r1} \sin \beta_1 \Rightarrow V_{r1} = \frac{V_{n1}}{\sin \beta_1}$$

$$\begin{aligned} V_{t1} &= u_1 - V_{r1} \cos \beta_1 \\ &= V_1 \cos \alpha_1 \end{aligned}$$

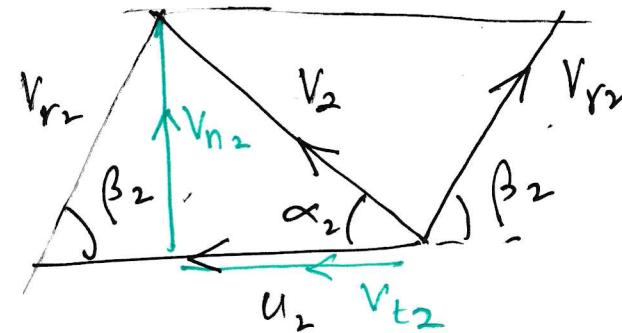
$$\rightarrow u_1 - \left( \frac{V_{r1}}{\sin \beta_1} \right) \cos \beta_1$$

$$V_{t1} = u_1 - V_{n1} \cot \beta_1$$

Special Case:  $V_{t1} = 0$  (Radial entry)



Outlet



$$V_{n2} = V_2 \sin \alpha_2 = V_{r2} \sin \beta_2 \Rightarrow V_{r2} = \frac{V_{n2}}{\sin \beta_2}$$

$$\begin{aligned} V_{t2} &= V_2 \cos \alpha_2 = u_2 - V_{r2} \cos \beta_2 \\ &= u_2 - \frac{V_{n2}}{\sin \beta_2} \cos \beta_2 \\ &= u_2 - V_{n2} \cot \beta_2 \end{aligned}$$

$$\begin{aligned} T &= \dot{m} r_2 V_{t2} - \dot{m} r_1 V_{t1} \\ &= \dot{m} [r_2 V_{t2} - r_1 V_{t1}] \end{aligned}$$

(4)

$$\begin{aligned} P &= T \omega \\ &= \dot{m} [\omega r_2 V_{t2} - \omega r_1 V_{t1}] \\ &= \dot{m} [u_2 V_{t2} - u_1 V_{t1}] \end{aligned}$$

$$w = \frac{P}{\dot{m}} = u_2 V_{t2} - u_1 V_{t1}$$

Work

← Euler's Turbomachinery equation

done per unit mass

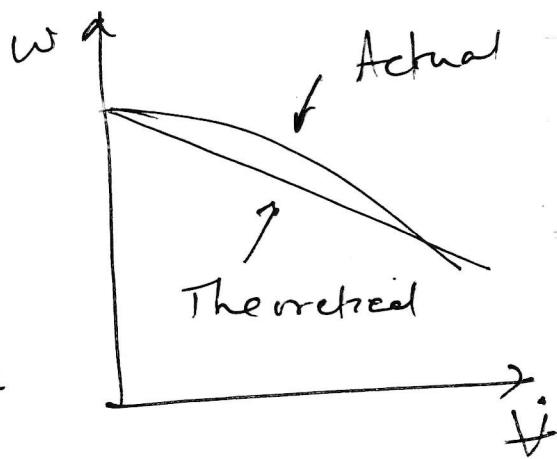
Special Case:  $V_{t1} = 0$

$$\begin{aligned} w &= u_2 V_{t2} \\ &= u_2 (u_2 - V_{n2} \cot \beta_2) \\ &= u_2^2 - u_2 V_{n2} \cot \beta_2 \end{aligned}$$

Since  $\dot{\tau} = 2\pi R_2 b_2 V_{n2} \Rightarrow V_{n2} = \frac{\dot{\tau}}{2\pi R_2 b_2}$

$$w = u_2^2 - u_2 \frac{\cot \beta_2}{2\pi R_2 b_2} \dot{\tau}$$

$$\frac{\dot{\tau}}{2\pi R_2 b_2}$$



(5)

## Thermodynamics

↳ Energy in + work done = Energy out

$$h_1 + \frac{V_1^2}{2} + w = h_2 + \frac{V_2^2}{2}$$

$\underbrace{\quad}_{h_{01}}$

Stagnation  
enthalpy at  
inlet

↑  
work done from  
Bernoulli's equation.

$$h_2 + \frac{V_2^2}{2}$$

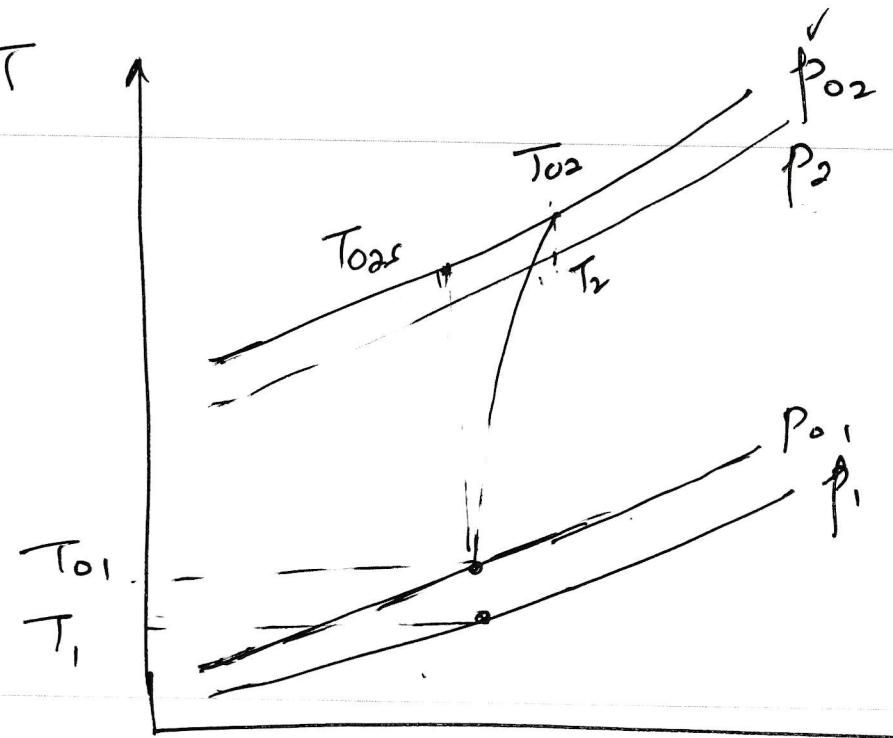
$\underbrace{\quad}_{h_{02}}$   
Stagnation enthalpy  
at exit

$$h_{01} + w = h_{02} \Rightarrow w = h_{02} - h_{01}$$

$$= Cp(T_{02} - T_{01})$$

$$\Rightarrow T_{02} = T_{01} + \frac{w}{Cp}$$

(6)



$$h_{01} = h_1 + \frac{V_1^2}{2} \Rightarrow C_p T_{01} = C_p T_1 + \frac{V_1^2}{2} \Rightarrow T_{01} = T_1 + \frac{V_1^2}{2C_p}$$

↑ stagnation temp at inlet

$$\frac{P_{01}}{P_1} = \left( \frac{T_{01}}{T_1} \right)^{\frac{r}{r-1}}$$

must be J/kg k

$$T_{02} = T_{01} + \frac{w}{C_p}$$

$$\eta = \frac{T_{02s} - T_{01}}{T_{02} - T_{01}} \leftarrow \text{Fund } T_{02s}$$

$$\frac{P_{02}}{P_{01}} = \left( \frac{T_{02s}}{T_{01}} \right)^{\frac{r}{r-1}}$$

$$T_2 = T_{02} - \frac{V_2^2}{2C_p}$$

$$\frac{P_2}{P_{02}} = \left( \frac{T_2}{T_{02}} \right)^{\frac{r}{r-1}}$$