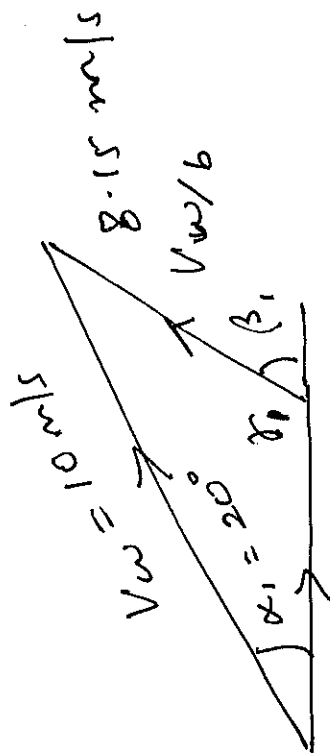
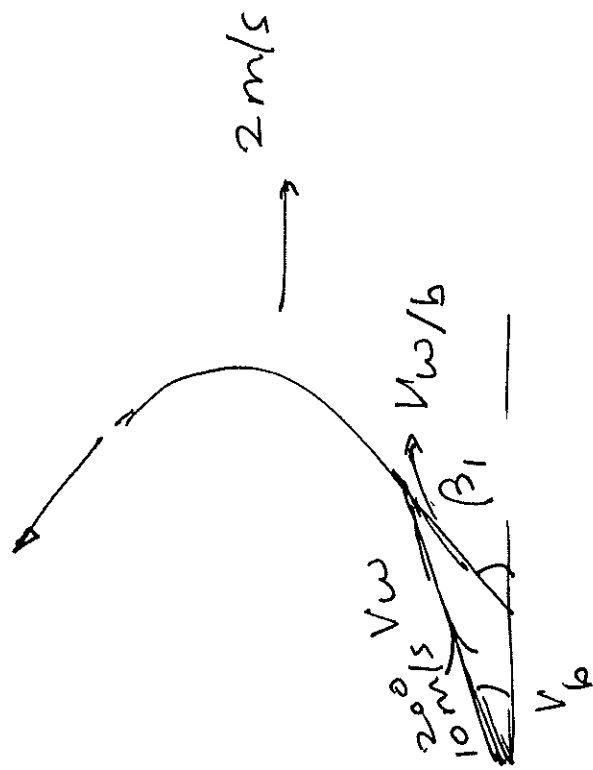


①



$$V_w = V_{w/b} + V_b$$



$$V_{w/b}^2 = V_b^2 + V_w^2 - 2V_b V_w \cos 20$$

$$= 2^2 + 10^2 - (2)(10)(2) \cos 20$$

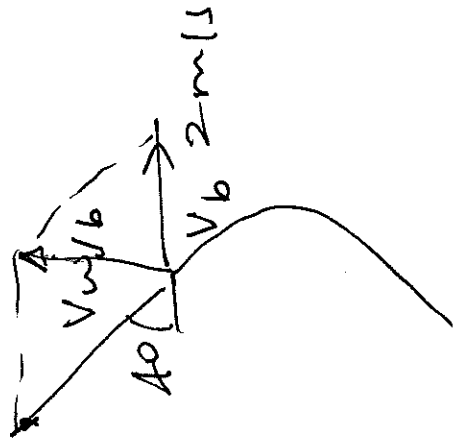
$$(6) \quad V_{w/b} = 8.15 \text{ m/s}$$

$$\cos \beta_1 = \frac{2^2 + 8.15^2 - 10^2}{(2)(8.15)(2)} = 155.1$$

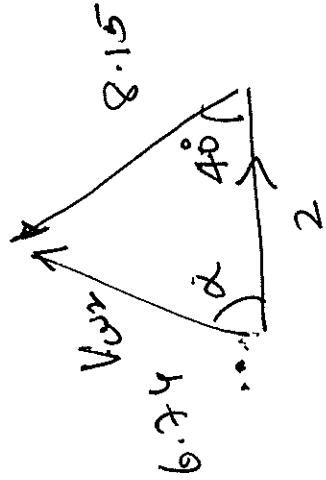
$$(5) \quad \beta_1 = 180 - 155 = 24.8^\circ$$

②

$$\begin{aligned}\vec{V}_{w/b} &= -V_{w/b} \cos 40^\circ \hat{i} + V_{w/b} \sin 40^\circ \hat{j} \\ &= -8.15 \cos 40^\circ \hat{i} + 8.15 \sin 40^\circ \hat{j} \\ &= -6.74 \hat{i} + 5.23 \hat{j}\end{aligned}$$



$$\begin{aligned}V_{w_2} &= V_{w_2/b} + V_b \\ V_{w_2} &= \sqrt{V_{w/b}^2 + V_b^2} = 2 V_{w/b} V_b \cos 40^\circ \\ &= \sqrt{8.15^2 + 2^2} = (2)(8.15) \cos 40^\circ \\ &= 6.74 \text{ m/s} \\ \alpha &= \cos^{-1} \frac{6.74^2 + 2^2 - 8.15^2}{(2)(6.74)(2)} = 129^\circ\end{aligned}$$



$$\alpha_2 = 180 - 129 = 51^\circ$$

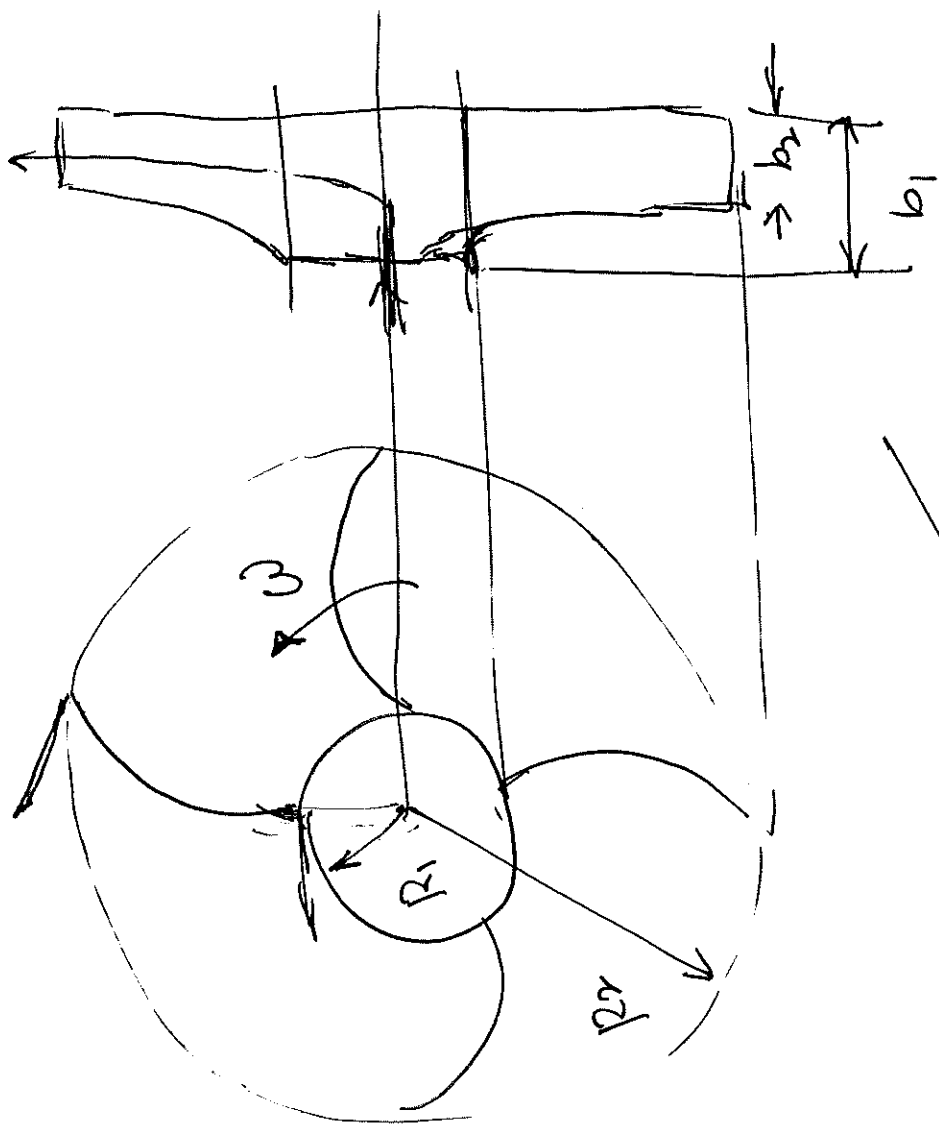
$$\dot{m} = \rho A V_r = (10^3) \frac{\pi}{4} (0.04)^2 \times 8.15 = 10.23 \text{ kg/s}$$

Horizontal: $\dot{m} V_{ix} - \dot{m} V_{out_x} - F_x = 0 \Rightarrow F_x = (10.23) [10 \cos 20 - 6.74 \cos 129] = 139.5 \text{ N}$

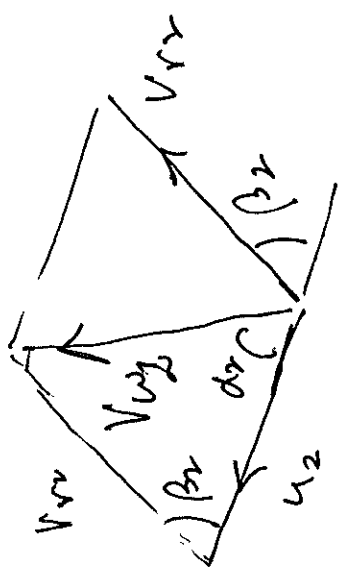
Vertical: $\dot{m} V_{iy} - \dot{m} V_{out_y} - F_y = 0 \Rightarrow F_y = (10.23) [10 \sin 20 - 6.74 \sin 129] = -18.6 \text{ N}$

$$u_1 = R_1 \omega$$

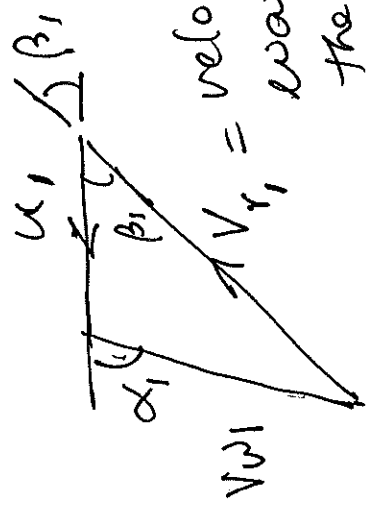
$$u_2 = R_2 \omega$$



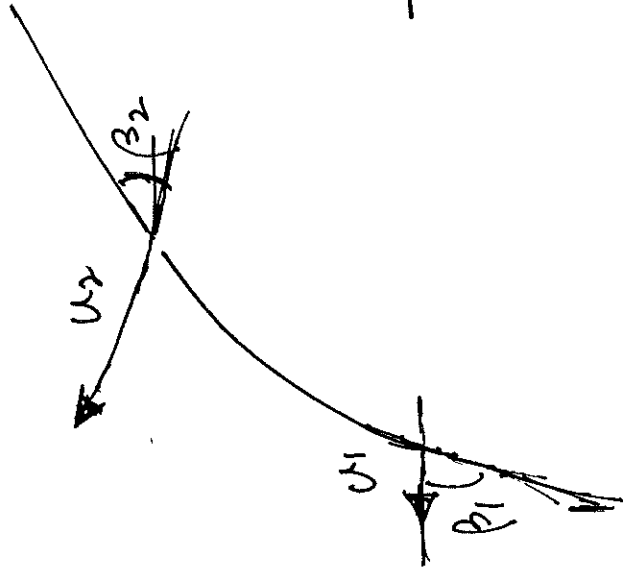
Outlet triangle



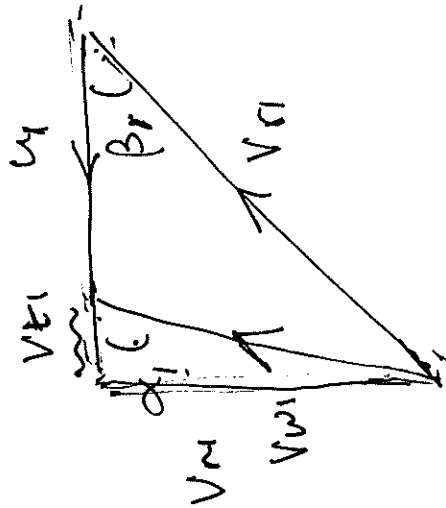
Inlet Triangle



V_{r1} = velocity of the jet
water relative to
the blade



Inlet

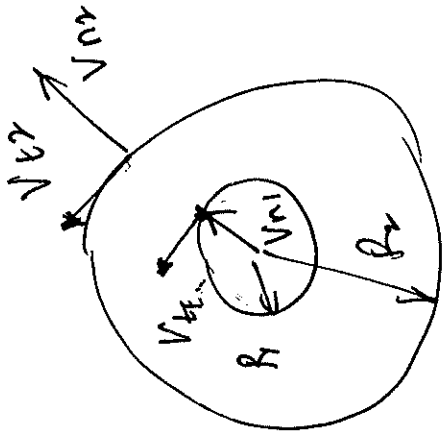


$V_{w1} = V_{t1}, V_{n1} = \text{tangential \& normal components of } V_{w1}$

of V_{w1}

$$V_{n1} = V_{w1} \sin \alpha_1 = V_{r1} \sin \beta_1$$

$$V_{t1} = V_{r1} \cos \beta_1 - u_1$$



$$V_{n2} = V_{r2} \sin \beta_2 = V_{w2} \sin \alpha_2$$

$$V_{t2} = u_2 - V_{r2} \cos \beta_2$$

$$AU = 2\pi R_1 b_1 V_{n1} = 2\pi R_2 b_2 V_{n2}$$

$$Q = AU = 2\pi R_1 b_1 V_{n1} = 2\pi R_2 b_2 V_{n2}$$

Conservation of Angular Momentum

$$\dot{H}_{in} - \dot{H}_{out} + \sum T = \frac{dH_{cv}}{dt}$$

$$\dot{H}_{cv} = \dot{m} r_1 V_{t1} - \dot{m} r_2 V_{t2} + T = 0$$

$$\boxed{T = \dot{m} (r_2 V_{t2} - r_1 V_{t1})}$$

(3)

(4)

Euler's
Turbo Machinery
Equations.

$$\begin{aligned}\text{Power } P &= T\omega \\ &= \dot{m} \omega (r_2 V_{t2} - r_1 V_{t1}) \\ &= \dot{m} [V_{t2} (r_2 \omega) - (r_1 \omega) V_{t1}]\end{aligned}$$

$$P = \dot{m} [V_{t2} u_2 - V_{t1} u_1]$$

$$\omega = \frac{P}{\dot{m}} \quad \left(V_{t2} u_2 - V_{t1} u_1 \right)$$

work done
per unit mass

$$h_{\text{pump}} = \frac{\omega}{g} = \frac{P}{\dot{m} g} = \frac{V_{t2} u_2 - V_{t1} u_1}{g}$$

← Pump
head.

Special Case: $\alpha_1 = 90^\circ \Rightarrow$ radial entry $\Rightarrow V_{t1} = 0$

$$h_{\text{pump}} = \frac{V_{t2} u_2}{g}$$

Assume Radial entry

$$h_{\text{pump}} = \frac{V_{t2} u_2}{g}$$

$$= \frac{u_2}{g} (u_2 - V_{r2} \cos \beta_2)$$

$$= \frac{u_2}{g} \left(u_2 - \frac{V_{r2}}{\sin \beta_2} \cos \beta_2 \right)$$

$$= \frac{u_2}{g} \left(u_2 - \frac{Q}{2\pi R_2 b_2} \frac{\cos \beta_2}{\sin \beta_2} \right)$$

$$= \frac{u_2^2}{g} - Q \left[\frac{u_2}{2\pi R_2 b_2} \frac{\cot \beta_2}{g} \right]$$

$$\text{Let } a = \frac{u_2^2}{g}$$

$$b = \frac{u_2}{2\pi R_2 b_2} \frac{\cot \beta_2}{g}$$

$$h_{\text{pump}} = a - bQ$$

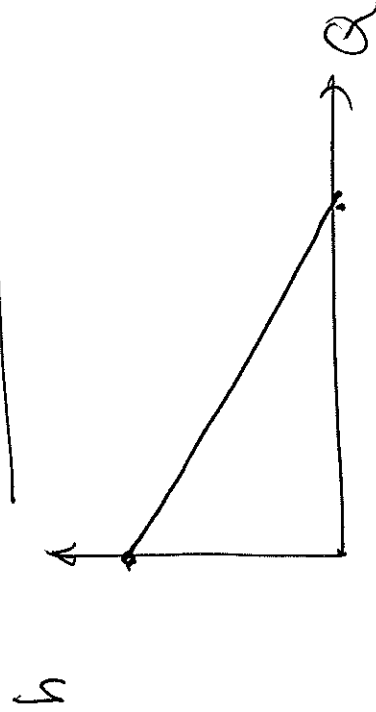
$$\text{When } Q=0 \quad h_{\text{pump}} = a = \frac{u_2^2}{g}$$

Shut-off head.

$$Q = \frac{a}{b}$$

$$h_{\text{pump}} = 0$$

Maximum flow-rate



6

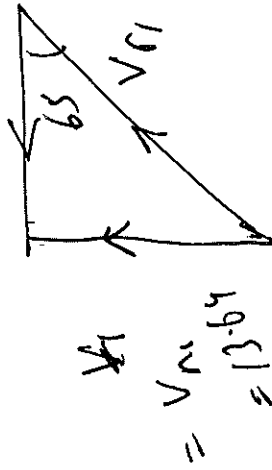
Parameter	Inlet	Outlet
Radius-(mm)	175	500
Blade width (b mm)	50	30
Blade angle β	65	70

$$\omega = (750) \frac{2\pi}{60} = 78.54 \text{ rad/s}$$

Assume Radial entry

Inlet triangle

$$u_1 = 13.74$$



$$V_{t1} = 13.64$$



$$V_{n1} = \frac{Q}{2\pi R_1 b_1} = \frac{0.75}{(2\pi)(0.175)(0.05)} = 13.64 \text{ m/s}$$

$$V_{r1} = \frac{V_{n1}}{\sin \beta_1} = \frac{13.64}{\sin 65^\circ} = 15.05 \text{ m/s}$$

$$V_{t1} = u_1 - V_{r1} \cos \beta_1 = 13.64 - 15.05 \cos 65^\circ = 7.28 \text{ m/s}$$

$$N = 750 \text{ rpm}$$

$$Q = 0.75 \text{ m}^3/\text{s}$$

Theoretical head?

$$P = ?$$

$$u_1 = (78.54)(0.175) = 13.74 \text{ m/s}$$

$$u_2 = (78.54)(0.5) = 39.27 \text{ m/s}$$

Outlet triangle

$$Q = 2\pi R_2 b V_{n2}$$

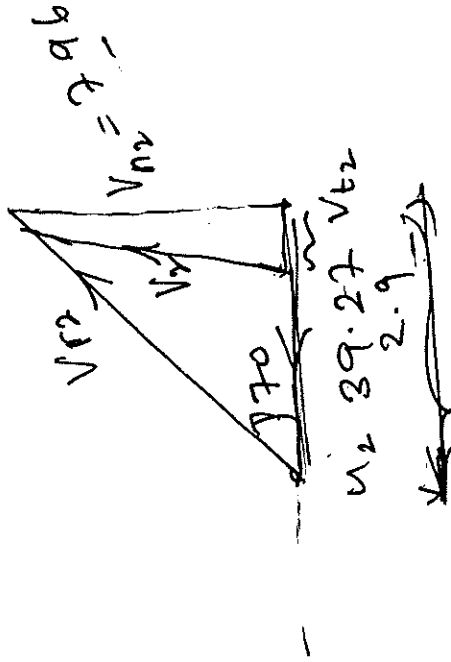
(7)

$$V_{n2} = \frac{Q}{2\pi R_2 b} = \frac{0.75}{(2\pi)(0.5)(0.03)}$$

$$= 7.96 \text{ m/s}$$

$$V_{r2} = u_2 - \underbrace{V_{t2} \cos \beta_2}_{39.27 - 2.9}$$

$$= 36.37 \text{ m/s}$$



$$V_{t2} = \frac{V_{n2}}{\tan \beta_2}$$

$$h_{\text{pump}} = \frac{V_{t2} u_2 - V_{t1} u_1}{g} = \frac{(36.37)(39.27) - (13.74)(7.28)}{9.81} = 135 \text{ m.}$$

$$\text{Power} = \dot{m} g h_{\text{pump}}$$

$$= \rho Q g h_{\text{pump}} = (10^3)(0.75)(9.81)(135) = 996 \text{ kW.}$$

$$\text{Torque} = \dot{m} (r_2 V_{t2} - r_1 V_{t1}) = (10^3)(0.75)[(0.5)(36.37) + (0.875)(7.28)] = 12.7 \text{ kN-m}$$