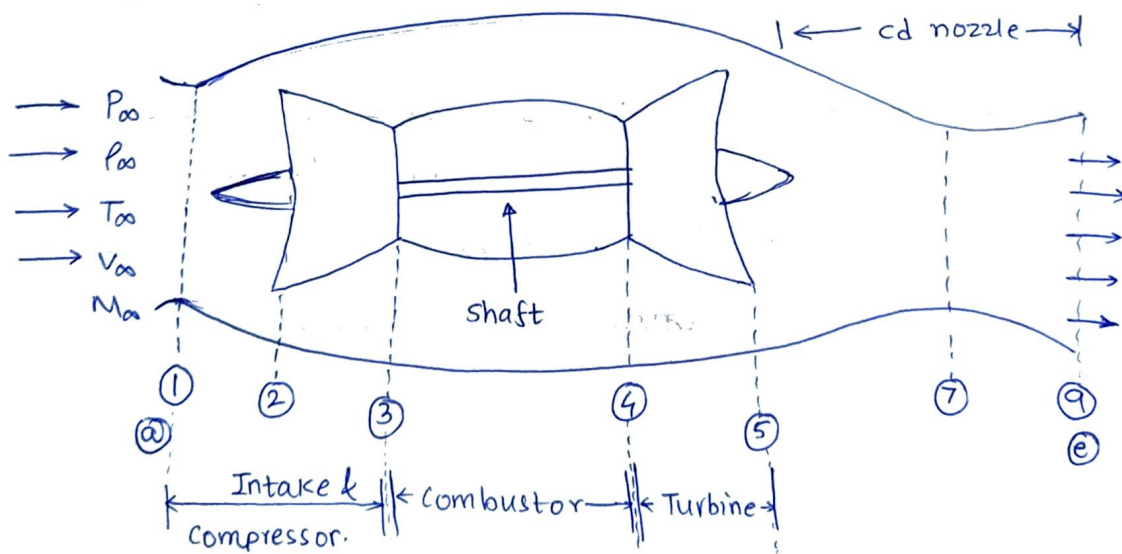


→ Given Data:

1.  $M_{cruise} = 0.85$
2.  $h_{cruise} = 41000 \text{ ft}$
3. Max Takeoff weight = 170600 kg
4. empty weight = 71700 kg.
5. Max Fuel carried at Takeoff = 75750 kg.
6. Fuel required for Takeoff = 0 kg
7. Fuel required for Landing = 0 kg
8. Weight of 1 F118 engine = 1450 kg.
9. Overall compression ratio =  $\frac{P_{stg, \text{ after compressor}}}{P_{\text{ambient}}} = 35.1$
10. Maximum Thrust produced = 85 kN.
11.  $\eta_{\text{compressor}} = \eta_t = 0.90$  and  $Q_R = 45 \text{ MJ/kg}$

Q1. Engine schematic. (Turbojet)



- ① @ Intake    ② Compressor entry    ③ Compressor exit & combustor entry
- ④ combustor exit    ⑤ Turbine entry    ⑥ Turbine exit
- ⑦ @ throat of cd nozzle.
- ⑧ nozzle exit ( $P_e, T_e, \rho_e, M_e, V_e$ ) etc.

here  $P_{amb} = T_{amb}$ ,  $P_{amb}$  will be calculated from the International Standard Atmosphere tables.

@ 41,000 ft  $\approx$  40,000 ft  $T_{amb} = -56.5^\circ\text{C} = 216.5\text{ K}$   
 [entry]  $P_{amb} = 18.8\text{ kPa}$

$\therefore P_{amb} = 0.2462 \times 1.225 = 0.301595\text{ kg/m}^3$   
 $T = \frac{P_{amb}}{\rho_{SL}} = 0.2462$

Now as we don't have losses in the diffuser we have

$P_{stg,2} = P_{stg,amb} = 30.1523\text{ kPa} = P_{t2}$  (1)

$T_{stg,2} = T_{stg,amb} = 247.796\text{ K} = T_{t2}$  (2)

Now we use Isentropic gas tables. @  $M = 0.85$

$\frac{T}{T_0} = 0.8737 \Rightarrow T_{0,amb} = 247.796\text{ K}$

$\frac{P}{P_0} = 0.6235 \Rightarrow P_{0,amb} = 30.1523\text{ kPa}$

Next we have non-isentropic compression with  $\pi_c = 35.1$  and

$\eta_{compressor} = 0.9$  But  $\pi_c = \frac{P_{t3}}{P_{amb}} = 35.1 \Rightarrow \pi_{c,actual} = \frac{P_{t3}}{P_{t2}} = 21.88$

$\therefore \pi_c = \frac{V_{amb}}{V_3} = 35.1$  hence we use the relation,

$\frac{T_{t3}}{T_{t2}} = 1 + \frac{1}{\eta_c} \left[ \pi_c^{\frac{\gamma-1}{\gamma}} - 1 \right] = 1 + \frac{1}{0.9} \left[ \frac{35.1}{21.88}^{\frac{1.4-1}{1.4}} - 1 \right]$   
 $= \frac{2.9598}{2.5719} \Rightarrow T_{t3} = 733.4381\text{ K}$  (3)  $T_{t3} = 637.315$

Now we do apply ideal gas equation.

$\frac{P_{t2} V_2}{T_{t2}} = \frac{P_{t3} V_3}{T_{t3}}$  But  $\pi_c = 35.1 = \frac{V_2}{V_3} = \frac{V_{amb}}{V_3}$

$\therefore P_{t3} = \frac{P_{t2} \cdot T_{t3}}{T_{t2}} \cdot 35.1 = [30.1523\text{ kPa}] \times \left[ \frac{733.4381}{247.796} \right] \times 35.1$

$\therefore P_{t3} = 3132.5408\text{ kPa}$  (4)  $P_{t3} = P_{amb} \times 35.1 = 659.88\text{ kPa}$

Next, we have the combustor stage where we have,

$\dot{q}_{burner} = \dot{m}_a C_p (T_{t4} - T_{t3}) = \dot{m}_a \times \frac{\gamma R}{\gamma - 1} (T_{t4} - T_{t3}) = \dot{m}_f Q_R$

at maximum conditions we have Thrust<sub>max</sub> = 85 kN. >>>

$$\text{Thrust} = \dot{m}_a((1+f)v_e - v_a) = 85 \text{ kN} \quad \dots f = \frac{\dot{m}_f}{\dot{m}_a}$$

also we have  $\dot{m}_f Q_R = \dot{m}_a C_p (T_{t4} - T_{t3})$

$$\Rightarrow f = \frac{C_p (T_{t4} - T_{t3})}{Q_R} \quad \text{also as it is not specified we can take perfect expansion of nozzle} \therefore [\pi_t = \pi_c]$$

$$\therefore \text{Thrust} = \dot{m}_a((1+f)M_e \sqrt{\gamma R T_e} - M_a \sqrt{\gamma R T_a})$$

We know, that  $T_{t4} = \text{maximum allowable temperature} = 1400 \text{ K}$  (5)

$\therefore$  Now we have.  $f = \frac{\gamma R}{\gamma - 1} \times \frac{1400 - \cancel{733.4381}}{45 \times 10^6}$

$$\therefore f = \frac{0.01703}{\cancel{0.01488}} \approx 0.017 \quad (6) \quad f = 0.017$$

Now, after that, we have non isentropic expansion, &  $\pi_t = \pi_c$

$$\therefore \frac{T_{t5}}{T_{t4}} = 1 - \eta_t \left[ 1 - \pi_t^{\frac{1-\gamma}{\gamma}} \right] = \frac{0.4256}{0.4727} \quad \therefore T_{t5} = \frac{\cancel{595.88}}{661.794} \text{ K} \quad (7)$$

We must remember that we have isobaric combustion in the combustor. Hence  $P_{t4} = P_{t3} = \frac{659.88}{\cancel{3132.5408}} \text{ kPa}$  (8)

Now, applying ideal gas law in stages (4) & (5)

$$\frac{P_{t4} V_4}{T_{t4}} = \frac{P_{t5} V_5}{T_{t5}} \quad \text{But } \pi_t = \frac{35.1}{21.88} = \frac{V_5}{V_4} = \frac{P_{t4}}{P_{t5}}$$

$$\therefore P_{t5} = \frac{P_{t4}}{\frac{V_5}{V_4}} \times \frac{1}{35.1} \times T_{t5} = \frac{37.985}{30.159} \text{ kPa} \quad (9)$$

Now as as we don't have any losses in the nozzle and we have perfect expansion, we have isentropic process from stage (5) Turbine exit to stage (9) nozzle exit.

$$\therefore T_{t9} = T_{t5} = \frac{661.794}{\cancel{595.88}} \text{ K} \quad (10) \quad \text{and} \quad M_e = 0.85 \quad (12)$$

$$P_{t9} = P_{t5} = \frac{37.985}{30.159} \text{ kPa} \quad (11) \quad \therefore T_e = \frac{520.646}{578.238} \text{ K} \quad (13)$$

$$\therefore V_e = M_e \sqrt{\gamma R T_e} = \frac{\cancel{388.84}}{409.782} \text{ m/s} \quad (14)$$

$$V_a = M_a \sqrt{\gamma R T_a} = 250.74 \text{ m/s} \quad (15)$$



∴ we had expression as

$$\text{Thrust max} = \dot{m}_a [(1+f) V_e - V_a]$$

$$85 \times 10^3 = \dot{m}_a \left( (1 + \frac{0.017}{0.015}) V_e - V_a \right)$$

$$85 \times 10^3 = \dot{m}_a \left( \left( \frac{1.017}{0.015} \times \frac{409.782}{388.84} \right) - 250.74 \right)$$

$$85 \times 10^3 = \dot{m}_a \times \frac{166.008}{143.9326} \Rightarrow \dot{m}_a = \frac{540.55}{512.048} \text{ kg/s} \quad (16)$$

$$\text{also } \dot{m}_f = f \cdot \dot{m}_a = \frac{0.017}{0.015} \times \frac{540.55}{512.048} \text{ kg/s} = \frac{8.7048}{8.858} \text{ kg/s} \quad (17)$$

$$\dot{Q}_{\text{burner}} = \dot{m}_f Q_R = \frac{398.62}{391.716} \times 10^6 \frac{\text{kJ}}{\text{s}} \quad (18)$$

Now, just for extra, we had Mass fuel = 75750 kg

$$\& \dot{m}_f = 8.588 \text{ kg/s} \quad \therefore \text{General} = \frac{75750}{8.588} = 8820.44 \text{ sec}$$

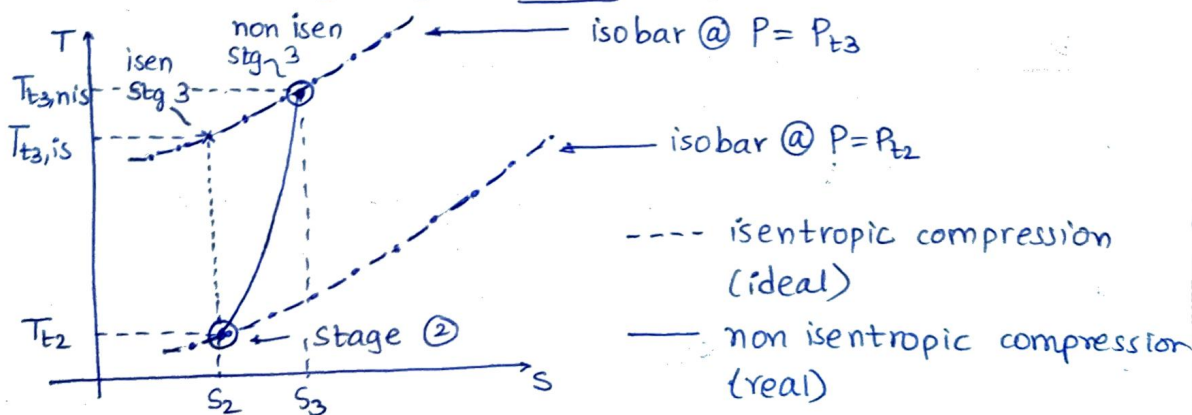
$$\text{Endurance} = \frac{8.7048}{8.7048} = 8702.095$$

$$(\text{for engine}) = \frac{147.007}{145.034} \text{ minutes}$$

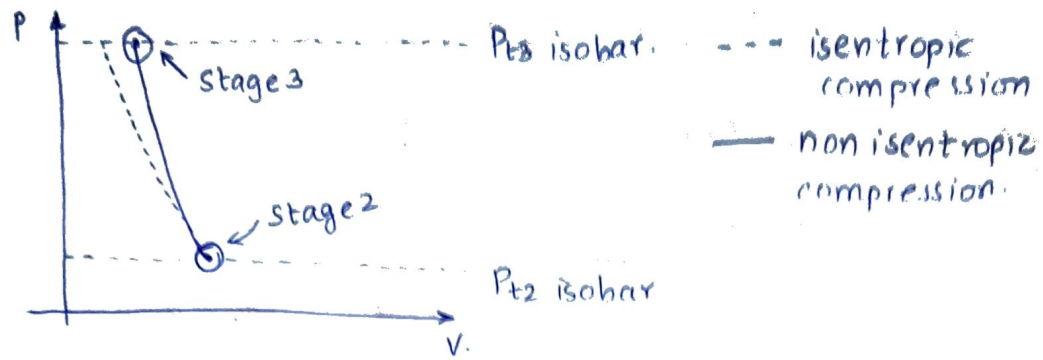
Q2 We basically have the following processes:

- ① - ② i.e ambient to compressor entry : isentropic
- ② - ③ i.e compressor entry to compressor exit  
: non-isentropic compression ( $\because \eta_c \neq 1$ )
- ③ - ④ i.e combustor entry to combustor exit  
: isobaric combustion ('Brayton's cycle')
- ④ - ⑤ i.e combustor exit or turbine entry to turbine exit  
: non isentropic expansion ( $\because \eta_t \neq 1$ )
- ⑤ - ⑥ isentropic flow (perfectly expanded nozzle).

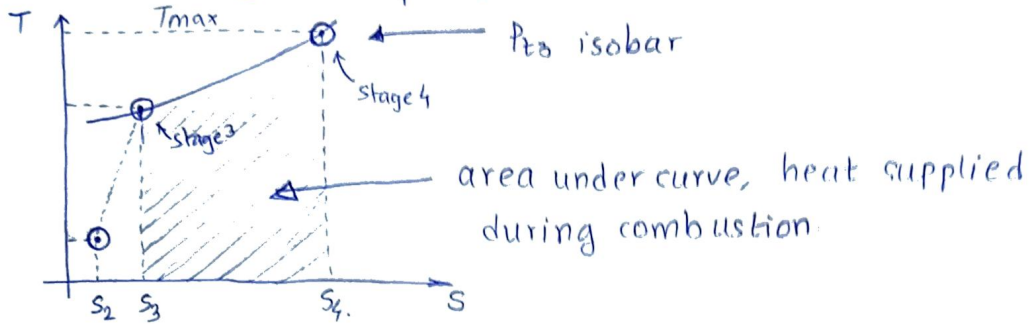
Now. Process ② → ③. PLOTS. T-s.



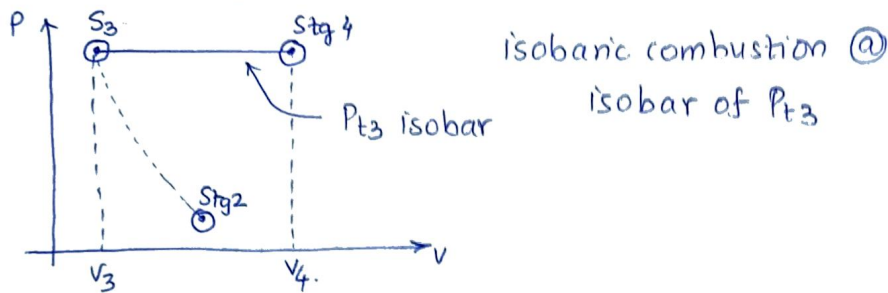
## P-V plots - for (2) - (3)



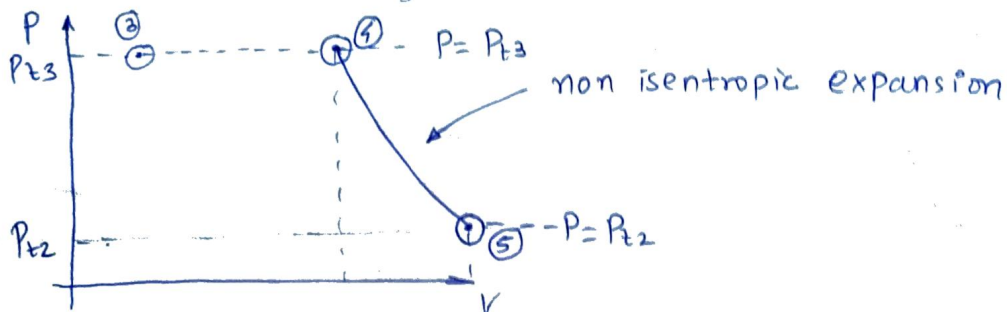
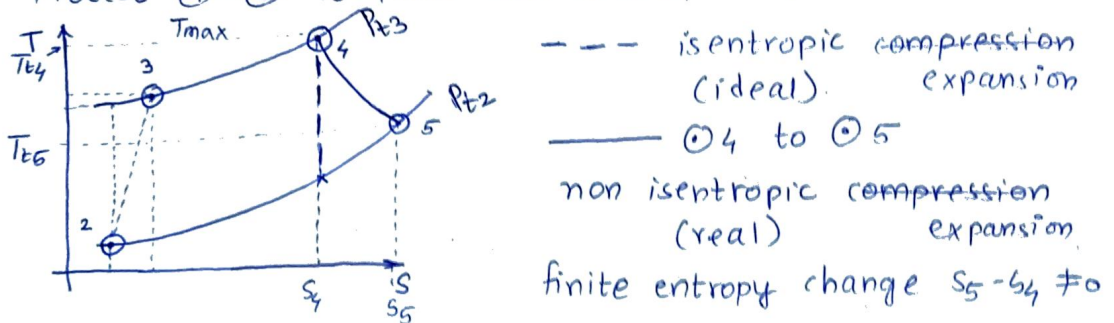
## Process (3) - (4) Ts plots. Isobaric combustion.



## Process (3) - (4) PV plots.



## Process (4) - (5) Ts plots non-isentropic expansion in turbine



expressions for these are same as stated above  
for process ② → ③

$$\frac{T_{t3}}{T_{t2}} = 1 + \frac{1}{\eta_c} \left[ \pi_c^{\frac{\gamma-1}{\gamma}} - 1 \right] \quad \text{and units are 'K'}$$

$$\frac{P_{t3}}{P_{t2}} = \left( \frac{T_{t3}}{T_{t2}} \right)^{\frac{\gamma}{\gamma-1}} \pi_c \quad \dots \text{units kPa} \quad \dots (\text{just ideal gas eqn})$$

for process ③ → ④

$$P_{t3} = P_{t4} \quad \dots \text{units kPa} \quad (\text{isobaric}).$$

$$\dot{q}_{\text{burner}} = \dot{m}_a c_p (T_{t4} - T_{t3}) \quad (\text{isobaric combustion})$$

units MJ/s.

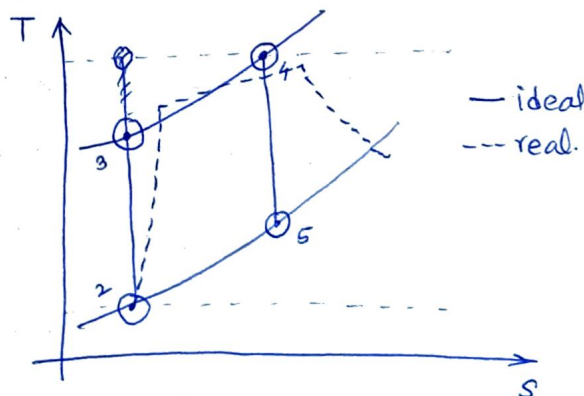
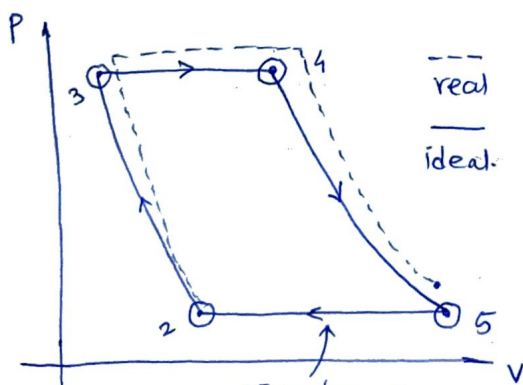
for process ④ → ⑤

$$\frac{T_{t5}}{T_{t4}} = 1 - \eta_t \left[ 1 - \pi_t^{\frac{\gamma-1}{\gamma}} \right] \quad \dots \text{units 'K'}$$

$$\frac{P_{t5}}{P_{t4}} = \frac{T_{t5}}{T_{t4}} \cdot \frac{1}{\pi_t} \quad \dots \text{units kPa}$$

(ideal gas law).

In general we see that for  $\eta_c, \eta_t \neq 1$  to achieve the same compressor ratio in compressor,  $T_{t3}$  &  $P_{t3}$  are higher than their normal isentropic values. In nozzle expansion the resulting  $P_t$  &  $T_t$  are again higher, if it were isentropic expansion then the  $T_{t, \text{amb}}$ ,  $P_{t, \text{amb}}$  &  $P_{t5}$  must be same as the ideal Brayton cycle proposes that. But since we have losses. Brayton cycle (~turbojet cycle)





Q3. TSFC

Now we had  $f = \frac{\dot{m}_f}{\dot{m}_a} = 0.015$  ~~0.017~~

also we have that  $TSFC = \frac{\dot{m}_f}{\tau} = \frac{\dot{m}_f}{\dot{m}_a((1+f)V_e - V_a)}$

$$\begin{aligned} \therefore TSFC &= \frac{f}{(1+f)V_e - V_a} = \frac{0.015}{(1.015 \times 388.84) - (250.74)} \\ &= \frac{0.015}{1.017 \times 409.782 - (250.74)} \\ &= \frac{0.015}{1.024 \times 10^{-4}} \frac{s}{m} = 1.4215 \times 10^{-4} \frac{kg}{Ns} \\ &= \frac{1.4215 \times 10^{-4} \times 10^3 \times 10^{-4}}{1.024} \frac{gm}{kNs} = \frac{1.4215}{102.4} \frac{gm}{kN-s} \end{aligned}$$

$$\begin{aligned} \eta_{thermal} &= \eta_{Brayton\ cycle} = \frac{Work_{net}}{heat\ added} \\ &= 1 - \left( \frac{T_{t5} - T_{t2}}{T_{t4} - T_{t3}} \right) = 1 - \left( \frac{661.794}{595.88 - 247.796} \right) \\ &= 1 - \left( \frac{348.084}{666.5619} \right) = \frac{0.47779}{0.45718} \approx \boxed{0.48} = \eta_{Th} \end{aligned}$$

$$\begin{aligned} \eta_{propulsive} &= \frac{2V_a}{V_a + V_e} = \frac{2 \times 250.74}{250.74 + 388.84} = \frac{501.48}{639.58} \\ \eta_{propulsive} &= \boxed{0.784} \quad 0.75922 \end{aligned}$$

$$\eta_{overall} = \eta_{propulsive} \times \eta_{Thermal} = 0.784 \times 0.48 = 0.376$$

$$0.759 \times 0.457 = 0.3468$$

$$\therefore \boxed{\eta_{overall} = 0.3468}$$

Q4. Intake  $d = 1.2\ m \Rightarrow A_{intake} = 1.13097\ m^2$

Thrust@ cruise =  $\dot{m}_a((1+f)V_e - V_a)$

But  $\dot{m}_a = \rho_\infty A_\infty V_\infty = 0.301595 \frac{kg}{m^3} \times 1.13097\ m^2 \times 350.74$

$$= 85.526\ kg/s$$

$\therefore Thrust@cruise = 85.526\ kg/s \times ((1.017 \times 409.782) - 250.74)$

1 engine = 14.198 kN. all four engines = 56.792 kN

Q5. Range for constant attitude and speed

$$R = \frac{V_a}{g_0 TSFC} \cdot \frac{L}{D} \cdot \log \frac{W_{ini}}{W_{fin}}$$

We have  $W_{ini} = 170600 \text{ kg}$

$W_{fuel} = 75750 \text{ kg}$  assuming  $M_{fuel, T0} = 0$

and all of the fuel is consumed during  $M_{fuel, land} = 0$

the flight we have,  $W_{final} = 170600 - 75750$

$$= 94850 \text{ kg}$$

$$\therefore \text{Range} = \frac{250.74}{9.81 \times 1.024 \times 10^{-4}} \times 20 \times \log \left( \frac{170600}{94850} \right)$$

$$= \boxed{2930.496 \text{ km}}$$

Q6. Now after the combustor we had properties,

$$P_{t2} = 30.1523 \text{ kPa}$$

$$P_{t3} = \pi_c \cdot P_{amb} = 659.88$$

$$T_{t2} = 247.796 \text{ K}$$

$$T_{t3} = 637.315 \text{ K}$$

$$P_{t4} = 659.88 \text{ kPa}$$

$$P_{t5} = 30.159 \text{ kPa}$$

$$T_{t4} = 1400 \text{ K}$$

$$T_{t5} = 661.794 \text{ K}$$

after combustor we have stage ④ & ⑤

Now we have overall compression  $= 35.1 = \frac{P_{t3}}{P_{amb}}$

$$\text{but } \pi_c = \frac{P_{t3}}{P_{t2}} = \frac{P_{amb} \times 35.1}{P_{amb}} = \frac{659.88}{30.1523} = 21.88$$

but we have perfect expansion hence  $\pi_t = 21.88$

now if we have  $\gamma = 1.33$  we have

$$\frac{T_{t5}}{T_{t4}} = 1 - \eta_t \left[ 1 - \pi_t^{\frac{1-\gamma}{\gamma}} \right] = 0.5185 \Rightarrow \boxed{T_{t5} = 725.972 \text{ K}}$$

$$\text{also we have } \pi_t = 21.88 = \frac{P_{t4}}{P_{t5}} \Rightarrow \boxed{P_{t5} = 30.159 \text{ K}}$$

But as we have no losses in the nozzle we have



$$T_{t9} = T_{t5} = 725.972 \quad P_{t9} = P_{t5} = 30.159 \text{ kPa}$$

no loss in stagnation pressure as there is no heat addition, isentropic flow and no losses.

$$\therefore M_{\text{exit}} = 0.85 \Rightarrow T_{\text{exit}} = 648.645 \text{ K}$$

$$P_{\text{exit}} = 18.80 \text{ kPa.}$$

$$\therefore a_{\text{exit}} = \sqrt{\gamma R T_{\text{exit}}} = 497.675$$

$$V_{\text{exit}} = M_{\text{exit}} a_{\text{exit}} = 423.023 \text{ m/s}$$

$$V_a = 250.74 \text{ m/s.}$$

$$\text{we have } f = \frac{C_p (T_{t4} - T_{t3})}{Q_R} = 0.017$$

$$\text{TSFC} = \frac{f}{(1+f)V_e - V_a} = \frac{0.017}{(1.017 \times 423.023) - 250.74}$$

$$= \frac{0.017}{179.474} = 9.472 \times 10^{-5} = \cancel{0.947 \times 10^{-4}} = 94.72 \times 10^{-4} \text{ (TSFC decreased)}$$

(earlier it was around  $102.4 \times 10^{-4}$ )

$$\eta_{\text{thermal}} = 1 - \left( \frac{T_{t5} - T_{t2}}{T_{t4} - T_{t3}} \right) = 1 - \left( \frac{725.972 - 247.796}{1400 - 637.315} \right)$$

$$= 0.373 \text{ (decreased) (earlier } \eta_{th} = 0.457)$$

$$\eta_{\text{propulsive}} = \frac{2V_a}{V_e + V_a} = \frac{2 \times 250.74}{423.023 + 250.74} = 0.7442$$

(decreased) (earlier  $\eta_{\text{propulsive}} = 0.75922$ )

$$\eta_{\text{overall}} = 0.7442 \times 0.373 = 0.2775$$

(decreased) (@  $\gamma = 1.4$   $\eta_{\text{overall}} = 0.3468$ )

Q7. Engine Improvement.

$$\text{Overall compression ratio} = \frac{P_{t3}}{P_{\text{amb}}} = 45$$

$$P_{t3} = P_{\text{amb}} \times 45 = 18.8 \text{ kPa} \times 45 = 846 \text{ kPa}$$

$$\text{actual compression ratio} = \frac{P_{t3}}{P_{t2}} = \pi_c = \frac{846 \text{ kPa}}{30.1523 \text{ kPa}}$$

$$\therefore \pi_c = 28.057$$

Hence what we have now is,

$$T_{t2} = 247.296 \text{ K} \quad P_{t2} = 30.1523 \text{ kPa}$$

$$\boxed{P_{t3} = 846 \text{ kPa}} \quad (1)$$

$$\therefore \frac{T_{t3}}{T_{t2}} = 1 + \frac{1}{\gamma_c} [\pi_c^{\frac{\gamma-1}{\gamma}} - 1] \quad \left[ \text{here } \gamma = 1.33 \text{ after combustor} \right]$$

hence use  $\gamma = 1.4$  here]

$$\therefore \frac{T_{t3}}{T_{t2}} \Rightarrow \boxed{T_{t3} = 686.269 \text{ K}} \quad (2)$$

$$\text{Now, } \boxed{T_{t4} = 1400 \text{ K}} \quad \therefore f = \frac{c_p(T_{t4} - T_{t3})}{\dot{Q}_R} = 0.01593$$

(here too, as process before combustor using  $\gamma = 1.4$ )  $\approx 0.016$ .

Now as perfect expansion,  $\pi_t = \pi_c = 28.057$

$$\therefore \pi_t = \frac{P_{t4}}{P_{t5}} = 28.057 \quad \text{But combustion is isobaric}$$

$$\text{hence } \boxed{P_{t4} = 846 \text{ kPa}}$$

$$\therefore \boxed{P_{t5} = 30.1529 \text{ kPa}} \quad \text{and as we have that, in turbine}$$

$$\frac{T_{t5}}{T_{t4}} = 1 - \eta_t \left( 1 - \pi_t^{\frac{1-\gamma}{\gamma}} \right) \quad \text{now use } \gamma = 1.33 \text{ as after the combustor process.}$$

$$\therefore \frac{T_{t5}}{T_{t4}} = 0.4935 \quad \boxed{T_{t5} = 690.9118 \text{ K}}$$

$$\therefore T_{t9} = T_{t5} = 690.9118 \quad \left. \vphantom{T_{t9}} \right\} \text{(no losses in the nozzle)}$$

$$P_{t9} = P_{t5} = 30.1529 \text{ kPa}$$

and for perfect expansion  $Me = 0.85$

$$\therefore T_{\text{exit}} \text{ or } T_9 = \frac{T_{\text{stg},9}}{\left( 1 + \frac{\gamma-1}{2} M^2 \right)} = \frac{690.9118 \text{ K}}{\left( 1 + \left( \frac{1.33-1}{2} \right) (0.85)^2 \right)}$$

$$\therefore \boxed{T_{\text{exit}} = 617.319 \text{ K}}$$

$$V_{\text{exit}} = Me \sqrt{\gamma R T_e} = 412.682 \text{ m/s.}$$

We have Thrust =  $\dot{m}_a((1+f)v_e - v_a)$

$$\begin{aligned}\text{Thrust} &= \dot{m}_a(1+0.017) 412.682 - 250.74) \\ &= \dot{m}_a(168.957)\end{aligned}$$

Hence Thrust per unit  
mass flow  
rate =  $168.957 \frac{\text{N}}{\text{kg/s}}$

$$\text{also TSFC} = \frac{\dot{m}_f}{\tau} = \frac{f}{(1+f)v_e - v_a} = \frac{0.017}{168.957}$$

$$\therefore \text{updated TSFC} = 1.00617 \times 10^{-4} \frac{\text{kg}}{\text{Ns}}$$