

# Assignment #3.

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Let the Prandtl's stress function be.

$$\phi = c \left( \frac{x^2}{a^2} + \frac{y^2}{b^2} - 1 \right) \quad \text{--- (1)}$$

since ellipse of eq. is

$$\frac{x^2}{a^2} + \frac{y^2}{b^2} = 1.$$

so B.C i.e.

$\phi = 0$  ; at boundary is already satisfied

and C is const.

Since we know that

Compatibility  $\leftarrow \nabla^2 \phi = F = -2G \frac{d\theta}{dz}$   
eq.

$$\Rightarrow \frac{\partial^2 \phi}{\partial x^2} + \frac{\partial^2 \phi}{\partial y^2} = -2G \frac{d\theta}{dz}$$

$$\Rightarrow 2c \left( \frac{1}{a^2} + \frac{1}{b^2} \right) = -2G \frac{d\theta}{dz}$$

$$\Rightarrow c = -G \frac{d\theta}{dz} \frac{a^2 b^2}{(a^2 + b^2)}$$

putting it back in eq. (1)

$$\phi = -G \frac{d\theta}{dz} \frac{a^2 b^2}{(a^2 + b^2)} \left( \frac{x^2}{a^2} + \frac{y^2}{b^2} - 1 \right) \quad \text{--- (2)}$$

Now we have  $\phi$ . so

$$\tau = 2 \int \phi \, dx \, dy$$

$$\Rightarrow \tau = -2G \frac{d\theta}{dz} \frac{a^2 b^2}{(a^2 + b^2)} \left( \frac{1}{a^2} \iint x^2 \, dx \, dy + \frac{1}{b^2} \iint y^2 \, dx \, dy - \iint dx \, dy \right)$$

Here first integral =  $\frac{\pi a^3 b}{4} = I_{yy}$

Second integral =  $\frac{\pi a b^3}{4} = I_{xx}$

third integral  $A = \pi ab$

Finally  $T$  becomes

$$T = G \frac{d\theta}{dz} \frac{\pi a^3 b^3}{(a^2 + b^2)} \quad - (3)$$

Now we know that

$$T_{yz} = -\frac{\partial \phi}{\partial x} ; \quad T_{xz} = \frac{\partial \phi}{\partial y}$$

Using eq. (3) putting  $G \frac{d\theta}{dz}$  in eq. (2) &

then diff. according to above relation

$$T_{yz} = + T \frac{(a^2 + b^2)}{\pi a^3 b^3} \times \frac{a^2 b^2}{(a^2 + b^2)} \times \frac{2x}{a^2}$$

$$\Rightarrow T_{yz} = \frac{2T_x}{\pi a^3 b}$$

Similarly we can solve for  $T_{xz}$

$$\Rightarrow T_{xz} = -\frac{2T_y}{\pi a b^3}$$

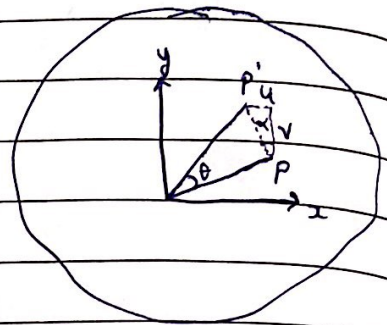
Now for warping displacement. since we know

$$\gamma_{xz} = \frac{T_{xz}}{G} = -\frac{2T_y}{G \pi a b^3} \frac{\partial w}{\partial x} + \frac{\partial u}{\partial z}$$

$$\text{and } \gamma_{yz} = \frac{T_{yz}}{G} = \frac{2T_x}{G \pi a^3 b} \frac{\partial w}{\partial y} + \frac{\partial v}{\partial z}$$



Here  $u = -y \theta \sin \alpha$   
 $= -\theta y$   
 $v = \theta x$



So.

$$\frac{\partial w}{\partial y} = \frac{T_{xz}}{G} - \frac{\partial u}{\partial z}$$

$$\& \frac{\partial w}{\partial x} = \frac{T_{yz}}{G} - \frac{\partial v}{\partial z}$$

$$\rightarrow \frac{\partial w}{\partial x} = -\frac{2Ty}{\pi a b^3 G} + y \frac{d\theta}{dz} \quad \text{--- (I)}$$

$$\& \frac{\partial w}{\partial y} = \frac{2Tx}{\pi a^3 b G} - x \frac{d\theta}{dz} \quad \text{--- (II)}$$

Using value of  $\frac{d\theta}{dz} = \frac{T}{G} \frac{(a^2 + b^2)}{\pi a^3 b^3}$  from previous.

and integrating both (I) & (II) we get.

$$w = \frac{T(b^2 - a^2)}{\pi a^3 b^3 G} xy + f_1(y)$$

$$\& w = \frac{T(b^2 - a^2)}{\pi a^3 b^3 G} + f_2(x)$$

Since both w are same so

$f_1(y)$  &  $f_2(x)$  should be zero so we get

$$w = \frac{T(b^2 - a^2)}{\pi a^3 b^3 G} xy$$

Now,

Case 1, So  $a = b$  & let  $a^4 = b^4 = C$ .

$$\text{So } T_{yz} = \frac{2Tx}{\pi C}$$

$$\& T_{xz} = -\frac{2Ty}{\pi C}$$

Case 2.  $\frac{a}{b} = 46 \Rightarrow a = 46b$ .

And taking  $b = 1$ .

We know that for  $a = b = 1$ , the eq. will be.

~~Ans~~

$$x^2 + y^2 = 1$$

$$\Rightarrow T_{yz}^2 + T_{xz}^2 = 1$$

For ellipse.

$$\frac{x^2}{a^2} + \frac{y^2}{b^2} = 1$$

$$\frac{T_{yz}^2}{4T^2} \frac{\pi^2}{a^2} + \frac{T_{xz}^2}{4T^2} \frac{\pi^2}{b^2} = 1$$

$$\Rightarrow \frac{a^3 b^3 \pi^2}{4T^2} \left[ T_{yz}^2 \left( \frac{1}{a} \right) + T_{xz}^2 \left( \frac{1}{b} \right) \right] = 1$$

Now for  $\frac{b}{a} = 1$ .

Normal eq. will be  $T_{yz}^2 + T_{xz}^2 = 1$ .

that will be a circle.

& otherwise  $\frac{b}{a} = 46$

$$\Rightarrow \frac{T_{yz}^2}{46} + T_{xz}^2 \times 46 = 1$$

$$\Rightarrow T_{yz}^2 = 46 - 46^2 T_{xz}^2$$

(d) Maximum shear stress ~~at~~ at

$$T_{\max} = \sqrt{T_{xz}^2 + T_{yz}^2}$$

$$= 2 \frac{G d \theta}{dz} \frac{1}{(a^2 + b^2)} \sqrt{b^4 x^2 + a^4 y^2}$$

$$= 2 \frac{G d \theta}{dz} \sqrt{\frac{b^4 x^2 + a^4 y^2}{(a^2 + b^2)^2}}$$



$$\Rightarrow T_{\text{int}} = 2 \frac{h d \theta}{dz} \sqrt{\frac{r^4 x^2 + y^2}{(1 + r^2)^2}} \quad \text{when } r = \frac{b}{a}$$

i) if  $r=1$   $T_{\text{int}} = 2 \frac{h d \theta}{dz} \sqrt{\frac{x^2 + y^2}{(1 + 1^2)^2}}$

ii) if  $r=46$   $T_{\text{int}} = 2 \frac{h d \theta}{dz} \sqrt{\frac{(46)^4 x^2 + y^2}{(1 + 46^2)^2}}$

So we can see that max strain stress will occur at boundaries.

