## AE 308: Control Theory AE 775: System Modelling, Dynamics and Control

## **Lecture 20: PD and PID Control Designs**



# Dr. Arnab Maity Department of Aerospace Engineering Indian Institute of Technology Bombay Powai, Mumbai 400076, India

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#### Introduction

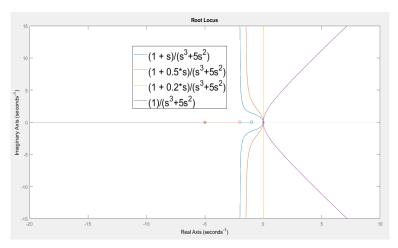
ullet Given below is the basic form of the PD controller

$$G_{PD} = K_p + K_d s = K_p \left( 1 + T_d s \right)$$

- PD controller adds a zero at  $s=-(1/T_d)=-K_p/K_d$ , where  $K_d$  is the derivative gain and  $K_p$  is proportional gain
- PD controllers are used to improve the damping of dominant system behaviour, as well as speed of response in terms of rise time

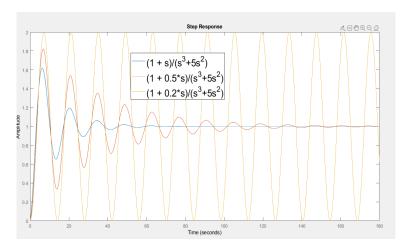


#### **Effect on Root Locus**



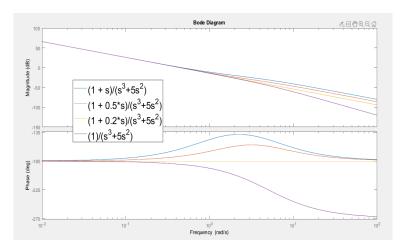


#### **Effect on Step Response**





#### **Effect on Bode Diagram**



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#### PD Controller - Control Design



#### **Control Design using Root Locus**

- Design of PD controllers is mainly concerned with determining the location of zero, based on the closed loop transient response specifications.
- In this method, we attempt to modify the root locus such that it passes through the desired dominant closed loop poles.
- The procedure makes use of angle and magnitude conditions, commonly used for drawing the root locus.
- This is done by first calculating the angle deficiency at the required dominant poles, which is used to set the zero location.
- Next, gain is determined from the magnitude condition.



**Example:** A system is defined by the following transfer function,

$$G(s) = \frac{20}{(s+1)(s+2)(s+3)}$$

- Design a PD controller to achieve following performance in the closed loop.
  - **1** 2% Settling time  $\leq 4$  seconds
  - 2 Peak time  $\leq 1$  second



**Solution:** Step Response of system,

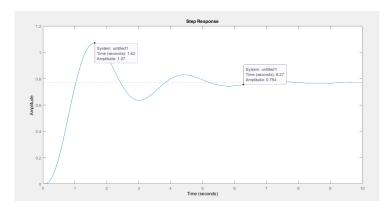


Figure: Step Response without Controller



- $T_s(2\%) = \frac{4}{\sigma} \le 4 \implies \sigma \ge 1$
- $T_p = \frac{\pi}{\omega_d} \le 1 \implies \omega_d \ge \pi$
- Therefore desired pole,

$$p_c = -\sigma \pm j\omega_d = -1 \pm j3.14$$

•

$$G(s) = \frac{20K(s+z)}{(s+1)(s+2)(s+3)}$$



Substituting,

$$s = -1 + j3.14$$

Thus,

$$G(-1+j3.14) = \frac{20K(-1+j3.14+z)}{(j3.14)(j3.14+1)(j3.14+2)}$$

• Satisfying Angle Condition,  $\angle G(-1+j3.14)=-180^{\circ}$ 

$$\theta_1 = \tan^{-1} \frac{3.14}{0} = 90^{\circ}, \quad \theta_2 = \tan^{-1} \frac{3.14}{1} = 72.3^{\circ}$$
 $\theta_3 = \tan^{-1} \frac{3.14}{2} = 57.5^{\circ}, \quad \phi = \tan^{-1} \frac{3.14}{z - 1}$ 

•  $\phi = -180 + \theta_1 + \theta_2 + \theta_3 = 39.8$ 

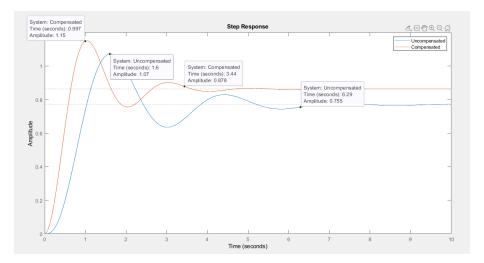


- $\tan^{-1} \frac{3.14}{z-1} = 39.8 \implies z = 4.77$
- $G_{PD} = K(s + 4.77)$
- As at  $p_c$ ,  $|G_{PD}G| = 1$

$$\implies K = \frac{|p_c + 1| \times |p_c + 2| \times |p_c + 3|}{20|p_c + 4.77|}$$
$$= \frac{3.14 \times 3.297 \times 3.724}{20 \times 4.906} = 0.3929$$

•  $G_{PD} = 0.3929(s + 4.77)$ 





## PD Control Design - Observations



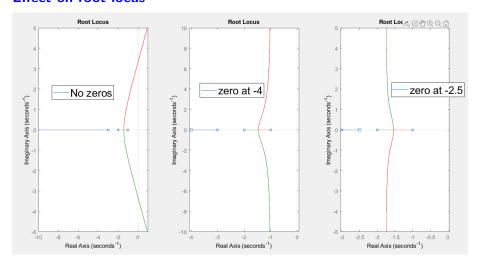
#### **Observations**

- Design is meeting the setting time and peak time requirements.
- Tracking error is also reduced

#### PD Control - Effects on root locus



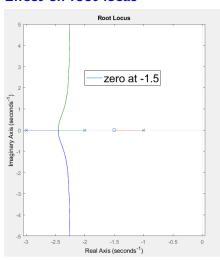
#### Effect on root locus

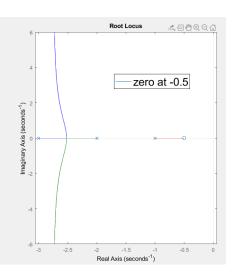


#### PD Control - Effects on root locus



#### **Effect on root locus**





#### PD Control - Observations



#### **Observations**

- Large improvements in transient response are possible with PD controllers.
- The addition of 'zero' changes the root locus shape and influences both  $\sigma$  and  $\omega_d$ , so that all attributes of the transient response are influenced.

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## PD Controller - Control Design



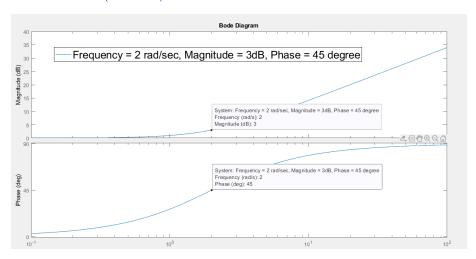
#### Control Design using Bode Plots

- Design of PD controller in frequency domain is primarily governed by the requirements on PM.
- In addition, a condition is put that DC gain remains unchanged. Therefore, in case there are requirements also on error constants, these are satisfied first, before designing PD
- The general form of PD in this case is  $K_p(1+T_ds)$ , where corner frequency  $1/T_d$  is chosen such that the positive phase to be added, occurs close to the GCO.

## PD Controller - Control Design



**Bode Plot of** (1 + 0.5s)



## **PD Control Design - Observations**



#### Observations

- We see that at frequencies  $>1/T_d$ , increase in phase is accompanied by an increase in gain as well.
- This has the effect of pushing the GCO of the compensated system to a higher value.
- Thus, the design of PD controller has to take this fact into account and add the required additional phase at the new GCO.
- This also results in a kind of iteration as the additional phase is actually calculated at the original GCO.



**Example:** A system is defined by the following transfer function,

$$G(s) = \frac{K_x}{s(s^2 + 4.2s + 14.4)}$$

- Design a PD controller to achieve following performance in the closed loop.
  - **1**  $K_v \geq 3$
  - ② GM > 6dB,  $PM > 30^{\circ}$



#### Solution:

• First step is to achieve the specified  $K_v$  which can be done by making  $K_x \ge 43.2$  (How??)



#### **Solution:**

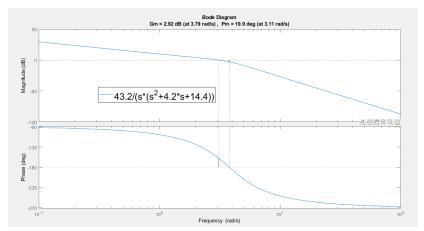
• First step is to achieve the specified  $K_v$  which can be done by making  $K_x \ge 43.2$  (How??)

$$K_v = \lim_{s \to 0} sG(s)$$

$$\implies \frac{K_x}{14 \cdot 4} \ge 3, \quad K_x \ge 43.2$$



**Bode plot of** 
$$\frac{43.2}{s(s^2+4.2s+14.4)}$$



Both GM and PM are below the desired values.



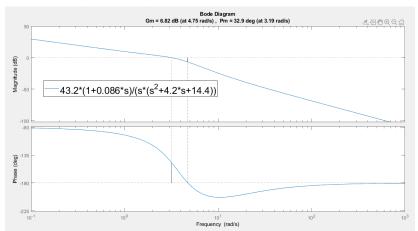
- $\bullet$  PM is to be increased by  $10^{\circ}$  at 3.11 rad/sec.
- Approximate solution for PD controller can be the one which add  $15^{\circ}$  phase.

$$\angle (1 + T_d s)|_{\omega = 3.11} = 15^{\circ}$$
 $\implies 3.11T_d = \tan 15^{\circ}, \quad T_d = 0.086$ 

$$G_{PD} = 1 + 0.086s$$



**Bode plot of**  $\frac{43.2(1+0.086s)}{s(s^2+4.2s+14.4)}$ 



All requirements are met.

#### PD Controller - Drawbacks



#### **Drawbacks**

- PD controllers are improper transfer functions and hence, reduce relative degree (n m), and may result in unexpected changes.
- $\bullet$  Further, we may also wish to preserve (n m) in order to ensure a desired slope of high frequency asymptote in bode plot.
- Therefore, we need an alternative to PD control to ensure (n m), which is the lead compensator.

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## **Lead Compensator - Structure**



#### **Structure**

• Lead compensator structure is as shown below.

$$G_{Lead}(s) = K_c \frac{\alpha(Ts+1)}{\alpha Ts+1} = K_c \frac{(s+1/T)}{s+1/\alpha T}, \quad \alpha < 1$$

- Here,  $K_c$  is compensator gain, T is the compensator time constant and  $(\alpha)$  is a parameter that decides the amount of lead added by the compensator.
- We see that above form will preserve relative degree.

## **Lead Compensator - Features**



#### **Features**

- Lead compensator adds a zero at s=-1/T and a pole at  $s=-1/(\alpha T)$ , to the plant, so that (n m) is constant.
- Further, as a bonus, we also get additional design degree of freedom, to better achieve the specifications.
- When  $\alpha \to 0$ , pole lies at  $-\infty$ , resulting in PD controller.
- Also, if  $\alpha \to 0$  and  $T \to \infty$  , the zero moves towards the origin, leading to a pure D control.
- DC gain of lead compensator is  $K_c\alpha$ , and is usually kept 1.0, which fixes  $K_c$  once  $\alpha$  is determined.

## **Lead Compensator - Features**



Phase is given by,

$$\phi = tan^{-1}\omega T - tan^{-1}\alpha\omega T$$

• Maximum phase occurs at,

$$\frac{d\phi}{d\omega} = 0 \to \omega_m = \frac{1}{\sqrt{\alpha}T}, \text{ as } \frac{d^2\phi}{d\omega^2}|_{\omega=\omega_m} < 0$$

• Substituting  $\omega = \omega_m = \frac{1}{\sqrt{\alpha}T}$  in phase equation, we obtain

$$\tan \phi_m = \frac{1 - \alpha}{2\sqrt{\alpha}} \to \sin \phi_m = \frac{1 - \alpha}{1 + \alpha}$$

## **Lead Compensator - Example**



**Example :** Find maximum phase and the frequency at which it occurs of the following system,

$$G(s) = \frac{0.01(1+s)}{0.01s+1}$$

## **Lead Compensator - Example**



#### Solution:

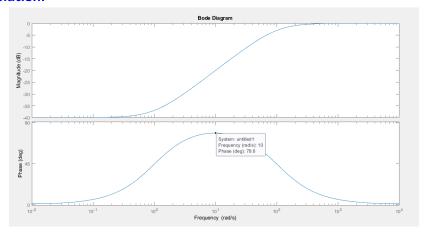


Figure:  $\omega_m = 10 rad/sec$ ,  $\phi_m = 78.6^{\circ}$ 

## Limitations of P, PI, PD



#### Limitations

- With the design of P, PI and PD controllers, we are in a position to ensure a wide range of tracking and transient responses for any given plant.
- However, the above assurance is usually under the condition that either tracking or transient response features drive the design of control element.
- In reality, we are likely to encounter a combination of steady-state and transient response specifications, so that employing any one of these would not be adequate.

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#### PID controller - Introduction



#### Introduction

- PID controller aims to achieve both tracking and transient response simultaneously and, hence, is a better option in comparison to either PI or PD.
- This is so because it includes all three actions which help in achieving a wide range of performance.
- Further, it manages the overall design effort well, while increasing the overall design degrees of freedom.

#### PID controller - Zeigler - Nichols



#### Zeigler - Nichols PID Design

- Zeigler Nichols is a methodology for designing PID controllers, based on the specific assumptions about the unit step response of the plant.
- $\bullet$  The controller transfer function is rewritten in terms of the overall gain  $K_p$  and time constants  $T_i$  and  $T_d$  , as shown below.

$$G_c(s) = K_p \left( 1 + \frac{1}{T_i s} + T_d s \right)$$

• There are two methods for arriving at the controller.



#### First Method

 This method applies if the response to a step input exhibits an S-shaped curve

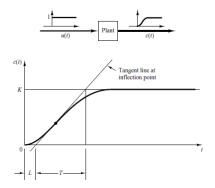


Figure: Source - "Modern Control Engineering" by Katsuhiko Ogata



- ullet The S-shaped curve is characterized by two constants, delay time L and time constant T.
- ullet The delay time and time constant are determined by drawing a tangent line at the inflection point of the S-shaped curve and determining the intersections of the tangent line with the time axis and line c(t)=K
- Transfer function may then be approximated by a first-order system with a transport lag as follows:

$$\frac{C(s)}{U(s)} = \frac{Ke^{-Ls}}{Ts+1}$$



• Ziegler and Nichols suggested to set the values of  $K_p, T_i, T_d$  according to the following table,

Type of Controller	$K_p$	$T_i$	$T_d$
Р	$\frac{T}{L}$	$\infty$	0
PI	$0.9\frac{T}{L}$	$\frac{L}{0.3}$	0
PID	$1.2\frac{T}{L}$	2L	0.5L



PID controller tuned by the first method of Ziegler–Nichols rules gives:

$$G_c(s) = K_p \left( 1 + \frac{1}{T_i s} + T_d s \right)$$
$$= 1.2 \frac{T}{L} \left( 1 + \frac{1}{2Ls} + 0.5 L s \right)$$
$$= 0.6 T \frac{\left(s + \frac{1}{L}\right)^2}{s}$$

 $\bullet$  Thus, the PID controller has a pole at the origin and double zeros at s=-1/L.

## **Ziegler Nichols - Second Method**



#### Second Method

- In the second method, we first set  $T_i=\infty$  and  $T_d=0$ . Using the proportional control action only, increase  $K_p$  from 0 to a critical value  $K_{cr}$  at which the output first exhibits sustained oscillations.
- $\bullet$  If the output does not exhibit sustained oscillations for whatever value  $K_p$  may take, then this method does not apply.
- Thus, the critical gain  $K_{cr}$  and the corresponding period  $P_{cr}$  are experimentally determined.

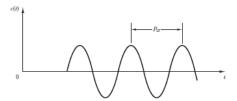


Figure: Source - "Modern Control Engineering" by Katsuhiko Ogata

## **Ziegler Nichols - Second Method**



 $\bullet$  Ziegler and Nichols suggested that we set the values of the parameters  $K_p$  ,  $T_i$  , and  $T_d$  according to the table

Type of Controller	$K_p$	$T_i$	$T_d$
Р	$0.5K_{cr}$	$\infty$	0
PI	$0.45K_{cr}$	$\frac{1}{1.2}P_{cr}$	0
PID	$0.6K_{cr}$	$0.5P_{cr}$	$0.125P_{cr}$

## **Ziegler Nichols - Second Method**



 PID controller tuned by the second method of Ziegler–Nichols rules gives:

$$G_c(s) = K_p \left( 1 + \frac{1}{T_i s} + T_d s \right)$$

$$= 0.6 K_{cr} \left( 1 + \frac{1}{0.5 P_{cr} s} + 0.125 P_{cr} s \right)$$

$$= 0.075 K_{cr} P_{cr} \frac{\left( s + \frac{4}{P_{cr}} \right)^2}{s}$$

• Thus, the PID controller has a pole at the origin and double zeros at  $s=-4/P_{cr}$ .

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## PID Controller Design - Bode



#### PID control design using Bode

- Zeigler-Nichols method of tuning broadly aims to arrive at a stable closed loop system with acceptable transient response.
- However, a more focused design can be done using frequency domain methods, which take into account the design specifications.
- In this method, following generic form of the PID controller is assumed

$$G_{PID}(s) = \frac{K(as+1)(bs+1)}{s}$$



**Example:** Consider a system as given below.

$$G(s) = \frac{1}{s^2 + 1}$$

• Design a PID controller so that  $K_v$  is 4, PM is at least  $50^\circ$  and GM is more than 10dB.



#### **Solution:**

• PID controller is assumed as,

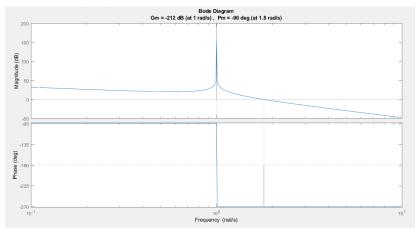
$$G_{PID}(s) = \frac{K(as+1)(bs+1)}{s}$$

ullet First step is to achieve the specified  $K_v$  which can be done by making K=4

$$K_v = \lim_{s \to 0} sG(s)G_{PID}(s)$$
  
 $\implies \frac{K}{1} = 4, \quad K = 4$ 



# Bode plot of $\frac{4}{s(s^2+1)}$



• We see that we need to add a large positive phase at the GCO.

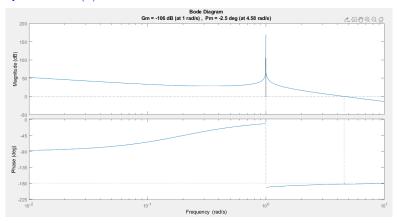


- Large positive phase at the GCO can be added by first choosing value of 'a' to be large (say 5) (i.e. zero at -0.2)
- This acts as the PI controller, as shown below.

$$G''(s) = \frac{5s+1}{s} \times \frac{4}{s^2+1}$$



#### Bode plot of G''(s)



 $\bullet$  We see that, while GCO increases from 1.8 to 4.58, PCO remains unchanged.

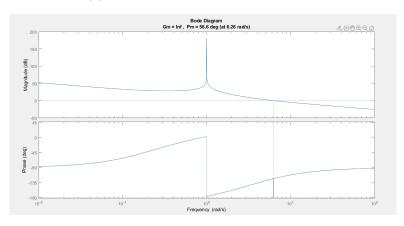


- We now choose 'b' so that PM requirement is met. This also will change PCO by adding more positive phase in low frequency regime.
- $\bullet$  Approximate solution for PD controller can be the one which add  $50^\circ$  phase.

$$\angle (1+bs)|_{\omega=4.58} = 50^{\circ}$$
  
 $\implies 4.58b = \tan 50^{\circ}, \quad b = 0.26$   
 $G'''(s) = (1+0.26s)\frac{5s+1}{s} \times \frac{4}{s^2+1}$ 



#### Bode plot of G'''(s)



### PID Controller Design - Observations



#### **Observations**

- Desired conditions are met
- We see that, as GCO increases to 6.14, PM is more than required.
- Further, as there is no PCO, GM becomes infinite.

#### References



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