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# Comparison of BEM and Full Navier-Stokes CFD Methods for Prediction of Aerodynamics Performance of HAWT Rotors

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**Abstract**— The essential contribution of this study consists of comparing between two radically different aerodynamic methods which were applied to predict the aerodynamic performance of horizontal axis wind turbines (HAWTs). The classical blade element momentum theory (BEM) and full rotor geometry computational fluid dynamics (CFD) based on the Reynolds Averaged Navier-Stokes (RANS) approach were used in order to discover their strengths and weaknesses for a range of wind speeds where the flow over the rotor varied from fully attached flow to massively separated flow (i.e. Tip Speed Ratio, TSR = 4–10). New MEXICO measurements (Model rotor EXperiments In Controlled conditions) from German-Dutch wind tunnel (DNW) which were recently carried out between June and July 2014 are used for validating and testing the both BEM and CFD codes. In general, it was founded that, RANS-CFD simulations give good agreements with uniform accuracy level in all studied cases, whereas BEM calculations give reasonable results only at low wind speeds and it fails at higher wind speeds due to the separated flow conditions.

**Keywords**—Wind energy; Aerodynamics analysis; New MEXICO Measurements; CFD; RANS; Classical BEM.

## I. INTRODUCTION

With increasing global energy needs and growing environmental worry, the search for new efficient and inexhaustible energy sources is an obligation and not a choice. The importance of renewable energy in tomorrow's energy market becomes evident from the energy policies. According to the International Energy Agency's report *World Energy Outlook* [1], the projected share of renewable in the worldwide electricity generation reaches 31% by 2035, a quarter of which is from wind power alone.

At the last two decades, the wind energy has seen an unprecedented progress and great success in the industry application where the annual growth rate has exceeded the 30%[2]. According to the recent Global Wind Energy Council (GWEC-2017)[3], the total installed capacity produced by the

wind turbines has reached at 486 (GW) today, this value practically has been multiplied by 20 in these 16 last years.

It is well recognized that a great part linked to the development of the wind energy was the results of the development of wind turbine aerodynamics [4]. Many numerical methods based on aerodynamic concepts have been developed and used to compute rotor performance, to predict the capacity of wind energy and to simulate the flow around wind turbines. Historically, the developed methods for helicopter and propeller rotors were the starting point for the development of wind turbine calculation methods [5].

The first predictions of the performances of wind turbine rotors, based on a simple energy balance of a perfect fluid, were realized in the first decades of the 20th century. The result obtained by the German Betz [6] in 1920 is famous and known as (Betz limit) which demonstrates that the ideal maximum power of a wind turbine could not exceed 59.3%. In 1935, Glauert [7] developed the 1D momentum theory into a more general theory, which is commonly known as the Blade Element Momentum (BEM) theory.

In the BEM theory, the rotor is modeled as a series of elements, this theory operates with two fundamental assumptions, it assumes that there is no radial dependence of the quantity of momentum through the rotor, and the forces can be calculated on average in the individual rings. According to these assumptions, the forces and velocities within this annular element are not influenced by any other annular element and consequently, a wind turbine rotor can be divided into (N) rings number and the forces in each ring can be calculated independently. On the basis of these two hypotheses, the idea is to find a balance between the force exerted on the flow field by the blade elements and the variation of the axial momentum quantity of the fluid between the upstream and downstream locations of the rotor. Thereafter, an iterative process is used to obtain this equilibrium between the forces on the blade elements and the forces of the flow field, then, by the integration of the elementary forces in each ring along the blade

makes it possible to determine the performance of the rotor, such as total power, rotor thrust, aerodynamic forces, etc.

because these methods are fast, easily programmed and do not require significant memory space, this is what left it to be the frequently used method in the industrial design of wind turbine blades[8-10].

With the advent of the computer age since the 1960s, aerodynamics experienced a large unprecedented explosion. An advanced category of methods which allows us to obtain more detailed information of the flows around wind turbines appeared is the so-called Computational Fluid Dynamics (CFD) methods. The most used CFD methods is which based on the numerical resolution of the Navier-Stokes equations with full blade geometry, this method has been used extensively in the literature by different authors [11-14].

The development, application, and acceptance of aerodynamic methods (e.g. BEM, CFD) for a wind turbine will be greatly dependent on the availability of good experimental data under controlled conditions. The NREL/NASA AMES Wind Tunnel Experiment in 1999 [15, 16] might be the most well-known.

Here, the focus is on the recent New MEXICO (Model Experiments in Controlled Conditions) measurements. The MEXICO experiments were carried out at the large-scale low-speed facility (LLF) of the German-Dutch wind tunnels (DNW), which is a high-quality wind tunnel with a 9.5x9.5 m<sup>2</sup> open test section. The first series of experiments were performed in December 2006 [17-19]. Between 20th June and 4th, July 2014 the New MEXICO measurements were followed up of the first test keeping the same model [20, 21].

The MEXICO rotor [22] is a three-bladed rotor model of 4.5 m diameter equipped with a speed controller and pitch actuator. The blade of MEXICO rotor is aerodynamically complicated, it consists of three different families of aerodynamic profiles (DU91-W2-250, RISØ-A1-21, and NACA 64-418), for more details, see [20, 21].

The main goal of this work is to compare results of two different aerodynamic methods: the classical BEM theory, and full Navier-Stokes CFD method. The recent New MEXICO measurements were selected for validating and testing of both BEM and CFD codes. The BEM code was programmed using MATLAB software based on the aerodynamic loads which extracted experimentally at appropriate Reynolds number for each MEXICO blade profile in the German-Dutch Wind tunnel (DNW) and the CFD simulations are performed using the finite volume flow solver, ANSYS Fluent 17.2 based on resolution of Reynolds Averaged Navier-Stokes (RANS) equations closed by  $k - \varepsilon$  RNG high Reynolds turbulence model [23], this model can give the best results with reasonable computational time [14, 24, 25]. The simulations include three cases in axial flow conditions with constant blade pitch angle (-2.3°), under constant rotational speed (425.1 rpm). The comparisons of results showed the strength and weakness of both BEM and CFD codes in attached and separated flow conditions.

## II. NUMERICAL METHOD

In this investigation, three wind speeds were investigated using classical BEM and full Navier-Stokes CFD methods, these cases represent the turbulent wake state (TSR=10), design conditions (TSR=6.65), and separated flow conditions (TSR=4.16). The operational conditions for the three simulated cases can be found in Table 1.

TABLE I. THE OPERATIONAL CONDITIONS OF STUDIED CASES

Case	TSR	$U_\infty$ (m/s)	$\rho$ (kg/m <sup>3</sup> )	$P_\infty$ (Pa)
1	9.9664	10.05	1.197	101398
2	6.6509	15.06	1.191	101345
3	4.1647	24.05	1.195	101407

### A. BEM code

According to the BEM theory, the angle of attack (AOA) at each section of the blade is computed by[26]:

$$\tan(\phi) = \frac{(1-a)V_0}{(1-a')r\omega} \quad (1)$$

$$AOA = \phi - \theta \quad (2)$$

In Eq. (1),  $a$  is the axial induction factor,  $a'$  is the tangential factor,  $V_0$  is the upstream wind velocity and  $\omega$  is the rotor rotational speed. In Eq. (2),  $\theta$  is the sum of blade angles (twist and pitch).

By projection the lift and drag forces on the profile in the normal and parallel directions to the rotor plane, the normal and tangential force coefficients (normalized with respect to  $[1/2\rho V_{rel}C]$ , where  $C$  is the chord length in each section) can be calculated as:

$$C_N = C_L \cos \phi + C_D \sin \phi \quad (3)$$

$$C_T = C_L \sin \phi - C_D \cos \phi \quad (4)$$

The new value of the  $a$  and  $a'$  based on the Prandtl's tip loss corrections can be calculated as [9, 27]:

$$a = \frac{1}{4F \sin^2 \phi / \sigma C_N + 1} \quad (5)$$

$$a' = \frac{1}{4F \sin \phi \cos \phi / \sigma C_T - 1} \quad (6)$$

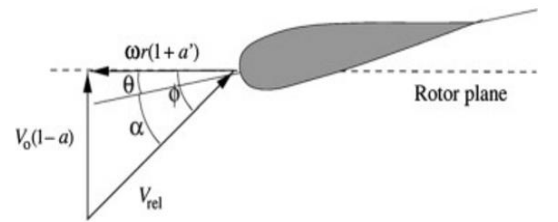


Figure 1. Velocity components on the profile.

Where,  $\sigma$  is the solidity given as a function of the chord at each section of the blade,  $C(r)$  by:

$$\sigma = \frac{C(r)N}{2\pi R} \quad (7)$$

Where  $N$  is the blades number of the studied wind turbine and  $R$  is the rotor radius.

In Eqs. 5 and 6 the correction function  $F$  proposed by Prandtl is defined as [28]:

$$F = \frac{2}{\pi} \arccos(e^{-f}) \quad (8)$$

$$f = \frac{N}{2} \frac{R-r}{r \sin \phi} \quad (9)$$

In the first iteration, the induction factors  $a$  and  $a'$  are assumed as zero value and the AOA is calculated via Eq. (2). Then, using the  $C_L$  and  $C_D$  of the corresponding profile of the blade at each AOA, the corresponding  $C_N$  and  $C_T$  can be calculated using Eqs. (3) and (4). Thereafter, the new value of the  $a$  and  $a'$  are calculated again using Eqs. (5) and (6). And the calculation is continuous until the convergence, Fig.2.

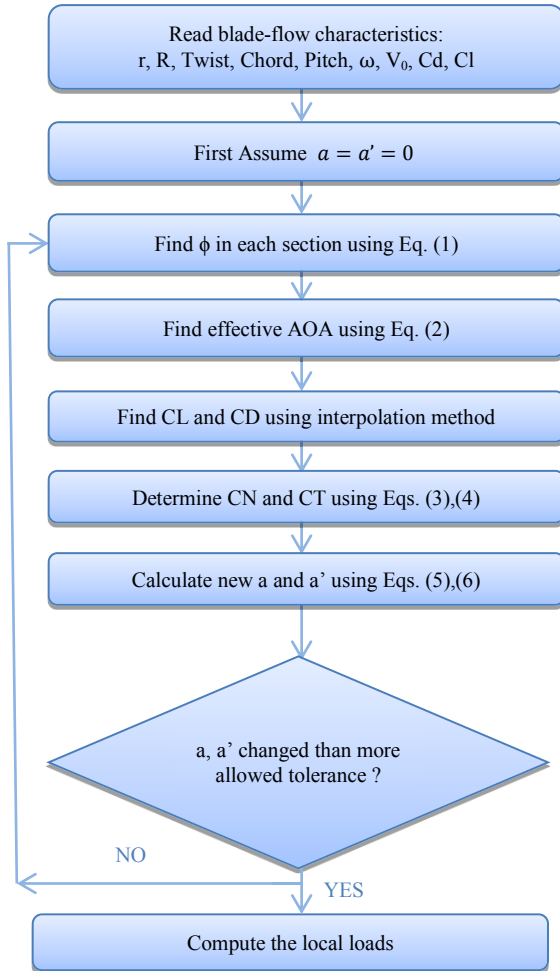


Figure.2. Blade element momentum theory (BEM) algorithm.

The algorithm illustrated in Fig.2 has been programmed using MATLAB software (version-16), based on the experimental normal and tangential forces at each profile of the MEXICO blade measured at the appropriate Reynolds number ( $Re=0.7 \times 10^6$ ), see Fig.3.

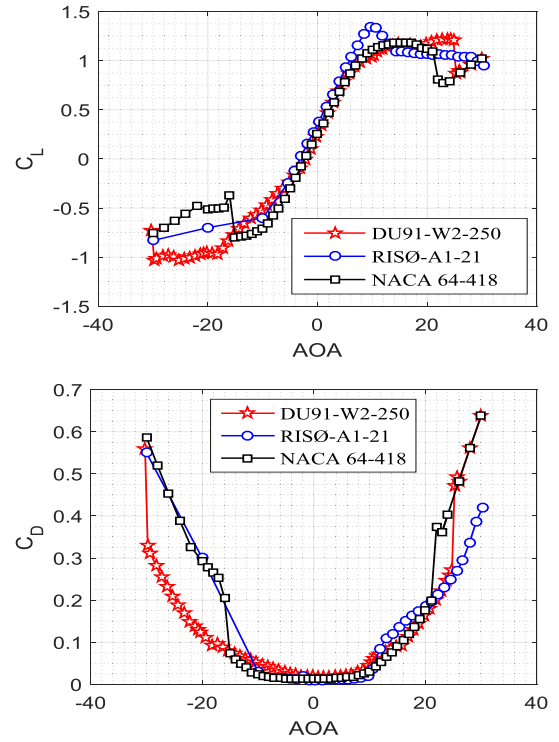


Figure 3. Lift and drag coefficients distribution for three MEXICO blade airfoils, at  $Re=0.7 \times 10^6$ .

### B. CFD code

The 3D geometry of the MEXICO blade was generated based on the 2D airfoils coordinates and the detailed blades geometry using SolidWorks Software, version-14. The computational domain represents a cylindrical tunnel configuration of 5 blade radii upstream and 10 blade radii downstream with a radius of 5 blade radii (Fig.4). A small region close to the blade was designed with dimensions of two blade radii in diameter and one-blade radii in length for separate the stationary part into the rotating part. The multiple reference frame (MRF) [29] approach is applied in this zone for taking into the account the rotating of the blades by additional terms in the Navier-Stokes equations (Coriolis and centripetal accelerations) [30]. In this study, and in order to reduce the computational grid near the wall, a universal standard wall function which is first proposed by Launder and Spalding [31] was used as boundary condition near the wall.

The independence of the mesh will not be presented here, for more details on the independence of the grid and the boundary conditions, see ref [14]. The selected mesh contains about 2.7 million tetrahedral nodes in the one-third computational domain. The height of the first-floor mesh element is about  $4 \times 10^{-4}m$ , which assures that  $y^+$  is about 30 on the blade surface for applying the wall law function.

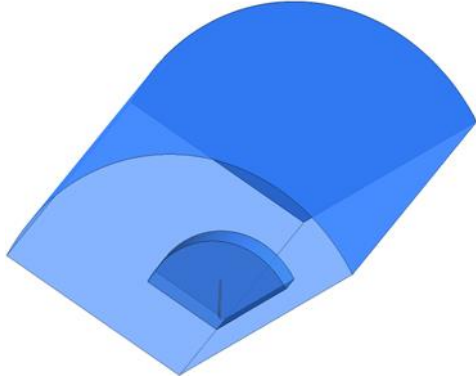


Figure 4. Computational domain

### III. RESULTS AND DISCUSSION

#### A. Normal and Tangential Forces

The comparison between BEM and CFD-RANS results of normal and tangential forces distribution is shown in Fig.5. At low wind speed, (TSR =10), where the flow is mostly attached on the whole of MEXICO blade, a good agreement between the measured results and calculations for both BEM and CFD, especially for CFD results was found. With augmentation of wind speed and a decrease of TSR, was founded that, the under-prediction for the normal forces distribution and the over-prediction for the tangential forces for BEM results, especially in high wind speeds, at TSR =6.65 and 4.16, (i.e. with increase of wind speed, the accuracy of BEM decreases). This weakness of BEM results is mainly caused by the phenomenon of the stall delay produced by 3D turbulence effects, which cannot predicted it using classical BEM. The CFD-RANS results give a good agreement with experimental data for all cases, this reflects the ability of the RANS-CFD to predict the aerodynamic performance of the rotor.

#### B. Torque curve

The comparison of the aerodynamic torque calculated by BEM and RANS-CFD codes and measured from New MEXICO experiments as a function of TSR are represented in Fig.6. The torque was calculated in CFD code based on a simple linear variation between the tangential forces in five-span sections of the blade (25%, 35%, 60%, 82% and 92%) assuming zero value at the root and tip and the integration is based on the Trapezoid method. From the figure, it can be seen that the CFD results give a good agreement with experimental with uniform accuracy level for the three simulated cases, and the curve of BEM torque was over-prediction by about 31% in all cases.

### IV. CONCLUSION AND FUTURE WORK

In this study, the comparison between two radically different aerodynamic methods on the prediction of the performance of horizontal axis wind turbines (HAWTs) for attached and separated flow conditions was carried out. The classical blade element momentum (BEM) with Prandtl tip loss factor model

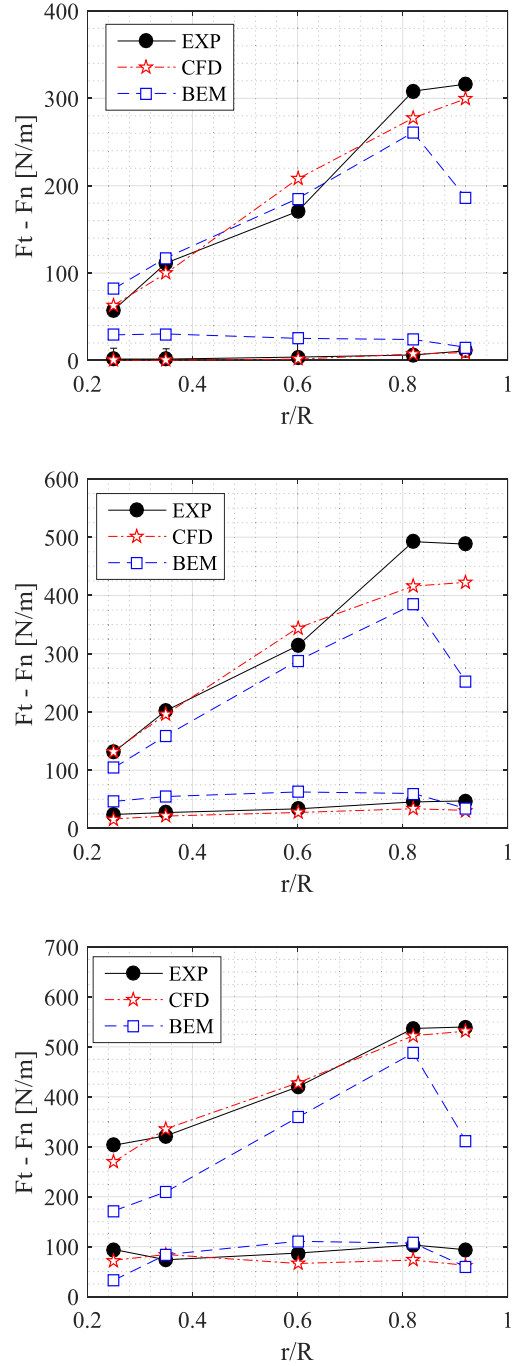


Figure 5. Normal and tangential forces distribution for three cases.

and full rotor geometry CFD-RANS approach were performed to predict the aerodynamic performance of the MEXICO wind turbine.

This work shows that, on the one hand, the classical BEM calculation (without any stall correction model) is not appropriate to predict of wind turbine performance especially for high wind speeds in the presence of the flow separation phenomenon. The classical BEM calculations give under-prediction results for the normal force distribution and the over-



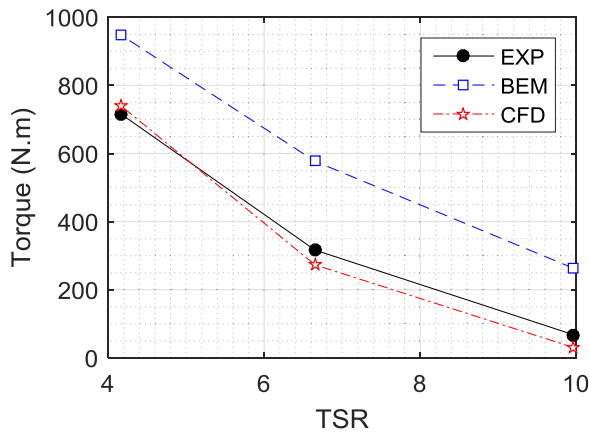


Figure.6. Comparison of torque measured from BEM and CFD methods.

prediction of the tangential forces, where the torque is overestimated by more than 30%, this weakness due to the rotational augmentation caused by 3D turbulence effects, this due to the 2D data used in BEM is different from the 3D results. On the other hand, the found results using CFD-RANS method were in good agreement on a uniform level of accuracy with experimental data for all studied cases (i.e. attached and separated flows), this reflects the ability of the RANS-CFD code to predicting the separation of the flows and this is what makes them are the most appropriate methods for predicting the aerodynamic performance of the wind turbines.

As a future work linked to this study, it is recommended to perform a comparison between different models available for correcting the BEM (as, Tip loss factors models, Stall-delay models, Dynamic-stall models). In addition, the accuracy of BEM codes should be improved by developing new correction models based on the 3D turbulence effects and separated flows phenomenon.

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