

Books

14 January 2018 20:43

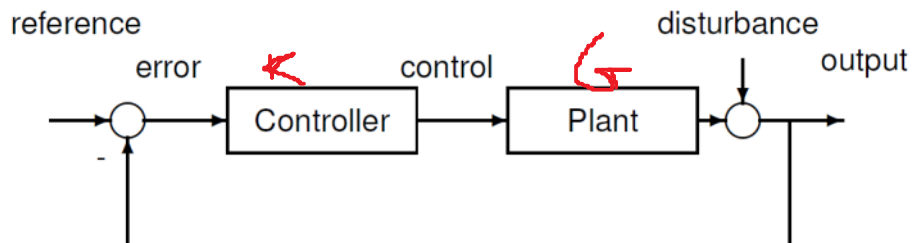
(Use latest editions)

- *Modern Control System Theory and Design*, S.M. Shinnars, Wiley. Good for the state-space approach.
- *Modern Control Systems*, R.C. Dorf and R.H. Bishop, Addison-Wesley. Good for the transfer function approach.
- *Feedback Control of Dynamic Systems*, G.F. Franklin, J.D. Powell and A. Emami-Naeini, Addison-Wesley. Good for practical design issues.
- *Feedback Control Systems*, C.L. Phillips and R.D Harbor, Prentice-Hall. Follows the lecture course closely, and is useful for third year course too.

Aims of Control

14 January 2018 20:47

Use of feedback to force a given plant to exhibit desired characteristics



$error = reference - output$
keep it small - output close to reference

Base Model around existing G 's (plant) frequency response
Design Controller

Feedback For Static Systems

14 January 2018 21:16

Block Diagrams:

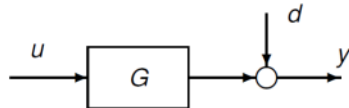
- A way of representing equations involving signals

Problem with open loop: Can't make $y=u$

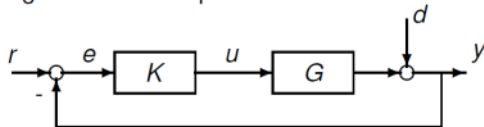
Don't know disturbance before hand

Consider the open loop static system:

$$y = Gu + d.$$



Introducing feedback compensation:



R and D - external signals

E, U, Y - internal signals

Start with E: solve for

$$e = F(r, d, K, G)$$

Then solve for U and Y, eliminating e with above equation

$$\rightarrow Y = \frac{GK}{1 + GK} r + \frac{1}{1 + GK} d, |GK| \gg 1$$

$$y \rightarrow r$$

Good tracking and **disturbance rejection require high loop gain**

Sensitivity:

- $S = \frac{1}{1 + GK}$ (system to the left)
 - reduce sensitivity - high loop gain

Use of Laplace Transforms in Circuit

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convert ODE into simple quadratic

Using LT tables convert back into time

Impedance

$$Z(s) = \frac{V(s)}{I(s)}$$

- Resistor, $Z(s) = R$
- Inductor, $Z(s) = sL$
- Capacitor, $Z(s) = \frac{1}{sC}$

Transfer Function

$$G(s) = \frac{V_2(s)}{V_1(s)}$$

- ratio of LT of output / input
- for a linear system
- all initial conditions assumed to be zero

Standard Format

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Decompose an nth order DE into a set of n 1st order DE's
written in matrix-vector form

$$\dot{x}(t) = Ax(t) + Bu(t) \text{ (state equation), } x(0) = x_0 \text{ (initial condition)}$$

$$y(t) = Cx(t) + Du(t) \text{ (output equation)}$$

$$A : n \times n \text{ (system matrix); } B : n \times n_u \text{ (input matrix)}$$

$$C : n_y \times n \text{ (output matrix); } D : n_y \times n_u \text{ (direct feedthrough)}$$

$$x(t) : n \times 1 \text{ (state vector); } u(t) : n_u \times 1 \text{ (input vector)}$$

$$y(t) : n_y \times 1 \text{ (output vector)}$$

Example: 2-input 2-output state-variable model

Suppose $\ddot{y}_1 + k_1 \dot{y}_1 + k_2 y_1 = u_1 + k_3 u_2$, $\dot{y}_2 + k_4 y_2 + k_5 \dot{y}_1 = k_6 u_1$.
Define $x_1 = y_1$, $x_2 = \dot{y}_1$, $x_3 = y_2$. Then

$$\begin{aligned} \dot{x}_1 &= x_2 \\ \dot{x}_2 &= -k_2 x_1 - k_1 x_2 + u_1 + k_3 u_2 \\ \dot{x}_3 &= -k_5 x_2 - k_4 x_3 + k_6 u_1 \\ y_1 &= x_1, \quad y_2 = x_3 \end{aligned}$$

$$\underbrace{\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix}}_{\dot{x}} = \underbrace{\begin{bmatrix} 0 & 1 & 0 \\ -k_2 & -k_1 & 0 \\ 0 & -k_5 & -k_4 \end{bmatrix}}_A \underbrace{\begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}}_x + \underbrace{\begin{bmatrix} 0 & 0 \\ 1 & k_3 \\ k_6 & 0 \end{bmatrix}}_B \underbrace{\begin{bmatrix} u_1 \\ u_2 \end{bmatrix}}_u$$

$$\underbrace{\begin{bmatrix} y_1 \\ y_2 \end{bmatrix}}_y = \underbrace{\begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 1 \end{bmatrix}}_C \underbrace{\begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}}_x$$

Simulation Diagrams

05 February 2018 22:45

Three Elements:

- **integrator** - transfer function $\frac{1}{s}$
- **pure gain**
- **summer**

Models from transfer functions:

- Divide numerator and denominator by s^n :

$$y(s) = \frac{b_{n-1}s^{-1} + b_{n-2}s^{-2} + \dots + b_0s^{-n}}{1 + a_{n-1}s^{-1} + a_{n-2}s^{-2} + \dots + a_0s^{-n}} u(s).$$

- Set

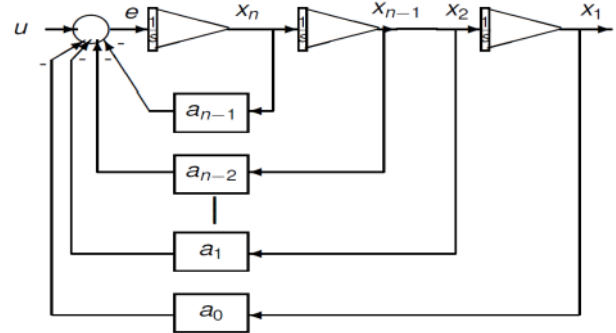
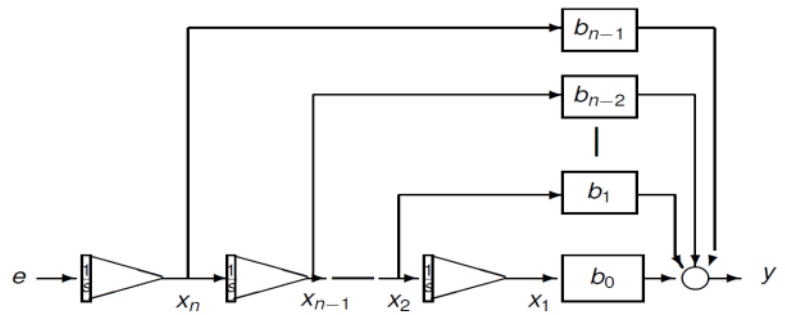
$$e(s) := \frac{u(s)}{1 + a_{n-1}s^{-1} + a_{n-2}s^{-2} + \dots + a_0s^{-n}}$$

so that

$$y(s) = [b_{n-1}s^{-1} + b_{n-2}s^{-2} + \dots + b_0s^{-n}] e(s).$$

- This gives

$$e(s) = u(s) - [a_{n-1}s^{-1} + a_{n-2}s^{-2} + \dots + a_0s^{-n}] e(s).$$



Matrix Exponential function

21 January 2018 23:41

- In matrix form this yields the state-variable model:

$$\dot{x} = \begin{bmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 \\ \vdots & \vdots & \ddots & \ddots & \vdots \\ 0 & 0 & \cdots & 0 & 1 \\ -a_0 & -a_1 & \cdots & -a_{n-2} & -a_{n-1} \end{bmatrix} x + \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} u,$$

$$y = [b_0 \ b_1 \ \cdots \ b_{n-2} \ b_{n-1}] x.$$

- Note the direct connection with the coefficients of the transfer function:

$$G(s) = \frac{b_{n-1}s^{n-1} + b_{n-2}s^{n-2} + \cdots + b_0}{s^n + a_{n-1}s^{n-1} + a_{n-2}s^{n-2} + \cdots + a_0}.$$

For a Square Matrix A - **Matrix Exponential Function**

$$e^{At} = I + At + \frac{A^2 t^2}{2!} + \frac{A^3 t^3}{3!} + \cdots + \frac{A^k t^k}{k!} + \cdots$$

Useful Properties:

- P1: $e^0 = I$
- P2: $e^{T \Lambda T^{-1}} = T e^{\Lambda} T^{-1}$ for any nonsingular T
- P3: $e^{(\alpha+\beta)A} = e^{\alpha A} e^{\beta A}$
- P4: $e^{-A} = (e^A)^{-1}$
- P5: $\frac{d}{dt} e^{At} = A e^{At} = e^{At} A$
- P6: $\mathcal{L}(e^{At}) = (sI - A)^{-1}$

Test signals

02 June 2018 12:27

Unit impulse:

- $\delta(t) \rightarrow 1$

Unit Step:

- $u(t) = 1, t \geq 0 \rightarrow \frac{1}{s}$

Unit Ramp

- $u(t) = t, t \geq 0 \rightarrow \frac{1}{s^2}$

$$\cos \omega t \rightarrow \frac{s}{s^2 + \omega^2}$$

$$\sin \omega t \rightarrow \frac{\omega}{s^2 + \omega^2}$$

First-order

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Steady-state - not affected by T

Transient - affected by T

Suppose that

$$y(s) = G(s)u(s),$$

where $G(s)$ is a given transfer function.

Suppose that the limit

$$\lim_{t \rightarrow \infty} y(t)$$

exists.

Using the **final value theorem**:

$$\lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} sy(s) = \lim_{s \rightarrow 0} sG(s)u(s).$$

If $u(t)$ is a unit step, this becomes:

$$\lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} sG(s) \frac{1}{s} = \lim_{s \rightarrow 0} G(s) = G(0).$$

$G(0)$ is often called the **d.c. gain** of the system.

Second-order

02 June 2018 12:23

for $ky(t) + B\dot{y}(t) + M\ddot{y}(t) = u(t)$

$$G(s) = K \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$K = \frac{1}{k}, \quad \omega_n = \sqrt{\frac{k}{M}}, \quad \zeta = \frac{B}{2\sqrt{kM}}$$

$G(s)$ has poles at:

- $p_1, p_2 = -\zeta\omega_n \pm \omega_n\sqrt{\zeta^2 - 1}$

ζ	Case	Description
>1	Overdamped	p_1 and p_2 are real, negative and unequal
$=1$	Critically Damped	p_1 and p_2 are real, negative and equal
$0 \leq 1$	Underdamped	p_1 and p_2 are complex conjugates, negative real parts
< 0	Unstable	p_1 and p_2 positive real parts

Unit Step response:

- underdamped system
- damped natural frequency: $\omega_d = \omega_n\sqrt{1 - \zeta^2}$

$$y(t) = 1 - \frac{1}{\sqrt{1 - \zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_d t + \alpha)$$

- exponential - envelope
 - time constant $\tau = \frac{1}{\zeta\omega_n}$
- sine - Im part of the poles

Specifications: underdamped 2nd order

- T_r - rise time (10% to 90%)
- M_p - peak value $= 1 + e^{-\frac{\zeta\pi}{\sqrt{1-\zeta^2}}}$
- T_p - time to first peak $= \frac{\pi}{\sqrt{1-\zeta^2}\omega_n}$
- y_{ss} - steady state value $= 1$ - unit step
- %overshoot $= \frac{M_p - y_{ss}}{y_{ss}} \times 100 = 100e^{-\frac{\zeta\pi}{\sqrt{1-\zeta^2}}}$
- T_s - settling time
 - time to enter range of $y_{ss} \pm d$ tube
 - $T_s \approx 4\tau = \frac{4}{\zeta\omega_n}$ for 2%

5% overshoot, $\zeta = 0.7 \approx \frac{1}{\sqrt{2}}$

As ζ increases from 0 to 1

- tracking properties, T_s and %overshoot decrease
- speed of response, T_p and T_r increase

Overdamped: T_r, T_s and y_{ss} well defined,
but not peak values, M_p and T_p

Interpret Pole locations

02 June 2018 12:23

Each pole is either real - contributes a pure exponential
Or occur in complex conjugate pairs - contributing an exponentially decaying sinusoid

Can approximate a system a system if $G(s)$ has **dominant poles of 2nd order**,
i.e. other poles are located far into left half plane

Steady-state accuracy

02 June 2018 12:23

System type N: number of free integrators in the loop (s^{-1} 's)

Table:

The following table gives e_{ss} for type 0, 1, and 2 systems for step, ramp and parabolic inputs.

Type	$r(s) = \frac{1}{s}$	$r(s) = \frac{1}{s^2}$	$r(s) = \frac{1}{s^3}$	Error constants
0	$\frac{1}{1 + K_p}$	∞	∞	$K_p = \lim_{s \rightarrow 0} Q(s)$
1	0	$\frac{1}{K_v}$	∞	$K_v = \lim_{s \rightarrow 0} sQ(s)$
2	0	0	$\frac{1}{K_a}$	$K_a = \lim_{s \rightarrow 0} s^2 Q(s)$

Steady-state error constants

Bode to Nyquist

05 June 2018 03:17

If phase decreasing - moving clockwise

reducing gain - moving towards origin

BIBO stability

02 June 2018 14:51

System is **unstable** when one or more poles in the **RHS plane**

BIBO stability when poles all poles in **open LHS plane (not including Im axis)**

Marginally stable - when all poles lie in **closed LHS** (plane) - poles on **Im axis must also be simple poles**

Criteria

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For 1st and 2nd order, sufficient to have all coefficients have same sign

For higher order, this is necessary but not sufficient

Routh Array

Degree N, -> **N+1 rows**

First two rows:

Nth Coefficient, skipping one until end

Nth-1 Coefficient skipping one until end

Next rows:

$$i^{th} \text{ entry} = - \frac{1}{\text{row above, first element}} \begin{vmatrix} \text{two rows above, 1st el} & \text{two rows above, } i+1 \text{ entry} \\ \text{one row above, 1st el} & \text{one row above, } i+1 \text{ entry} \end{vmatrix}$$

first element in a row = 0, and other elements will be non-zero in same row

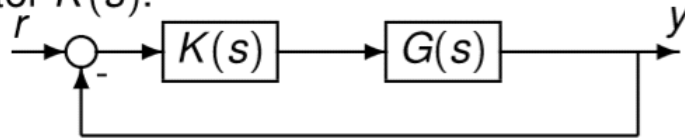
replace 0 with ϵ

unstable roots = number of sign changes in first column

Application

02 June 2018 14:52

- Consider the feedback system involving a plant $G(s)$ and compensator $K(s)$.



- The closed-loop transfer function $H(s)$ is

$$H(s) = \frac{G(s)K(s)}{1 + G(s)K(s)}.$$

Can use Routh Array to see if can find a **constant K**, if $K(s) = K$,

can make feedback loop stable, if $G(s)$ by itself gives an unstable closed-loop

If not, $K(s)$ needs to be dynamic

Frequency Response

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A transfer function $G(s)$:

- **frequency-response function** $G(j\omega) = |G(j\omega)|e^{\angle G(j\omega)}$
- **Nyquist Diagram** characterisation
 - $Re\{G(j\omega)\}$
 - $Im\{G(j\omega)\}$
- Real and Imaginary axis
- π to $-\pi$ clockwise
- top half is for $-\omega$, bottom half for $+\omega$

Nyquist Analysis

02 June 2018 18:42

$$F(s) = 1 + G(s) = 0 \rightarrow \text{characteristic equation of closed-loop, } H(s)$$
$$= \frac{G(s)}{1 + G(s)}$$

Z – zeros of $F(s)$ – closed loop poles – unknown

P – poles of $F(s)$ – open loop poles – known

Nyquist Contour - covers entire RHP

As s traverses Nyquist contour, let plot of F be value of $F(s)$ as s traverses

N – number of clockwise encirclement of origin by $F(s)$

$$N = Z_{RHP} - P_{RHP}$$

If we have plot of $G(s)$

- can just **number of encirclements of $-1 + 0j$** for $G(s)$ plot

Nyquist Stability Criterion

02 June 2018 18:42

- Consider the feedback loop shown in the figure.
 - Assume that $G(s)$ has no imaginary axis poles.
 - Let P be the number of unstable poles of $G(s)$.
 - Let Z denote the number of unstable poles of the closed-loop system $H(s) = G(s)/(1 + G(s))$
 - Let Γ denote the Nyquist contour.
 - Let Γ_G , called the **Nyquist diagram** of $G(s)$, denote the closed curve defined by the mapping $s \rightarrow G(s)$ as s traverses Γ clockwise.
 - Let N be the number of clockwise encirclement of the $-1 + j0$ point by Γ_G .
- Then $N = Z - P$.

Only stable when **$Z = N + P = 0$**

Nyquist Contour has 4 regions:

- the point $s=0$, $G(0) \rightarrow$ D.C. gain
- jw-axis $\rightarrow G(j\omega)$
 - use fact that $j\omega \rightarrow \infty$
 - look what happens to magnitude and phase of poles
 - phase \rightarrow **sum of effect on individual poles**
 - if net of poles is increase, angle of G decreases**
 - usual poles:**
 - 0 to 90**
 - 180 to 90**
- Infinite arc, generally maps to the origin
 - $\lim_{s \rightarrow \infty} G(s)$
- lower half of jw-axis reflect upper half (opposite direction)

find intersection with real axis:

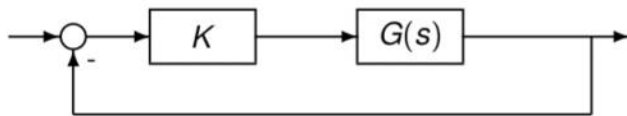
- Routh Array for $1+KG(s) = 0$, gives $G(j\omega_1)$**
 - for K making system marginally stable
- to find ω for $G(j\omega_1)$ form aux equation from first row involving K**

alternatively, set Im part of $G(j\omega)$ to 0

Then do $Z=N+P$

clockwise encirclement of $-\frac{1}{K} + j0$

- It follows that the Nyquist diagram of $KG(s)$ encircles the $-1 + j0$ point if and only if the Nyquist diagram of $G(s)$ encircles the $-\frac{1}{K} + j0$ point, and we have the following modified Nyquist criterion.
- Consider the feedback loop in the figure. Let
 - ① P be the number of unstable poles of $G(s)$.
 - ② Z be the number of unstable closed-loop poles.
 - ③ N be the number of clockwise encirclement of the $-\frac{1}{K} + j0$ point by the Nyquist diagram of $G(s)$.
- Then $N = Z - P$.
- That is, the closed-loop is stable if and only if $-N = P$.



Relative Stability

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gain margin - factor to make a stable closed-loop gain become marginally stable

at least 2

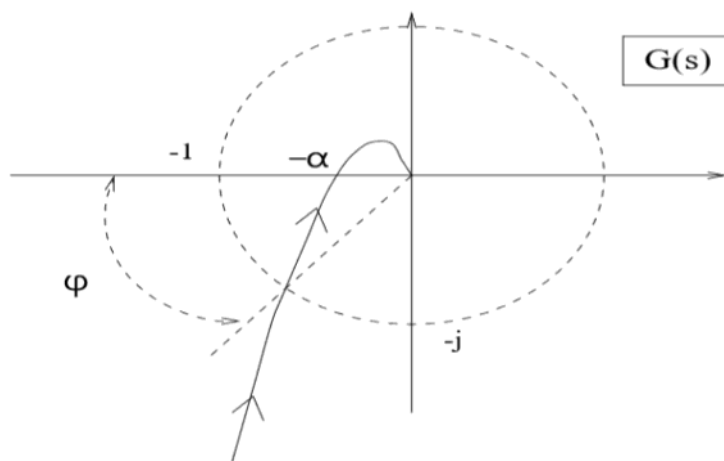
phase margin - min angle the Nyquist diagram must be rotated to intersect with $-1 + j0$

without changing the gain

at least 45 degree

set $|G(j\omega)| = 1$

solve for ω_1 , sub into $\angle G(j\omega)$



Gain Margin = $1/\alpha$, Phase Margin = ϕ

Prototype feedback system

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$Q(s) = G(s)K_c(s)$: open – loop gain

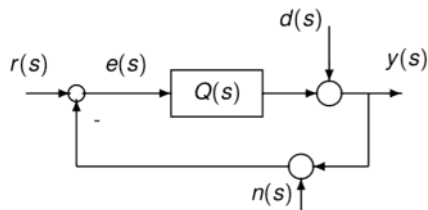
$r(s)$: reference signal

$y(s)$: output signal

$e(s)$: error signal

$d(s)$: output disturbance

$n(s)$: sensor noise



Error signal:

$$e(s) = \frac{1}{1 + Q(s)} [r(s) - d(s) - n(s)]$$

Output signal:

$$y(s) = \frac{1}{1 + Q(s)} d(s) + \frac{Q(s)}{1 + Q(s)} [r(s) - n(s)]$$

Closed-loop TF:

$$H(s) = \frac{Q(s)}{1 + Q(s)}$$

Sensitivity TF:

$$S(s) = \frac{1}{1 + Q(s)}$$

$$H(s) + S(s) = 1$$

Design Objectives

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Steady-state accuracy:

- need y to track r , so want
- e_{ss} **to be small**
- if input $r(t) = e^{j\omega_0 t}$
 - $e_{ss}(t) = S(j\omega_0)e^{j\omega_0 t}$
 - need $|S(j\omega_0)|$, the steady-state gain of error at frequency ω_0 , to be $\ll 1$
 - hence, $Q(j\omega_0) \gg 1$
- good steady-state accuracy requires **large open-loop gain** over wide range of frequencies
 - **large closed-loop bandwidth**

Output disturbance rejection:

- **also requires large open-loop gain**
 - same as large closed-loop bandwidth

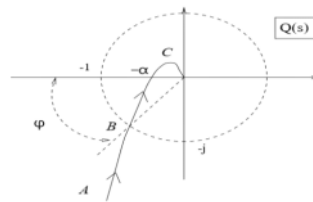
Sensor noise attenuation

- Need $Q(j\omega_0) \ll 1$
- requires small open-loop gain
 - same as small closed-loop bandwidth

Trade-off:

- SSA and ODR conflict with SNA
- usually noise signal has much higher frequency
- so:
 - **Large gain at low frequencies**
 - **Small gain at high frequencies**
 - **usually aim for 40dB decrease per decade after, i.e. $1/s^2$**

Transient Response and Stability margins:



The diagram is divided into three regions:

- A is the low frequency region ($|Q(j\omega)| \gg 1$).
- B is the **crossover** region ($|Q(j\omega)| \approx 1$).
- C is the high frequency region ($|Q(j\omega)| \ll 1$).

Also want low open-loop gain at high frequency ($s \sim 0$)
to have good gain and phase margin

Shape **transient** response by **regions B and C**

Steady-state response from region A

Compensation

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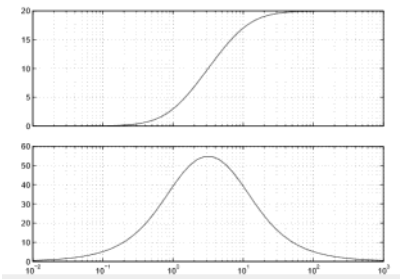
Constant, $K_C(s) = K$

- multiplies gain uniformly for all frequencies
- if $K < 1$
 - **increase gain and phase margins**
 - **reduce overshoot** and closed-loop bandwidth
- but
 - **slower response time** - increase in rise time

Phase-lead controller: - hard to balance

$$K_C(s) = K \frac{1 + s/\omega_0}{1 + s/\omega_p} = K \frac{\omega_p s + \omega_0}{\omega_0 s + \omega_p}, \quad \omega_0 < \omega_p$$

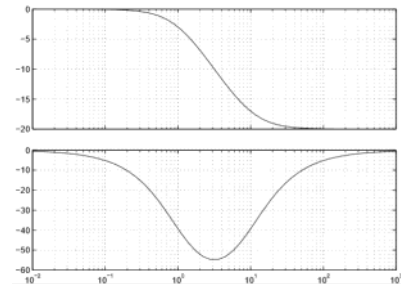
- increasing gain above ω_p
 - bad for margins
- adds **phase-lead (+)** between ω_0 and ω_p
 - so choose ω_0 and ω_p to be in the crossover region B
 - Crossover freq = **geometric mean** between ω_0 and ω_p



Phase-lag controller: - choose K to increase low freq. gain

$$K_C(s) = K \frac{1 + s/\omega_0}{1 + s/\omega_p} = K \frac{\omega_p s + \omega_0}{\omega_0 s + \omega_p}, \quad \omega_p < \omega_0$$

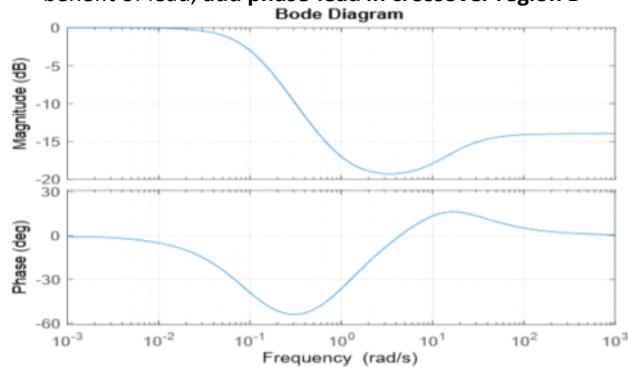
- decreasing gain above ω_0
 - good for margins
- adds **phase-lag (-)** between ω_p and ω_0
 - so choose ω_p and ω_0 to be in the lower frequency range A



Lag-lead controller:

$$K_C(s) = K \frac{1 + s/\omega_{0_1}}{1 + s/\omega_{p_1}} \frac{1 + s/\omega_{0_2}}{1 + s/\omega_{p_2}}, \quad \omega_{0_1} > \omega_{p_1}, \omega_{0_2} < \omega_{p_2}$$

- Main benefits of Lag, **decrease gain, add lag in region A**
- benefit of lead, **add phase-lead in crossover region B**



Ziegler-Nichols tuning rules

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Special form of	Name	K(s)
lag	PI - improve steady state accuracy up to a point much lower than crossover frequency	$K_p + \frac{K_i}{s}$
lead	PD - add phase lead at crossover, but can cause instability via high freq. gain	$K_p + K_d s$
lag-lead	PID - best of both	$K_p + \frac{K_i}{s} + K_d s$

For a G(s) which is

- stable
- Type 0 - rules out PD
- Overdamped (real poles and zeros)

By experimenting with a constant gain compensator, get K_{po} and T_o

- when closed-loop becomes marginally stable
 - K_{po} value of the constant
 - T_o period of oscillations

Compensator is defined either by:

- **P:** $K(s) = 0.5K_{po}$

- **PI:** $K(s) = 0.45K_{po} + \frac{0.54K_{po}/T_o}{s}$

- **PID:** $K(s) = 0.6K_{po} + \frac{1.2K_{po}/T_o}{s} + 0.075K_{po}T_o s$