TZ SOLUTIONS (WATTZ)

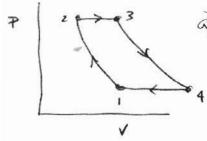
(1 OF 2)

MY DATA WERE OBTAINED 9/16/03 16:50 hrs.

AMBIENT AIR TEMP = 75°F = 297K
ATMORPHERIC PRESSURE (ASSUMED) = 19tm = 101.3 kPg

COMPRESSOR DELIVERY PRESSURE = 200 PSIG = 214.77 PSIA ABSOLUTE (relative to

atmospheric) SO PRESSURE RATIO = 214.77 = 14.54



a) 0 P = 101.3 Hz, T = 297K

(2) $P_2 = 14.54 (101.3) = 1473 kPa$ $\frac{P_2}{P_1} r^{-1} k_{=} \frac{T_2}{T_1} = 2.149 i. T_2 = 638 K$ as adiab.

3 constant pressure heating

@ g-s adiabatic expansion by P3=14.54 :. Pa=101.3 kPa

$$\left(\frac{P_4}{P_3}\right)^{8-1/8} = \frac{T_4}{T_3} = 0.465$$
; $T_4 = 652 \text{ K}$

b) THERMAL EFFICIENCY

 $W = C_{p} \left(T_{3} - T_{z} + T_{1} - T_{4} \right) = 1003.5 \left(1400 - 638 + 247 - 652 \right) = 408 \frac{1}{44}$

C) Tz FIXED. COLD DAY TI = Z73K

HOT DAY TI = Z03K

$$N_{SID} = 1 - \frac{273}{638} = 0.57 \qquad y_{HOT} = 1 - \frac{303}{638} = 0.525$$
d) TOTAL GAS ENERGY FLOW = 66x10³ BTU/S = 69.63 MJ/S

ACTIVE LOAD = ZOMW

$$M = \frac{20}{69.63} = \frac{Vkycle}{QIN} = 0.287$$
A VARIETY OF NON-IDEAL PROCESSES CAUSE THE EFFICIENCY TO BSE SIGNIFICANTLY LESS THAN THE VALUE OBTAINED FOIL THE IDEAL CYCLE.

2) FOR PRESSURE RATIO OF 14.54, CALCULATED Tz = 2.149

$$MEASURED T_2 (COMPR DISCHARGE TEMP) = 730 °F$$

*HIGHER THAN DIAB PROCEST TO MEAS. = 661K

BUT NOTE THERE IS ALSO SOME COOLING BETWEEN THE AMBIENT (TETTS F) AND THE COMPRESSOR INLET

(COMP INVET TEMP= 62°F) WHICH WAS NOT ACCOUNT FOR THIS OUR G-S, ADIABATIC MODEL IS EVEN WORSE

(i.e. SHOWS A SMALLER AT THAN IN REAL DENCE)