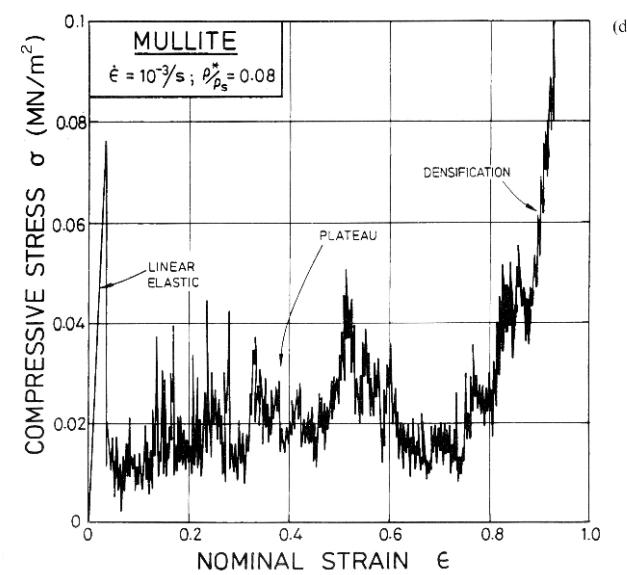
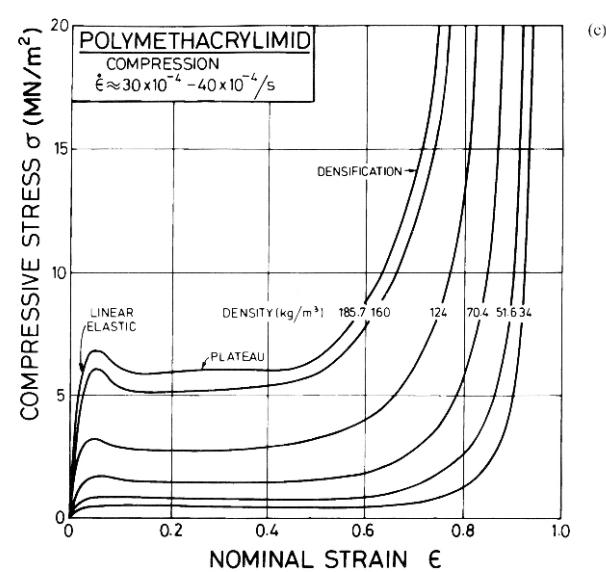
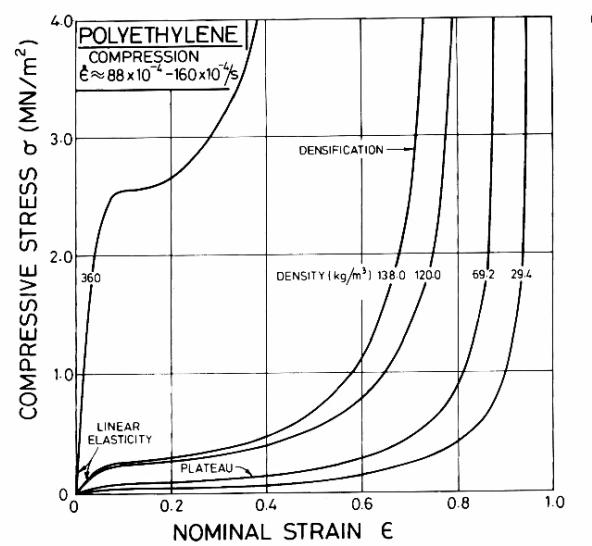
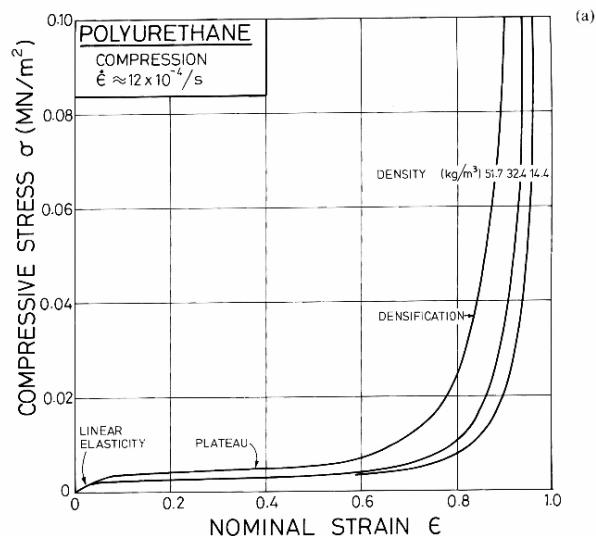


## Open-cell foams

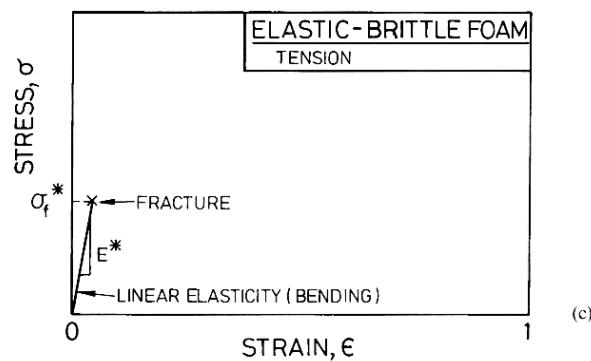
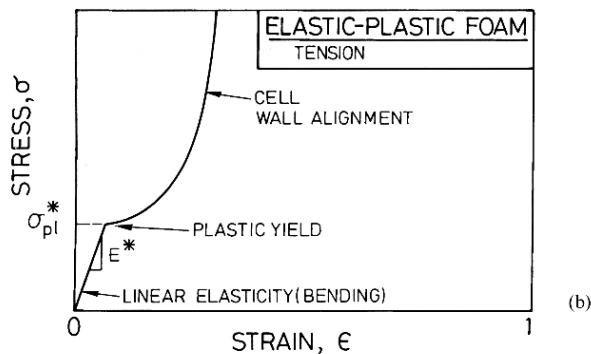
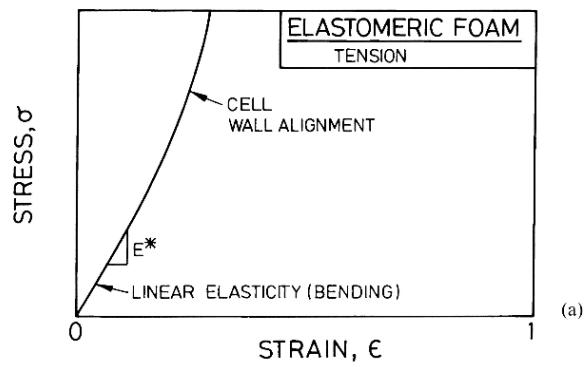
- stress-strain curves: deformation + failure mechanisms
  - compression - 3 regimes - linear elastic - bending
    - stress plateau - cell collapse by buckling
    - yielding
    - crushing
  - tension - no buckling
    - yielding can occur
    - brittle fracture
- 

## Linear elastic behaviour

- initial linear elasticity - bending of cell edges (small  $t/l$ )
- as  $t/l \uparrow$ , axial deformation becomes more significant
- consider dimensional argument, which models mechanism of deformation + failure, but not cell geometry
- Consider cubic cell, square cross-section members of area  $t^2$ , length  $l$



Gibson, L. J., and M. F. Ashby. *Cellular Solids: Structure and Properties*. 2nd ed. Cambridge University Press, © 1997. Figures courtesy of Lorna Gibson and Cambridge University Press.

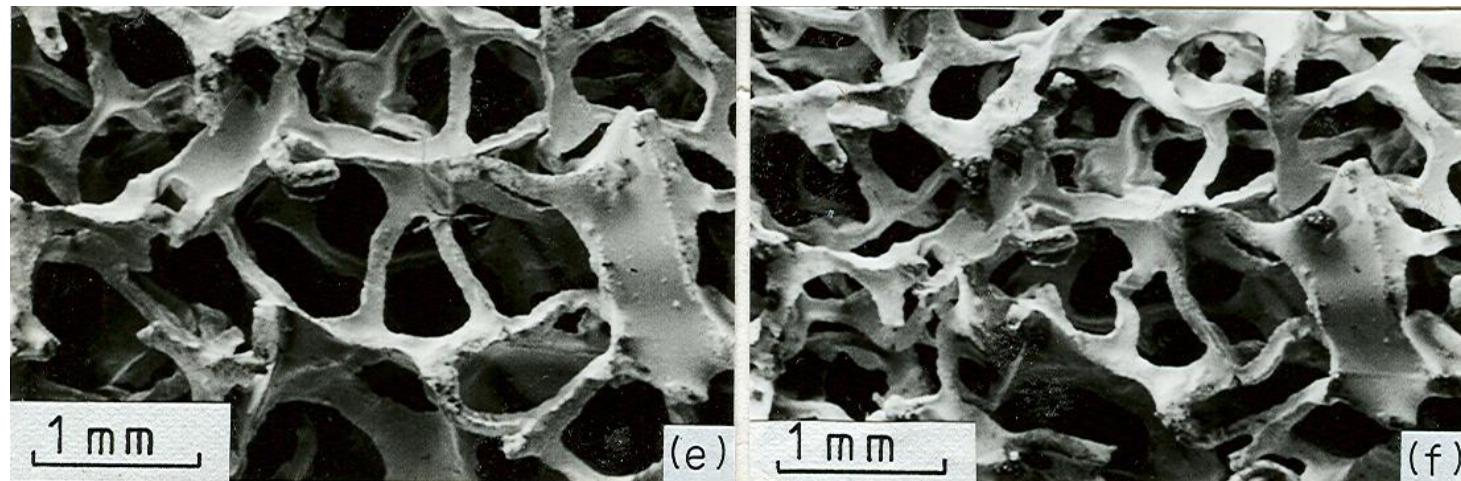


Gibson, L. J., and M. F. Ashby. *Cellular Solids: Structure and Properties*. 2nd ed. Cambridge University Press, © 1997. Figure courtesy of Lorna Gibson and Cambridge University Press.

# Foams: Bending, Buckling

Figure removed due to copyright restrictions. See Fig. 3: Gibson, L. J., and M. F. Ashby. "[The Mechanics of Three-Dimensional Cellular Materials](#)." *Proceedings of The Royal Society of London A* 382 (1982): 43-59.

# Foams: Plastic Hinges



Gibson, L. J., and M. F. Ashby. *Cellular Solids: Structure and Properties*. 2nd ed. Cambridge University Press, © 1997. Figure courtesy of Lorna Gibson and Cambridge University Press.

# Foams: Cell Wall Fracture

Figure removed due to copyright restrictions. See Fig. 3: Gibson, L. J., and M. F. Ashby. "[The Mechanics of Three-Dimensional Cellular Materials](#)." *Proceedings of The Royal Society of London A* 382 (1982): 43-59.

- regardless of specific cell geometry chosen:

$$\rho^*/\rho_s \propto (t/l)^2 \quad I \propto t^4$$

$$\sigma \propto F/l^2 \quad \epsilon \propto \delta/l \quad \delta \propto Fl^3/E_s I$$

$$E^* \propto \frac{\sigma}{\epsilon} \propto \frac{F}{l^2} \frac{l}{\delta} \propto \frac{F}{l} \frac{E_s t^4}{Fl^3} \propto E_s \left(\frac{t}{l}\right)^4 \propto E_s \left(\frac{\rho^*}{\rho_s}\right)^2$$

$$E^* = C_1 E_s \left(\frac{\rho^*}{\rho_s}\right)^2 \quad C_1 \text{ includes all geometrical constants.}$$

Data:  $C_1 \approx 1$

- data suggest  $C_1 = 1$
- analysis of open cell tetrakaidecahedral cells with Plateau borders gives  $C_1 = 0.98$
- shear modulus  $G^* = C_2 E_s \left(\frac{\rho^*}{\rho_s}\right)^2 \quad C_2 \sim 3/8$  if foam isotropic

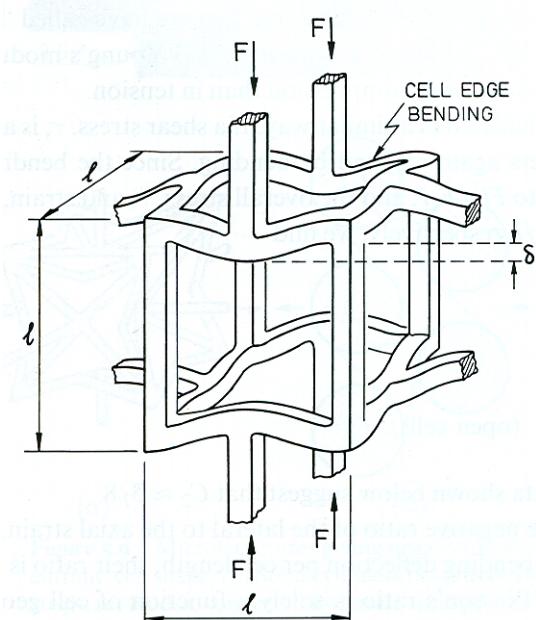
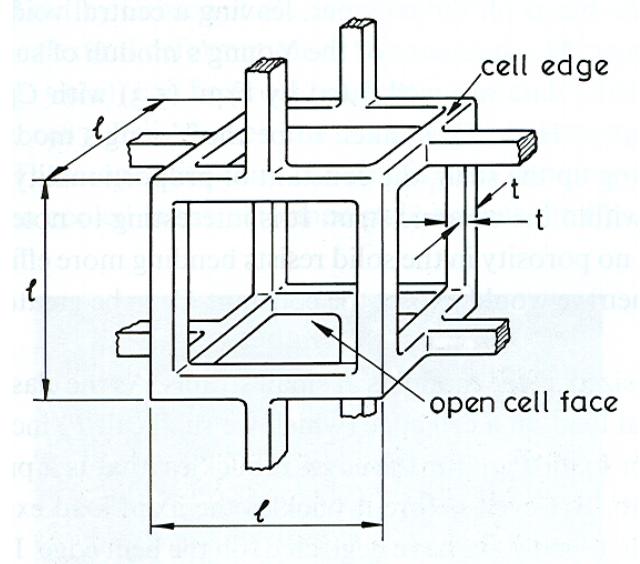
isotropy:  $G = \frac{E}{2(1+\nu)}$

- Poisson's ratio  $\nu^* = \frac{E}{2G} - 1 = \frac{C_1}{2C_2} - 1 = \text{constant, independent of } E_s, t/l$

$$\nu^* = C_3$$

(analogous to honeycombs in-plane)

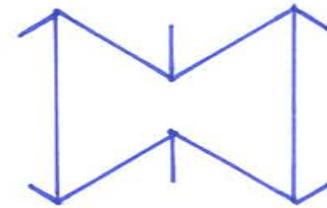
# Foam: Edge Bending



Gibson, L. J., and M. F. Ashby. *Cellular Solids: Structure and Properties*. 2nd ed. Cambridge University Press, © 1997. Figure courtesy of Lorna Gibson and Cambridge University Press.

## Poisson's ratio

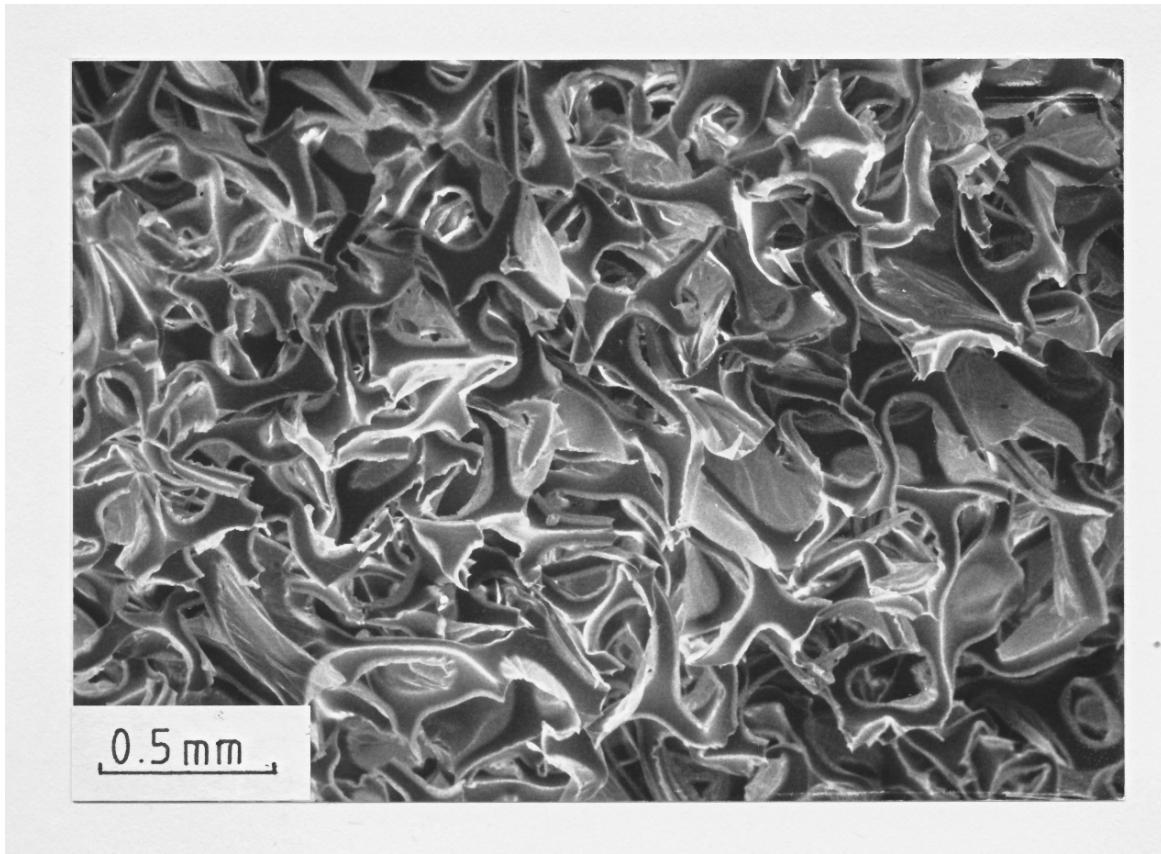
- can make negative Poisson's ratio foams
- invert cell angles (analogous to honeycomb)
- eg. thermoplastic foams - load hydrostatically + heat to  $T > T_g$ , then cool + release load  
so that edges of cell permanently point inward



## Closed-cell foams

- edge bending as for open-cell foams
- also: face stretching + gas compression
- polymer foams: surface tension draws material to edges during processing
  - define  $t_e$ ,  $t_f$  in figure
- apply  $F$  to the cubic structure

# Negative Poisson's Ratio



Gibson, L. J., and M. F. Ashby. *Cellular Solids: Structure and Properties*. 2nd ed. Cambridge University Press, © 1997. Figure courtesy of Lorna Gibson and Cambridge University Press.

• external work done  $\propto F\delta$

• internal work bending edges  $\propto \frac{F_e}{\delta e} \delta e^2 \propto \frac{E_s I}{l^3} \delta^2$

• internal work stretching faces  $\propto \sigma_f \epsilon_f V_f \propto E_s \epsilon_f^2 V_f \propto E_s (\delta/l)^2 t_f l^2$

$$\therefore F\delta = \alpha \frac{E_s t_e^4}{l^3} \delta^2 + \beta E_s \left(\frac{\delta}{l}\right)^2 t_f l^2$$

$$E^* \propto \frac{F}{l^2} \frac{l}{\delta} \Rightarrow F \propto E^* \delta l$$

$$\therefore E^* \delta^2 l = \alpha \frac{E_s t_e^4}{l^3} \delta^2 + \beta E_s \left(\frac{\delta}{l}\right)^2 t_f l^2$$

$$E^* = \alpha E_s \left(\frac{t_e}{l}\right)^4 + \beta E_s \left(\frac{t_f}{l}\right)^2$$

Note: open cells, uniform : if  $\phi$  = volume fraction of solid in cell edges:

$$\rho^* / \rho_s \propto (t/l)^2 \quad t_e/l = C \phi^{1/2} (\rho^*/\rho_s)^{1/2}$$

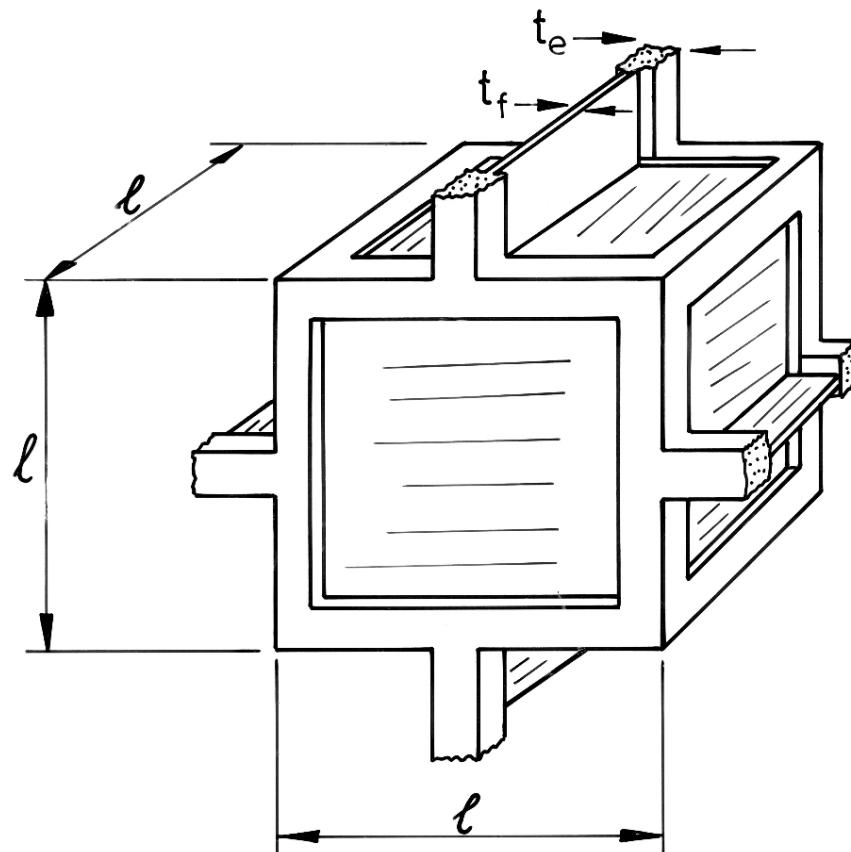
Closed cells, uniform

$$\rho^* / \rho_s \propto (t_e)$$

$$t_f/l = C' (1-\phi) (\rho^*/\rho_s)$$

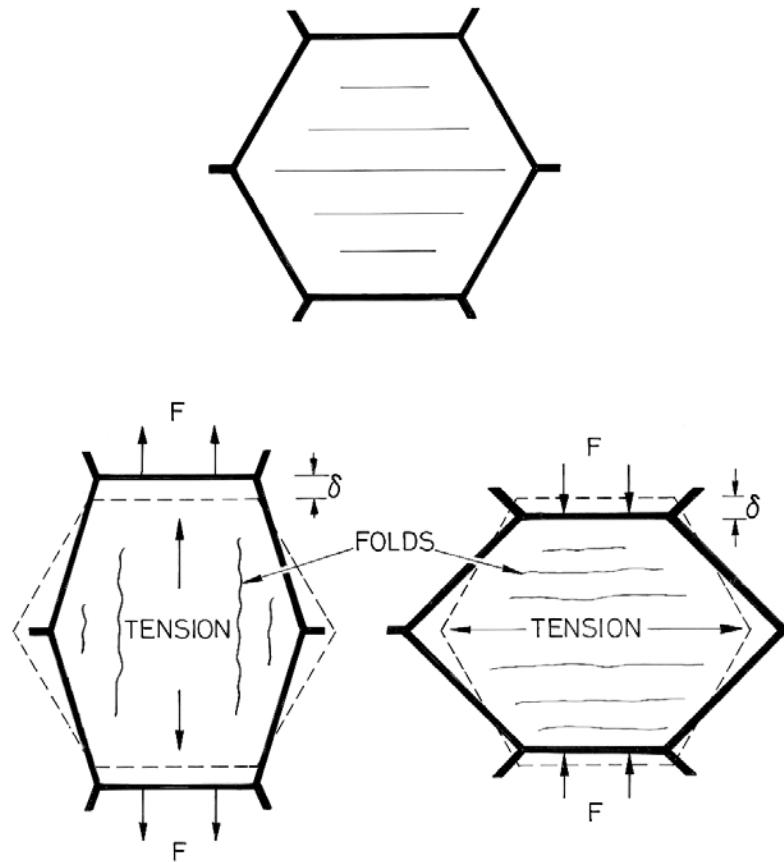
$$\boxed{\frac{E^*}{E_s} = C_1 \phi^2 \left(\frac{\rho^*}{\rho_s}\right)^2 + C'_1 (1-\phi) \frac{\rho^*}{\rho_s}}$$

# Closed-Cell Foam



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# Cell Membrane Stretching



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Closed cell foams - gas within cell may also contribute to  $E^*$

- cubic element of foam of volume  $V_0$
- under uniaxial stress, axial strain in direction of stress is  $\epsilon$
- deformed volume  $V$  is:

$$\frac{V}{V_0} = 1 - \epsilon(1 - 2v^*)$$

taking compressive strain as positive  
neglecting  $\epsilon^2, \epsilon^3$  terms

$$\frac{V_g}{V_g^0} = \frac{1 - \epsilon(1 - 2v^*) - \rho^*/\rho_s}{1 - \rho^*/\rho_s}$$

$V_g$  = volume gas  
 $V_g^0$  = " " initially

- Boyle's law:  $p V_g = p_0 V_g^0$
- pressure that must be overcome is  $p' = p - p_0$

$$p' = \frac{p_0 \epsilon (1 - 2v^*)}{1 - \epsilon(1 - 2v^*) - \rho^*/\rho_s}$$

$p$  = pressure after strain  $\epsilon$   
 $p_0$  = pressure initially

- contribution of gas compression to the modulus,  $E^*$ :

$$E_g^* = \frac{dp'}{d\epsilon} = \frac{p_0 (1 - 2v^*)}{1 - \rho^*/\rho_s}$$

(4a)



$$V_0 = l_0^3$$

$$\epsilon_1 = \frac{l_1 - l_0}{l_0} \Rightarrow l_1 = l_0 + \epsilon_1 l_0 = l_0(1 + \epsilon_1)$$

$$V = l_1 l_2 l_3$$

$$\epsilon_2 = \frac{l_2 - l_0}{l_0} \Rightarrow l_2 = l_0 + \epsilon_2 l_0 \quad \nu = -\frac{\epsilon_2}{\epsilon_1}$$

$$= l_0 - \nu \epsilon_1 l_0 \quad \epsilon_2 = -\nu \epsilon_1$$

$$= l_0 (1 - \nu \epsilon_1)$$

$$G_3 = l_0 (1 - \nu \epsilon_1).$$

$$V = l_1 l_2 l_3 = l_0 (1 + \epsilon_1) l_0 (1 - \nu \epsilon_1) l_0 (1 - \nu \epsilon) = l_0^3 (1 + \epsilon_1) (1 - \nu \epsilon)^2$$

$$\frac{V}{V_0} = \frac{l_0^3 (1 + \epsilon_1) (1 - \nu \epsilon)^2}{l_0^3} = \frac{(1 + \epsilon_1) (1 - 2\nu \epsilon + \nu^2 \epsilon^2)}{(1 + \epsilon_1)}$$

$$= (1 - 2\nu \epsilon + \cancel{\nu^2 \epsilon^2}) + \epsilon - \cancel{2\nu \epsilon} + \cancel{\nu^2 \epsilon^2}$$

$$= 1 \cancel{\epsilon} + 2\nu \epsilon \quad / \text{taking comp. as } +)$$

$$= 1 - \epsilon (1 - 2\nu)$$

(5b)

$$p' = p - p_0$$

$$p = \frac{p_0 V_g^*}{V_f}$$

$$p' = p - p_0 = \frac{p_0 V_g^*}{V_f} - p_0 = p_0 \left( \frac{V_g^*}{V_f} - 1 \right) \quad \cancel{\text{CPD} \left( \frac{V_g^* - V_f}{V_f} \right)}$$

$$= p_0 \left[ \frac{1 - \rho/p_s}{1 - \epsilon(1 - 2\rho^*) - \rho^*/p_s} - 1 \right]$$

$$= p_0 \left[ \frac{\frac{1 - \rho/p_s}{1 - \epsilon(1 - 2\rho^*) - \rho^*/p_s} - \left( 1 - \epsilon(1 - 2\rho^*) - \frac{\rho^*}{p_s} \right)}{1 - \epsilon(1 - 2\rho^*) - \rho^*/p_s} \right]$$

$$= p_0 \left[ \frac{\epsilon(1 - 2\rho^*)}{1 - \epsilon(1 - 2\rho^*) - \rho^*/p_s} \right] \checkmark$$

## Closed cell foam

$$\frac{E^*}{E_s} = \phi^2 \left( \frac{\rho^*}{\rho_s} \right)^2 + (1-\phi) \left( \frac{\rho^*}{\rho_s} \right) + \frac{p_0 (1-2v^*)}{E_s (1-\rho^*/\rho_s)}$$

↓                    ↓                    ↓  
 edge bending      face stretching    gas compression

- note: if  $p_0 = p_{atm} = 0.1 \text{ MPa}$ , gas compression term negligible,  
except for closed-cell elastomeric foams
- gas comp. can be significant if  $p_0 \gg p_{atm}$ ; also modifies shape of stress plateau  
in elastomeric closed cell foams

Shear modulus: edge bending, face stretching; shear  $\Delta V=0$  gas contrib. = 0

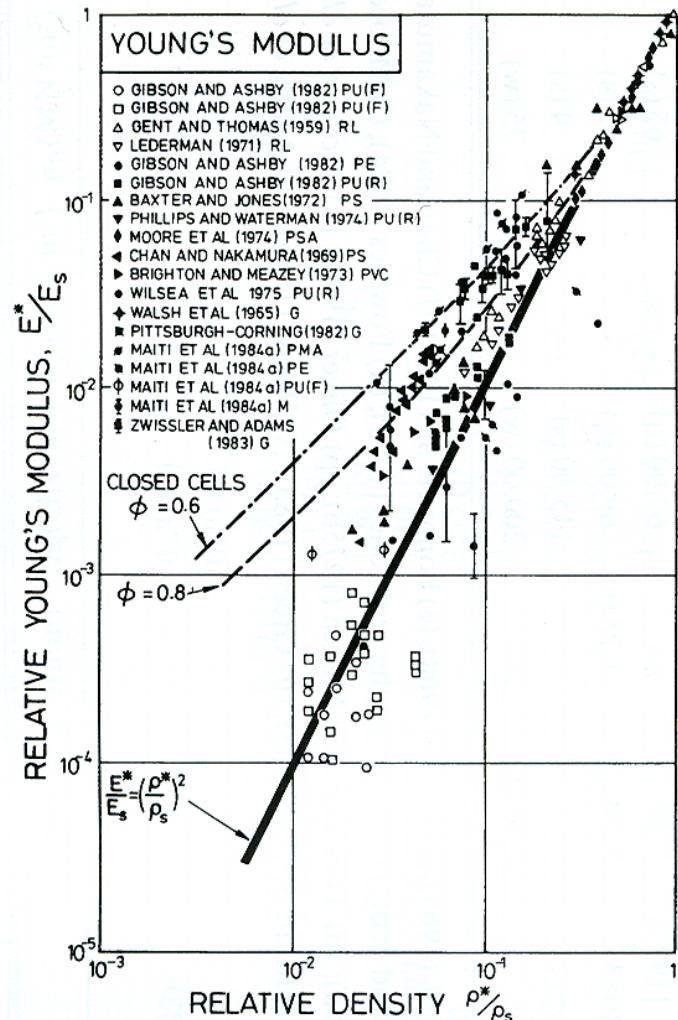
$$\frac{G^*}{E_s} = \frac{3}{8} \left[ \phi^2 \left( \frac{\rho^*}{\rho_s} \right)^2 + (1-\phi) \left( \frac{\rho^*}{\rho_s} \right) \right] \quad (\text{isotropic foam})$$

Poisson's ratio = f (cell geometry only)       $v^* \approx 1/3$

## Comparison with data

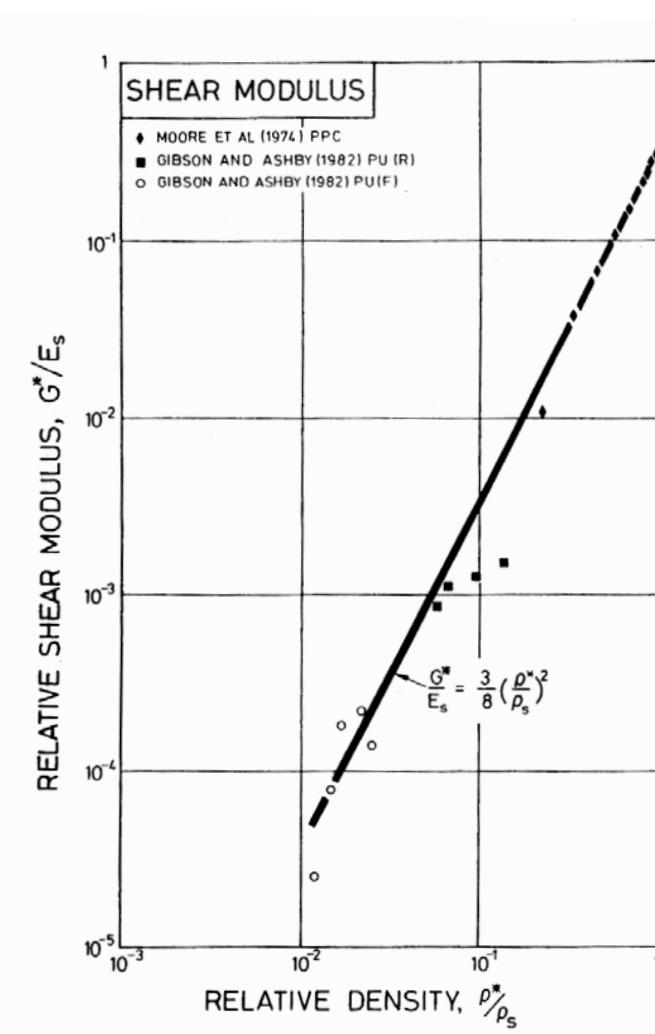
- data for polymers, glasses, elastomers
- $E_s, \rho_s$  Table S.1 in book
- open cells - open symbols
- closed cells - filled symbols

# Young's Modulus



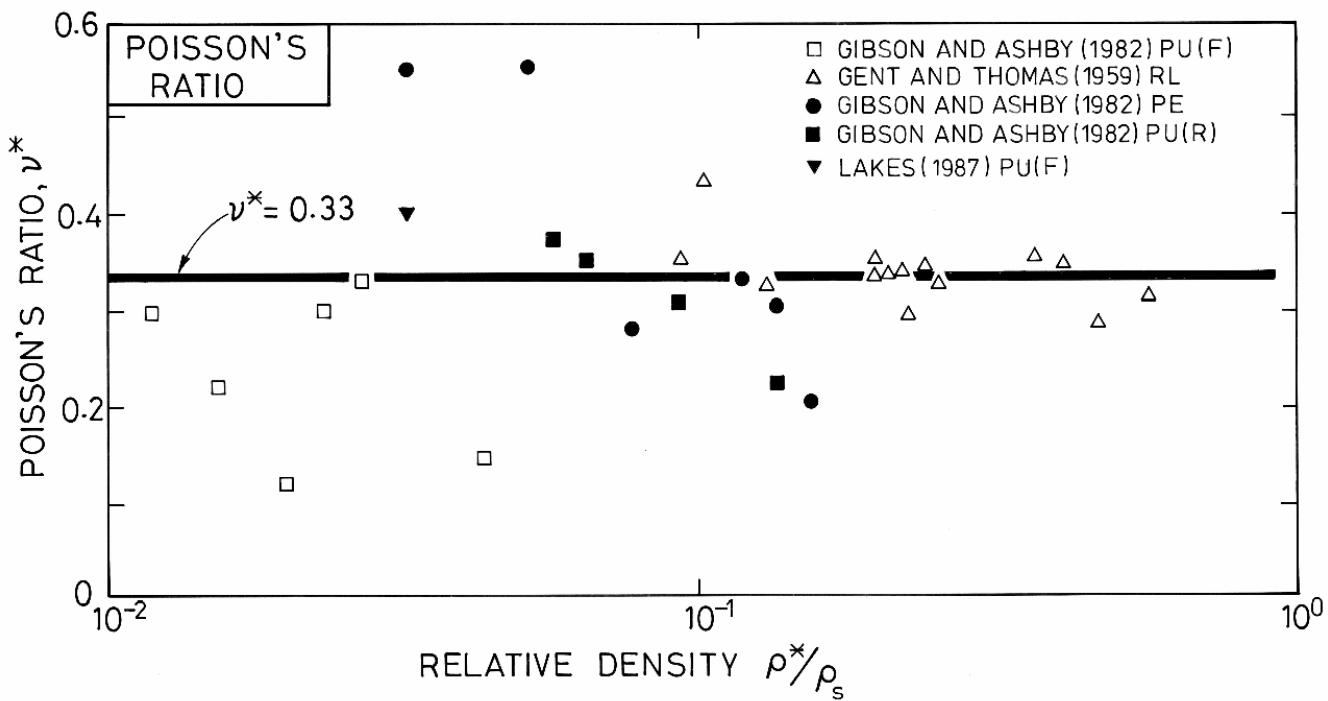
Gibson, L. J., and M. F. Ashby. *Cellular Solids: Structure and Properties*. 2nd ed. Cambridge University Press, © 1997. Figure courtesy of Lorna Gibson and Cambridge University Press.

# Shear Modulus



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# Poisson's Ratio



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## Non-linear elasticity

### Open cells:

$$P_{cr} = \frac{n^2 \pi^2 E_s I}{l^2}$$

$$\sigma_{el}^* \propto \frac{P_{cr}}{l^2} \propto E_s \left(\frac{l}{\ell}\right)^4$$

$$\sigma_{el}^* = C_4 E_s (\ell^* / \rho_s)^2$$

Data:  $C_4 \approx 0.05$ , corresponds to strain when buckling initiates, since  $E^* = E_s (\ell^* / \rho_s)^2$

### Closed cells

- $t_f$  often small compared to  $t_e$  (surface tension in processing) - contrib. small
- if  $p_o \gg p_{atm}$ , cell walls pretensioned; buckling stress has to overcome this

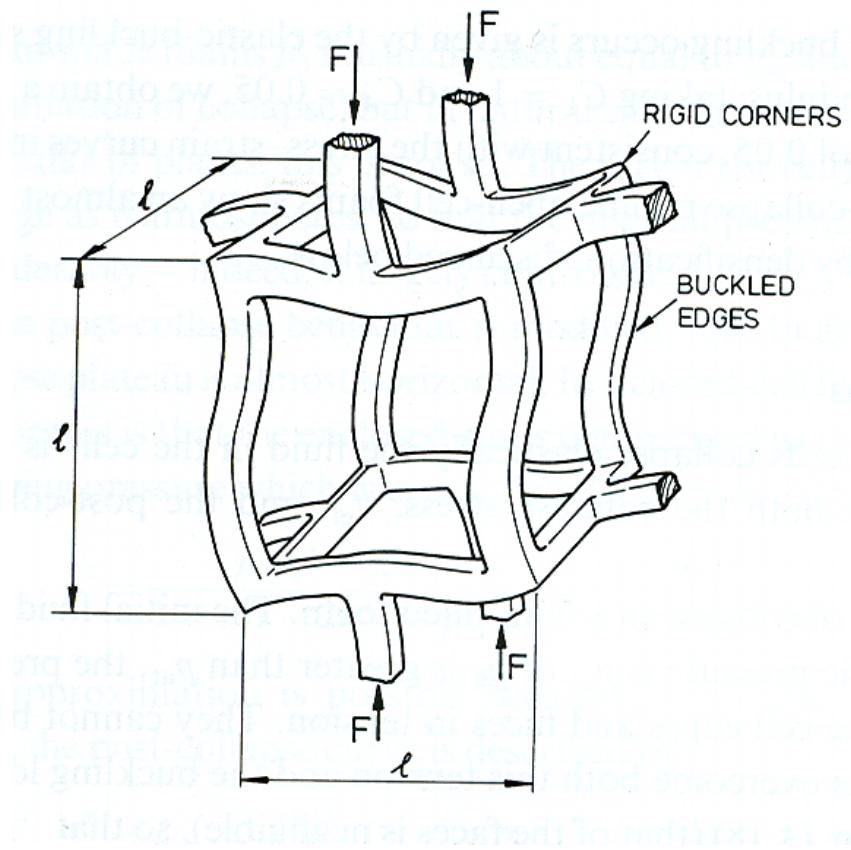
$$\sigma_{el}^* = 0.05 E_s (\ell^* / \rho_s)^2 + P_o - P_{atm}$$

- post-collapse behaviour - stress plateau rises due to gas compression (if faces don't rupture)  $\nu^* = 0$  in post-collapse regime

$$P' = \frac{P_o \epsilon (1 - 2\nu^*)}{1 - \epsilon(1 - 2\nu^*) - \rho^* / \rho_s} = \frac{P_o \epsilon}{1 - \epsilon - \rho^* / \rho_s}$$

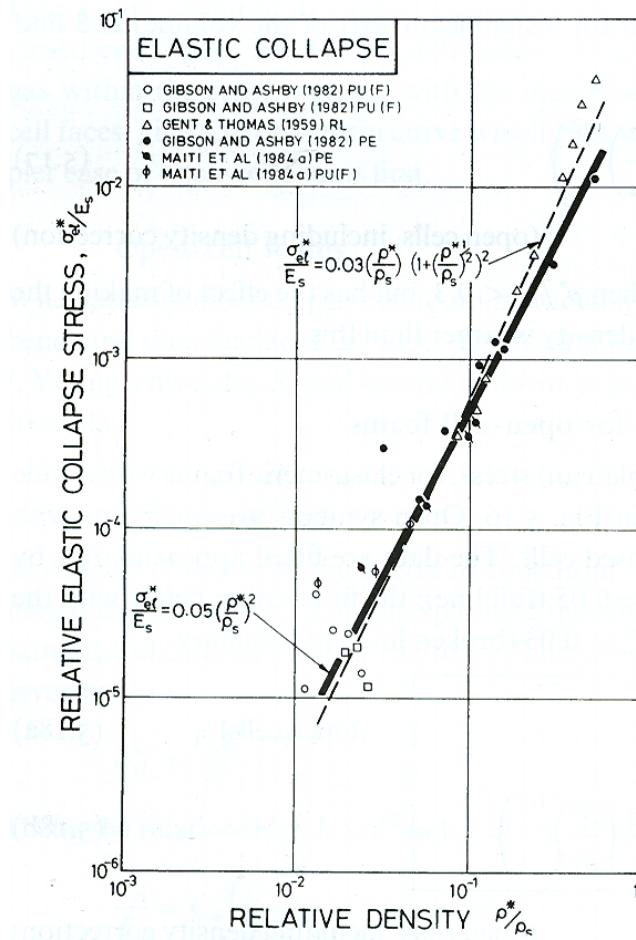
$$\sigma_{post-collapse}^* = 0.05 E_s \left(\frac{\ell^*}{\rho_s}\right)^2 + \frac{P_o \epsilon}{1 - \epsilon - \rho^* / \rho_s}$$

# Elastic Collapse Stress



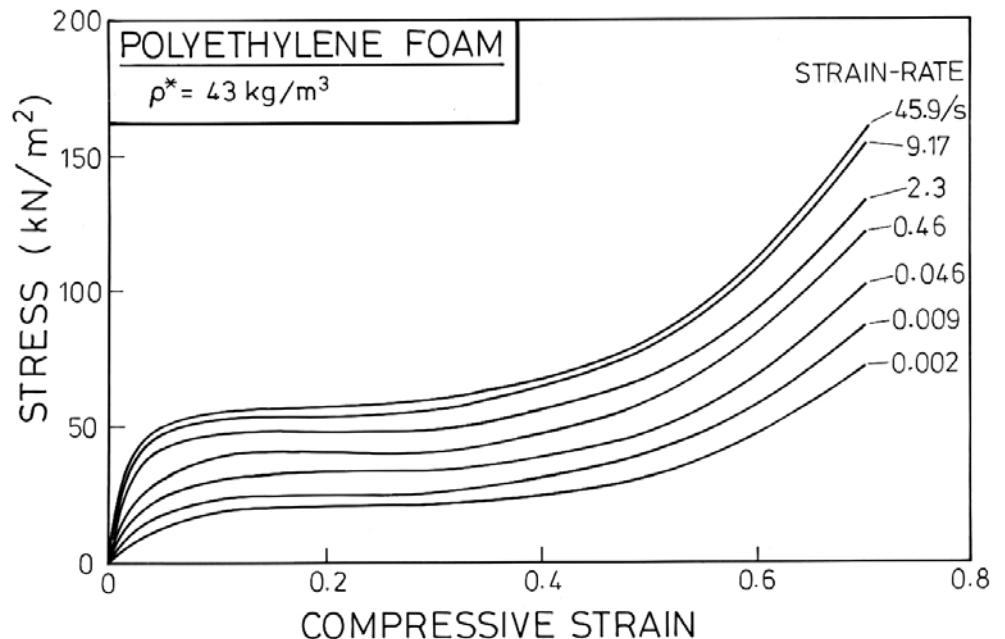
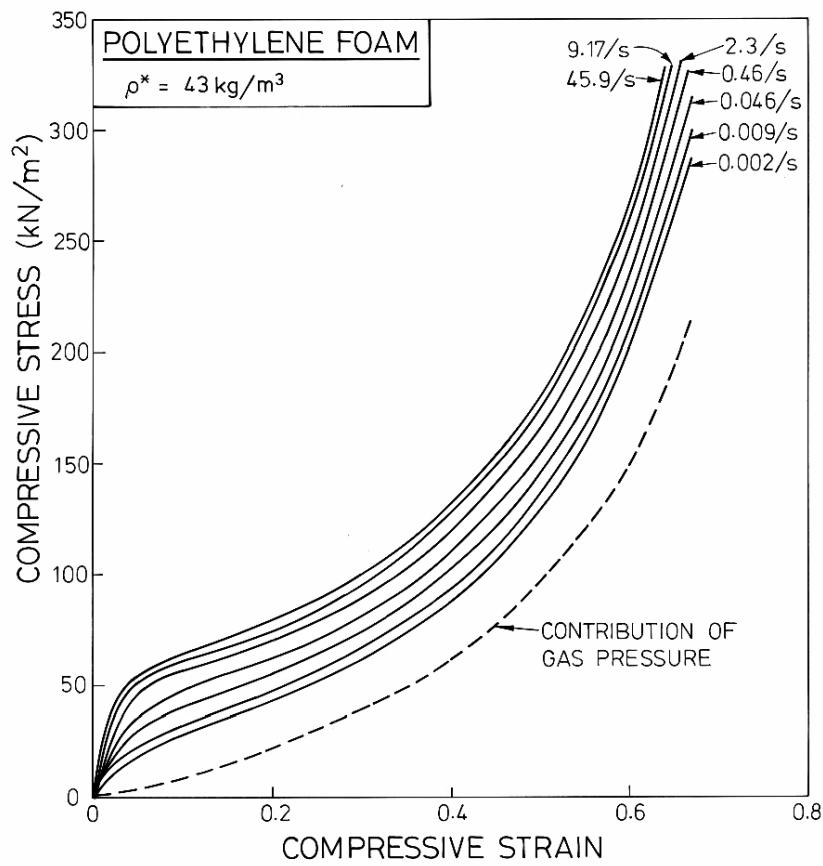
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# Elastic Collapse Stress



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# Post-collapse stress strain curve



## Plastic collapse

### Open cells

- failure when  $M = M_p$
- $M_p \propto \sigma_{ys} t^3$        $M \propto \sigma_{pl}^* l^3$

$$\boxed{\sigma_{pl}^* = C_s \sigma_{ys} (\rho^*/\rho_0)^{3/2}} \quad C_s \approx 0.3, \text{ from data.}$$

- elastic collapse precedes plastic collapse if  $\sigma_{el}^* < \sigma_{pl}^*$

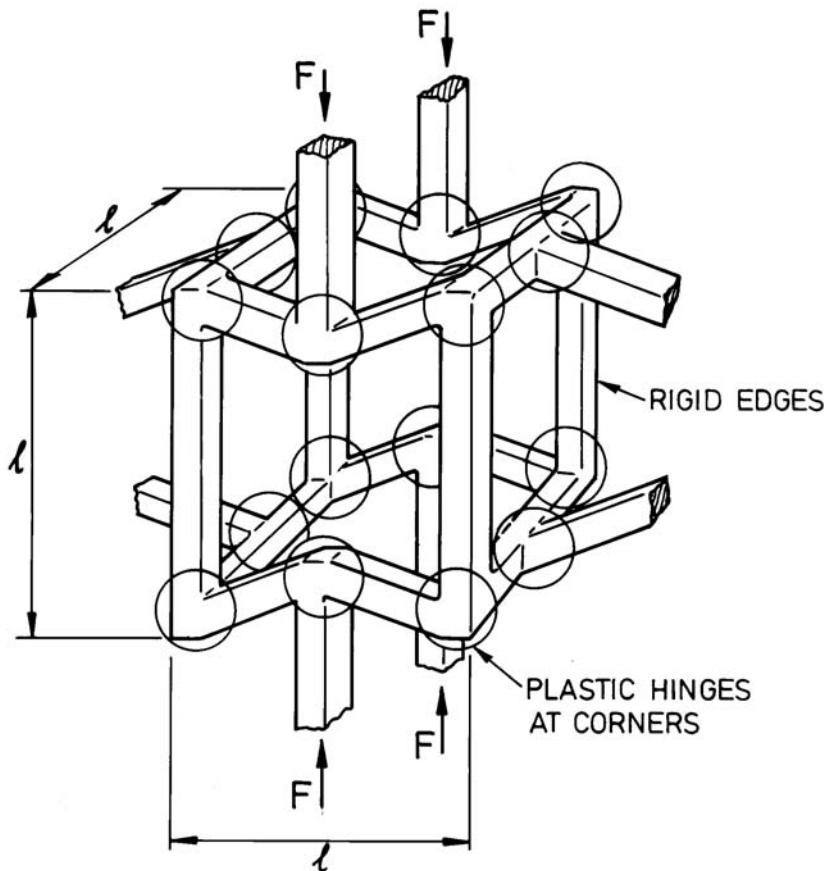
$$0.05 E_s (\rho^*/\rho_0)^2 \leq 0.3 \sigma_{ys} (\rho^*/\rho_0)^{3/2} \quad \text{rigid polymers } (\rho^*/\rho_0)_{cr} < 0.04 \quad \left( \frac{\sigma_{ys}}{E_s} \approx \frac{1}{30} \right)$$

$$(\rho/\rho_0)_{critical} \leq 36 (\sigma_{ys}/E_s)^2 \quad \text{metals } (\rho^*/\rho_0)_c < 10^{-5} \quad \left( \frac{\sigma_{ys}}{E_s} \approx \frac{1}{1000} \right)$$

### Closed cells

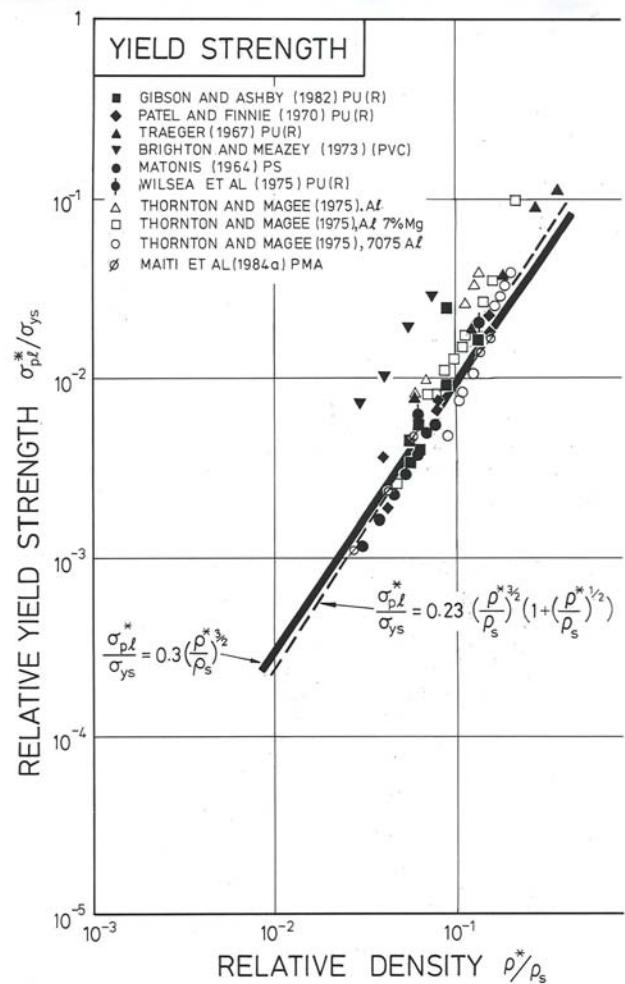
- including all terms  $\sigma_{pl}^* = C_s \sigma_{ys} (\phi \rho^*/\rho_0)^{3/2} + C_s \sigma_{ys} (1-\phi) (\rho/\rho_0) + P_i - P_{atm}$ 
  - $\uparrow$  edge bending
  - $\uparrow$  face stretching
  - $\uparrow$  gas
- but in practice, faces often rupture around  $\sigma_{pl}^*$  - often  $\sigma_{pl}^* = 0.3 (\rho^*/\rho_0)^{3/2} \sigma_{ys}$

# Plastic Collapse Stress



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# Plastic Collapse Stress



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## Brittle crushing strength

### Open cells

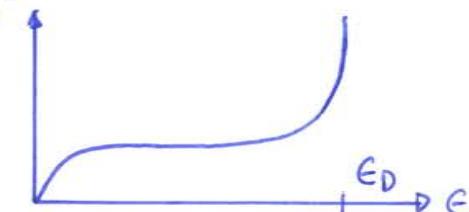
- failure when  $M = M_f \quad M \propto \sigma_{cr}^* l^3 \quad M_f \propto \sigma_{fs} t^3$

$$\sigma_{cr}^* = C_6 \sigma_{fs} \left( \rho^*/\rho_s \right)^{3/2} \quad C_6 \approx 0.2$$

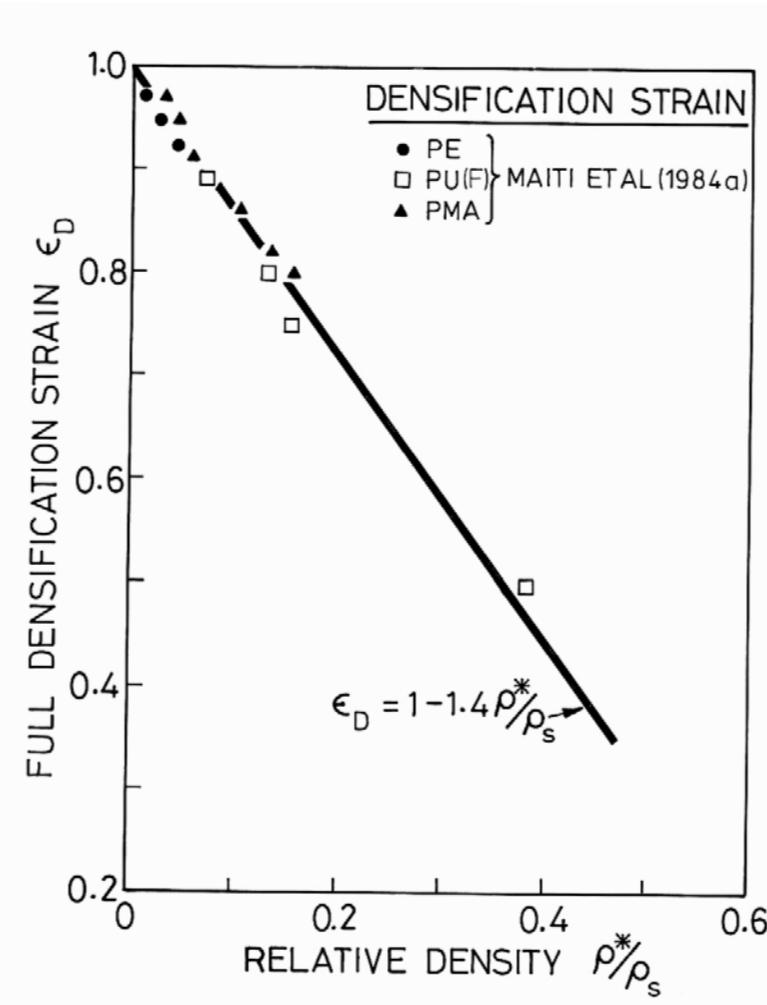
### Densification strain, $\epsilon_D$

- at large comp. strain, cell walls begin to touch,  $\sigma$ - $\epsilon$  rises steeply
- $E^* \rightarrow E_s$ ;  $\sigma$ - $\epsilon$  curve looks vertical, at limiting strain
- Might expect  $\epsilon_D = 1 - \rho^*/\rho_s$
- walls jam together at slightly smaller strain than this:

$$\epsilon_D = 1 - 1.4 \rho^*/\rho_s$$



# Densification Strain



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## Fracture toughness

Open cells : crack length  $2a$ , local stress  $\sigma_l$ , remote stress  $\sigma^\infty$

$$\sigma_l = \frac{C \sigma^\infty \sqrt{\pi a}}{\sqrt{2\pi r}} \quad \text{a distance } r \text{ from crack tip}$$

- next unbroken cell wall a distance  $r \approx l/2$  ahead of crack tip subject to a force (integrating stress over next cell)

$$F \propto \sigma_l l^2 \propto \sigma^\infty \sqrt{\frac{a}{l}} l^2$$

- edges fail when applied moment,  $M$  = fracture moment,  $M_f$

$$M_f \propto \sigma_{fs} t^3$$

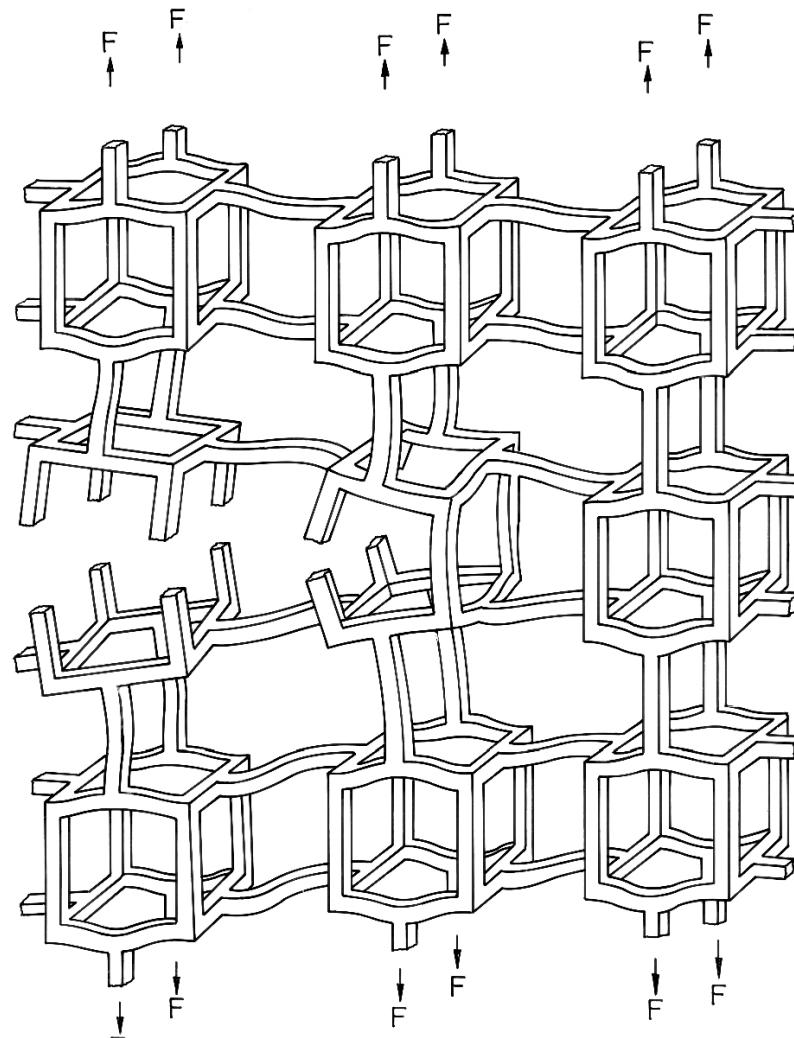
$$M \propto Fl^2 \propto \sigma^\infty \left(\frac{a}{l}\right)^{1/2} l^3 \quad M = M_f \Rightarrow \sigma^\infty \left(\frac{a}{l}\right)^{1/2} l^3 \propto \sigma_{fs} t^3$$

$$\sigma^\infty \propto \sigma_{fs} \left(\frac{l}{a}\right)^{1/2} \left(\frac{t}{l}\right)^3$$

$$K_{Ic}^* = \sigma^\infty \sqrt{\pi a} = C_8 \sigma_{fs} \sqrt{\pi l} \left(\rho^*/\rho_s\right)^{3/2}$$

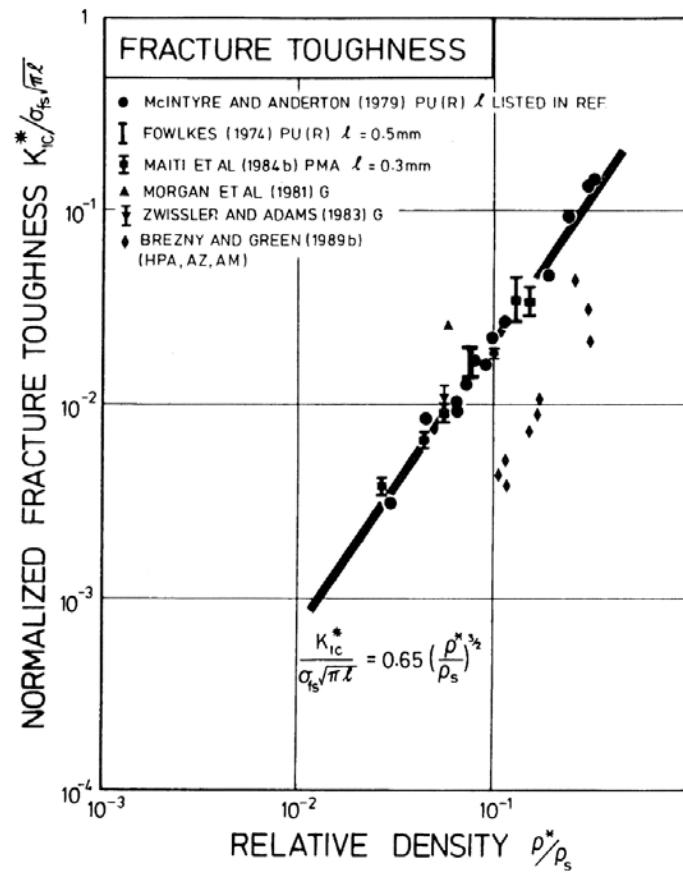
Data:  $C_8 \sim 0.65$

# Fracture Toughness



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# Fracture Toughness



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3.054 / 3.36 Cellular Solids: Structure, Properties and Applications

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