

# Linear elasticity and weak formulation of the elastostatic problem

## 1 Linear elasticity

### 1.1 Total energy of a system

The total energy of a mechanical system consists of:

- *internal potential energy*;
- *kinetic energy*
- *work done by external forces*;

Let us take an elastic body  $B$ , with its boundary being  $\partial B$ . We can associate a vector  $\mathbf{x}$  to every point of the body  $B$ . We will divide the boundary  $\partial B$  in two parts  $S_u$  and  $S_t$ , such that  $S_u \cup S_t = \partial B$  and  $S_u \cap S_t = 0$ . On  $S_t$  we impose an external mechanical traction  $\mathbf{t}(\mathbf{x})$  (it corresponds to a contact force per unit area). On  $S_u$  we impose a constraint on the displacement field, such that  $\mathbf{u} = \hat{\mathbf{u}}$ . We can also apply mechanical forces per unit volume inside the body  $B$ : we define them with the field  $\mathbf{b}(\mathbf{x})$ .

The total energy of the body  $B$  will be

$$\Pi = E_{int} + W_{ext} \tag{1}$$

where  $E_{int}$  is the internal potential energy of the body, whereas  $W_{ext}$  is the work done by external forces. We have neglected the kinetic energy, as we assume that the behavior of the body  $B$  is *quasi-static*.

Let  $\mathbf{u}$  be one of the possible displacement field of the body, and  $\boldsymbol{\sigma}$  the stress associated with the displacement field, the internal potential energy can be defined as

$$E_{int}\{\nabla\mathbf{u}, \boldsymbol{\sigma}\} = \frac{1}{2} \int_B \boldsymbol{\sigma} \cdot \nabla\mathbf{u} \, dv \quad (2)$$

where  $\boldsymbol{\sigma}$  is the Cauchy stress tensor.

Since  $\boldsymbol{\sigma}$  is symmetric, if we define  $\boldsymbol{\epsilon} = \text{sym}\{\nabla\mathbf{u}\} = \frac{1}{2} [\nabla\mathbf{u} + (\nabla\mathbf{u})^T]$ , we can rewrite equation 2:

$$E_{int}\{\nabla\mathbf{u}, \boldsymbol{\sigma}\} = \frac{1}{2} \int_B \boldsymbol{\sigma} \cdot \boldsymbol{\epsilon} \, dv \quad (3)$$

Furthermore, with the constitutive relation of linear elasticity

$$\boldsymbol{\sigma} = \mathbb{C}\boldsymbol{\epsilon} \quad (4)$$

with  $\mathbb{C}$  being a symmetric fourth-order tensor, we notice that the internal energy  $E_{int}$  does only depend on the field  $\nabla\mathbf{u}$ :

$$E_{int}\{\nabla\mathbf{u}\} = \frac{1}{2} \int_B \mathbb{C}\boldsymbol{\epsilon} \cdot \boldsymbol{\epsilon} \, dv \quad (5)$$

The work done by external forces  $W_{ext}$  can be easily written as

$$W_{ext}\{\mathbf{u}\} = - \int_B \mathbf{b} \cdot \mathbf{u} \, dv - \int_{S_t} \mathbf{t} \cdot \mathbf{u} \, da \quad (6)$$

where we have assumed that forces and displacement are related.

Now we have an expression for the total energy of an elastostatic system:

$$\Pi\{\mathbf{u}\} = E_{int}\{\nabla\mathbf{u}\} + W_{ext}\{\mathbf{u}\} = \frac{1}{2} \int_B \mathbb{C}\boldsymbol{\epsilon} \cdot \boldsymbol{\epsilon} \, dv - \int_B \mathbf{b} \cdot \mathbf{u} \, dv - \int_{S_t} \mathbf{t} \cdot \mathbf{u} \, da \quad (7)$$

## 1.2 Principle of minimum potential energy

The *principle of minimum potential energy* states that every system tends to achieve a state which minimizes its total energy.

Let  $\Pi\{\mathbf{x}, \mathbf{y}, \mathbf{z}\}$  be the total energy of a system, which depends on the generic fields  $\mathbf{x}, \mathbf{y}, \mathbf{z}$ . If  $\bar{\mathbf{x}}, \bar{\mathbf{y}}, \bar{\mathbf{z}}$  are the actual fields which allow the system to be in equilibrium, then  $\delta\Pi\{\bar{\mathbf{x}}, \bar{\mathbf{y}}, \bar{\mathbf{z}}; \delta\mathbf{x}, \delta\mathbf{y}, \delta\mathbf{z}\} = 0 \quad \forall$  admissible  $\delta\mathbf{x}, \delta\mathbf{y}, \delta\mathbf{z}$ .

That said, if the displacement field  $\mathbf{u}$  is actually the equilibrium field of our system, then, recalling equation 7:

$$\delta\Pi\{\mathbf{u}; \delta\mathbf{u}\} = \delta E_{int}\{\nabla\mathbf{u}; \nabla\delta\mathbf{u}\} + \delta W_{ext}\{\mathbf{u}; \delta\mathbf{u}\} = 0 \quad \forall \text{ admissible } \delta\mathbf{u} \quad (8)$$

noted that  $\delta(\nabla\mathbf{u}) = \nabla\delta\mathbf{u}$ . In this case with the expression  $\forall$  admissible  $\delta\mathbf{u}$  we mean all the possible fields for which  $\mathbf{u} = \hat{\mathbf{u}}$  on  $S_u$ . If  $\tilde{\mathbf{u}} = \mathbf{u} + \zeta\delta\mathbf{u}$ , also for the variation  $\tilde{\mathbf{u}} = \hat{\mathbf{u}}$  on  $S_u$ . It is therefore obvious that  $\delta\mathbf{u} = 0$  on  $S_u$ .

Remembering how to compute the variation of a functional (??), then

$$\delta\Pi\{\mathbf{u}; \delta\mathbf{u}\} = \frac{1}{2} \int_B \frac{\partial(\mathbb{C}\boldsymbol{\epsilon} \cdot \boldsymbol{\epsilon})}{\partial(\nabla\mathbf{u})} \cdot \nabla\delta\mathbf{u} \, dv - \int_B \frac{\partial}{\partial\mathbf{u}}(\mathbf{b} \cdot \mathbf{u}) \cdot \delta\mathbf{u} \, dv - \int_{S_t} \frac{\partial}{\partial\mathbf{u}}(\mathbf{t} \cdot \mathbf{u}) \cdot \delta\mathbf{u} \, da \quad (9)$$

We will avoid the passages that lead to the final expression to not weigh the paragraph down. That said, we can finally exploit the *weak formulation of the linear elastostatic problem*.

$$\int_B \boldsymbol{\sigma} \cdot \nabla\delta\mathbf{u} \, dv - \int_B \mathbf{b} \cdot \delta\mathbf{u} \, dv - \int_{S_t} \mathbf{t} \cdot \delta\mathbf{u} \, da = 0 \quad \forall \text{ admissible } \delta\mathbf{u} \quad (10)$$

It can be easily demonstrated that the *weak formulation* is equivalent to the *local balance equations* and to the *principle of virtual power*.