

Nis Bjørn Møller

M.Sc. Danish Technical University

- Acoustics
- Servo Techniques

Professional Sound Reproduction

Brüel & Kjær since 1969

- Structural Dynamics
- Machine Diagnostics
- Vibration testing



Author of many Conference Papers

- International Modal Analysis Conference (IMAC)
- SAE – USA – Brazil
-

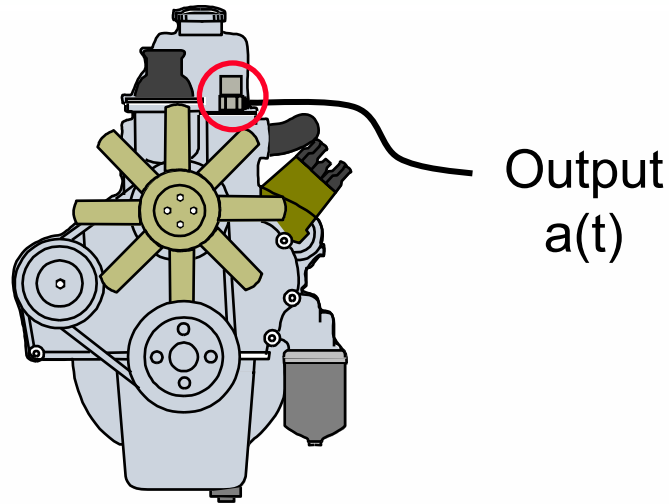
System Analysis

System Analysis

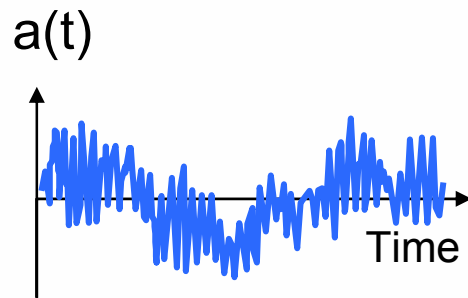
- Introduction
- System Descriptors
- Dual-channel FFT Analysis
- Cross Spectrum
- Coherence Function
- Frequency Response Function

Signal Analysis

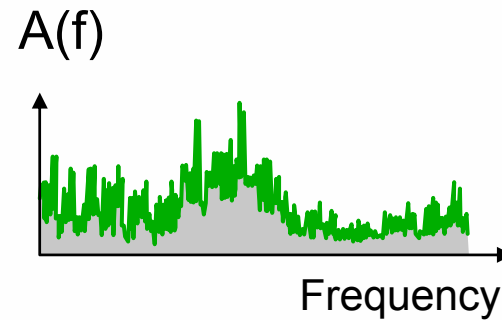
Definition



Time signal

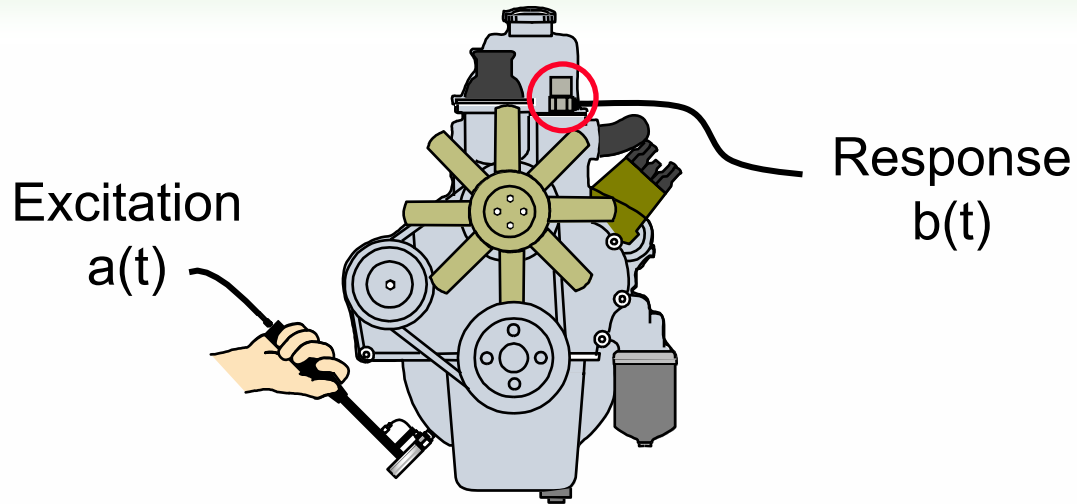


Frequency spectrum

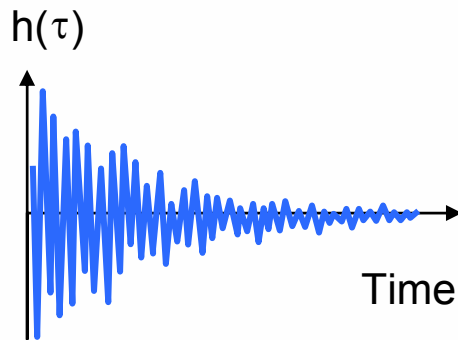


System Analysis

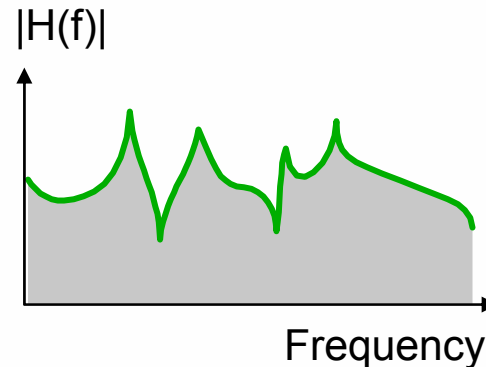
Definition



Impulse Response Function



Frequency Response Function

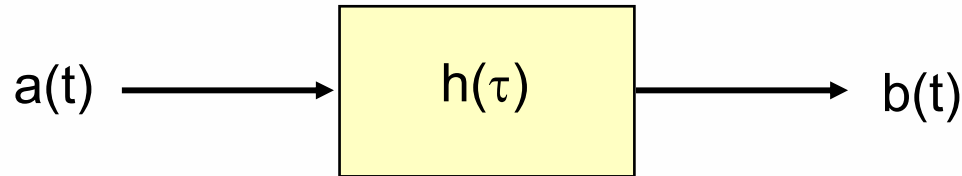


System Analysis

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- Frequency Response Function

System Descriptors

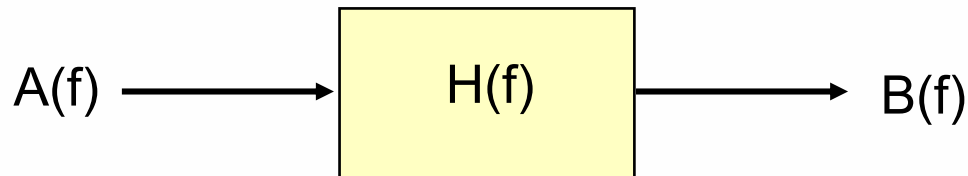
Impulse Response Function



Convolution

$$b(t) = \int_{-\infty}^{\infty} h(\tau) \cdot a(t - \tau) d\tau = h(t) * a(t)$$

Frequency Response Function

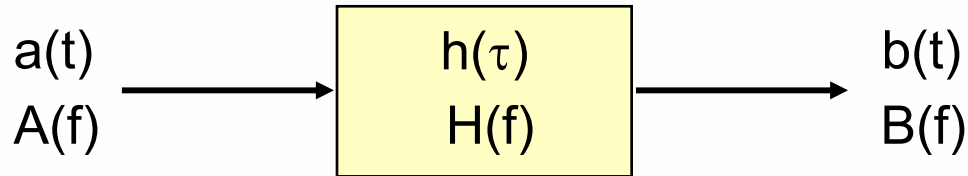


Multiplication

$$B(f) = H(f) \cdot A(f)$$

$h(\tau)$ and $H(f)$ are system descriptors
independent of the signals involved

The Ideal Physical System



Constant parameters

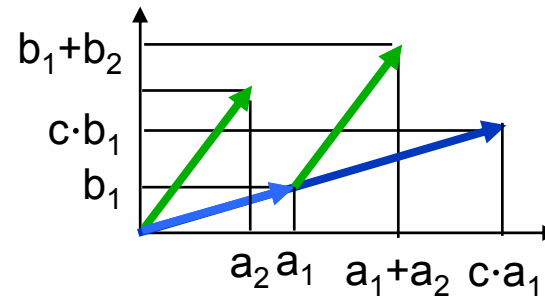
$$h(\tau, t) = h(\tau)$$

$$H(f, t) = H(f)$$

Linearity

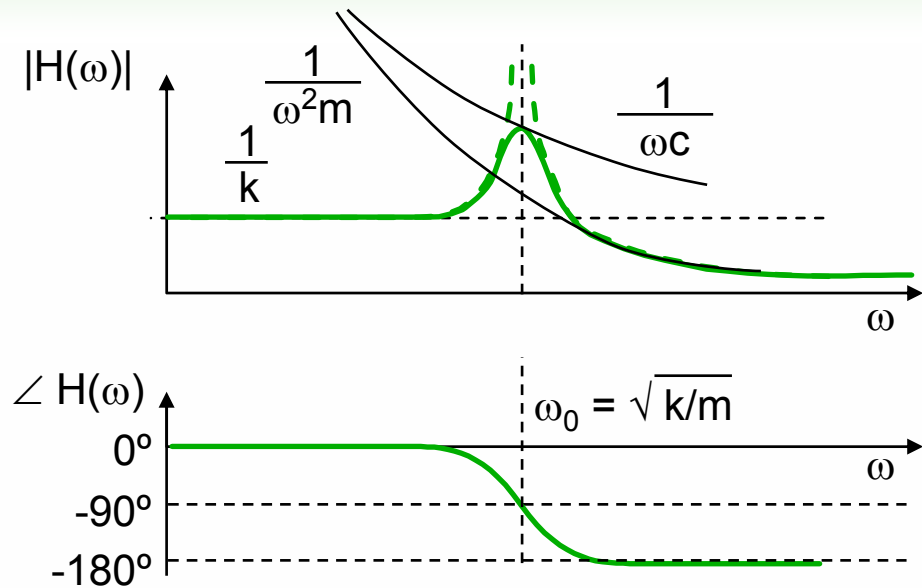
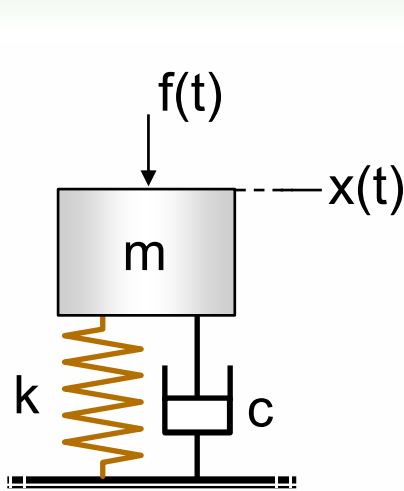
If $a_1(t)$ produces $b_1(t)$
and $a_2(t)$ produces $b_2(t)$

then $a_1(t) + a_2(t)$ produces $b_1(t) + b_2(t)$
and $c \cdot a_1(t)$ produces $c \cdot b_1(t)$



Additive and homogeneous in
time and frequency domain

Frequency Response Function for SDOF system



Time domain

$$m a = f - c v - k x \Leftrightarrow m \ddot{x} + c \dot{x} + k x = f$$

Frequency domain

$$H(\omega) = \frac{X(\omega)}{F(\omega)} = \frac{1}{-\omega^2 m + j\omega c + k} = \frac{R}{j\omega - (-\sigma + j\omega_d)} + \frac{R^*}{j\omega - (-\sigma - j\omega_d)}$$

where

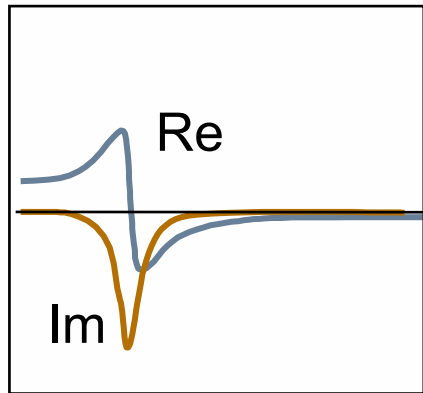
$$R = \frac{1}{j2m\omega_d} \quad \omega_d = \sqrt{\frac{k}{m} - \frac{c^2}{4m^2}} \quad \sigma = \frac{c}{2m}$$

Frequency Response Function for SDOF system

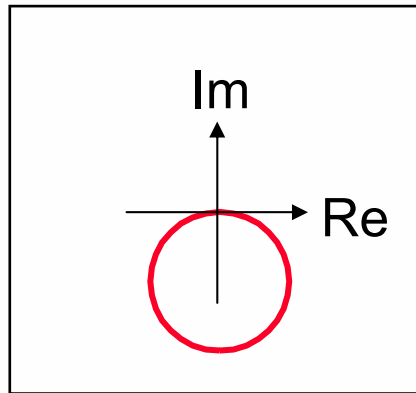
$$H(\omega) = \frac{R}{j\omega - p} + \frac{R^*}{j\omega - p^*} \quad \text{where } p = -\sigma + j\omega_d$$

At resonance: $|H(\omega)| \approx \frac{R}{\sigma} \quad R = \text{Residue}$

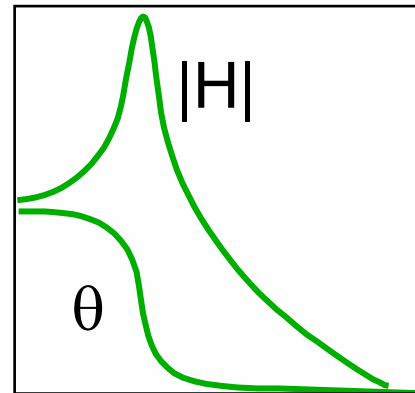
Real & Imaginary



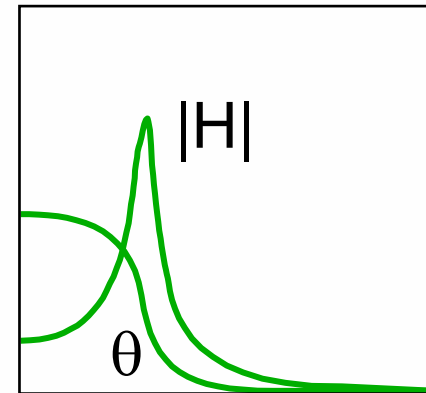
Nyquist



Log. Magnitude
& Phase



Magnitude
& Phase



Damping Parameters

3 dB bandwidth	$\Delta f_{3\text{dB}} = \frac{2\sigma}{2\pi} \quad , \quad \Delta \omega_{3\text{dB}} = 2\sigma$
Loss factor	$\eta = \frac{1}{Q} = \frac{\Delta f_{3\text{dB}}}{f_0} = \frac{\Delta \omega_{3\text{dB}}}{\omega_0}$
Damping ratio	$\zeta = \frac{\eta}{2} = \frac{\Delta f_{3\text{dB}}}{2f_0} = \frac{\Delta \omega_{3\text{dB}}}{2\omega_0}$
Decay constant	$\sigma = \zeta \omega_0 = \pi \Delta f_{3\text{dB}} = \frac{\Delta \omega_{3\text{dB}}}{2}$
Quality factor	$Q = \frac{f_0}{\Delta f_{3\text{dB}}} = \frac{\omega_0}{\Delta \omega_{3\text{dB}}}$

System Analysis

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Dual-channel FFT Analysis

- Simultaneous measurements at the input and output are performed
- The input and output autospectra, and the cross spectrum between the input and output are measured
- Many other functions can be calculated from the basic, measured data

Advantages of Dual-channel FFT Analysis

- Phase information is available
- Effects of noise are minimised
- A controlled input signal is not needed
- Easy to use
- Extension, for instance to Modal Analysis

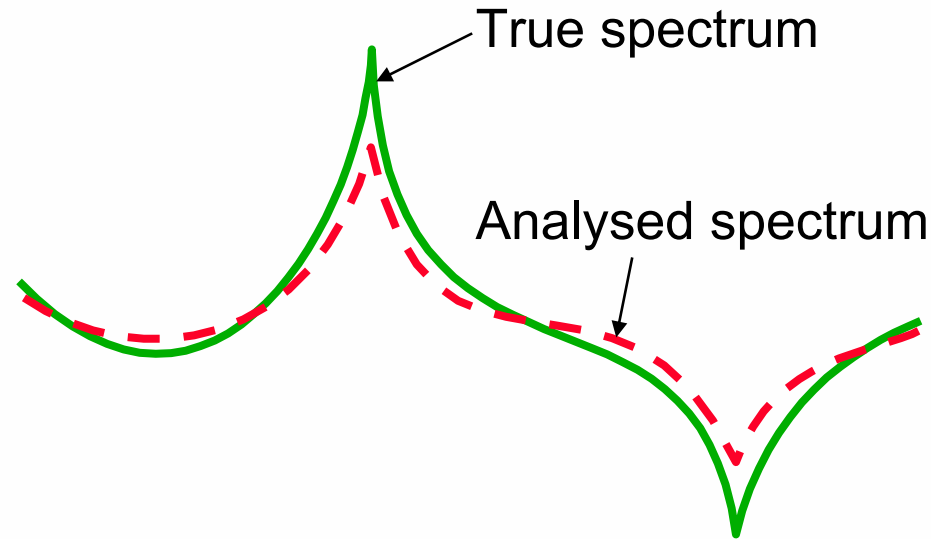
Pitfalls of Dual-channel FFT Analysis

Pitfalls of Dual-channel Analysis include:

- Leakage
- A linear system is assumed
- Compensation for system delays necessary

Leakage

- Leakage occurs because FFT analyzers operate on truncated time signals
- Leakage means that measured peaks can be too low and measured valleys too high
- To combat leakage in system analysis:
 - increase the resolution
 - use optimum time weighting function
 - choose optimum excitation

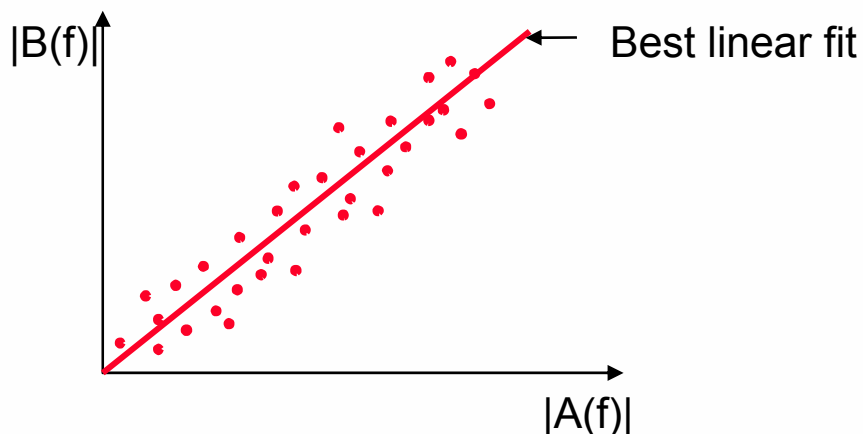


Linearisation

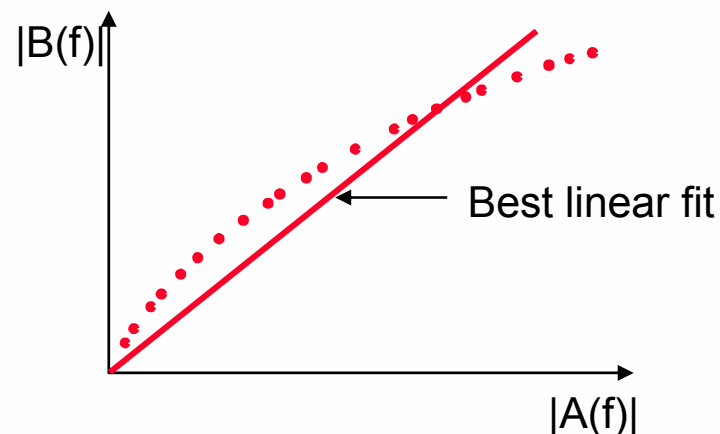
All FFT calculations assume a linear system

Using random excitation, the best linear estimate of the response functions can be obtained with a FFT analyzer

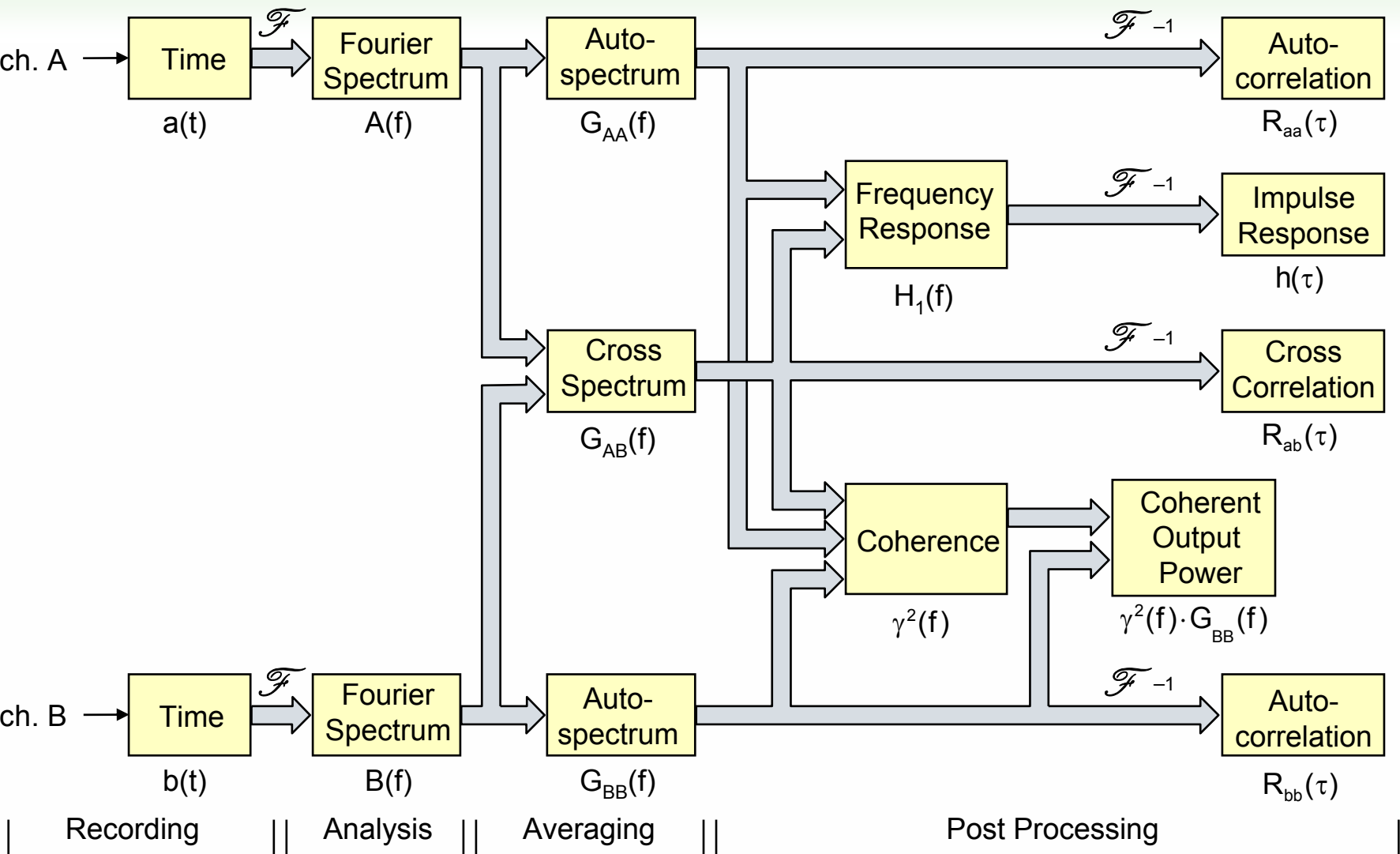
Noise



Non-linear system



Dual Channel Spectrum Averaging



System Analysis

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- Frequency Response Function
- Impulse Response Function
- Summary

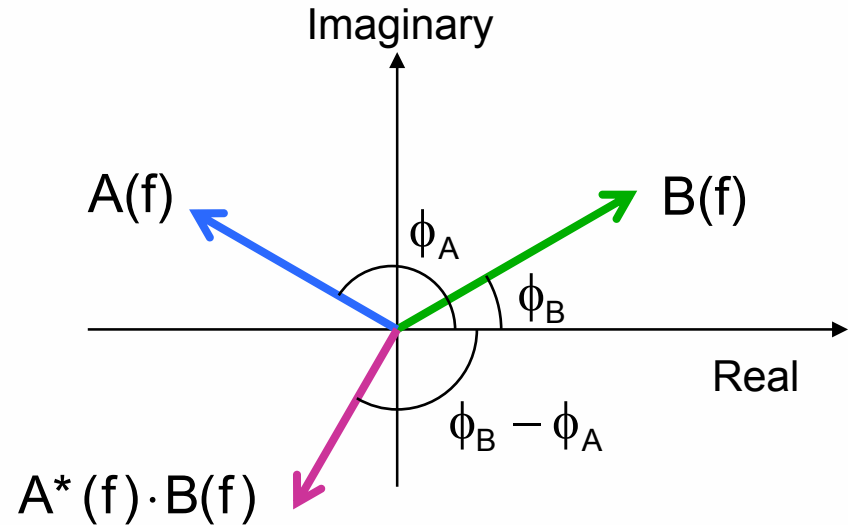
Cross Spectrum

$$S_{AB}(f) = E[A^*(f) \cdot B(f)]$$

$$A(f) = |A(f)|e^{i\phi_A(f)}$$

$$B(f) = |B(f)|e^{i\phi_B(f)}$$

$$S_{AB}(f) = E[|A(f)| \cdot |B(f)| e^{i(\phi_B(f) - \phi_A(f))}]$$



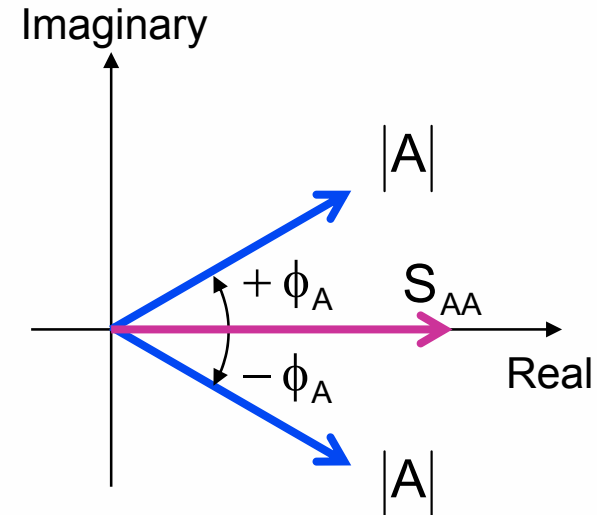
Phase of Cross Spectrum is phase of system

Auto Spectrum

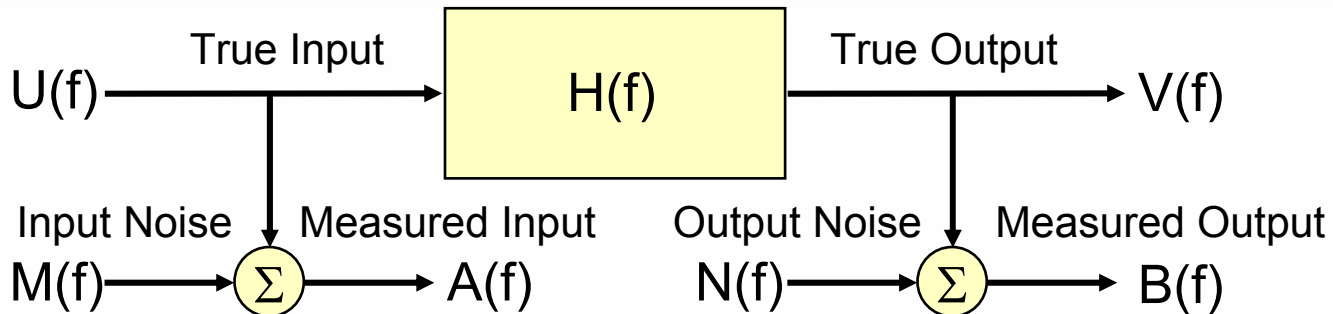
$$\begin{aligned} S_{AA}(f) &= E[A(f) \cdot A^*(f)] \\ &= \mathcal{F} \quad E[a(t) * a(-t)] \\ &= \mathcal{F} [R_{aa}(\tau)] \end{aligned}$$

$$A(f) = |A(f)|e^{i\phi_A(f)} \quad A^*(f) = |A(f)|e^{-i\phi_A(f)}$$

$$S_{AA}(f) = E[|A(f)| \cdot |A^*(f)| \cdot e^{i0}]$$



Influence of Noise



Autospectrum Ch.A

$$\begin{aligned}
 S_{AA} &= E[(U + M)^* \cdot (U + M)] \\
 &= E[(U^* \cdot U)] + E[(U^* \cdot M)] + E[(M^* \cdot U)] + E[(M^* \cdot M)] \\
 &= S_{UU} + S_{MM}
 \end{aligned}$$

Autospectrum Ch.B

$$S_{AA} = S_{VV} + S_{NN}$$

Cross Spectrum

$$\begin{aligned}
 S_{AB} &= E[(U + M)^* \cdot (V + N)] \\
 &= E[(U^* \cdot V)] + E[(U^* \cdot N)] + E[(M^* \cdot V)] + E[(M^* \cdot N)] \\
 &= S_{UV}
 \end{aligned}$$

Summary

Cross Spectrum

$$S_{AB}(f) = E[A^*(f) \cdot B(f)]$$

- Measures phase difference
- The effect of noise is reduced through averaging

ANY



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The Coherence Function

- Input-Output Relationship

$$[\text{Cross Power}]^2 \leq [\text{Input Power}] \cdot [\text{Output Power}]$$

$$|G_{AB}(f)|^2 \leq |G_{AA}(f)| \cdot |G_{BB}(f)|$$

- Definition

$$\gamma^2(f) \equiv \frac{|G_{AB}(f)|^2}{|G_{AA}(f)| \cdot |G_{BB}(f)|}$$

- It expresses degree of *linear* relationship between $A(f)$ and $B(f)$

$$0 \leq \gamma^2(f) \leq 1$$

Coherence vs Correlation Coefficient

- Coherence

$$\gamma^2(f) \equiv \frac{|G_{AB}(f)|^2}{G_{AA}(f) \cdot G_{BB}(f)} \quad 0 \leq \gamma^2(f) \leq 1$$

- Correlation Coefficient

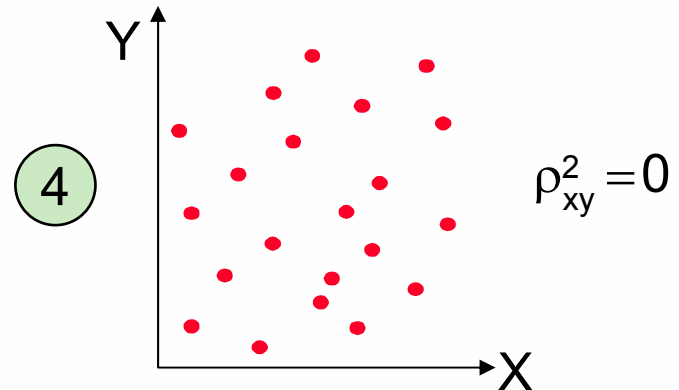
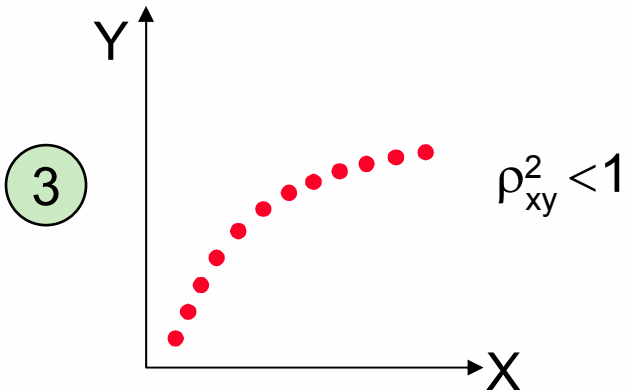
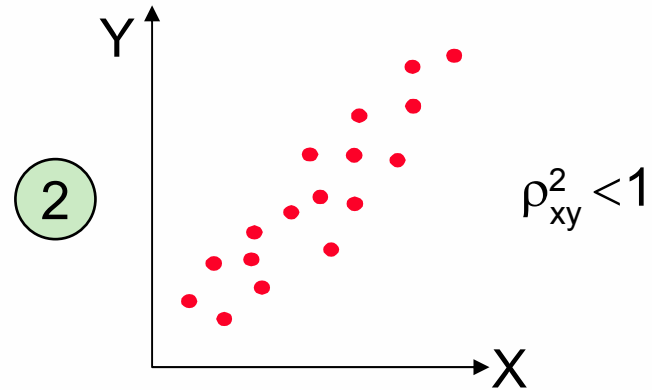
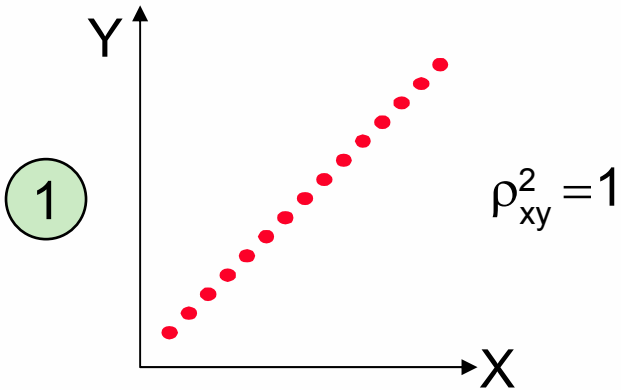
$$\rho_{xy}^2 \equiv \frac{\sigma_{xy}^2}{\sigma_x^2 \cdot \sigma_y^2} \quad 0 \leq \rho_{xy}^2 \leq 1$$

Variance ~ Autospectrum

Covariance ~ Cross Spectrum

The Correlation Coefficient

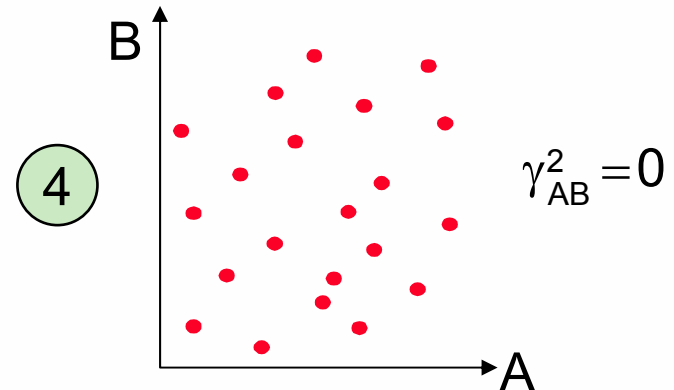
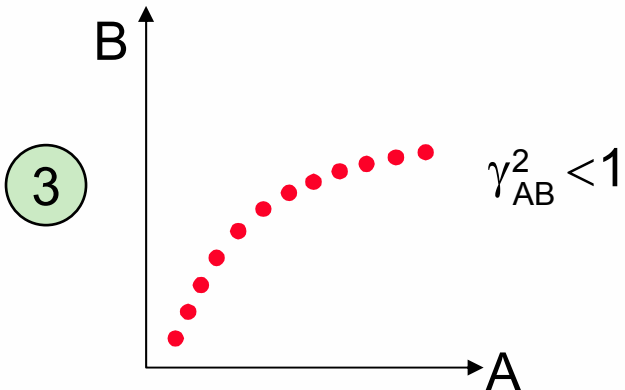
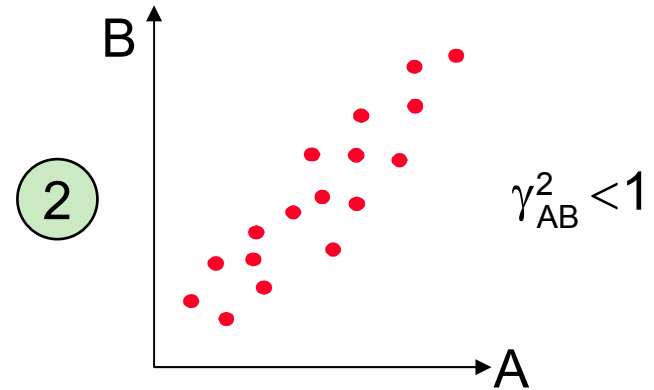
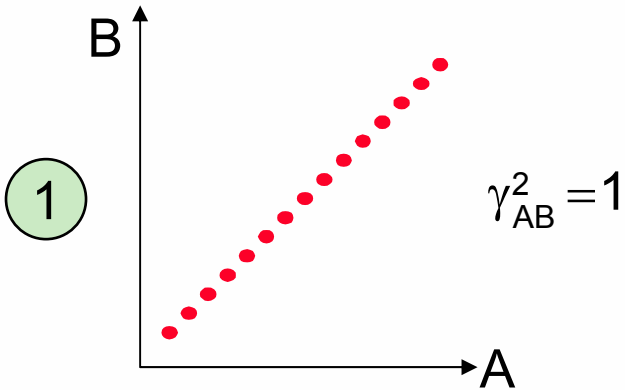
$$\rho_{xy}^2 = \frac{\sigma_{xy}^2}{\sigma_x^2 \cdot \sigma_y^2}$$



The Coherence Function

$$\gamma^2(f) \equiv \frac{|G_{AB}(f)|^2}{G_{AA}(f) \cdot G_{BB}(f)}$$

For each frequency



Averaging of The Coherence Function

$$\gamma^2(f) \equiv \frac{|G_{AB}(f)|^2}{G_{AA}(f) \cdot G_{BB}(f)}$$

- The coherence only provides useful information when $G_{AB}(f)$, $G_{AA}(f)$ and $G_{BB}(f)$ are estimates, i.e averaged over many records
- For one record only (no averaging):

$$\gamma^2(f) = 1$$

Reasons for Low Coherence

Difficult measurements:

- Noise in measured output signal
- Noise in measured input signal
- Other inputs not correlated with measured input signal

Bad measurements:

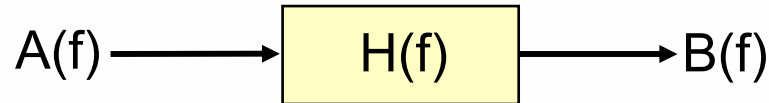
- Leakage
- Time varying systems
- Non-linearities of system
- DOF-jitter
- Propagation time not compensated for

System Analysis

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Definition of the Frequency Response Function

- Definition for the **ideal** system:



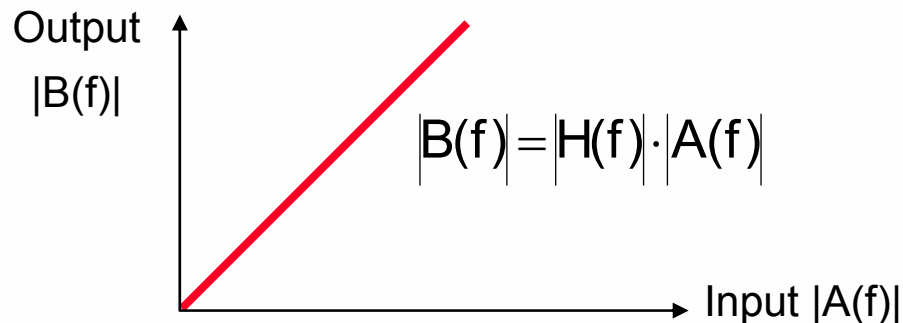
Output:

$$B(f) = H(f) \cdot A(f)$$

- Frequency Response Function:

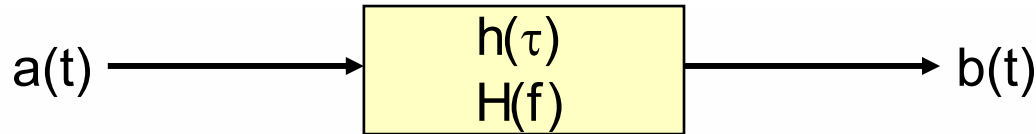
$$H(f) \equiv \frac{B(f)}{A(f)}$$

- For each frequency:



H is the slope of the straight line describing the output as function of input

Alternative Estimators



$$H(f) \equiv \frac{B(f)}{A(f)}$$

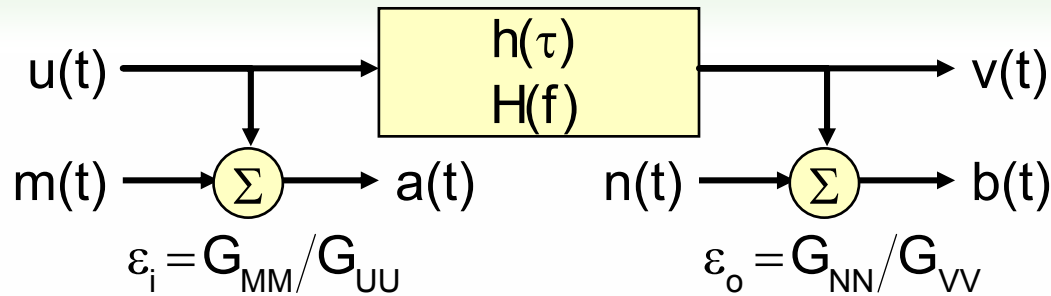
$$H_1(f) = \frac{G_{AB}(f)}{G_{AA}(f)}$$

$$H_2(f) = \frac{G_{BB}(f)}{G_{BA}(f)}$$

$$H_3(f) = \sqrt{\frac{G_{BB}}{G_{AA}}} \cdot \frac{G_{AB}}{|G_{AB}|} = \sqrt{H_1 \cdot H_2}$$

$$\gamma^2(f) = \frac{|G_{AB}|^2}{G_{AA} \cdot G_{BB}} = \frac{G_{AB}}{G_{AA}} \cdot \frac{G_{AB}^*}{G_{BB}} = \frac{H_1}{H_2}$$

Noise at Input and Output



$$H_1 = \frac{G_{AB}}{G_{AA}} = H \frac{1}{1 + \varepsilon_i}$$

$$H_2 = \frac{G_{BB}}{G_{BA}} = H [1 + \varepsilon_o]$$

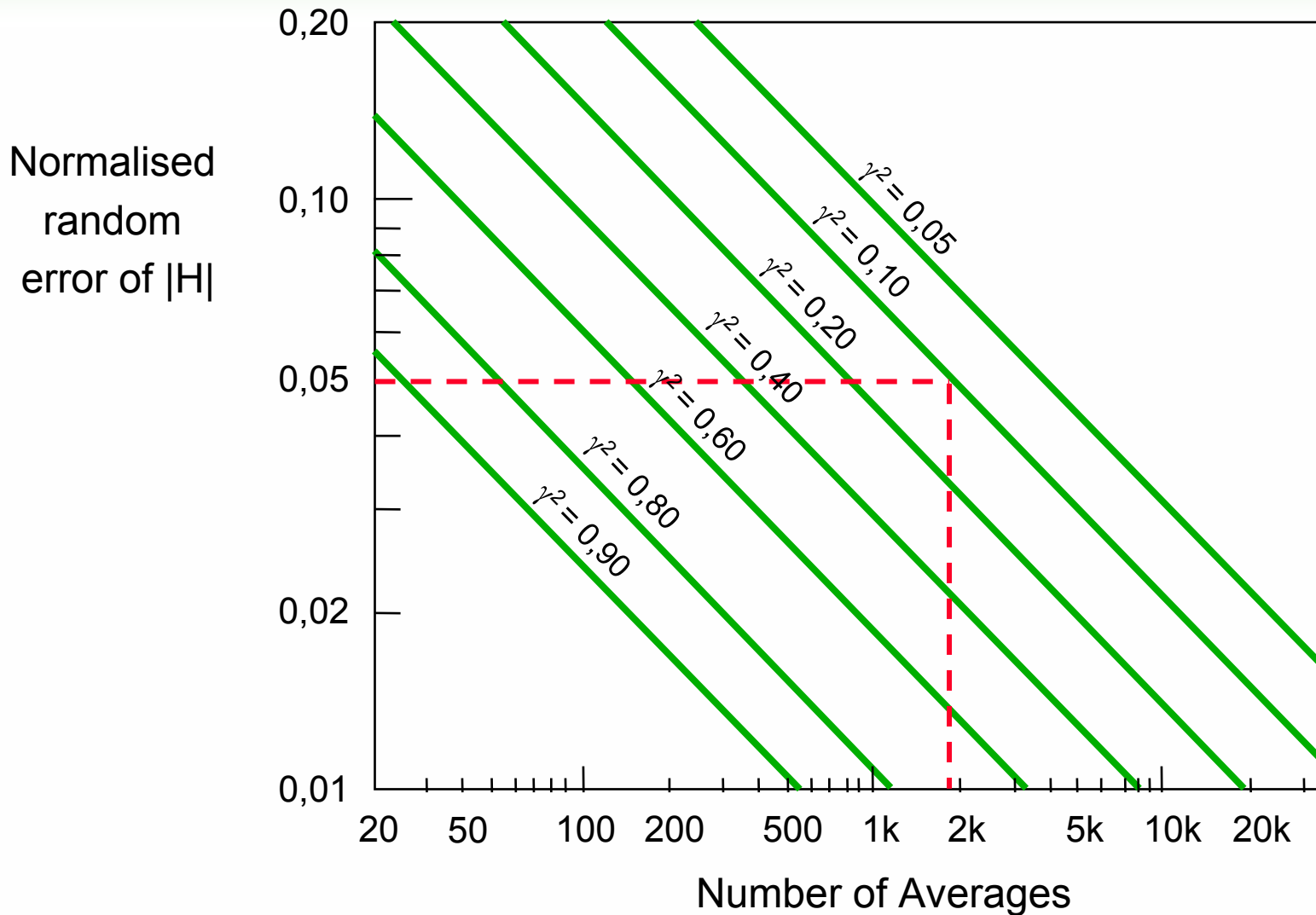
$$|H_3|^2 = \frac{G_{BB}}{G_{AA}} = |H|^2 \frac{1 + \varepsilon_o}{1 + \varepsilon_i}$$

$$\gamma^2 = \frac{H_1}{H_2} = \frac{1}{(1 + \varepsilon_i)(1 + \varepsilon_o)}$$

$$|H_3|^2 = |H_1| \cdot |H_2|$$

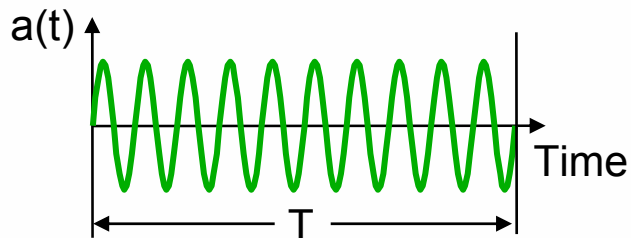
$$|H_1| \leq |H| \leq |H_2|$$

Noise at Input and Output

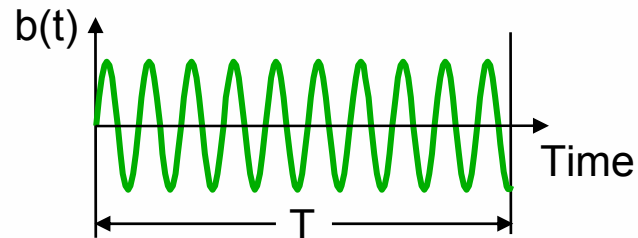


Leakage

Signal periodic
with record length

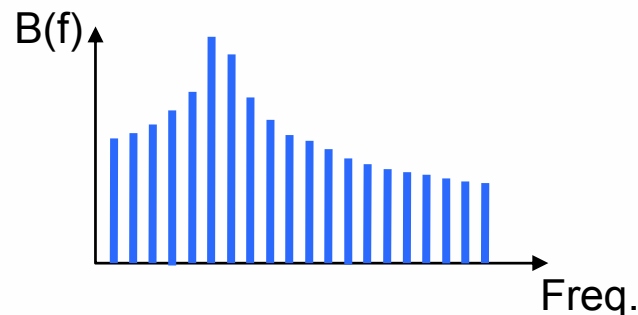
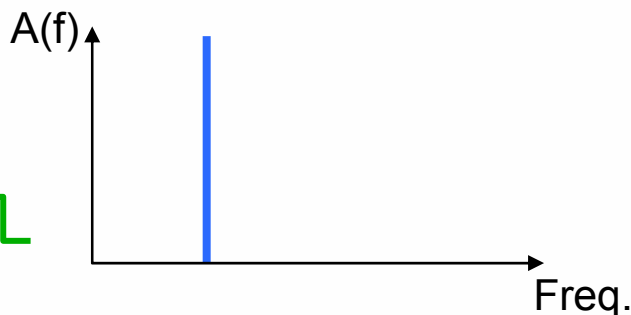


Signal **not** periodic
with record length



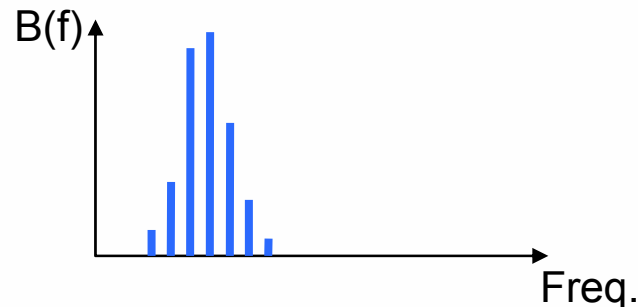
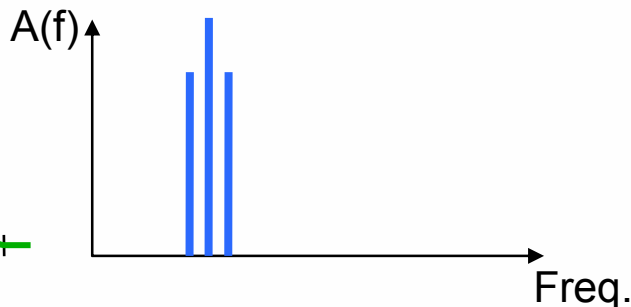
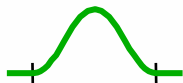
Rectangular
weighting

(no weighting)

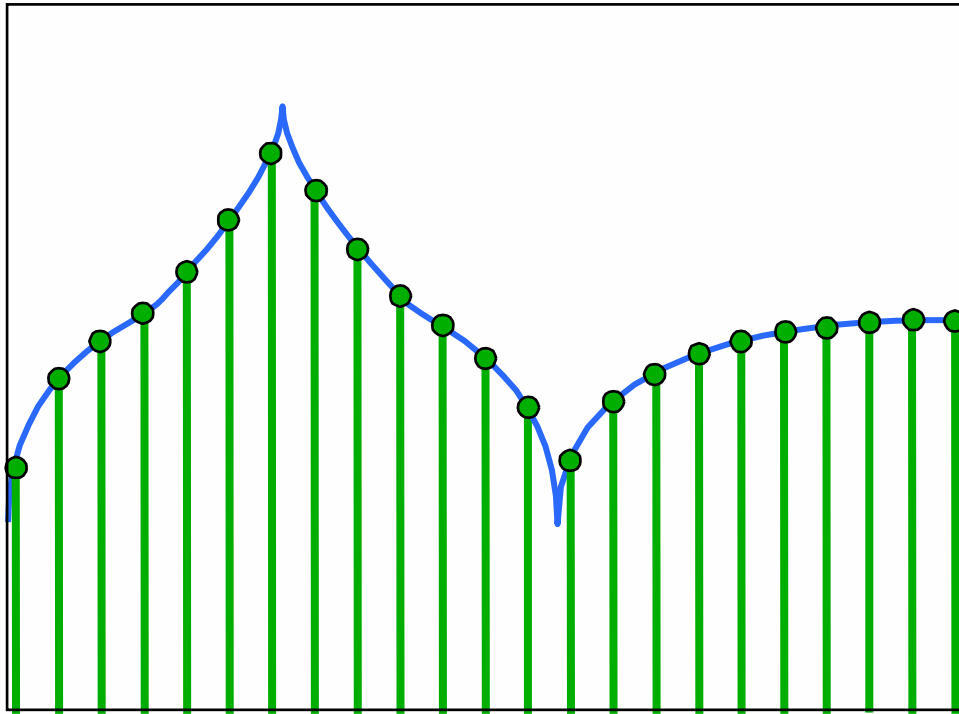


Hanning
weighting

$$\left(1 - \cos \frac{2\pi t}{T}\right)$$



FFT Fundamentals - Picket Fence Effect



The measured points give the correct values, but the exact frequency and magnitude of the resonance may not be found.

Frequency Response Function Estimates

Accuracy

Definitions: $H_1 \equiv \frac{G_{AB}}{G_{AA}}$ $H_2 \equiv \frac{G_{BB}}{G_{BA}}$ $H_3 \equiv \sqrt{\frac{G_{BB}}{G_{AA}}} \frac{G_{AB}}{|G_{AB}|}$

Accuracy for systems with:	H_1	H_2	H_3
Input noise	-	Best	-
Output noise	Best	-	-
Input + output noise	-	-	Best
Peaks (leakage)	-	Best	-
Valleys (leakage)	Best	-	-

User can choose H_1 , H_2 or H_3 after measurement

Summary

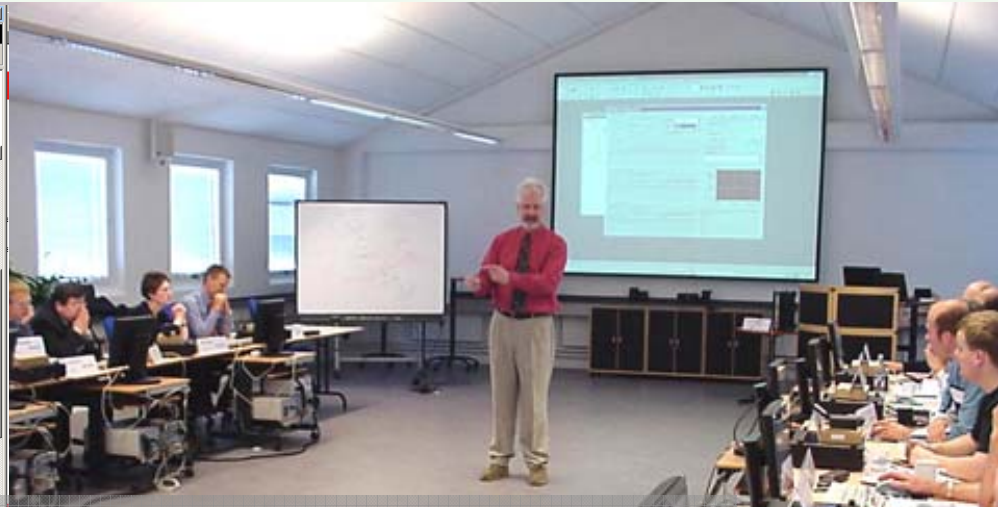
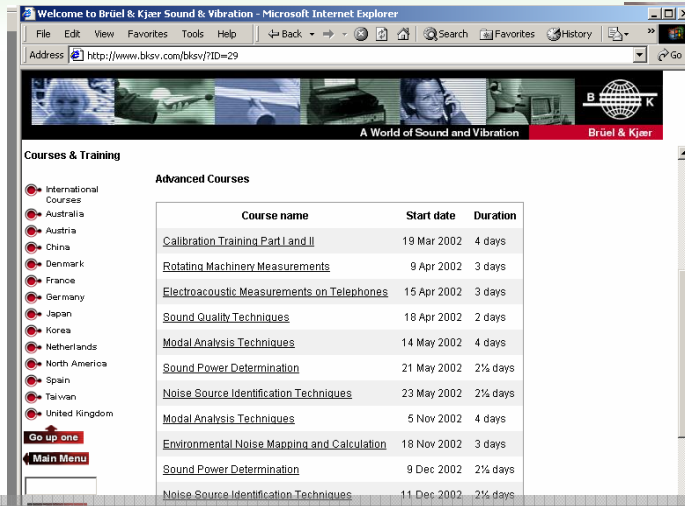
- Measure both Input and Output
- Using Dual (Multi-) channel FFT
- Averaging - improving accuracy
- Leakage
 - Increase Frequency Resolution
 - Use Zoom
 - Use Time weighting
- Noise at
 - Input
 - Output

ANY



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 - **Objectives** To give an understanding of Modal Testing, its possibilities and limitations.
- **Rotating Machinery Diagnostics - 5 Dec 2007 - 3 days**
 - **Objectives** To teach and explain about the different vibration sources of rotating machinery.

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