



# NON LINEAR CONTROL

## Collaborative Laboratory

097469 – NonLinear Control Course – A.Y. 2023/2024 – Semester I

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# NonLinear Control – Collaborative Laboratory

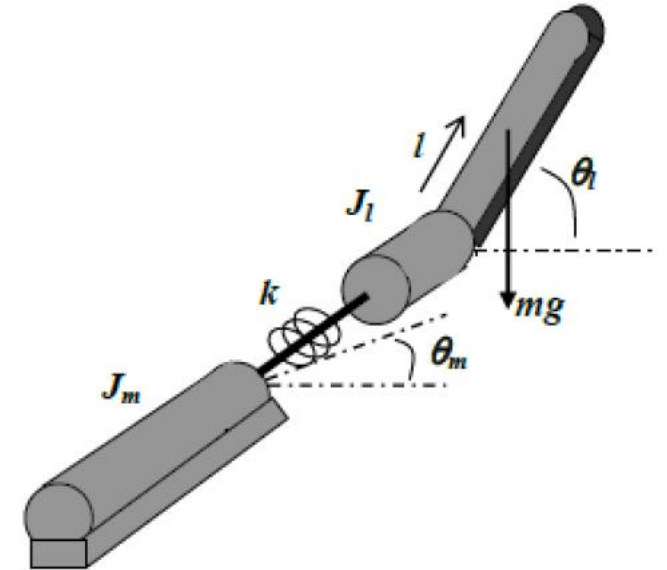
- **Introduction**
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# Introduction

Model Dynamical Equations :

$$\begin{aligned} J_l \ddot{\theta}_l + B_l \dot{\theta}_l + k(\theta_l + \theta_m) + mgl \cos(\theta_l) &= 0 \\ J_m \ddot{\theta}_m + B_m \dot{\theta}_m - k(\theta_l - \theta_m) &= u \end{aligned}$$



Parameters :

$k$	$J_l$	$J_m$	$B_l$	$B_m$	$m$	$l$
0.8 Nm/rad	4e-4 Nms <sup>2</sup> /rad	4e-4 Nms <sup>2</sup> /rad	0 Nms/rad	0.015Nms/rad	0.3 kg	0.3 m



# State-Space Representation

## Model and Simulation scheme

**Request 1 : Give a state space representation of the system and provide a related simulation scheme**

### State-Space Representation:

$$J_l \ddot{\theta}_l + B_l \dot{\theta}_l + k(\theta_l - \theta_m) + mgl \cos(\theta_l) = 0$$

$$J_m \ddot{\theta}_m + B_m \dot{\theta}_m - k(\theta_l - \theta_m) = u$$

$$x_1 = \theta_l; x_2 = \dot{\theta}_l; x_3 = \theta_m; x_4 = \dot{\theta}_m$$

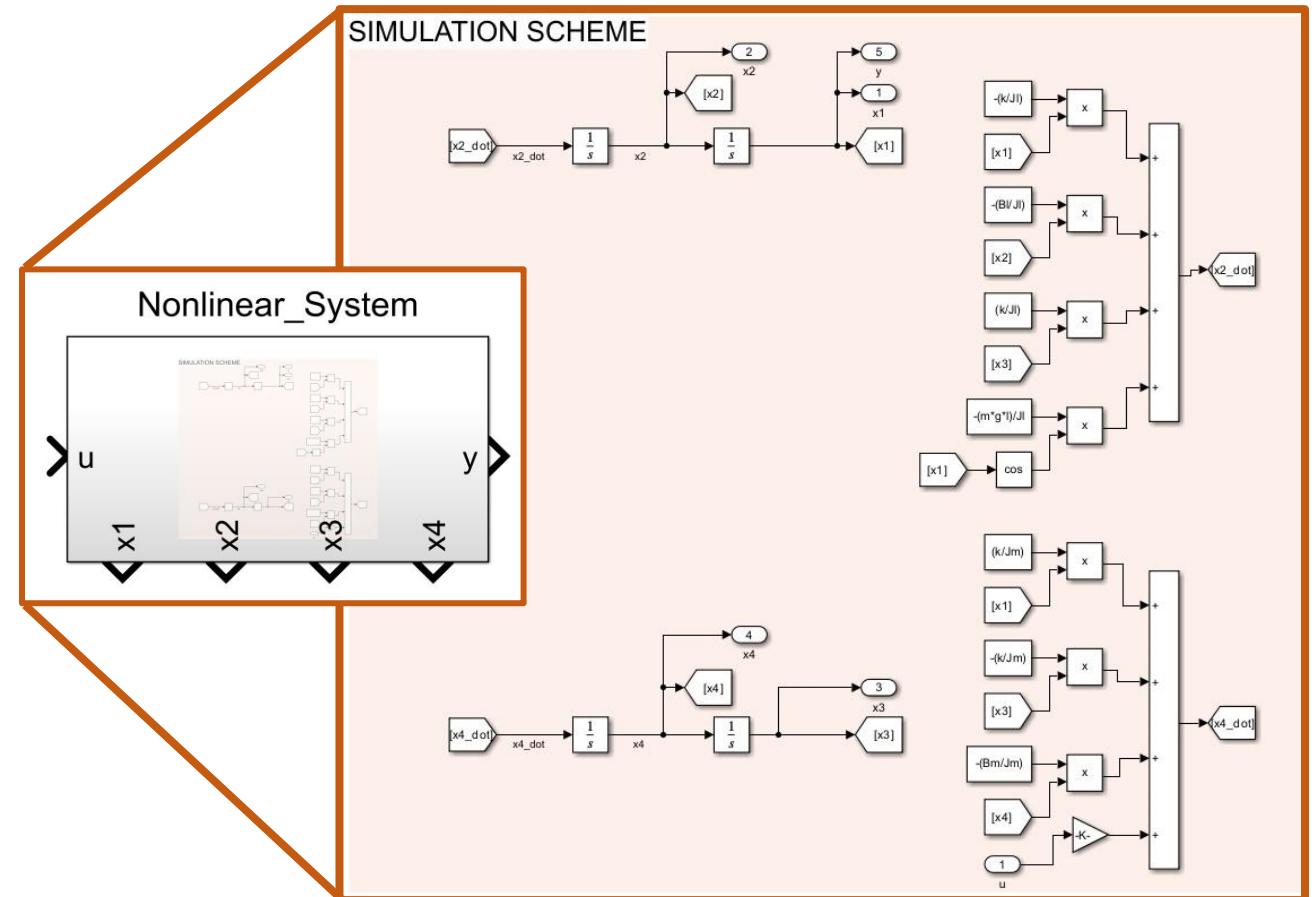
$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = -\frac{B_l}{J_l} x_2 - \frac{k}{J_l} x_1 + \frac{k}{J_l} x_3 - \frac{mgl}{J_l} \cos x_1$$

$$\dot{x}_3 = x_4$$

$$\dot{x}_4 = -\frac{B_m}{J_m} x_4 + \frac{k}{J_m} x_1 - \frac{k}{J_m} x_3 + \frac{1}{J_m} u$$

$$y = x_1$$



# Linear Tangent Approximation

## Linearized Model around the Equilibrium

**Request 2 : Give Linear Tangent Approximation of the System at the point corresponding to  $\theta_l = \frac{\pi}{4}$  and propose a stabilizing feedback of this basis**

### Linear Tangent Approximation :

$$\begin{aligned} \dot{x} &= A x + B u \\ y &= C x \end{aligned} \quad \Rightarrow \quad A = \begin{bmatrix} 0 & 1 & 0 & 0 \\ -\frac{k}{J_l} & -\frac{\beta}{J_l} & \frac{k}{J_l} & 0 \\ 0 & 0 & 0 & 1 \\ \frac{k}{J_m} & 0 & -\frac{k}{J_m} & -\frac{\beta_m}{J_m} \end{bmatrix} \quad B = \begin{bmatrix} 0 \\ 0 \\ 0 \\ \frac{1}{J_m} \end{bmatrix} \quad C = [1 \quad 0 \quad 0 \quad 0] \quad D = 0$$

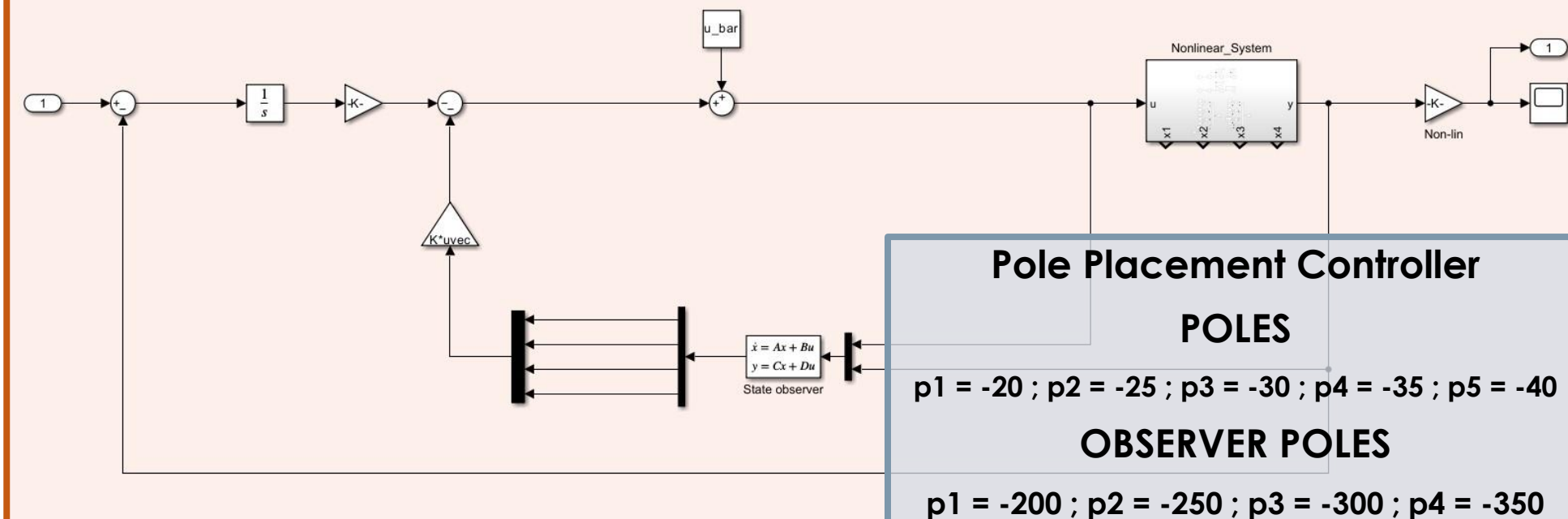


# Linear Tangent Approximation

## Stabilizing Control Law

As stabilizing control law we used a Pole Placement controller where we estimated the state through a Luenberger estimator and we enlarged the system with an integrator to track a reference on  $\theta_l$

PP + OBS ON THE TANGENT MODEL OF THE NON LINEAR SYSTEM

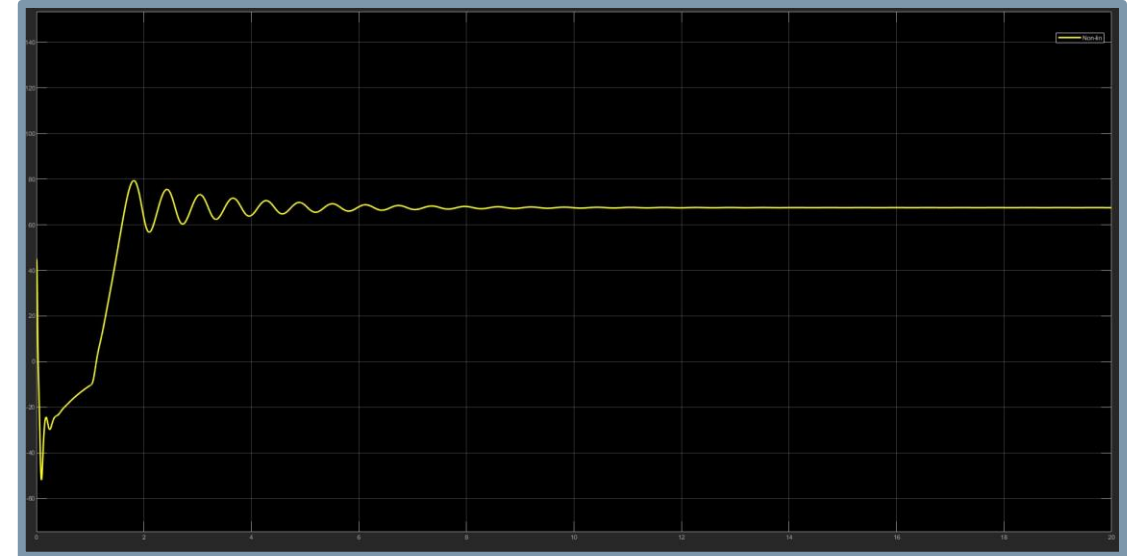


# Linear Tangent Approximation

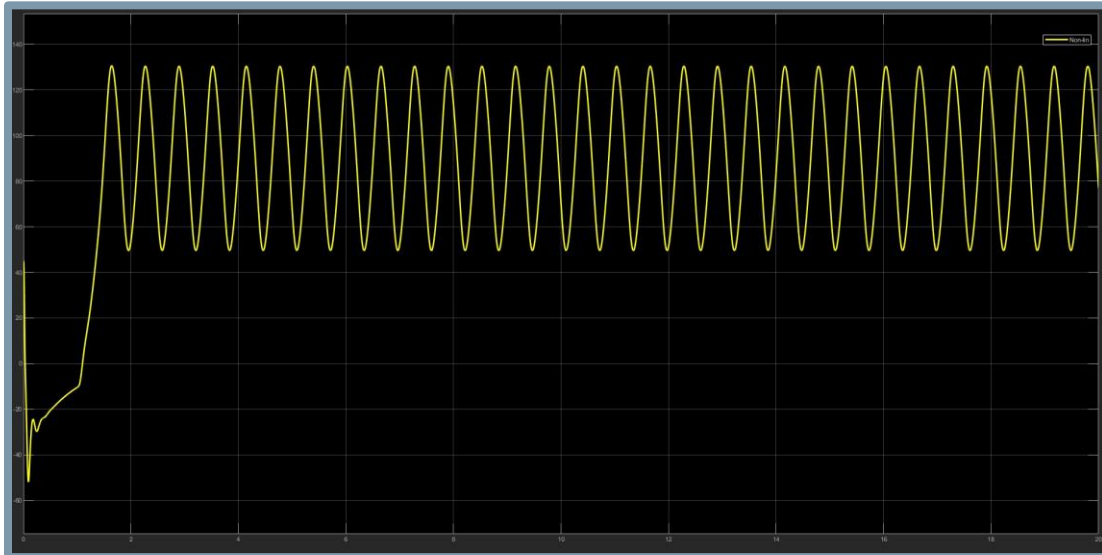
## Simulation Performance

**Request 3 : Implement the obtained control law in the simulation scheme and test its behavior**

We run **two different simulations** to show the behaviour of this controller. In the first case (right image), we set a reference of  $\pi/4 + \pi/8$ , and as we can see the output tracks the reference after some initial small oscillations.



In the second case (left image) we used a reference of  $\pi/2$  and from the plot we can see that the tracking performances are not so good as in the other case. This is due to the fact that the tangent model is accurate only in a neighborhood of the equilibrium point.



# FeedBack Linearization

## Fully Linearizability Condition

**Request 4 : Check that the original nonlinear system is fully exactly I/O linearizable by diffeomorphism and state feedback, with  $\frac{\pi}{4} - \theta_l$  as the output, and propose a stabilizing state feedback law on this basis**

First step is to check if our NonLinear, affine in the input, TI system is Fully Linearizable (Input/State Linearizable). This is equal to check that our system  $S$  has relative degree  $r$  in  $x^\circ$ , equal to the grade of the system,  $n$ . Then we can obtain a (locally) linear system via state feedback :

$$v := L_a^r c + u L_b L_a^{r-1} c \quad \longrightarrow \quad u = \frac{1}{L_b L_a^{r-1} c} (v - L_a^r c)$$

System  $S$  has relative degree  $r$  in  $x^\circ$  if, in a neighborhood of  $x^\circ$  we got :

$$L_b L_a^k c \equiv 0, \quad k = 0, 1, 2, \dots, r-2 \quad \text{and} \quad [L_b L_a^{r-1} c]_x \neq 0$$

Then, to build the feedback linearization block we need to compute 2 functions :  $L_a^r c$  and  $L_b L_a^{r-1} c$

In order to do so we define the Lie derivative operator along a vector field  $f$  as : 
$$Lf := \sum_{i=1}^n f_i(x) \frac{\delta}{\delta x_i}$$

We have done all the other computations through a MatLab script

(results are on “symbolic\_derivatives.m” file attached to the presentation)





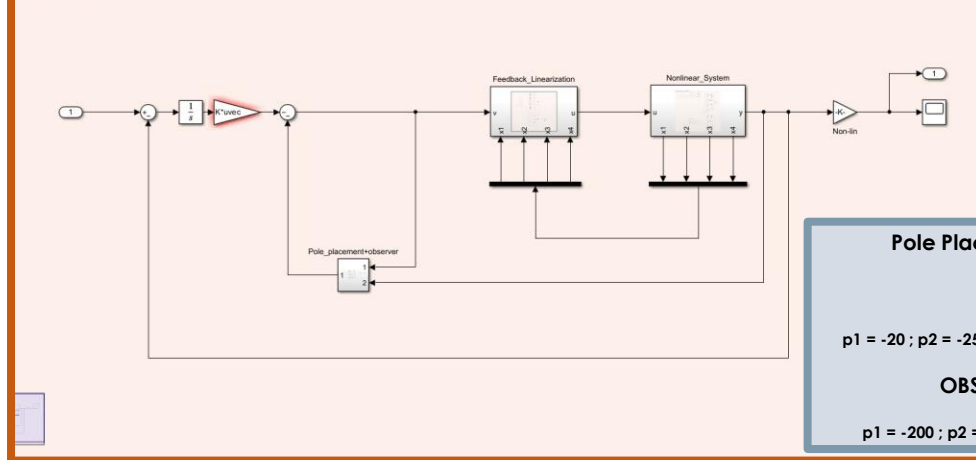
# FeedBack Linearization

## Model + Stabilizing State FeedBack Law

$$M = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \quad N = \begin{bmatrix} 0 \\ 0 \\ 0 \\ 1 \end{bmatrix}$$

$$C_{fl} = \begin{bmatrix} 1 & 0 & 0 & 0 \end{bmatrix} \quad D_{fl} = 0$$

FEEDBACK LINEARIZATION + POLE PLACEMENT



Pole Placement Controller

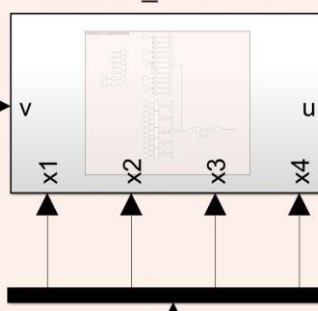
POLES

p1 = -20 ; p2 = -25 ; p3 = -30 ; p4 = -35 ; p5 = -40

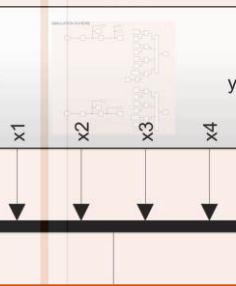
OBSERVER POLES

p1 = -200 ; p2 = -250 ; p3 = -300 ; p4 = -350

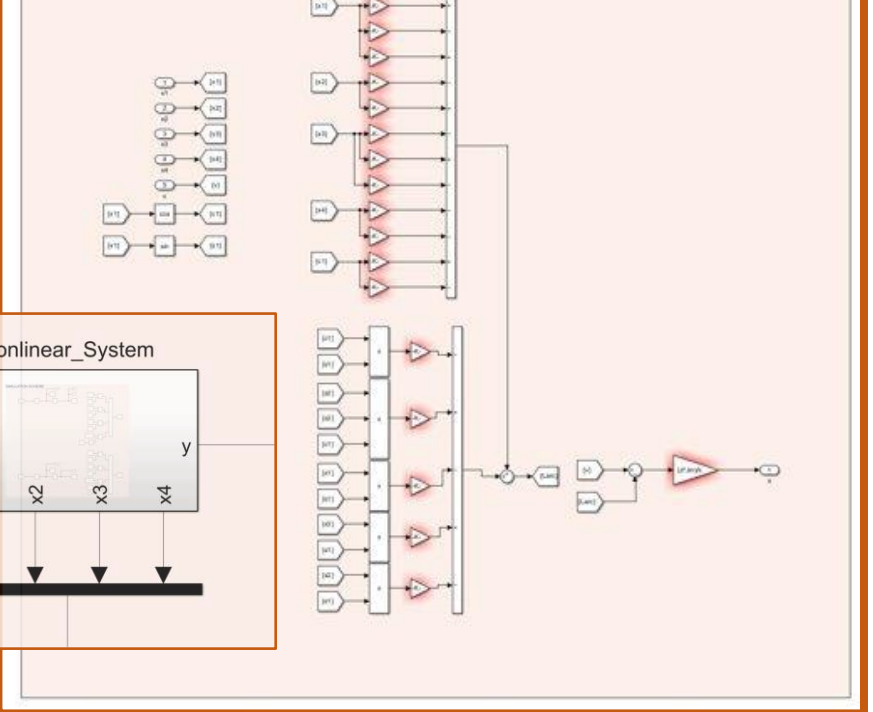
Feedback\_Linearization



Nonlinear\_System



FEEDBACK LINEARIZATION



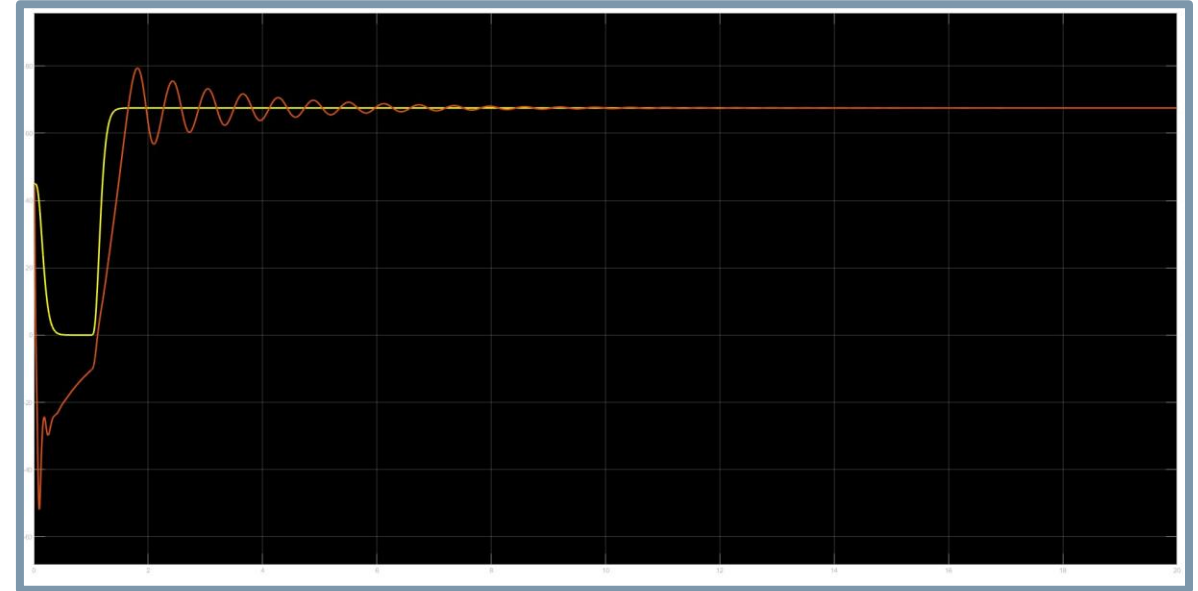
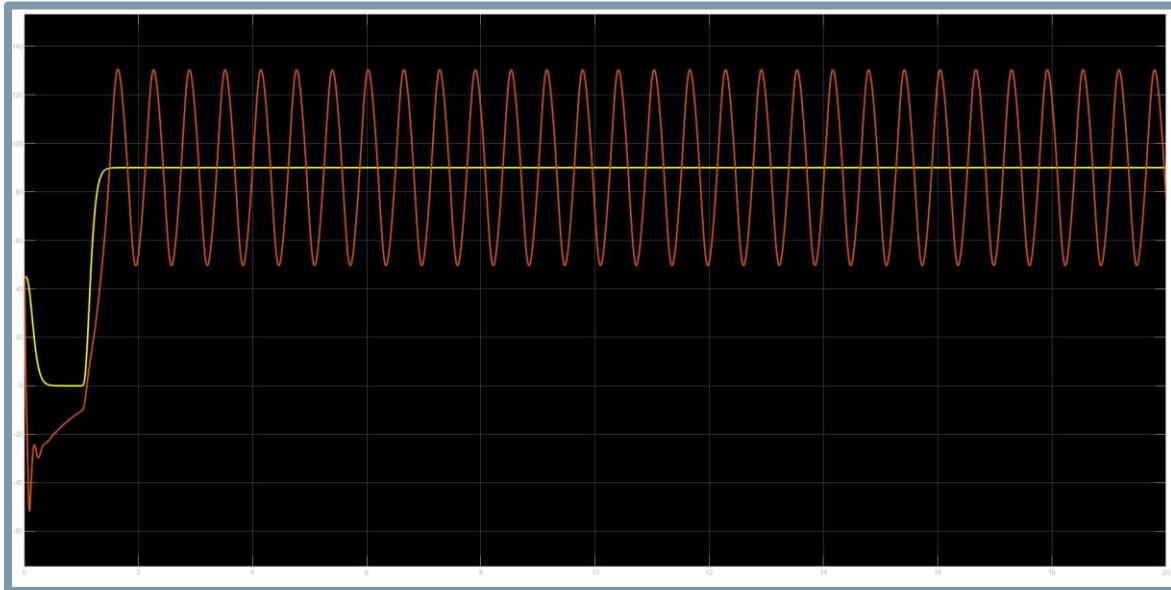
As stabilizing control law we used a **pole placement approach** and we extended the system with an integrator in order to track a reference of  $\theta_l$

# FeedBack Linearization

## Simulation Performance

### Request 5 : Verify the results in simulation and comment

We run **two different simulations** with the same reference values we used for the linear tangent approximation. As we can see the feedback linearization performs better than the linear tangent approximation model tracking the reference way faster without oscillations.

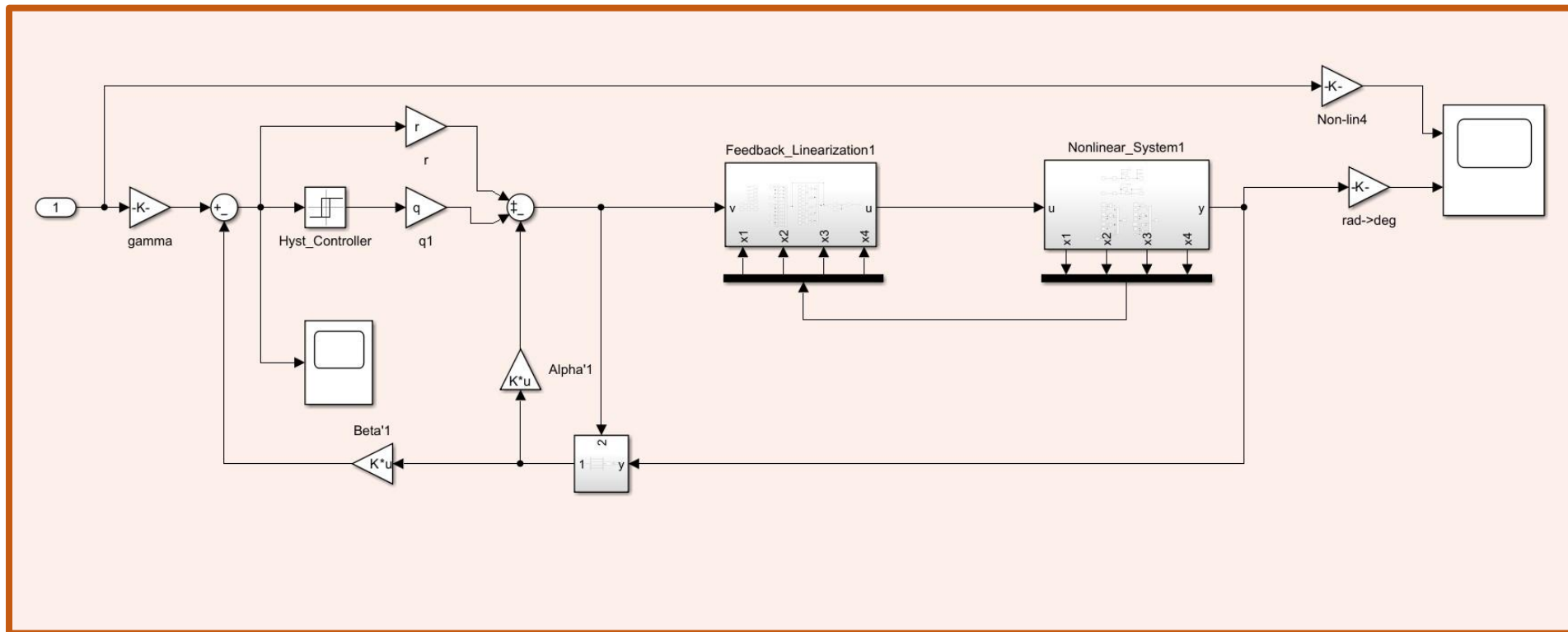


Moreover we can also see that the feedback linearized system can track a reference far from the equilibrium point in a more performant way with respect to the Linear Tangent Approximation.

# Variable Structure Control

## Model

**Request 6 : Propose a Variable Structure Controller for the system linearized via feedback linearization to make  $y(t) = \frac{\pi}{4} - \theta_l$  track some (constant) reference signal  $y^\circ$**



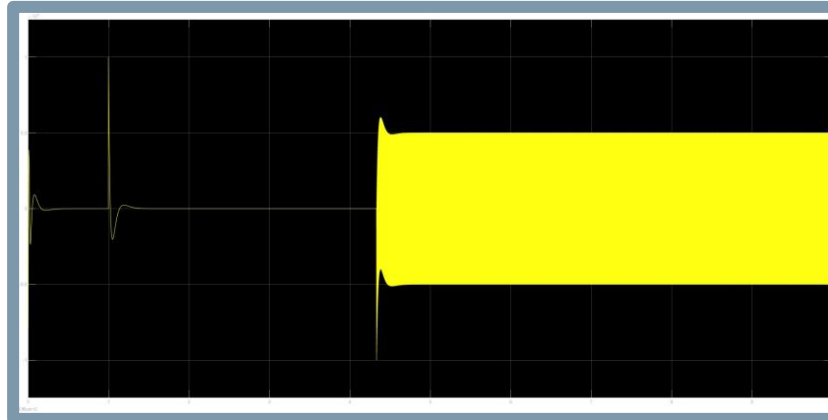
# Variable Structure Control

## Simulation Performance

**Request 7 : Analyze the performance achieved by the variable structure controller devised in point 6 focusing on high frequency switching**



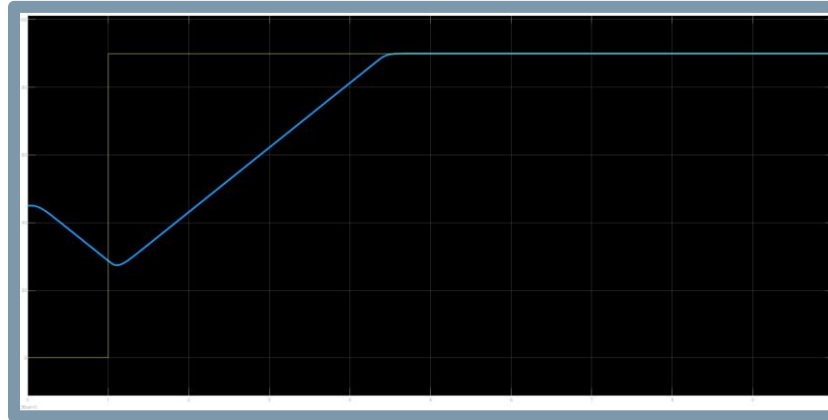
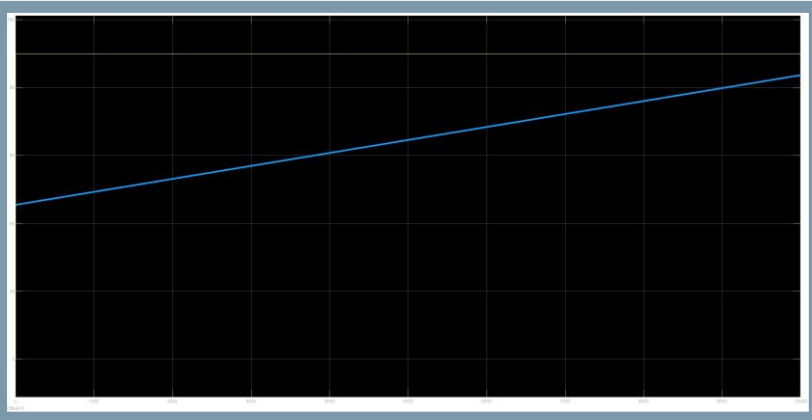
$q = 1 ; r = 0$



$q = 5000 ; r = 0$

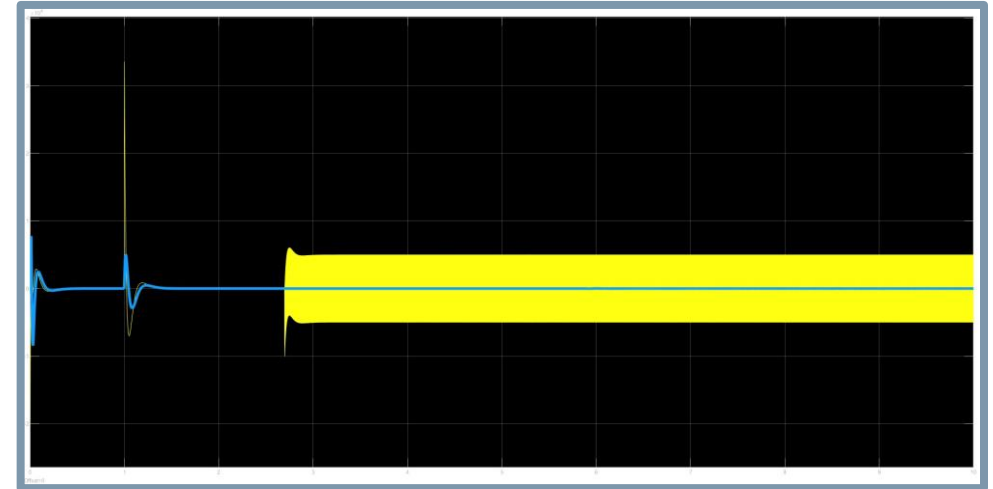
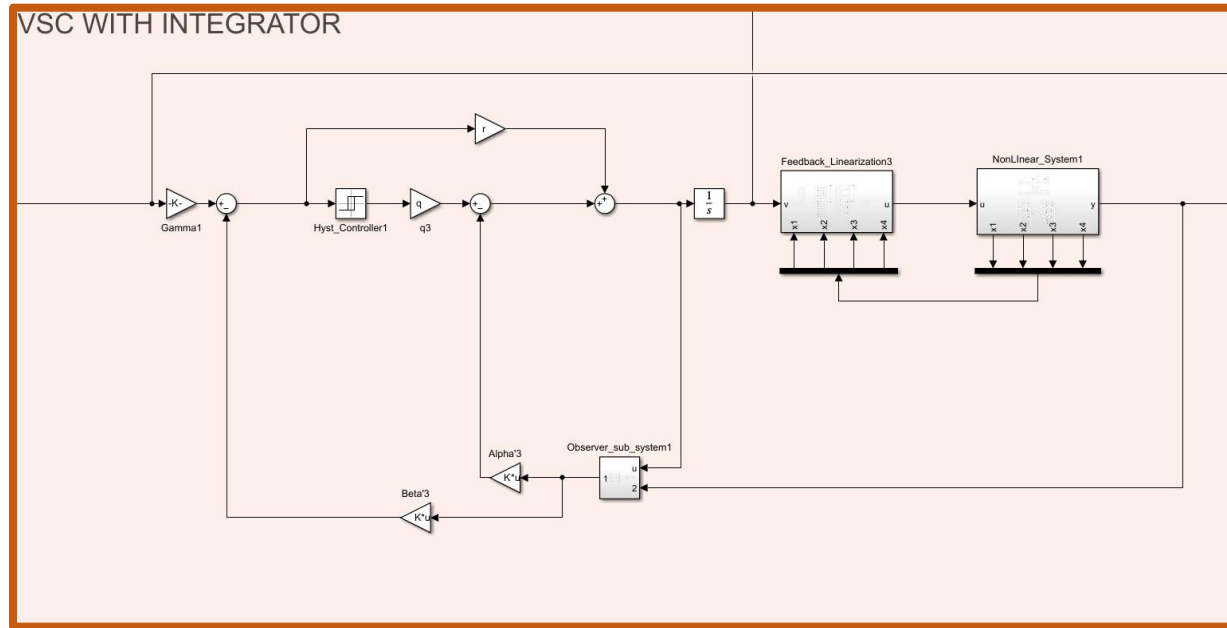


$q = 10 ; r = 100$



# Variable Structure Control

High Frequency Switching countermeasure + Simulation performance



$q = 5000 ; r = 1$

