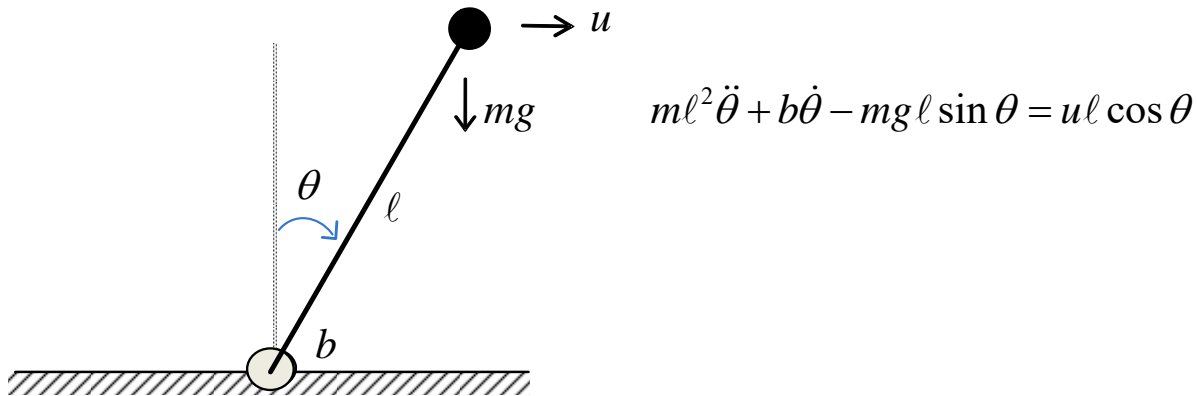


MEM 255 (W2019) Introduction to Controls HW#7

Consider the following simple inverted pendulum system. Note that unlike the cart-inverted pendulum, the base of the simple pendulum is fixed, and the control input force $u(t)$ is perpendicular to the gravity.



- (a) Use the Lagrange approach to verify the above equation of motion for the simple inverted pendulum system.
- (b) Define the state variables $x_1 = \theta$, $x_2 = \dot{\theta}$, and then find the nonlinear state equation of the system.
- (c) Assume $\frac{g}{\ell} = 9$, $\frac{b}{m\ell^2} = 0.6$, $\frac{1}{m\ell} = 1$, and rewrite the nonlinear state equation.
- (d) Show that $\bar{x}_U = [0 \ 0]^T$ and $\bar{x}_D = [\pi \ 0]^T$ are the equilibriums of the system associated with the upright and downward pendulum positions, respectively.
- (e) Find the linearized state equation $\dot{x}(t) = A_U x(t) + B_U u(t)$ associated with the upright equilibrium $\bar{x}_U = [0 \ 0]^T$ and show that the equilibrium is unstable based on the eigenvalues of A_U .
- (f) Build a Simulink/MATLAB program based on the nonlinear state equation obtained in (b), and use it to verify that the upright equilibrium $\bar{x}_U = [0 \ 0]^T$ is unstable.
- (g) Find the linearized state equation $\dot{x}(t) = A_D x(t) + B_D u(t)$ associated with the downward equilibrium $\bar{x}_D = [\pi \ 0]^T$ and show that the equilibrium is stable based on the eigenvalues of A_D .
- (h) Use the same Simulink/MATLAB program you built in (f) to verify that the downward equilibrium $\bar{x}_D = [\pi \ 0]^T$ is stable.
- (i) Find a state feedback controller, $u(t) = Fx(t) = F_1 x_1(t) + F_2 x_2(t)$, so that the closed-loop system eigenvalues (the eigenvalues of $A_U + B_U F$) are in the strictly left half of the complex plane.
- (j) Modify the Simulink/MATLAB program you used in (f) and (h) to incorporate the state feedback controller you designed in (i). Conduct simulation to verify that the originally unstable upright equilibrium $\bar{x}_U = [0 \ 0]^T$ has become stable.

Hint:

$$(d) \quad \dot{x} = \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} x_2 \\ ? \end{bmatrix} = f(x, u) = \begin{bmatrix} f_1(x_1, x_2, u) \\ f_2(x_1, x_2, u) \end{bmatrix}$$

$$u = 0, \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = 0 \rightarrow \begin{cases} x_2 = 0 \\ ?? = 0 \end{cases} \rightarrow \begin{bmatrix} \bar{x}_1 \\ \bar{x}_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \quad \text{or} \quad \begin{bmatrix} \pi \\ 0 \end{bmatrix}$$

$$(e) \quad \dot{x}(t) = A_U x(t) + B_U u(t)$$

$$\text{where} \quad A_U = J_x = \left[\frac{\partial f}{\partial x} \right]_{\bar{x}} = \begin{bmatrix} \partial f_1 / \partial x_1 & \partial f_1 / \partial x_2 \\ \partial f_2 / \partial x_1 & \partial f_2 / \partial x_2 \end{bmatrix}_{\bar{x} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}} = \dots$$

$$B_U = J_u = \left[\frac{\partial f}{\partial u} \right]_{\bar{x}} = \begin{bmatrix} \partial f_1 / \partial u \\ \partial f_2 / \partial u \end{bmatrix}_{\bar{x} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}} = \dots$$

Note that the eigenvalues of A_U are ? and ?, which verify that the equilibrium is unstable.

(f) Modify from the program in (j)

$$(g) \quad \dot{x}(t) = A_D x(t) + B_D u(t)$$

$$\text{where} \quad A_D = J_x = \left[\frac{\partial f}{\partial x} \right]_{\bar{x}} = \begin{bmatrix} \partial f_1 / \partial x_1 & \partial f_1 / \partial x_2 \\ \partial f_2 / \partial x_1 & \partial f_2 / \partial x_2 \end{bmatrix}_{\bar{x} = \begin{bmatrix} \pi \\ 0 \end{bmatrix}} = \dots$$

$$B_D = J_u = \left[\frac{\partial f}{\partial u} \right]_{\bar{x}} = \begin{bmatrix} \partial f_1 / \partial u \\ \partial f_2 / \partial u \end{bmatrix}_{\bar{x} = \begin{bmatrix} \pi \\ 0 \end{bmatrix}} = \dots$$

Note that the eigenvalues of A_D are ??, which verify that the equilibrium is stable.

(h) Modify from the program in (j)

$$(i) \quad \dot{x}(t) = A_U x(t) + B_U u(t)$$

Let the state feedback be $u(t) = Fx(t) = [F_1 \quad F_2]x(t)$, then the closed-loop system will be

$$\dot{x}(t) = A_U x(t) + B_U u(t) = A_U x(t) + B_U Fx(t) = (A_U + B_U F)x(t)$$

Choose the desired close-loop systems poles to be λ_1 and λ_2 , which are in the left half of the complex plane. Then find F_1 and F_2 as follows.

$$\det[\lambda I - (A_U + B_U [F_1 \quad F_2])] = (\lambda - \lambda_1)(\lambda - \lambda_2) \rightarrow F_1 \quad \text{and} \quad F_2$$

(j) MATLAB programs

```
% filename: HW7_SIP_1.m, Simple Inverted Pendulum
% BC Chang, Drexel University, on 2/22/2019
% Program running sequence: MATLAB R2015a or later versions
```

```

% 1. HW7_SIP_1.m
% 2. HW7_SIP_tmprpns_2.mdl
% 3. HW7_plot_3.m

% x1_dot=x2
% x2_dot=9*sinx1-0.6*x2+cosx1*u
% Initial Conditions
x10_deg = 15 %in degrees
x10=x10_deg*3.1416/180 % in radians
x20=0

AU=[0 1; 9 -0.6]
BU=[0; 1]
eig(AU)

% u=F*x
%Choose desired damping ratio ze and natural frequency wn
ze=0.8
wn=5
F=[-9-wn^2 0.6-2*ze*wn] % Use this to obtain closed-loop response

%Zero Feedback
%F=[0 0];

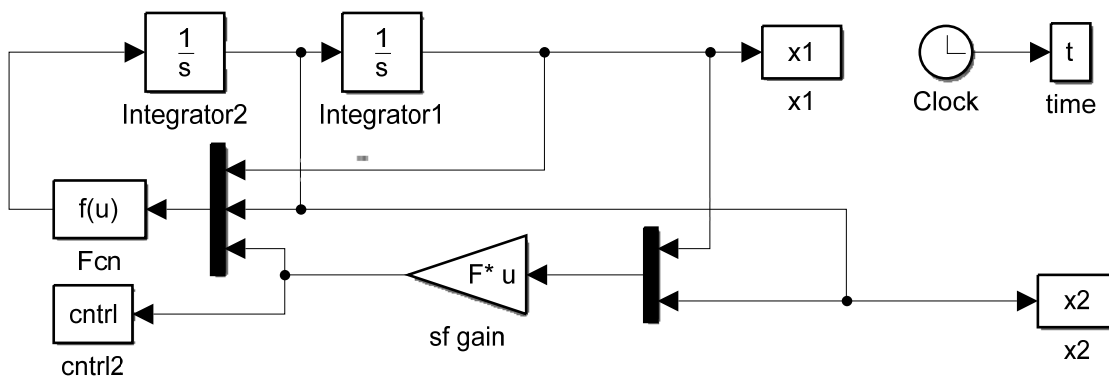
Acl=A+B*F
eig(Acl)

%Part 3 Simulation
% Using the the following script to run the simulink and plot the
% simulation results
sim_time=5
%sim_time = 20

sim_options = simset('SrcWorkspace ', 'current',...
    'DstWorkspace ', 'current');
%open('CSDInvP_sim_2_wFriction') %
%sim('CSDInvP_sim_2_wFriction', [0, sim_time], sim_options);
open('HW7_SIP_tmprpns_2') %
sim('HW7_SIP_tmprpns_2', [0, sim_time], sim_options);
run('HW7_plot_3')

```

HW7_SIP_tmprpns_2.mdl



In Fcn Expression box: type the following

$$9*\sin(u(1))-0.6*u(2)+\cos(u(1))*u(3)$$

In Integrator1 Initial condition box: type x10

In Integrator1 Initial condition box: type 0

In cntrl2 Variable name box: type cntrl

In x1 Variable name box: type x1

In x2 Variable name box: type x2

In sf gain Gain box: type F

In time Variable name box: type t

```
%Filename: HW7_plot3.m
%Run this program after HW7_SIP-1.m, and HW7_SIP_tmprns_2.mdl
x1_deg=x1*180/pi;
x2_deg=x2*180/pi;

figure(10)
subplot(1,3,1)
plot(t,cntrl) % plot cntrl
grid on
grid minor
xlabel('t'), ylabel('cntrl')

subplot(1,3,2)
plot(x1_deg,x2_deg) % plot phase plane trajectory
grid on
grid minor
xlabel('x1'), ylabel('x2')

subplot(1,3,3)
plot(t,x1_deg,t,x2_deg) % plot x1, x2 time responses
legend('x1 deg','x2 (deg/s)')
grid on
grid minor
xlabel('t'), ylabel('x1 and x2')
```

