

# Lecture 13: Rigid body kinematics – Kinematic differential equations

- Brief recap of representations of rotation
  - Rotation matrices (6.4)
  - Euler angles (6.5)
    - 3-parameter representation of rotations
    - Roll-pitch-yaw
  - Angle-axis, Euler-parameters (6.6, 6.7)
    - 4-parameter representation of rotations
  - Angular velocity (6.8)
  - Kinematic differential equations
- Today:
  - Rigid body kinematics: Configuration
  - Newton Euler equation

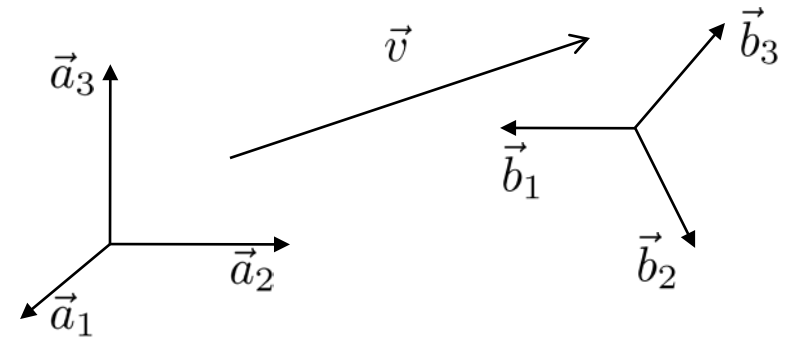
Book: Ch. 6.9, 6.12, 6.13, 7.1

# Rotation matrices

The rotation matrix from  $a$  to  $b$   $\mathbf{R}_b^a$  is used to

- **Transform** a coordinate vector from  $b$  to  $a$

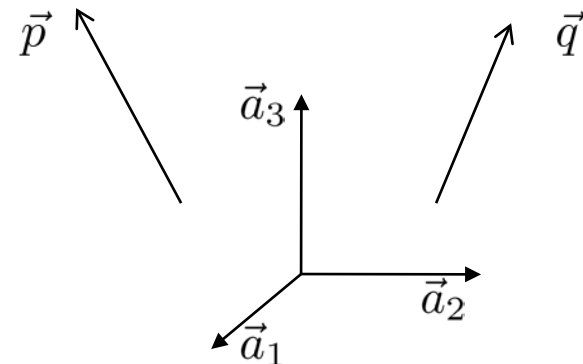
$$\mathbf{v}^a = \mathbf{R}_b^a \mathbf{v}^b$$



- **Rotate** a vector  $\vec{p}$  to vector  $\vec{q}$ . If decomposed in  $a$ ,

$$\mathbf{q}^a = \mathbf{R}_b^a \mathbf{p}^a$$

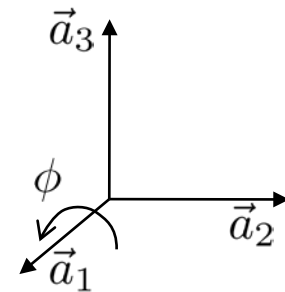
such that  $\mathbf{q}^b = \mathbf{p}^a$ .



# Simple rotations

- Simple rotation = rotation about an axis
- Example: Rotation matrix for rotation about x-axis:

$$\mathbf{R}_{x,\phi} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & \cos \phi & -\sin \phi \\ 0 & \sin \phi & \cos \phi \end{pmatrix}$$



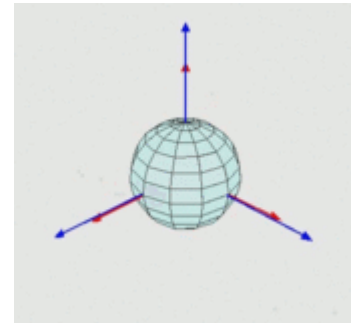
# Representations of rotations

- Rotation matrix
  - Easy to use, but not to visualize (also over-parameterized, 9 parameters)

## Euler's Theorem:

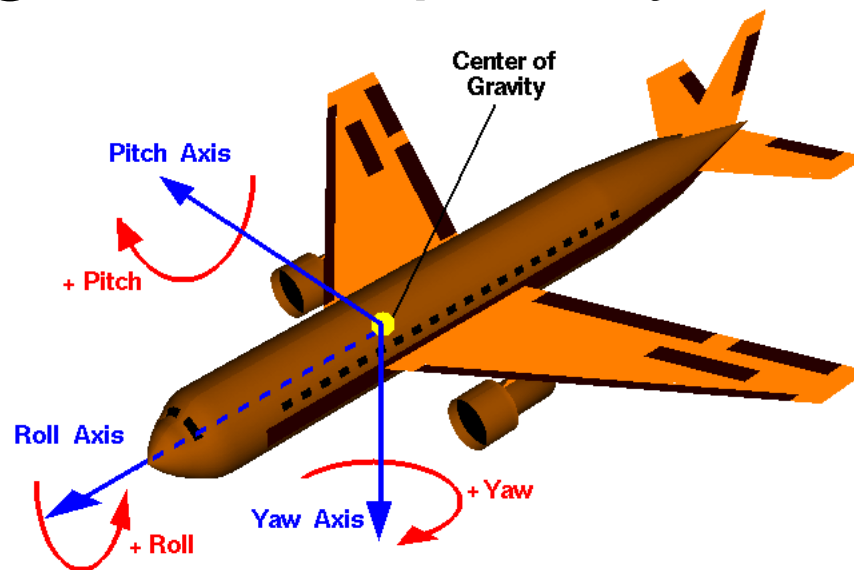
“Any two independent orthonormal coordinate frames can be related by a sequence of rotations (not more than three) about coordinate axes, where no two successive rotations may be about the same axis.”

- Three rotations about axes are enough to specify any rotation
  - These representations are called Euler angles
    - 12 different combinations possible
    - Most common(?): Roll-pitch-yaw
  - Natural and (in many cases) simple to use, very much used
  - Problem: Singularity (more on this today)
- Angle-axis, Euler-parameters
  - 4-parameter representations of rotations
  - No singularity problems



Source: Wikipedia

# Euler-angles: Roll-pitch-yaw

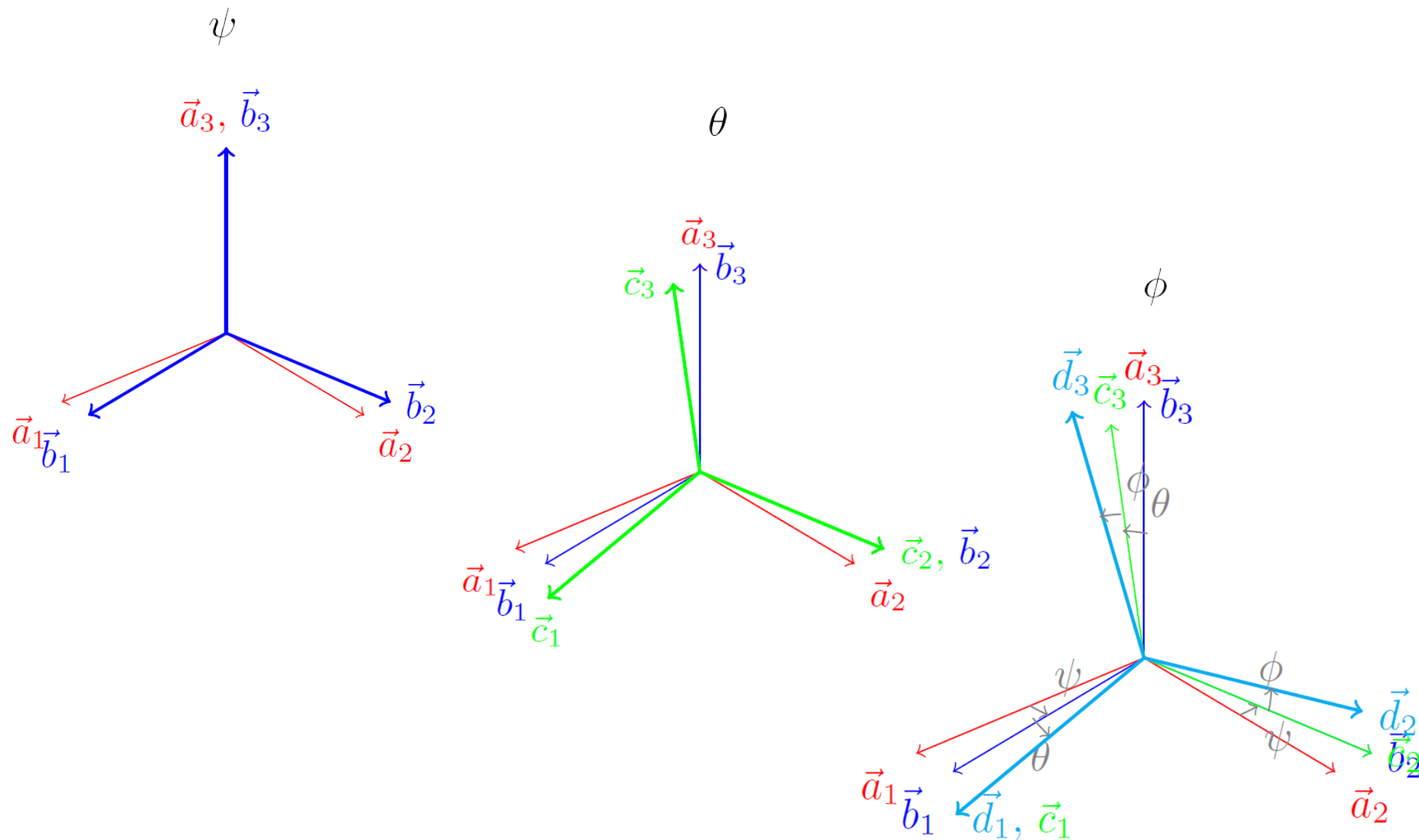


- Rotation  $\psi$  about z-axis,  $\theta$  about (rotated) y-axis,  $\phi$  about (rotated) x-axis

$$\mathbf{R}_b^a = \mathbf{R}_{z,\psi} \mathbf{R}_{y,\theta} \mathbf{R}_{x,\phi}$$

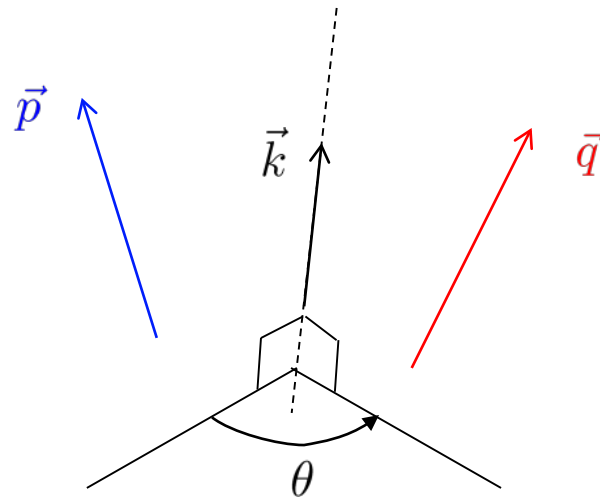
$$\mathbf{R}_b^a = \begin{pmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{pmatrix} \begin{pmatrix} \cos \theta & 0 & \sin \theta \\ 0 & 1 & 0 \\ -\sin \theta & 0 & \cos \theta \end{pmatrix} \begin{pmatrix} 1 & 0 & 0 \\ 0 & \cos \phi & -\sin \phi \\ 0 & \sin \phi & \cos \phi \end{pmatrix}$$

# Euler angles



# Angle-axis representation of rotations

All rotations can be represented as a simple rotation around an axis



- Angle-axis parameters:

- Coordinate free:  $\vec{k}, \theta$

$$\vec{q} = \underbrace{\left( \cos \theta \vec{I} + \sin \theta \vec{k}^\times + (1 - \cos \theta) \vec{k} \vec{k} \right)}_{\vec{R}_{\vec{k}, \theta}} \cdot \vec{p}$$

- With coordinates:  $\mathbf{k}^a, \theta$

$$\mathbf{R}_b^a = \mathbf{R}_{\mathbf{k}, \theta} = \cos \theta \mathbf{I} + \sin \theta (\mathbf{k}^a)^\times + (1 - \cos \theta) \mathbf{k}^a (\mathbf{k}^a)^\top$$

# Euler parameters

- Euler parameters are closely related to angle-axis:

- Coordinate-free:

$$\eta = \cos \frac{\theta}{2}$$

$$\vec{\epsilon} = \vec{k} \sin \frac{\theta}{2}$$

- With coordinates:

$$\eta = \cos \frac{\theta}{2}$$

$$\epsilon = \mathbf{k} \sin \frac{\theta}{2}$$

- Rotation matrix (on coordinate form):

$$\mathbf{R}(\eta, \epsilon) = \mathbf{I} + 2\eta\epsilon^\times + 2\epsilon^\times\epsilon^\times$$

- Much used, since:
  - Compact, **singularity-free** representation of orientation
  - No trigonometric terms in expression for rotation matrix
  - $\eta^2 + \vec{\epsilon} \cdot \vec{\epsilon} = 1$ : Easy to normalize (avoid roundoff errors)
    - Rotation matrices may tend to become non-orthogonal when simulated
  - Euler parameters are (*unit*) *quaternions*:
    - Quaternions are generalized complex numbers
    - Can use algebra of quaternions for calculations and analysis



# Derivatives of rotations

- Derivative of position  $\mathbf{r}$  is velocity,  $\dot{\mathbf{r}} = \mathbf{v}$ .
- Derivative of rotation matrix  $\mathbf{R}_b^a$  is  $\dot{\mathbf{R}}_b^a$ . What is this?
- Seems natural that a concept of angular velocity should be involved, but how?
- What are derivatives of representations of rotations?
  - Derivatives of Euler angles? Euler parameters?
  - These are the kinematic differential equations!

# Angular velocity

- The rotation matrix is orthogonal:

$$\mathbf{R}_b^a (\mathbf{R}_b^a)^\top = \mathbf{I}$$

- Differentiate:

$$\frac{d}{dt} [\mathbf{R}_b^a (\mathbf{R}_b^a)^\top] = \dot{\mathbf{R}}_b^a (\mathbf{R}_b^a)^\top + \mathbf{R}_b^a (\dot{\mathbf{R}}_b^a)^\top = \mathbf{0}$$

- If we define  $\mathbf{S} = \dot{\mathbf{R}}_b^a (\mathbf{R}_b^a)^\top$ , this says that  $\mathbf{S} + \mathbf{S}^\top = \mathbf{0}$  which means that  $\mathbf{S}$  is **skew symmetric**.

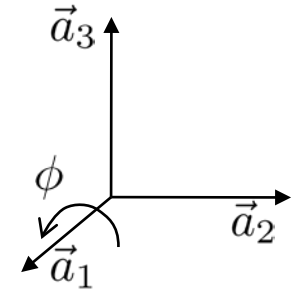
$$\mathbf{S} = \begin{pmatrix} 0 & -\omega_3 & \omega_2 \\ \omega_3 & 0 & -\omega_1 \\ -\omega_2 & \omega_1 & 0 \end{pmatrix} = (\boldsymbol{\omega}_{ab}^a)^\times$$

- The vector  $\boldsymbol{\omega}_{ab}^a$  defined by  $(\boldsymbol{\omega}_{ab}^a)^\times = \dot{\mathbf{R}}_b^a (\mathbf{R}_b^a)^\top$  is the **angular velocity of frame  $b$  relative to frame  $a$**  (decomposed in  $a$ )
- The equation  $\dot{\mathbf{R}}_b^a = (\boldsymbol{\omega}_{ab}^a)^\times \mathbf{R}_b^a$  is the **kinematic differential equation** for rotation matrices

# Angular velocity of simple rotations

- Rotation about x-axis:

$$\mathbf{R}_{x,\phi} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & \cos \phi & -\sin \phi \\ 0 & \sin \phi & \cos \phi \end{pmatrix}$$



- We calculate  $(\omega_{ab}^a)^\times = \dot{\mathbf{R}}_b^a (\mathbf{R}_b^a)^\top$ :

$$\dot{\mathbf{R}}_{x,\phi} (\mathbf{R}_{x,\phi})^\top = \begin{pmatrix} 0 & 0 & 0 \\ 0 & -\sin \phi & -\cos \phi \\ 0 & \cos \phi & -\sin \phi \end{pmatrix} \dot{\phi} \begin{pmatrix} 1 & 0 & 0 \\ 0 & \cos \phi & \sin \phi \\ 0 & -\sin \phi & \cos \phi \end{pmatrix} = \begin{pmatrix} 0 & 0 & 0 \\ 0 & 0 & -\dot{\phi} \\ 0 & \dot{\phi} & 0 \end{pmatrix}$$

- That is:

$$\omega_x = \begin{pmatrix} \dot{\phi} \\ 0 \\ 0 \end{pmatrix}$$

- Similar for rotations around y- and z-axis:  $\omega_y = \begin{pmatrix} 0 \\ \dot{\theta} \\ 0 \end{pmatrix}, \quad \omega_z = \begin{pmatrix} 0 \\ 0 \\ \dot{\psi} \end{pmatrix}$

- Angle-axis representations (constant axis):

$$\omega_{ab}^a = \dot{\theta} \mathbf{k}^a$$

# Composite rotations

- Given
  - composite rotation  $\mathbf{R}_d^a = \mathbf{R}_b^a \mathbf{R}_c^b \mathbf{R}_d^c$ , and
  - individual angular velocities  $\omega_{ab}^a$ ,  $\omega_{bc}^b$ , and  $\omega_{cd}^c$

How to calculate the composite angular velocity  $\omega_{ad}^a$ ?

- It can be shown (easy, see book p. 241) that

$$\vec{\omega}_{ad} = \vec{\omega}_{ab} + \vec{\omega}_{bc} + \vec{\omega}_{cd}$$

- On coordinate form:

$$\omega_{ad}^a = \omega_{ab}^a + \omega_{bc}^a + \omega_{cd}^a$$

- So:

$$\omega_{ad}^a = \omega_{ab}^a + \mathbf{R}_b^a \omega_{bc}^b + \mathbf{R}_b^a \mathbf{R}_c^b \omega_{cd}^c$$

# Kinematic differential equation of Euler angles

$$\mathbf{R}_d^a = \mathbf{R}_b^a \mathbf{R}_c^b \mathbf{R}_d^c = \mathbf{R}_{z,\psi} \mathbf{R}_{y,\theta} \mathbf{R}_{x,\phi}$$

$$\vec{\omega}_{ad} = \vec{\omega}_{ab} + \vec{\omega}_{bc} + \vec{\omega}_{cd} = \dot{\psi} \vec{a}_3 + \dot{\theta} \vec{b}_2 + \dot{\phi} \vec{c}_1$$

$$\begin{aligned} \underline{\omega}_{ad}^a &= \begin{pmatrix} 0 \\ 0 \\ \dot{\psi} \end{pmatrix} + \mathbf{R}_z(\psi) \begin{pmatrix} 0 \\ \dot{\theta} \\ 0 \end{pmatrix} + \mathbf{R}_z(\psi) \mathbf{R}_y(\theta) \begin{pmatrix} \dot{\phi} \\ 0 \\ 0 \end{pmatrix} \\ &= \mathbf{E}_a(\underline{\Phi}) \dot{\underline{\Phi}} \qquad \underline{\Phi} = \begin{pmatrix} \phi \\ \theta \\ \psi \end{pmatrix} \end{aligned}$$

$$\dot{\underline{\Phi}} = \mathbf{E}_a^{-1}(\underline{\Phi}) \underline{\omega}_{ad}^a \qquad \theta \neq 90^\circ$$

# Kinematic differential equation of Euler parameter

$$\mathbf{R}_b^a = \mathbf{R}(\eta, \underline{\varepsilon})$$

$$\dot{\mathbf{R}}_b^a = (\underline{\omega}_{ab}^a)^\times \mathbf{R}_b^a$$

- It can be derived (quaternion algebra p. 248)

$$\dot{\eta} = -\frac{1}{2} \underline{\varepsilon}^T \underline{\omega}_{ab}^a$$

$$\dot{\underline{\varepsilon}} = \frac{1}{2} (\eta \mathbf{I} - \underline{\varepsilon}^\times) \underline{\omega}_{ab}^a$$

# Differentiation of vectors (6.8.5, 6.8.6)

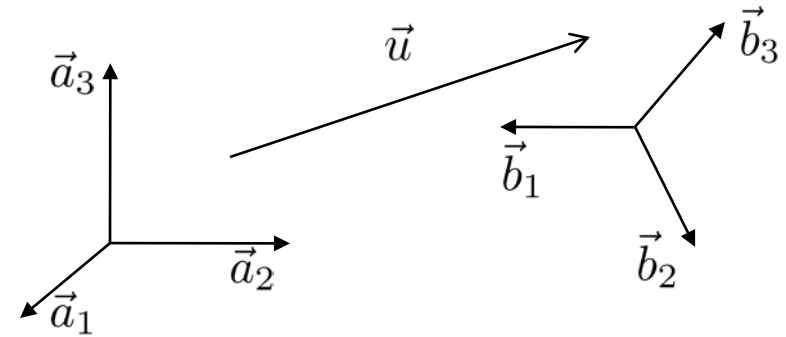
- Coordinate representation:

$$\mathbf{u}^a = \mathbf{R}_b^a \mathbf{u}^b$$

- Differentiation:

$$\dot{\mathbf{u}}^a = \mathbf{R}_b^a \dot{\mathbf{u}}^b + \dot{\mathbf{R}}_b^a \mathbf{u}^b$$

$\dot{\mathbf{R}}_b^a = \mathbf{R}_b^a (\boldsymbol{\omega}_{ab}^b)^\times$



$$\dot{\mathbf{u}}^a = \mathbf{R}_b^a \left[ \dot{\mathbf{u}}^b + (\boldsymbol{\omega}_{ab}^b)^\times \mathbf{u}^b \right]$$

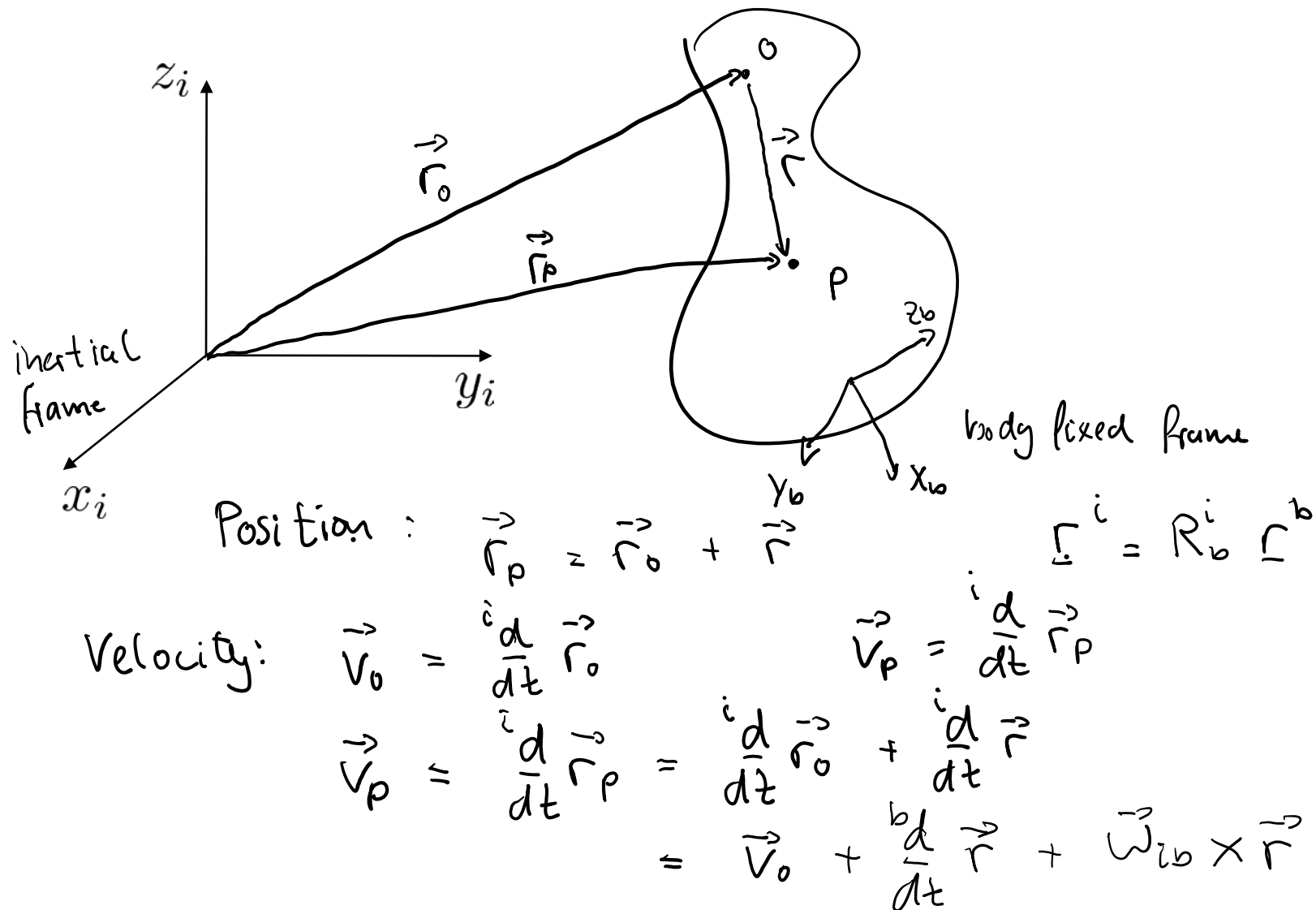
- On vector form:

$$\frac{{}^a d}{dt} \vec{u} = \frac{{}^b d}{dt} \vec{u} + \vec{\omega}_{ab} \times \vec{u}$$

Note! Generally,

$$\dot{\mathbf{u}}^a \neq \mathbf{R}_b^a \dot{\mathbf{u}}^b$$

# Kinematics of rigid body I





# Kinematics of rigid body II

$$\vec{a}_o = \frac{d^2}{dt^2} \vec{r}_o$$

$$\vec{a}_p = \frac{d^2}{dt^2} \vec{r}_p$$

$$\vec{a}_{ib} : \frac{d}{dt} \vec{\omega}_{ib} = \frac{d}{dt} \vec{\omega}_{ib} + \cancel{\vec{\omega}_{ib} \times \vec{\omega}_{ib}}$$

$$\frac{d^2}{dt^2} \vec{r}_p = \frac{d^2}{dt^2} \vec{r}_o + \frac{d^2}{dt^2} \vec{r}$$

$$\vec{a}_p = \vec{a}_o + \frac{d}{dt} \left( \frac{d}{dt} \vec{r} \right)$$

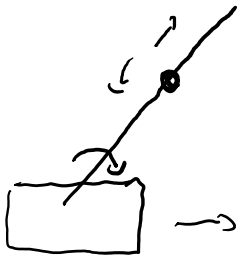
$$= \vec{a}_o + \frac{d}{dt} \left( \frac{d}{dt} \vec{r} + \vec{\omega}_{ib} \times \vec{r} \right)$$

$$= \vec{a}_o + \frac{d}{dt} \frac{d}{dt} \vec{r} + \vec{\omega}_{ib} \times \frac{d}{dt} \vec{r} + \frac{d}{dt} \vec{\omega}_{ib} \times \vec{r} +$$

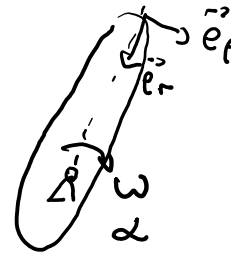
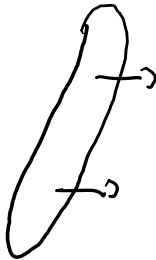
$$\vec{\omega}_{ib} \times (\vec{\omega}_{ib} \times \vec{r}) + \vec{\omega}_{ib} \times \frac{d}{dt} \vec{r}$$

# Kinematics of rigid body III

$$\vec{a}_p = \vec{a}_o + \underbrace{\frac{b}{dt^2} \vec{r}}_{\int \vec{e}_r} + \underbrace{2\vec{\omega}_{ib} \times \frac{b}{dt} \vec{r}}_{\text{Coriolis} \int \vec{e}_e} + \underbrace{\vec{\alpha}_{ib} \times \vec{r}_g}_{\text{transversal} \int \vec{e}_e} + \underbrace{\vec{\omega}_{ib} \times (\vec{\omega}_{ib} \times \vec{r}_g)}_{\text{centripetal} \int -\vec{e}_r}$$



$\Rightarrow$



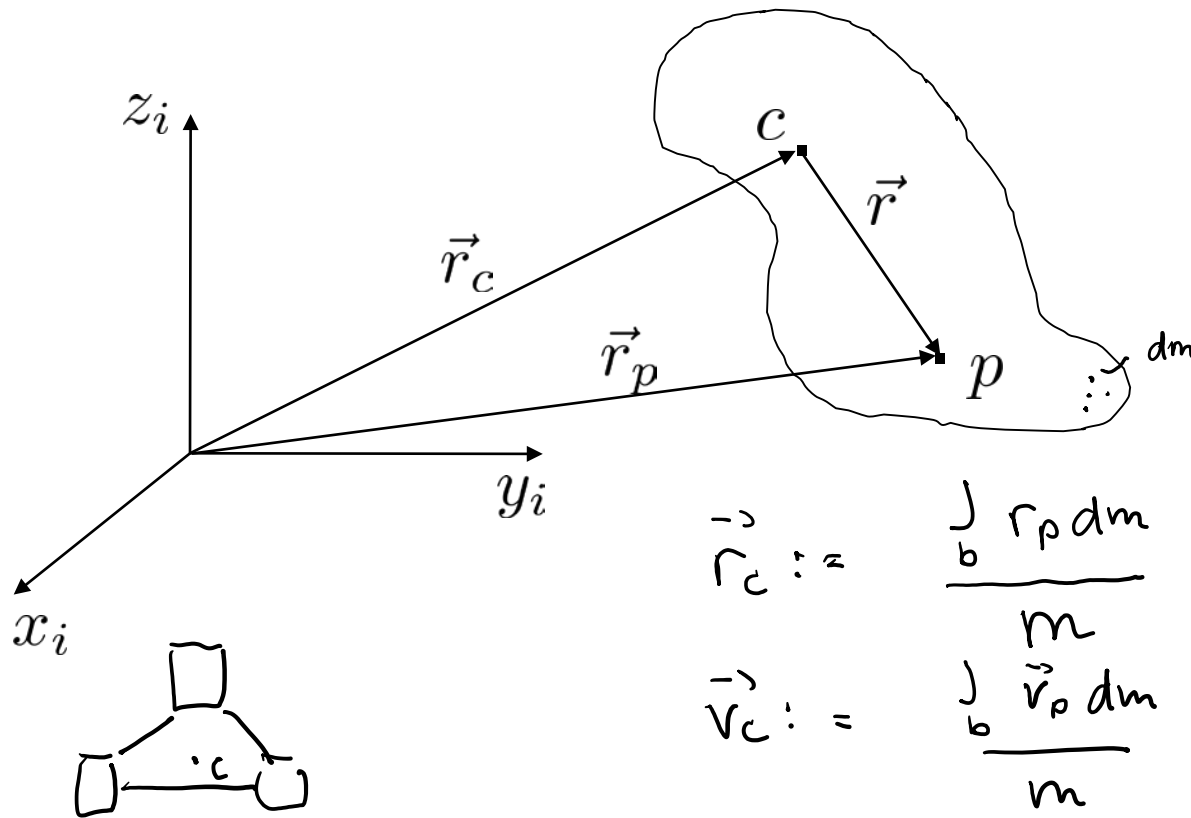
if  $\vec{r}$  is fixed

$$\frac{b}{dt} \vec{r} = 0$$

$$\vec{V}_p = \vec{V}_o + \vec{\omega}_{ib} \times \vec{r}$$

$$\vec{a}_p = \vec{a}_o + \vec{\alpha}_{ib} \times \vec{r} + \vec{\omega}_{ib} \times (\vec{\omega}_{ib} \times \vec{r})$$

# Center of mass



$$\vec{r} = \vec{r}_p - \vec{r}_c$$

$$m = \int_b dm$$

$$\vec{a}_c := \frac{\int_b \vec{a}_p dm}{m}$$

$$\vec{r}_c := \frac{\int_b \vec{r}_p dm}{m}$$

$$\vec{v}_c := \frac{\int_b \vec{v}_p dm}{m}$$

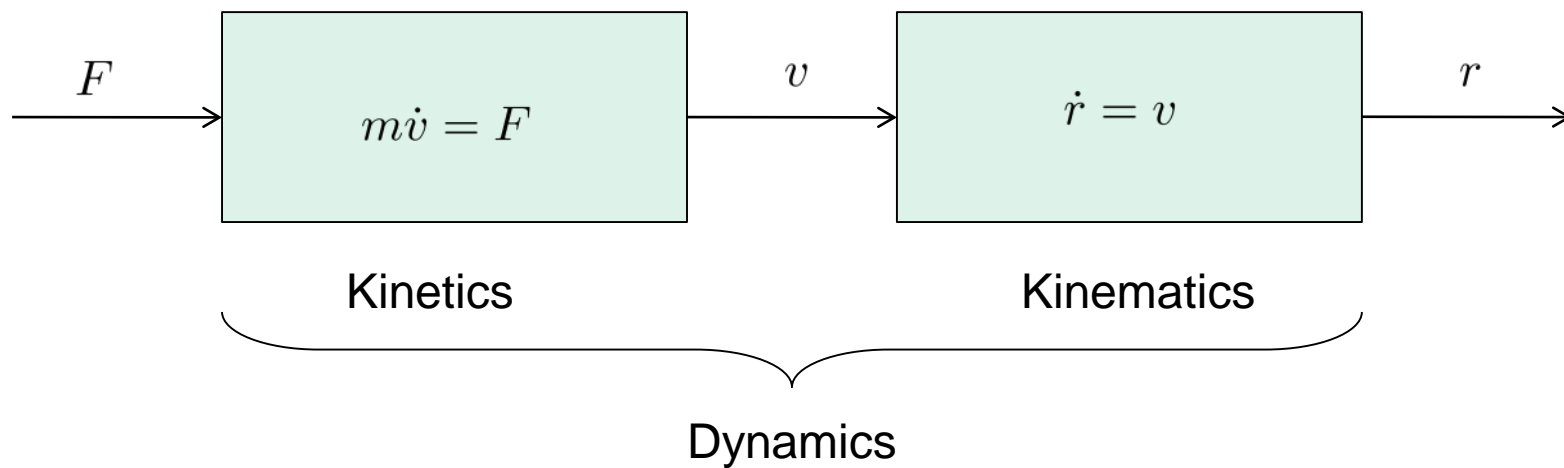
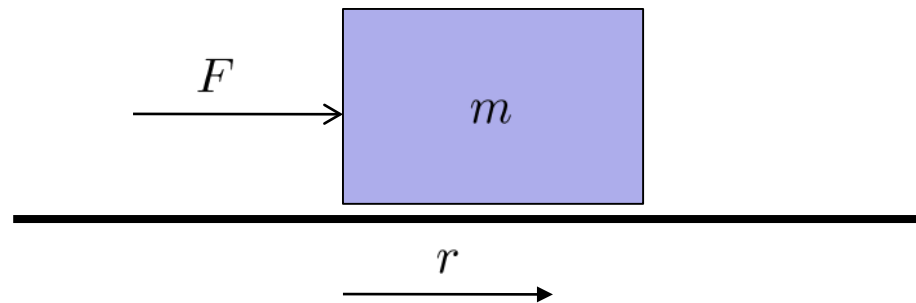
$$\int_b \vec{r} dm = \int_b \vec{r}_p dm - m \vec{r}_c = m \vec{r}_c - m \vec{r}_c = 0$$

# What is rigid body dynamics?

- Rigid body:
  - Wikipedia: “...a rigid body is an idealization of a solid body of finite size in which deformation is neglected.”
- Dynamics = Kinematics + Kinetics
- Kinematics
  - eb.com: “...branch of physics (...) concerned with the geometrically possible **motion** of a body or system of bodies **without consideration of the forces involved** (i.e., causes and effects of the motions).”
  - Book: Ch. 6
- Kinetics
  - eb.com: “...**the effect of forces and torques** on the **motion** of bodies having mass.”
  - Book: Ch. 7, 8.

Remark: Sometimes “dynamics”  
is used for “kinetics” only

# Simplest scalar case



# Newton-Euler equation of motion I

Newton: For a particle :

$$dm \vec{a}_p = \vec{f}_p = \vec{f}_{p,ext} + \vec{f}_{p,int}$$

$$m \vec{a}_c = \underbrace{\int_b \vec{f}_{p,ext} dm}_{=: F_{ext}} + \underbrace{\int_b \vec{f}_{p,int} dm}_{=0}$$

↖ resultant force

Newton's law for center of mass

$$m \vec{a}_c = \vec{F}_{bc}$$

↗  
"resulting force" on  
the center of mass (COM)

# Newton-Euler equation of motion II

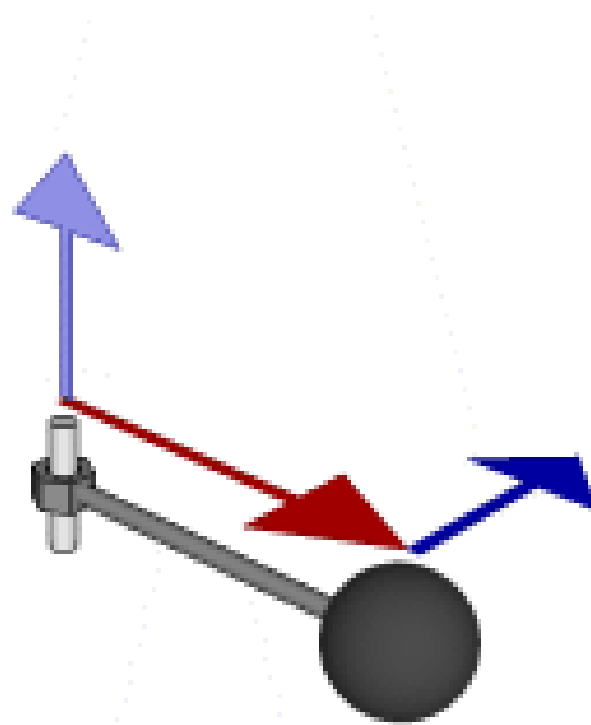
alternatively :  $\vec{p}_c = m \vec{v}_c$   $\frac{d}{dt} \vec{p}_c = F_{bc}$

Define angular momentum of a particle

$$\vec{h}_p = \vec{r}_p \times \vec{p}_p \qquad \vec{p}_p = dm \vec{v}_p$$

# Torque, and linear/angular momentum

$$\begin{aligned}\boldsymbol{\tau} &= \mathbf{r} \times \mathbf{F} \\ \mathbf{L} &= \mathbf{r} \times \mathbf{p}\end{aligned}$$



Source: Wikipedia

- Book:
  - Torque:  $\vec{N}, \vec{T}$
  - Angular momentum:  $\vec{h}$



# Euler's 2nd law of motion I

torque for a particle:  $\vec{\tau}_p = \vec{r}_p \times \vec{f}_p$

$$^i \frac{d}{dt} \vec{h}_p = \underbrace{^i \frac{d}{dt} \vec{r}_p \times \vec{p}_p}_{=0} + \vec{r}_p \times \frac{d}{dt} \vec{p}_p := \vec{\tau}_p$$

Integrate (sum) over the mass

$$\int_b ^i \frac{d}{dt} \vec{h}_p = \int_b \vec{\tau}_p$$

$$\textcircled{1} \quad ^i \frac{d}{dt} \int_b \vec{r}_p \times \vec{v}_p dm = \int_b \vec{r}_p \times \vec{f}_p \textcircled{2}$$

# Euler's 2nd law of motion II

$$\textcircled{1} \quad = \frac{d}{dt} \int_b (\vec{r} + \vec{r}_c) \times \vec{v}_p \, dm$$

$$= \frac{d}{dt} \int_b \vec{r} \times \vec{v}_p \, dm + \int_b \vec{r}_c \times \frac{d}{dt} \vec{p}_p + \underbrace{\int_b \frac{d}{dt} \vec{r}_c \times \vec{p}_p}_{\text{red bracket}}$$

$$= \frac{d}{dt} \int_b \vec{r} \times \vec{v}_p \, dm + \underbrace{\int_b \vec{r}_c \times \vec{f}_p}_{\text{blue circle}}$$

$$\begin{aligned} & \frac{d}{dt} \vec{r}_c \times \int_b \vec{v}_p \, dm \\ &= \frac{d}{dt} \vec{r}_c \times \vec{V}_c \cdot m \\ &= 0 \end{aligned}$$

$$\textcircled{2} \quad = \int_b (\vec{r} + \vec{r}_c) \times \vec{f}_p = \int_b \vec{r} \times \vec{f}_p + \underbrace{\int_b \vec{r}_c \times \vec{f}_p}_{\text{blue circle}}$$

# Euler's 2nd law of motion III

Therefore

$$\frac{d}{dt} \int_b \underbrace{\vec{r} \times \vec{v}_p}_{\vec{h}_{b/c}} dm = \int_b \underbrace{\vec{r} \times \vec{f}_p}_{\vec{T}_{b/c}} = \int_b \vec{r} \times \vec{f}_{p, \text{ext}}$$

$\vec{h}_{b/c}$ : angular momentum about CoM

$\vec{T}_{b/c}$ : torque about CoM

# Newton-Euler for center of mass

- That shows:
  - Newton's laws can be formulated for the center of mass

$$\frac{{}^i d}{dt} \vec{p}_c = \vec{F}_{bc} \quad (\text{linear moments})$$

$$\frac{{}^i d}{dt} \vec{h}_{b/c} = \vec{T}_{bc} \quad (\text{rotation})$$

# Angular momentum

$$\vec{h}_{b/c} = \int_b \vec{r} \times \vec{v}_p dm$$

$$\vec{v}_p = \vec{v}_c + \vec{\omega}_{ib} \times \vec{r}$$

$$= \int_b \vec{r} \times (\vec{v}_c + \vec{\omega}_{ib} \times \vec{r}) dm$$

$$= \underbrace{\int_b \vec{r} dm}_{=0} \times \vec{v}_c + \int_b \vec{r} \times (\vec{\omega}_{ib} \times \vec{r}) dm$$

$$= - \int_b \vec{r} \times (\vec{r} \times \vec{\omega}_{ib}) dm$$

$$= - \underbrace{\int_b \vec{r}^x \cdot \vec{r}^x dm}_{\vec{M}_{b/c}} \cdot \vec{\omega}_{ib}$$

$\vec{M}_{b/c}$  : inertia dyadic

$$= \vec{M}_{b/c} \cdot \vec{\omega}_{ib}$$

# Euler's 2nd law of motion about CoM

$$\begin{aligned}
 \vec{T}_{bc} &= {}^i \frac{d}{dt} \vec{h}_{b/c} = {}^i \frac{d}{dt} (\vec{M}_{b/c} \cdot \vec{w}_{ib}) \\
 &= {}^b \frac{d}{dt} (\vec{M}_{b/c} \cdot \vec{w}_{ib}) + \vec{w}_{ib} \times (\vec{M}_{b/c} \cdot \vec{w}_{ib}) \\
 &= \vec{M}_{b/c} \cdot \vec{a}_{ib} + \vec{w}_{ib} \times (\vec{M}_{b/c} \cdot \vec{w}_{ib}) \\
 \vec{a}_{ib} &= {}^b \frac{d}{dt} \vec{w}_{ib} = {}^i \frac{d}{dt} \vec{w}_{co} \\
 {}^b \frac{d}{dt} \vec{M}_{b/c} &= 0
 \end{aligned}$$

# EoM with reference of CoM

$$\vec{F}_{bc} = m\vec{a}_c$$

$$\vec{T}_{bc} = \vec{M}_{b/c} \cdot \vec{\alpha}_{ib} + \vec{\omega}_{ib} \times \left( \vec{M}_{b/c} \cdot \vec{\omega}_{ib} \right)$$

# Inertia dyadic I

$$\begin{aligned}\vec{M}_{b/c} &= - \int_b \vec{r}^{\times} \cdot \vec{r}^{\times} \, dm \\ &= \int_b [ \vec{r} \cdot \vec{r}, I - \vec{r} \vec{r} ] \, dm\end{aligned}$$

Evaluate in frame

$$\vec{M}_{b/c} = \sum_{i=1}^3 \sum_{j=1}^3 m_{ij}^b b_i b_j \quad m_{ij}^b: \text{constant}$$

$$\text{matrix form: } M_{b/c}^b = \begin{bmatrix} m_{11}^b & - & - \\ m_{21}^b & - & - \\ \cdot & \cdot & \cdot \end{bmatrix}$$



# Inertia dyadic II

$$\vec{M}_{b/c} = - \int_b \vec{r}^{\times} \cdot \vec{r}^{\times} dm$$

$$M_{b/c}^b = \int_b \left[ (\underline{r}^b)^T \cdot \underline{r}^b \mathbf{I} - \underline{r}^b (\underline{r}^b)^T \right] dm$$

$$= \int_b \begin{bmatrix} y^2 + z^2 & -xy & -xz \\ -xy & x^2 + z^2 & -yz \\ -xz & -yz & x^2 + y^2 \end{bmatrix} dm$$

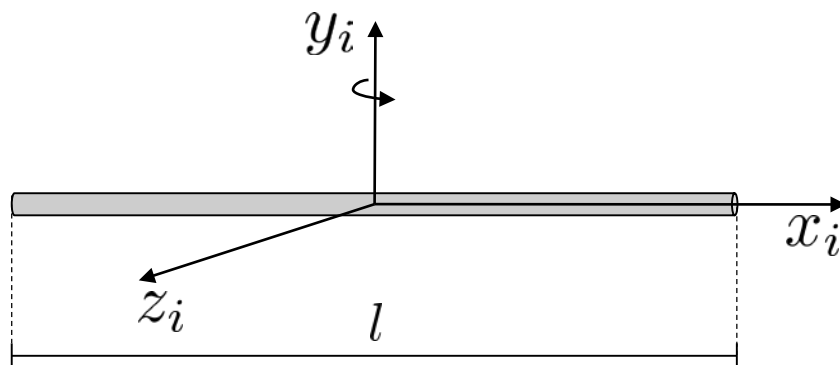
$$\underline{r}^b = \begin{bmatrix} x \\ y \\ z \end{bmatrix}$$



$M_{b/c}^b$  constant

$$M_{b/c}^i = R_b^i M_{b/c}^b R_i^b \quad \text{not constant}$$

# Example: Slender beam



# Homework

- Try to derive the moment of inertia for the slender beam (slide 34) using the information on slide 33.
- Derive the acceleration of the point  $p$  on a rigid body with the help of the body fixed frame and the CoM.
- Read 7.3

# Kahoot

- <https://play.kahoot.it/#/k/4152faff-75ee-49ea-bb9e-b4c79dd85785>