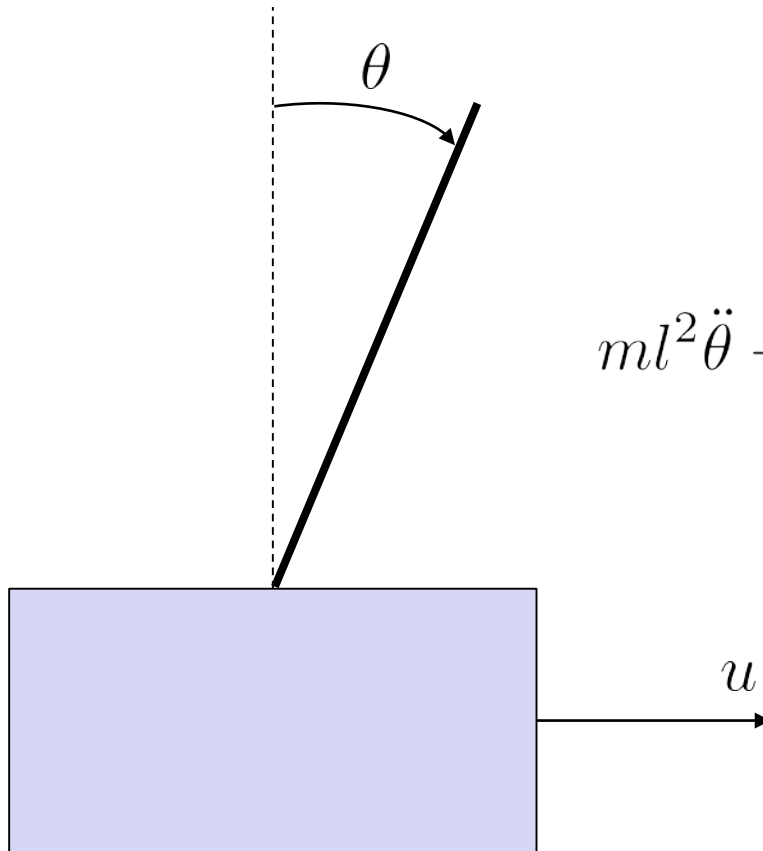


Lecture 2:

- Model types (E1.1-1.3,E2.1-2.2)
 - State space models, transfer functions
 - Linear models, nonlinear models

Example: “Stick balancing”



$$ml^2\ddot{\theta} - mg \sin \theta = -mlu \cos \theta$$

Example: "Stick balancing" [Example 4]

$$ml^2\ddot{\theta} - mg \sin \theta = -mlu \cos \theta$$

Linearise around $\theta = \dot{\theta} = 0 \quad u = 0$

$$\ddot{\theta} = -\frac{u}{l} \cos \theta + \frac{g}{l^2} \sin \theta$$

$$\Delta \ddot{\theta} = \frac{\partial}{\partial \theta} \left[-\frac{u}{l} \cos \theta + \frac{g}{l^2} \sin \theta \right]_{\substack{\theta=\dot{\theta}=0 \\ u=0}} \Delta \theta +$$

$$\frac{\partial}{\partial \dot{\theta}} \left[-\frac{u}{l} \cos \theta + \frac{g}{l^2} \sin \theta \right]_{\substack{\theta=\dot{\theta}=0 \\ u=0}} \Delta \dot{\theta} +$$

$$\frac{\partial}{\partial u} \left[-\frac{u}{l} \cos \theta + \frac{g}{l^2} \sin \theta \right]_{\substack{\theta=\dot{\theta}=0 \\ u=0}} \Delta u$$

$$= \left[\frac{u}{l} \sin \theta + \frac{g}{l^2} \cos \theta \right]_{\substack{\theta=\dot{\theta}=0 \\ u=0}} \Delta \theta + 0 \cdot \Delta \dot{\theta} +$$

$$\left[-\frac{1}{l} \cos \theta \right]_{\substack{\theta=\dot{\theta}=0 \\ u=0}} \Delta u$$

$$= \frac{g}{l^2} \Delta \theta - \frac{1}{l} \Delta u$$

1.3 Transfer functions

Linear time invariant model (LTI)

$$\dot{\underline{x}} = \mathbf{A}\underline{x} + \mathbf{B}\underline{u}$$

System matrix

Control matrix

$$\underline{y} = \mathbf{C}\underline{x} + \mathbf{D}\underline{u}$$

Output matrix

Feed-Forward matrix

Laplace notation

+ important analysis
and design methods

$$\underline{x}(s) = \mathcal{L}\{\underline{x}(t)\}$$

$$\underline{u}(s) = \mathcal{L}\{\underline{u}(t)\}$$

$$\underline{y}(s) = \mathcal{L}\{\underline{y}(t)\}$$

$$\mathcal{L}\{\dot{\underline{x}}(t)\} = s\mathcal{L}\{\underline{x}(t)\} - \underbrace{x(t=0)}$$

Assume = 0

Transform LTI system

$$\dot{\underline{x}} = \underline{A}\underline{x} + \underline{B}\underline{u} \quad \underline{y} = \underline{C}\underline{x} + \underline{D}\underline{u}$$

$$s \underline{x}(s) = \underline{A} \underline{x}(s) + \underline{B} \underline{u}(s) \quad (1)$$

$$\underline{y}(s) = \underline{C} \underline{x}(s) + \underline{D} \underline{u}(s) \quad (2)$$

$$(s\mathbf{I} - \underline{A}) \underline{x}(s) = \underline{B} \underline{u}(s)$$

$$\underline{x}(s) = [s\mathbf{I} - \underline{A}]^{-1} \underline{B} \underline{u}(s)$$

$$\underline{y}(s) = \underbrace{(\underline{C} [s\mathbf{I} - \underline{A}]^{-1} \underline{B} + \underline{D})}_{H(s)} \underline{u}(s)$$

$H(s)$: transfer function

Rational transfer function

$$\frac{y(s)}{u(s)} = H(s)$$

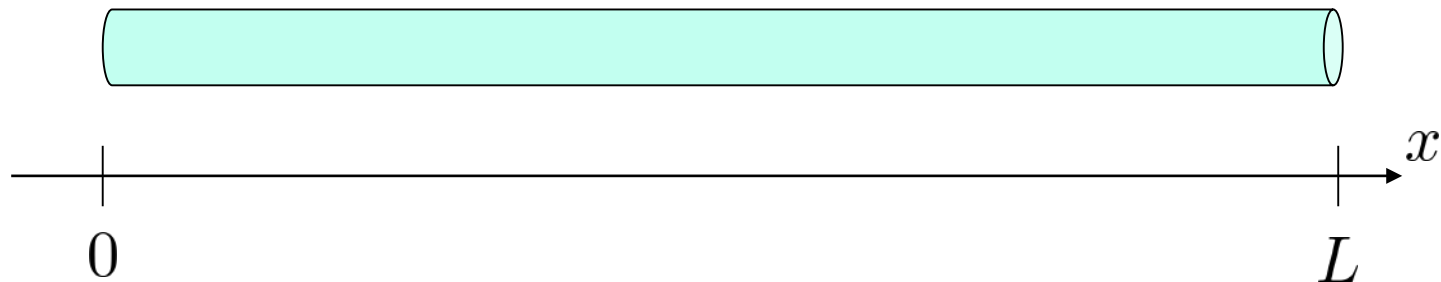
- Rational transfer function if it can be expressed as:

$$\begin{aligned} H(s) &= K \frac{P(s)}{Q(s)} \\ &= K \frac{(s + z_1) \dots (s + z_m)}{(s + p_1) \dots (s + p_n)} \end{aligned}$$

- m: zeros ; n: poles

Partial differential equations

- Partial differential equations (pde) lead to irrational transfer functions
- They can be approximated by rational transfer functions with infinitely order
→ infinite dimension system
- Example: Transport equation/advection /wave equation



PDE – Example I



PDE:

$$\frac{\partial v(x,t)}{\partial t} = -C \frac{\partial v(x,t)}{\partial x} \quad v(0,t) = v_1(t)$$

$$\mathcal{L} \left\{ \frac{\partial v(x,t)}{\partial t} \right\} = s \mathcal{L} \{ v(x,t) \} = s v(x,s)$$

$$s v(x,s) = -C \frac{\partial v(x,s)}{\partial x}$$

→ ODE of x

$$- \frac{s}{C} \partial x = \frac{\partial v(x,s)}{v(x,s)}$$

$$\int_0^x - \frac{s}{C} \partial x = \int_{v(0,s)}^{v(x,s)} \frac{\partial v(x,s)}{v(x,s)}$$

PDE – Example II



$$-\frac{s}{c} x = \ln \left(\frac{v(x,s)}{v(0,s)} \right)$$

$$\exp\left(-\frac{s}{c} x\right) = \frac{v(x,s)}{v(0,s)}$$

$$v(x,s) = v(0,s) \exp\left(-\frac{s}{c} x\right)$$

boundary
conditions:

$$v(L,s) = v_2(s)$$

$$v_2(s) = v_1(s) \exp\left(-\frac{s}{c} L\right)$$

$$\frac{v_2(s)}{v_1(s)} = \underbrace{\exp(-Ts)}$$

irrational
transfer function

$$T = \frac{L}{c}$$

propagation
time

Lecture 3: Energy functions and passivity

Using "energy" as a concept for characterizing system behavior

- Energy functions (aka Lyapunov functions)
 - If the "internal energy" of a system decreases, the system is stable
 - "Introvert" (not concerned with surroundings)
- Passivity
 - Does a system produce "energy" to its surroundings?
 - "Extrovert" (mainly concerned with surroundings, via inputs and outputs)
- The above concepts are connected via storage functions (next time)

Book: E2.3, E2.4

Energy function

- The system: $\dot{\underline{x}} = \underline{f}(\underline{x}, \underline{u}, t)$
- Assume we have a function $V(x, t) \geq 0$, which describes the «energy» of the system

- The derivative of the energy function $V(x, t)$ is

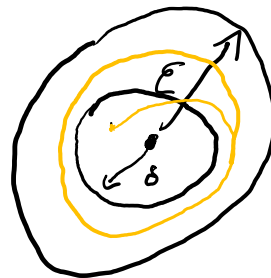
$$\dot{V} = \frac{\partial V}{\partial t} + \frac{\partial V}{\partial x} \frac{dx}{dt} = \frac{\partial V}{\partial t} + \frac{\partial V}{\partial x} f(x, u, t)$$

- If we have $\dot{V} \leq 0$
 - Energy of the system decreases monotonically
 - stability

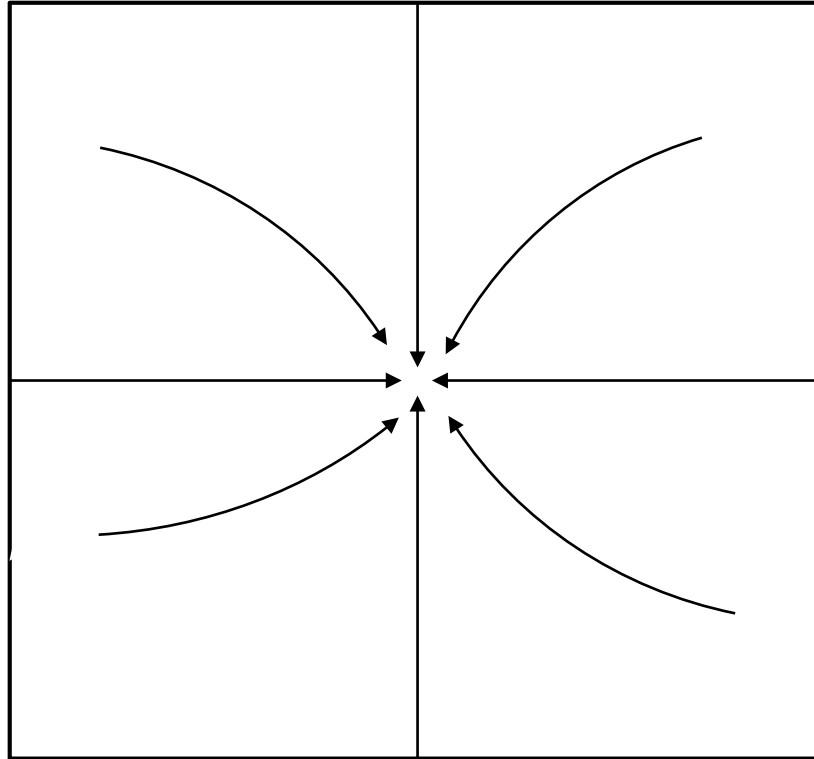
Stable system

- Equilibrium point is stable if for any possible $\varepsilon > 0$ radius around the steady state point a region with the radius δ exist, such that for all initial values $|x_0 - x_e| < \delta$ the solution $x(t)$ fullfils for all $t > t_0$ the following condition:

$$|x(t) - x_e| < \varepsilon$$

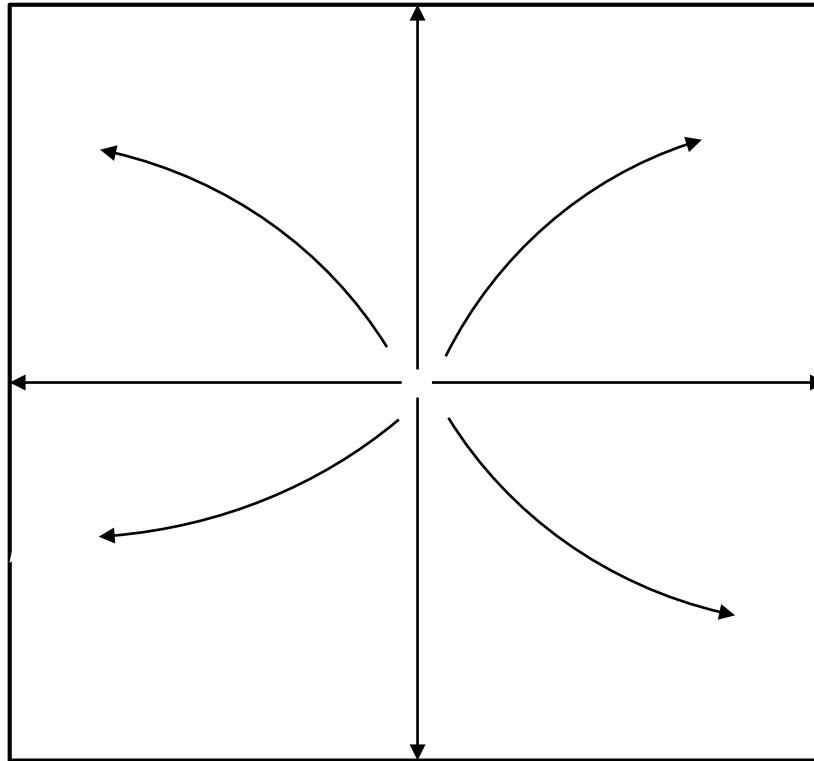


Phase diagram for system with real Eigenvalues



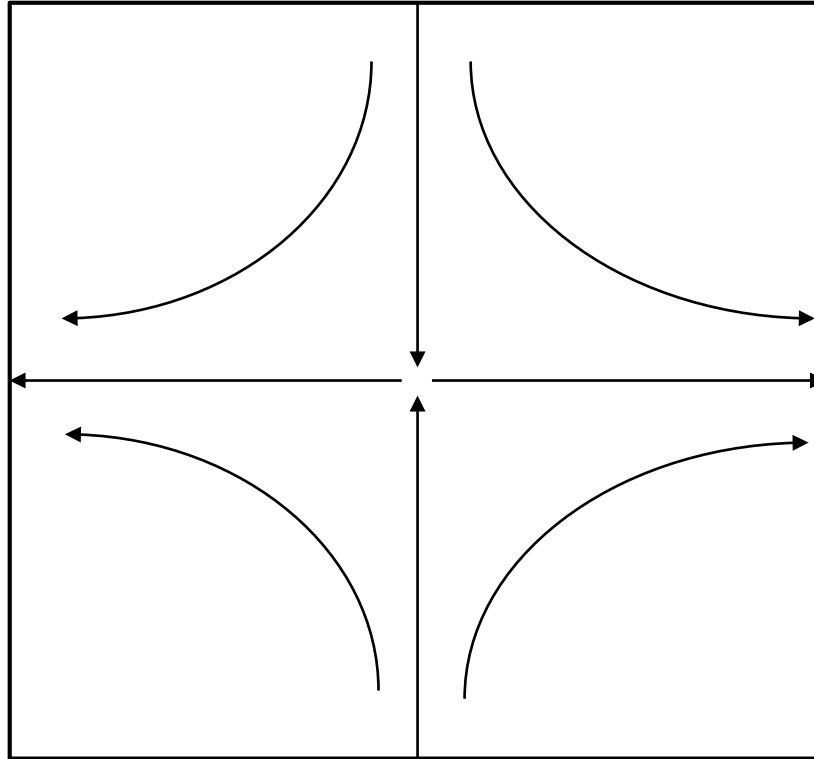
Stable

Phase diagram for system with real Eigenvalues



Unstable

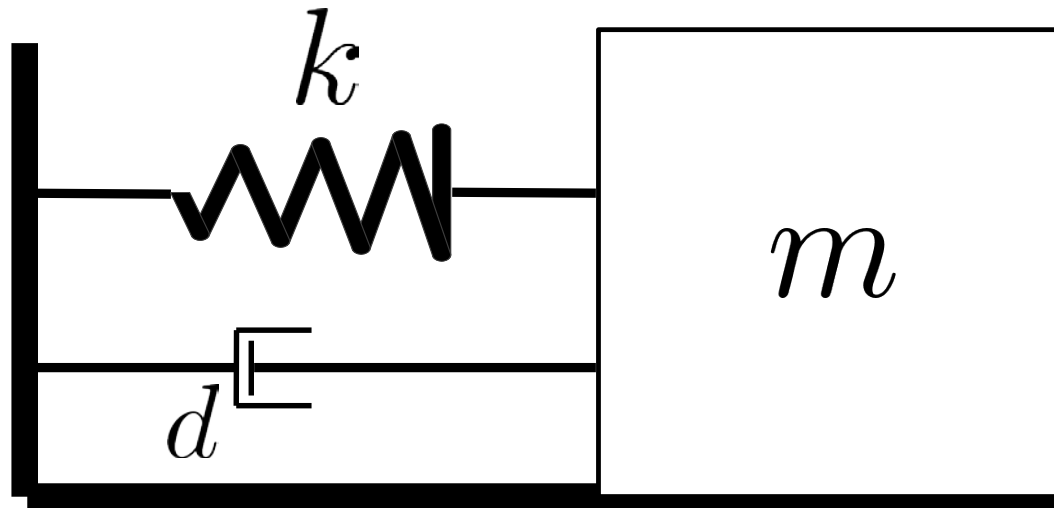
Phase diagram for system with real Eigenvalues



Saddle \rightarrow unstable

Mass-spring-damper I (2.3.4)

$$m\ddot{x} + d\dot{x} + kx = 0$$



Mass-spring-damper II $m\ddot{x} + d\dot{x} + kx = 0$

$$\left. \begin{aligned} x_1 &= x \\ x_2 &= \dot{x} \end{aligned} \right\}$$

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = -\frac{k}{m}x_1 - \frac{d}{m}x_2$$

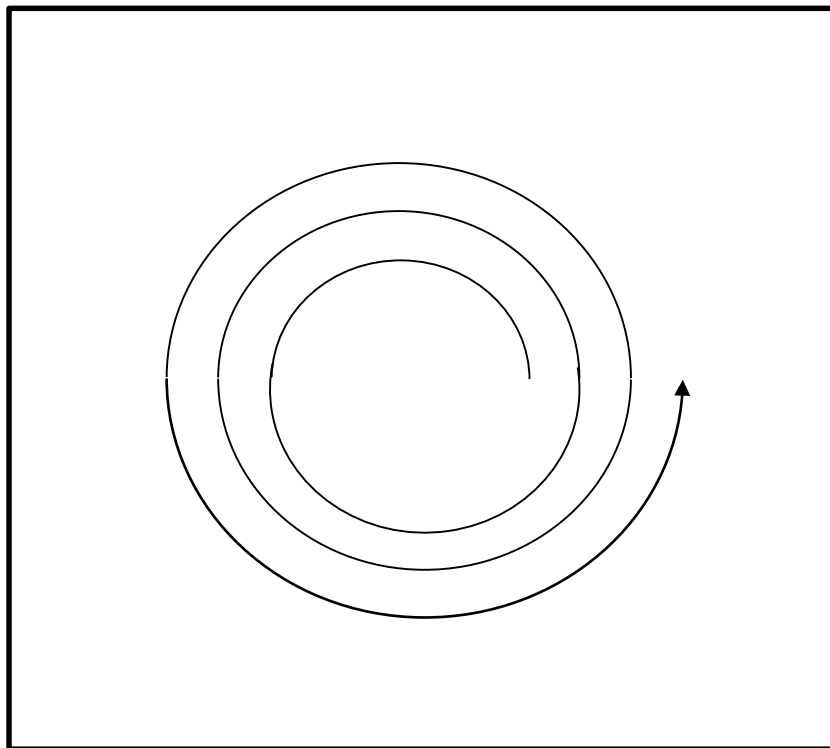
$$V = \underbrace{\frac{1}{2} m x_2^2}_{\text{Kin. energy}} + \underbrace{\frac{1}{2} k x_1^2}_{\text{pot. energy}} \geq 0$$

Mass-spring-damper III $m\ddot{x} + d\dot{x} + kx = 0$

$$\begin{aligned}
 \dot{V} &= m x_2 \dot{x}_2 + k x_1 \dot{x}_1 \\
 &= m x_2 \left(-\frac{k}{m} x_1 - \frac{d}{m} x_2 \right) + k x_1 x_2 \\
 &= -d x_2^2 \leq 0 \\
 &\rightarrow V(t) \leq V(t_0) = V_0 \\
 &\rightarrow \text{stable}
 \end{aligned}$$

Mass-spring-damper

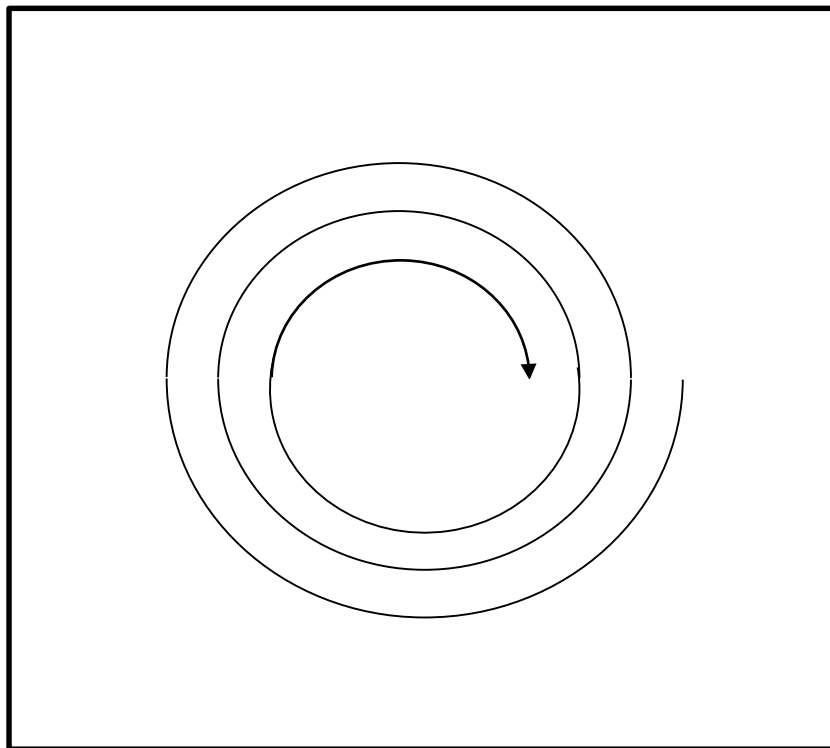
$$\lambda_{1,2} = u \pm iv$$



unstable $u > 0$

Mass-spring-damper

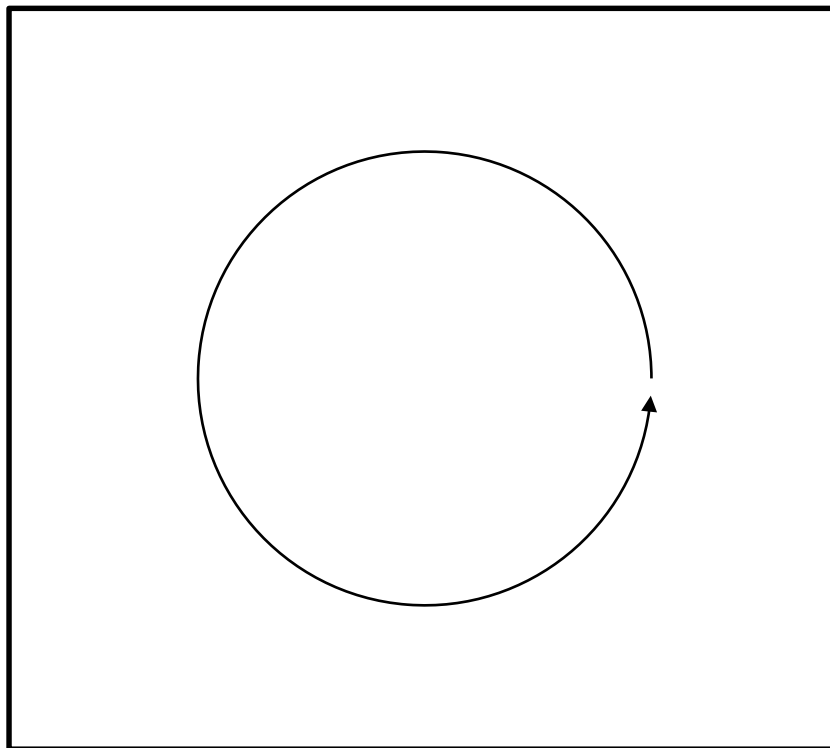
$$\lambda_{1,2} = u \pm iv$$



stable $u < 0$

Mass-spring-damper

$$\lambda_{1,2} = u \pm iv$$

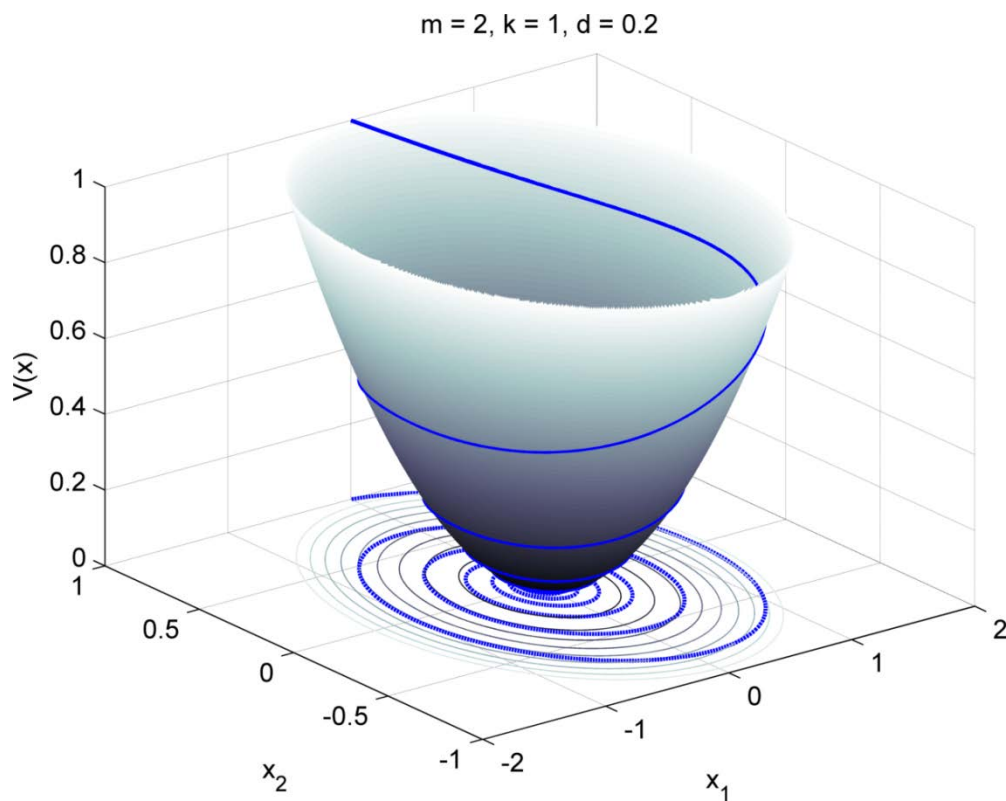


centre $u = 0$

Mass-spring-damper

$$m\ddot{x} + d\dot{x} + kx = 0$$

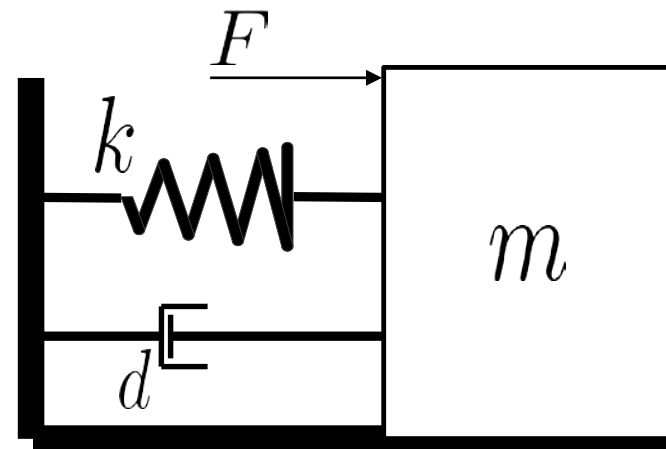
$$V(x) = \frac{m}{2}\dot{x}^2 + \frac{k}{2}x^2$$



Mass-spring-damper with force

$$m\ddot{x} + d\dot{x} + kx = F$$

$$V = \frac{1}{2} k x_1^2 + \frac{1}{2} m x_2^2$$



$$\left. \begin{array}{l} x_1 = x \\ x_2 = \dot{x} \end{array} \right\} \quad \begin{array}{l} \dot{x}_1 = x_2 \\ \dot{x}_2 = \frac{F}{m} - \frac{k}{m}x_1 - \frac{d}{m}x_2 \end{array}$$

$$\dot{V} = -d x_2^2 + F x_2$$

If F supplied by a controller: $F = -k_d \cdot x_2$
 $\rightarrow \dot{V} = -(k_d + d) x_2^2 \leq 0 \rightarrow \text{stable}$

General: Energy-based controller design

$$\left. \begin{aligned} \dot{x} &= f(x, u, t) \\ \dot{V} &= \frac{\partial V}{\partial t} + \frac{\partial V}{\partial x} f(x, u, t) \end{aligned} \right\} \begin{array}{l} \text{Choose a} \\ \text{controller that} \\ \dot{V} \leq 0 \end{array}$$

Why learn about passivity? Preview...

- Say you have several systems (or models), and you want to interconnect them
 - For instance, a process and a controller, or a motor and a load, or two buffer tanks in series, ...
 - Will the interconnection be stable?
- Bad news: The interconnection of stable systems is not necessarily stable
- **Good news: The interconnection of passive systems is passive (and therefore stable)!**

Homework (recommended)

- Derive the derivative of the energy function of the mass-spring-damper system with force

- $\dot{V} = -dx_2^2 + Fx_2$

- Read section 2.4.1, 2.4.2, 2.4.3 in the book

- Try to proof passivity of the transfer-function:

$$H(s) = \frac{1}{1 + Ts}$$

by first transferring the function to the time domain