

Formulas

1) Fundamentals

Static load on each axle:

$$W_{fs} = W \frac{c}{L}$$

$$W_{rs} = W \frac{b}{L}$$

Assuming not static:

$$W_f = \frac{1}{L} \left[W \cos(\theta) \cdot c - W \sin(\theta) \cdot h - D_A h_a - \frac{W}{g} a_x h \right]$$

$$W_r = \frac{1}{L} \left[W \cos(\theta) \cdot b + W \sin(\theta) \cdot h + D_A h_a + \frac{W}{g} a_x h \right]$$

2) Longitudinal Dynamics

Upper limit:

$$a_x = \frac{F_x}{M} = \frac{gP}{VW}$$

$$1 \text{ kW} = 0,746 \text{ HP}$$

Power-limited acceleration

Engine: $T_c = T_E - I_E a_E$

Transmission: $T_D = (T_c - I_T a_E) \eta_T$

Differential: $T_A = (T_D - I_D a_D) \eta_F = F_x r + I_w a_w$

Kinematic relationships:

$$a_x = \alpha_w r \quad \alpha_D = \alpha_w n_F \quad \alpha_E = \alpha_D n_T = \alpha_w n_F n_T$$

Rotational equilibrium of powertrain:

$$F_x = \frac{T_E n_{TF} n_{TF}}{r} - \left[(I_E + I_T) n_{TF}^2 + I_D n_F^2 + I_w \right] \frac{a_x}{r^2}$$

Translational equilibrium of vehicle:

$$M a_x = F_x - R_x - D_A - R_{hx} - W \sin \theta$$

Simplified power-limited acceleration:

$$(M + M_r) a_x = \frac{T_E n_{TF} n_{TF}}{r} - R_x - D_A - R_{hx} - W \sin \theta$$

Mass factor:

$$\frac{M + M_r}{M} = 1 + 0.04 + 0.0025 n_{TF}^2$$

Traction-limited Acceleration

$$F_x = \mu W, \quad \mu \text{ friction coefficient}$$

Transverse weight shift due to drive torque:

Axis in equilibrium:

$$\sum T = \left(\frac{W_r}{2} + W_y \right) \frac{t}{2} - \left(\frac{W_r}{2} - W_y \right) \frac{t}{2} + T_s - T_D = 0$$

$$\Rightarrow W_y = \frac{T_D - T_s}{t}, \quad T_D \text{ torque from driveshaft on axle}$$

T_s torque due to suspension roll stiffness

$$T_D = F_x \frac{r}{\eta_F}$$

Roll torque:

$$T_{sf} = K_{\phi f} \phi$$

, $K_{\phi f}$: front susp. roll stiffness

$$T_{sr} = K_{\phi r} \phi$$

, $K_{\phi r}$: rear susp. roll stiffness

$$K_{\phi} = K_{\phi f} + K_{\phi r}$$

$$\phi = \frac{T_D}{K_{\phi}} = \frac{T_D}{K_{\phi f} + K_{\phi r}}$$

$$T_{sr} = K_{\phi r} \frac{T_D}{K_{\phi f} + K_{\phi r}}$$

$$W_y = \frac{F_x r}{\eta_F t} \left[1 - \frac{K_{\phi r}}{K_{\phi f} + K_{\phi r}} \right] = \frac{F_x r}{\eta_F t} \frac{K_{\phi f}}{K_{\phi}}$$

Net load on rear axle during acceleration:

$$W_r = W \left(\frac{b}{L} + \frac{a_x}{g} \frac{h}{L} \right)$$

Neglecting rolling resistance and drag

$$W_r = W \left(\frac{b}{L} + \frac{F_x}{mg} \frac{h}{L} \right)$$

Maximum tractive force;

Solid rear axle with non-locking diff.:

$$F_{x, \max} = \frac{\mu \frac{W_b}{L}}{1 - \frac{h}{L} \mu + \frac{2\mu r}{n_F t} \frac{k_{\phi f}}{k_{\phi}}}$$

Solid rear axle with locking diff. / independent rear susp.:

$$F_{x, \max} = \frac{\mu \frac{W_b}{L}}{1 - \frac{h}{L} \mu}$$

Rolling Resistance

$$R_x = f_r W, \quad f_r \text{ rolling resistance coefficient}$$

Aerodynamic drag

$$D_A = \frac{1}{2} \rho C_D A V^2$$

Road grade

$$R_G = W \sin \theta$$

Braking Force

$$F_b = \frac{T_b - F_w a_w}{r}$$

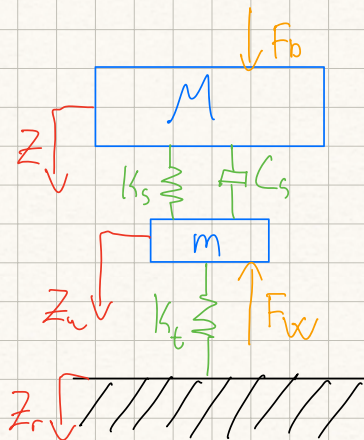
Simplified: $F_b = \frac{T_b}{r}$

Braking coefficient:

$$\mu = \frac{F_x}{F_z}$$

Slip: $Slip = \frac{V - \omega r}{V}$

4) Ride



Ride rate: effective stiffness

$$RR = \frac{k_s k_t}{k_s + k_t}$$

Undamped frequency:

$$\omega_n = \sqrt{\frac{RR}{m}}$$

Damped frequency:

$$\omega_d = \omega_n \sqrt{1 - \xi_s^2}$$

ξ_s damping ratio: 0,2 - 0,4

$$M \ddot{z} + C_s \dot{z} + k_s z = C_s \dot{z}_u + k_s z_u + F_b$$

$$m\ddot{z}_u + c_s \dot{z}_u + (k_s + k_t) z_u = c_s \dot{z} + k_s z + k_t z_r + F_w$$

Analytical frequencies:

$$(if \ k_t \gg k_s \rightarrow \omega_1 = \sqrt{\frac{k_s}{m}}, \quad \omega_2 = \sqrt{\frac{k_t}{m}}$$

Skyhook Control

$$c_s = \begin{cases} c_{max} & , \quad \dot{z}_s(\dot{z}_s - \dot{z}_u) > 0 \\ c_{min} & , \quad \dot{z}_s(\dot{z}_s - \dot{z}_u) \leq 0 \end{cases}$$

5) Lateral Dynamics

low speed cornering

Steer angles (assuming small angles):

$$\delta_o = \frac{L}{R + \frac{t}{2}}$$

$$\delta_i = \frac{L}{R - \frac{t}{2}}$$

Average angle, Ackerman angle:

$$\delta = 57.3 \frac{L}{R}$$

High speed cornering

Cornering force:

$$F_y = \alpha C_\alpha \quad , \quad C_\alpha \text{ cornering stiffness}$$

Sum of lateral forces:

$$\sum F_y = F_{yf} + F_{yr} = \frac{MV^2}{R}$$

To have moment equilibrium:

$$F_{yf} \cdot b - F_{yr} \cdot c = 0 \quad \Rightarrow \quad F_{yf} = F_{yr} \frac{c}{b} \Leftrightarrow F_{yr} = F_{yf} \frac{b}{c}$$

This gives us:

$$F_{yr} = \frac{Mb}{L} \frac{V^2}{R} = \frac{W_r}{g} \frac{V^2}{R}$$

$$F_{yf} = \frac{Mc}{L} \frac{V^2}{R} = \frac{W_f}{g} \frac{V^2}{R}$$

Need to find slip angle:

$$F_y = C_a \alpha \Rightarrow \alpha = \frac{F_y}{C_a}$$

$$\alpha_f = \frac{W_f V^2}{C_{af} g R}$$

$$\alpha_r = \frac{W_r V^2}{C_{ar} g R}$$

Steering angle:

$$\delta = 57.3 \frac{L}{R} + \alpha_f - \alpha_r$$

$$= 57.3 \frac{L}{R} + \underbrace{\left(\frac{1}{g} \left(\frac{W_f}{C_{af}} - \frac{W_r}{C_{ar}} \right) \right)}_{\rightarrow k} \underbrace{\left(\frac{V^2}{R} \right)}_{\rightarrow a_y}$$

$$\delta = 57.3 \frac{L}{R} + k a_y$$

Characteristic speed:

$$V_{\text{char}} = \sqrt{57.3 \frac{L g}{k}}$$

Critical speed:

$$V_{\text{crit}} = \sqrt{-57.3 \frac{L g}{k}}$$

Sideslip angle:

$$\begin{aligned} \beta &= 57.3 \frac{c}{R} - \alpha_r \\ &= 57.3 \frac{c}{R} - \frac{W_r V^2}{C_{ar} g R} \end{aligned}$$

$$V_{\beta=0} = \sqrt{57.3 \frac{C_{ar} g C_{af}}{W_r}}$$

Load transfer during cornering

Equilibrium in vertical direction:

$$F_{in} + F_{out} = W$$

Equilibrium to rotation around roll center

$$F_y(h_{gs} - h_r) = \underbrace{F_{out} \frac{s}{2} - F_{in} \frac{s}{2}}_{M_\phi}$$

M_ϕ roll torque

$$F_{in} = \frac{W}{2} - \frac{F_y(h_{gs} - h_r)}{s}$$

$$F_{out} = \frac{W}{2} + \frac{F_y(h_{gs} - h_r)}{s}$$

$$\frac{\Delta F_z}{2} = \frac{F_y(h_{cg} - h_r)}{s}$$

Roll angle sprung mass (for small angles):

$$\phi = \frac{x}{s} \Rightarrow x = \phi s$$

Torque generated by suspension springs around roll center

$$M_\phi = \frac{s}{2} (F_{out} - F_{in}) = \frac{s}{2} k_s x = \frac{1}{2} s^2 k_s \phi = K_\phi \phi$$

Rotational equilibrium around ground point:

$$F_{z0} \frac{t}{2} - F_{zi} \frac{t}{2} = F_y h_r + K_\phi \phi$$

$$\Rightarrow F_{z0} - F_{zi} = \underbrace{\frac{2 F_y h_r}{t}}_1 + \underbrace{\frac{2 K_\phi \phi}{t}}_2 = 2 \Delta F_z$$

1
load transfer
lateral force

2
load transfer
vehicle roll

6) Tires

Characteristics:

P225/60R16 93Z

1 2 3 4 5 6 7

1: Application (Passenger, Truck, ...)

2: Section width [mm]

3: Aspect ratio $\frac{\text{height}}{\text{width}}$

4: Construction (Radial, Bias)

5: Rim diameter [in] 1 in = 2,54 cm

6: Load index

7: Speed rating