Ride

Excitation sources

- Road profile Variations in the road sur face such as lamps, potholes, and rough patches, Directly introduces vertical displacement to the tires and suspension. Higher frequency bumps can cause Vibrations felt in the calain. hargo obstacles or unever tenain can local to suspension compression and body motion Susponsion design aims to isolate the cabin from road disturbances Use of dampors and springs tened to absorb road irregularities - Tires/wheels Irregularities in fire shape uneven fire wear, or imbalance in the wheel assembly Generates cyclic vertical forces as the tire notates Causes vibrations felt at higher speeds, often percieved as reheel wobble or shaking. Proper tire buluncing Use of uniform fire materials and precision manufacturing - Drive shaft Rotational imbulances or unisalignment in the driveshoft or transmission

companents	
Introduces periodic vibrations !	alked to the drive shaft rotation
speed	
Can lead to noticeable oscillati	ons in the cubin, especially under
acceleration or deceleration	
Precision balancing of rotating	- components
Proper alignment of drive ghaft	
- Engine	
Vibrations created by recipron	cating motion of pistons,
votating crankshafts, and ofher	
Introduces high-freq vibration	s ento the chassis
Engine mounts designed to isola	
Balancing shafts and harmonic	
Quarter-car model	
Dynamic model	Ride vate: effective stiffness
M Sprung Muss	$RR = \frac{ \zeta_s \zeta_t}{ \zeta_s + \zeta_t }$
2	Undamped frequency
M Unsprung muss	$\omega_{n} = \sqrt{\frac{RR}{M}}$
Zu K. S. Fix. Tire	
Road	Damped frequency
t-1//////	$\omega_{d} = \omega_{n} \sqrt{1 - \frac{62}{5}}$
	75

(: damping valio: 0.2-0.4 MZ+(, Z+ K, Z= C, Zu+K, Zu+F) mzu + (zzu+(14+14,) Zu = Cz + 142+142+ Fx Analytical frequencies (f K_t) K₃ -7 K₄ + K_t & K_t - K₅ & K₆ - 7 6 = 1 m₅ , 6 2 = 1 m₆ Suspension types Passive Semi-active Active Semi-active suspension - Electromagnetic coils located in the piston to generate a localized magnetic field around the piston's passages - The hydraulic fluid inside the dampers contains tyny iron particles, distributed randomly before electric current is applied to the piston coils - Applying current to the coils create a magnetic field, which arranges the

particles into lines, making the fluid more resistant to flow
Active Suspension
Control the suspension with actuators
Skyhook control for semi-active suspension
magine vehicle body is suspended not from the
whods, but from an imaginary fixed point in cosky I Zs
The sky. This point doesn't move, no the sody Ks Zu Zu
Oscillations, independent of road distorbances.
Since this connection to the sky is impossible,
the semi-active damps is used to minic this behavior as closely
as possible.
The damping force applied by the semi-active damper is adjusted based on the vertical velocity of the vehicle body relative to the
wheel.
$C_{5} = \begin{cases} C_{mox}, & Z_{5}(\hat{z}_{5} - \hat{z}_{u}) \\ C_{5} = C_{5}(\hat{z}_{5} - \hat{z}_{u}) \\ C_{6} = C_{5}(\hat{z}_{5} - \hat{z}_{u}) \\ C_{7} = C_{7}(\hat{z}_{5} - \hat{z}_{u}$
$Z_s(\dot{z}_5 - \dot{z}_a) \langle 0 \rangle$

Lets say: body moving upwards: 2,40,2,70 When do we want high damping? $\frac{2}{3}(2-2)70 = 7 = \frac{2}{3} - \frac{2}{3} = \frac{2$ When unspring mass is also moving operads with the same or higher speed (negative direction) we want high dumping What about low damping?

2/3(2/3-2/a) 40 => 2/3 - 2/3 Zu 20 When unsprung mass is moving either downwards or unwards but with lower speal than the spring mass, we want low damping, Downward motion: 2,70 When do we want high damping? $\frac{1}{2}(\frac{2}{2}, -\frac{1}{2}, \frac{1}{2}) = \frac{1}{2}(\frac{2}{2}, -\frac{1}{2}, \frac{1}{2}, \frac{1}{2}) = \frac{1}{2}(\frac{2}{2}, -\frac{1}{2}, \frac{1}{2}, \frac{1}{2}) = \frac{1}{2}(\frac{1}{2}, \frac{1}{2})$ When unsprung mass is moving upwards or downwards but with slower speed, we want high damping

