

Control System Design Solutions

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Chapter 2 Answers

Answer (2.5a). For each cart, we know that acceleration is equal to the net Force divided by the mass. Thus,

$$\ddot{z}_i = \frac{F_N}{M}.$$

We let the positive direction be to the right. Thus,

$$\begin{aligned}\ddot{z}_1 &= \frac{1}{M} (F_{m1} - F_s) \\ \ddot{z}_2 &= \frac{1}{M} (F_{m2} + F_s),\end{aligned}$$

where F_{mi} are the forces from each respective motor, and F_s is the force from the intermediate spring. Now lets derive the equations for the force F_{mi} . This will follow from Example 2B of the textbook.

$$\begin{aligned}e_i - v_i &= Ri_i \\ e_i - K_2 w_i &= Ri \\ \tau_i &= K_1 i_i \\ e_i - K_2 w_i &= \frac{R\tau_i}{K_1} \\ \tau_i &= \frac{K_1}{R} e_i - \frac{K_1 K_2}{R} w_i \\ F_{mi} &= \frac{\tau_i}{r} = \frac{K_1}{rR} e_i - \frac{K_1 K_2}{rR} w_i \\ w_i r &= \dot{z}_i \\ F_{mi} &= \frac{K_1}{rR} e_i - \frac{K_1 K_2}{r^2 R} \dot{z}_i\end{aligned}$$

Now we will use Hooke's law to derive the force of the spring,

$$F_s = kd = k(z_2 - z_1).$$

Thus, for our differential equations, we have

$$\begin{aligned}\ddot{z}_1 &= \frac{k}{M} z_1 - \frac{k}{M} z_2 - \frac{K_1 K_2}{r^2 RM} \dot{z}_1 + \frac{K_1}{rRM} e_1 \\ \ddot{z}_2 &= -\frac{k}{M} z_1 + \frac{k}{M} z_2 - \frac{K_1 K_2}{r^2 RM} \dot{z}_2 + \frac{K_1}{rRM} e_2.\end{aligned}$$

Finally for our state-space representation of the system, we have,

$$\begin{bmatrix} \dot{z}_1 \\ \dot{z}_2 \\ \ddot{z}_1 \\ \ddot{z}_2 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ \frac{k}{M} & -\frac{k}{M} & -\frac{K_1 K_2}{r^2 RM} & 0 \\ -\frac{k}{M} & \frac{k}{M} & 0 & -\frac{K_1 K_2}{r^2 RM} \end{bmatrix} \begin{bmatrix} z_1 \\ z_2 \\ \dot{z}_1 \\ \dot{z}_2 \end{bmatrix} + \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ \frac{K_1}{rRM} & 0 \\ 0 & \frac{K_1}{rRM} \end{bmatrix} \begin{bmatrix} e_1 \\ e_2 \end{bmatrix}.$$

Answer (2.5b). Using the differential equations above, we can create the following block diagram.

