

# Continuous Measurements and Feedback Control of a Quantum Harmonic Oscillator

Max Eriksson

Bachelor Thesis, 15 ECTS, Spring 2025

Supervised by Kalle Kansanen and Peter Samuelsson  
Division of Mathematical Physics | Department of Physics





Frontcover graphic generated using *ChatGPT-4o*, OpenAI, by using the prompt: “Can you create an image in standing A4 format. The image should imagine a quantum harmonic oscillator. I want the main two colors used to be R0 G0 B128 and R156 G97 B20. The style should be watercolor and not too complex.” Image generated January 2025.

# Acknowledgements

First and foremost I would like to thank my supervisors, Peter and Kalle, for guiding me through this project, and providing me with invaluable understanding of the field. I want to give a special thanks to Kalle for always being available when I had any questions, and to all the discussions during those impromptu hour long meetings. I would also like to thank the rest of the division of mathematical physics, especially Peter's group, for creating a welcoming working environment and great fredagsfika.

Furthermore, I want to thank everyone who helped relieve stress during this period. Thank you, Hugo, Lukas, Fredrik, Elis, Kaspian, Kornelius, Josef, Amanda, Alma, Liam, Edvin, Herman, and all others. To all those late Wednesday nights at Pub Rydberg and the all too short weekends. To all the late night shenanigans. To Valborg and Skanör. Thank you to Tequilafredag and Utopia-eftersläppet. My gratitude goes out to Systembolaget for bringing joy to the weekends.

A special thanks is also deserved to the coffee machine at MatPhys for always providing me with much-needed caffeine. Thank you to the chair and the floor for giving me something to sit and rest on. Tack du vackra Vår för solljus och energi. Thanks to all the food I've eaten that has given me sustenance. Thanks to L<sup>A</sup>T<sub>E</sub>X for making the report pleasing to the eyes.

I want to thank all the teachers that motivated my interest for science and knowledge. For opening my eyes to the wonderful world of physics. On that note I also want to give a special thanks to Henrik, whom without I would not be here.

Last, but certainly not least, I want to thank my family for being the reason that I am where I am today. Thank you for supporting my education and always believing in me, even when I didn't. Mamma och Pappa, tack för allt.

# Abstract

Quantum mechanical technologies often relies on the system being cooled to low temperatures. One way to achieve this is with measurement and feedback based cooling. In this thesis measurement and feedback on a quantum harmonic oscillator coupled to a thermal reservoir is considered. The Wiseman-Milburn equation is used with a linear feedback control to find the steady-state solutions for the first and second moments of the system. The solutions are performed analytically. The results show that there is a feedback parameter such that the system is cooled below the temperature of the thermal reservoir, more generally the magnitude and phase of the feedback parameter can control the temperature in both directions. It is also shown that the system is stable for such a parameter. There also exist feedback parameters which can cool the system, but is unstable, as well as feedback parameters which break one or more of the approximations and assumptions made about the system. Furthermore, it looks promising using feedback as a way to stabilize an unstable system, since the stability of the system is heavily dependent on the feedback parameter.

# Populärvetenskaplig sammanfattning

Kvantmekaniken har sedan början av 1900-talet revolutionerat vår förståelse av fysik, framförallt hur ljus och partiklar fungerar och interagerar. En unik egenskap av kvantmekaniken är att den i grunden är en probabilistisk teori, vilket låter oss beskriva system som befinner sig i flera olika tillstånd samtidigt. Detta fenomenet kallas för superposition och är vad som gör kvantmekanik unikt från klassisk fysik.

En konsekvens av att kvantmekaniska system kan finnas i superposition är att mätningar av systemet ger olika resultat fördelade enligt en sannolikhetsfördelning. Detta öppnar också upp för frågan vad en kvantmekanisk mätning är för något. I grund och botten kan vi säga att en mätning är en interaktion mellan vårt system och ett yttre system, en detektor. Det är denna interaktion som gör att systemet kollapsar från en superposition till det observerade tillståndet. En mätning, eller interaktion, påverkar alltså vårt kvantmekaniska system. I detta arbete undersöker vi hur man kan använda mätningar och återkoppling för att kyla ett kvantmekaniskt system.

Det system som vi studerar är ett av de simplesta kvantmekaniska systemen, en kvantharmonisk oscillator. Det är ett av få system som kan lösas analytiskt, vilket är en av anledningarna till att vi väljer att studera det. En annan fördel med en kvantharmonisk oscillator är att det är en bra approximation av många system som befinner sig nära sitt jämviktsläge. I modellen som används betraktar vi även ett värmebad som systemet är kopplat till, vilket är ett sätt att modellera en omgivning som påverkar systemet och ger oss en mer realistisk bild.

Vi undersöker sedan hur svaga mätningar påverkar systemet. Med en svag mätning menar vi att istället för att kollapsa systemet helt till ett tillstånd och förstöra superpositionen så får vi ut en begränsad mängd information om systemet, samtidigt som systemet fortfarande är i superposition. Vi matar sedan in denna information i ett återkopplingssystem, som i sin tur påverkar systemet. Genom att justera styrkan och fasen av vår återkopplingsparameter kan vi kyla systemet genom att minimera dessa fluktuationer. Det intressanta resultatet visar att det är möjligt att kyla systemet till en temperatur som är lägre än den i värmebadet. Det vill säga att systemet blir kallare än sin omgivning. Vi visar även i arbetet att systemet är stabilt under återkopplingen som kan kyla systemet. Detta är viktigt för att kunna vara praktiskt tillämpbart då ett instabilt system hade varit väldigt känsligt för störningar, något som alltid finns i verkligheten. Det öppnar också upp för möjligheten att istället för att se kylning som det primära funktionen för återkopplingen kan vi istället tänka oss en återkoppling som försöker stabilisera ett annars instabilt system.

En anledning att vi vill kunna kyla kvantystem är för att kunna observera och utnyttja kvantmekaniska fenomen. Exempelvis kan höga temperaturer excitera systemet till högre energinivåer, och om energinivåer är ett av de tillstånden som är viktiga för vår applikation så kan det leda till brus och informationsförluster. Ett exempel skulle kunna vara kvantdatorer där vi utnyttjar superposition för att utföra beräkningar. Detta bygger på användandet av kvantbitar, som är en superposition av två tillstånd. Ofta kan den fysiska realiseringen av en kvantbit vara två energinivåer i ett system, där grundtillståndet motsvarar 0 och det första exciterade tillståndet motsvarar 1. Här är det lätt att se att om systemet exciteras av vår omgivning så förlorar vi den informationen som är lagrad i systemet. Kylning av kvantsystem är därför en viktig del för framtida forskning inom kvantteknologier.

## List of Abbreviations

**EOM** Equations of Motion. 10–12, 15, 18

**POVM** Positive Operator-Valued Measurement. 7

**QHO** Quantum Harmonic Oscillator. 3, 4, 6, 10, 21, 22, 24, 25

# Contents

Acknowledgements	i
Abstract	ii
Populärvetenskaplig sammanfattning	iii
List of Abbreviations	iv
<b>1 Introduction</b>	<b>1</b>
1.1 Outline . . . . .	2
<b>2 Theoretical Framework</b>	<b>3</b>
2.1 Quantum Harmonic Oscillator . . . . .	3
2.2 Open Quantum Systems . . . . .	4
2.2.1 Master Equation . . . . .	5
2.3 Continuous Measurements . . . . .	7
2.4 Feedback Control . . . . .	8
<b>3 Results</b>	<b>10</b>
3.1 Measurement Without Feedback . . . . .	10
3.2 Feedback . . . . .	15
<b>4 Discussion and Conclusions</b>	<b>22</b>
4.1 Cooling of a Quantum Harmonic Oscillator . . . . .	22
4.2 Stability with Feedback . . . . .	23
4.3 Applications . . . . .	24
<b>5 Outlook</b>	<b>25</b>
<b>References</b>	<b>26</b>



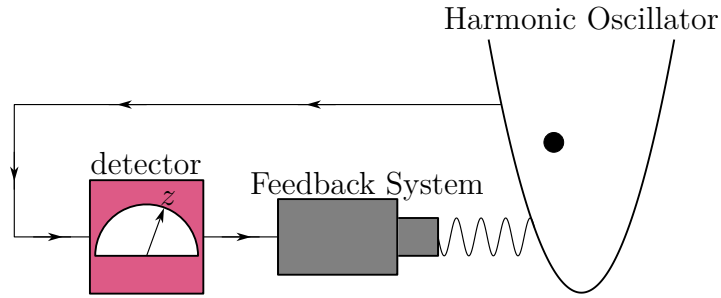
# 1 Introduction

As our society is becoming increasingly dependent on technology, the demand for better and more efficient technologies is growing. One kind of technology which has been really important since its conception during the second world war by Alan Turing is computers [1]. The field of computer science has been researched a lot during the second half of the 20th century but has hit a fundamental problem. Our current classical computers have reached a limit where the transistors cannot get any smaller without quantum mechanics causing issues [1]. This has prompted research into quantum technologies such as quantum computers and quantum simulations [1].

To properly understand how quantum mechanical systems work, it is important to understand what happens to them when they are interacted with, during for example a measurement [2]. It is also this interaction with a quantum system, which has prompted many interpretations of quantum mechanics and given rise to what is known as the measurement problem [2]. The measurement problem is a fundamental philosophical problem in quantum mechanics, which arises since a quantum mechanical state evolves deterministically according to the Schrödinger equation, but collapses probabilistically when measured or interacted with [2].

It is also interesting to see how such a system can be manipulated to create a certain state which can be used for a specific purpose. This could include, but is not limited to, creating a qubit state to be used in a quantum computer or a state which can simulate a certain physical system [1]. Here it is important to understand the effect feedback has on a measured system, and how to utilize this to create a desired state [3].

To create technologies based on quantum mechanical phenomena, we need systems where the quantum mechanical properties dominate. One way to achieve this is to cool the system to very low temperatures. There are several ways to do this: Laser cooling, evaporative cooling, dilution, and cavity cooling [4]. This thesis however, will focus on measurement and feedback based cooling, where the system is continuously measured, and the result is fed to a feedback loop that affects the system in a way that decreases the energy.



**Figure 1:** The system and feedback apparatus considered in this thesis. The measurement outcome is shown as  $z$  on the detector. The feedback system is drawn as a laser creating a field interacting with the harmonic potential.

The schematic system considered in the thesis can be seen in Fig. 1 showing a harmonic potential coupled to a detector and a feedback system. There are however, numerous physical realizations. It could be mechanical motion of a particle confined in a harmonic potential, for example an electron confined in a potential generated by lasers. It could also be realized as an optical or microwave cavity where photons are oscillating. Superconducting circuits are also a good example of a physical realization. Similarly, the feedback system also have numerous physical realizations. In Fig. 1 it is drawn as a laser, but in the case of a cavity, it could be realized as moving one of the mirrors in response to the measurement outcome.

## 1.1 Outline

The remaining text is organized as follows: Section 2 starts by introducing the theoretical framework central to this thesis by first defining what we mean by a quantum harmonic oscillator as well as shortly introducing the density matrix formalism of quantum mechanics. Then, we move on to discuss open quantum systems and the mathematical framework for the evolution of this type of system, and here we define a Markovian master equation in Lindblad form. We then move on to discussing measurements on open systems as well as feedback control, in the same mathematical formalism. In section 3, we use what has been discussed to derive equations of motion for the quantum harmonic oscillator as well as then computing the energy of the system. These results also allow analysing the stability of the system both with and without feedback. In section 4, there is a discussion of the relevancy of the results obtained in section 3, as well as mentions of a few applications.

Lastly, in section 5, there is a short outlook on future possible research in the field.

## 2 Theoretical Framework

In this section the theoretical framework needed is introduced. The Markovian master equations discussed are of interest and explain what types of approximations and assumptions are made about the model used in this thesis.

### 2.1 Quantum Harmonic Oscillator

The Quantum Harmonic Oscillator (QHO) is a quantum mechanical system useful for many applications. This stems from the fact that many systems can be approximated as harmonic, that is quadratic close to their equilibrium position, and since the QHO is a simple system, which is possible to solve analytically it is a good starting approximation. Let us start by assuming that a particle with mass  $m$  is confined in a harmonic potential, then the system has the Hamiltonian

$$\hat{H} = \frac{\hat{p}^2}{2m} + \frac{1}{2}m\omega^2\hat{x}^2 = \hbar\omega \left( \hat{a}^\dagger\hat{a} + \frac{1}{2} \right), \quad (1)$$

where  $\hat{p}$  and  $\hat{x}$  are the momentum and position operators,  $m$  is the mass of the particle, and  $\omega$  is the angular frequency of the oscillator. The operators  $\hat{a}^\dagger$  and  $\hat{a}$  are the creation and annihilation operators, collectively referred to as the ladder operators, which are defined as

$$\hat{a} = \sqrt{\frac{m\omega}{2\hbar}} \left( \hat{x} + \frac{i}{m\omega}\hat{p} \right) \quad \text{and} \quad \hat{a}^\dagger = \sqrt{\frac{m\omega}{2\hbar}} \left( \hat{x} - \frac{i}{m\omega}\hat{p} \right). \quad (2)$$

These operators can be used to define the number operator  $\hat{n} = \hat{a}^\dagger\hat{a}$  which has the number states, or Fock states,  $|n\rangle$  as its eigenstates with eigenvalue  $n$  [5]. The ladder operators have a useful the commutation relation  $[\hat{a}, \hat{a}^\dagger] = \mathbb{1}$ , that follows from the canonical commutation relation  $[\hat{x}, \hat{p}] = i\hbar$ . The Fock states are also eigenstates to the

ladder operators with properties

$$\hat{a} |n\rangle = \sqrt{n} |n-1\rangle, \quad (3)$$

$$\hat{a}^\dagger |n\rangle = \sqrt{n+1} |n+1\rangle, \quad (4)$$

which means that they change the excitation level of the QHO [5].

## 2.2 Open Quantum Systems

Before introducing open quantum systems, we shortly introduce the language of density matrices. A density matrix describes an ensemble of states defined as

$$\hat{\rho} = \sum_i p_i |\psi_i\rangle \langle\psi_i|, \quad (5)$$

where there is probability  $p_i$  for the system to be prepared in the state  $|\psi_i\rangle$  [1]. Two conditions imposed on a density matrix is that it I) has unit trace, and II) is positive semi-definite [1]. These conditions ensure that the probabilities are  $0 \leq p_i \leq 1$  and  $\sum_i p_i = 1$  and that the density matrix is hermitian. With this definition of the density matrix object, we can reformulate the postulates of quantum mechanics allowing for open systems and mixed states [1].

With an open quantum system we mean a quantum system which in some ways interacts with an environment [3]. This interaction could be described as a thermal coupling between the main system and some temperature bath. This will cause the system to be in a thermal equilibrium with the environment if left alone, and therefore it will be dependent on the temperature. Unless the temperature of the bath is zero, the system will be in a mixed state, described by a density matrix  $\hat{\rho}$ . Notably, if the temperature is zero, the system is purely dissipative since the bath cannot excite the system, and thus energy can only be transferred from the system to the bath.

The thermal coupling to the environment will lead to dissipation of quantum information from the system to the environment [3]. During this process, the system loses coherence. That is, the quantum mechanical properties of the system are lost and a clas-

sical description of the state becomes more appropriate. The coherence of the system is manifested in the off-diagonal elements of the density matrix. If the off-diagonal elements are zero, either by dissipation to the environment or by other means of decoherence, the system will exist in a classical probabilistic state, and any superposition of states will be lost.

The combination of the system and environment can be considered a closed system, though more complicated than the main system itself. Then, by performing a partial trace over the environment, a description of the system alone arises at the cost of losing information about the correlation between the two parts [3]. This introduces an uncertainty in the state, and it is therefore necessary to treat the resulting system to be in a mixed state. To describe the evolution of this system with a master equation two approximations about the coupling need to be performed. First, we consider the Born approximation, which says that the coupling between the system and environment is weak enough that only negligible excitations appear in the environment [6]. The other approximation is the Markov approximation saying that the excitations that do appear in the environment will decay much faster than the timescale that the system varies on, and that the system's time evolution is only affected by the current state of the system and not previous states [6]. Together these approximations allow us to write the total density matrix as

$$\rho_{SE} = \rho_S \otimes \rho_E, \quad (6)$$

where  $\otimes$  is the tensor product, and derive a Markovian master equation. For ease of notation we will drop the subscript S and always consider the system unless otherwise stated.

### 2.2.1 Master Equation

The evolution of an open quantum system can be described by a master equation, which is a differential equation and generalization of the Schrödinger equation to involve open

quantum systems instead of pure states [3]. By introducing the superoperator

$$\mathcal{D}[\hat{L}]\hat{\rho} = \hat{L}\hat{\rho}\hat{L}^\dagger - \frac{1}{2}\left\{\hat{L}^\dagger\hat{L}, \hat{\rho}\right\}, \quad (7)$$

where  $\hat{L}$  is called a Lindblad jump operator, the master equation on Lindblad form can be written as

$$\partial_t \hat{\rho} = -\frac{i}{\hbar}[\hat{H}, \hat{\rho}] + \sum_k \gamma_k \mathcal{D}[\hat{L}_k]\hat{\rho}, \quad (8)$$

where  $\hat{H}$  is the Hamiltonian of the system, and  $\gamma_k$  are the decay rates of the system, relating the decoherence to the environment depending on the coupling to the system [3]. If  $\gamma_k = 0$  for all  $k$ , and the coupling to the bath is removed, the equation reduces to the von Neumann equation for a closed quantum system. The remaining term thus describes the unitary time evolution of the system and is the analogue of the Schrödinger equation for the density matrix formalism [3]. At this stage, one might also introduce the Liouvillian superoperator  $\mathcal{L}$ , and write the master equation (8), more compactly as

$$\partial_t \hat{\rho} = \mathcal{L}\hat{\rho}. \quad (9)$$

This compactness will be useful when considering other types of perturbing effects on the system such as measurements and feedback [3].

In the case considered in this thesis with a QHO coupled to a thermal reservoir we can imagine that we have two types of Lindblad jump operators, one of which transfers energy into the system and one transfers energy out of the system [5]. As mentioned in Sec. 2.1 the ladder operators can be used to excite or deexcite a system. Since an increased temperature indicates larger fluctuations, it is also reasonable to assume that the decay, or the energy flowing between the systems and the environment, is proportional to the thermal occupation  $\bar{n}$ , defined by

$$\bar{n} = \frac{1}{e^{\hbar\omega/k_B T} - 1}, \quad (10)$$

where  $T$  is the temperature of the bath and  $k_B$  is the Boltzmann constant [5]. If one

starts from a microscopic point of view of a QHO coupled to a thermal bath, it is possible to show that the Lindblad jump operators become  $\hat{L}_1 = \hat{a}$  and  $\hat{L}_2 = \hat{a}^\dagger$  with coefficients  $\gamma_1 = \gamma(\bar{n} + 1)$  and  $\gamma_2 = \gamma\bar{n}$ , where  $\gamma$  is a decay rate. We can also note that  $\hat{L}_1$  and  $\gamma_1$  refer to the spontaneous emission from the system to the environment while  $\hat{L}_2$  refer to spontaneous absorption from the environment to the system, consistent with what we know about ladder operators from Sec. 2.1 [5]. Notably, for  $T = 0$ , the thermal occupation is  $\bar{n} = 0$  and the system will only exhibit emission and will decay.

### 2.3 Continuous Measurements

Measurement is a process which allow us to know the state of the system and has the consequence of introducing decoherence, and it is therefore interesting to look at its effects [2]. The simplest view on measurements takes the form of von Neumann measurements. This type of measurement is described by a set of measurement operators which projects the system onto the eigenstates of the observable [3]. This essentially means that all quantum information in the system is lost and full decoherence has happened. By generalizing the measurement theory one can derive what is called Positive Operator-Valued Measurement (POVM) [3].

Since the POVM is not necessarily a projective von Neumann measurement all coherence need not be lost after the measurement. Thus, this opens up for the possibility of performing time-continuous weak measurement [3]. To describe this type of POVM, we first consider a Gaussian measurement operator

$$\hat{K}(z) = \left(\frac{2\bar{\lambda}}{\pi}\right)^{1/4} e^{-\bar{\lambda}(z-\hat{A})^2}, \quad (11)$$

where  $\bar{\lambda}$  represents the strength of the measurement,  $z$  is a continuous outcome of the measurement, and  $\hat{A}$  is the measured observable [3]. We note that the post measurement state of such a measurement is described by

$$\hat{\rho}_{\text{post}} = \frac{\hat{K}(z)\hat{\rho}\hat{K}^\dagger(z)}{p(z)}, \quad (12)$$

where the probability is defined as  $p(z) = \text{tr}(\hat{K}^\dagger(z)\hat{K}(z)\hat{\rho})$  [3]. In this thesis we will only consider the case where  $\hat{A} = \hat{x}$ , that is we only measure the position quadrature.

Then by discretizing the time interval to segments of  $dt$  and defining  $\bar{\lambda} = \lambda dt$  we approach a situation where in the limit  $dt \rightarrow 0$  all measurements will be weak, and the coherence of the system is minimally affected [3]. Considering the stochastic nature of the process and averaging the possible trajectories one can derive the master equation [3] in Lindblad form to be

$$\partial_t \hat{\rho} = \mathcal{L}\hat{\rho} + \lambda \mathcal{D}[\hat{A}]\hat{\rho}. \quad (13)$$

That is, we do not use the actual measurement outcome, but instead consider the system as affected when averaging over the measurement outcomes.

## 2.4 Feedback Control

Until this point we have only considered measurements where we omit the information about the measurement outcome. That is, we interact with the system and look at how it evolves due to this interaction on average, instead of looking at the specific outcome of any given measurement [3]. However, now we want to consider feedback control of the system, and thus we will need to include the information about the measurement outcome [3]. By feedback control we mean a process by which we manipulate the evolution of a system using a measurement outcome [7]. Since we are dealing specifically with quantum systems, we can further talk about quantum feedback control, where quantum mechanical effects play a role in the modelling of the feedback mechanism's effect on the system [7]. However, it is worth noting that the physical realization of the feedback mechanism does not necessarily need to be entirely quantum mechanical, but at least part of the mechanism needs to incorporate quantum mechanics in its description [7]. An example of this could be a cavity where the feedback is moving one of the ends of the cavity, which is a mechanical process, the effect on the field inside the cavity however, need to be treated quantum mechanically.

In this thesis, for a measurement outcome  $z$  we will consider a linear feedback modi-



fication of  $\mathcal{L}$  such that

$$\mathcal{L} \rightarrow \mathcal{L} + z\mathcal{K}, \quad (14)$$

where  $\mathcal{K}$  is a superoperator describing the feedback on the system [3] which takes the form

$$\mathcal{K}\hat{\rho} = -\frac{i}{\hbar}[\hat{H}_c, \hat{\rho}], \quad (15)$$

where  $\hat{H}_c$  is the control Hamiltonian of the system. We consider a control Hamiltonian which is linear and has the form

$$\hat{H}_c = f^*\hat{a} + f\hat{a}^\dagger, \quad (16)$$

where  $f$  is the feedback amplitude, which essentially changes the system Hamiltonian to

$$\hat{H}_{\text{tot}} = \hat{H}_{\text{QHO}} + \hat{H}_c = \hbar\omega \left( \hat{a}^\dagger\hat{a} + \frac{1}{2} \right) + z(f^*\hat{a} + f\hat{a}^\dagger). \quad (17)$$

Thus, for a real  $f$ ,  $\Im\{f\} = 0$ , we have  $\hat{H}_c \propto \hat{x}$  and for an imaginary  $f$ ,  $\Re\{f\} = 0$ , we have  $\hat{H}_c \propto -\hat{p}$ . If the physical realization of the feedback is for example a laser, then  $|f|$  represent the power while  $\arg f$  represent the relative phase difference between the oscillator's measured quadrature and the feedback field. Starting from the same place as one derives Eq. (13) we can derive a master equation including feedback [3] to be

$$\partial_t \hat{\rho} = \mathcal{L}\hat{\rho} + \lambda \mathcal{D}[\hat{A}]\hat{\rho} + \frac{1}{2}\mathcal{K}\{\hat{A}, \hat{\rho}\} + \frac{1}{8\lambda}\mathcal{K}^2\hat{\rho}, \quad (18)$$

where the square on  $\mathcal{K}$  means  $\mathcal{K}^2\hat{\rho} = \mathcal{K}(\mathcal{K}\hat{\rho})$ . This result is also not dependent on the measurement outcome but describes the system as one averages over all measurement outcomes. The first additional term introduced is the drive arising from the feedback on the system, while the second term is attributed to the noise and fluctuations of the feedback [3]. We can see that as the measurement strength,  $\lambda \rightarrow \infty$ , the noise term vanishes. However, this would also make the measurement term diverge and leave us with a von Neumann measurement.

### 3 Results

This section looks at the equations of motion and steady-state solutions of a QHO, both with and without feedback. The first subsection deals with a QHO which is measured continuously and without feedback, while the second subsection adds feedback into the scheme.

#### 3.1 Measurement Without Feedback

Consider a QHO described by the Hamiltonian in Eq. (1) which is coupled to a thermal bath with temperature  $T$ . If the oscillator's position quadrature is also continuously measured, the evolution of the system can be described by the master equation in Eq. (13) using the Lindblad operators mentioned in Sec. 2.2.1. We want to solve for the Equations of Motion (EOM) for the first and second moments, when measuring for the position quadrature  $\hat{x}$ .

We start by looking at the first moments which are derived from

$$\partial_t \langle \hat{x} \rangle = \text{tr}(\hat{x} \partial_t \hat{\rho}), \quad (19)$$

$$\partial_t \langle \hat{p} \rangle = \text{tr}(\hat{p} \partial_t \hat{\rho}). \quad (20)$$

Inserting the master equation in Eq. (13) we find

$$\partial_t \langle \hat{x} \rangle = -\frac{\gamma}{2} \langle \hat{x} \rangle - \frac{1}{m} \langle \hat{p} \rangle, \quad (21)$$

$$\partial_t \langle \hat{p} \rangle = -\frac{\gamma}{2} \langle \hat{p} \rangle + m\omega^2 \langle \hat{x} \rangle. \quad (22)$$

Then when solving for the steady state we find

$$\langle \hat{x} \rangle = \langle \hat{p} \rangle = 0. \quad (23)$$

This result also confirms the intuition that for a harmonic oscillator, the system has a steady state around the origin.

It is also important to investigate the stability of the system. That is, if and when the system, in the long time limit, will end up in a steady state, and that the steady state is stable to small perturbations. To check for stability we can rewrite the equation as an eigenvalue problem and solve for the eigenvalues.

$$\partial_t \begin{pmatrix} \langle \hat{x} \rangle \\ \langle \hat{p} \rangle \end{pmatrix} = \mathcal{M} \begin{pmatrix} \langle \hat{x} \rangle \\ \langle \hat{p} \rangle \end{pmatrix}, \quad (24)$$

for the matrix

$$\mathcal{M} = \begin{pmatrix} -\gamma/2 & -1/m \\ m\omega^2 & -\gamma/2 \end{pmatrix}. \quad (25)$$

The eigenvalues of  $\mathcal{M}$  are

$$\varepsilon_1 = -\frac{\gamma}{2} - i\omega, \quad (26)$$

$$\varepsilon_2 = -\frac{\gamma}{2} + i\omega. \quad (27)$$

Since the real part of the eigenvalues are negative the system is stable. This is because we can write the time-dependent solutions of the dynamical equations as an exponential with the eigenvalues while the coefficients are chosen by the initial condition, the value of the eigenvalues will determine the behaviour of the system. The consequence of the real part of the eigenvalue being negative is that it will make the function decay with time and is thus considered stable. If both eigenvalues are negative the system will decay no matter what the initial conditions are since both terms in the solution will decay. Positive eigenvalues on the other hand will correspond to the system growing with time and will thus be unstable. When one of the eigenvalues is positive and the other is negative the stability of the system will be determined by the initial conditions. That is, the initial condition will determine which eigenvalue will dominate the solution.

We now solve the EOM for the second moments using the same method of inserting

the master equation, derived from

$$\partial_t \langle \hat{x}^2 \rangle = \text{tr}(\hat{x}^2 \partial_t \hat{\rho}), \quad (28)$$

$$\partial_t \langle \hat{p}^2 \rangle = \text{tr}(\hat{p}^2 \partial_t \hat{\rho}), \quad (29)$$

$$\partial_t \langle \{\hat{x}, \hat{p}\} \rangle = \text{tr}(\{\hat{x}, \hat{p}\} \partial_t \hat{\rho}). \quad (30)$$

The choice of looking at the second moments has to do with their relation to the variance and thus the fluctuations of the system. These fluctuations are then related to the energy contained in the oscillator, as the expectation value of the Hamiltonian. When looking at feedback control one application is to minimize the energy in the oscillator which is thus also an argument for looking at the second moments. Inserting the quantum master equation in Eq. (13) and solving Eqs. (28) to (30) yields the following EOM

$$\partial_t \langle \hat{x}^2 \rangle = -\gamma \langle \hat{x}^2 \rangle - \frac{1}{m} \langle \{\hat{x}, \hat{p}\} \rangle + \frac{\gamma \hbar}{m\omega} (\bar{n} + 1/2), \quad (31)$$

$$\partial_t \langle \hat{p}^2 \rangle = -\gamma \langle \hat{p}^2 \rangle + m\omega^2 \langle \{\hat{x}, \hat{p}\} \rangle + \gamma m\omega \hbar (\bar{n} + 1/2) + \lambda \hbar^2, \quad (32)$$

$$\partial_t \langle \{\hat{x}, \hat{p}\} \rangle = -\gamma \langle \{\hat{x}, \hat{p}\} \rangle + 2m\omega^2 \langle \hat{x}^2 \rangle - \frac{2}{m} \langle \hat{p}^2 \rangle. \quad (33)$$

Performing a change of variable to make the equations dimensionless

$$\tilde{x} = \sqrt{\frac{m\omega}{\hbar}} x \quad \text{and} \quad \tilde{p} = \sqrt{\frac{1}{m\omega\hbar}} p, \quad (34)$$

we can solve for the steady state, by setting the time derivative to zero. Introducing the quality factor  $Q = \omega/\gamma$  we obtain the steady-state solutions

$$\langle \tilde{x}^2 \rangle_{\text{ss}} = (\bar{n} + 1/2) + \frac{\lambda \hbar}{m\omega^2} \frac{2Q^3}{4Q^2 + 1}, \quad (35)$$

$$\langle \tilde{p}^2 \rangle_{\text{ss}} = (\bar{n} + 1/2) + \frac{\lambda \hbar}{m\omega^2} \left( Q - \frac{2Q^3}{4Q^2 + 1} \right), \quad (36)$$

One aspect of the steady-state solutions is that both equations have identical terms capturing the thermal aspect of the fluctuations. Without measurement, the steady states are only thermal, which is to be expected. When calculating the energy of the steady

state the complicated fraction above cancels, and we are left with a purely linear term in the quality factor.

$$E_{\text{ss}} = \left\langle \tilde{H} \right\rangle_{\text{ss}} = \frac{\hbar\omega}{2} (\langle \tilde{p}^2 \rangle_{\text{ss}} + \langle \tilde{x}^2 \rangle_{\text{ss}}) = \hbar\omega(\bar{n} + 1/2) + \frac{\lambda\hbar^2}{2m\omega}Q \quad (37)$$

To simplify plotting we might perform the change of variable

$$\tilde{E} = \frac{E}{\hbar\omega}, \quad \Lambda = \frac{\lambda\hbar}{m\omega^2}. \quad (38)$$

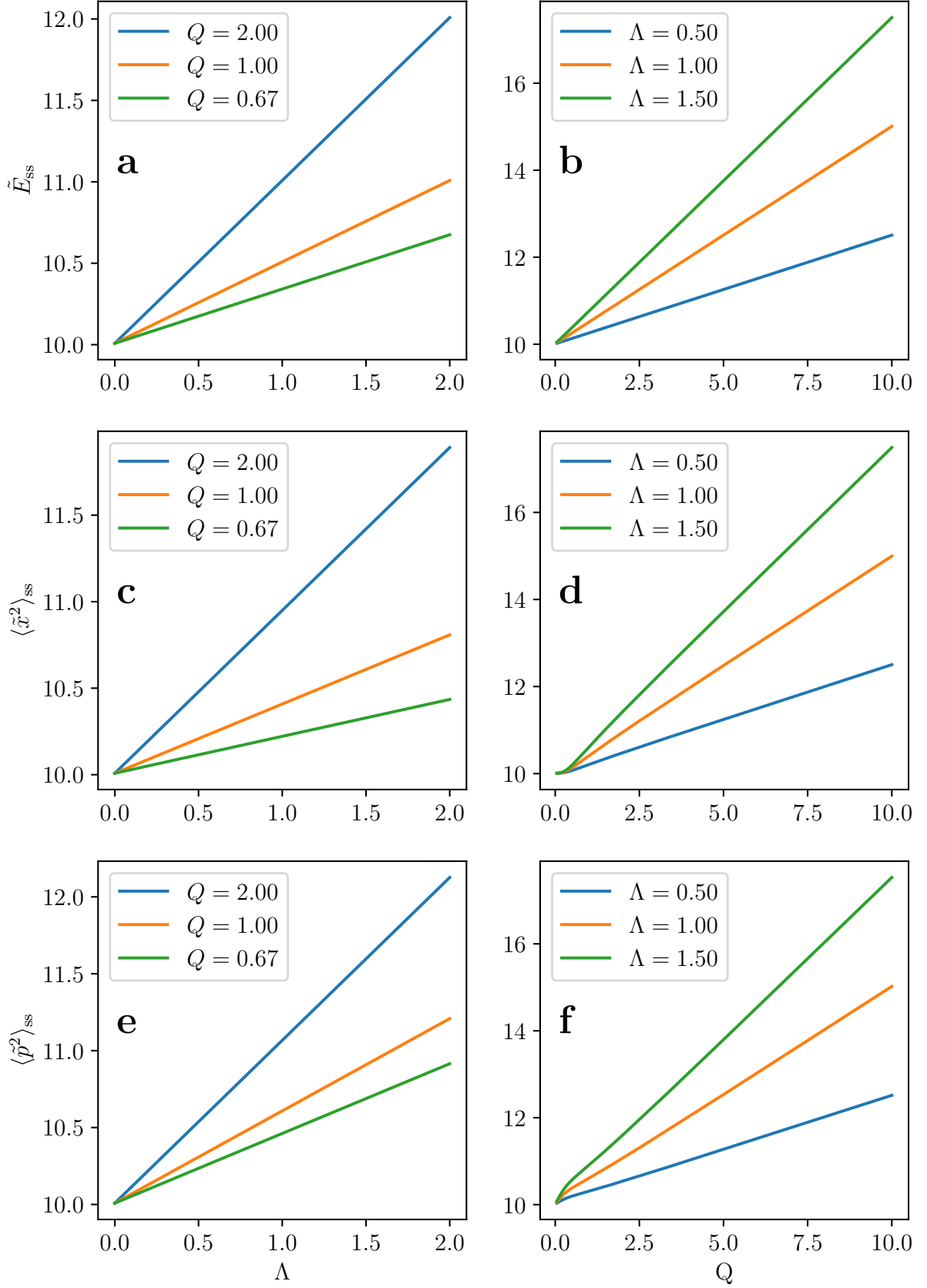
Looking at panels **a** and **b** in Fig. 2 one can see that a stronger measurement correlates to the system's steady state increasing in energy, as does it for an increasing quality factor. Both also affect the system linearly. Thus, by continuously measuring the system we add energy into it, which makes the steady state higher in energy than what the thermal effects from the bath would otherwise place it. That is, if we do not perform any measurement the system would be stable at around  $\bar{n} + 1/2$  which for the parameters used here would give  $E_{\text{ss}} \approx 10\hbar\omega$ .

In panels **d** and **f** we can see that there is some non-linear behaviour near  $Q = 0$ . However, due to the approximations made in Sec. 2.2 we cannot trust the results in the region of a low quality factor. This is because this regime has a relatively high coupling to the environment, and thus the excitations in the environment might not decay fast enough and therefore affect the oscillator. Panels **c** and **e** show the linear dependence on  $\Lambda$  for the second moments.

Eq. (23) together with Eqs. (35) and (36) also shows that the variance of the system is only dependent on the second moments, since

$$\sigma_{\hat{A}}^2 = \left\langle \hat{A}^2 \right\rangle - \left\langle \hat{A} \right\rangle^2 \quad (39)$$

for an operator  $\hat{A}$ . It is also an easy calculation then to show that for temperature  $T = 0$  and without measurement we have equality in the Heisenberg uncertainty relation,  $\sigma_{\hat{x}}\sigma_{\hat{p}} = \hbar/2$ , justifying the accuracy of the results, and the approximations made in the



**Figure 2:** The top panels show Eq. (37), the middle panels show Eq. (35) and the bottom panels show Eq. (36). All plots use the parameter  $k_B T = 10\hbar\omega$ . The left panels are plotted against  $\Lambda$  and with three different values for  $Q$ , while the right panels are plotted against  $Q$  and three different values of  $\Lambda$ .

derivation of the master equation.

### 3.2 Feedback

We now consider a feedback mechanism on the oscillator described by Eq. (18) which is a linear feedback scheme. Solving for the first moments' EOM we find

$$\partial_t \langle \hat{x} \rangle = - \left( \frac{\gamma}{2} + \frac{2\Im\{f\}}{\sqrt{2m\omega\hbar}} \right) \langle \hat{x} \rangle - \frac{1}{m} \langle \hat{p} \rangle, \quad (40)$$

$$\partial_t \langle \hat{p} \rangle = -\frac{\gamma}{2} \langle \hat{p} \rangle + \left( \Re\{f\} \sqrt{\frac{2m\omega}{\hbar}} + m\omega^2 \right) \langle \hat{x} \rangle. \quad (41)$$

We see that when measuring  $\hat{x}$  the feedback terms enter the equations on the position quadrature. A real  $f$ ,  $\Im\{f\} = 0$ , removes the feedback for the equation of the position, while an imaginary  $f$ ,  $\Re\{f\} = 0$ , removes the feedback for the equation of the momentum. The equations are still coupled however, since the position is dependent on the momentum and vice versa. Setting  $f = 0$ , yields the previous result. The steady state solutions for this system are  $\langle \hat{x} \rangle = \langle \hat{p} \rangle = 0$ .

We also notice that it is possible to remove  $\langle \hat{x} \rangle$  from the equations, so the system reduces to

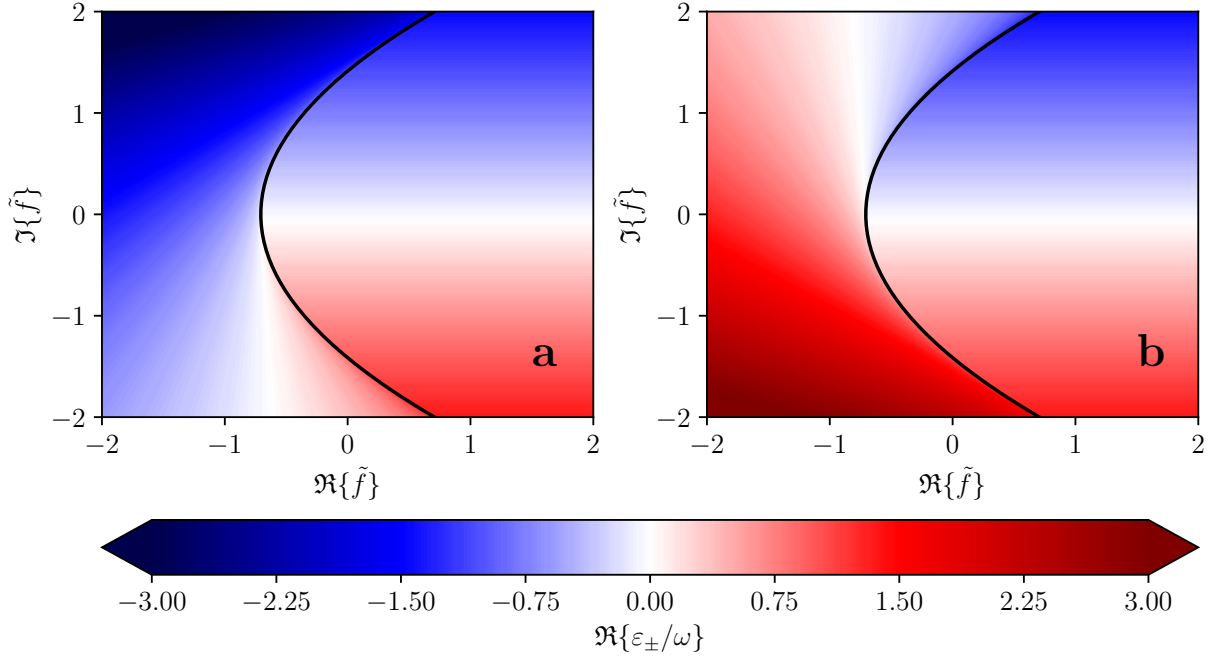
$$\partial_t \langle \hat{x} \rangle = -\frac{1}{m} \langle \hat{p} \rangle, \quad (42)$$

$$\partial_t \langle \hat{p} \rangle = -\frac{\gamma}{2} \langle \hat{p} \rangle, \quad (43)$$

by choosing  $f$  such that

$$\Re\{f\} = -\sqrt{\frac{m\omega^3\hbar}{2}} \quad \text{and} \quad \Im\{f\} = -\frac{\gamma\sqrt{m\omega\hbar}}{2\sqrt{2}}, \quad (44)$$

which has a steady-state solution for  $\langle \hat{p} \rangle = 0$  and any  $\langle \hat{x} \rangle$ . We find the eigenvalues for this choice of  $f$  to be  $\varepsilon_- = -\gamma/2$  and  $\varepsilon_+ = 0$ . This choice of  $f$  removes all the oscillatory behaviour from the system, which can be seen by the eigenvalues being real. This puts it in the region of being overdamped by the choice of  $f$ . Since one of the eigenvalues is zero,



**Figure 3:** Eq. (47) plotted as a contour plot against the real and imaginary part of  $f$ . The parameter used is  $Q = 10$ . Panel **a** shows  $\Re\{\varepsilon_-/\omega\}$  and panel **b** shows  $\Re\{\varepsilon_+/\omega\}$ . The parabola that can be seen in both panels is the points where the first square root in Eq. (47) is zero. Thus, points to the left of this parabola are real, while points to the right are complex, only the real part is plotted, however.

the system is also one dimensional, which is reasonable as the choice of  $f$  is such that the first moments' dynamics are independent of the position average.

To check for stability of the first moments, we first perform the change of variable

$$\tilde{f} = \frac{f}{\omega\sqrt{m\omega\hbar}}, \quad (45)$$

and rewrite the equations as an eigenvalue problem with matrix

$$\mathcal{M} = \begin{pmatrix} -\left(\frac{\gamma}{2} + \sqrt{2}\Im\{\tilde{f}\}\omega\right) & -\frac{1}{m} \\ m\omega^2\left(\sqrt{2}\Re\{\tilde{f}\} + 1\right) & -\frac{\gamma}{2} \end{pmatrix}, \quad (46)$$

which has eigenvalues

$$\frac{\varepsilon_{\pm}}{\omega} = -\frac{\Im\{\tilde{f}\}}{\sqrt{2}} - \frac{1}{2Q} \pm \sqrt{\frac{\Im\{\tilde{f}\}^2}{2} - \sqrt{2}\Re\{\tilde{f}\} - 1} \quad (47)$$

Looking at Fig. 3 we can see that  $\Re\{\varepsilon_-/\omega\}$  is mostly negative while  $\Re\{\varepsilon_+/\omega\}$  is



mostly positive for the parameters used in the region closest to the origin. The only part where both eigenvalues are negative is in top right region of the figures. That is, the region where  $\Im\{\tilde{f}\} > 0$  and  $\Re\{\tilde{f}\} \gtrsim -0.7$ . The reason for the approximation is that it is not an analytical result, but instead apparent from looking at panel **b**, where we can see that line where the eigenvalue is zero, and has a slant and is not vertical. Another thing to note is that the eigenvalues are complex to the right of the black parabola in the Fig. 3, and thus the solution will have oscillatory motion in this region.

It is important to remember that we can solve the dynamics of the system as exponentials with the eigenvalues, and some coefficients determined by the initial conditions. For a vector equation on the form

$$\partial_t \mathbf{x}(t) = \mathcal{A} \mathbf{x}(t), \quad (48)$$

with a vector  $\mathbf{x}$  and a diagonalizable matrix  $\mathcal{A}$ , we can write the solution as

$$\mathbf{x}(t) = \sum_i c_i e^{\lambda_i t} \mathbf{u}_i, \quad (49)$$

where  $\lambda_i$  is the  $i$ 'th eigenvalue,  $\mathbf{u}_i$  is the  $i$ 'th eigenvector, and  $c_i$  is set by the initial conditions. It is apparent that in the case when  $c_i \neq 0$  for all  $i$ , the eigenvalues  $\lambda_i$ , must have a negative real part for the system to be stable, as otherwise one of the terms in the sum would diverge. If  $\mathcal{A}$  has degenerate eigenvalues, the solution would be a linear combination of  $e^{\lambda t}$  and  $te^{\lambda t}$ , however the real part of the eigenvalue still need to be negative for stability to be present.

Solving for the EOM for the second moments we obtain

$$\begin{aligned} \partial_t \langle \hat{x}^2 \rangle = & - \left( \gamma + \frac{4\Im\{f\}}{\sqrt{2m\omega\hbar}} \right) \langle \hat{x}^2 \rangle - \frac{1}{m} \langle \{\hat{x}, \hat{p}\} \rangle + \frac{\gamma\hbar}{m\omega} (\bar{n} + 1/2) \\ & - \frac{1}{4\lambda m\omega\hbar} \Re\{f\}^2, \end{aligned} \quad (50)$$

$$\begin{aligned} \partial_t \langle \hat{p}^2 \rangle = & -\gamma \langle \hat{p}^2 \rangle + \left( m\omega^2 + \Re\{f\} \sqrt{\frac{2m\omega}{\hbar}} \right) \langle \{\hat{x}, \hat{p}\} \rangle + \gamma m\omega\hbar (\bar{n} + 1/2) \\ & + \lambda\hbar^2 + \frac{m\omega}{4\lambda\hbar} \Re\{f\}^2, \end{aligned} \quad (51)$$

$$\begin{aligned} \partial_t \langle \{\hat{x}, \hat{p}\} \rangle = & - \left( \gamma + \frac{2\Im\{f\}}{\sqrt{2m\omega\hbar}} \right) \langle \{\hat{x}, \hat{p}\} \rangle + \left( 2m\omega^2 + 2\sqrt{\frac{2m\omega}{\hbar}} \Re\{f\} \right) \langle \hat{x}^2 \rangle - \frac{2}{m} \langle \hat{p}^2 \rangle \\ & - \frac{\Re\{f\} \Im\{f\}}{2\lambda\hbar}. \end{aligned} \quad (52)$$

We can solve for the steady state by first rewriting the system of equations to a matrix equation  $\partial_t X = \mathcal{A}X + \mathcal{B}$  where

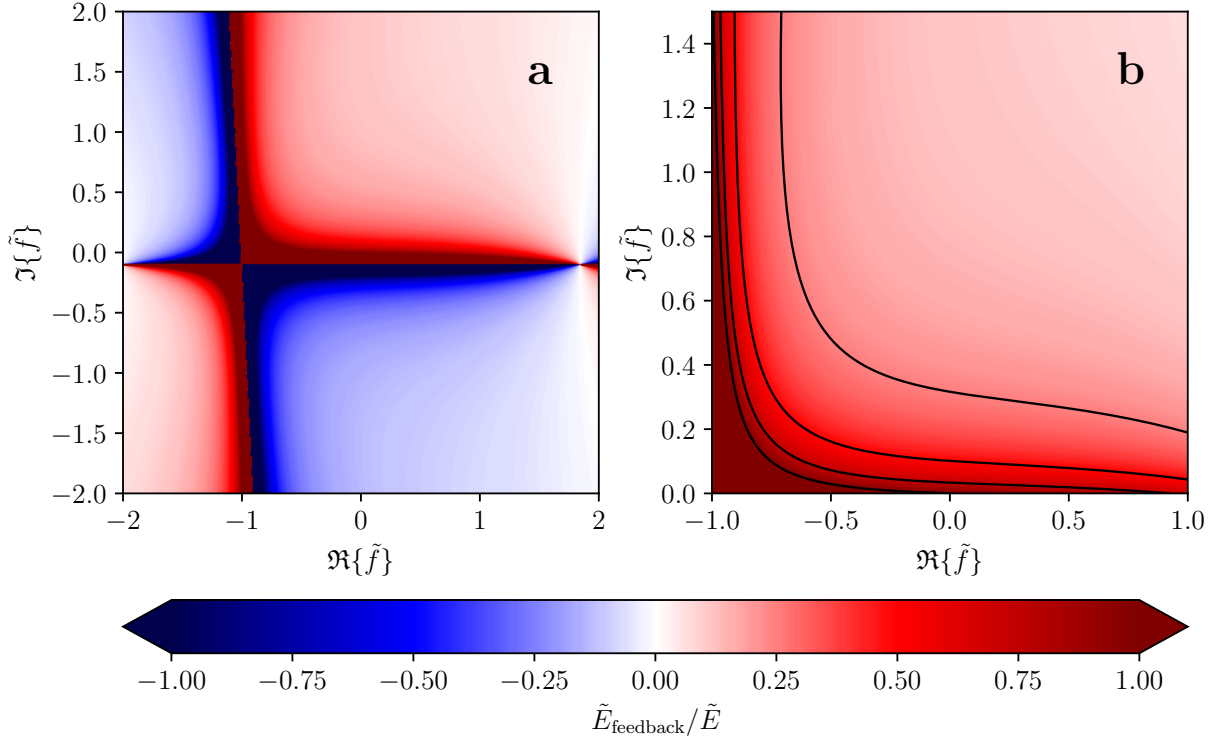
$$X = \begin{pmatrix} \langle \hat{x}^2 \rangle \\ \langle \hat{p}^2 \rangle \\ \langle \{\hat{x}, \hat{p}\} \rangle \end{pmatrix}, \quad \mathcal{A} = \begin{pmatrix} -\left(\gamma + \frac{4\Im\{f\}}{\sqrt{2m\omega\hbar}}\right) & 0 & -\frac{1}{m} \\ 0 & -\gamma & m\omega^2 + \Re\{f\} \sqrt{\frac{2m\omega}{\hbar}} \\ 2m\omega^2 + 2\sqrt{\frac{2m\omega}{\hbar}} \Re\{f\} & -\frac{2}{m} & -\left(\gamma + \frac{2\Im\{f\}}{\sqrt{2m\omega\hbar}}\right) \end{pmatrix} \quad (53)$$

and

$$\mathcal{B} = \begin{pmatrix} \frac{\gamma\hbar}{m\omega} (\bar{n} + 1/2) - \frac{\Re\{f\}^2}{4\lambda m\omega\hbar} \\ \gamma m\omega\hbar (\bar{n} + 1/2) + \lambda\hbar^2 + \frac{m\omega \Re\{f\}^2}{4\lambda\hbar} \\ -\frac{\Re\{f\} \Im\{f\}}{2\lambda\hbar} \end{pmatrix}. \quad (54)$$

We notice the addition of a constant term in all three equations corresponding to the noise of the feedback, that is, all these terms comes from the second feedback term in Eq. (18). Furthermore, we see that a driving feedback term has entered all three equations on the position quadrature and the anticommutator, which contains  $\hat{x}$ . It's worth noting the similarities of the terms between equations.

To obtain the steady-state solutions we solve  $X_{\text{ss}} = -\mathcal{A}^{-1}\mathcal{B}$ . To simplify this we perform the same change of variables as before. The full analytical solutions are not shown here due to the length of the equations. It would also be possible to solve the



**Figure 4:** The ratio between the energy with feedback and without plotted as a contour plot against  $\Re\{\tilde{f}\}$  and  $\Im\{\tilde{f}\}$ . The parameters used are  $k_{\text{B}}T = 10\hbar\omega$ ,  $Q = 10$ ,  $\Lambda = 2$ . Panel **a** shows a large variation of the parameters. There are divergences in the plot which almost follow a  $(\Re\{\tilde{f}\} + 1)\Im\{\tilde{f}\} = 1$  curve. Panel **b** is a zoomed in version of panel **a** and shows the behaviour of the system in a region where the ratio always is positive. The contour lines in panel **b** are placed at the values of the tick marks in the colourbar.

matrix equation numerically. To get an understanding of the steady-state solution we look at figures generated by the solution.

In Fig. 4 we can see the effect the feedback has on the energy of the system. Looking at panel **a**, we can see areas which are negative. If the energy of the system without measurement is constant, it must be the feedback energy that becomes negative. In the way we have set up the model, a negative energy is unreasonable, since a temperature of  $T = 0$  would correspond to  $E = \hbar\omega(1/2 + \Lambda Q/2)$ , which without measurement would mean an energy of  $E = \hbar\omega/2$ . A possible explanation for the negative energy is that the specific feedback that give rise to this is affecting the system in such a way that at least one of our assumptions is no longer accurate. This could, for example, be that the system no longer have a physical steady state. Another reason might be that the Markovian approximation no longer holds.

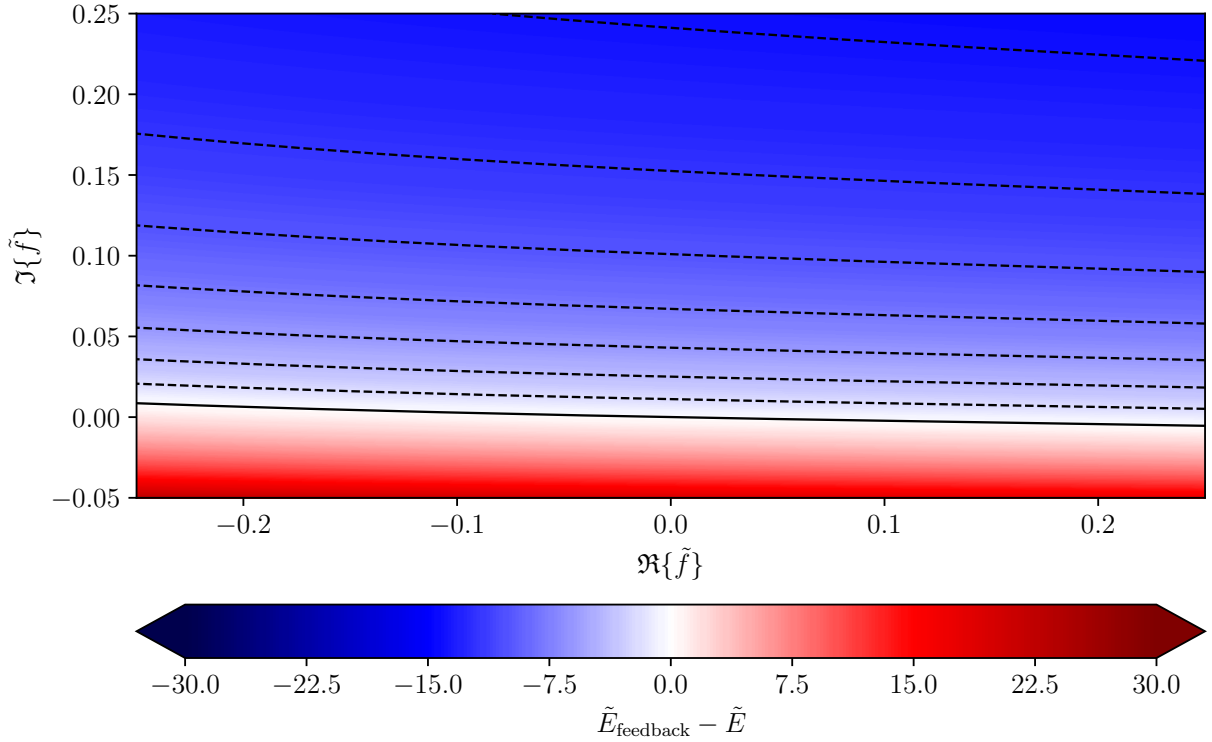
Another interesting aspect of Fig. 4 is that when looking at panel **b** in conjunction with Eq. (37) is that it is possible to cool the system to a lower energy than the thermal energy. Using the same parameters in Eq. (37) as are used in the figure we find that the thermal energy is half of the total energy in the system without feedback, and looking at the figure we see that there exist a region which has a ratio of less than 0.5.

If one tweaks the parameters used in Fig. 4 the general form of the plot remains the same, with some differences. The measurement strength  $\Lambda$  does not affect the plot much, but it does however increase the energy, the contour lines in panel **b** remains almost the same. Thus, the same feedback would remove the same proportion of energy, it does however mean that a larger proportion need to be removed before cooling below the thermal equilibrium. Similarly, changing  $k_B T$  does not dramatically change how the plot looks, but it does increase the energy of the system as well. However, since the thermal energy now constitute a larger part of the total energy of the system, a smaller feedback parameter is needed to cool the system below the thermal equilibrium. The quality factor  $Q$  however, changes the plot quite a lot, decreasing the spacing between the contour lines and increasing the efficiency of the cooling.

Using the parameters as in Fig. 5 the energy without feedback is  $\langle \tilde{E} \rangle \approx 20$  with the thermal part accounting for around 10 of that. Looking at Fig. 5 it is possible to see that even with a relatively low amount of feedback the system will be cooled to some degree. However, depending on how the feedback is applied it can also make the energy in the system diverge which can be seen for  $\Im \left\{ \tilde{f} \right\} < 0$ .

Due to the uncertainty relation and the canonical commutation relation  $\hat{x}^2$ ,  $\hat{p}^2$ , and  $\{\hat{x}, \hat{p}\}$  are not independent [8]. This has the consequence that the stability analysis cannot be performed with  $\mathcal{A}$  and  $\mathcal{B}$ . Instead, we rewrite  $X$  as a quantum variance matrix [8], and write Eqs. (50) to (52) as a Lyapunov equation

$$\partial_t S = \mathcal{M}S + S\mathcal{M}^T + \mathcal{C} = 0 \quad (55)$$



**Figure 5:** The difference between the energy of the system with feedback plotted and the energy without feedback plotted against  $\Re\{\tilde{f}\}$  and  $\Im\{\tilde{f}\}$ . The parameters used are  $k_B T = 10\hbar\omega$ ,  $Q = 10$ ,  $\Lambda = 2$ . The plot is done in a small region around the origin, thus showing the effect a small feedback has on the system. The spacing between the contour lines is 2 and the solid lines is at 0.

where

$$S = \begin{pmatrix} \langle \hat{x}^2 \rangle & \langle \{\hat{x}, \hat{p}\} \rangle / 2 \\ \langle \{\hat{x}, \hat{p}\} \rangle / 2 & \langle \hat{p}^2 \rangle \end{pmatrix}, \quad (56)$$

$$\mathcal{C} = \begin{pmatrix} \frac{\gamma\hbar}{m\omega}(\bar{n} + 1/2) - \frac{\Re\{f\}^2}{4\lambda m\omega\hbar} & -\frac{\Re\{f\}\Im\{f\}}{4\lambda\hbar} \\ -\frac{\Re\{f\}\Im\{f\}}{4\lambda\hbar} & \gamma m\omega\hbar(\bar{n} + 1/2) + \lambda\hbar^2 + \frac{m\omega\Re\{f\}^2}{4\lambda\hbar} \end{pmatrix}, \quad (57)$$

and  $\mathcal{M}$  is the same matrix as in Eq. (46), but using  $f$  instead of  $\tilde{f}$ .  $S$  has a unique steady-state solution if the real part of the eigenvalues of  $\mathcal{M}$  are negative [9]. We can therefore refer back to Eq. (47) and Fig. 3 to see the stability of the system. In conjunction with Fig. 4 and 5 we can see that the system is stable in the region where we cool the system. Thus, it is not only possible to cool the system, but the system is also stable. We can therefore see that this type of feedback mechanism is a valid way to cool a QHO.

## 4 Discussion and Conclusions

The results presented in this thesis show the possibility of cooling a QHO using a measurement and feedback loop. The discussion analyses the results, comparing to previous work as well as discussing different applications of cooled QHOs. The stability of the system under feedback is also discussed, as well as mentioning other ways to affect the system using feedback.

### 4.1 Cooling of a Quantum Harmonic Oscillator

Technologies relying on quantum phenomena such as quantum computers, generally have a need for a low temperature system [1]. It is therefore an interesting and active research topic to find ways to cool a quantum system. Currently, there exist multiple types of cooling, and the work performed in this thesis is in the form of measurement and feedback based cooling, similar to [4].

As briefly mentioned in Sec. 2.4, the physical realization of the feedback scheme could be a driving laser acting on the system. The successfulness of the cooling would then also depend on the possibilities to physically control the parameters. Especially the feedback strength and phase. It is especially apparent from the results in Fig. 4 that the phase plays a large role in if the feedback is successful, which can be seen from quadrant 2 and 4 having negative energy, which is not physical. The explanation for this is that the system most likely breaks down and, for a steady state to exist given these parameters, the temperature would have to be negative. Thus, the only physically relevant solutions are in quadrant 1 and 3. We can then also conclude that in general a functional feedback scheme has a control Hamiltonian which is either proportional to  $\hat{x} - \hat{p}$  or  $-\hat{x} + \hat{p}$ . It is possible to have a feedback Hamiltonian proportional to  $-\hat{x} - \hat{p}$ , but this will not be as effective at cooling the system. In panel **b** in Fig. 4 this area can be seen for  $\Re\{\tilde{f}\} < 0$ . So even if it is possible to cool in this area, the same amplitude but smaller phase would yield a better result.

The feedback employed in this thesis takes the form of an external force acting on the system, for example a laser interacting with the field inside a cavity, which linearly

changes the system Hamiltonian. A different way to cool the system would be to modify the frequency of the oscillator [10], which has applications in quantum optomechanics. Quantum optomechanics has the ability to use light to prepare macroscopic objects in quantum states [11], allowing for quantum mechanical control over macroscopic systems, which have application in for example LIGO. Changing the frequency could prove more difficult to model since the feedback would be non-linear. It's therefore interesting to ask if the results in this thesis, and the feedback scheme used, could be realized in optomechanical systems.

We only consider the case of infinite detector bandwidth and instantaneous feedback in this thesis. In reality, this idealized picture would not hold, and the effectiveness of the feedback scheme might decrease [3], affecting the efficiency of the cooling. The addition of finite bandwidth is a new addition in the field of quantum feedback control, with [4] being one of the first to consider this.

Increasing the quality factor of the system has a positive impact on the effectiveness of the cooling, allowing smaller feedback parameters to have greater control of the system. A consequence however, is that the system will also be more sensitive to the feedback and measurement. Thus, if the parameters used are prone to noise, the system will vary more than with a lower quality factor.

To get an accurate conclusion about the usefulness of the feedback scheme, it is important to consider the stability of the system in the physical regions. Thus, it is not certain that just because there exist a steady-state solution, we have a stable solution at that point.

## 4.2 Stability with Feedback

The stability of a quantum system is of importance when considering applications based on said system. That is, a stable system can be used and its properties exploited, while an unstable system might behave unpredictably, and cause issues. Specifically, we call a system stable, if in the long time limit it decays to a specific state, and does not diverge from this if affected by a small disturbance, barring any external factors which

may affect the stability. In the case of the system modelled in this thesis it means that the expectation value of the variance in the system decays to a finite value.

We can see from the results in Fig. 3 that the stability of the system is highly dependent on the feedback parameter. It is also important to note that when looking at Fig. 4 only quadrant 1 is in the region of stability of the system. Thus, even if quadrant 3 cools the system, the state will not be stable, and it would be difficult to isolate the system from external fluctuations and keep it in this state. This essentially solidifies the fact that the only relevant feedback parameters are those which are in quadrant 1.

A compelling use-case for feedback could also be stabilizing unstable systems. That is, for a system that is unstable for  $f = 0$ , it could be possible to adjust the feedback parameter such that the system becomes stable. This is something that is common for classical feedback protocols, where you have an unstable mechanical system, for example an aeroplane, and want to stabilize it [12].

### 4.3 Applications

One useful application of QHOs is in the field of quantum computation. A particular application is the use of QHOs in conjunction with qubits to create higher dimensional systems, qudits, to be used for calculations [13]. A problem noted in [13] is that when coupled to a bath, which all physical system are, the qudit will experience noise due to the fluctuations of the bath, which would require quantum error correction to combat [13]. However, with too much noise these protocols could fail, and thus cooling the system may help by reducing the noise.

Another interesting use for QHOs is as thermal baths themselves inside quantum circuits as described by [14]. In order to realize this thermalization an ensemble of QHOs would be needed with variable energies [14]. The results in this thesis show that a possible protocol to control the temperature of a QHO which therefore would allow QHOs to be used in this way. Quantum circuits themselves are interesting since they are a way to physically realize a quantum computer [1].



## 5 Outlook

The field of measurement and feedback based cooling seems to be a promising way to cool quantum systems. In this thesis a QHO has been considered, but it is not the only system that in theory could be cooled in this way. However, it has served as a good proof of concept, and has application that are relevant. As mentioned in the discussion, implementing detectors with finite bandwidth is an important consideration for future research as this would more accurately model a real system.

Furthermore, this thesis only consider one type of feedback scheme, looking at the feedback Hamiltonian as a linear modification of the position and momentum quadratures. There is promising work to be done either considering modifying the position of the potential akin to [4], or modifying the frequency of the oscillator similar to [10]. The latter would be interesting as it seems to have a more direct application in quantum optomechanics.

This thesis has also mainly looked at feedback with the purpose of cooling the system, with the stability secondary. For further research it would be intriguing to look at the stability as the main point of interest. It could also be interesting to look at feedback as a way to prepare different types of states which can later be used for other experiments, for example squeezed states.

## References

- [1] M. A. Nielsen and I. L. Chuang, *Quantum Computation and Quantum Information: 10th Anniversary Edition*. Cambridge: Cambridge University Press, 2010.
- [2] A. N. Jordan and I. A. Siddiqi, *Quantum Measurement: Theory and Practice*. Cambridge: Cambridge University Press, 2024.
- [3] B. Annby-Andersson, *Continuous measurements of small systems: Feedback control, thermodynamics, entanglement*. Doctoral thesis (compilation), Lund University, 4 2024.
- [4] G. De Sousa, P. Bakhshinezhad, B. Annby-Andersson, P. Samuelsson, P. P. Potts, and C. Jarzynski, “Continuous feedback protocols for cooling and trapping a quantum harmonic oscillator,” *Physical Review E*, vol. 111, no. 1, p. 014152, 2025.
- [5] P. Meystre, *Quantum Optics: Taming the Quantum*. Cham: Springer, 2021.
- [6] H.-P. Breuer and F. Petruccione, *The Theory of Open Quantum Systems*. Oxford: Oxford University Press, 2007.
- [7] H. M. Wiseman and G. J. Milburn, *Quantum Measurement and Control*. Cambridge: Cambridge University Press, 2009.
- [8] R. Simon, N. Mukunda, and B. Dutta, “Quantum-noise matrix for multimode systems: U(n) invariance, squeezing, and normal forms,” *Physical Review A*, vol. 49, no. 3, pp. 1567–1583, 1994.
- [9] A. Purkayastha, “Lyapunov equation in open quantum systems and non-hermitian physics,” *Physical Review A*, vol. 105, no. 6, p. 062204, 2022.
- [10] H. Habibi, E. Zeuthen, M. Ghanaatshoar, and K. Hammerer, “Quantum feedback cooling of a mechanical oscillator using variational measurements: tweaking Heisenberg’s microscope,” *Journal of Optics*, vol. 18, no. 8, p. 084004, 2016.
- [11] Y. Chen, “Macroscopic quantum mechanics: theory and experimental concepts of optomechanics,” *Journal of Physics B: Atomic, Molecular and Optical Physics*, vol. 46, no. 10, p. 104001, 2013.
- [12] J. Bechhoefer, “Feedback for physicists: A tutorial essay on control,” *Reviews of Modern Physics*, vol. 77, no. 3, pp. 783–836, 2005.
- [13] Y. Liu, J. Sinanan-Singh, M. T. Kearney, G. Mintzer, and I. L. Chuang, “Constructing qudits from infinite-dimensional oscillators by coupling to qubits,” *Physical Review A*, vol. 104, no. 3, p. 032605, 2021.
- [14] J. P. Pekola and B. Karimi, “Heat bath in a quantum circuit,” *Entropy*, vol. 26, no. 5, 2024.