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HOMWORK-5

COURSE :- AE-686A (Helicopter Theory)

### Question-01

#### \* Steps of calculation :-

- Blade rotor plane is divided in 360, and the forces and moments are calculated at each location of azimuth ( $\psi$ ) with difference of  $1^\circ$ .
- To calculate forces & Moments, blade is divided into 20 parts, and the gaussian numerical integration is done using gaussian quadrature.
- $\beta, \beta^*, \beta^{**}$  values are obtained from Newmark's algorithm (Homework-04).
- Uniform inflow is considered.

#### # Formulas to calculate loads :

##### \* Velocities in non-dimensional form

$$u_T = 1 + \mu \sin \psi$$

$$u_P = \lambda \cos \beta + a \beta^* + \mu \cos \psi \sin \beta$$

$$u_R = \mu \cos \psi$$

##### \* Velocities in dimensional form

$$U_T = \Omega r + \mu \Omega R \sin \psi$$

$$U_P = \lambda \Omega R \cos \beta + \gamma \beta^* + \mu \Omega R \cos \psi \sin \beta$$

$$U_R = \mu \Omega R \cos \psi$$

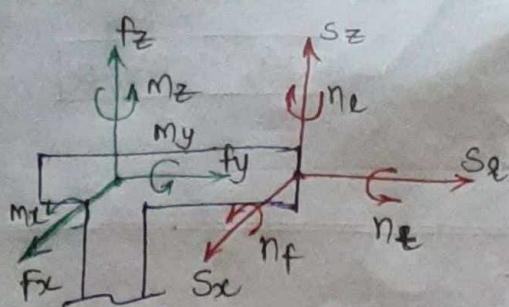
##### \* Forces on section $dr$ :

$$dF_z = \frac{1}{2} \rho C c_{\alpha} \cdot U_T^2 \left( 1 - \frac{U_P}{U_T} \right) \cdot dr$$

$$dF_x = \frac{1}{2} \rho C c_{\alpha} \cdot (U_P U_T \theta - U_P^2 + \frac{C_d}{C_{\alpha}} \cdot U_T^2) dr$$

$$dF_y = \frac{1}{2} \rho C c_{\alpha} \cdot \beta (U_T^2 \theta - U_P U_T) \cdot dr$$





\* Root shear loads (Rotating frame)

$$S_x = \int_e^R \left[ \frac{dF_x}{dr} \right] dr$$

$$S_z = \int_e^R \left[ \frac{dF_z}{dr} - m(r-e) \ddot{\beta} \right] dr$$

$$S_y = \int_e^R \left[ \frac{dF_y}{dr} - \beta \cdot \frac{dF_z}{dr} \right] dr$$

\* Hub loads & Moments (Rotating frame)

$$F_x = S_x$$

$$F_y = S_y$$

$$F_z = S_z$$

$$M_x = n_t + e S_z$$

$$M_y = n_t$$

$$M_z = -n_x - e S_x$$

\* Root bending loads (Rotating frame)

$$n_t = \int_e^R (r-e) dF_z - \int_e^R (m dr) \ddot{\beta} (r-e) \beta - \int_e^R m(r-e) \ddot{\beta} dr$$

$$= K_\beta (\beta - \beta_p) = \left( \frac{1}{2} \frac{R^2}{\rho} - 1 - \frac{3}{2} \frac{e}{R} \right) \frac{1}{\rho} R^2 (\beta - \beta_p)$$

$$n_x = \int_e^R \left[ (r-e) \cdot \frac{dF_x}{dr} \right] dr$$

$$n_t = \int_e^R \left[ \frac{dM_x}{dr} \right] dr$$

- Here,  $dM_x$  is about elastic axis, which is considered at quarter-chord.

Hence,  $n_t = 0$  (twisting moment)

$$M_x = 0$$

\* Hub loads & Moments (Fixed frame)

Rotor Drag :  $H = F_x = \sum_{m=1}^{N_b} \left[ f_y^{(m)} \cos(\psi_m) + f_x^{(m)} \sin(\psi_m) \right]$

Rotor side Force :  $Y = F_y = \sum_{m=1}^{N_b} \left[ f_y^{(m)} \sin(\psi_m) - f_x^{(m)} \cos(\psi_m) \right]$

Thrust :  $T = F_z = \sum_{m=1}^{N_b} \left[ f_z \right]$

Rotor roll Moment :  $M_x = \sum_{m=1}^{N_b} \left[ m_x \sin(\psi_m) + m_y \cos(\psi_m) \right]$

Rotor pitch Moment :  $M_y = \sum_{m=1}^{N_b} \left[ -m_x \cos(\psi_m) + m_y \sin(\psi_m) \right]$

Rotor Torque :  $\phi = \sum_{m=1}^{N_b} \left[ m_z \right]$

$$\rightarrow \dot{\beta} = \Omega \beta^*$$

$$\ddot{\beta} = \Omega^2 \beta^{**}$$

$$\rightarrow \beta(\psi) = \beta_0 + \beta_{tw} \cdot \psi + \beta_{1s} \cdot \sin \psi + \beta_{1c} \cdot \cos \psi$$

$$\rightarrow \beta(\psi) = \beta_0 + \beta_{1c} \cdot \cos \psi + \beta_{1s} \cdot \sin \psi$$



## # OBSERVATIONS :-

### (1) Plot 1 (Shear forces at hub - Rotating Frame)

- $f_z$  (force in z-direction) will be dependent on periodic flap dynamics. (The force varies highly over one revolution)
- $f_z$  depends upon lift, lift depend on azimuth ( $\psi$ ) location because it has different relative velocity and hence different forces.
- $f_y$  (force in y-direction) is very high because of presence of centrifugal term ( $m\omega^2 r$ ) in sq. ~~form~~ integration.
- $f_x$  (force in x-direction) is minimum, because drag is very small force compared to other lift force.

### (2) Plot 2 (Bending moments at hub - Rotating Frame)

- $M_y$  - Moment w.r.t. y axis of blade is zero, because elastic axis coincide with quarter chord axis.  
(Hence,  $M_y = M_t = 0$ ).

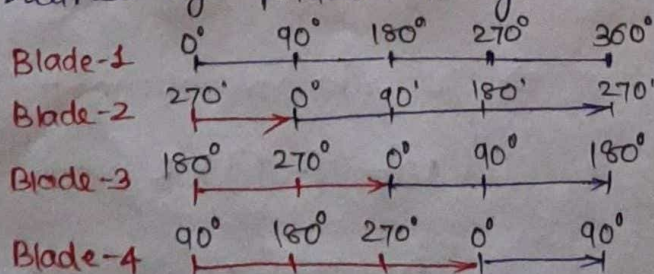
### (3) Plot-3 (Non-dimensional vertical shear forces - Rotating Frame)

- All blades produce same magnitude of shear-force in z-direction, however they are shifted by  $90^\circ$  (4-bladed rotor).

### (4) Plot 4, 5 (Forces & Moments at hub - Fixed Frame)

- $F_z(\psi) = 4a_0 + 4a_4 \cdot \sin 4\psi$
- Hub only transfers  $p \cdot N_b$  /rev frequency. (Here,  $p=4$ ).  
hence, it acts as a filter which removes all other frequencies except  $p \cdot N_b$  /rev <sup>hub</sup>
- Here, fixed frame forces & Moments have 4 /rev frequency.

⇒ The data for 1 blade is calculated and other blade data is obtained by phase shifting.





## [Question-02]

# Steps for calculation :-

⇒ Residual equations

(1) Vertical Force equilibrium :

$$R_1 = W - T \cdot \cos(\alpha_s) \cdot \cos(\phi_s) + Y \cdot \sin(\phi_s) - H \cdot \sin(\alpha_s) \cdot \cos(\phi_s) + Y_F \cos(\alpha_s) \cdot \sin(\phi_s) + D \cdot \sin(\theta_{fp}) = 0$$

(2) Longitudinal equilibrium :

$$R_2 = D \cos(\theta_{fp}) + H \cdot \cos(\alpha_s) - T \cdot \sin(\alpha_s) = 0$$

(3) Lateral force equilibrium :

$$R_3 = (Y + Y_F) \cdot \cos(\phi_s) + T \cdot \cos(\alpha_s) \cdot \sin(\phi_s) + H \cdot \sin(\alpha_s) \cdot \sin(\phi_s) = 0$$

(4) Pitch Moment equilibrium :

$$R_4 = M_y + M_{y_F} - W(x_{cg} \cos \alpha_s - h \sin \phi_s) - D(x_{cg} \sin \alpha_s + h \cos \alpha_s) = 0$$

(5) Roll Moment equilibrium :

$$R_5 = M_x + M_{x_F} + Y_F \cdot h + W(h \cdot \sin \phi_s - Y_{cg} \cos \phi_s) = 0$$

(6) Yaw moment / Torque equilibrium

$$R_6 = Q - Y_F l_T = 0 \quad (\text{we have not considered this, we assumed this is satisfied}).$$

(7) Inflow equation

$$R_7 = \lambda - M \cdot \tan \alpha - \frac{C_T}{2 \sqrt{M^2 + \lambda^2}} = 0$$

⇒ Jacobian

$$J = \begin{bmatrix} \frac{\partial R_1}{\partial \theta_1} & \frac{\partial R_1}{\partial \theta_2} & \dots & \frac{\partial R_1}{\partial \theta_6} \\ \frac{\partial R_2}{\partial \theta_1} & & & \\ \vdots & & & \\ \frac{\partial R_6}{\partial \theta_1} & \frac{\partial R_6}{\partial \theta_2} & \dots & \frac{\partial R_6}{\partial \theta_6} \end{bmatrix}$$

⇒ Finite difference

i<sup>th</sup> residue, j<sup>th</sup> input

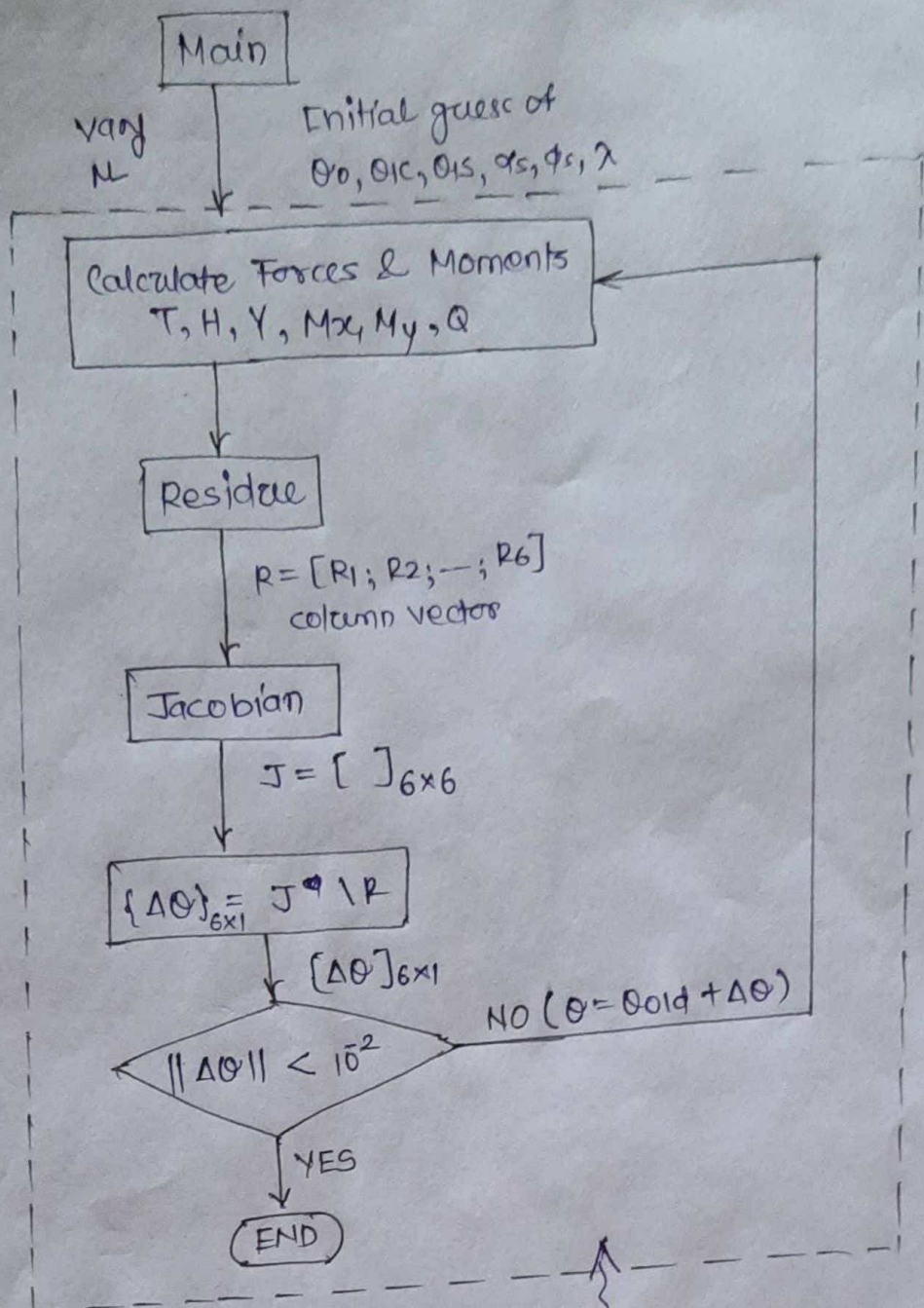
$$\frac{\partial R_i}{\partial \theta_j} = \frac{R_i(\theta_j + \Delta \theta_j) - R_i(\theta_j)}{\Delta \theta_j}$$

⇒ Change in control input

$$\{\Delta \theta\} = [-J] \setminus R$$

$$\theta = \theta_0 + \Delta \theta$$





This block will repeat several times for each value of  $N$ .



## # OBSERVATIONS :-

(1) Plot 1 (Control inputs:  $\theta_0, \theta_{1c}, \theta_{1s}$  v/s  $u$ )

$\theta_0$  (collective pitch ~~control~~)  $\rightarrow$  The RPM of rotor blade is ~~is~~ almost constant.

- To increase forward speed, the thrust vector needs to be tilted which requires more lift/thrust generation to support weight as well as forward speed component. which is achieved by changing cyclic variation ( $\theta_{1s}$ ) in longitudinal direction.
- Remains constant with  $u$ .

$\theta_{1s}$  (Longitudinal cyclic)  $\Rightarrow$   $\theta_{1s}$  decreases with increase in  $u$ , to balance out high pitch moments generated because of shaft tilting.

- It stabilizes forces & moments in forward flight.

$\theta_{1c}$  (lateral cyclic)  $\rightarrow$  Remains almost constant with  $u$ .

- There is no lateral movement at the time of trim. (No sideslip velocity).

(2) Plot 2 (shaft angles:  $\alpha_s, \phi_s$  v/s  $u$ )

$\alpha_s$  (Longitudinal shaft angle)  $\rightarrow$   $\alpha_s$  increases with increase in  $u$  (ie Advance ratio - Nondimensional forward speed)

- This is because forward speed is obtained by tilting the thrust vector in forward direction.

$\phi_s$  (Lateral shaft angle)  $\rightarrow$   $\phi_s$  does not change with  $u$ .

- There is no sideward movements ~~or~~ considered or no sideslip in velocity is considered. hence,  $\phi_s$  remains constant.