

# Video Game Physics

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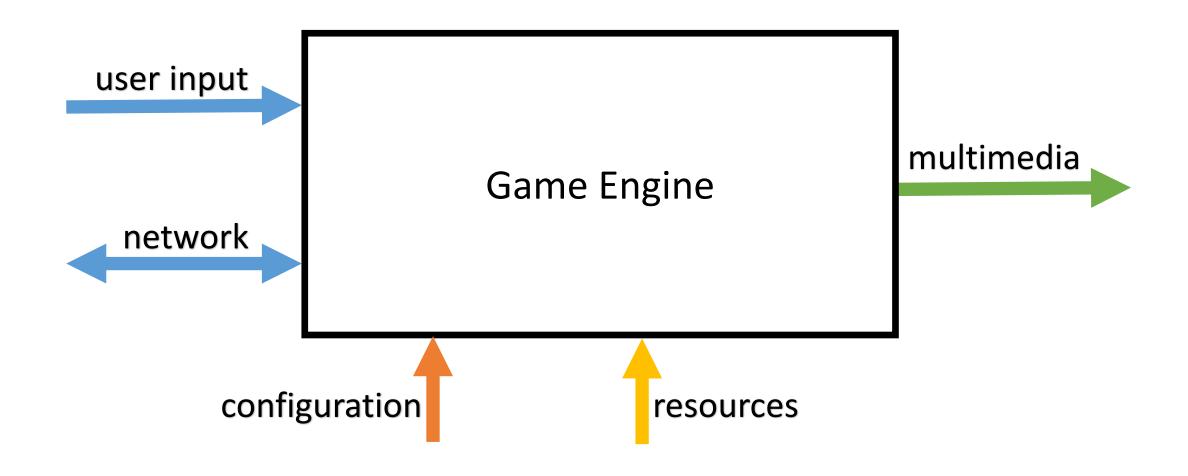


### References

- Game Physics Second Edition, David Eberly
- Erin Catto (<a href="https://box2d.org/publications/">https://box2d.org/publications/</a>)

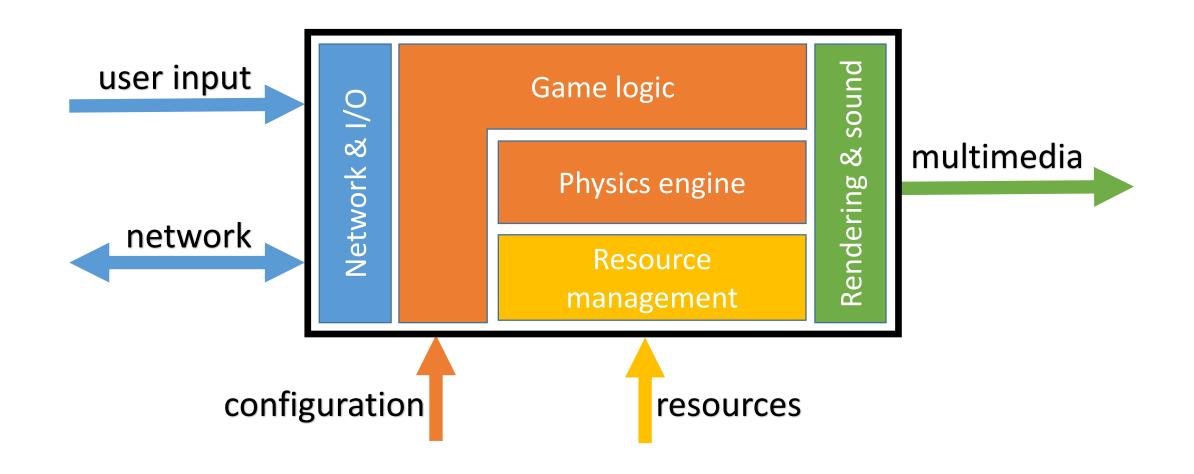


## Game Engine





### Game Engine





## Physics Engine

#### Fields of study:

- 1. Physics (classical mechanics)
- 2. Control theory
- 3. Numerical algorithms
- 4. Graphics
- 5. Mathematical optimization



## Physics Engine

#### 1. Model

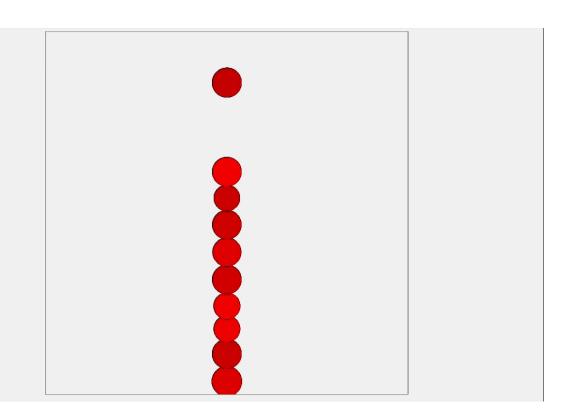
- I. Space model
- II. Physics body model

#### 2. Kinematics

- 1. Newton's laws of motion
- 2. Numerical integration
- 3. External forces

#### 3. Constrained motion:

- I. Collision detection
- II. Constraint modeling
- III. Constraint solving

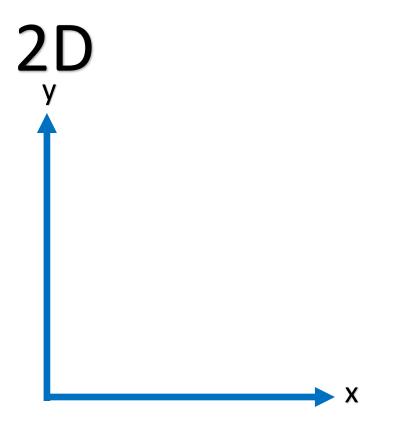


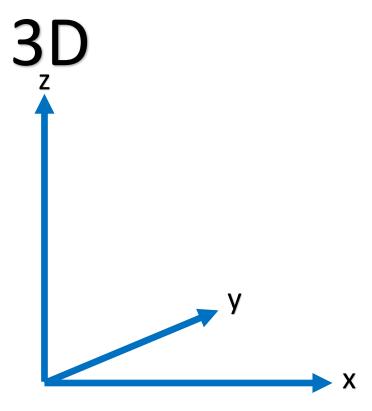


The way to describe the state of a physical system in any given moment?



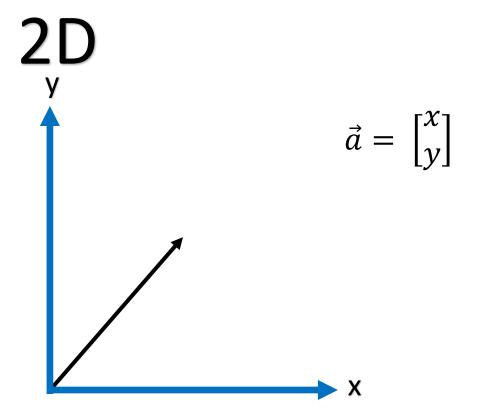
1. Space model

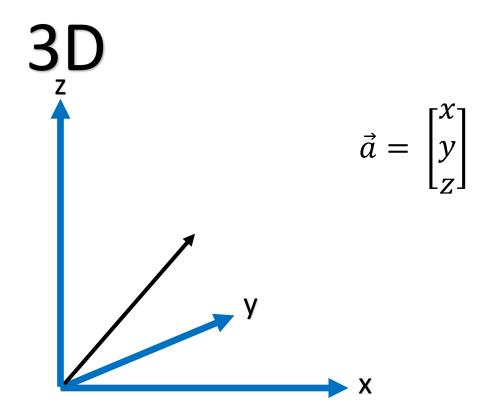






- 1. Space model
  - Vectors





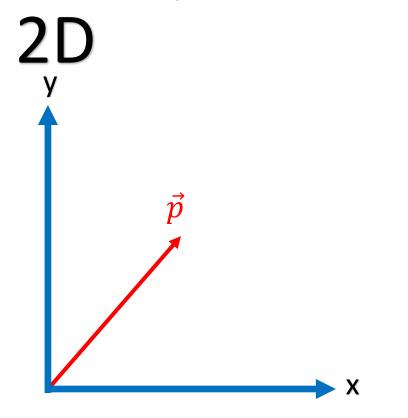


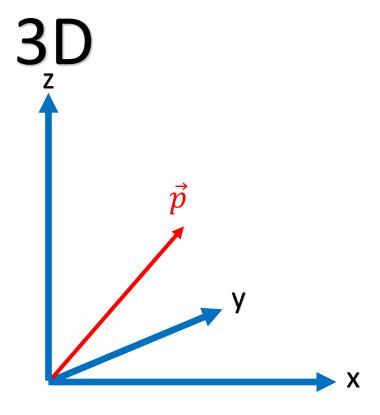
#### 1. Space model

- Vectors
- + vectors generalize the problem (same solutions apply to 2D and 3D)
- + vectors simplify the equations (operations are written simultaneously across all dimensions of space)
- + vectors faster execution (operations are performed simultaneously across all dimensions of space)
- + vectors are suitable for modern hardware
- + most graphics APIs provide implementations of vector algebra
- vectors require familiarity with vector algebra (takes some time to get used to it)



- 2. Physics body model
  - I. position

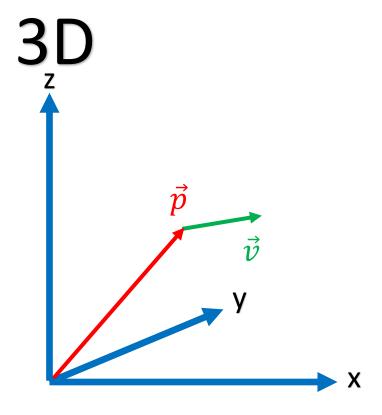






- 2. Physics body model
  - I. position

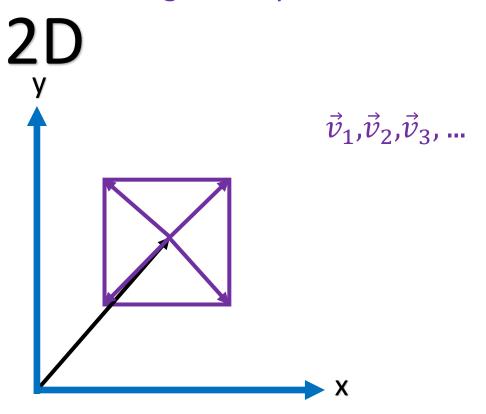
2D II. current velocity  $\vec{p}$   $\vec{v}$ 

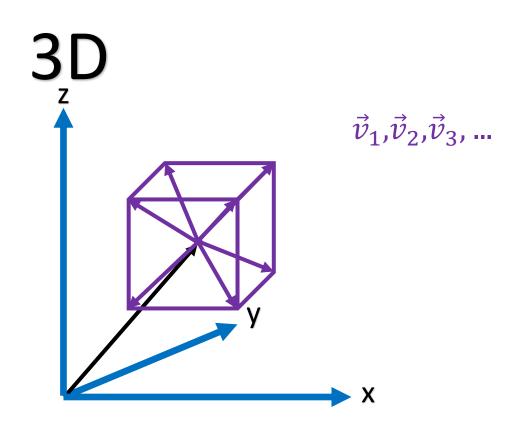




### 2. Physics body model

IV. geometry

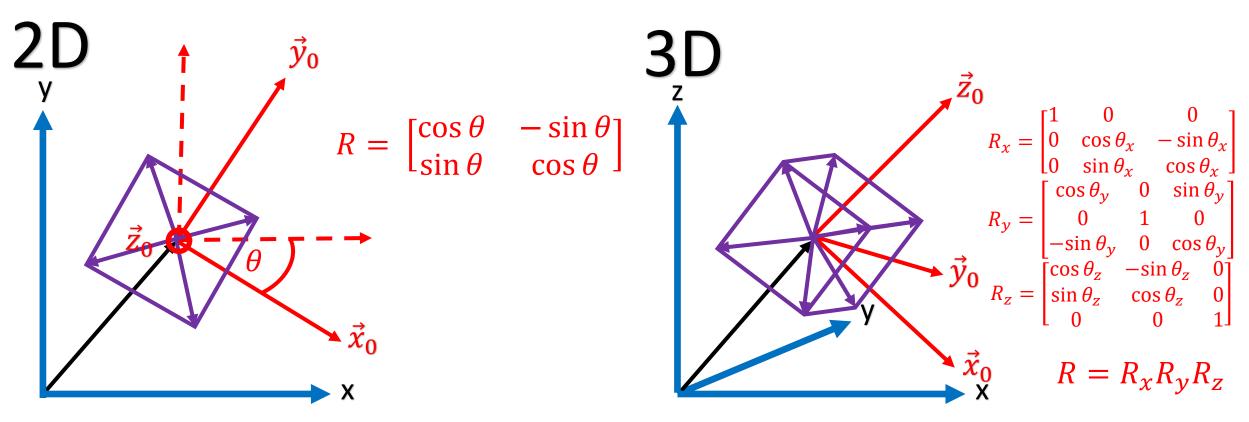






#### 2. Physics body model

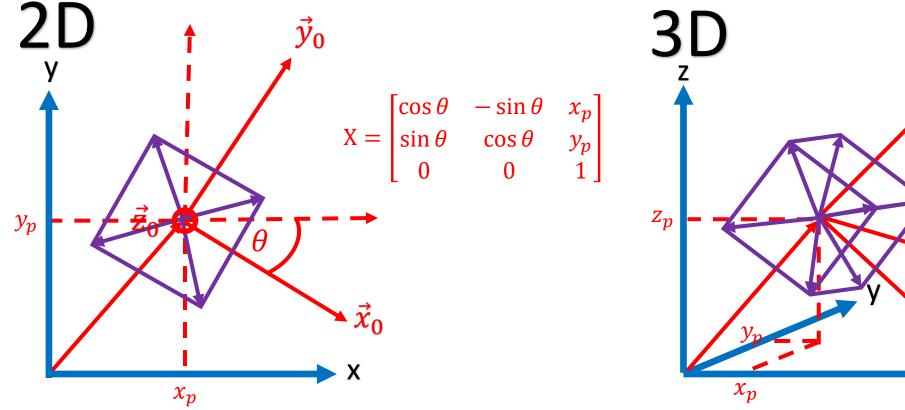
V. orientation

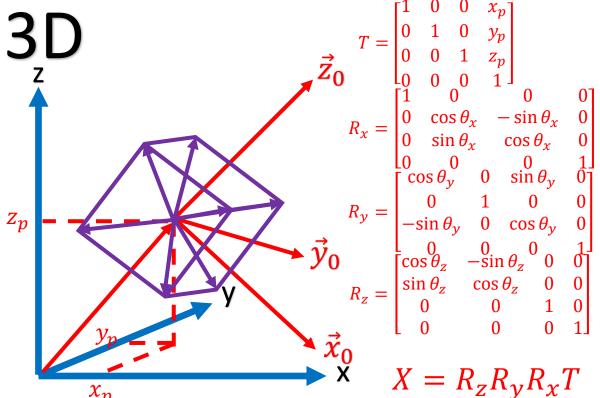




#### 2. Physics body model

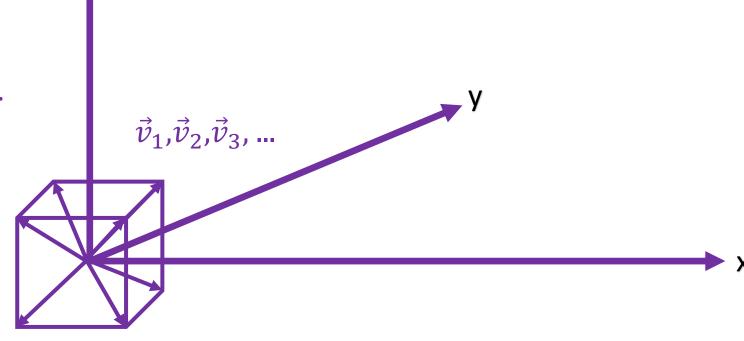
• transformation matrix: position + orientation (homogeneous coordinates)





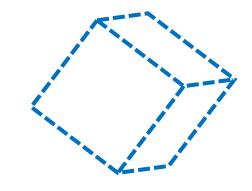


- 2. Physics body model
  - transformation matrix
  - I. local coordinate.system





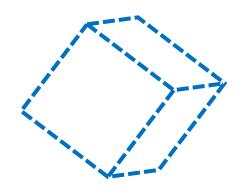
- 2. Physics body model
  - transformation matrix
  - local coordinate system
  - II. global coordinate system







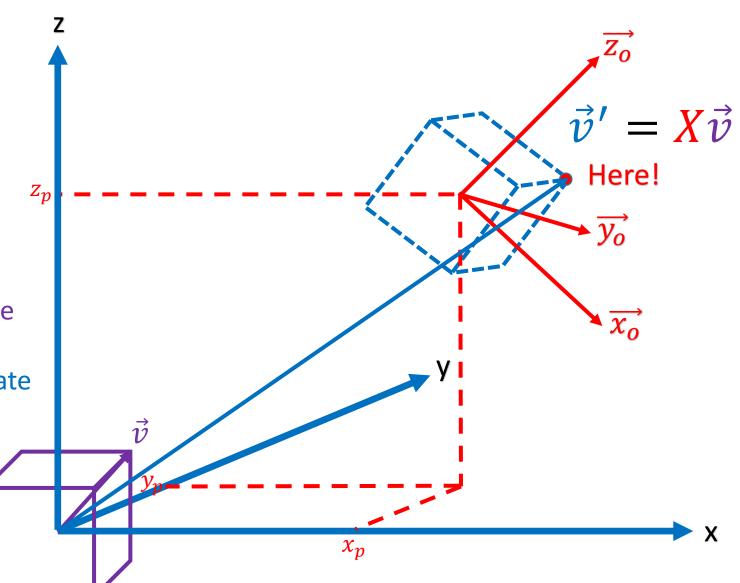
- Physics body model
  - transformation matrix
  - local coordinate system
  - II. global coordinate system



Where is this position found in global coordinate system?



- 2. Physics body model
  - transformation matrix
  - I. local coordinate system
  - II. global coordinate system



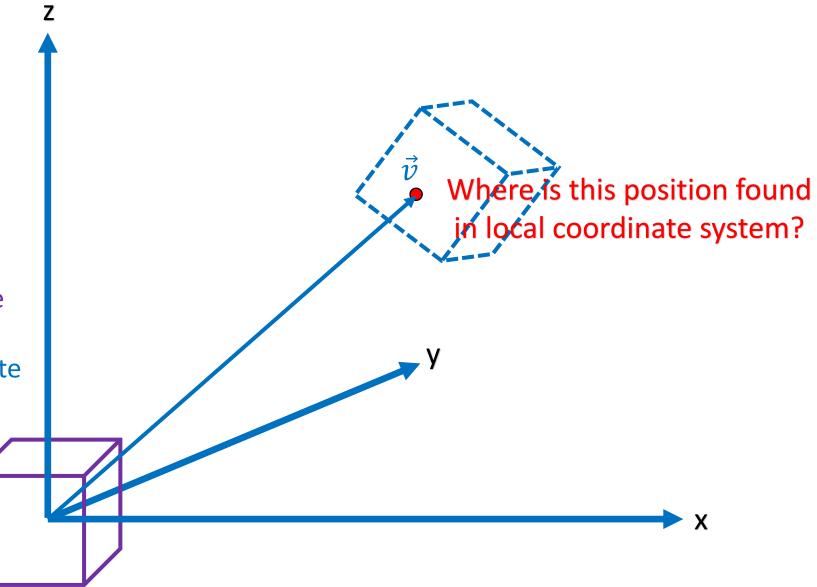


2. Physics body model

• transformation matrix

 local coordinate system

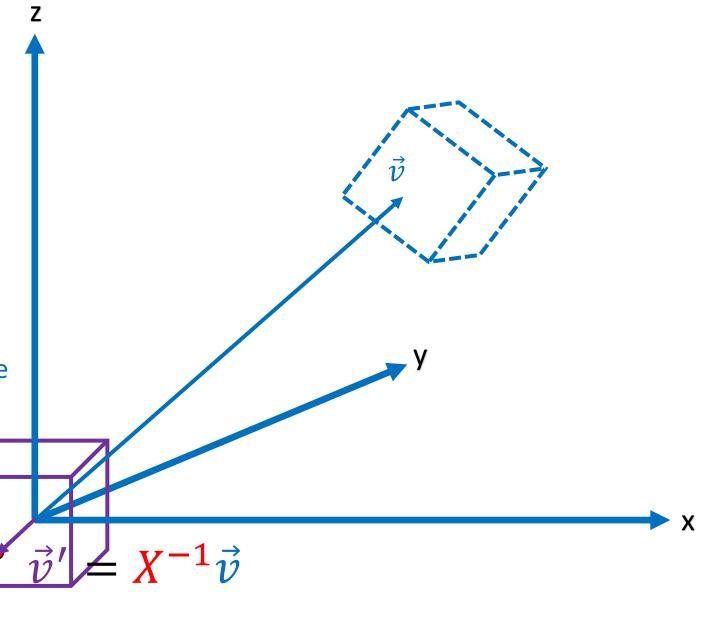
II. global coordinate system





- 2. Physics body model
  - (inverse) transformation matrix
  - I. local coordinate system
  - II. global coordinate system

Here!





#### 2. Physics body model

- Transformation matrix
- + body geometry is precomputed in local coordinate system and remains constant
- + as the body moves, only it's position and orientation are updated, but not the entire geometry (much faster)
- + graphics APIs use precomputed body geometry and transformation matrices to render the result, thus the graphics hardware does all the work
- + most graphics APIs provide transformation matrices implementation
- in case of calculations which require interaction of multiple bodies (e.g. collision detection), geometries of both bodies must be transformed to the same coordinate system (global coordinate system or local coordinate system of either of those bodies); these calculations are usually done on CPU, but are much less frequent than rendering

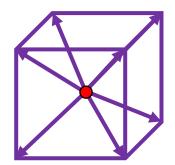


### 2. Physics body model

• geometric center (centroid)

$$\vec{\mathcal{C}}_g = \frac{\sum_{i=1}^n \vec{v}_i}{n}$$
 
$$\vec{v}_1, \vec{v}_2, \vec{v}_3, \dots$$

$$\vec{v}_1, \vec{v}_2, \vec{v}_3, \dots$$





#### 2. Physics body model

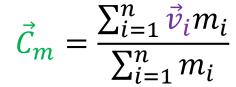
- geometric center (centroid)
- center of mass

$$\vec{C}_g = \frac{\sum_{i=1}^n \vec{v}_i}{n}$$

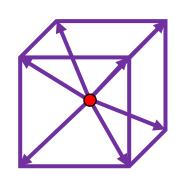
$$\vec{v}_1, \vec{v}_2, \vec{v}_3, \dots$$

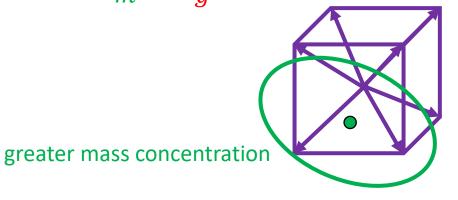
for homogenous bodies:

$$\vec{C}_m = \vec{C}_g$$



$$\vec{v}_1, \vec{v}_2, \vec{v}_3, \dots \\ m_1, m_2, m_3 \dots$$

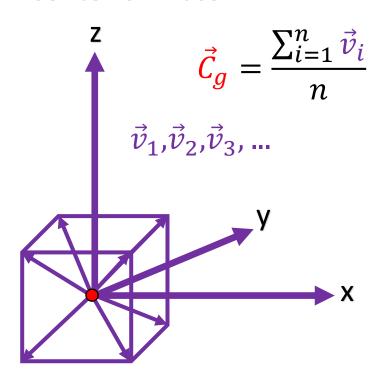




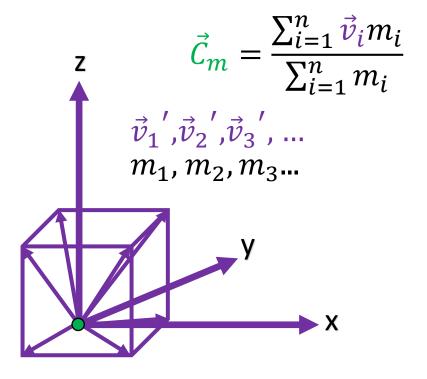


#### 2. Physics body model

- geometric center (centroid)
- center of mass



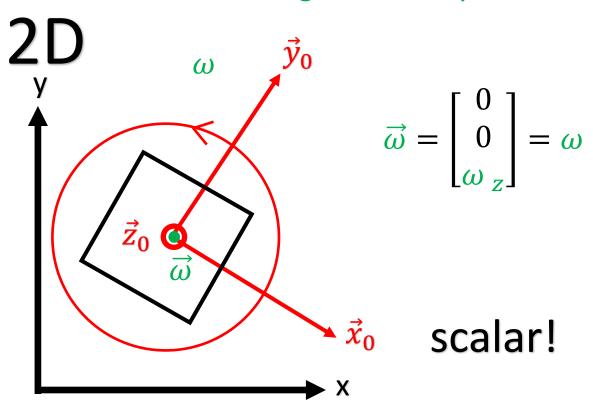
Origin of a local coordinate system should always be precomputed against  $\vec{C}_g$  (or  $\vec{C}_m$  if exists) so that it becomes the pivot point for all the transformations!

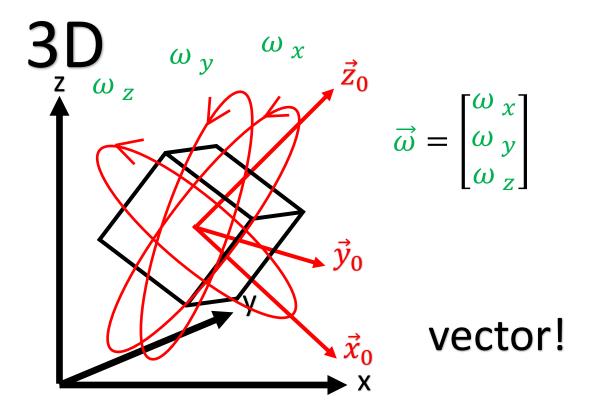




#### 2. Physics body model

VI. current angular velocity

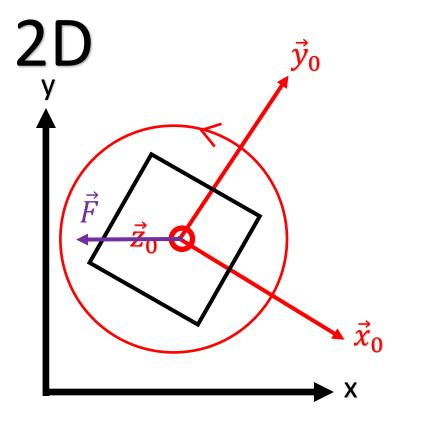


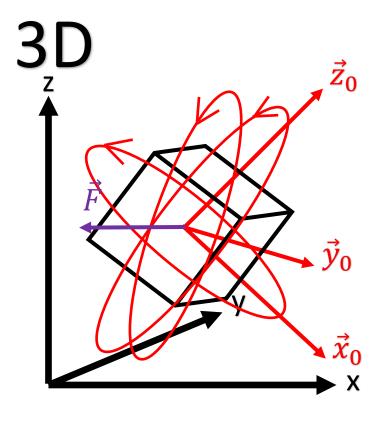




### 2. Physics body model

VIII. current total linear force

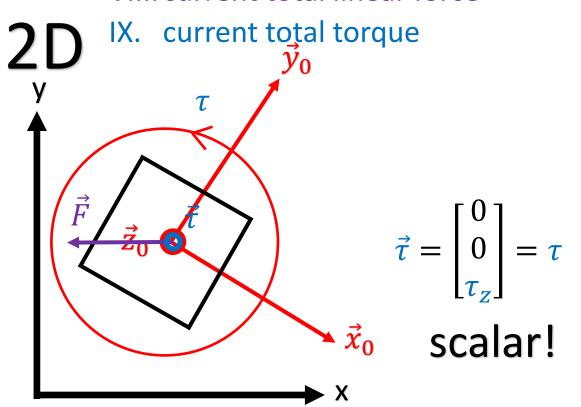


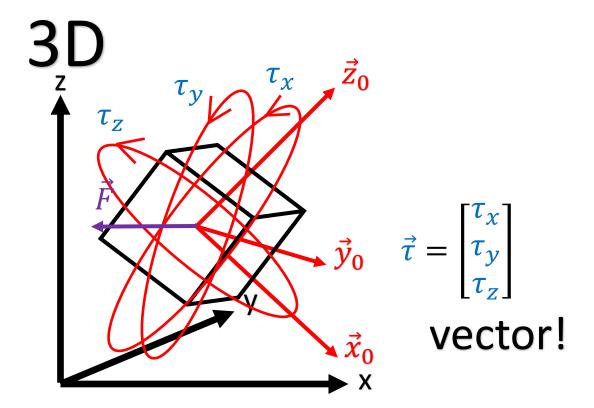




#### 2. Physics body model

VIII. current total linear force







2. Physics body model

X. mass

Linear motion	Rotational motion
2D/3D	
mass	
m	

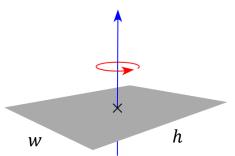


#### 2. Physics body model

X. mass

XI. moment of inertia

Moments of inertia in the table are given for rotation around  $\vec{\mathcal{C}}_g$ 



Linear motion	R		
2D/3D	2D		
mass	moment of inertia (scalar)		
	circle: $I = \frac{mr^2}{2}$		
m	rectangle: $I = \frac{m}{12}(w^2 + h^2)$		
	:		

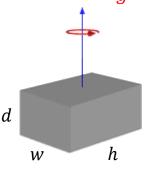


# 2. Physics body model

X. mass

XI. tensor of inertia

Tensors of inertia in the table are given for rotation around  $\vec{C}_g$ 



Linear motion	Rotational motion					
2D/3D	2D	3D				
mass	moment of inertia (scalar)	tensor of inertia (matrix)				
	circle: $I = \frac{mr^2}{2}$	sphere: $I = \begin{bmatrix} \frac{2}{5}mr^2 & 0 & 0\\ 0 & \frac{2}{5}mr^2 & 0\\ 0 & 0 & \frac{2}{5}mr^2 \end{bmatrix}$				
m	rectangle: $I = \frac{m}{12}(w^2 + h^2)$	prism: $I = \begin{bmatrix} \frac{m}{12}(h^2 + d^2) & 0 & 0 \\ 0 & \frac{m}{12}(w^2 + d^2) & 0 \\ 0 & 0 & \frac{m}{12}(w^2 + h^2) \end{bmatrix}$				



2. Physics body model

Linear motion			Rotational motion							
2D/3D			2D		3D					
geometry:	$\vec{v}_1, \vec{v}_2, \vec{v}_3, \dots$	[m]	<b>{</b>	geo	metry:	$\vec{v}_1, \vec{v}_2, \vec{v}_3, \dots$	[m]			
position:	$ec{p}$	[m]	angle:	θ	[rad]	orientation:	$\overrightarrow{\theta} = \begin{bmatrix} \theta_x \\ \theta_y \\ \theta_z \end{bmatrix}$			[rad]
velocity:	$\vec{v}$	$\left[\frac{m}{s}\right]$	angular velocity:	ω	$\left[\frac{rad}{s}\right]$	angular velocity:	$\overrightarrow{\omega} = \begin{bmatrix} \omega_x \\ \omega_y \\ \omega_z \end{bmatrix}$			$\left[\frac{rad}{s}\right]$
force:	$ec{F}$	[ <i>N</i> ]	torque:	τ	[Nm]	torque:	$\vec{\tau} = \begin{bmatrix} \tau_x \\ \tau_y \\ \tau_z \end{bmatrix}$			[ <i>Nm</i> ]
mass:	m	[kg]	moment of inertia:	Ι	$\left[\frac{kg}{m^2}\right]$	tensor of inertia:	$I = \begin{bmatrix} I_{xx} \\ I_{yx} \\ I_{zx} \end{bmatrix}$	$I_{xy}$ $I_{yy}$ $I_{zy}$	$\begin{bmatrix} I_{xz} \\ I_{yz} \\ I_{zz} \end{bmatrix}$	$\left[\frac{kg}{m^2}\right]$



### Kinematics

How to update the state of a physical system in discrete time steps?



### Kinematics

#### Given initial conditions:

Linear motion	Rotational motion					
2D/3D	2D	3D				
initial position: $ec{p}_0$	initial angle: $ heta_0$	initial orientation: $ec{ heta}_0$				
initial velocity $ec{v}_{0}$	initial angular velocity $\omega_0$	initial angular velocity: $ ec{\omega}_0 $				

#### Find:

Linear motion	Rotational motion			
2D/3D	2D	3D		
position: $\vec{p}(t) = ?$	angle: $\theta(t) = ?$	orientation: $\overrightarrow{\theta}(t) = ?$		



### Kinematics

• exact solution (assuming constant  $\vec{F}$ ):

$$\vec{p}(t) = \vec{p}_0 + \vec{v}_0 t + \frac{1}{2} \vec{a} t^2$$

$$\vec{p}(t) = \vec{p}_0 + \vec{v}_0 t + \frac{1}{2} \frac{\vec{F}}{m} t^2$$

- this form can't describe complex motion
- this form can't describe interactive motion (affected by user interaction)

Back to square #1! Where does this solution come from?

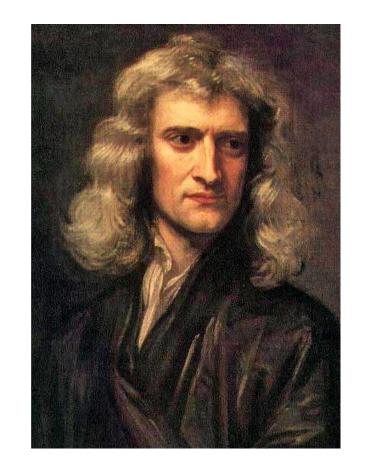


• 2nd Newton's Law of Motion:

$$\vec{F} = m\vec{a}$$

$$\vec{a} = \frac{\vec{F}}{m}$$

$$\frac{d^2\vec{p}(t)}{dt^2} = \frac{\vec{F}}{m}$$

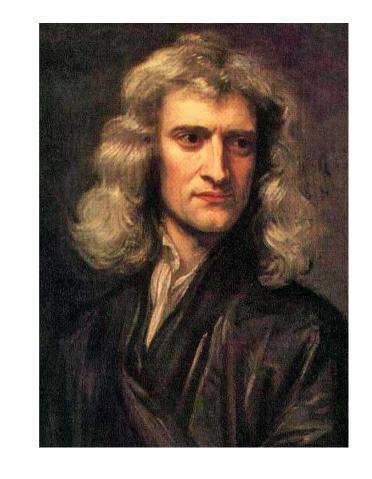


2nd order differential equation!



• 2nd Newton's Law of Motion:

$$\frac{d^2\vec{p}(t)}{dt^2} = \frac{\vec{F}(t)}{m}$$



What is the solution assuming variable  $\vec{F}(t)$ ? It depends on  $\vec{F}(t)$ , and it further depends on user interaction!



At any given moment in time  $(t + \Delta t)$  a function can be expanded into Taylor series:

$$f(t + \Delta t) = f(t) + \Delta t \frac{df(t)}{dt} + \frac{\Delta t^2}{2} \frac{d^2 f(t)}{dt^2} + \dots + \frac{\Delta t^n}{n!} \frac{d^n f(t)}{dt^n}$$

This form is suitable for finding an approximate numerical solution!



Euler's Method

$$f(t + \Delta t) = f(t) + \Delta t \frac{df(t)}{dt} + \frac{\Delta t^2}{2} \frac{d^2 f(t)}{dt^2} + \dots + \frac{\Delta t^n}{n!} \frac{d^n f(t)}{dt^n}$$

$$f(t + \Delta t) \approx f(t) + \Delta t \frac{df(t)}{dt}$$



• Euler's Method

Position function:

$$\vec{p}(t + \Delta t) = \vec{p}(t) + \Delta t \frac{d\vec{p}(t)}{dt}$$



Euler's Method

Differentiate both sides once:

$$\vec{p}(t + \Delta t) = \vec{p}(t) + \Delta t \frac{d\vec{p}(t)}{dt}$$

$$\frac{d\vec{p}(t+\Delta t)}{dt} = \frac{d\vec{p}(t)}{dt} + \Delta t \frac{d^2\vec{p}(t)}{dt^2}$$



Euler's Method

substitution: 
$$\frac{d\vec{p}(t)}{dt} = \vec{v}(t)$$

$$\vec{p}(t + \Delta t) = \vec{p}(t) + \Delta t \vec{v}(t)$$

$$\vec{v}(t + \Delta t) = \vec{v}(t) + \Delta t \frac{\vec{F}(t)}{m}$$



• Euler's Method

Iterative form:

$$\vec{p}_i = \vec{p}_{i-1} + \Delta t \vec{v}_{i-1}$$

$$\vec{v}_i = \vec{v}_{i-1} + \Delta t \frac{\vec{F}_{i-1}}{m}$$



• Euler's Method

$$\vec{p}_i = \vec{p}_{i-1} + \Delta t \vec{v}_{i-1}$$

$$\vec{v}_i = \vec{v}_{i-1} + \Delta t \frac{\vec{F}_{i-1}}{m}$$

- Plain Euler's method uses old velocity for finding new position.



Semi-implicit Euler's Method

$$\vec{v}_i = \vec{v}_{i-1} + \Delta t \frac{\vec{F}_{i-1}}{m}$$

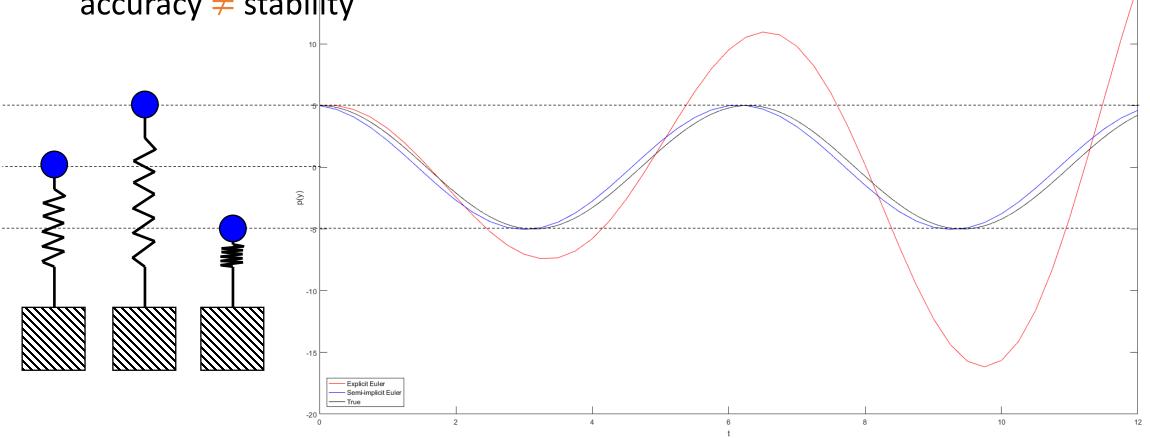
$$\vec{p}_i = \vec{p}_{i-1} + \Delta t \vec{v}_i$$

- + Calculates new velocity first, and then uses it to find new position.
- + Offers improved method stability without additional calculations!











0	
$ec{v}_0$	
$ec{p}_0$	

0
$ec{\omega}_0$
$ec{ heta}_0$



0	$0 + \Delta t$	
$ec{v}_0$	$\vec{v}_1 = \vec{v}_0 + \Delta t \frac{\vec{F}_0}{m}$	
$ec{p}_0$	$\vec{p}_1 = \vec{p}_0 + \Delta t \vec{v}_1$	

0	$0 + \Delta t$	
$ec{\omega}_0$	$\vec{\omega}_1 = \vec{\omega}_0 + \Delta t \frac{\vec{\tau}_0}{I}$	
$ec{ heta}_0$	$\vec{\theta}_1 = \vec{\theta}_0 + \Delta t \vec{\omega}_1$	



0	$0 + \Delta t$	$0 + 2\Delta t$
$ec{v}_0$	$\vec{v}_1 = \vec{v}_0 + \Delta t \frac{\vec{F}_0}{m}$	$\vec{v}_2 = \vec{v}_1 + \Delta t \frac{\vec{F}_1}{m}$
$ec{p}_0$	$\vec{p}_1 = \vec{p}_0 + \Delta t \vec{v}_1$	$\vec{p}_2 = \vec{p}_1 + \Delta t \vec{v}_2$

0	$0 + \Delta t$	$0 + 2\Delta t$
$ec{\omega}_0$	$\vec{\omega}_1 = \vec{\omega}_0 + \Delta t \frac{\vec{\tau}_0}{I}$	$\vec{\omega}_2 = \vec{\omega}_1 + \Delta t \frac{\vec{\tau}_1}{I}$
$ec{ heta}_0$	$\vec{\theta}_1 = \vec{\theta}_0 + \Delta t \vec{\omega}_1$	$\vec{\theta}_2 = \vec{\theta}_1 + \Delta t \vec{\omega}_2$



0	$0 + \Delta t$	$0 + 2\Delta t$	$0 + \mathbf{i}\Delta t$
$ec{v}_0$	$\vec{v}_1 = \vec{v}_0 + \Delta t \frac{\vec{F}_0}{m}$	$\vec{v}_2 = \vec{v}_1 + \Delta t \frac{\vec{F}_1}{m}$	$\vec{v}_i = \vec{v}_{i-1} + \Delta t \frac{\vec{F}_{i-1}}{m}$
$ec{p}_0$	$\vec{p}_1 = \vec{p}_0 + \Delta t \vec{v}_1$	$\vec{p}_2 = \vec{p}_1 + \Delta t \vec{v}_2$	$\vec{p}_i = \vec{p}_{i-1} + \Delta t \vec{v}_i$

0	$0 + \Delta t$	$0 + 2\Delta t$	$0 + \mathbf{i}\Delta t$
$ec{\omega}_0$	$\vec{\omega}_1 = \vec{\omega}_0 + \Delta t \frac{\vec{\tau}_0}{I}$	$\vec{\omega}_2 = \vec{\omega}_1 + \Delta t \frac{\vec{\tau}_1}{I}$	$\vec{\omega}_i = \vec{\omega}_{i-1} + \Delta t \frac{\vec{\tau}_{i-1}}{I}$
$ec{ heta}_0$	$\vec{\theta}_1 = \vec{\theta}_0 + \Delta t \vec{\omega}_1$	$\vec{\theta}_2 = \vec{\theta}_1 + \Delta t \vec{\omega}_2$	$\vec{\theta}_i = \vec{\theta}_{i-1} + \Delta t \vec{\omega}_i$



• Engine pipeline:

Apply external forces

Integrate

Render

Sleep



$$\vec{v}_i = \vec{v}_{i-1} + \Delta t \frac{\vec{F}_{i-1}}{m}$$

$$\vec{p}_i = \vec{p}_{i-1} + \Delta t \vec{v}_i$$

$$\vec{\omega}_i = \vec{\omega}_{i-1} + \Delta t \frac{\vec{\tau}_{i-1}}{l}$$

$$\vec{\theta}_i = \vec{\theta}_{i-1} + \Delta t \vec{\omega}_i$$

Which are known and which are unknown variables?



1st Newton's Law of Motion:

assuming 
$$\vec{F}_{i-1} = 0$$
 and  $\vec{\tau}_{i-1} = 0$ :

a) assuming  $\vec{v}_{i-1} \neq 0$  and  $\vec{\omega}_{i-1} \neq 0$ :

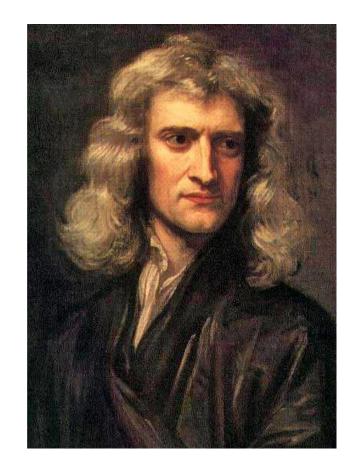
$$\vec{v}_i = \vec{v}_{i-1} + \Delta t \frac{0}{m} = \vec{v}_{i-1} = const.$$

$$\vec{\omega}_i = \vec{\omega}_{i-1} + \Delta t \frac{0}{I} = \vec{\omega}_{i-1} = const.$$

b) assuming  $\vec{v}_{i-1} = 0$  and  $\vec{\omega}_{i-1} = 0$ :

$$\vec{v}_i = 0 + \Delta t \frac{0}{m} = 0 = const.$$

$$\vec{\omega}_i = 0 + \Delta t \frac{0}{I} = 0 = const.$$



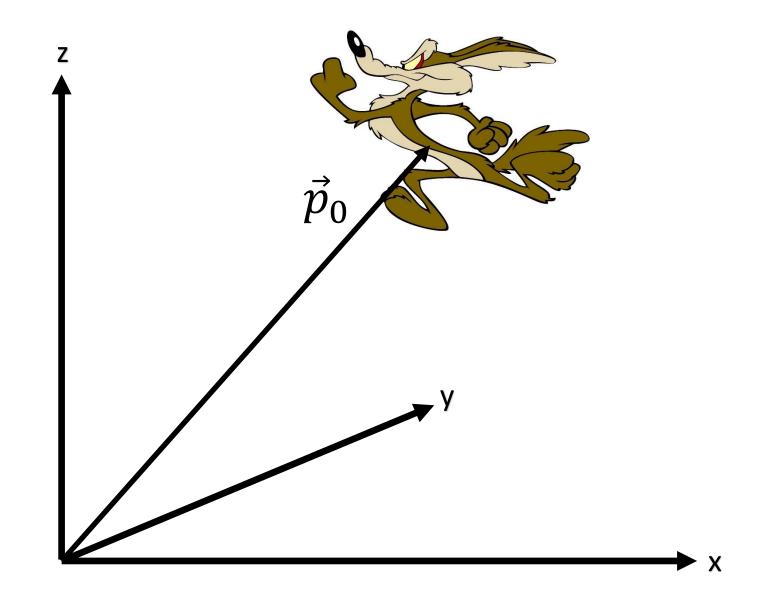


$$\vec{F}_0 = \begin{bmatrix} 0.00 \\ 0.00 \\ 0.00 \end{bmatrix}$$

$$\vec{v}_0 = \begin{bmatrix} 0.00 \\ 0.00 \\ 0.00 \end{bmatrix}$$

$$\vec{p}_0 = \begin{bmatrix} 5.00 \\ 5.00 \\ 5.00 \end{bmatrix}$$

$$t = 0$$



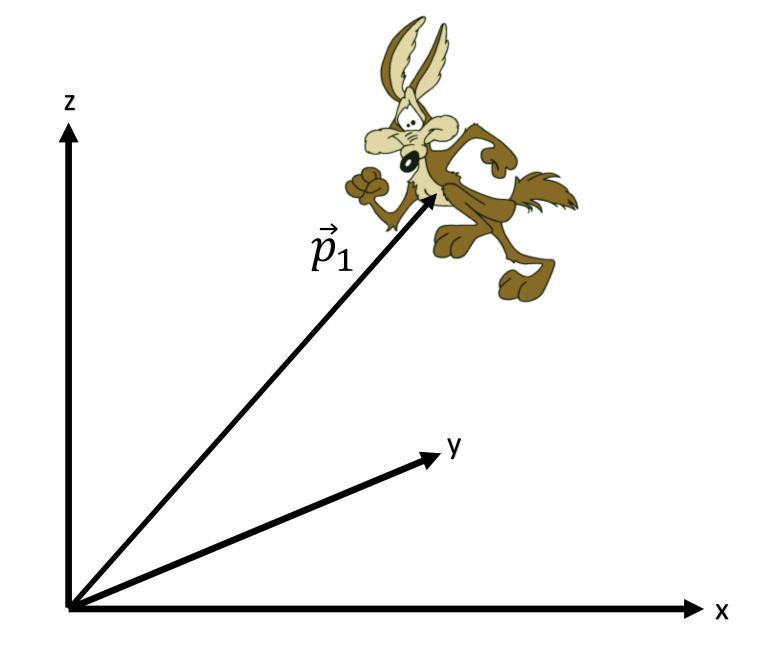


$$\vec{F}_0 = \begin{bmatrix} 0.00 \\ 0.00 \\ 0.00 \end{bmatrix}$$

$$\vec{v}_1 = \begin{bmatrix} 0.00 \\ 0.00 \\ 0.00 \end{bmatrix}$$

$$\vec{p}_1 = \begin{bmatrix} 5.00 \\ 5.00 \\ 5.00 \end{bmatrix}$$

$$t = \Delta t$$



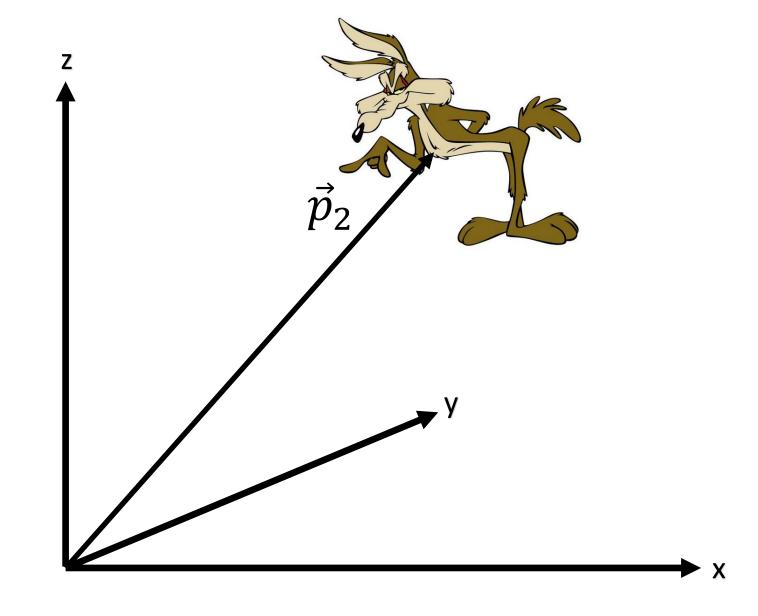


$$\vec{F}_1 = \begin{bmatrix} 0.00 \\ 0.00 \\ 0.00 \end{bmatrix}$$

$$\vec{v}_2 = \begin{bmatrix} 0.00 \\ 0.00 \\ 0.00 \end{bmatrix}$$

$$\vec{p}_2 = \begin{bmatrix} 5.00 \\ 5.00 \\ 5.00 \end{bmatrix}$$

$$t = 2\Delta t$$



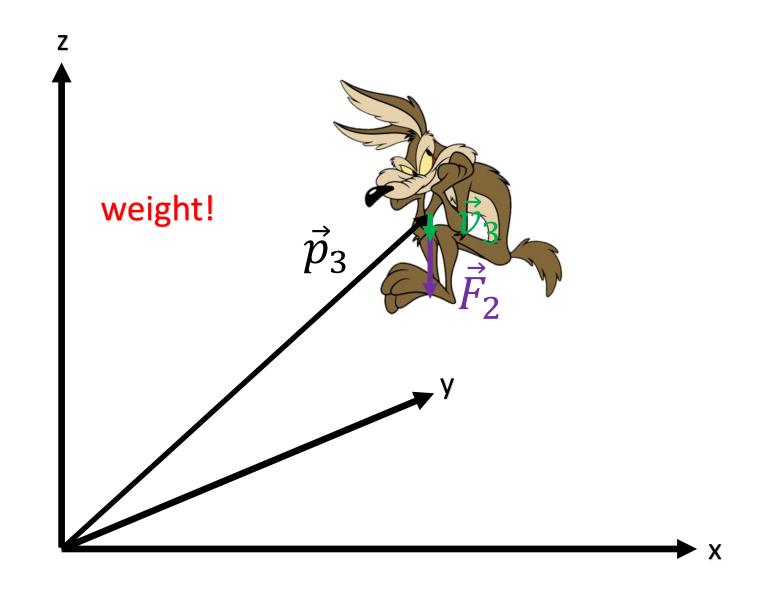


$$\vec{F}_2 = \begin{bmatrix} 0.00 \\ 0.00 \\ -mg \end{bmatrix}$$

$$\vec{v}_3 = \begin{bmatrix} 0.00 \\ 0.00 \\ -2.45 \end{bmatrix}$$

$$\vec{p}_3 = \begin{bmatrix} 5.00 \\ 5.00 \\ 4.38 \end{bmatrix}$$

$$t = 3\Delta t$$



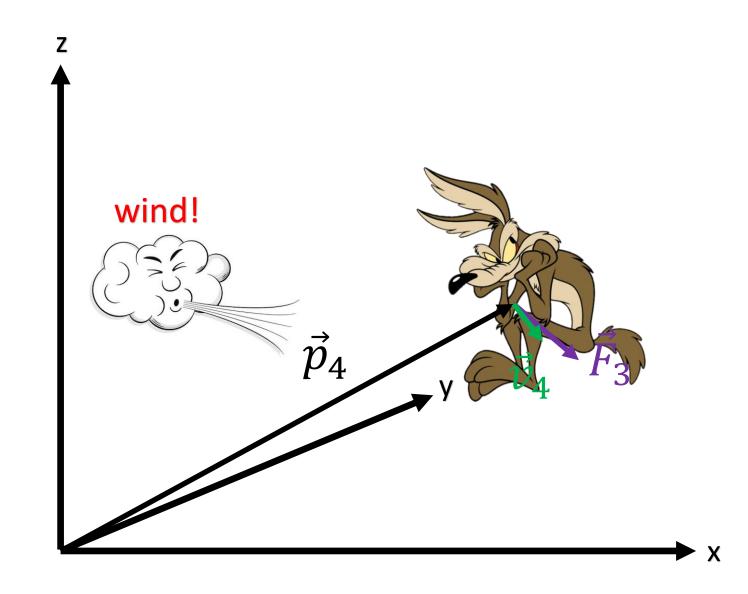


$$\vec{F}_3 = \begin{bmatrix} 10.0 \\ 0.00 \\ -mg \end{bmatrix}$$

$$\vec{v}_4 = \begin{bmatrix} 3.33 \\ 0.00 \\ -4.90 \end{bmatrix}$$

$$\vec{p}_4 = \begin{bmatrix} 6.11 \\ 5.00 \\ 3.15 \end{bmatrix}$$

$$t = 4\Delta t$$



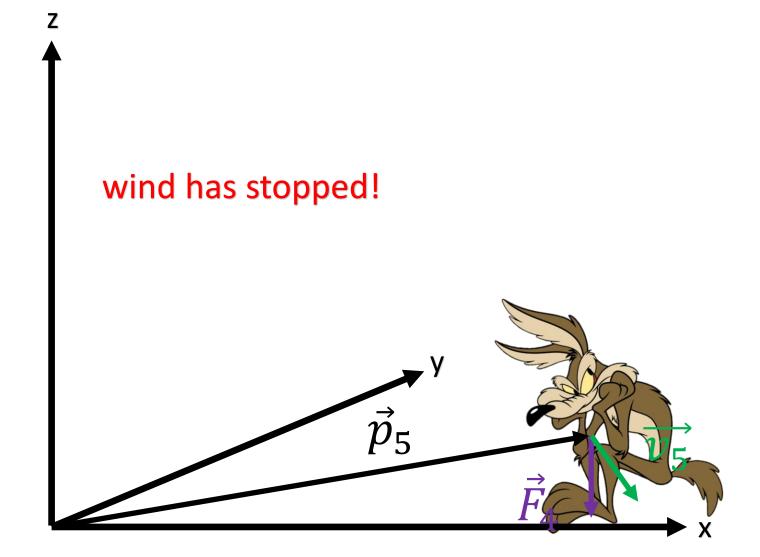


$$\vec{F}_4 = \begin{bmatrix} 0.00 \\ 0.00 \\ -mg \end{bmatrix}$$

$$\vec{v}_5 = \begin{bmatrix} 3.33 \\ 0.00 \\ -9.81 \end{bmatrix}$$

$$\vec{p}_5 = \begin{bmatrix} 7.22 \\ 5.00 \\ 1.31 \end{bmatrix}$$

$$t = 5\Delta t$$



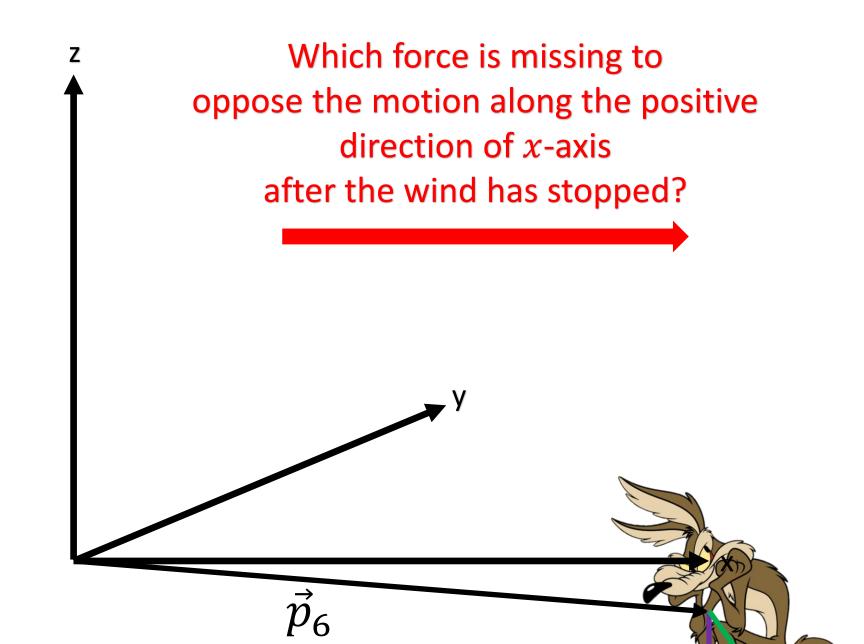


$$\vec{F}_5 = \begin{bmatrix} 0.00 \\ 0.00 \\ -mg \end{bmatrix}$$

$$\vec{v}_6 = \begin{bmatrix} 3.33 \\ 0.00 \\ -7.35 \end{bmatrix}$$

$$\vec{p}_6 = \begin{bmatrix} 8.33 \\ 5.00 \\ -1.14 \end{bmatrix}$$

$$t = 6\Delta t$$





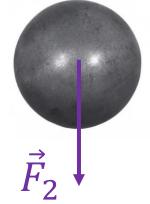
#### vacuum:

$$\vec{F} = \vec{F}_{weight}$$

(only the force of weight acts on the body)







$$v_1 = 0 + \Delta t \frac{m_1 g}{m_1}$$
  $v_2 = 0 + \Delta t \frac{m_2 g}{m_2}$ 

X

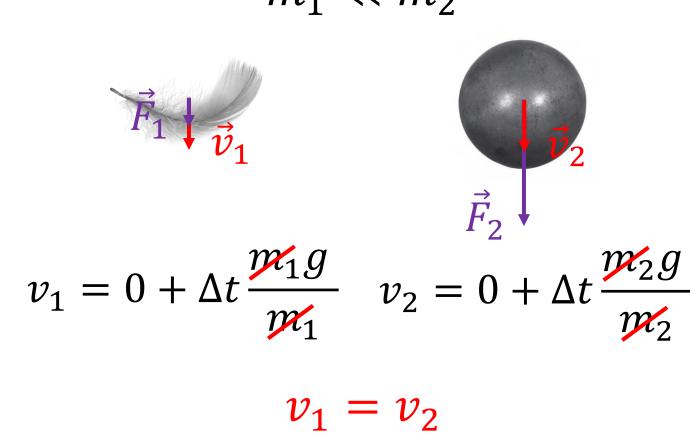


#### vacuum:

$$\vec{F} = \vec{F}_{weight}$$

(only the force of weight acts on the body)





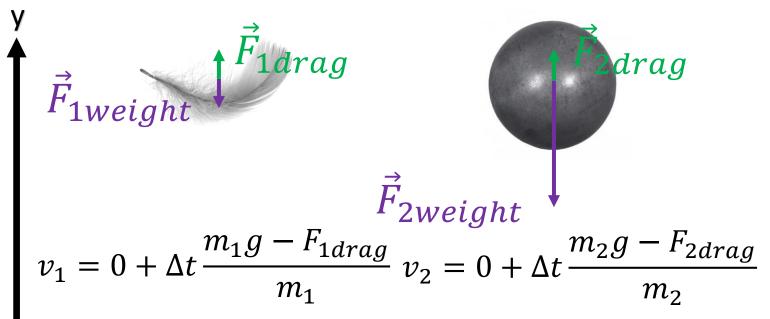


#### fluid:

$$\vec{F} = \vec{F}_{weight} + \vec{F}_{drag}$$

(weight + fluid drag force)





Х

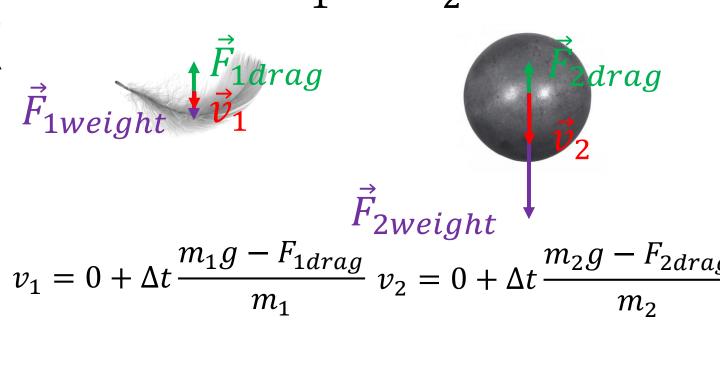


#### fluid:

$$\vec{F} = \vec{F}_{weight} + \vec{F}_{drag}$$

(weight + fluid drag force)





$$v_1 < v_2$$



Two basic types of bodies:

- 1. Particles
- 2. Rigid bodies

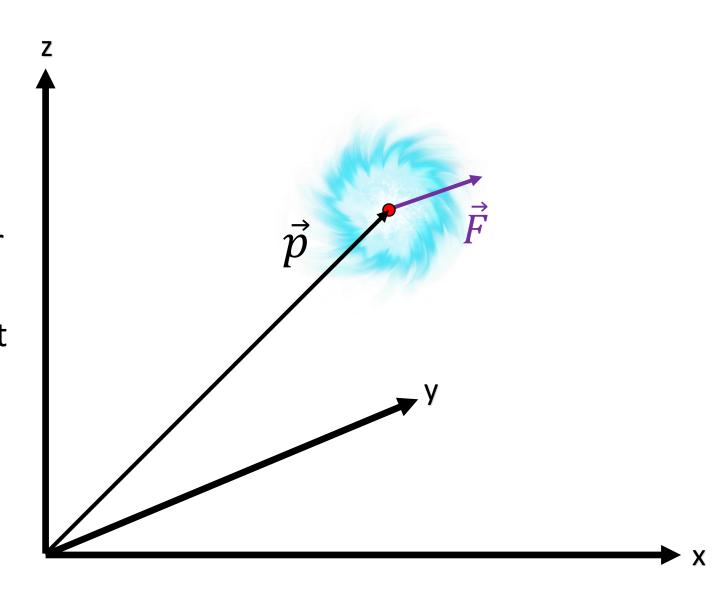
Complex types of bodies (various combinations of basic types):

- 1. Soft bodies
- 2. Fluids
- 3. Cloth
- 4. Ragdolls
- 5. etc.



• Particle:

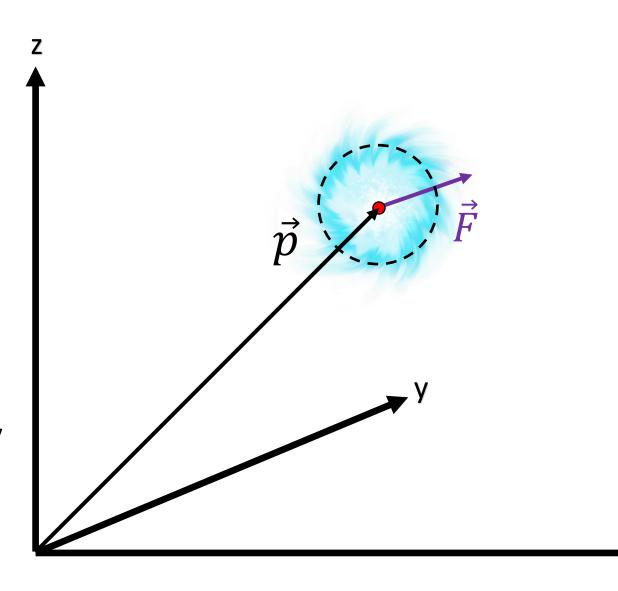
- + has no orientation or rotation
- + all external forces act on  $\overrightarrow{C_g}$





- Particle:
- + has no orientation or rotation
- + all external forces act on  $\overrightarrow{C_g}$

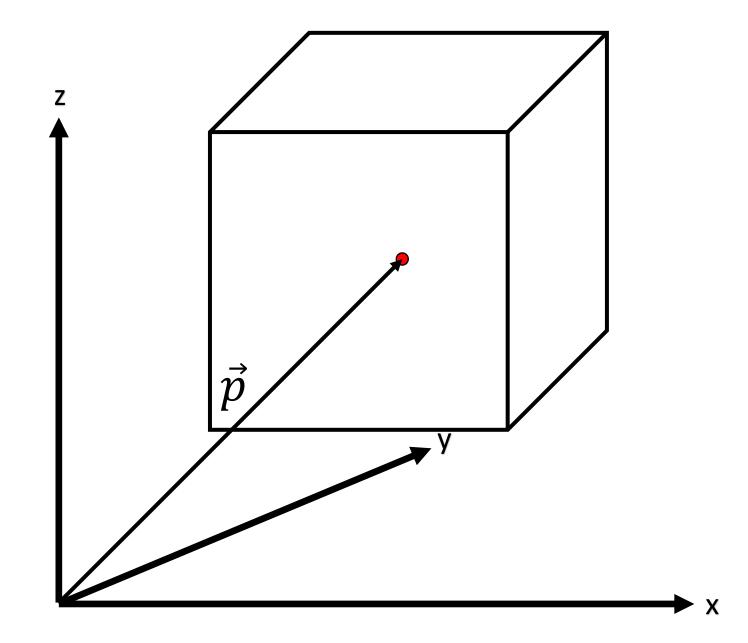
(a symmetrical geometry is usually used for collision detection)





• Rigid body:

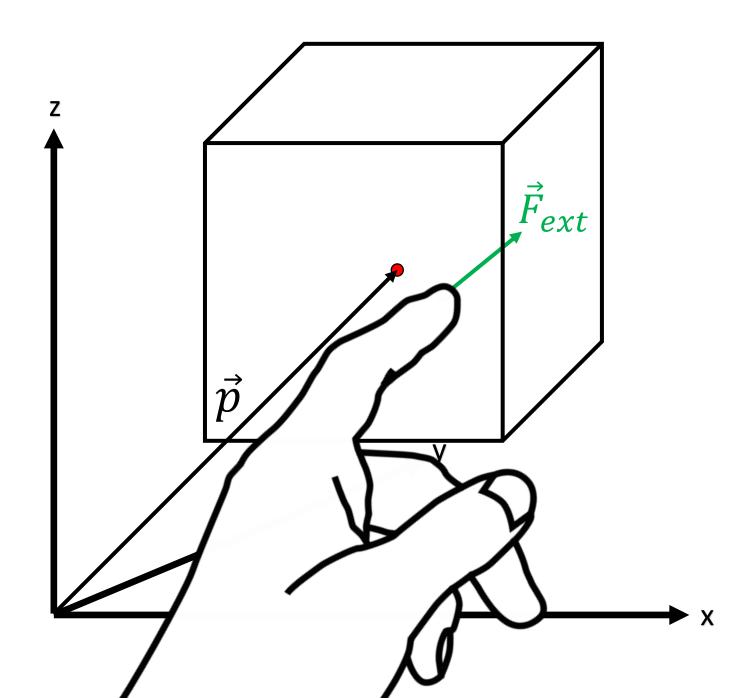
has orientation and rotation





• Rigid body:

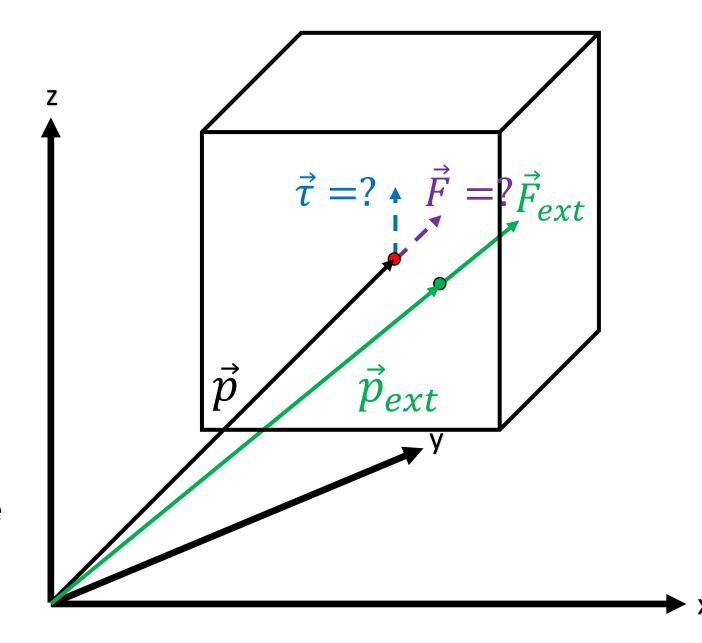
has orientation and rotation





• Rigid body:

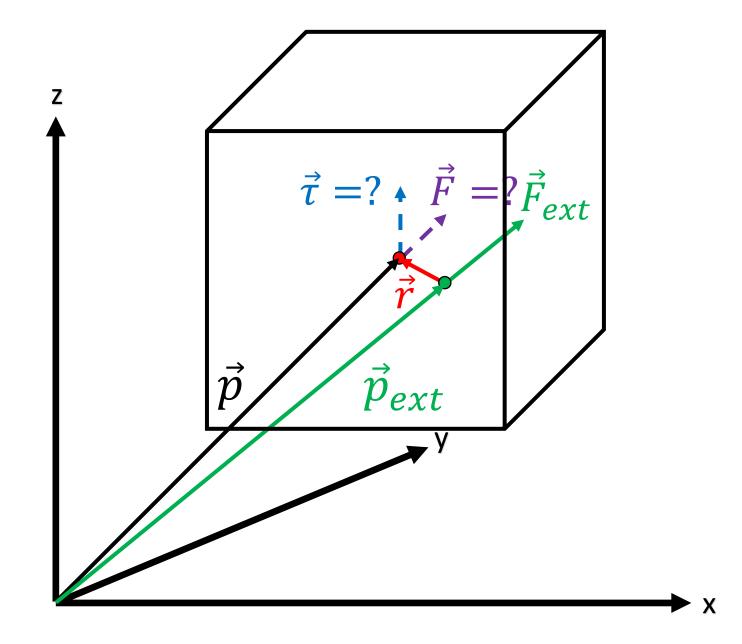
- has orientation and rotation
- Linear force  $\vec{F}$  and torque  $\vec{t}$  must be computed from an applied external force  $\vec{F}_{ext}$





• Rigid body:

$$1. \quad \vec{r} = \vec{p} - \vec{p}_{ext}$$

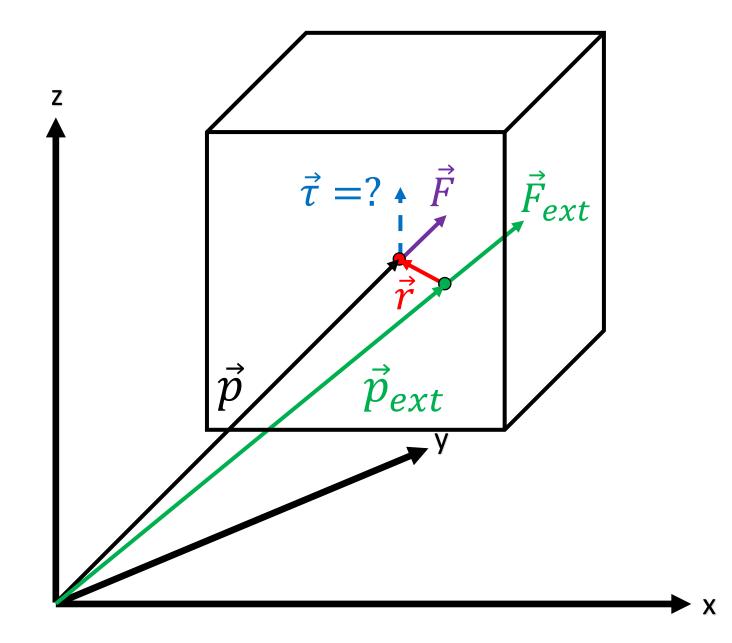




• Rigid body:

$$1. \quad \vec{r} = \vec{p} - \vec{p}_{ext}$$

1. 
$$\vec{r} = \vec{p} - \vec{p}_{ext}$$
2.  $\vec{F} = \vec{F}_{ext} \cdot \vec{r} \frac{\vec{F}_{ext}}{|\vec{F}_{ext}|}$ 



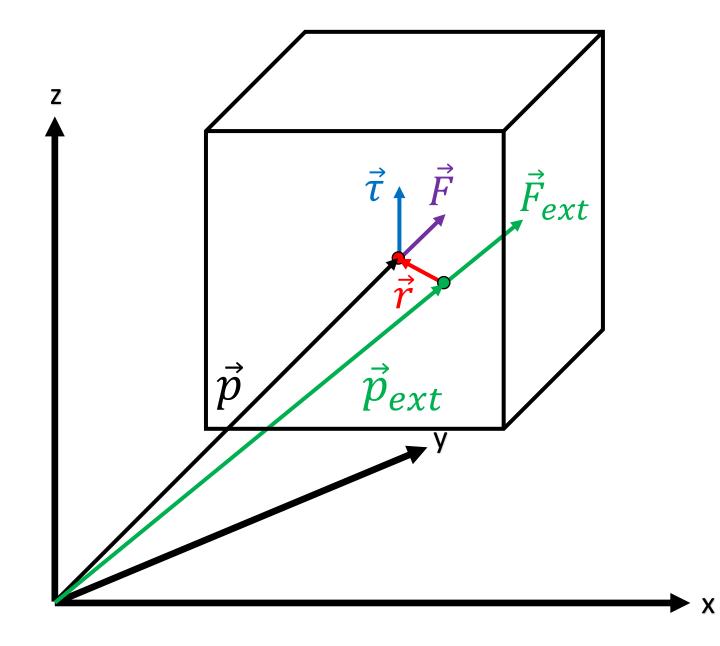


• Rigid body:

$$1. \quad \vec{r} = \vec{p} - \vec{p}_{ext}$$

2. 
$$\vec{F} = \vec{F}_{ext} \cdot \vec{r} \frac{\vec{F}_{ext}}{|\vec{F}_{ext}|}$$
3.  $\vec{\tau} = \vec{F}_{ext} \times \vec{r}$ 

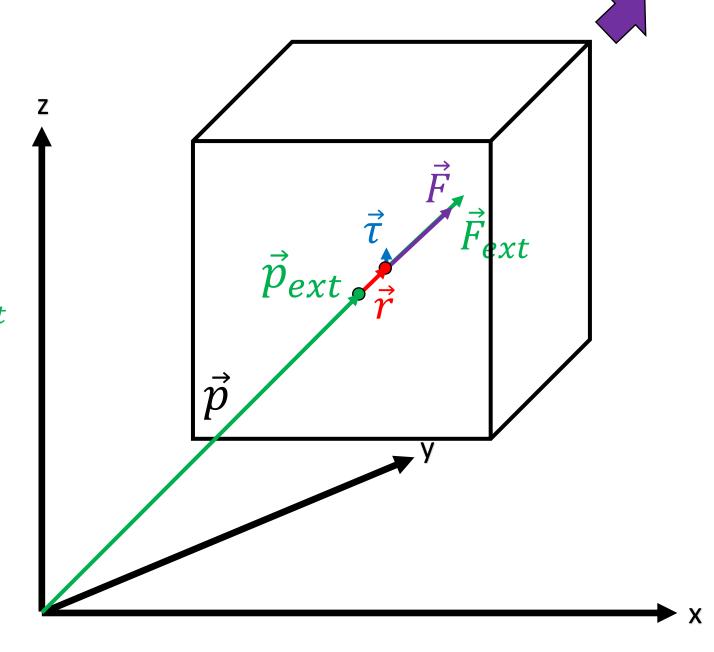
3. 
$$\vec{\tau} = \vec{F}_{ext} \times \vec{r}$$





• Rigid body:

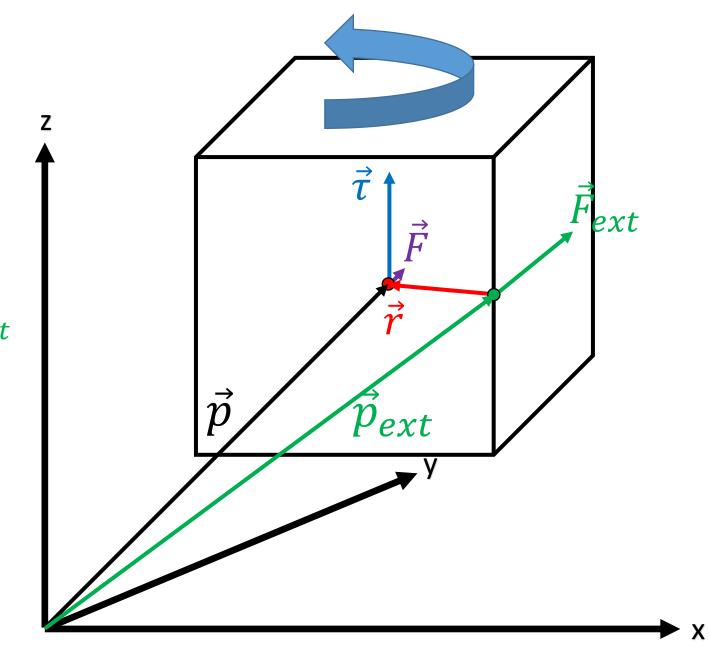
The more the  $\vec{r}$  and  $\vec{F}_{ext}$  are parallel to each other, the linear motion becomes more dominant





• Rigid body:

The more the  $\vec{r}$  and  $\vec{F}_{ext}$  are perpendicular to each other, the rotational motion becomes more dominant

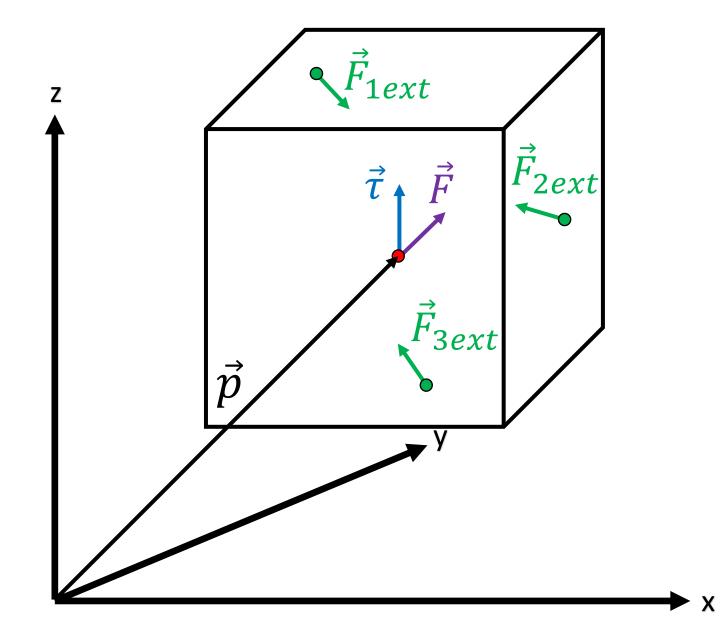




• Rigid body:

$$\vec{F} = \vec{F}_1 + \vec{F}_2 + \cdots$$

$$\vec{\tau} = \vec{\tau}_1 + \vec{\tau}_2 + \cdots$$





#### • Scaling:

If, for example, a body should travel:

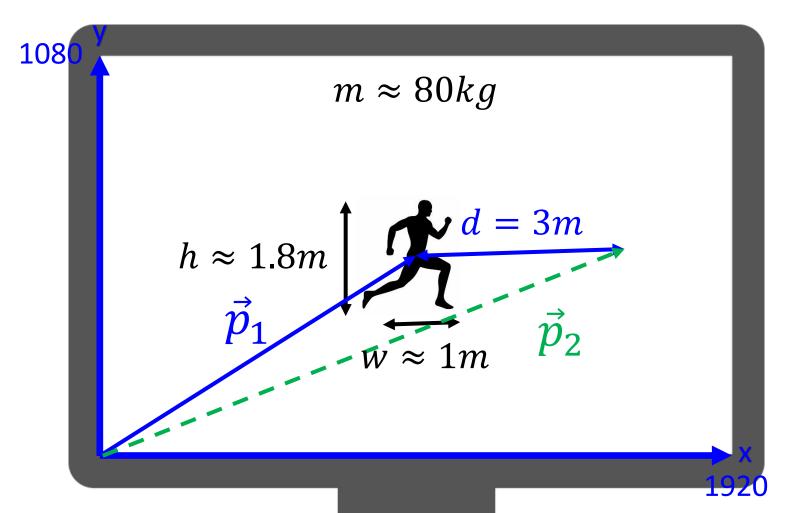
$$t = 1s$$

from position  $\vec{p}_1$  to position  $\vec{p}_2$  over a distance of:

$$d = 3m$$

it's velocity should be:

$$\vec{v} = \begin{bmatrix} 3 \\ 0 \end{bmatrix} \frac{m}{s}$$



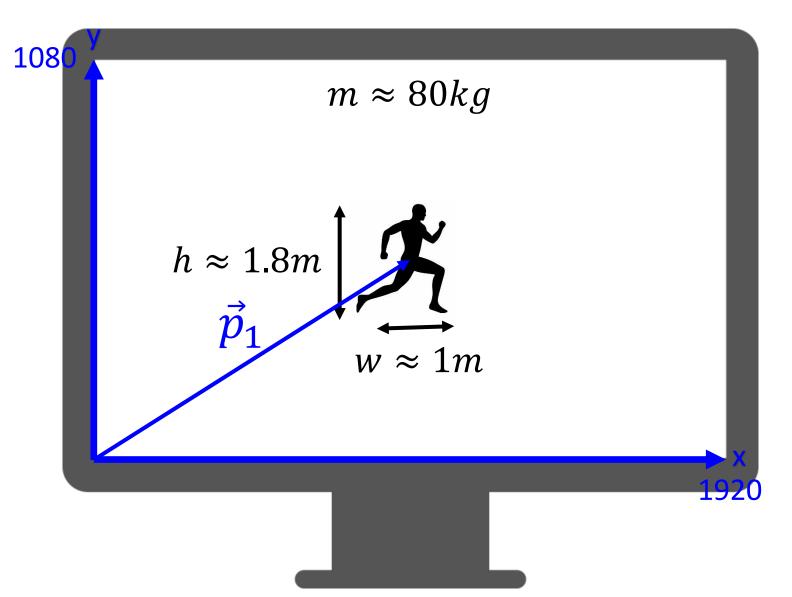


• Scaling:

without the scaling: 1m = 1px

$$\vec{p}_1 = \begin{bmatrix} 960 \\ 540 \end{bmatrix} px$$

$$\vec{v} = \begin{bmatrix} 3 \\ 0 \end{bmatrix} \frac{px}{s}$$





#### • scaling:

without the scaling: 1m = 1px

$$\vec{p}_1 = \begin{bmatrix} 960 \\ 540 \end{bmatrix} px$$

$$\vec{v} = \begin{bmatrix} 3 \\ 0 \end{bmatrix} \frac{px}{s}$$

$$t = 1s$$

$$\vec{p}_2 = \begin{bmatrix} 963 \\ 540 \end{bmatrix} px$$

108  $m \approx 80 kg$  $h \approx 1.8m$  $w \approx 1m$ 

Too slow?



#### • scaling:

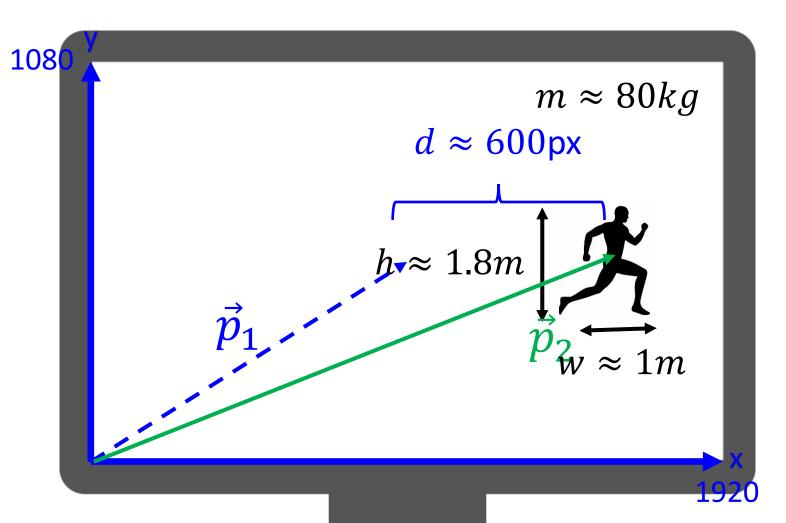
without the scaling: 1m = 1px

$$\vec{p}_1 = \begin{bmatrix} 960 \\ 540 \end{bmatrix} px$$

$$\vec{v} = \begin{bmatrix} 600 \\ 0 \end{bmatrix} \frac{px}{s}$$

$$t = 1s$$

$$\vec{p}_2 = \begin{bmatrix} 1560 \\ 540 \end{bmatrix} px$$



Solution?



#### • scaling:

without the scaling: 1m = 1px

$$\vec{p}_1 = \begin{bmatrix} 960 \\ 540 \end{bmatrix} px$$

$$\vec{v} = \begin{bmatrix} 600 \\ 0 \end{bmatrix} \frac{px}{s}$$

$$t = 1s$$

$$\vec{p}_2 = \begin{bmatrix} 1560 \\ 540 \end{bmatrix} px$$

108  $m \approx 80 kg$  $d \approx 600 \text{px}$  $h \approx 270 \text{px}$ 

Solution?



• scaling:

without scaling:

$$1m = 1px$$

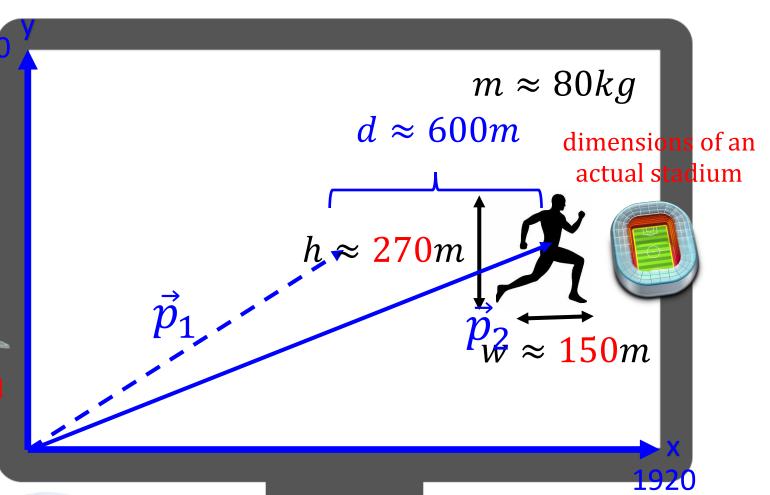
$$\vec{p}_1 = \begin{bmatrix} 960 \\ 540 \end{bmatrix} m$$

$$\vec{v} = \begin{bmatrix} 600 \\ 0 \end{bmatrix} \frac{m}{s}$$
F22's speed

108

To achieve such speed during 1s, a

force is required of:
$$F = 80kg \frac{{600 \brack 0} \frac{m}{s}}{1s} = {48000 \brack 0} N$$



thrust force of a jet engine



#### • scaling:

with the scaling:

$$1m = 200px$$

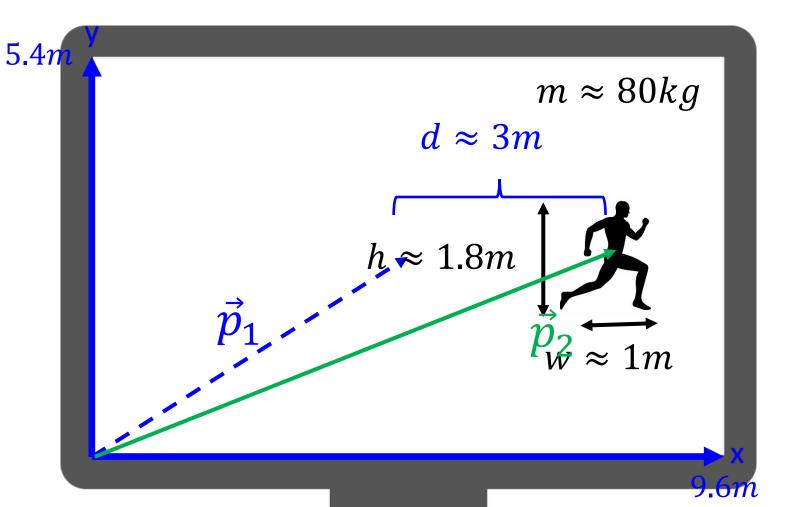
$$\vec{p}_1 = \begin{bmatrix} 4.8 \\ 2.8 \end{bmatrix} m$$

$$\vec{v} = \begin{bmatrix} 3 \\ 0 \end{bmatrix} \frac{m}{s}$$

$$t = 1s$$

$$\vec{p}_2 = \begin{bmatrix} 7.8 \\ 2.8 \end{bmatrix} m$$

Ratio is determined empirically and should be parametrized (zoom).

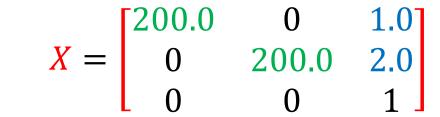


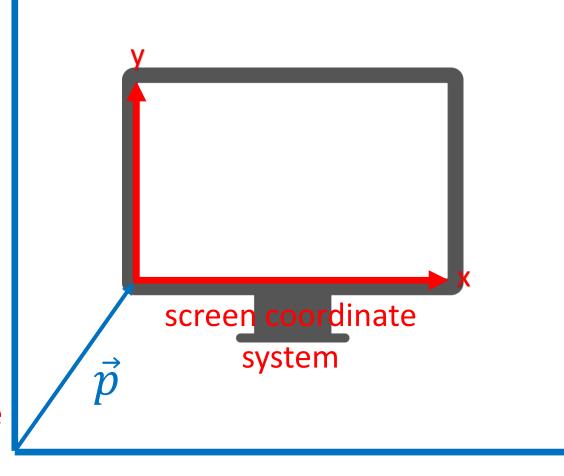


screen coordinate system:

$$X = \begin{bmatrix} s_x & 0 & p_x \\ 0 & s_y & p_y \\ 0 & 0 & 1 \end{bmatrix}$$

The greater the scaling factor, the greater portion of global space can fit onto screen!



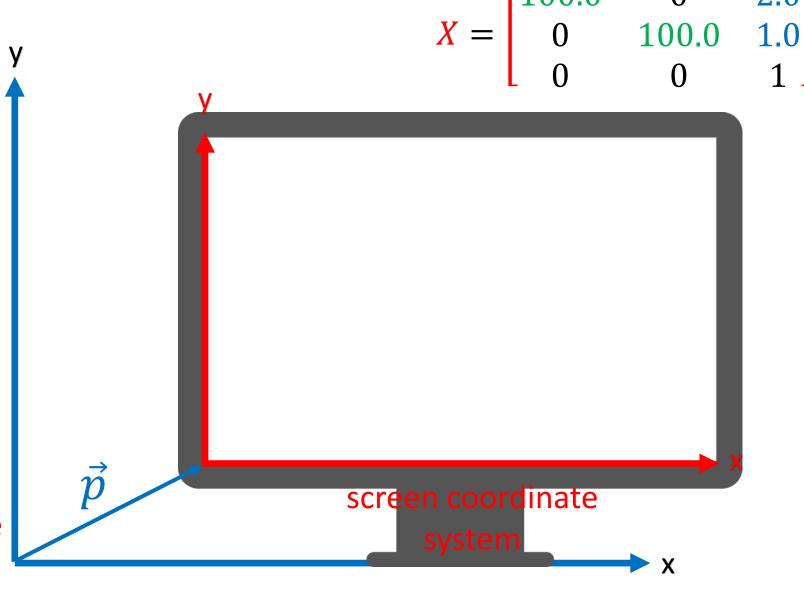




screen coordinate system:

$$X = \begin{bmatrix} s_x & 0 & p_x \\ 0 & s_y & p_y \\ 0 & 0 & 1 \end{bmatrix}$$

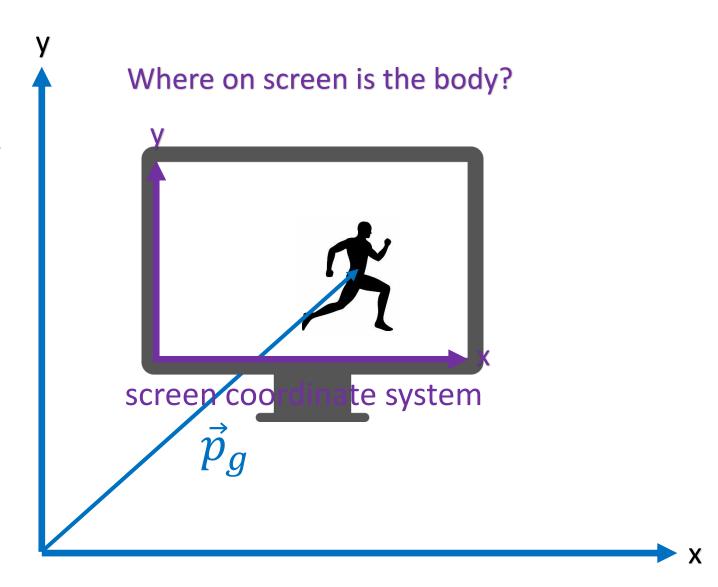
The smaller the scaling factor, the lesser portion of global space can fit onto screen!





screen coordinate system:

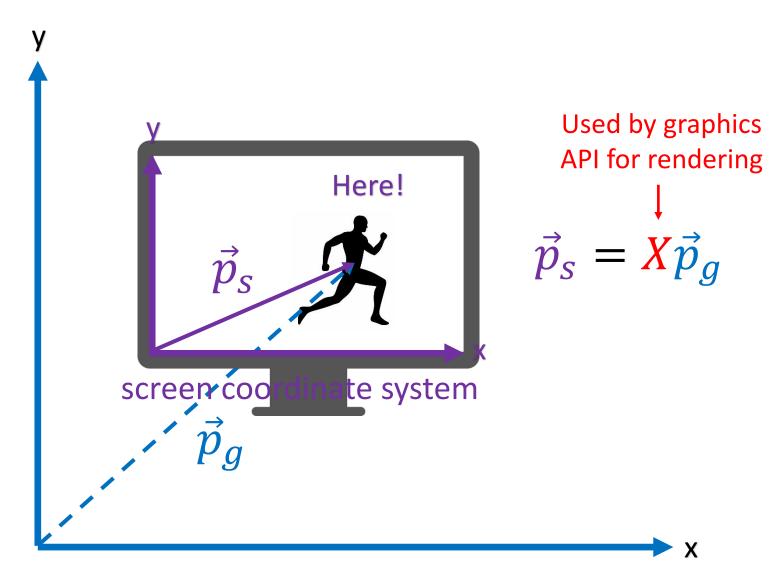
$$X = \begin{bmatrix} s_x & 0 & p_x \\ 0 & s_y & p_y \\ 0 & 0 & 1 \end{bmatrix}$$





screen coordinate system:

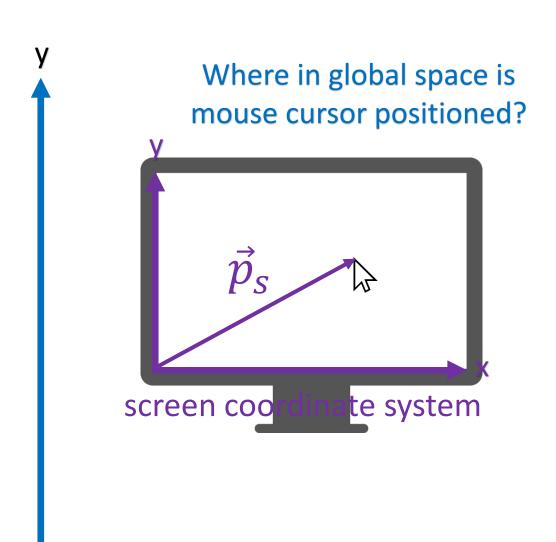
$$X = \begin{bmatrix} s_x & 0 & p_x \\ 0 & s_y & p_y \\ 0 & 0 & 1 \end{bmatrix}$$





• screen coordinate system:

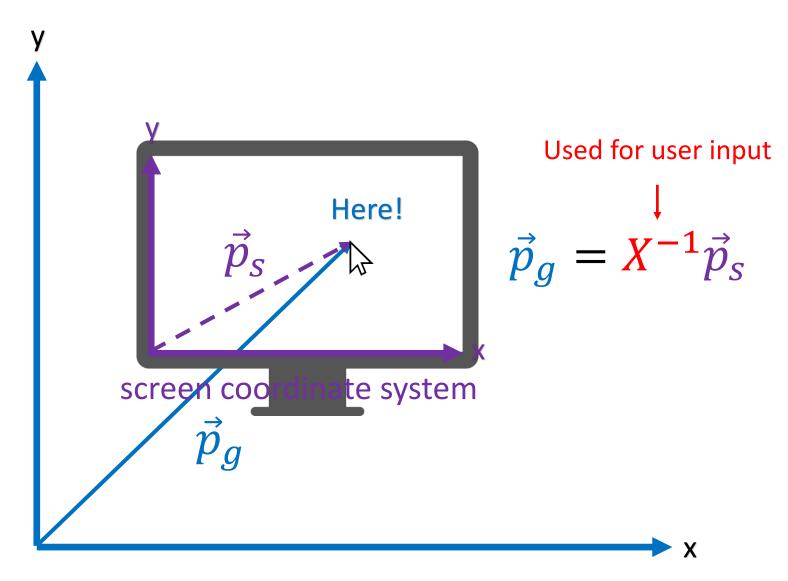
$$X = \begin{bmatrix} s_x & 0 & p_x \\ 0 & s_y & p_y \\ 0 & 0 & 1 \end{bmatrix}$$





screen coordinate system:

$$X = \begin{bmatrix} s_x & 0 & p_x \\ 0 & s_y & p_y \\ 0 & 0 & 1 \end{bmatrix}$$

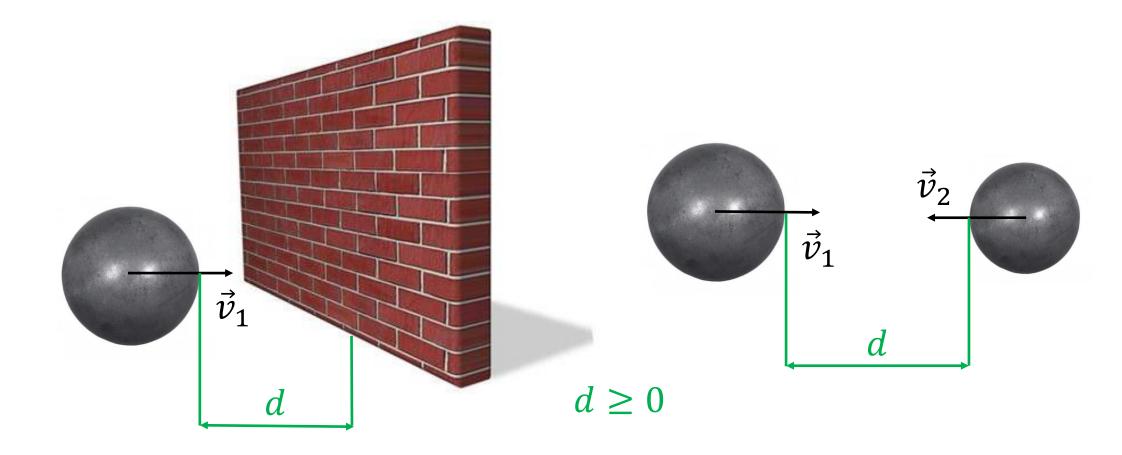




How to limit the motion of a physical system given predetermined conditions?

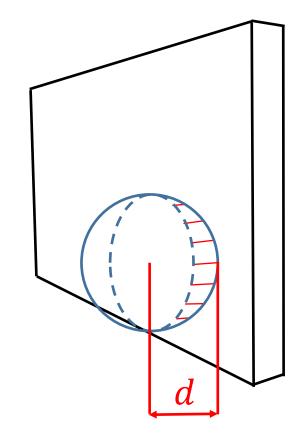


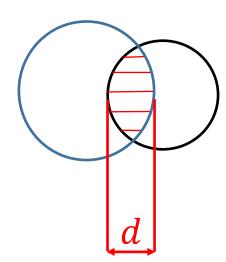
• Contact constraints: admissible state





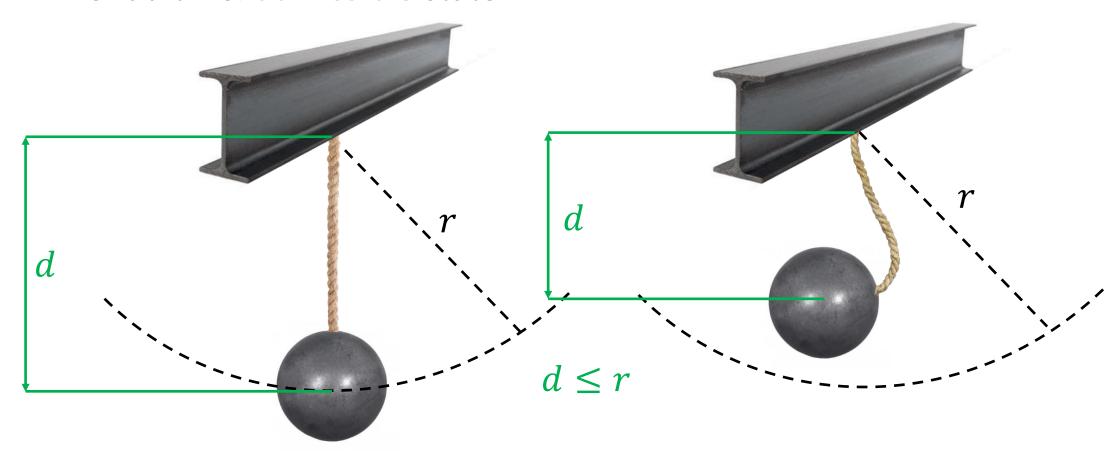
• Contact constraints: inadmissible state





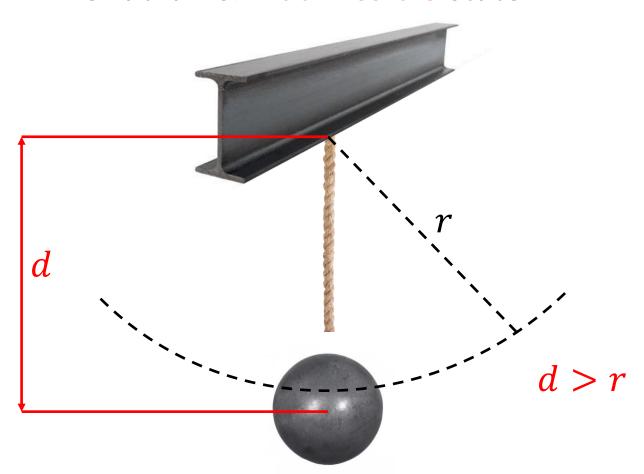


• Pendulums: admissible state



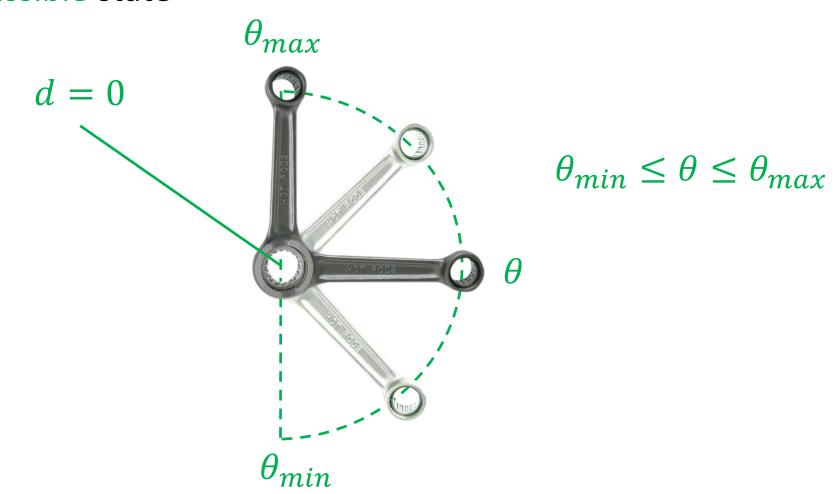


• Pendulums: inadmissible state



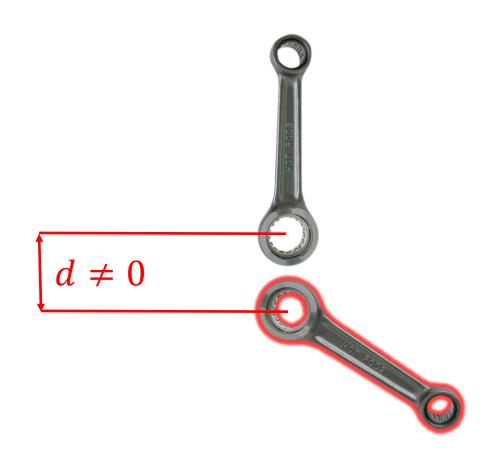


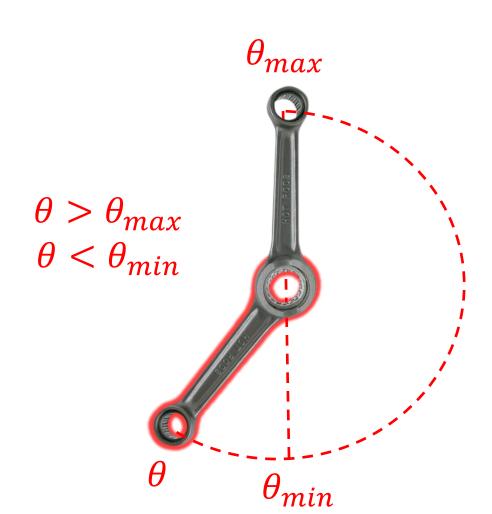
• Joints: admissible state





• Joints: inadmissible state







- springs
- fluids
- cloth
- ragdolls
- etc.



• Engine pipeline:

Apply external forces

Integrate

**Detect constraint violations** 

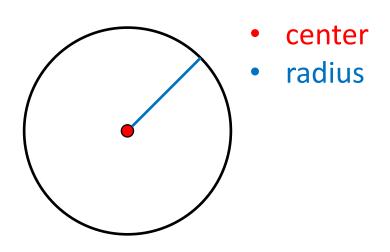
Solve constraint violations

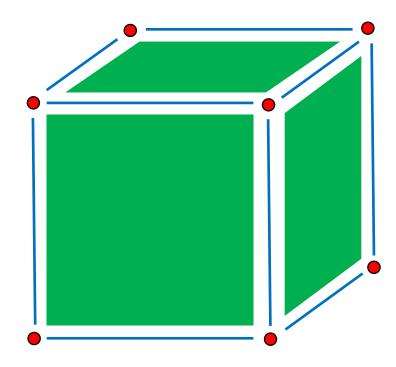
Render

Sleep



- 1. Collision detection
  - geometry features

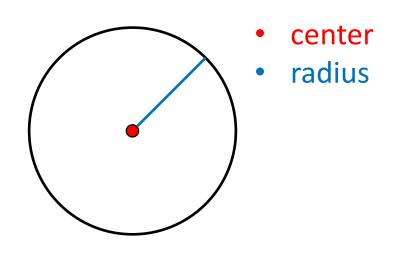


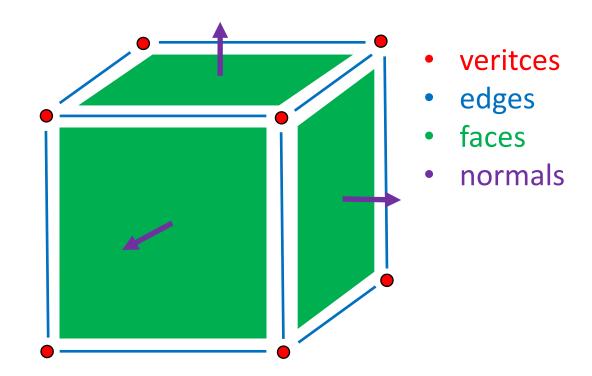


- veritces
- edges
- faces
- where is outside or inside?



- 1. Collision detection
  - geometry features

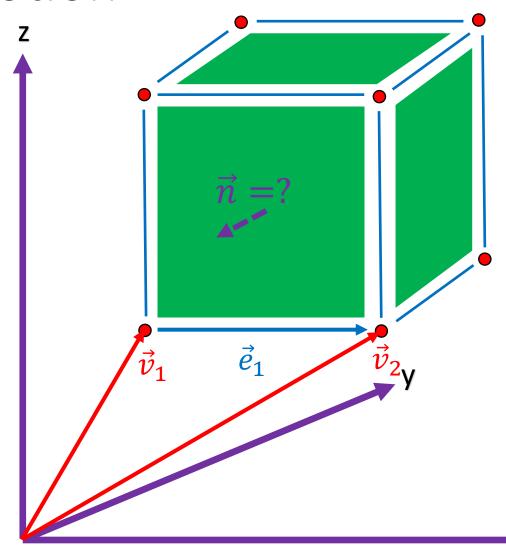






- 1. Collision detection
  - normals (3D)

$$\vec{e}_1 = \vec{v}_2 - \vec{v}_1$$

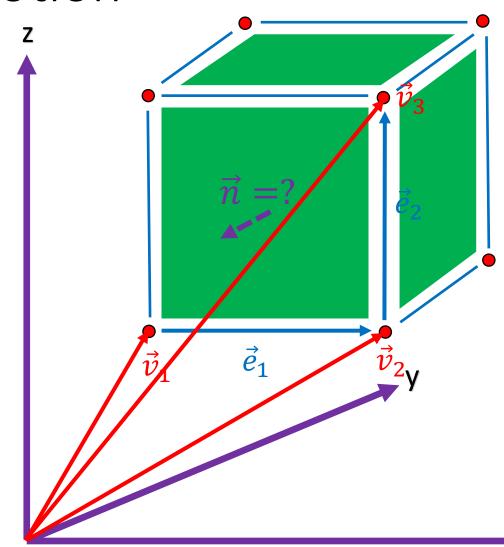




- 1. Collision detection
  - normals (3D)

$$\vec{e}_1 = \vec{v}_2 - \vec{v}_1$$

$$\vec{e}_2 = \vec{v}_3 - \vec{v}_2$$





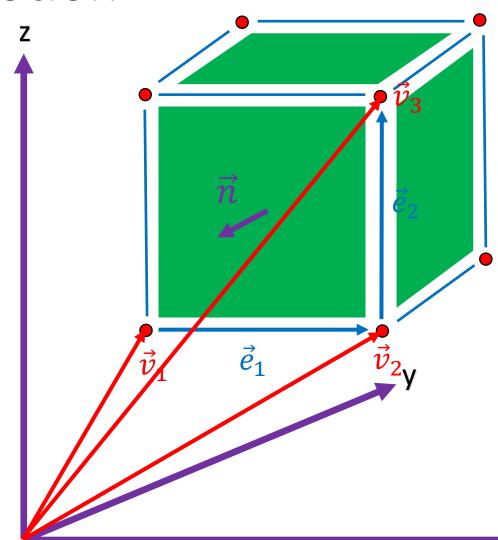
# 1. Collision detection

• normals (3D)

$$\vec{e}_1 = \vec{v}_2 - \vec{v}_1$$

$$\vec{e}_2 = \vec{v}_3 - \vec{v}_2$$

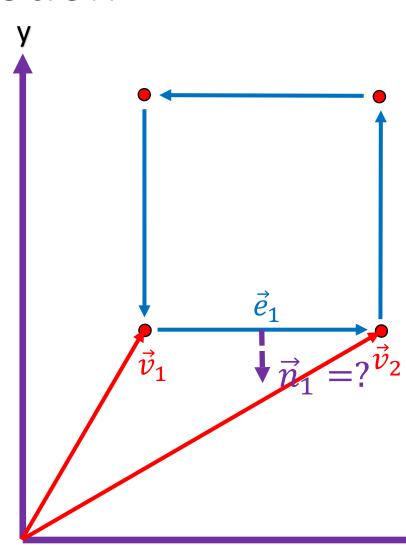
$$\vec{n} = \frac{\vec{e}_2 \times \vec{e}_1}{|\vec{e}_2 \times \vec{e}_1|}$$





- 1. Collision detection
  - normals (2D)

$$\vec{e}_1 = \vec{v}_2 - \vec{v}_1 = (x_1, y_1)$$

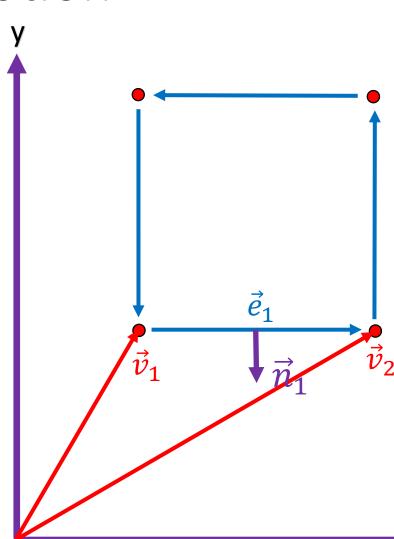




- 1. Collision detection
  - normals (2D)

$$\vec{e}_1 = \vec{v}_2 - \vec{v}_1 = (x_1, y_1)$$

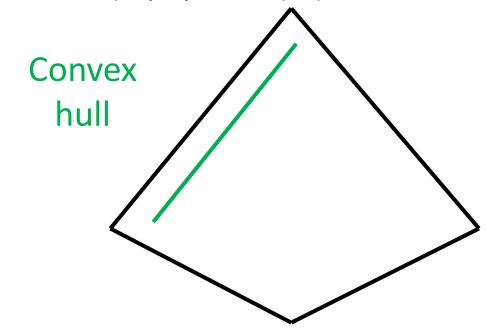
$$\vec{n}_1 = \frac{(y_1, -x_1)}{|(y_1, -x_1)|}$$

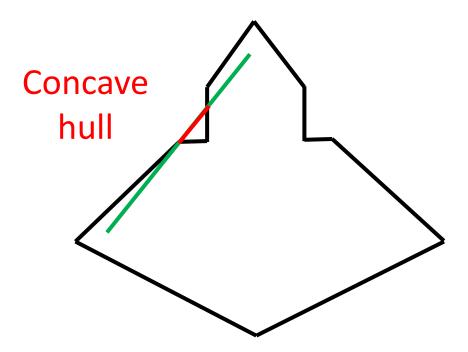




#### 1. Collision detection

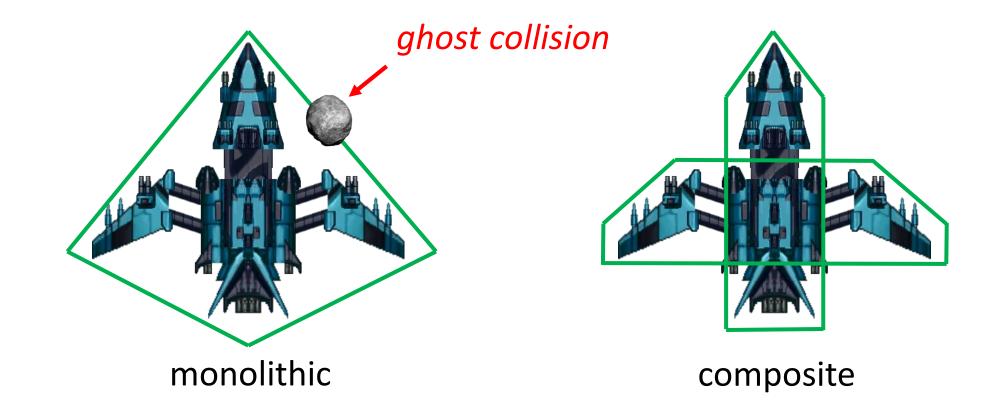
- Convex hull:
  - a) polygon (2D)
  - b) polyhedron (3D)







- 1. Collision detection
  - Convex hull:





1. Collision detection

#### algorithms:

- SAT (Separating Axis Test)
- Gilbert–Johnson–Keerthi Distance Algorithm + Expanding Polytope Algorithm

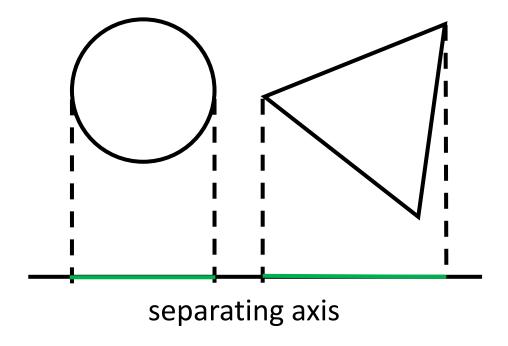
- Both use convex hulls!

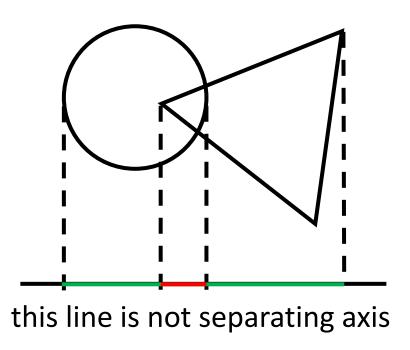


#### 1. Collision detection

Separating Axis Theorem:

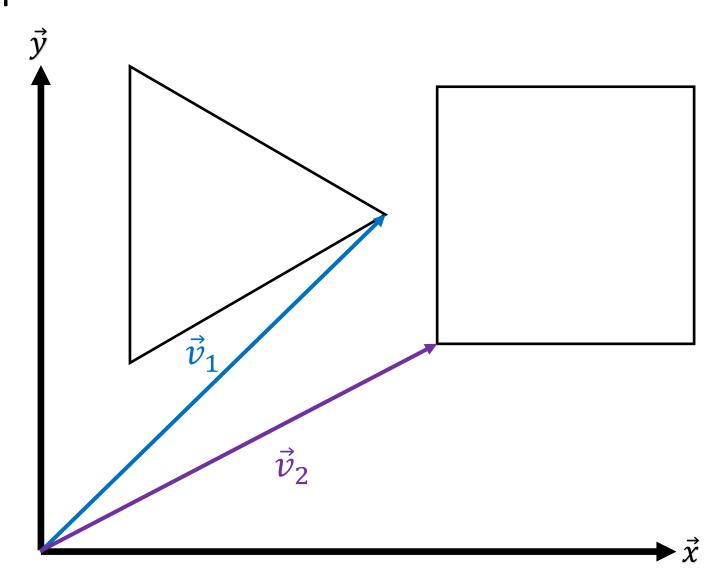
"Two convex objects do not overlap if there exists a line (called axis) onto which the two objects' projections do not overlap."







- 1. Collision detection
  - Separating Axis Test:





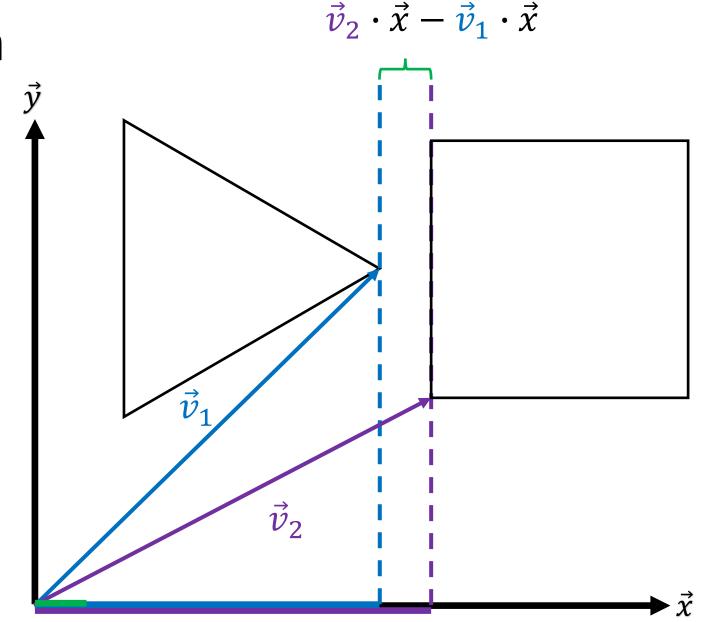
#### 1. Collision detection

• Separating Axis Test:

Dot product

$$\vec{v}_2 \cdot \vec{x} - \vec{v}_1 \cdot \vec{x} > 0$$

Bodies do not touch each other!





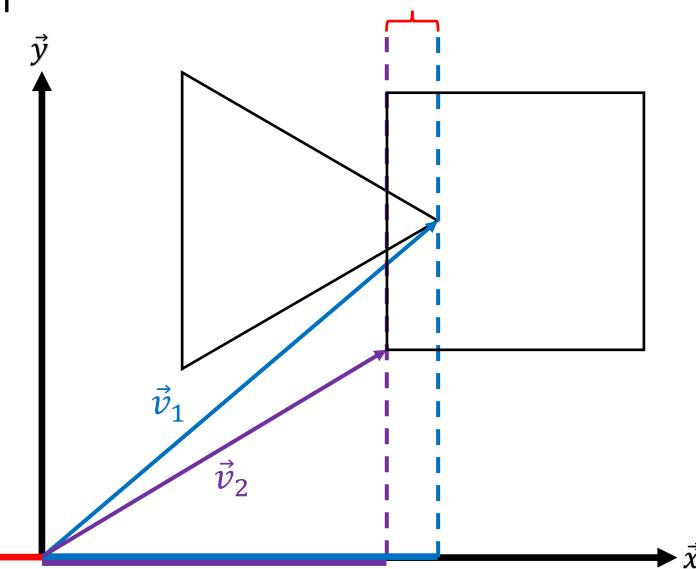
#### 1. Collision detection

• Separating Axis Test:

Dot product

$$\vec{v}_2 \cdot \vec{x} - \vec{v}_1 \cdot \vec{x} \le 0$$

Contact exists!



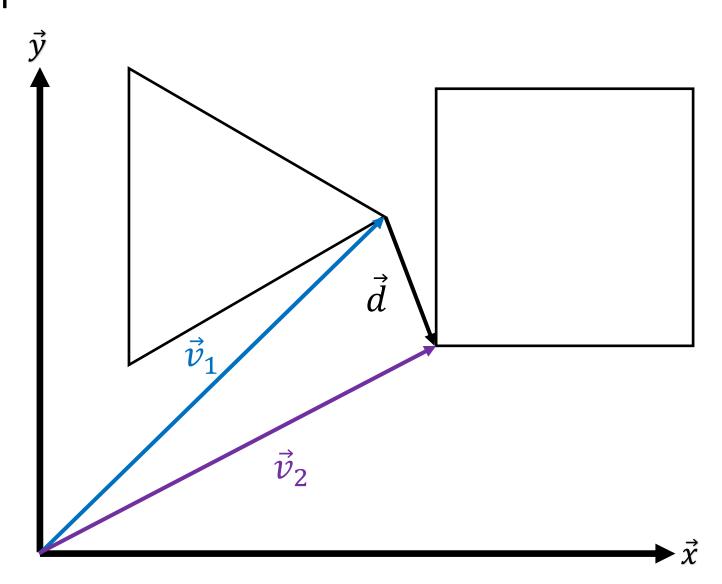
 $\vec{v}_2 \cdot \vec{x} - \vec{v}_1 \cdot \vec{x}$ 



- 1. Collision detection
  - Separating Axis Test:

Distance vector:

$$\vec{d} = \vec{v}_2 - \vec{v}_1$$

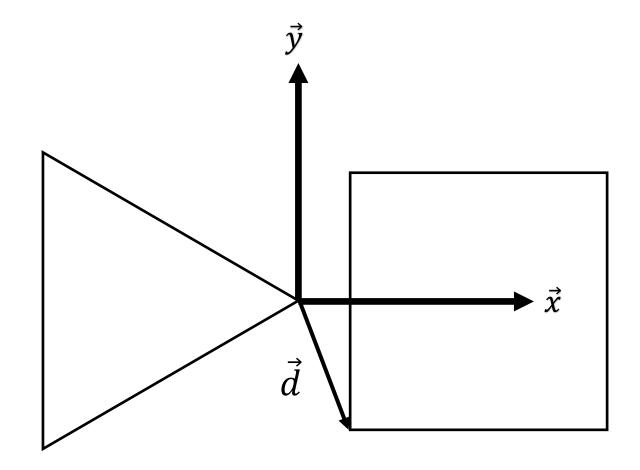




- 1. Collision detection
  - Separating Axis Test:

Distance vector:

$$\vec{d} = \vec{v}_2 - \vec{v}_1$$



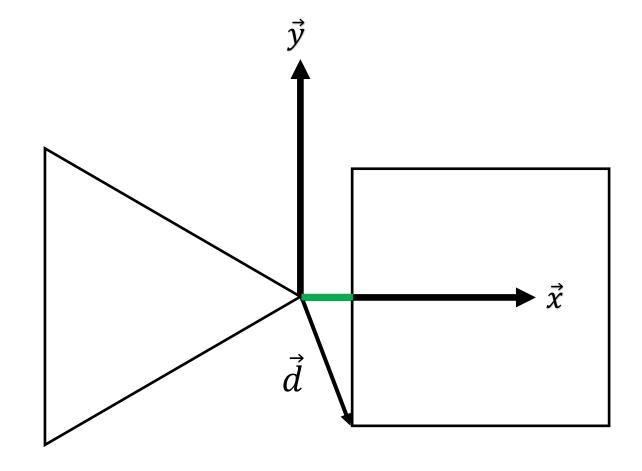


- 1. Collision detection
  - Separating Axis Test:

Dot product

$$\vec{d} \cdot \vec{x} > 0$$

Bodies do not touch each other!



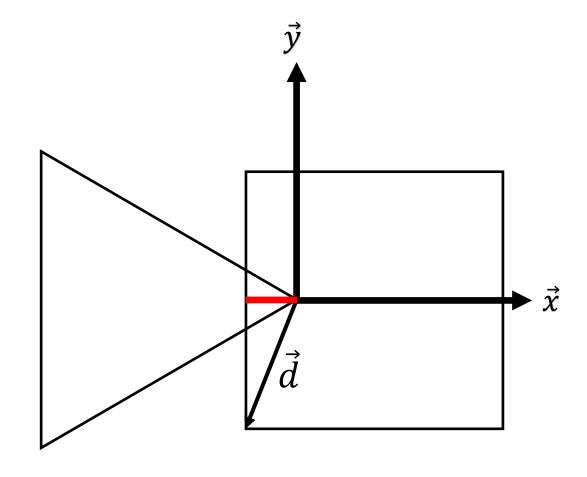


- 1. Collision detection
  - Separating Axis Test:

Dot product

$$\vec{d} \cdot \vec{x} \le 0$$

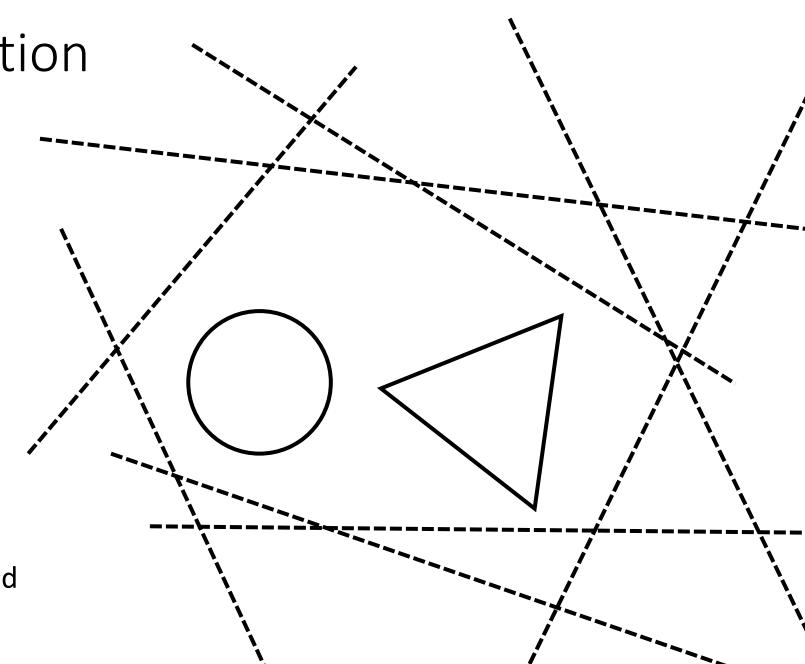
Contact exists!





- 1. Collision detection
  - Separating Axis Test:

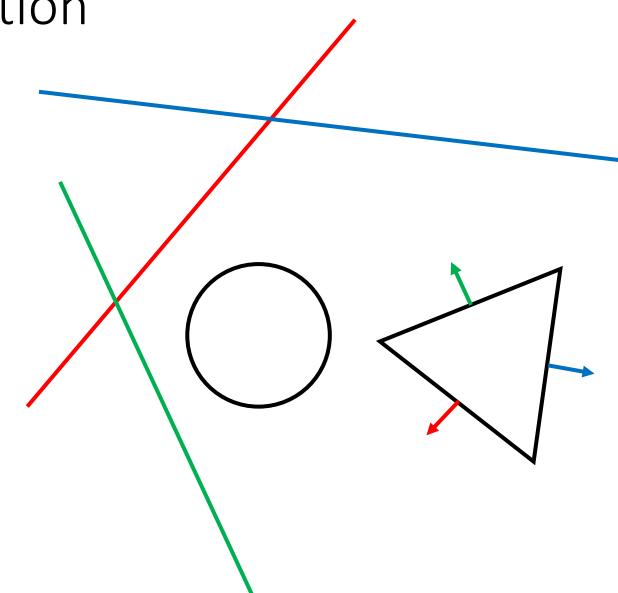
How many axes exist?
Against how many axes should test be performed?





- 1. Collision detection
  - Separating Axis Test:

In 2D space it is sufficient to test against convex hull normals!

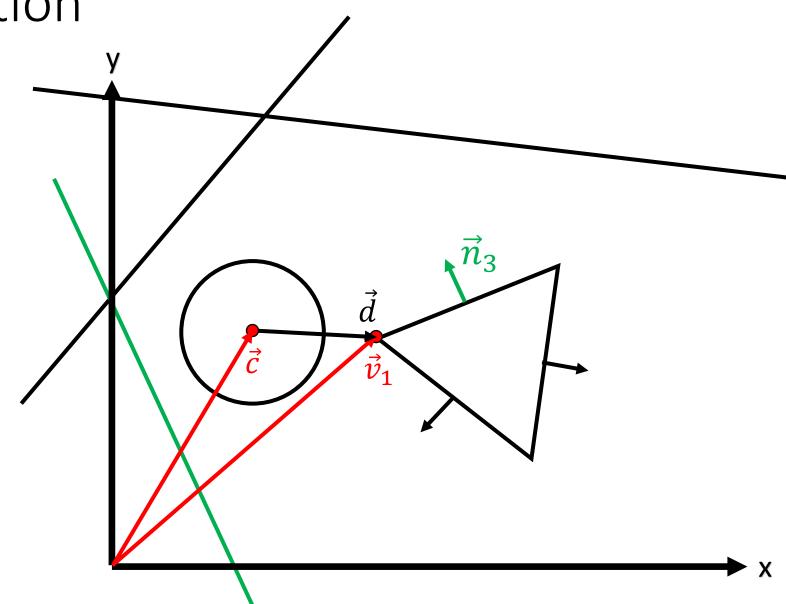




- 1. Collision detection
  - Separating Axis Test:

$$\vec{d} = \vec{v}_1 - \vec{c}$$

Geometries of both bodies must be transformed to the same coordinate system!

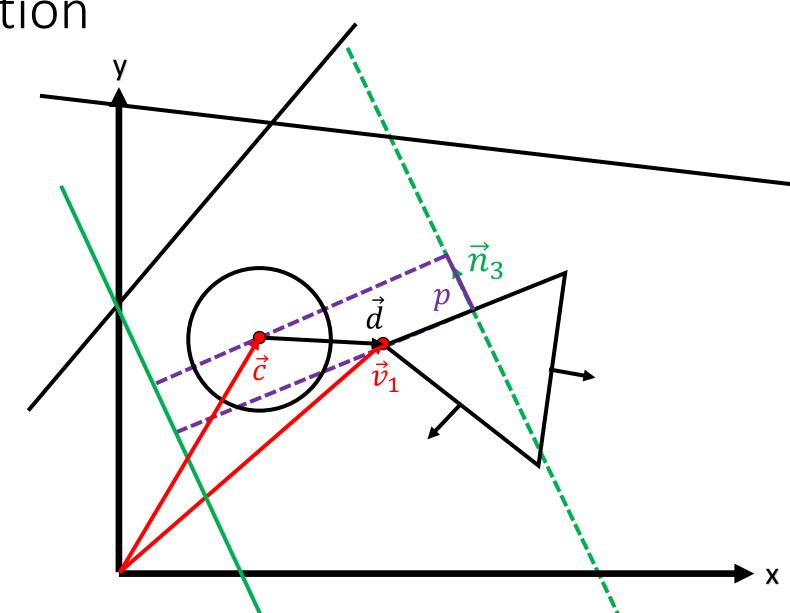




- 1. Collision detection
  - Separating Axis Test:

$$\vec{d} = \vec{v}_1 - \vec{c}$$

$$p = \vec{n}_3 \cdot \vec{d}$$





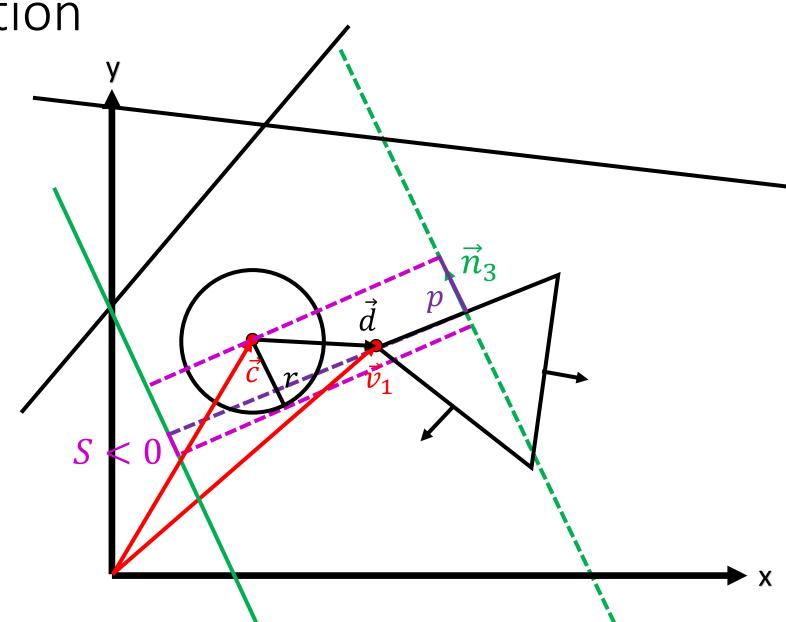
- 1. Collision detection
  - Separating Axis Test:

$$\vec{d} = \vec{v}_1 - \vec{c}$$

$$p = \vec{n}_3 \cdot \vec{d}$$

$$s = p - r$$

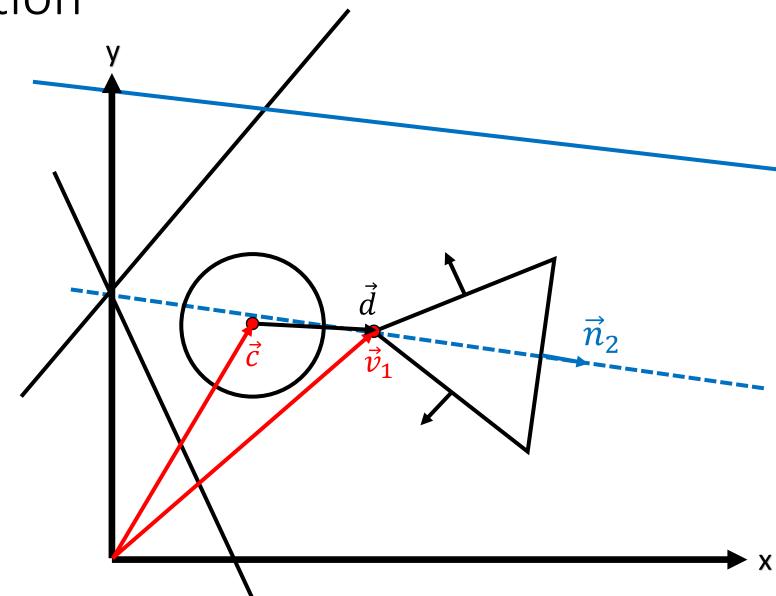
This normal is not a separating axis!





- 1. Collision detection
  - Separating Axis Test:

$$\vec{d} = \vec{v}_1 - \vec{c}$$

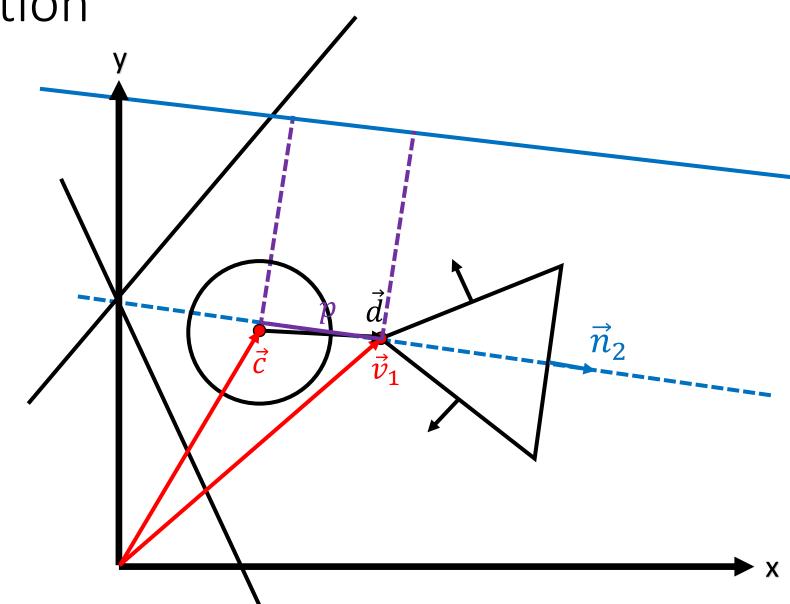




- 1. Collision detection
  - Separating Axis Test:

$$\vec{d} = \vec{v}_1 - \vec{c}$$

$$p = \vec{n}_2 \cdot \vec{d}$$





- 1. Collision detection
  - Separating Axis Test:

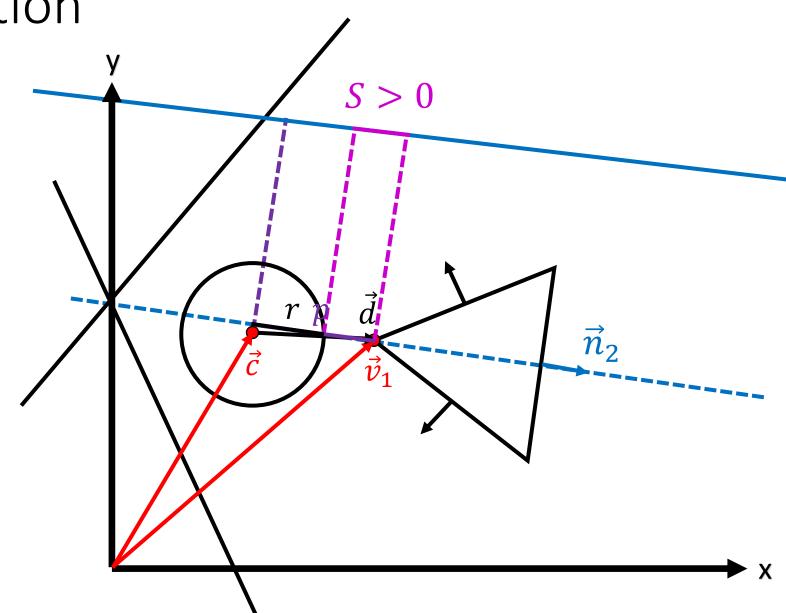
$$\vec{d} = \vec{v}_1 - \vec{c}$$

$$\vec{p} = \vec{n}_2 \cdot \vec{d}$$

$$s = p - r$$

This normal is separating axis!

No further testing is needed!



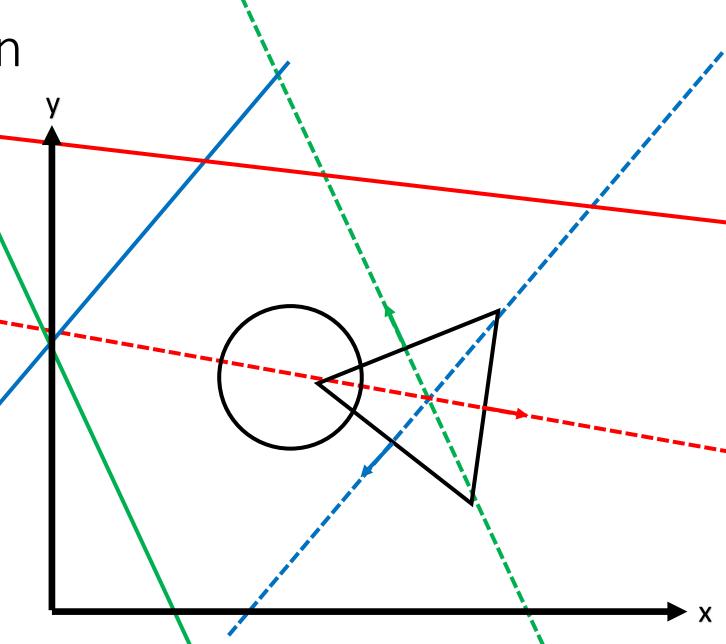


#### 1. Collision detection

• Separating Axis Test:

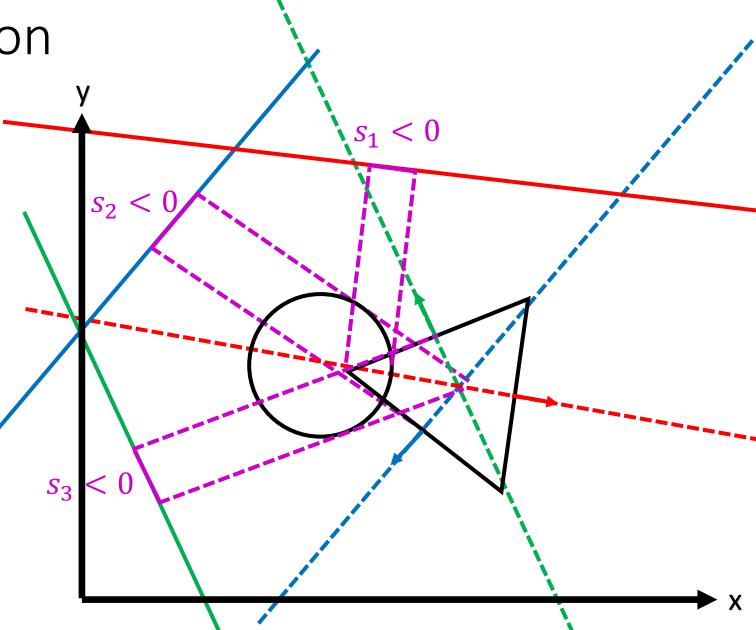
Separating axis test is performed on all the vertices of one body against all the vertices of the other body on all the convex hull normals of both bodies.

What if separating axis isn't found?



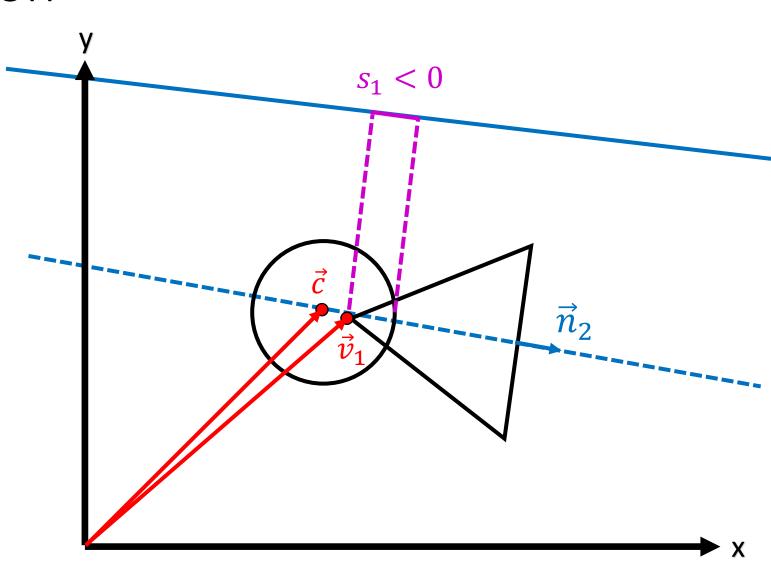


- 1. Collision detection
  - Separating Axis Test:
- 1. Negative separation most closer to 0 is used to describe the contact s (the one whose absolute value is the smallest).





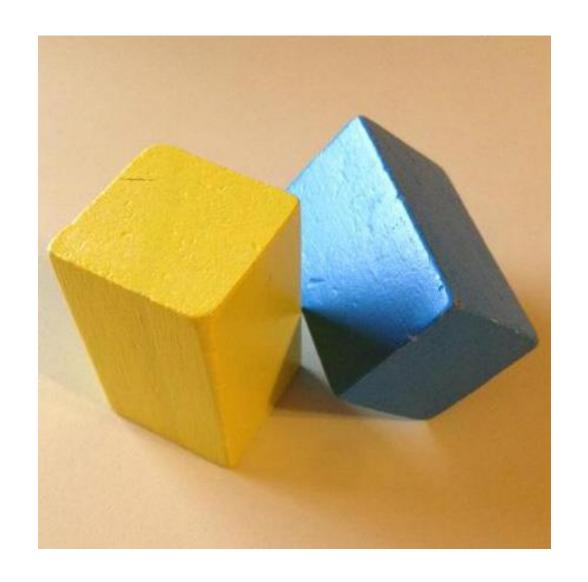
- 1. Collision detection
  - Separating Axis Test:
- 1. Negative separation most closer to 0 is used to describe the contact s (the one whose absolute value is the smallest).
- 2. Contact is described by:
  - contact normal (separating axis)
  - negative separation (penetration)
  - vertices of both bodies





- 1. Collision detection
  - Separating Axis Test:

In 3D space testing is needed against cross products of all the face normals too!





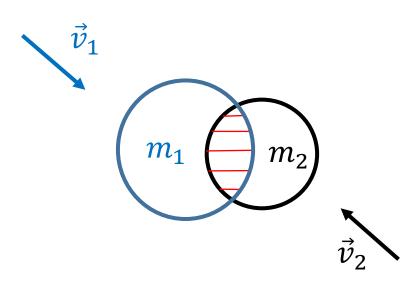
#### 1. Collision detection

- Separating Axis Test:
- + explicit geometrical approach to contact detection
- + everything needed to describe the contacts is calculated in the detection phase
- + as soon as the separating axis is found, further testing can be skipped (efficient in many cases).
- many special cases need to be considered, especially in 3D space



Constraint solving

Naive approach #1 (for simplicity sake, only particles are considered in this example):

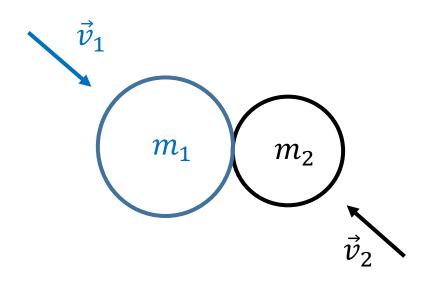




Constraint solving

Naive approach #1 (for simplicity sake, only particles are considered in this example):

1. position correction against contact normal





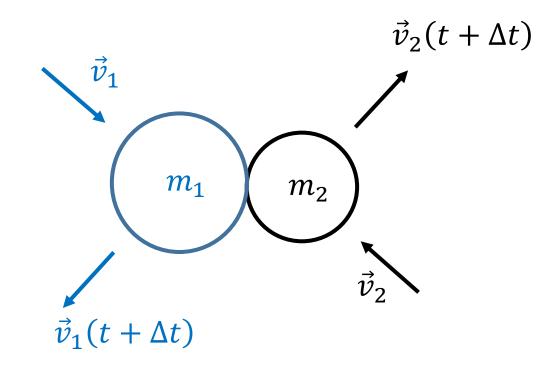
#### Constraint solving

Naive approach #1 (for simplicity sake, only particles are considered in this example):

- 1. position correction against contact normal
- 2. velocity update (conservation of momentum)

$$\vec{v}_1(t + \Delta t) = \frac{\vec{v}_1(m_1 - m_2) + 2m_2\vec{v}_2}{m_1 + m_2}$$

$$\vec{v}_2(t + \Delta t) = \frac{\vec{v}_2(m_2 - m_1) + 2m_1\vec{v}_1}{m_1 + m_2}$$

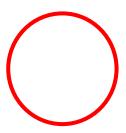


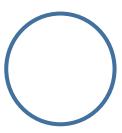


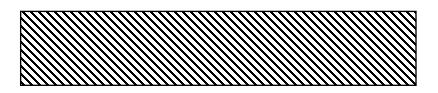
#### Constraint solving

```
while true
   applyExternalForces();
   integrate();
   if constraintsViolated()
       solveConstraints();
   end
   render();
   sleep();
```







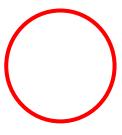


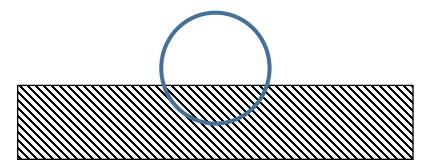


#### Constraint solving

```
while true
   applyExternalForces();
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   sleep();
end
```





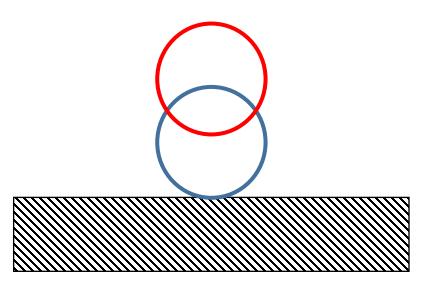




#### Constraint solving

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while true
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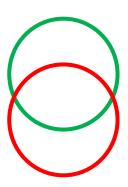


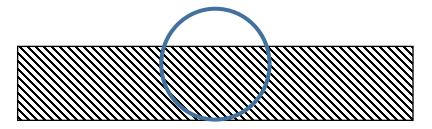




#### Constraint solving

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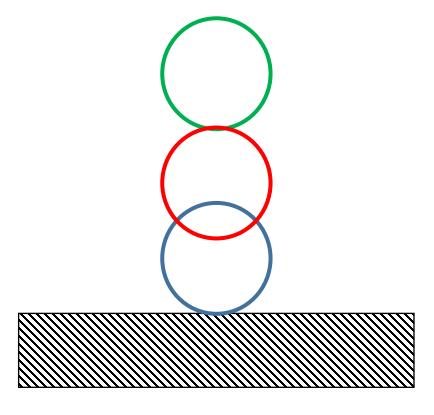






#### Constraint solving

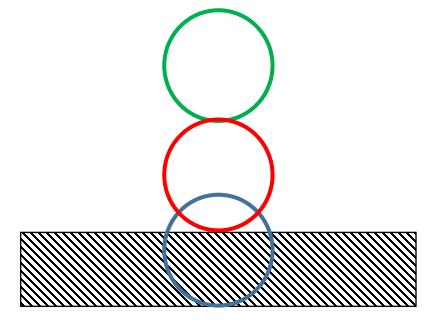
```
while true
    applyExternalForces();
    integrate();
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    end
    render();
    sleep();
end
```





#### Constraint solving

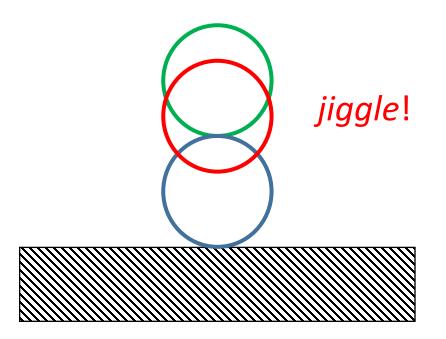
```
while true
   applyExternalForces();
   integrate();
   if constraintsViolated()
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   end
   render();
   sleep();
end
```





#### Constraint solving

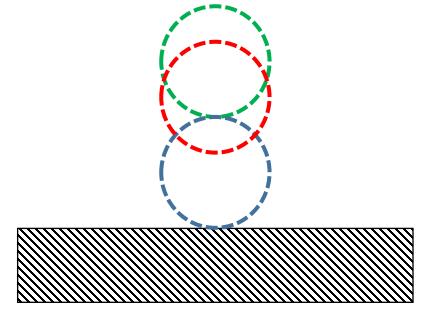
- + easy to implement
- + sufficient for occasional isolated contacts
- each constraint is solved separately so it can't support complex system of constraints
- constraint solving is performed at a position level inaccurate





#### Constraint solving

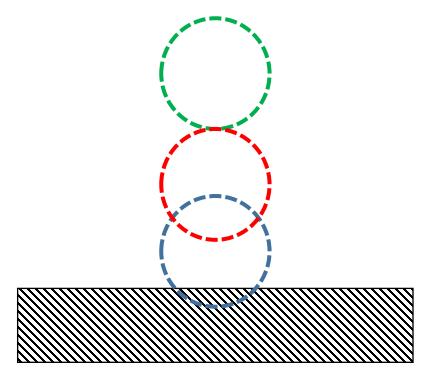
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while true
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```





#### Constraint solving

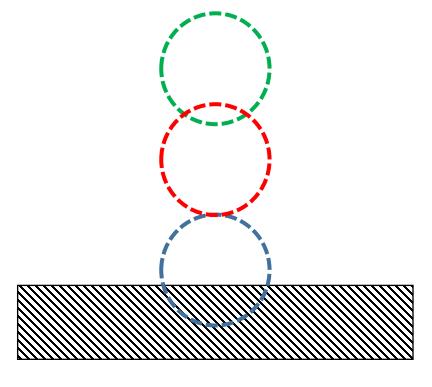
```
while true
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```





#### Constraint solving

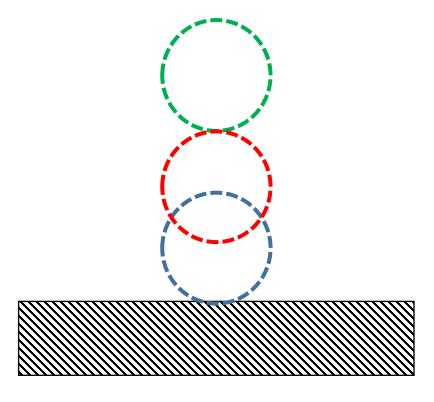
```
while true
   applyExternalForces();
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   sleep();
end
```





#### Constraint solving

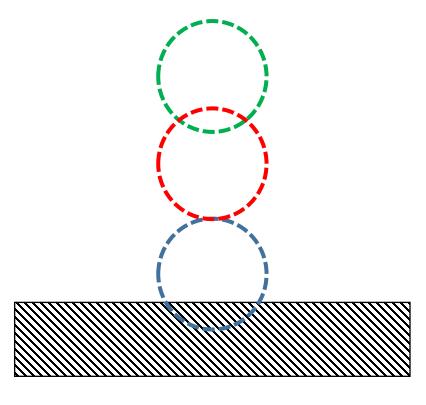
```
while true
   applyExternalForces();
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end
```





#### **Constraint solving**

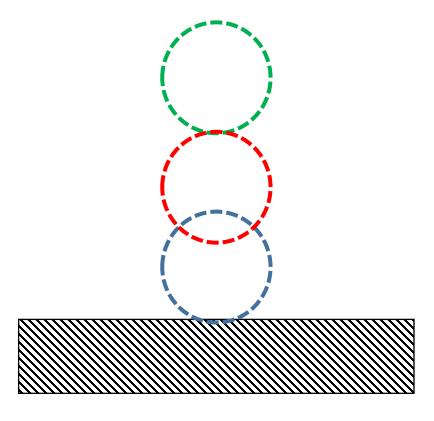
```
while true
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   render();
   sleep();
end
```





#### Constraint solving

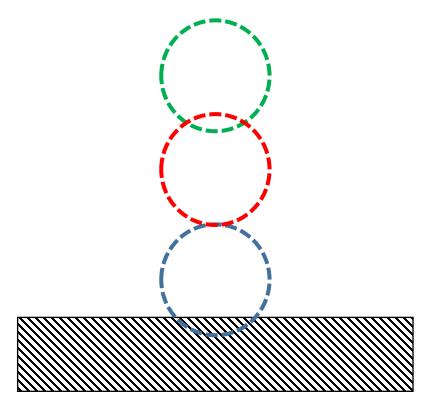
```
while true
   applyExternalForces();
   integrate();
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   end
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   sleep();
end
```





#### Constraint solving

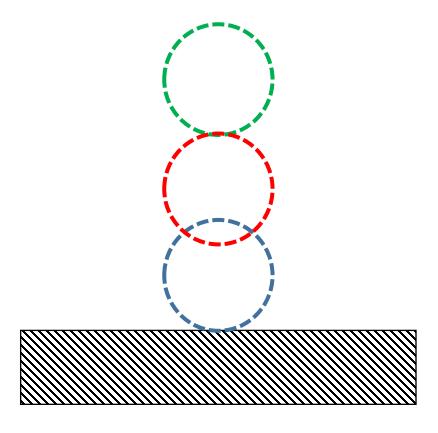
```
while true
   applyExternalForces();
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   end
   render();
   sleep();
end
```





#### Constraint solving

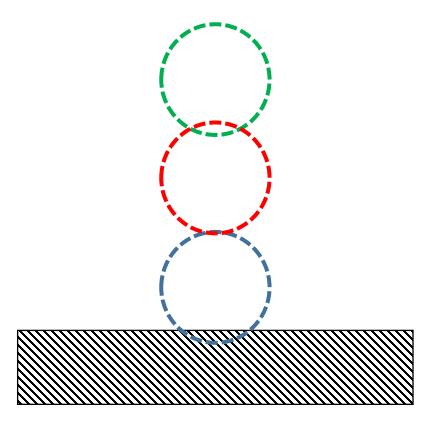
```
while true
   applyExternalForces();
   integrate();
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       solveConstraints();
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   render();
   sleep();
end
```





#### Constraint solving

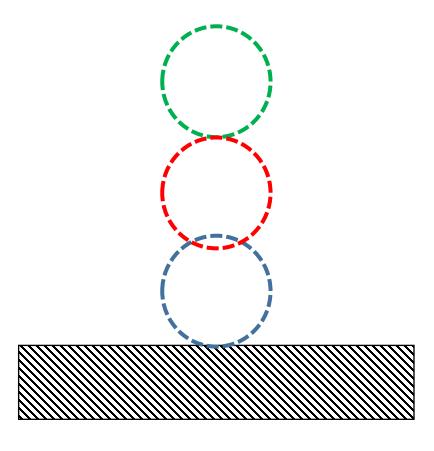
```
while true
   applyExternalForces();
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```





#### Constraint solving

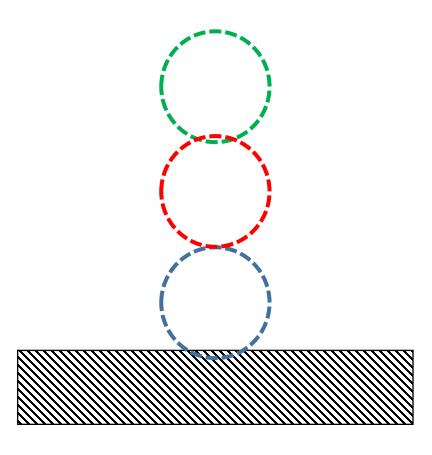
```
while true
   applyExternalForces();
   integrate();
   while constraintsViolated()
       solveConstraints();
   end
   render();
   sleep();
end
```





#### Constraint solving

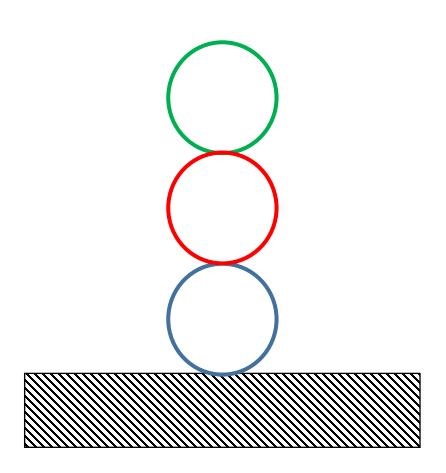
```
while true
   applyExternalForces();
   integrate();
   while constraintsViolated()
       solveConstraints();
   end
   render();
   sleep();
end
```





#### Constraint solving

- + easy to implement
- + solves more complex systems of constraints
- too slow (even for a small number of constraints)
- constraint solving is performed at a position level inaccurate

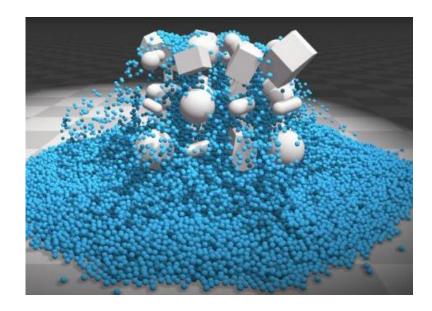




#### Constraint solving



- Define abstract model for all constraint types (so that all constraints are described in the same form)
- 2. Define system of linear equations to describe all the constraints together
- 3. Solve the system of linear equations (solve all the constraints together)
- 4. Use the solution to modify <u>forces and torques</u> which act on the bodies before the integration phase
- 5. During integration phase positions and rotations of bodies will be updated so that the constraints are satisfied afterwards





• Engine pipeline:

Apply external forces **Detect constraint violation** Solve constraints Integrate Render Sleep

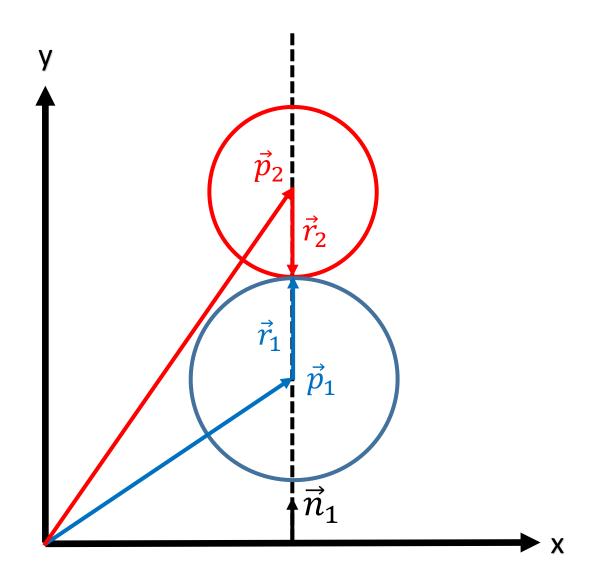


2. Constraint modeling (simple example)

Contact constraint for 2 particles (pairwise):

$$c(\vec{p}_1, \vec{p}_2) = 0$$

$$(\vec{p}_2 + \vec{r}_2 - (\vec{p}_1 + \vec{r}_1)) \cdot \vec{n}_1 = 0$$





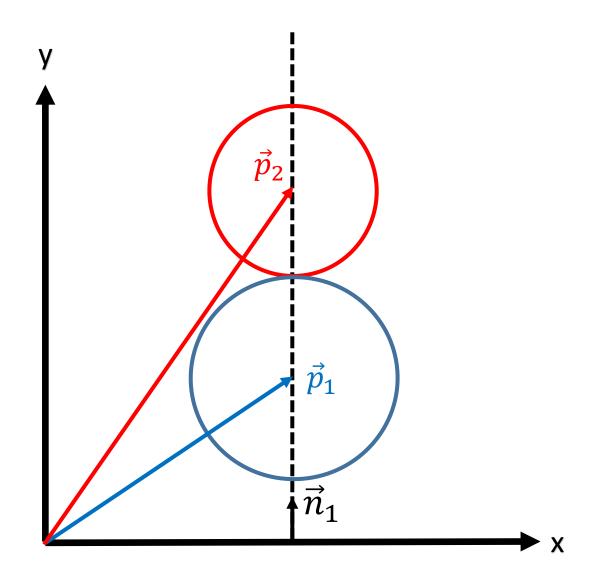
2. Constraint modeling (simple example)

In order for constraint to remain satisfied:

$$c(\vec{p}_1, \vec{p}_2) = 0$$

constraint derivative must be 0:

$$\frac{dc}{dt} = 0$$

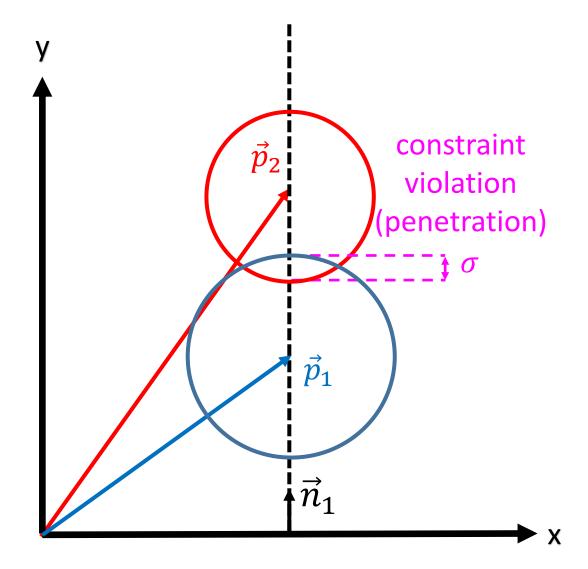




2. Constraint modeling (simple example)

However, after the detection phase constraints are already violated:

$$c(\vec{p}_1, \vec{p}_2) < 0$$





2. Constraint modeling (simple example)

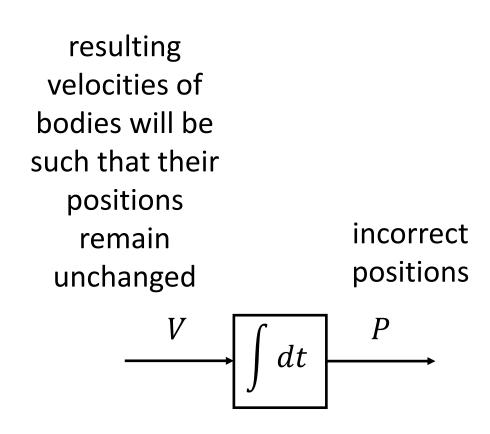
wrong assumption:

constraints are satisfied

$$c = 0$$

Solve:

$$\frac{dc}{dt} = 0$$





2. Constraint modeling (simple example)

Baumgarte stabilization (J. Baumgarte):

correct assumption:

constraints are violated

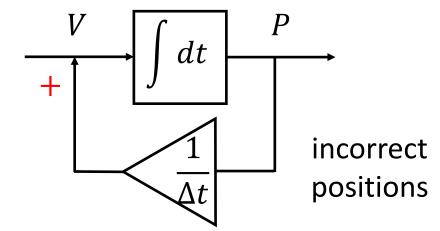
Solve:

$$\frac{dc}{dt} + \frac{c}{\Delta t} = 0$$

positive feedback loop

resulting
velocities of
bodies will be
such that their
positions are
modified

modified positions





2. Constraint modeling (simple example)

Baumgarte stabilization:

correct assumption:

constraints are violated

Solve:

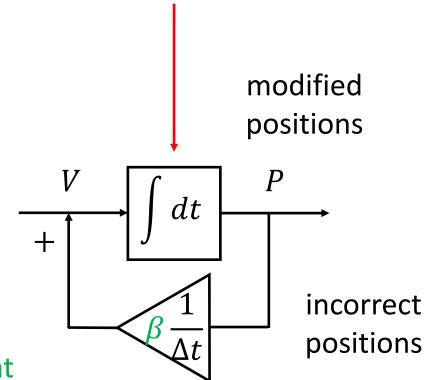
$$\frac{dc}{dt} + \beta \frac{c}{\Delta t} = 0$$

limit introducing the additional energy

$$0 \le \beta \le 1$$

predetermined constant

numerical integration inherently causes errors, thus the positions will not be accurately updated from velocities (numerical drift), which introduces additional energy into the system





## 2. Constraint modeling (simple example)

$$c(\vec{p}_1, \vec{p}_2) = (\vec{p}_2 + \vec{r}_2 - (\vec{p}_1 + \vec{r}_1)) \cdot \vec{n}_1$$

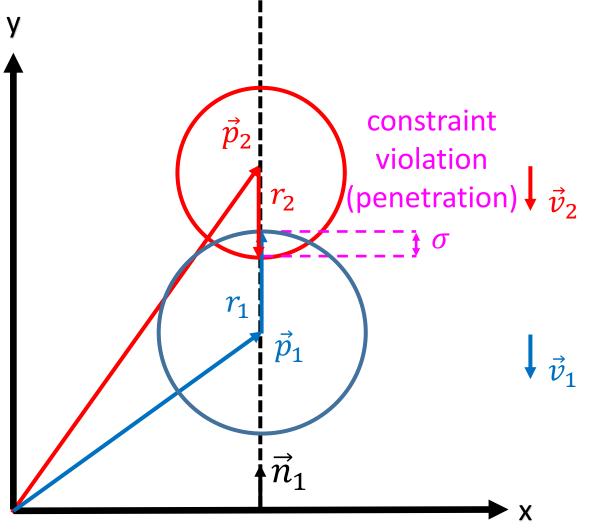
Solve:

$$\frac{dc}{dt} + \beta \frac{c}{\Delta t} = 0$$

$$\frac{dc}{dt} = -\beta \frac{c}{\Delta t}$$
normal is constant
$$\frac{d}{dt} \left( (\vec{p}_2 + \vec{r}_2 - (\vec{p}_1 + \vec{r}_1)) \cdot \vec{n}_1 \right) = -\beta \frac{c}{\Delta t}$$

$$\frac{d}{dt} (\vec{p}_2 + \vec{r}_2 - (\vec{p}_1 + \vec{r}_1)) \cdot \vec{n}_1 + (\vec{p}_2 + \vec{r}_2 - (\vec{p}_1 + \vec{r}_1)) \cdot \frac{d}{dt} \vec{n}_1 = -\beta \frac{c}{\Delta t}$$

$$\left( \frac{d}{dt} \vec{p}_2 + \frac{d}{dt} \vec{r}_2 - \frac{d}{dt} \vec{p}_1 - \frac{d}{dt} \vec{r}_1 \right) \cdot \vec{n}_1 = -\beta \frac{c}{\Delta t}$$



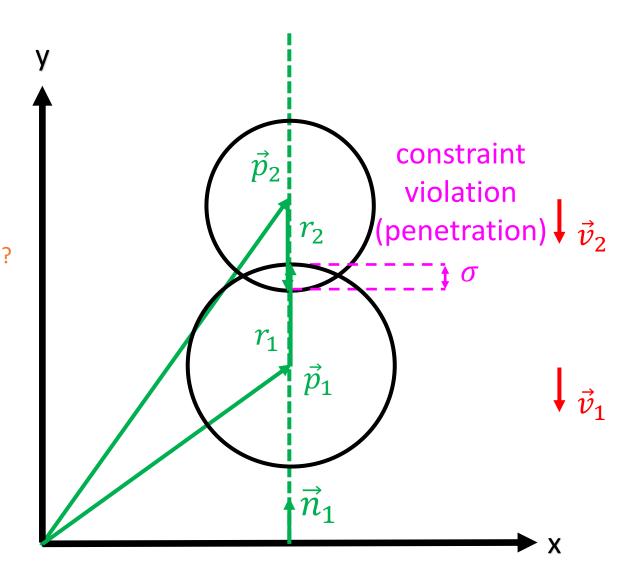


## 2. Constraint modeling (simple example)

$$c(\vec{p}_1, \vec{p}_2) = (\vec{p}_2 + \vec{r}_2 - (\vec{p}_1 + \vec{r}_1)) \cdot \vec{n}_1$$

Solve:

$$-\vec{v}_1 \cdot \vec{n}_1 + \vec{v}_2 \cdot \vec{n}_1 = -\beta \frac{c}{\Delta t}$$
unknown
known





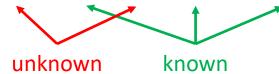
2. Constraint modeling (simple example)

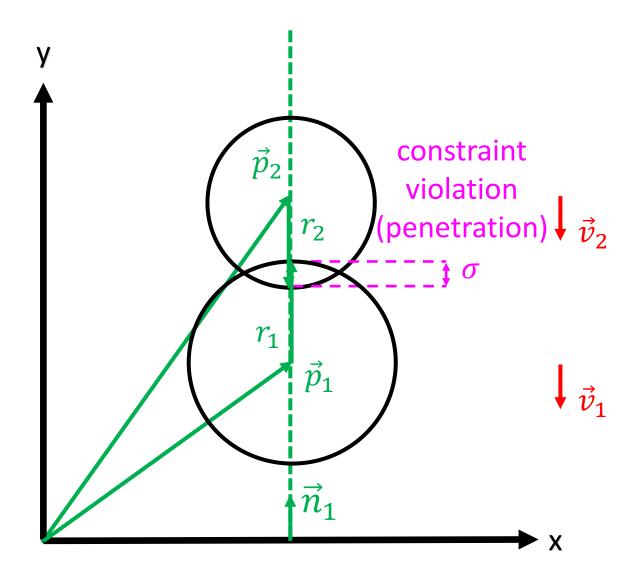
$$\sigma = (\vec{p}_2 + \vec{r}_2 - (\vec{p}_1 + \vec{r}_1)) \cdot \vec{n}_1$$

Solve:

found in detection phase

$$-\vec{v}_1 \cdot \vec{n}_1 + \vec{v}_2 \cdot \vec{n}_1 = -\beta \frac{\delta}{\Delta t}$$





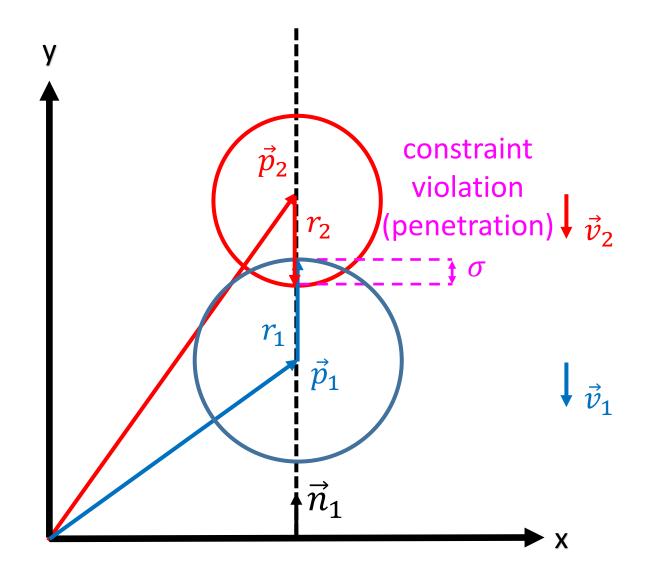


## 2. Constraint modeling (simple example)

$$c(\vec{p}_1, \vec{p}_2) = (\vec{p}_2 + \vec{r}_2 - (\vec{p}_1 + \vec{r}_1)) \cdot \vec{n}_1$$

vector form:

$$-\vec{v}_1 \cdot \vec{n}_1 + \vec{v}_2 \cdot \vec{n}_1 = -\beta \frac{\sigma}{\Delta t}$$
 variable form depending on the constraints 
$$[-\vec{n}_1 \quad \vec{n}_1] \begin{bmatrix} \vec{v}_1 \\ \vec{v}_2 \end{bmatrix} = -\beta \frac{\sigma}{\Delta t}$$
 same form always 
$$jv = -\beta \frac{\sigma}{\Delta t}$$

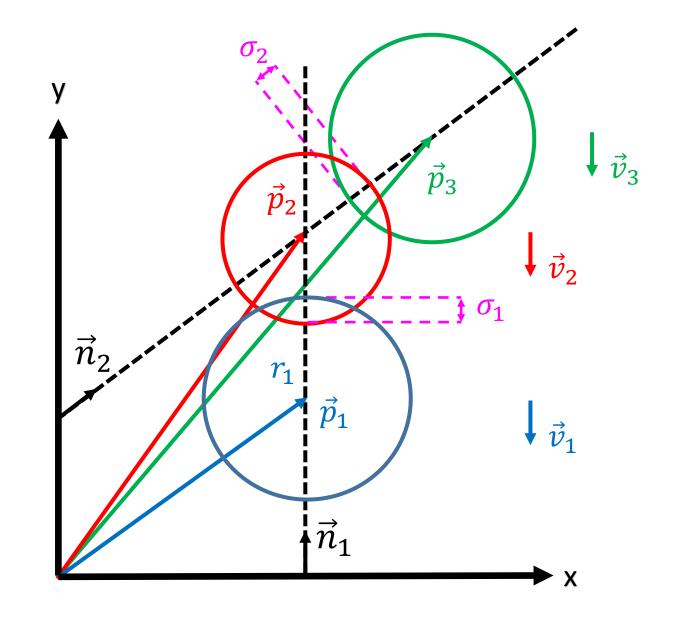




Constraint modeling (simple example)

For multiple contacts:

$$\begin{bmatrix} -\vec{n}_1 & \vec{n}_1 \end{bmatrix} \begin{bmatrix} \vec{v}_1 \\ \vec{v}_2 \end{bmatrix} = -\beta \frac{\sigma_1}{\Delta t}$$
$$\begin{bmatrix} -\vec{n}_2 & \vec{n}_2 \end{bmatrix} \begin{bmatrix} \vec{v}_2 \\ \vec{v}_3 \end{bmatrix} = -\beta \frac{\sigma_2}{\Delta t}$$
$$\vdots$$





2. Constraint modeling (simple example)

System of first order constraint derivatives:

$$\begin{aligned} \left[ -\vec{n}_1 \quad \vec{n}_1 \right] \begin{bmatrix} \vec{v}_1 \\ \vec{v}_2 \end{bmatrix} &= -\beta \frac{\sigma_1}{\Delta t} \\ \left[ -\vec{n}_2 \quad \vec{n}_2 \right] \begin{bmatrix} \vec{v}_2 \\ \vec{v}_3 \end{bmatrix} &= -\beta \frac{\sigma_2}{\Delta t} \\ &\vdots \\ \left[ -\vec{n}_n \quad \vec{n}_n \right] \begin{bmatrix} \vec{v}_n \\ \vec{v}_{n+1} \end{bmatrix} &= -\beta \frac{\sigma_n}{\Delta t} \end{aligned}$$



2. Constraint modeling (simple example)

Expanded to include all the space dimensions:

$$\begin{bmatrix} -n_{1x} & -n_{1y} & n_{1x} & n_{1y} \end{bmatrix} \begin{bmatrix} v_{1x} \\ v_{1y} \\ v_{2x} \\ v_{2y} \end{bmatrix} = -\beta \frac{\sigma_1}{\Delta t}$$

$$\begin{bmatrix} -n_{2x} & -n_{2y} & n_{2x} & n_{2y} \end{bmatrix} \begin{bmatrix} v_{2x} \\ v_{2y} \\ v_{3x} \\ v_{3y} \end{bmatrix} = -\beta \frac{\sigma_2}{\Delta t}$$

$$\begin{bmatrix} -n_{nx} & -n_{ny} & n_{nx} & n_{ny} \end{bmatrix} \begin{bmatrix} v_{nx} \\ v_{ny} \\ v_{(n+1)x} \\ v_{(n+1)y} \end{bmatrix} = -\beta \frac{\sigma_n}{\Delta t}$$



#### 2. Constraint modeling (simple example)

Jacobian

$$\begin{bmatrix} v_{1x} & v_{1y} & v_{2x} & v_{2y} \dots \\ -n_{1x} & -n_{1y} & n_{1x} & n_{1y} & 0 & 0 \\ 0 & 0 & -n_{2x} & -n_{2y} & n_{2x} & n_{2y} \\ \vdots & & & \ddots & & \vdots \\ & & & & 0 & 0 & -n_{nx} & -n_{ny} & n_{nx} & n_{ny} \end{bmatrix} \begin{bmatrix} v_{1x} \\ v_{1y} \\ v_{2x} \\ v_{2y} \\ \vdots \\ v_{nx} \\ v_{ny} \end{bmatrix} = -\beta \frac{1}{\Delta t} \begin{bmatrix} \sigma_1 \\ \sigma_2 \\ \vdots \\ \sigma_{n-1} \\ \sigma_n \end{bmatrix}$$

$$JV = -\beta \frac{1}{\Delta t} S$$

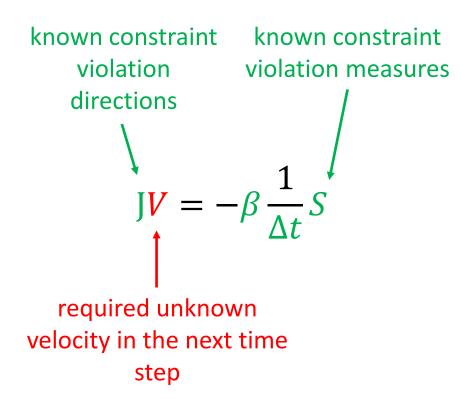
constraint

velocities violation

In order for constraint to <u>be solved</u> velocity along the constraint direction must cancel out constraint violation.



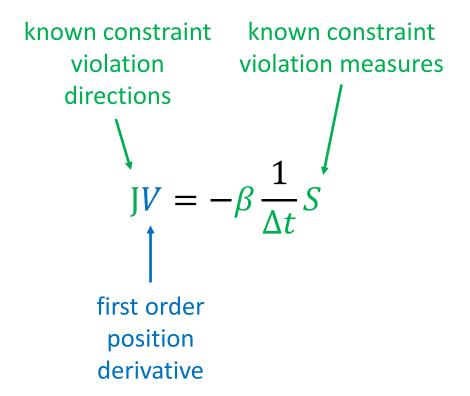
#### 3. Constraint solving:



Final solution is the constraint force  $F_c$  which will cancel out the constraint violation!



#### 3. Constraint solving:



$$\frac{d^2\vec{p}(t)}{dt^2} = \frac{\vec{F}(t)}{m}$$
$$\frac{d\vec{v}(t)}{dt} = \frac{\vec{F}(t)}{m}$$

matrix form:

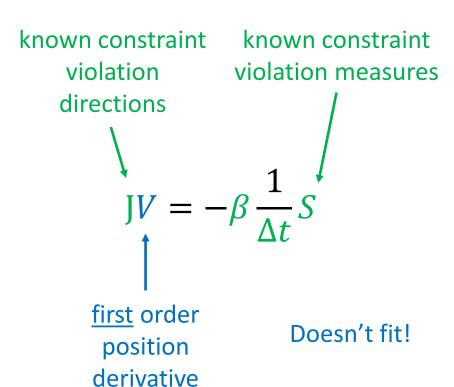
2nd Newton's Law of Motion!

$$\frac{dV}{dt} = \frac{1}{M}F \leftarrow \text{total force}$$

second order position derivative



#### 3. Constraint solving:



# $\frac{d^2\vec{p}(t)}{dt^2} = \frac{\vec{F}(t)}{m}$ $\frac{d\vec{v}(t)}{dt} = \frac{\vec{F}(t)}{m}$

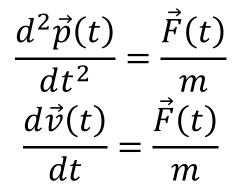
#### matrix form:

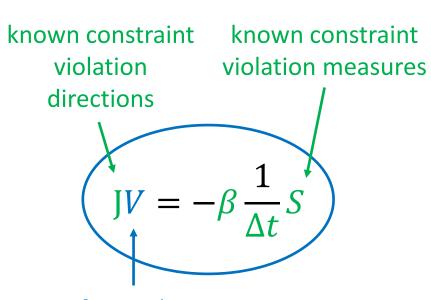
$$\frac{dV}{dt} = \frac{1}{M}F$$

$$\frac{dV}{dt} = \frac{1}{M}(F_{ext} + F_c) \leftarrow \begin{array}{c} \text{required} \\ \text{constraint forces} \\ \text{second} \\ \text{order} \end{array} \quad \text{known}$$
order masses external forces
position
derivative



#### 3. Constraint solving:





first order Solution #1:

position differentiate both sides once more Result: complicated!

#### matrix form:

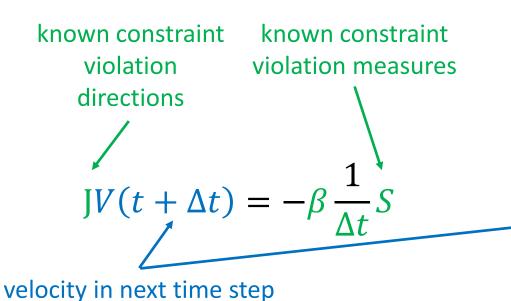
$$\frac{dV}{dt} = \frac{1}{M}F$$

$$\frac{dV}{dt} = \frac{1}{M}(F_{ext} + F_c) \leftarrow \begin{array}{c} \text{required} \\ \text{constraint forces} \\ \text{second} \\ \text{order} \end{array} \quad \text{known}$$
order masses external forces
position
derivative



## $\frac{d^2\vec{p}(t)}{dt^2} = \frac{\vec{F}(t)}{m}$ $\frac{d\vec{v}(t)}{dt} = \frac{\vec{F}(t)}{m}$

#### 3. Constraint solving:



matrix form:

$$\frac{dV}{dt} = \frac{1}{M}F$$

$$\frac{V(t + \Delta t) - V}{\Delta t} = \frac{1}{M}(F_{ext} + F_c) \leftarrow \text{required constraint force}$$

$$\frac{1}{M}(F_{ext} + F_c) \leftarrow \text{required constraint force}$$

$$\frac{1}{M}(F_{ext} + F_c) \leftarrow \text{required constraint force}$$

forces

Solution #2: Finite difference approximation!

known known velocity masses in current time step



#### 3. Constraint solving:

$$V(t + \Delta t) = -\beta \frac{1}{dt} J^T S$$

$$\frac{d^2\vec{p}(t)}{dt^2} = \frac{\vec{F}(t)}{m}$$
$$\frac{d\vec{v}(t)}{dt} = \frac{\vec{F}(t)}{m}$$

#### matrix form:

$$\frac{dV}{dt} = \frac{1}{M}F$$

$$V(t + \Delta t) - V = \Delta t \frac{1}{M} (F_{ext} + F_c)$$



3. Constraint solving:

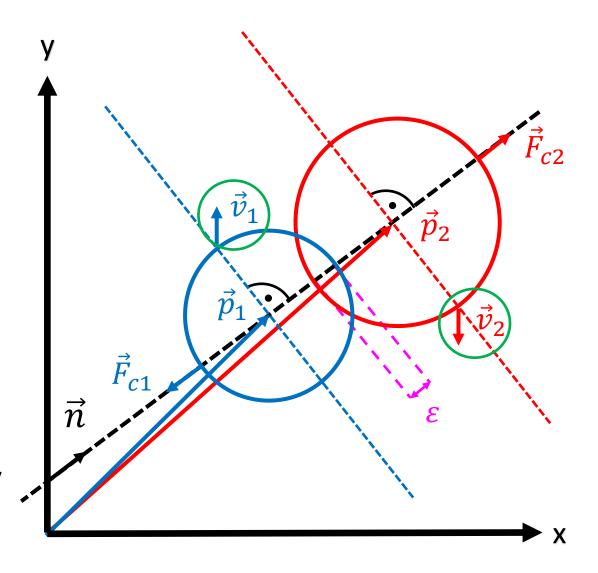
$$-\beta \frac{1}{\Delta t} J^T S - V = \Delta t \frac{1}{M} (F_{ext} + F_c)$$



## 3. Constraint solving: Constraint forces:

$$F_c \perp V(t + \Delta t)$$

- they cancel out fragments of current velocities which would further violate constraints
- they do not act along the directions of velocities in next time step (admissible velocities), but perpendicular to them
- they do not introduce additional energy into the system (they perform virtual work)

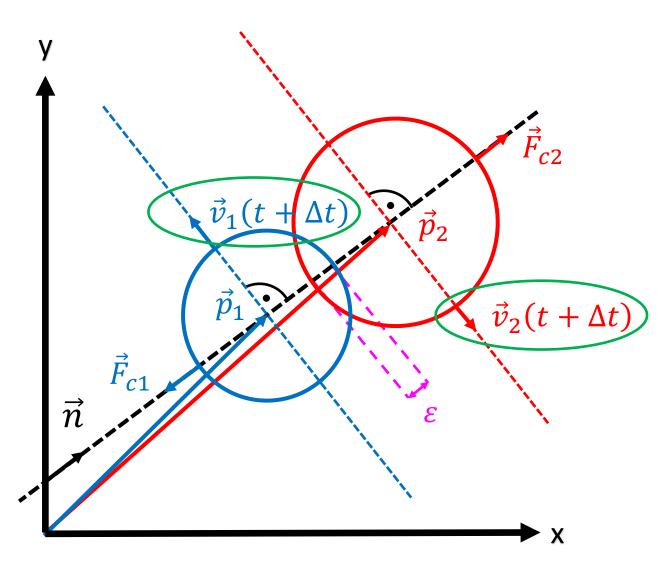




## 3. Constraint solving: Constraint forces:

$$F_c \perp V(t + \Delta t)$$

- they cancel out fragments of current velocities which would further violate constraints
- they do not act along the directions of velocities in next time step (admissible velocities), but perpendicular to them
- they do not introduce additional energy into the system (they perform virtual work)





#### 3. Constraint solving:

Jacobian rows are perpendicular to corresponding admissible velocities

$$\frac{dc}{dt} = 0$$

$$JV(t + \Delta t) = 0$$

#### Considering:

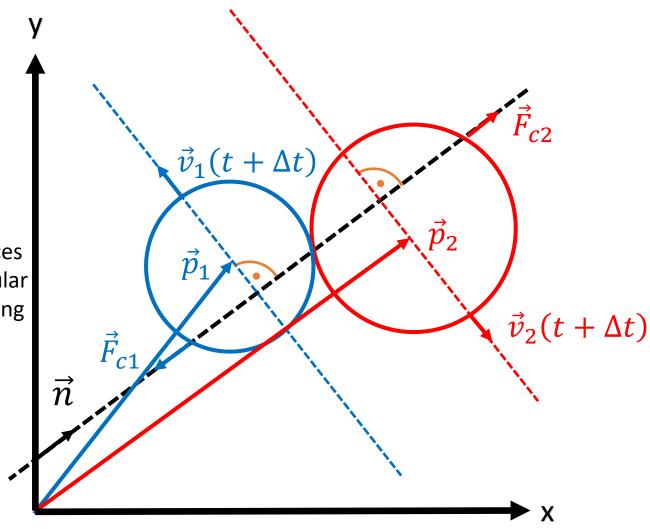
 $F_c \perp V(t + \Delta t) \leftarrow$  to corresponding

constraint forces
are perpendicular
to corresponding
admissible
velocities

the following is then true:

constraint forces must be parallel to corresponding rows of  $J^T$  (must have same directions)

$$F_{c} = J^{T} \lambda$$
known unknown direciton magnitudes





3. Constraint solving:

$$-\beta \frac{1}{\Delta t} J^T S - V = \Delta t \frac{1}{M} (F_{ext} + F_c)$$



3. Constraint solving:

$$-\beta \frac{1}{\Delta t} J^T S - V = \Delta t \frac{1}{M} (F_{ext} + J^T \lambda)$$



#### 3. Constraint solving:

$$-\beta \frac{1}{\Delta t} J^T S - V = \Delta t \frac{1}{M} (F_{ext} + J^T \lambda)$$

$$-\beta \frac{1}{\Delta t} J^T S - V = \Delta t \frac{1}{M} F_{ext} + \Delta t \frac{1}{M} J^T \lambda$$

$$\Delta t \frac{1}{M} J^T \lambda = -\beta \frac{1}{\Delta t} J^T S - V - \Delta t \frac{1}{M} F_{ext}$$

$$J \frac{1}{M} J^T \lambda = -\beta \frac{1}{\Delta t^2} S - \frac{1}{\Delta t} J V - J \frac{1}{M} F_{ext}$$



3. Constraint solving:

$$J\frac{1}{M}J^{T}\lambda = -\beta \frac{1}{\Delta t^{2}}S - \frac{1}{\Delta t}JV - J\frac{1}{M}F_{ext}$$





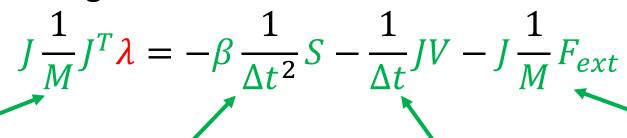
3. Constraint solving:

$$J\frac{1}{M}J^{T}\lambda = -\beta \frac{1}{\Delta t^{2}}S - \frac{1}{\Delta t}JV - J\frac{1}{M}F_{ext}$$

What is known and what is unknown?



#### 3. Constraint solving:



effective inverse mass

constraints resolution accelerations

accelerations needed to prevent further violation of constraints as a consequence of existing momentums

accelerations needed to prevent further violation of constraints as a consequence of applied external forces

$$\frac{1}{kg}$$

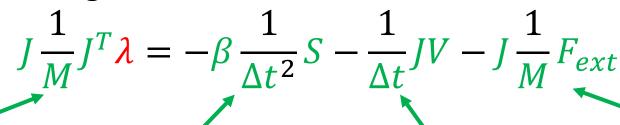
$$\frac{m}{s^2}$$

$$\frac{m}{s^2}$$

$$\frac{1}{ka}\frac{kg \cdot m}{s^2} = \frac{m}{s^2}$$



#### 3. Constraint solving:



effective inverse mass

constraints resolution accelerations

accelerations needed to prevent further violation of constraints as a consequence of existing momentums

accelerations needed to prevent further violation of constraints as a consequence of applied external forces

$$\frac{1}{kg}$$

matrix  $n \times n!$ 

$$\frac{m}{s^2}$$

vector  $n \times 1!$ 

$$\frac{m}{s^2}$$

vector  $n \times 1!$ 

$$\frac{1}{kg}\frac{kg \cdot m}{s^2} = \frac{m}{s^2}$$

vector  $n \times 1!$ 



#### 3. Constraint solving:

$$J\frac{1}{M}J^{T}\lambda = -\beta \frac{1}{\Delta t^{2}}S - \frac{1}{\Delta t}JV - J\frac{1}{M}F_{ext}$$

$$A\lambda = b$$



#### 3. Constraint solving:

$$J\frac{1}{M}J^{T}\lambda = -\beta \frac{1}{\Delta t^{2}}S - \frac{1}{\Delta t}JV - J\frac{1}{M}F_{ext}$$

$$A\lambda = b$$
$$\lambda = A \setminus b$$

numerical solution of system of linear equations



#### 3. Constraint solving:

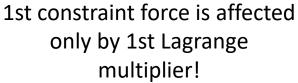
$$J\frac{1}{M}J^{T}\lambda = -\beta \frac{1}{\Delta t^{2}}S - \frac{1}{\Delta t}JV - J\frac{1}{M}F_{ext}$$

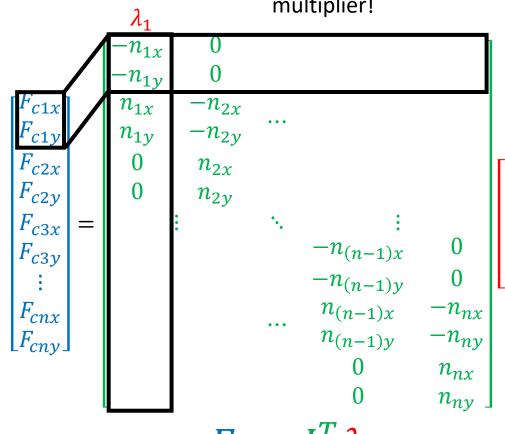
$$A\lambda = b$$
$$\lambda = A \setminus b$$

$$F_c = J^T \lambda$$



3. Constraint solving:





Lagrange multipliers (constraint forces' magnitudes)

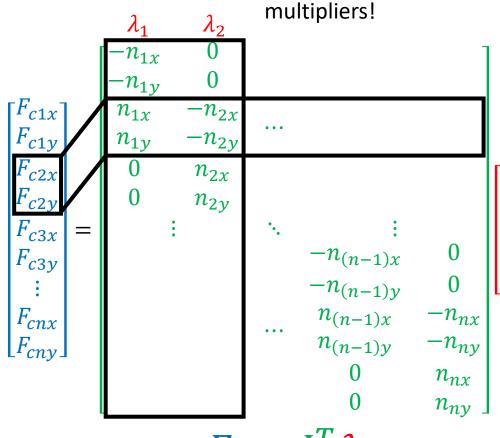
They are the solution of this system!

$$F_c = J^T \lambda$$



2nd constraint force is affected by first 2 Lagrange

3. Constraint solving:



Lagrange multipliers (constraint forces' magnitudes)

They are the solution of this system!

$$F_c = J^T \lambda$$



#### 3. Constraint solving:

$$J\frac{1}{M}J^{T}\lambda = -\beta \frac{1}{\Delta t^{2}}S - \frac{1}{\Delta t}JV - J\frac{1}{M}F_{ext}$$

$$A\lambda = b$$

$$\lambda = A \setminus b$$
numerical solution of
$$F_{c} = J^{T}\lambda$$
system of linear equations

integration:

$$\vec{v}_i = \vec{v}_{i-1} + \Delta t \frac{\vec{F}_{ext} + \vec{F}_c}{m}$$
$$\vec{p}_i = \vec{p}_{i-1} + \Delta t \vec{v}_i$$

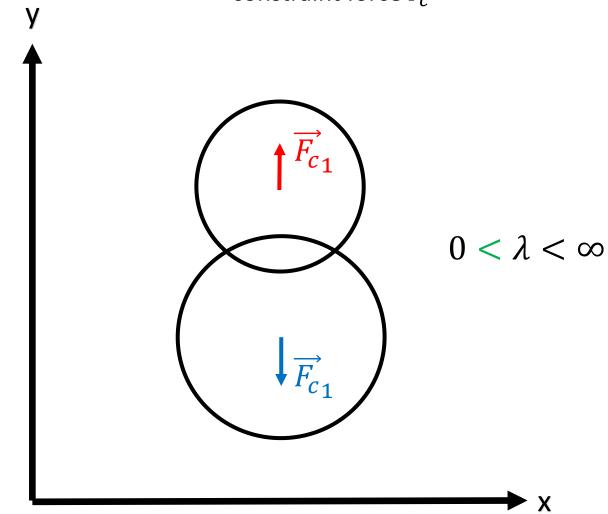


#### 3. Constraint solving:

**Equality constraints:** 

$$c(\vec{p}_1, \vec{p}_2) = 0$$

resulting positive (pushing) constraint force  $F_c$ 



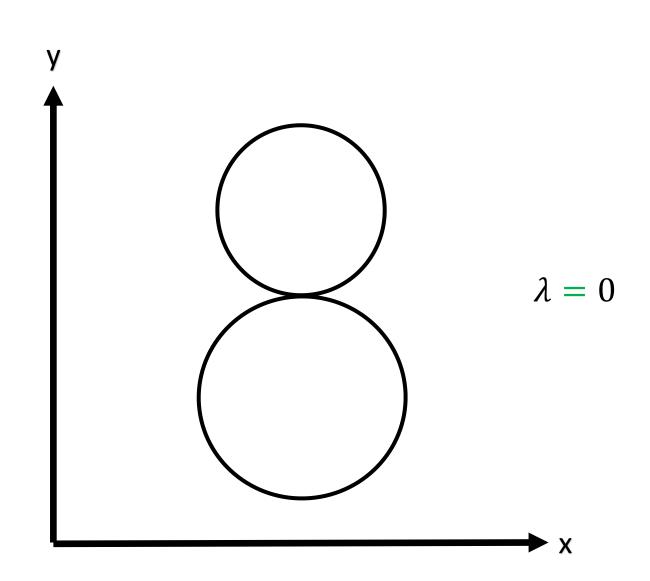


#### 3. Constraint solving:

**Equality constraints:** 

$$c(\vec{p}_1, \vec{p}_2) = 0$$

Penetration resolved!



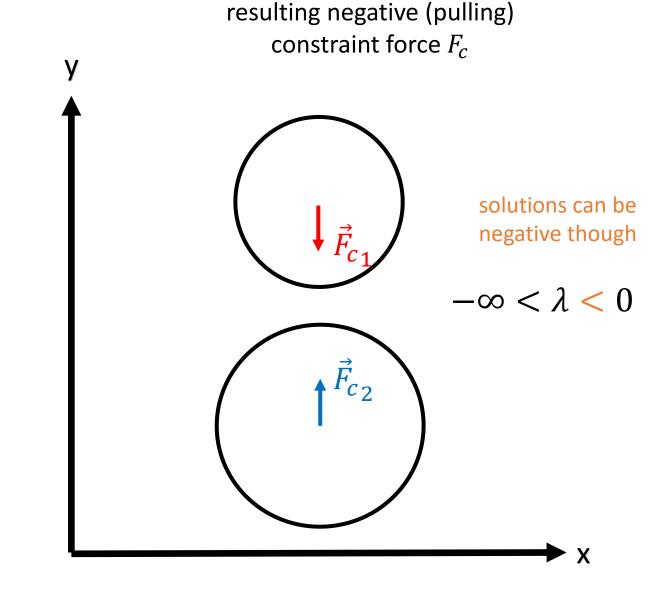


#### 3. Constraint solving:

Equality constraints:

$$c(\vec{p}_1, \vec{p}_2) = 0$$

However separation is prevented too!



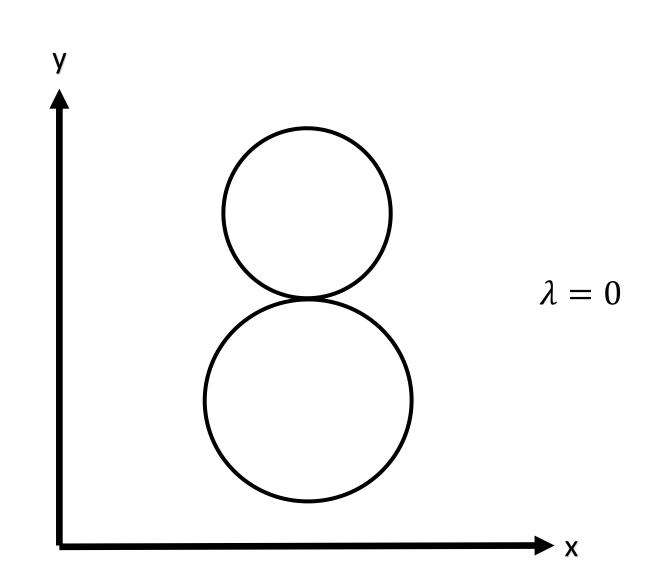


#### 3. Constraint solving:

Equality constraints:

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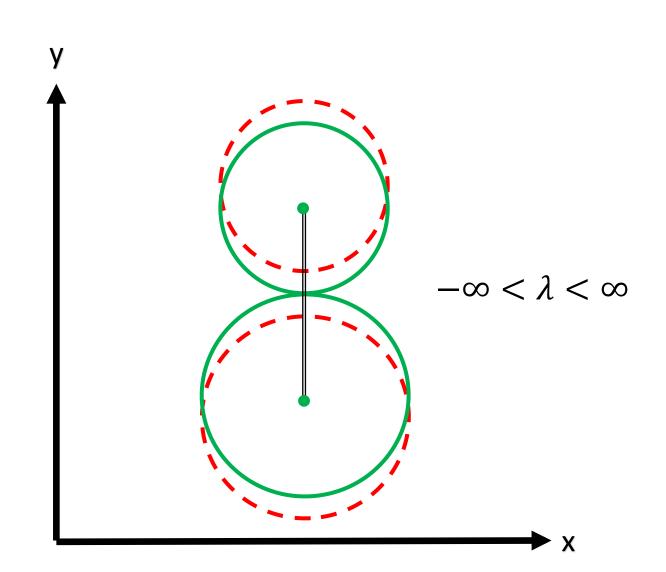


3. Constraint solving:

**Equality constraints:** 

$$c(\vec{p}_1, \vec{p}_2) = 0$$

e. g. distance joints!



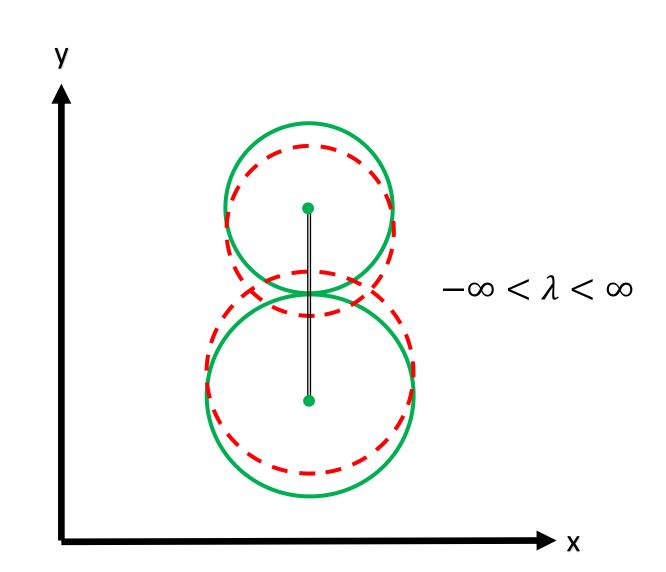


#### 3. Constraint solving:

**Equality constraints:** 

$$c(\vec{p}_1, \vec{p}_2) = 0$$

e.g. distance joints!



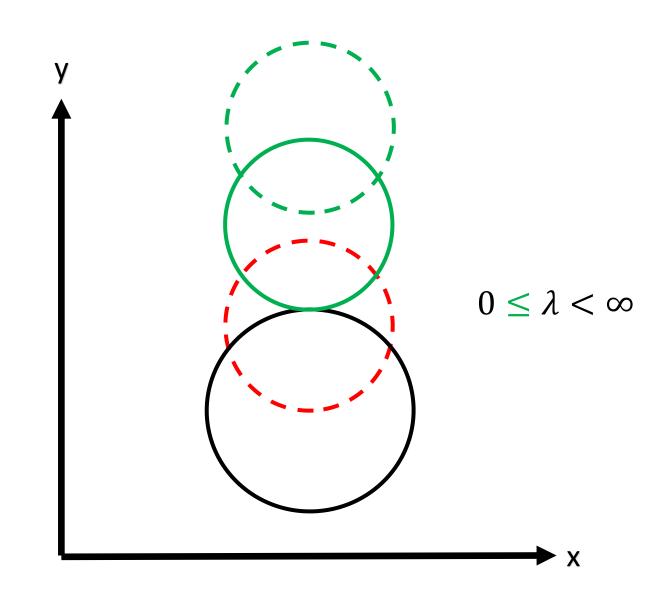


#### 3. Constraint solving:

Inequality constraints:

$$c(\vec{p}_1, \vec{p}_2) \ge 0$$

e.g. contact constraints!



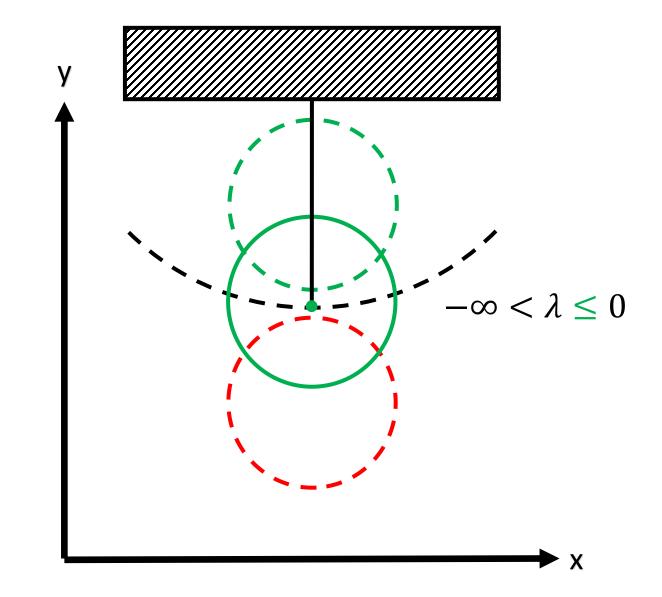


#### 3. Constraint solving:

Inequality constraints:

$$c(\vec{p}_1, \vec{p}_2) \le 0$$

e.g. pendulums!





#### 3. Constraint solving:

• System of linear equations

$$J\frac{1}{M}J^{T}\lambda = -\beta \frac{1}{\Delta t^{2}}S - \frac{1}{\Delta t}JV - J\frac{1}{M}F_{ext}$$
$$A\lambda = b$$

can only solve system of equality constraints!



#### 3. Constraint solving:

Equality: c(...) = 0

Inequality type  $\geq$ :  $c(...) \geq 0$ 

Inequality type  $\leq$ :  $c(...) \leq 0$ 



3. Constraint solving:

Equality:  $A\lambda = b$ 

Inequality type  $\geq$ :  $A\lambda \geq b$ 

Inequality type  $\leq$ :  $A\lambda \leq b$ 



- 3. Constraint solving:
  - I. transform inequalities into equalities

```
required (attempts to match F_c) slack variable (used to describe inequalities as equalities)
```

Equality:  $A\lambda + F_c = b$   $F_c = 0$ 

Inequality type  $\geq$ :  $A\lambda + F_c = b$   $F_c \leq 0$ 

Inequality type  $\leq$ :  $A\lambda + F_c = b$   $F_c \geq 0$ 



- 3. Constraint solving:
  - I. transform inequalities into equalities

```
found (matches F_c) slack variable (is no longer needed)

Equality: A\lambda + F_c = b \quad F_c = 0 \quad -\infty \leq \lambda \qquad \leq \infty
Inequality type \geq: A\lambda + F_c = b \quad F_c = 0 \qquad 0 \leq \lambda \leq \infty
Inequality type \leq: A\lambda + F_c = b \quad F_c = 0 \quad -\infty \leq \lambda \leq 0
```

• Constraints are satisfied if and only if  $\lambda$  is within corresponding limits (no additional  $F_c$  is needed to satisfy them)



- 3. Constraint solving:
  - I. transform inequalities into equalities
  - II. generalize  $\lambda$  limits to make them parametrized

• Constraints are satisfied if and only if  $\lambda$  is within corresponding limits (no additional  $F_c$  is needed to satisfy them)



has reached limits

- 3. Constraint solving:
  - I. transform inequalities into equalities
  - II. generalize  $\lambda$  limits to make them parametrized
  - III. complementarity

```
(unable to match F_c) slack variable (is needed once more)

Equality: A\lambda + F_c = b F_c \neq 0 \lambda_{min} = \lambda \forall \lambda = \lambda_{max}

Inequality type \geq: A\lambda + F_c = b F_c \geq 0 \lambda_{min} = \lambda

Inequality type \leq: A\lambda + F_c = b A\lambda + B\lambda = b A\lambda + B\lambda = b
```

• Constraints are violated if and only if  $\lambda$  has reached corresponding limits (a constraint force  $F_c$  should act to resolve them)





#### 3. Constraint solving:

(mixed) linear complementarity problem (MLCP)

$$J\frac{1}{M}J^{T}\lambda + F_{c} = -\beta \frac{1}{\Delta t^{2}}S - \frac{1}{\Delta t}JV - J\frac{1}{M}F_{ext}$$

$$A\lambda + F_c = b$$

$$F_{c_i} = 0 \iff \lambda_{min_i} \le \lambda_i \le \lambda_{max_i} , \forall i \in N$$

$$F_{c_i} \ge 0 \iff \lambda_i = \lambda_{min_i} , \forall i \in N$$

$$F_{c_i} \le 0 \iff \lambda_i = \lambda_{max_i} , \forall i \in N$$

- Constraints are satisfied if and only if  $\lambda$  is within corresponding limits (no additional  $F_c$  is needed to satisfy them)
- Constraints are violated if and only if  $\lambda$  has reached corresponding limits (a constraint force  $F_c$  should act to resolve them)





#### 3. Constraint solving:

• (mixed) linear complementarity problem (MLCP)

c() = 0	$c() \ge 0$	$c() \leq 0$
$-\infty < \lambda < \infty$	$\lambda \geq 0$	$\lambda \leq 0$

$$c_{1}(...) = 0$$

$$c_{2}(...) \geq 0$$

$$c_{3}(...) \leq 0 \qquad \lambda_{min} = \begin{bmatrix} -\infty \\ 0 \\ -\infty \end{bmatrix} \lambda_{max} = \begin{bmatrix} \infty \\ \infty \\ 0 \\ \vdots \\ \infty \end{bmatrix}$$

$$\vdots$$

#### In simple terms:

 $\lambda$  should only be found within constraints' corresponding limits!



#### 3. Constraint solving:

• Gauss-Seidel method (can only find solution to system of linear equations):



#### 3. Constraint solving:

• Projected Gauss-Seidel method (can solve MLCP):

```
function [x, it] = projectedGS(A, b, x0, xMin, xMax, itMax, errMax)
   rows = length(A);
   x = x0;
   for it = 1:itMax
        for row = 1:rows
           x(row) = 1/A(row, row)*(b(row) - A(row, 1:row - 1)*x(1:row - 1) - A(row, row + 1:end)*x0(row + 1:end));
            % projection
           if x(row) < xMin(row)</pre>
               x(row) = xMin(row);
           if x(row) > xMax(row)
               x(row) = xMax(row);
           end
        if abs(x - x0) < errMax
           return
       end
                                                           More algorithms exist, such as Lemke algorithm
       x0 = x;
   end
                                                                                (Carlton E. Lemke)
end
```



#### 3. Constraint solving:

$$J\frac{1}{M}J^{T}\lambda = -\beta \frac{1}{\Delta t^{2}}S - \frac{1}{\Delta t}JV - J\frac{1}{M}F_{ext}$$

$$A\lambda = b$$

$$\lambda = A \setminus b$$

$$F_{c} = J^{T}\lambda$$
numerical solution of MLCP given  $\lambda_{min}$  i  $\lambda_{max}$ 

integration:

$$\vec{v}_i = \vec{v}_{i-1} + \Delta t \frac{\vec{F}_{ext} + \vec{F}_c}{m}$$
$$\vec{p}_i = \vec{p}_{i-1} + \Delta t \vec{v}_i$$



#### 3. Constraint solving:

- Contact caching:
- + There is high probability that solution will not change much in between time steps. Caching dramatically reduces number of iterations needed to reach the solution in the next time step.
- + If system of equations is overdetermined, it could have infinite number of solutions. Without caching new solution would be found in each time step. This would introduce a *jiggle* effect.
- Constraint violations are evaluated in each time step. Vector  $\lambda$  can differ in size and order in between multiple time steps. Each element  $\lambda$  in current time step must be matched with corresponding element from previous time step. Some kind of search algorithm must be implemented.



#### 3. Constraint solving:

Contact caching:

```
time step \Delta t:

lambda1 = projectedGS(A, b, 0, lambdaMin, lambdaMax, iterations, 10^-4);

time step 2\Delta t:

lambda2 = projectedGS(A, b, lambda1, lambdaMin, lambdaMax, iterations, 10^-4);

timestep 3\Delta t:

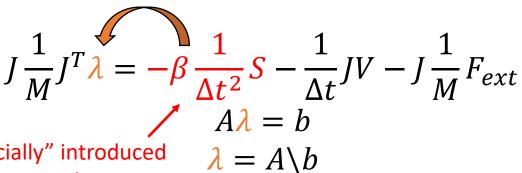
lambda3 = projectedGS(A, b, lambda2, lambdaMin, lambdaMax, iterations, 10^-4);

:
```



#### 3. Constraint solving:

- Fraction of force used to cancel out constraint violation becomes integrated into velocity and carried over to the next time step even though it is going to cancel out constraint violation in the current time step
- "Excess" velocity (which hasn't originated from external forces) is accumulated in between time steps



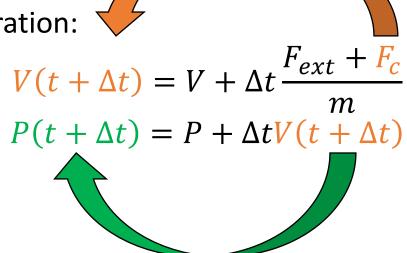
 $F_c = J^T \lambda$ 

"artificially" introduced term to cancel out existing violation of constraints as a consequence of:

1. them being discovered after they have happened

2. numerical errors

integration:





### 3. Constraint solving:

Problem should be separated:

- Fraction of force which modifies velocities should be integrated normally
- II. Fraction of force which cancels out constraints' violations should be integrated directly into positions to avoid introducing additional energy

$$J\frac{1}{M}J^{T}\lambda_{v} = -\frac{1}{\Delta t}JV - J\frac{1}{M}F_{ext}$$

$$A\lambda_{v} = b_{v}$$

$$\lambda_{v} = A \setminus b_{v}$$
(can be used in both systems)
$$F_{cv} = J^{T}\lambda_{v}$$

$$J\frac{1}{M}J^{T}\lambda_{p} = -\beta \frac{1}{\Delta t^{2}}S^{T}$$

$$A\lambda_{p} = b_{p}$$

$$\lambda_{p} = A \setminus b_{p}$$

$$F_{c_{p}} = J^{T}\lambda_{p}$$

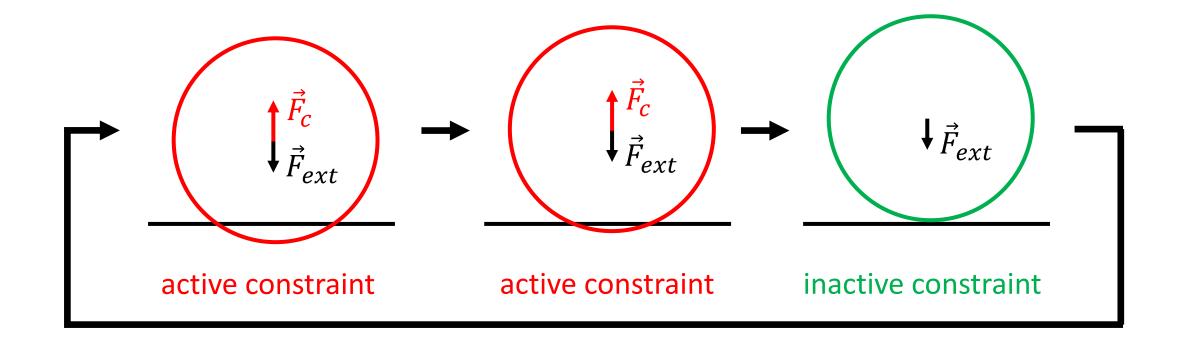
integration:

$$V(t + \Delta t) = V + \Delta t \frac{F_{ext} + F_{cv}}{m}$$

$$P(t + \Delta t) = P + \Delta t \left(V(t + \Delta t) + \Delta t \frac{F_{cp}}{m}\right)$$



- 3. Constraint solving:
  - Contact constraints:





- 3. Constraint solving:
  - Contact constraints:

$$J\frac{1}{M}J^T\lambda_p = -\beta \frac{1}{\Delta t^2} S$$

Prevent canceling out a small fraction of penetration (to keep constraints active as long as the bodies are in contact):

$$S = \sigma + \sigma_{slop}$$

†

 $0 < \sigma_{slop} \ll 1$ 

real separation

(negative value)



- 3. Constraint solving:
  - Contact constraints:

$$J\frac{1}{M}J^T\lambda_p = -\beta \frac{1}{\Delta t^2} S$$

Prevent positive separation or else solution will break down:

$$S = min(\sigma + \sigma_{slop}, 0)$$



## Demo

https://github.com/mbeocanin/MLCP-Particle-Phyiscs-Sandbox



# Q&A

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https://github.com/mbeocanin