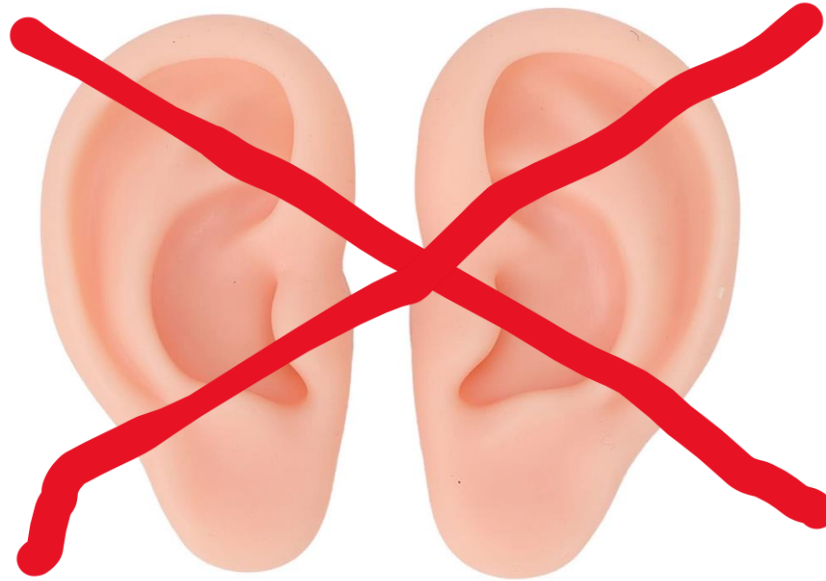


Introduction to EARS models

Mads Baungaard
University of Oxford
July 31, 2024

EARS

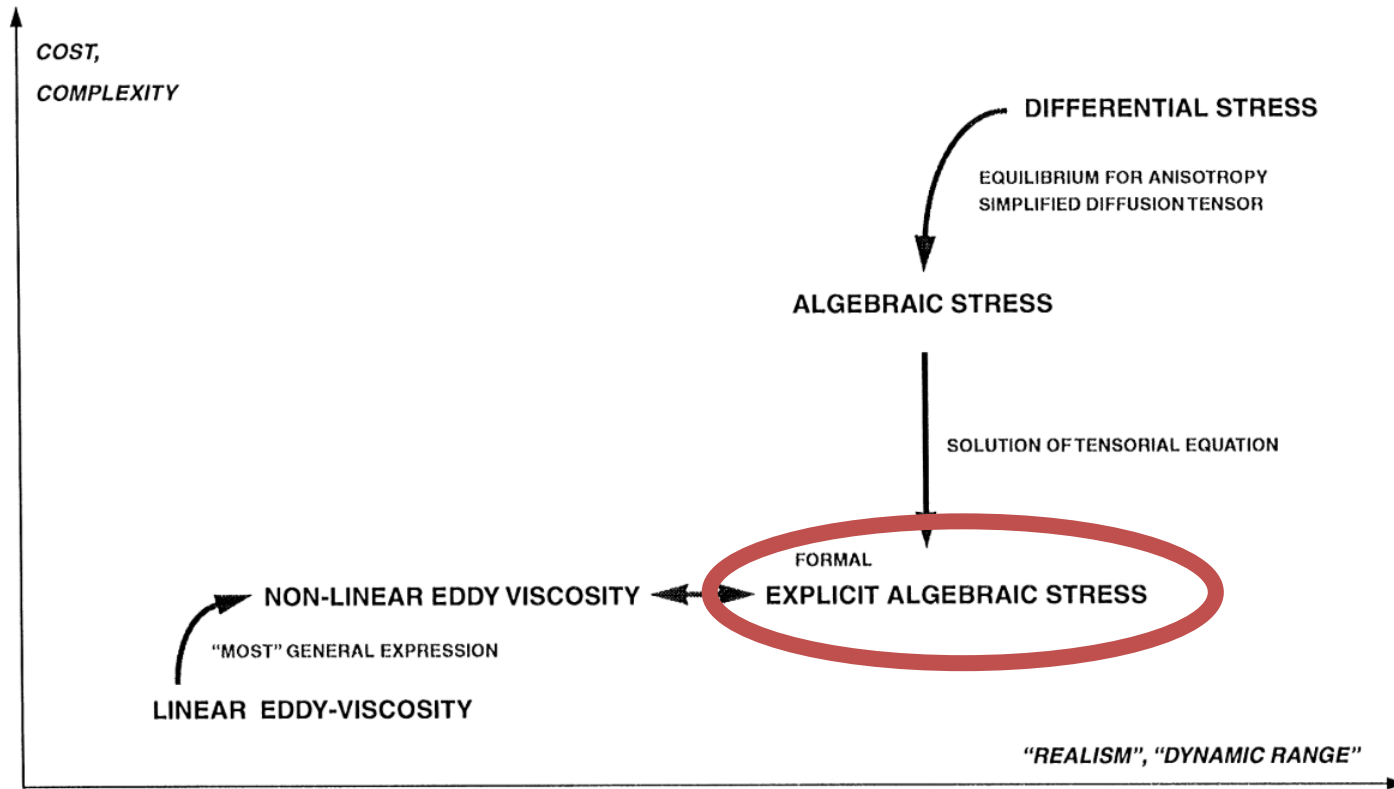


Explicit **A**lgebraic **R**eynolds **S**tress models



What is an EARS model?

- A class of turbulence models for RANS (and LES)



Gatski & Jongen (2000)

Example of an EARS model

- Model 2 of Wallin & Johansson (2000) with k-eps.

$$\frac{\partial k}{\partial t} + U_j \frac{\partial k}{\partial x_j} = \underbrace{-\overline{u'_i u'_j} \frac{\partial U_i}{\partial x_j}}_{\mathcal{P}} - \varepsilon + \underbrace{\frac{\partial}{\partial x_j} \left(\frac{\nu_t}{\sigma_k} \frac{\partial k}{\partial x_j} \right)}_{\mathcal{D}^{(k)}},$$

$$\frac{\partial \varepsilon}{\partial t} + U_j \frac{\partial \varepsilon}{\partial x_j} = (C_{\varepsilon 1} \mathcal{P} - C_{\varepsilon 2} \varepsilon) \frac{\varepsilon}{k} + \underbrace{\frac{\partial}{\partial x_j} \left(\frac{\nu_t}{\sigma_\varepsilon} \frac{\partial \varepsilon}{\partial x_j} \right)}_{\mathcal{D}^{(\varepsilon)}}.$$

$$\mathbf{S} = S_{ij} = \frac{1}{2} \frac{k}{\varepsilon} \left(\frac{\partial U_i}{\partial x_j} + \frac{\partial U_j}{\partial x_i} \right),$$

$$\mathbf{\Omega} = \Omega_{ij} = \frac{1}{2} \frac{k}{\varepsilon} \left(\frac{\partial U_i}{\partial x_j} - \frac{\partial U_j}{\partial x_i} \right),$$

$$II_S = S_{ij} S_{ji}$$

$$II_\Omega = \Omega_{ij} \Omega_{ji}$$

$$\mathbf{T}^{(1)} = \mathbf{S}$$

$$\mathbf{T}^{(4)} = \mathbf{S}\mathbf{\Omega} - \mathbf{\Omega}\mathbf{S}$$

$$P_1 = \left(\frac{1}{27} c_1'^2 + \frac{9}{20} II_S - \frac{2}{3} II_\Omega \right) c_1',$$

$$P_2 = P_1^2 - \left(\frac{1}{9} c_1'^2 + \frac{9}{10} II_S + \frac{2}{3} II_\Omega \right)^3.$$

$$N = \begin{cases} \frac{c_1'}{3} + (P_1 + \sqrt{P_2})^{1/3} + \text{sign}(P_1 - \sqrt{P_2}) |P_1 - \sqrt{P_2}|^{1/3}, & P_2 \geq 0 \\ \frac{c_1'}{3} + 2(P_1^2 - P_2)^{1/6} + \cos\left(\frac{1}{3} \left(\frac{P_1}{\sqrt{P_1^2 - P_2}} \right)\right), & P_2 < 0 \end{cases}$$

$$\beta_1 = -\frac{6}{5} \frac{N}{N^2 - 2II_\Omega}, \quad \beta_4 = -\frac{6}{5} \frac{1}{N^2 - 2II_\Omega}.$$

$$a_{ij} = \beta_1 T_{ij}^{(1)} + \beta_4 T_{ij}^{(4)}$$

$$\overline{u'_i u'_j} = k a_{ij} + \frac{2}{3} k \delta_{ij}$$

How I got interested in EARSMS

FLOW centre
KTH, Stockholm, Sweden




Visited dr. Stefan Wallin in Autumn 2021.

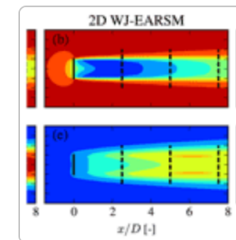
EARSM for wind applications

Research article | 

Wind turbine wake simulation with explicit algebraic Reynolds stress modeling

Mads Baungaard , Stefan Wallin, Maarten Paul van der Laan, and Mark Kelly

10 Oct 2022



PAPER • **OPEN ACCESS**

RANS simulation of a wind turbine wake in the neutral atmospheric pressure-driven boundary layer

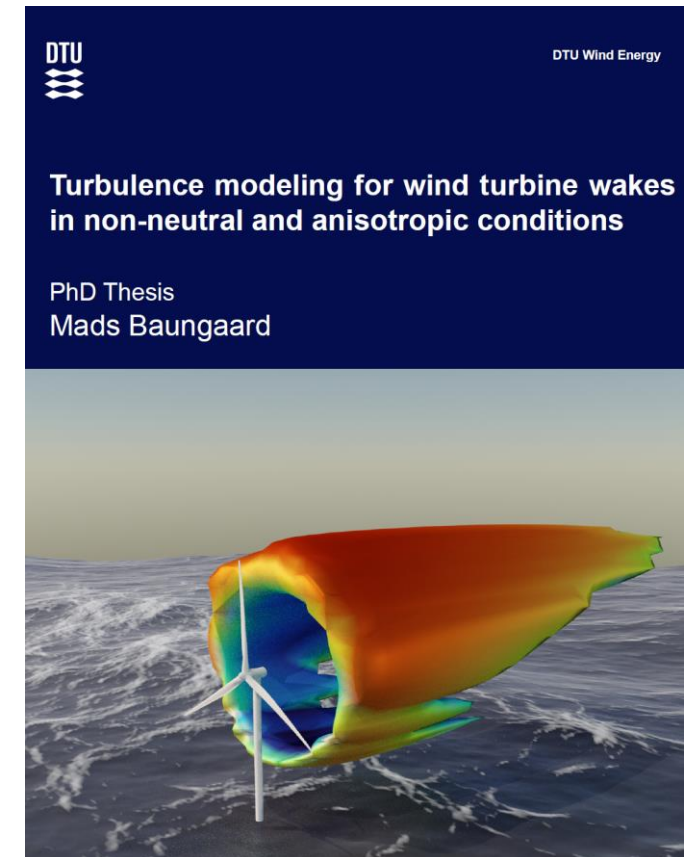
M Baungaard¹, M P van der Laan¹, S Wallin² and M Abkar³

Published under licence by IOP Publishing Ltd

[Journal of Physics: Conference Series](#), Volume 2505, [Wake Conference 2023](#), 20/06/2023 - 22/06/2023, Visby, Sweden

Citation M Baungaard et al 2023 *J. Phys.: Conf. Ser.* **2505** 012028

DOI 10.1088/1742-6596/2505/1/012028



Outline

Part 1:

Basics of turbulence modelling

Part 2:

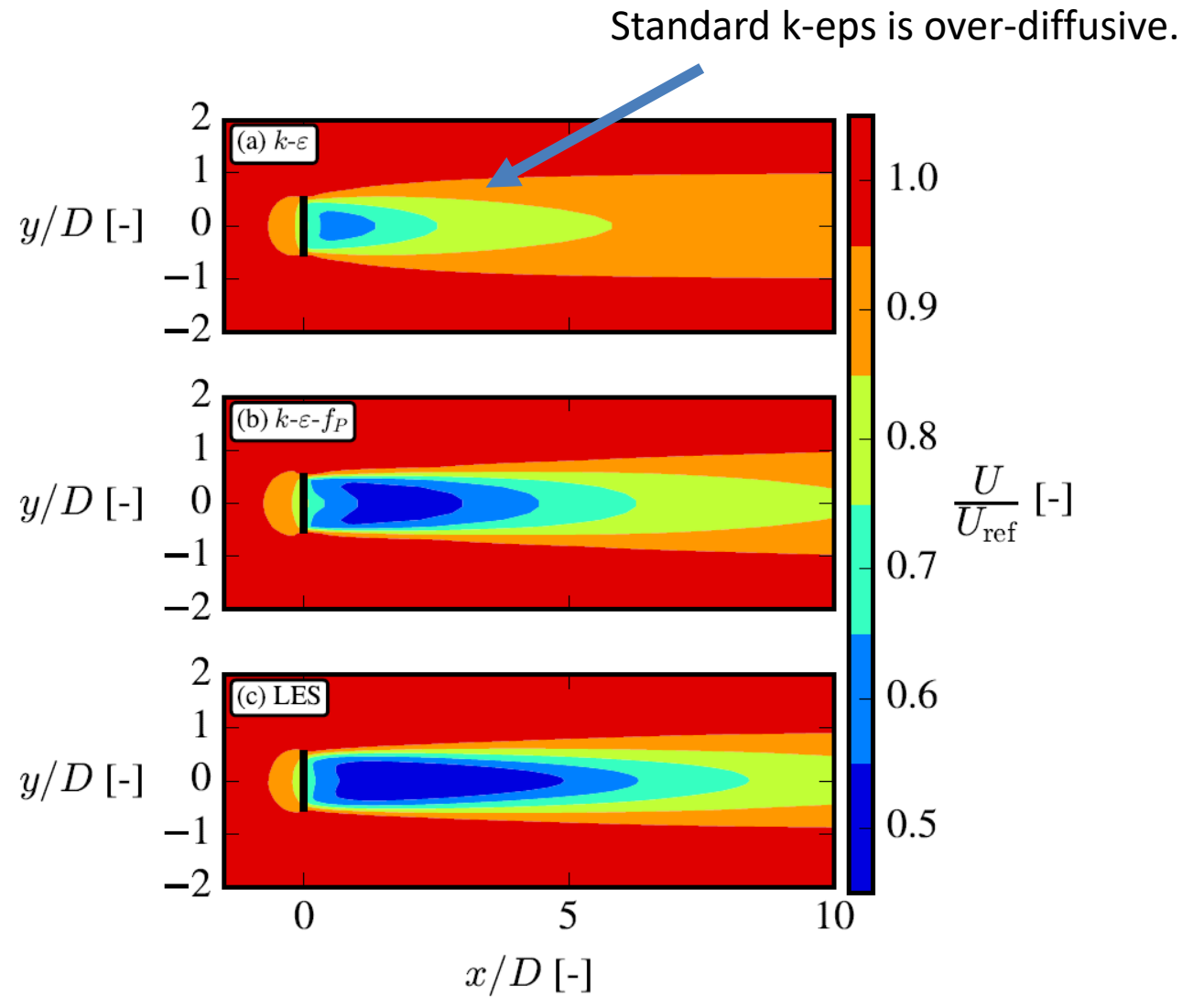
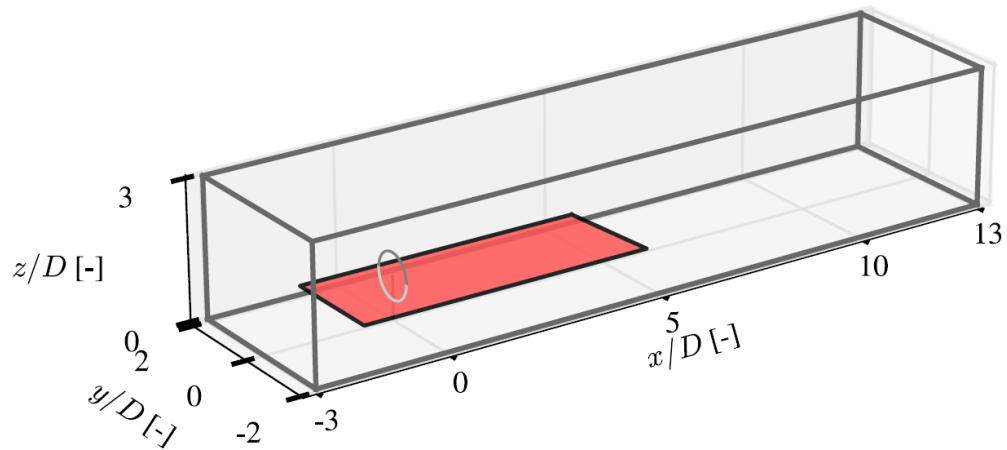
RANS equations

$$\frac{\partial U_i}{\partial t} + \frac{\partial (U_j U_i)}{\partial x_j} = -\frac{1}{\rho} \frac{\partial P}{\partial x_i} + \frac{\partial}{\partial x_j} \left(\nu \left(\frac{\partial U_i}{\partial x_j} + \frac{\partial U_j}{\partial x_i} \right) - \overline{u'_i u'_j} \right) + F_i$$

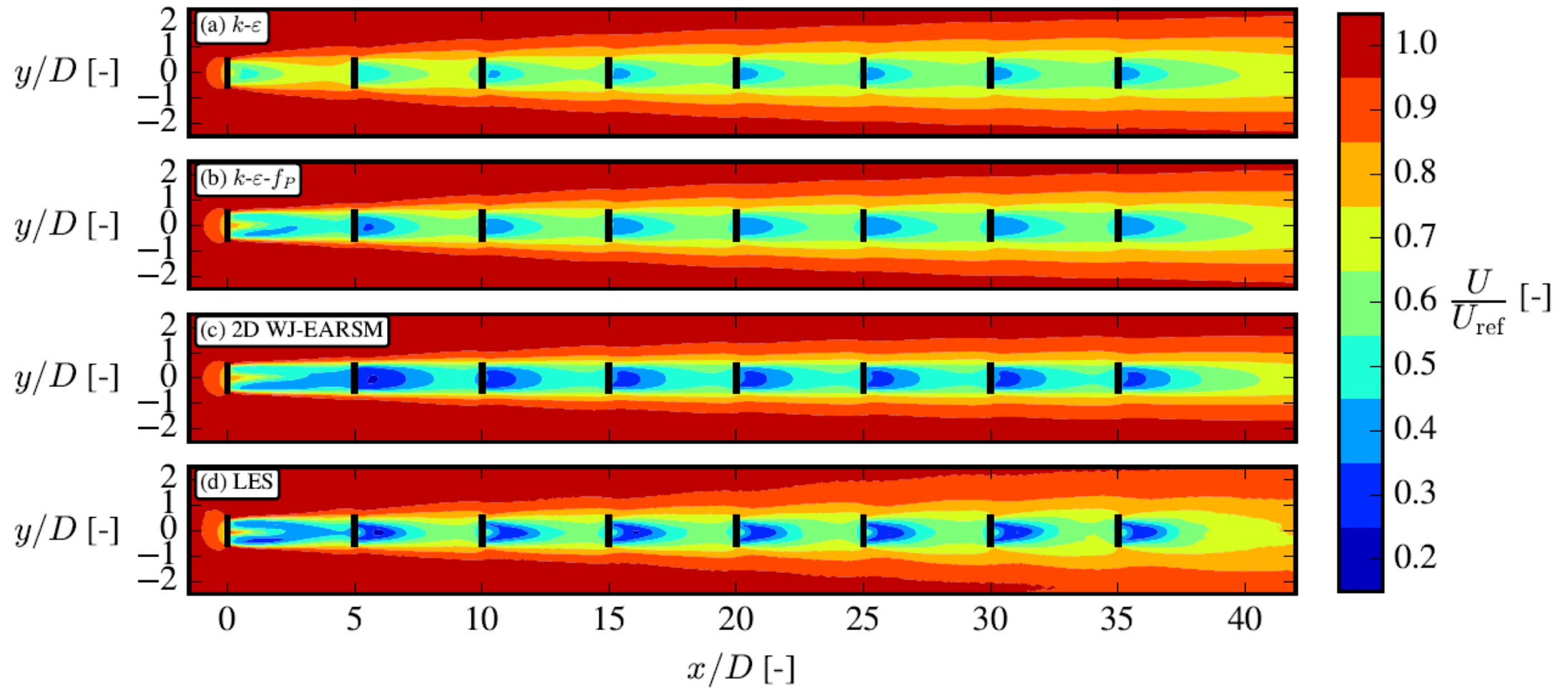
Very important for getting good results!

Turbulence modelling: *“How do we get the Reynolds stress tensor?”*

Example 1: a wind turbine wake



Example 2: a row of turbines



How to model $u_i u_j$?

- A natural starting point could be the exact equation

$$\frac{D\overline{u'_i u'_j}}{Dt} = \underbrace{-\overline{u'_j u'_k} \frac{\partial U_i}{\partial x_k} - \overline{u'_i u'_k} \frac{\partial U_j}{\partial x_k}}_{\mathcal{P}_{ij}} \underbrace{- \overline{\frac{\partial u'_j u'_i u'_k}{\partial x_k}}}_{\mathcal{D}_{ij}^t + \mathcal{D}_{ij}^v} + \nu \frac{\partial^2 \overline{u'_i u'_j}}{\partial x_k^2} \underbrace{- \frac{1}{\rho} \overline{u'_j \frac{\partial p'}{\partial x_i}} - \frac{1}{\rho} \overline{u'_i \frac{\partial p'}{\partial x_j}}}_{\text{vel pgrad corr}} - \underbrace{2\nu \overline{\frac{\partial u'_i}{\partial x_k} \frac{\partial u'_j}{\partial x_k}}}_{\varepsilon_{ij}}$$

Terms in **red** need to be modelled

A more compact form of the uiuj equation

- A typical decomposition

$$\underbrace{-\frac{1}{\rho} \overline{u'_j \frac{\partial p'}{\partial x_i}} - \frac{1}{\rho} \overline{u'_i \frac{\partial p'}{\partial x_j}}}_{\text{vel pgrad corr}} = \underbrace{\frac{1}{\rho} \overline{p' \left(\frac{\partial u'_i}{\partial x_j} + \frac{\partial u'_j}{\partial x_i} \right)}}_{\Pi_{ij}} - \underbrace{\frac{1}{\rho} \left(\overline{\frac{\partial p' u'_j}{\partial x_i}} + \overline{\frac{\partial p' u'_i}{\partial x_j}} \right)}_{\mathcal{D}_{ij}^p}$$

and a typical model assumption

$$\varepsilon_{ij} = \frac{2}{3} \varepsilon \delta_{ij},$$

allow the uiuj equation to be written as:

$$\boxed{\frac{D \overline{u'_i u'_j}}{Dt} = \mathcal{P}_{ij} + \mathcal{D}_{ij} + \Pi_{ij} - \frac{2}{3} \varepsilon \delta_{ij} .}$$

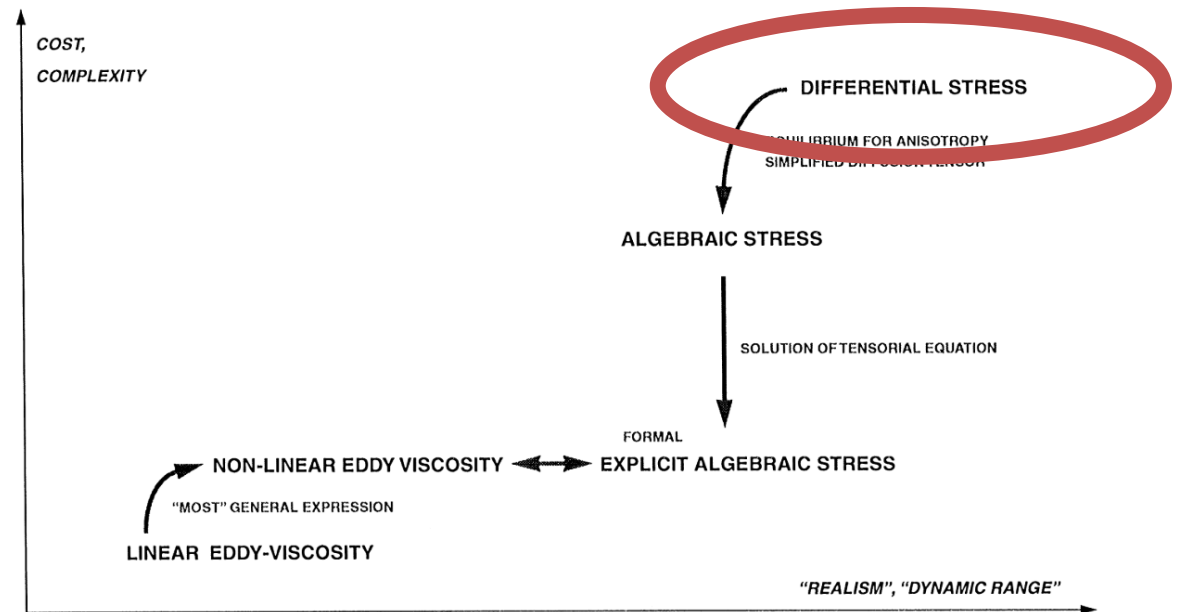
Differential Reynolds stress models (DRSMs)

$$\frac{D\overline{u'_i u'_j}}{Dt} = \mathcal{P}_{ij} + \mathcal{D}_{ij} + \Pi_{ij} - \frac{2}{3}\varepsilon\delta_{ij}.$$

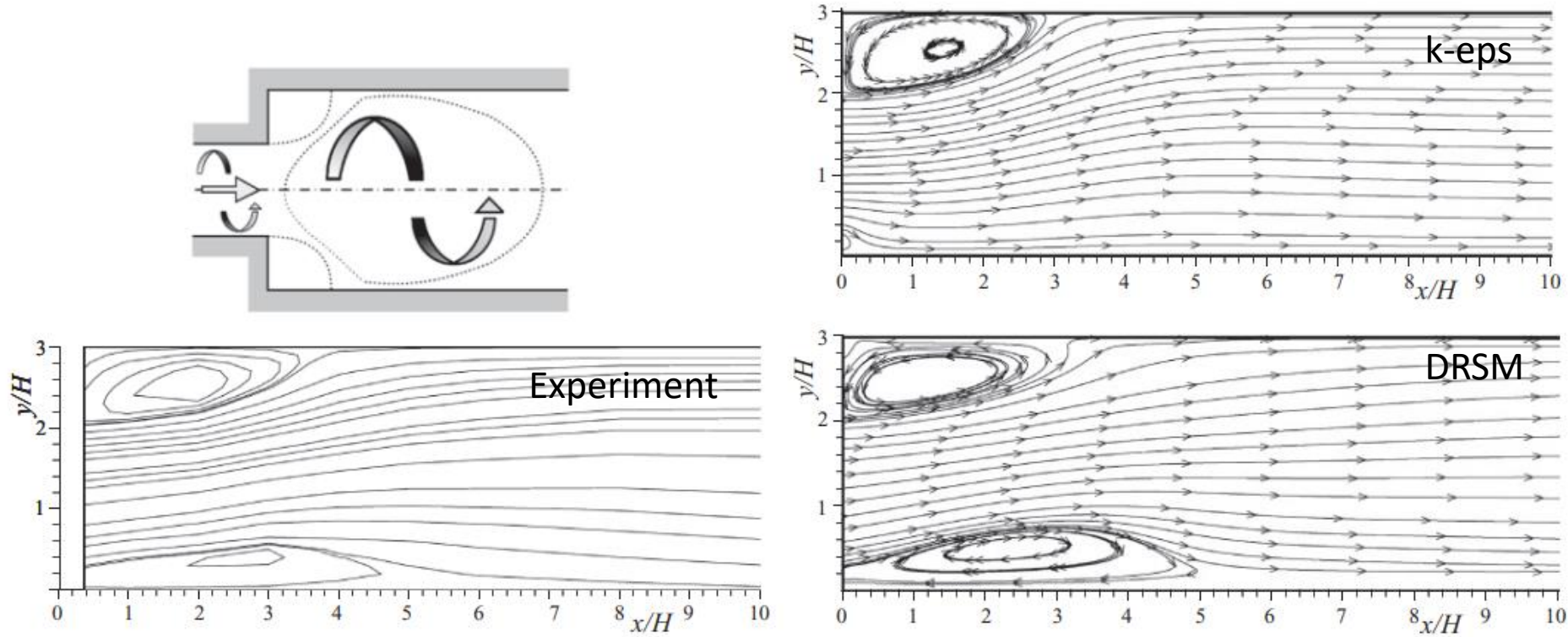
- 7 equations (6 for $u_i u_j$ + 1 for ε)
- Need models for

$$\Pi_{ij} \quad \mathcal{D}_{ij}^p \quad \mathcal{D}_{ij}^t$$

- Complicated, but possible. Two models (LRR and SSG) are available in OpenFOAM.



Example: swirling flow in an expanding pipe



Hanjalic & Launder (2011, p.81)

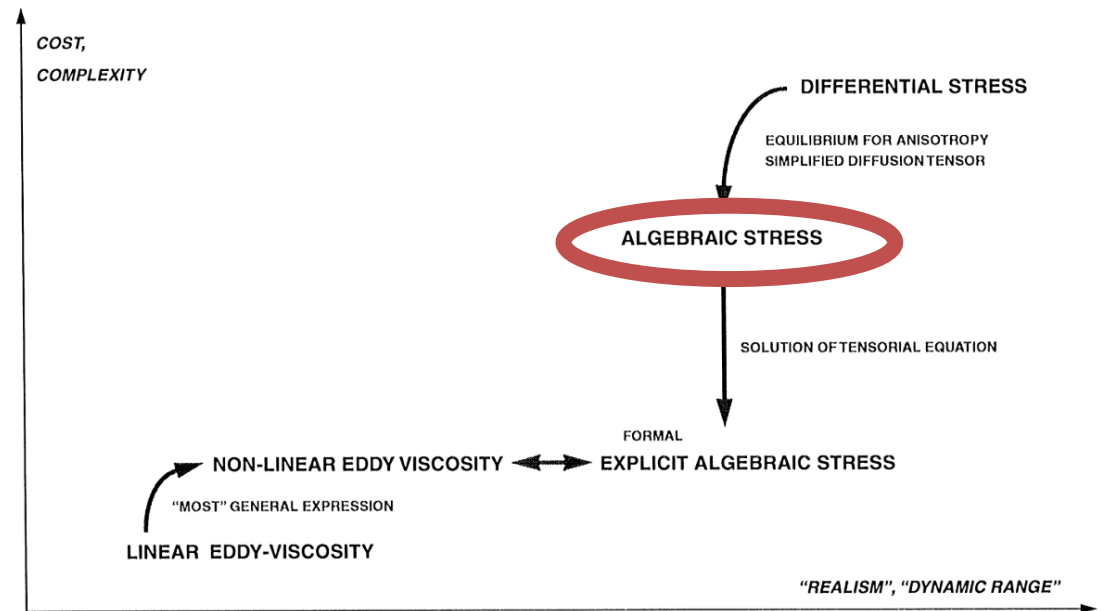
Motivation for the algebraic Reynolds stress (ARSM) equations

Disadvantages of DRSMs:

- More expensive (7 eqs)
- Difficult to implement
- Numerical robustness

Idea by Rodi (1972,1976):

“Transform the DRSM equations into a set of algebraic equations.”



Derivation of the ARSM equations 1/2

- It is convenient to use the anisotropy tensor

$$\mathbf{a} = a_{ij} \equiv \frac{\overline{u'_i u'_j}}{k} - \frac{2}{3} \varepsilon \delta_{ij},$$

Dimensionless
Symmetric
Traceless \rightarrow 5 unique elements

- Rewrite DRSM equations to

$$\frac{Da_{ij}}{Dt} = \frac{1}{k} \left(\mathcal{P}_{ij} + \Pi_{ij} + \mathcal{D}_{ij} - \frac{2}{3} \varepsilon \delta_{ij} - \left(\frac{\overline{u'_i u'_j}}{k} \right) (\mathcal{P} + \mathcal{D}^{(k)} - \varepsilon) \right)$$

and rearrange

$$\Rightarrow \frac{Da_{ij}}{Dt} - \underbrace{\left(\frac{\mathcal{D}_{ij}}{k} - \frac{\overline{u'_i u'_j}}{k^2} \mathcal{D}^{(k)} \right)}_{\mathcal{D}_{ij}^{(a)}} = \frac{1}{k} \left(\mathcal{P}_{ij} + \Pi_{ij} - \frac{2}{3} \varepsilon \delta_{ij} - \left(\frac{\overline{u'_i u'_j}}{k} \right) (\mathcal{P} - \varepsilon) \right)$$

“Weak-equilibrium assumption”

Derivation of the ARSM equations 2/2

- The “weak-equilibrium assumption” gives an algebraic set of equations

$$0 = \mathcal{P}_{ij} + \Pi_{ij} - \frac{2}{3}\varepsilon\delta_{ij} - \left(\frac{\overline{u'_i u'_j}}{k} \right) (\mathcal{P} - \varepsilon)$$

- Rewrite using definition of the anisotropy tensor

$$a_{ij} \left(\frac{\mathcal{P}}{\varepsilon} - 1 \right) = \frac{\mathcal{P}_{ij}}{\varepsilon} + \frac{\Pi_{ij}}{\varepsilon} - \frac{2}{3} \frac{\mathcal{P}}{\varepsilon} \delta_{ij}.$$

Anatomy of the Reynolds stress tensor

$$\overline{u'_i u'_j} = \begin{pmatrix} \overline{u'_1 u'_1} & \overline{u'_1 u'_2} & \overline{u'_1 u'_3} \\ \overline{u'_2 u'_1} & \overline{u'_2 u'_2} & \overline{u'_2 u'_3} \\ \overline{u'_3 u'_1} & \overline{u'_3 u'_2} & \overline{u'_3 u'_3} \end{pmatrix} \quad \text{Unit: m}^2/\text{s}^2$$

- Symmetric \rightarrow six unique elements
- Galilean invariant
- Half of the trace defined as the “turbulent kinetic energy” (TKE)

$$k = \frac{1}{2} \overline{u'_i u'_i} = \frac{1}{2} \left(\overline{u'_1 u'_1} + \overline{u'_2 u'_2} + \overline{u'_3 u'_3} \right)$$

Limits of the Reynolds stress tensor

Rule 1

Rule 2

$$0 \leq \overline{u'_\alpha u'_\alpha} \leq 2k$$

Greek indices imply that there is no summation over repeated indices!

$$\left(\overline{u'_\alpha u'_\beta} \right)^2 \leq \overline{u'_\alpha u'_\alpha} \overline{u'_\beta u'_\beta} \leq k^2$$

Rule 3

Rule 4

- Rule 1: Each normal stress must be positive.
- Rule 2: From the definition of TKE.
- Rule 3: Each shear stress must satisfy the Cauchy-Schwarz inequality.
- Rule 4: Comes from rule 2.

Motivation: Wind farm aerodynamics

Lots of parameters!

- Atmospheric conditions
- Oceanic conditions
- Turbine layout
- Turbine design
- Turbine operating conditions



Cause difficulties in...

- Modelling (wakes, blockage)
- Optimisation (AEP, LCoE, etc.)

Question:

- *What is the best way to tackle this **very high-dimensional** optimisation problem?*

Test

Test