

## LECTURE 11:

### TOPICS:

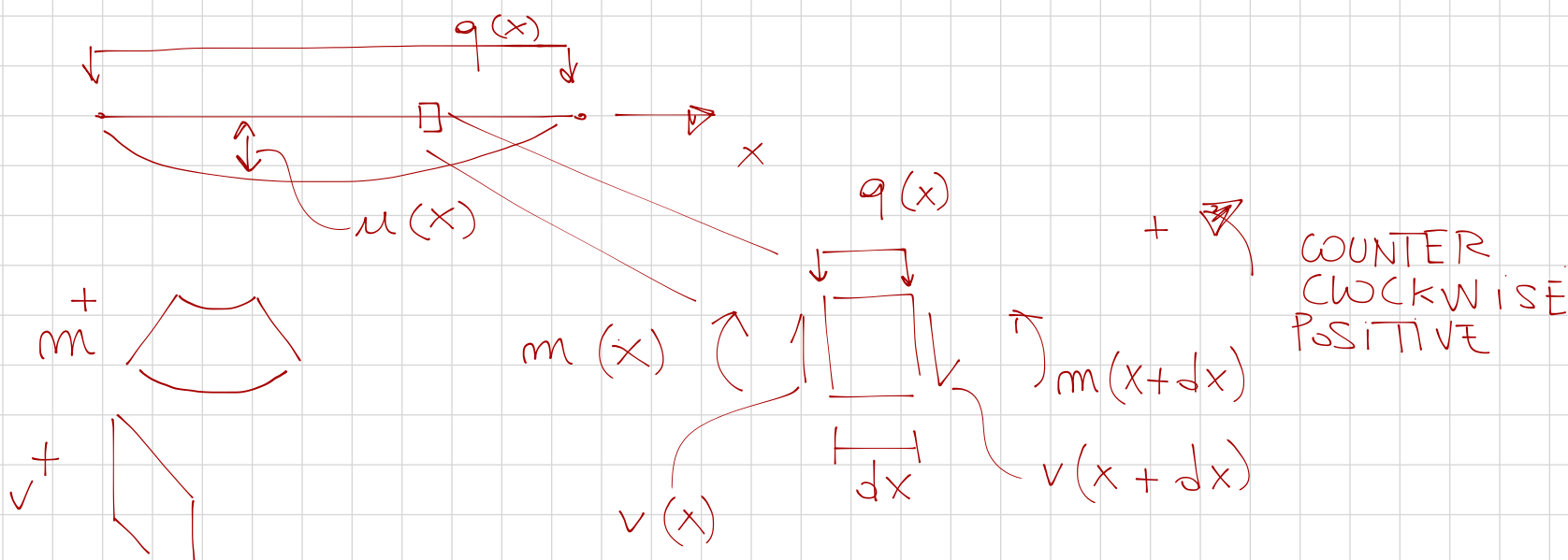
- REVIEW
- BEAM EQ IN 1D

### LOGISTICS

- HW 3 DUE TOM

## BEAM EQUATIONS

IN EULER-BERNOULLI WE ASSUME PLANE SECTIONS REMAIN PLANE



SUM OF FORCES IN THE VERTICAL DIRECTION GIVES

$$v(x) - v(x+dx) - q dx = 0 \Rightarrow \frac{dv}{dx} = -q$$

BALANCE OF MOMENT ABOUT  $x$

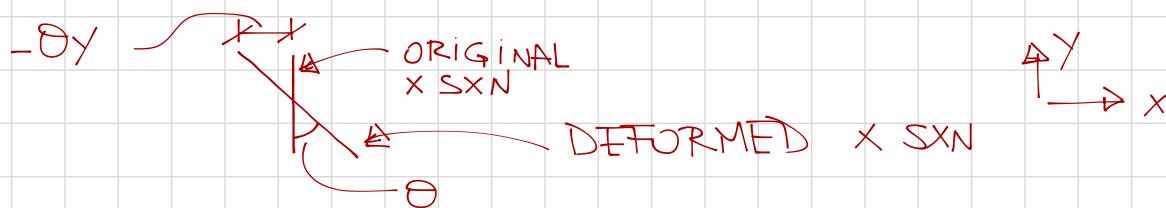
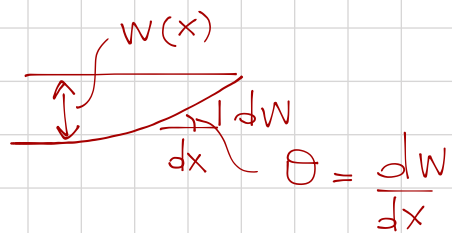
$$m(x+dx) - m(x) - v(x+dx)dx - (q dx)dx = 0$$

$$q dx = v(x) - v(x+dx)$$

$$m(x+dx) - m(x) - \cancel{v(x+dx)dx} - v(x)dx + \cancel{v(x+dx)dx} = 0$$

$$m(x+dx) - m(x) = v(x)dx \Rightarrow \frac{dm}{dx} = v$$

USING THE KINEMATIC ASSUMPTION OF PLANE SECTIONS  
REMAIN PLANE AND NORMAL TO THE NEUTRAL AXIS



$$\epsilon(x, y) = \frac{d}{dx}(-\theta y) = -\frac{d^2 w}{dx^2} y, \quad M(x) = -\int_A \sigma y = -\int_A E \epsilon y =$$

$$= \int_A E \frac{d^2 w}{dx^2} y \, dA = E \frac{d^2 w}{dx^2} \int_A y \, dA = EI \frac{d^2 w}{dx^2}$$

WE HAVE

$$\frac{dV}{dx} = \frac{d}{dx} \left( \frac{dM}{dx} \right) = \frac{d}{dx} \left( \frac{d}{dx} EI \frac{d^2 w}{dx^2} \right) = EI \frac{d^4 w}{dx^4} = -q$$

$$EI \frac{d^4 w}{dx^4} = -q \quad \leftarrow \text{BEAM EQUATION}$$

THE DIFF PROBLEM THEN READS

$$?, w: [x_i, x_j] \rightarrow \mathbb{R} \quad EI \frac{d^4 w}{dx^4} = -q \quad \text{ASSUME } = 0 \text{ FOR NOW} \quad \forall x \in (x_i, x_j)$$

$$w(x_i) = w_i$$

$$w(x_j) = w_j$$

$$\frac{dw}{dx}(x_i) = \theta(x_i) = \theta_i$$

$$\frac{dw}{dx}(x_j) = \theta(x_j) = \theta_j$$

IF WE INTEGRATE THE ABOVE THE SOLUTION TAKES THE FORM

$$w(x) = c_1 x^3 + c_2 x^2 + c_3 x + c_4$$

WE HAVE FOUR UNKNOWN AND FOUR BOUNDARY  
CONDITIONS SUCH THAT WE CAN SOLVE FOR  $c_i, i=1 \dots 4$ .

IF WE DO SO WE END UP WITH A FORM OF  $w(x)$   
AS

$$N(x) = N_1(x)\mu_i + N_2(x)\theta_i + N_3(x)\mu_j + N_4(x)\theta_j$$

WHERE, ASSUMING  $x_i = 0, x_j = \ell$

$$N_1(x) = 1 - 3\left(\frac{x}{\ell}\right)^2 + 2\left(\frac{x}{\ell}\right)^3$$

$$N_2(x) = \ell\left[\left(\frac{x}{\ell}\right) - 2\left(\frac{x}{\ell}\right)^2 + \left(\frac{x}{\ell}\right)^3\right]$$

$$N_3(x) = 3\left(\frac{x}{\ell}\right)^2 - 2\left(\frac{x}{\ell}\right)^3$$

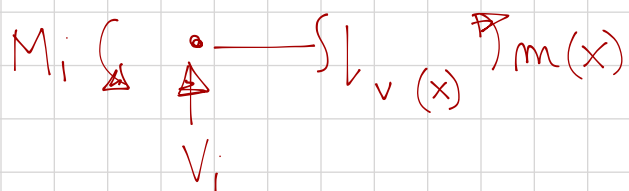
$$N_4(x) = \ell\left[\left(\frac{x}{\ell}\right)^3 - \left(\frac{x}{\ell}\right)^2\right]$$

LET LOWERCASE  $m(x)$  &  $v(x)$  DENOTE THE INTERNAL  
MOMENT AND SHEAR RESP.

RECALL

$$m(x) = EI \frac{d^2 w}{dx^2}(x) \quad v(x) = \frac{dm}{dx}(x) = EI \frac{d^3 w}{dx^3}(x)$$

AT NODE  $i$  WE APPLY EXTERNAL MOMENT & FORCES



SUM MOMENTS

$$M_i + m(0) = 0$$

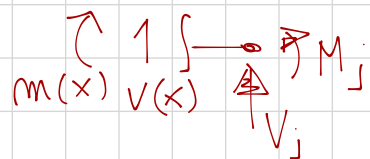
$$M_i = -m(0) = -EI \frac{d^2 w(0)}{dx^2} = EI \left\{ -N_1''(0) - N_2''(0) - N_3''(0) - N_4''(0) \right\} \begin{bmatrix} \mu_i \\ \theta_i \\ \mu_j \\ \theta_j \end{bmatrix}$$

SUM FORCES

$$V_i - v(x_i = 0) = 0$$

$$V_i = v(0) = EI \frac{d^3 w(0)}{dx^3} = EI \left\{ N_1'''(0) - N_2'''(0) - N_3'''(0) - N_4'''(0) \right\} \begin{bmatrix} \mu_i \\ \theta_i \\ \mu_j \\ \theta_j \end{bmatrix}$$

SIMILARLY AT NODE J



$$M_j - m(x_j) = 0$$

$$M_j = EI \frac{d^2 w}{dx^2}(e) = EI \{ N_1''(e) \quad N_2''(e) \quad N_3''(e) \quad N_4''(e) \} \begin{bmatrix} u_i \\ \theta_i \\ u_j \\ \theta_j \end{bmatrix}$$

$$V_j + v(x_j = l) = 0$$

$$V_j = -v(x_j = l) = -EI \{ N_1'''(e) \quad N_2'''(e) \quad N_3'''(e) \quad N_4'''(e) \} \begin{bmatrix} u_i \\ \theta_i \\ u_j \\ \theta_j \end{bmatrix}$$

THUS GIVING

$$\begin{bmatrix} V_i \\ M_i \\ V_j \\ M_j \end{bmatrix} = EI \begin{bmatrix} N_1'''(0) & N_2'''(0) & N_3'''(0) & N_4'''(0) \\ -N_1''(0) & -N_2''(0) & -N_3''(0) & -N_4''(0) \\ -N_1'''(l) & -N_2'''(l) & -N_3'''(l) & -N_4'''(l) \\ N_1''(l) & N_2''(l) & N_3''(l) & N_4''(l) \end{bmatrix} \begin{bmatrix} u_i \\ \theta_i \\ u_j \\ \theta_j \end{bmatrix}$$

SEE ATTACHED SLIDES FOR CONSTRAINTS

## A SIMPLE EXAMPLE:



CONSIDER A BEAM OF LENGTH \$l=10\$ WITH AN APPLIED LOAD \$M=10\$

$$\begin{Bmatrix} V_1 \\ M_1 \\ V_2 \\ M_2 \end{Bmatrix} = EI \begin{bmatrix} 12/e^3 & 6/e^2 & -12/e^3 & 6/e^2 \\ 6/e^2 & 4/e & -6/e^2 & 2/e \\ 12/e^3 & -6/e^2 & 12/e^3 & -6/e^2 \\ 6/e^2 & 2/e & -6/e^2 & 4/e \end{bmatrix} \begin{Bmatrix} w_1 \\ \theta_1 \\ w_2 \\ \theta_2 \end{Bmatrix}$$

~~$$\begin{Bmatrix} V_1 \\ M_1 \\ V_2 \\ M_2 \end{Bmatrix} = EI \begin{bmatrix} 12/e^3 & 6/e^2 & -12/e^3 & 6/e^2 \\ 6/e^2 & 4/e & -6/e^2 & 2/e \\ 12/e^3 & -6/e^2 & 12/e^3 & -6/e^2 \\ 6/e^2 & 2/e & -6/e^2 & 4/e \end{bmatrix} \begin{Bmatrix} w_1 \\ \theta_1 \\ w_2 \\ \theta_2 \end{Bmatrix}$$~~

$$K_{ff} = \begin{bmatrix} 4/e & 2/e \\ 2/e & 4/e \end{bmatrix}$$

$$\begin{Bmatrix} M_1 \\ M_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 10 \end{Bmatrix} = \begin{bmatrix} 4/e & 2/e \\ 2/e & 4/e \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}$$

$$\begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \frac{e^2}{16-4} \begin{bmatrix} 4/e & -2/e \\ -2/e & 4/e \end{bmatrix} \begin{Bmatrix} 0 \\ 10 \end{Bmatrix} = \frac{e^2}{12} \begin{Bmatrix} -20/e \\ 40/e \end{Bmatrix}$$

$$\begin{Bmatrix} \{Q_s\} \\ \{Q_f\} \end{Bmatrix} = \begin{bmatrix} [K_{ss}] & [K_{fs}] \\ [K_{sf}] & [K_{ff}] \end{bmatrix} \begin{Bmatrix} \{w_s\} \\ \{w_f\} \end{Bmatrix}$$

$$\begin{Bmatrix} V_1 \\ V_2 \end{Bmatrix} = \begin{bmatrix} 6/e^2 & 6/e^2 \\ -6/e^2 & -6/e^2 \end{bmatrix} \begin{Bmatrix} -20/12e^2 \\ 40/12e \end{Bmatrix}$$