Lesson 8

Trusses in 2D & 3D Space, Python codes

Minh-Chien Trinh, PhD

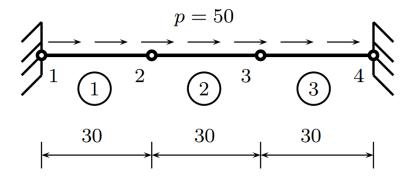
Division of Mechanical System Engineering Jeonbuk National University

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1. Python Codes - Isoparametric Bar under Uniform Load

Python code: https://github.com/mctrinh/fem-class



- An isotropic bar is clamped at two ends.
- Modulus of elasticity $E = 30 \cdot 10^6$
- Cross-section area A=1
- Uniform axial load p = 50
- Exact solution $u = \frac{pLx}{2EA} \left(1 \frac{x}{L} \right), \quad \sigma_x = \frac{pL}{A} \left(\frac{1}{2} \frac{x}{L} \right)$

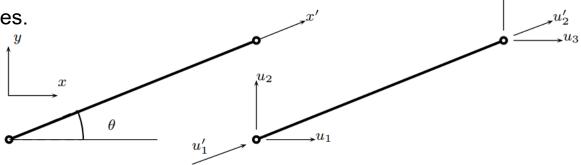
2. Trusses in 2D Space - Formulation - DOFs

- ❖ A transformation of coordinate basis is necessary to translate the local element matrices into the structural coordinate system.
- Trusses and Bars supports compressive and tensile forces only.
- All forces are applied at the nodes.
- 2 DOFs in local coordinate

$$\mathbf{u'}^T = \begin{bmatrix} u_1' & u_2' \end{bmatrix}$$

4 DOFs in global coordinate

$$\mathbf{u}^T = \begin{bmatrix} u_1 & u_2 & u_3 & u_4 \end{bmatrix}$$



2D truss element: local and global DOFs

Relation between local and global DOFs

$$\begin{cases} u_1' = u_1 \cos \theta + u_2 \sin \theta \\ u_2' = u_3 \cos \theta + u_4 \sin \theta \end{cases} \iff \mathbf{u}' = \mathbf{L}\mathbf{u}$$

$$\mathbf{L} = \begin{bmatrix} l & m & 0 & 0 \\ 0 & 0 & l & m \end{bmatrix} \quad \text{in which} \quad l = \cos \theta = \frac{x_2 - x_1}{L_e}; \quad m = \sin \theta = \frac{y_2 - y_1}{L_e}; \quad L_e = \sqrt{(x_2 - x_1)^2 + (y_2 - y_1)^2}$$

with θ is the angle between the local axis x' and global axis x.



2. Trusses in 2D Space — Formulation (Stiffness, Mass)

The local stiffness matrix of 2D truss elements is like bar elements:

$$\mathbf{K'} = \frac{EA}{L_e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

The strain energy of this element is transformed from local to global coordinate system

$$U^{e} = \frac{1}{2} \mathbf{u}'^{T} \mathbf{K}' \mathbf{u}' = \frac{1}{2} \mathbf{u}^{T} \left[\mathbf{L}^{T} \mathbf{K}' \mathbf{L} \right] \mathbf{u} = \frac{1}{2} \mathbf{u}^{T} \mathbf{K} \mathbf{u}$$

with the global stiffness matrix
$$\mathbf{K} = \frac{EA}{L_e} \begin{bmatrix} l^2 & lm & -l^2 & -lm \\ lm & m^2 & -lm & -m^2 \\ -l^2 & -lm & l^2 & lm \\ -lm & -m^2 & lm & m^2 \end{bmatrix}$$

The local mass matrix of 2D truss elements **

The consistent mass matrix

$$\mathbf{M}_C' = \frac{\rho A L_e}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix}$$

The lumped mass matrix

$$\mathbf{M}_{L}' = \frac{\rho A L_{e}}{2} \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

2. Trusses in 2D Space — Formulation (Stiffness, Mass, Stress)

The kinetic energy of this element is transformed from local to global coordinate system **

$$K^{e} = \frac{1}{2}\dot{\mathbf{u}}^{T}\mathbf{M}'\dot{\mathbf{u}}' = \frac{1}{2}\dot{\mathbf{u}}^{T} \left[\mathbf{L}^{T}\mathbf{M}'\mathbf{L}\right]\dot{\mathbf{u}} = \frac{1}{2}\dot{\mathbf{u}}^{T}\mathbf{M}\dot{\mathbf{u}}$$

$$\mathbf{M} = \frac{\rho A L_e}{6} \begin{bmatrix} 2l^2 & 2lm & l^2 & lm \\ 2lm & 2m^2 & lm & m^2 \\ l^2 & lm & 2l^2 & 2lm \\ lm & m^2 & 2lm & 2m^2 \end{bmatrix} \qquad \mathbf{M} = \frac{\rho A L_e}{2} \begin{bmatrix} l^2 & lm & 0 & 0 \\ lm & m^2 & 0 & 0 \\ 0 & 0 & l^2 & lm \\ 0 & 0 & lm & m^2 \end{bmatrix}$$

$$\mathbf{M} = \frac{\rho A L_e}{2} \begin{bmatrix} l^2 & lm & 0 & 0 \\ lm & m^2 & 0 & 0 \\ 0 & 0 & l^2 & lm \\ 0 & 0 & lm & m^2 \end{bmatrix}$$

(the global consistent mass matrix)

(the global lumped mass matrix)

The translational masses never vanish and must be retained in the local mass matrix. Setting lm = 0 and other terms are set to 1.

$$\mathbf{M} = \frac{\rho A L_e}{6} \begin{bmatrix} 2 & 0 & 1 & 0 \\ 0 & 2 & 0 & 1 \\ 1 & 0 & 2 & 0 \\ 0 & 1 & 0 & 2 \end{bmatrix}$$

$$\mathbf{M} = \frac{\rho A L_e}{2} \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

(the global consistent mass matrix)

(the global lumped mass matrix)

The stress is transformed from local to global coordinate system

$$\sigma_{x} = E\varepsilon_{x} = E\frac{u_{2}' - u_{1}'}{L_{e}} = \frac{E}{L_{e}}\begin{bmatrix} -1 & 1 \end{bmatrix}\begin{bmatrix} u_{1}' \\ u_{2}' \end{bmatrix} = \frac{E}{L_{e}}\begin{bmatrix} -1 & 1 \end{bmatrix}\mathbf{u}' = \frac{E}{L_{e}}\begin{bmatrix} -1 & 1 \end{bmatrix}\mathbf{L}\mathbf{u} = \frac{E}{L_{e}}\begin{bmatrix} -l & -m & l & m \end{bmatrix}\mathbf{u}$$



2. Trusses in 2D Space — 2D Truss Problem - Python

- Python code: https://github.com/mctrinh/fem-class
 - Modulus of elasticity

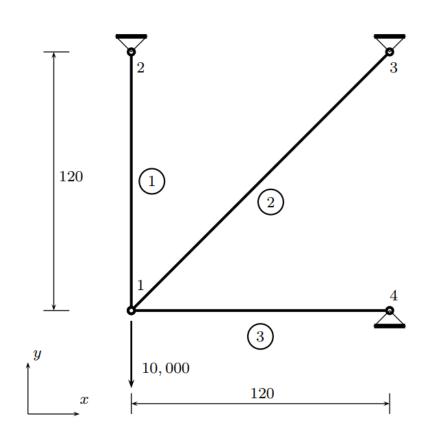
$$E = 30 \cdot 10^6$$

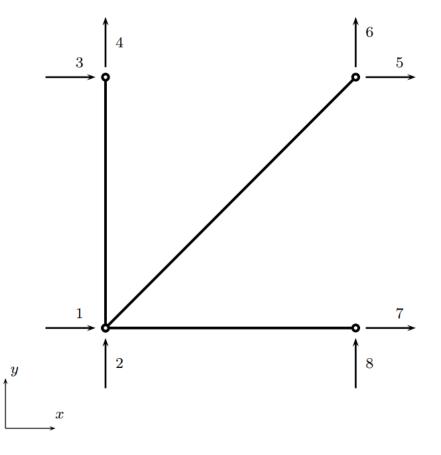
Cross-section area

$$A = 2$$

Applied load

$$P = 10000$$





3. Trusses in 3D Space - Formulation

- Generalize the 2D truss model in 3D Cartesian space.
- The element has 2 DOFs in local coordinate

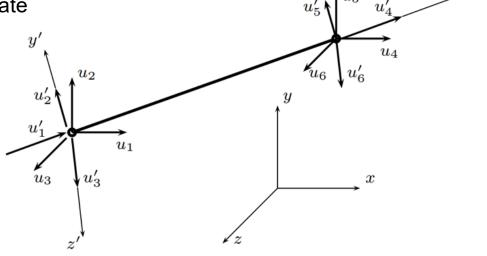
$$\mathbf{u'}^T = \begin{bmatrix} u_1' & u_2' & u_3' & u_4' & u_5' & u_6' \end{bmatrix}$$

- Each node has 3 global DOFs.
- Totally, 6 DOFs in global coordinate

$$\mathbf{u}^T = \begin{bmatrix} u_1 & u_2 & u_3 & u_4 & u_5 & u_6 \end{bmatrix}$$

Relation between local and global DOFs

$$\mathbf{u}' = \mathbf{L}\mathbf{u}$$



3D truss element: local and global DOFs

The direction cosines matrix

$$\mathbf{L} = \begin{bmatrix} l_x & l_y & l_z & 0 & 0 & 0 \\ 0 & 0 & 0 & l_x & l_y & l_z \end{bmatrix} \quad \text{with} \quad l_x = \frac{x_2 - x_1}{L_e}, \quad l_y = \frac{y_2 - y_1}{L_e}, \quad l_z = \frac{z_2 - z_1}{L_e}$$

The local stiffness matrix of 3D truss elements is like before

$$\mathbf{K}^e = \frac{EA}{L_e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$



3. Trusses in 3D Space — Formulation

The global stiffness matrix is

$$\mathbf{K} = \mathbf{L}^{T} \mathbf{K}^{e} \mathbf{L} = \frac{EA}{L_{e}} \begin{bmatrix} l_{x}^{2} & l_{x}l_{y} & l_{x}l_{z} & -l_{x}^{2} & -l_{x}l_{y} & -l_{x}l_{z} \\ l_{y}^{2} & l_{y}l_{z} & -l_{x}l_{y} & -l_{y}^{2} & -l_{y}l_{z} \\ l_{z}^{2} & -l_{x}l_{z} & -l_{y}l_{z} & -l_{z}^{2} \\ l_{x}^{2} & l_{x}l_{y} & l_{x}l_{z} \\ l_{y}^{2} & l_{y}l_{z} \end{bmatrix}$$

$$sym$$

The global mass matrix is

$$\mathbf{M} = \mathbf{L}^{T} \mathbf{M}^{e} \mathbf{L} = \frac{\rho A L_{e}}{6} \begin{bmatrix} 2 & 0 & 0 & 1 & 0 & 0 \\ & 2 & 0 & 0 & 1 & 0 \\ & & 2 & 0 & 0 & 1 \\ & & & 2 & 0 & 0 \\ & & & & 2 & 0 \\ sym & & & & 2 \end{bmatrix} \qquad \mathbf{M} = \mathbf{L}^{T} \mathbf{M}^{e} \mathbf{L} = \frac{\rho A L_{e}}{2} \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ & 1 & 0 & 0 & 0 & 0 \\ & & & 1 & 0 & 0 & 0 \\ & & & & 1 & 0 & 0 \\ & & & & & 1 & 0 \\ sym & & & & 1 \end{bmatrix}$$

$$\mathbf{M} = \mathbf{L}^{T} \mathbf{M}^{e} \mathbf{L} = \frac{\rho A L_{e}}{2}$$

$$sym$$

$$1 \quad 0 \quad 0 \quad 0 \quad 0$$

$$1 \quad 0 \quad 0$$

$$1 \quad 0 \quad 0$$

The global consistent mass matrix

The global lumped mass matrix



3. Trusses in 3D Space — 3D Truss Problem - Python

- Python code: https://github.com/mctrinh/fem-class
 - Given

$$E = 1.2 \cdot 10^{6}$$

$$P_{1} = (72, 0, 0)$$

$$P_{2} = (0, 36, 0)$$

$$P_{3} = (0, 36, 72)$$

$$P_{4} = (0, 0, -48)$$

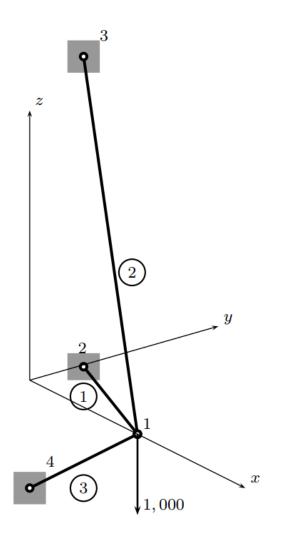
$$U_{2} = U_{3} = U_{4} = (0, 0, 0)$$

$$v_{1} = 0$$

$$A_{1} = 0.302$$

$$A_{2} = 0.729$$

$$A_{3} = 0.187$$



• Evaluate displacements, reactions, stresses of elements.

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