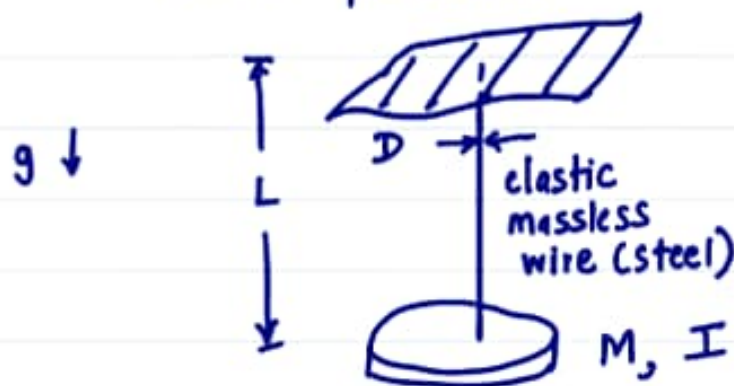


ME 202
LECTURE 8
TUTORIAL 2
MON 18 JAN 2022

Problem 1

Torsional pendulum



Gere &
Goodno
7th ed.

$$\text{Torque} = \frac{GJ}{L} \theta$$

$$I \ddot{\theta} + \frac{GJ}{L} \theta = 0$$

$$\ddot{\theta} + \omega_t^2 \theta = 0, \quad \omega_t = \sqrt{\frac{GJ}{LI}}$$

← wire
← mass moment of inertia

Next page

Max safe angle of twist under torsion only.

$$\begin{aligned} \text{from expt given } \tau_{\max} &= \frac{16T}{\pi D^3} = \frac{16}{\pi D^3} \frac{G \pi D^4}{L 32} \theta_{\max} \\ \theta_{\max} &= \frac{2L \tau_{\max}}{GD} \end{aligned}$$

However, wire is under both torsion & tension.

$$\sigma_{zz} = \frac{P}{A} = \frac{4Mg}{\pi D^2}, \quad \sigma_{xz} = \frac{16T}{\pi D^3}$$

$$\text{Max shear stress} = \left(\left(\frac{\sigma_{zz}}{2} \right)^2 + \sigma_{xz}^2 \right)^{1/2} = \tau_{\max}$$

$$\sqrt{\left(\frac{2Mg}{\pi D^2} \right)^2 + \left(\frac{GD \theta_{\max}}{2L} \right)^2} = \tau_{\max}$$

Problem 2

$$\begin{array}{cc} \omega = 2\pi f & \\ \uparrow & \uparrow \\ \text{rad/s} & \text{Hz} \end{array}$$

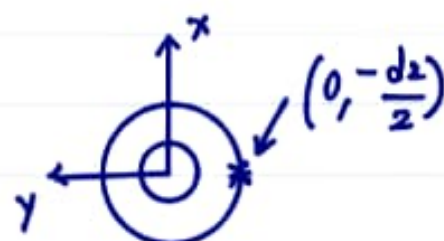
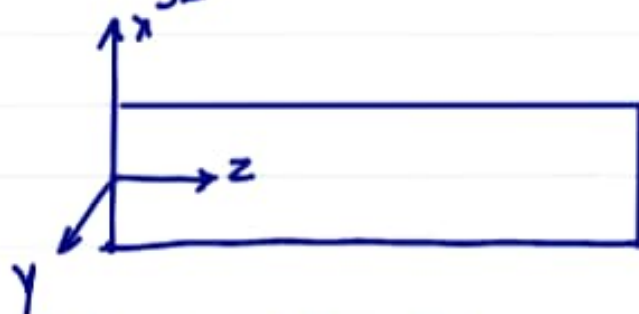
$$\text{Power} = T\omega = T 2\pi f$$

$$\begin{aligned} \text{Torque } T &= \frac{1800 \times 10^3}{2\pi \times 4} \\ &= 71.62 \times 10^3 \text{ Nmm} \\ P &= 540 \times 10^3 \text{ W} \end{aligned}$$

$$d_2 = 300 \text{ mm}, d_1 = 250 \text{ mm}$$

$$A = \frac{\pi}{4} (d_2^2 - d_1^2) = 21,598 \text{ mm}^2$$

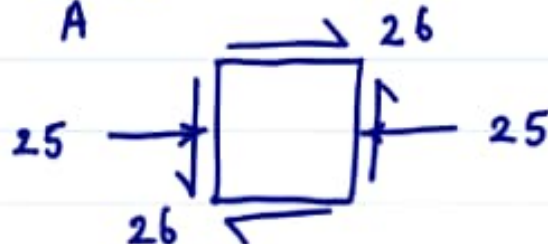
$$J = \frac{\pi}{32} (d_2^4 - d_1^4) = 411,720,443 \text{ mm}^4$$



$$\tau_{yz} = +G\alpha = 0$$

$$\tau_{xz} = -G\alpha y = \frac{T}{J} \frac{d_2}{2} = 26 \text{ N/mm}^2 = 26 \text{ MPa}$$

$$\sigma_{zz} = -\frac{P}{A} = -25 \text{ N/mm}^2$$



Principal Stresses

$$\sigma_1, \sigma_2 = \frac{\sigma_{zz} + \sigma_{xx}}{2} \pm \sqrt{\left(\frac{\sigma_{zz} - \sigma_{xx}}{2}\right)^2 + \tau_{zx}^2}$$

$$\sigma_1 = 16 \text{ MPa}, \sigma_2 = -41 \text{ MPa}$$

$$\text{Max Tensile Stress} = 16 \text{ MPa}$$

$$\text{Max Comp Stress} = 41 \text{ MPa}$$

$$\text{Max Shear Stress} = \frac{\sigma_1 - \sigma_2}{2} = 28.5 \text{ MPa}$$

Problem 3

$$\sigma_{zz} = \sigma_{zz}^{\text{bending}} = \frac{32 M}{\pi D^3}$$

$$\sigma_{zx} = \frac{16 T}{\pi D^3} \quad \text{given}$$

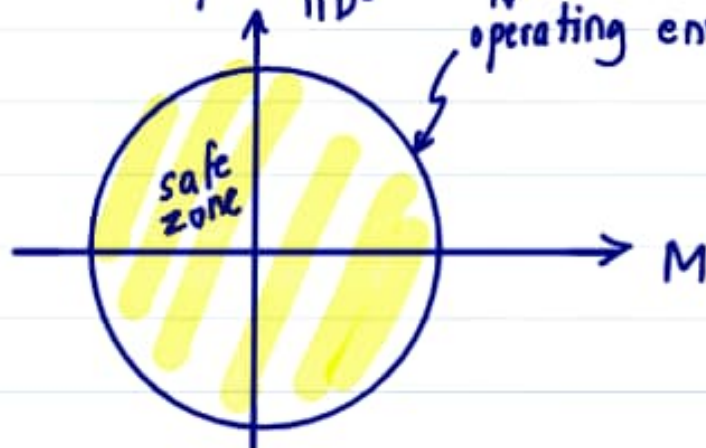
$$\tau_{\max} = \sqrt{\left(\frac{\sigma_{zz}}{2}\right)^2 + \sigma_{zx}^2} = S$$

$$\sqrt{\left(\frac{16 M}{\pi D^3}\right)^2 + \left(\frac{16 T}{\pi D^3}\right)^2} = S$$

$$\frac{(M^2 + T^2)^{1/2}}{T} \cdot \frac{16}{\pi D^3} = \frac{S}{N}$$

factor of safety > 1

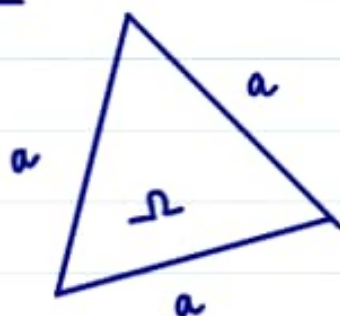
operating envelope



HW: P, M, T space

Problem 4

4.1



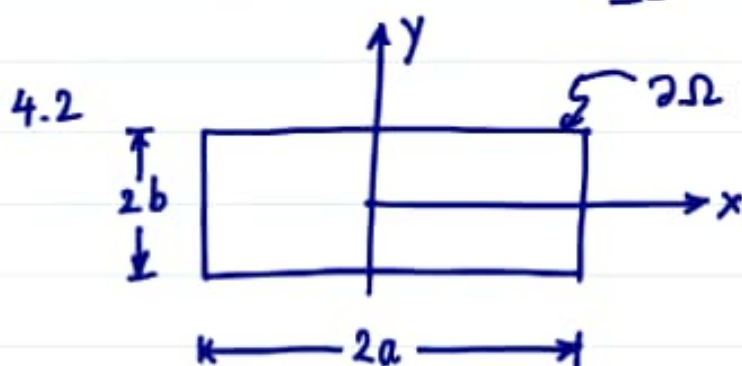
Goal $T = K_t \alpha$

Find $\varphi(x, y)$ s.t. $\nabla^2 \varphi = -2G\alpha$ inside Ω

$\varphi = 0$ on $\partial\Omega$

$$\varphi = K \underbrace{F(x, y)}$$

$$T = 2 \int_{\Omega} \varphi \, da$$



Try Same trick $\varphi(x, y) = K(x-a)(x+a)(y-b)(y+b)$
 $= K(x^2 - a^2)(y^2 - b^2)$

$$\varphi = 0 \text{ on } \partial\Omega$$

$$\nabla^2 \varphi = \frac{\partial^2 \varphi}{\partial x^2} + \frac{\partial^2 \varphi}{\partial y^2} = K(y^2 - b^2)2 + K(x^2 - a^2)2$$

$$\neq -2\epsilon a \quad \forall x, y \in \Omega$$

↑
for all

Work: Fourier series

Consider, thin rectangle $a \gg b$

