

# Control Systems

Continuous linear time-invariant systems (LT)

$$\begin{aligned} \text{"easy"} \\ \left\{ \begin{array}{l} \dot{q}(t) = A q(t) + B f(t) \\ y(t) = C q(t) + D f(t) \end{array} \right. \end{aligned}$$

Continuous linear time variant systems

$$\begin{aligned} \text{"hard"} \\ \left\{ \begin{array}{l} \dot{q}(t) = A(t) q(t) + B(t) f(t) \\ y(t) = C(t) q(t) + D(t) f(t) \end{array} \right. \end{aligned}$$

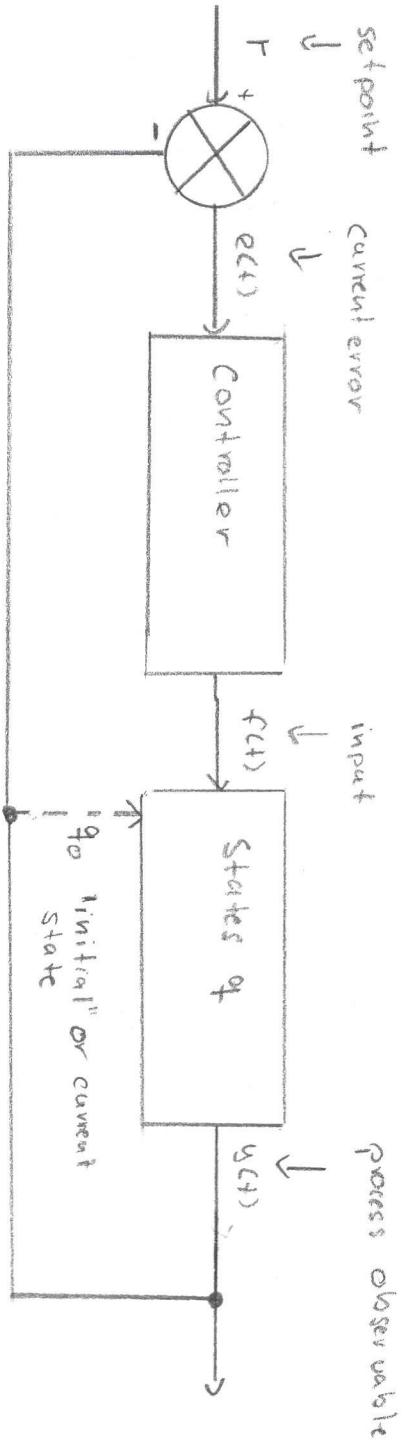
↑ linear but coefficients are time dependent

Non-linear systems

→ no matrix form

$$\begin{aligned} \text{"hard"} \\ \left\{ \begin{array}{l} \dot{q}_1 = a q_1 + b q_2 + c q_1 q_2 \\ \dot{q}_2 = -d q_1^3 + e \end{array} \right. \end{aligned}$$

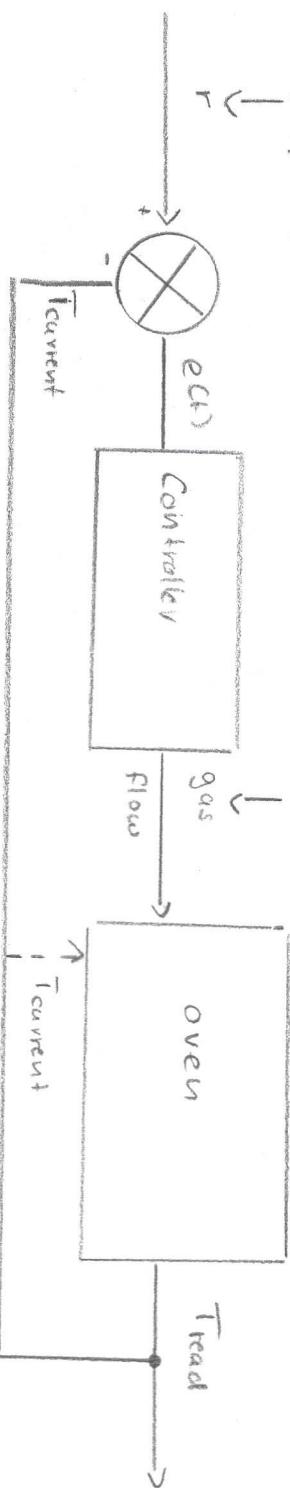
$$\begin{aligned} q' &= Aq + Bf \\ y &= Cq + Df \end{aligned} \quad \left. \begin{array}{l} \text{LTI system} \\ \text{ } \end{array} \right\}$$



Control problem: how to set input  $f(t)$  to drive the system from current state  $q_0$  to desired observable  $y(t)$

Example : oven

SP: Set point



MP: Manipulated Variable (Value per voltage)

Physical model requires knowledge  
of  $V$  vs. gas flow rate

t	$T_{read}$	$e(+) MV$
0	150	250 10 V
$10^5$	180	220 9 V
$2 \cdot 10^5$	200	200 8 V
$3 \cdot 10^5$	1	1 1 V
1	1	1 1 V

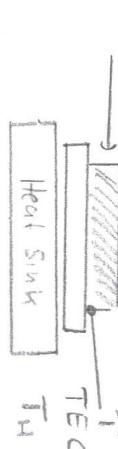
Model ( $e(T)$ ) if available or observe

- know  $e(+)$  current
- know  $e(+)$  past
- no knowledge of future

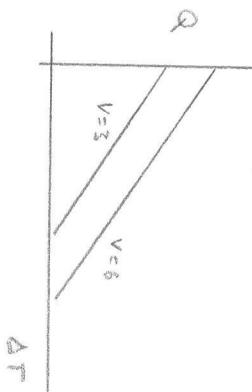
"controller is mapping  $e(+) to MV$ "

## Concrete Example : Cold Stage

$$\overline{T}_{env} \quad \text{heat flow [w]} = \left[ \frac{J}{S} \right]$$



$$Q(v, \Delta T) = a_1 \sqrt{v} + a_2 (T_H - \overline{T}) + a_3$$



How does  $\overline{T}$  change?

$$\frac{d\overline{T}}{dt} = k (\overline{T}_{env} - \overline{T}) = \frac{Q}{mc_p}$$

} physical model  
 of system

\underbrace{\qquad\qquad\qquad}\_{\text{Newton's law}} \quad \underbrace{\qquad\qquad\qquad}\_{\text{heat capacity}}

heat flux  
 + storage  
 of cooling

Convert model to state space form

$$\frac{d\bar{T}}{dt} = k\bar{T}_{env} - k\bar{T} - \frac{\alpha_1}{mc}\bar{V} + \frac{\alpha_2}{mc}\bar{T} - \frac{\alpha_3}{mc} + \frac{\alpha_2}{mc}\bar{T}_H$$

$$\frac{d\bar{T}}{dt} = (-k + \frac{\alpha_2}{mc})\bar{T} - \frac{\alpha_1}{mc}\bar{V} - \frac{\alpha_2}{mc}\bar{T}_H - \frac{\alpha_3}{mc} + k\bar{T}_{env}$$

State variables ( $s$ ):  $\begin{cases} \bar{q} = \bar{T} \\ \bar{f}_v = V_{env}, \bar{T}_H \\ \bar{y} = \bar{T} \end{cases}$

Inputs

Output ( $s$ )

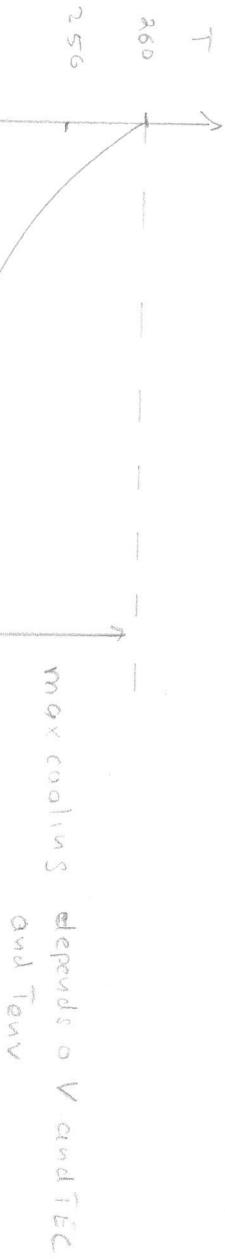
$$\bar{q} = \bar{T}$$

$$\bar{q}' = \bar{T}'$$

$$\begin{aligned} \bar{q}' &= \underbrace{\left[ -k + \frac{\alpha_2}{mc} \right] \bar{q}'}_{A} + \underbrace{\left[ -\frac{\alpha_1}{mc} - \frac{\alpha_2}{mc} - \frac{\alpha_3}{mc} - k \right]}_{B} \begin{bmatrix} \bar{V} \\ \bar{T}_H \\ \bar{T}_{env} \end{bmatrix} \\ \bar{q}' &= \left[ -k + \frac{\alpha_2}{mc} \right] \bar{q}' + \left[ -\frac{\alpha_1}{mc} - \frac{\alpha_2}{mc} - \frac{\alpha_3}{mc} - k \right] \begin{bmatrix} \bar{V} \\ \bar{T}_H \\ \bar{T}_{env} \end{bmatrix} \end{aligned}$$

$$y = \underbrace{\begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} \bar{q}'}_C + \underbrace{\begin{bmatrix} 0 & 0 & 0 \end{bmatrix} \bar{f}_v}_D$$

# Simple Solution : ODE solver



```
function odc(q, p, t)
```

$$A = \rho C_1 D$$

$$B = \rho L_2 D$$

$$f = \rho L_3 D$$

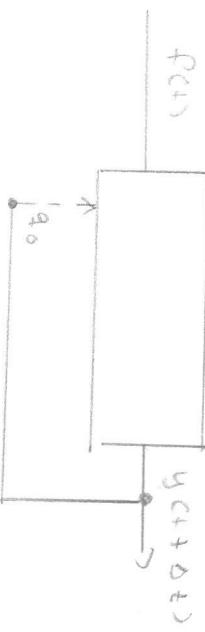
```
return Aq + Bf
```

```
end
```

# Discrete Event Simulation

$$\dot{q}^i = A q^i + B f$$

$$y = C q + D f$$



- initialize ODE at  $t=0$  with IVVP  $q_0, f$

- integrate to  $t + \Delta t$

- evaluate solution  $\rightarrow$  build time array! possibly update  $f$

$\leftarrow$   
loop



$\therefore$   $f$  may change each  $\Delta t$

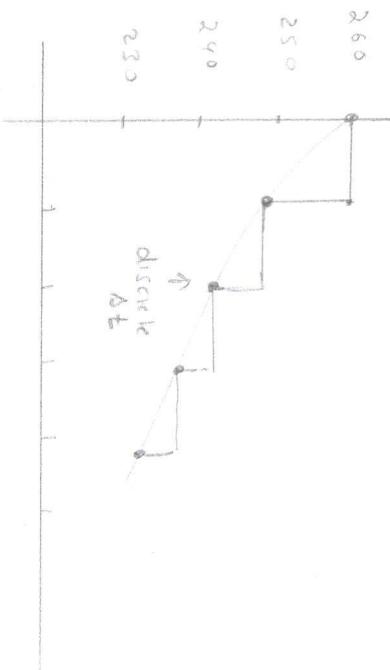
60

40

20

0

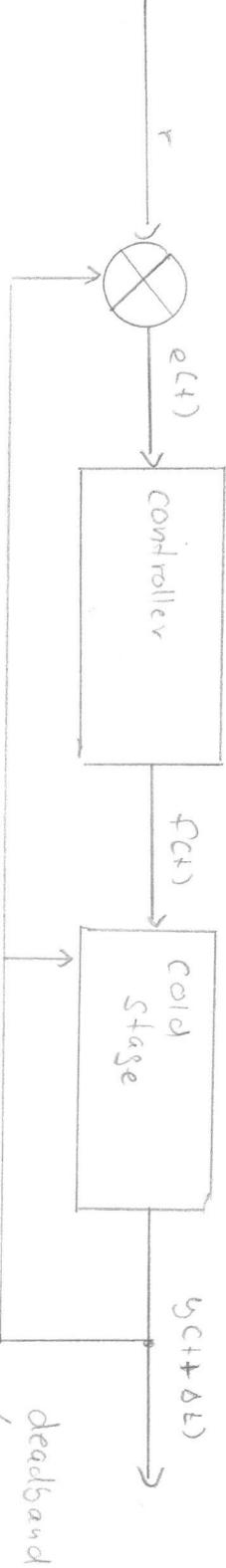
## Discrete Event Example



# On/Off Control

$r = 300$

$$+1 = \text{cooling} \rightarrow TEC = +12V \\ -1 = \text{heating} \rightarrow -12V$$



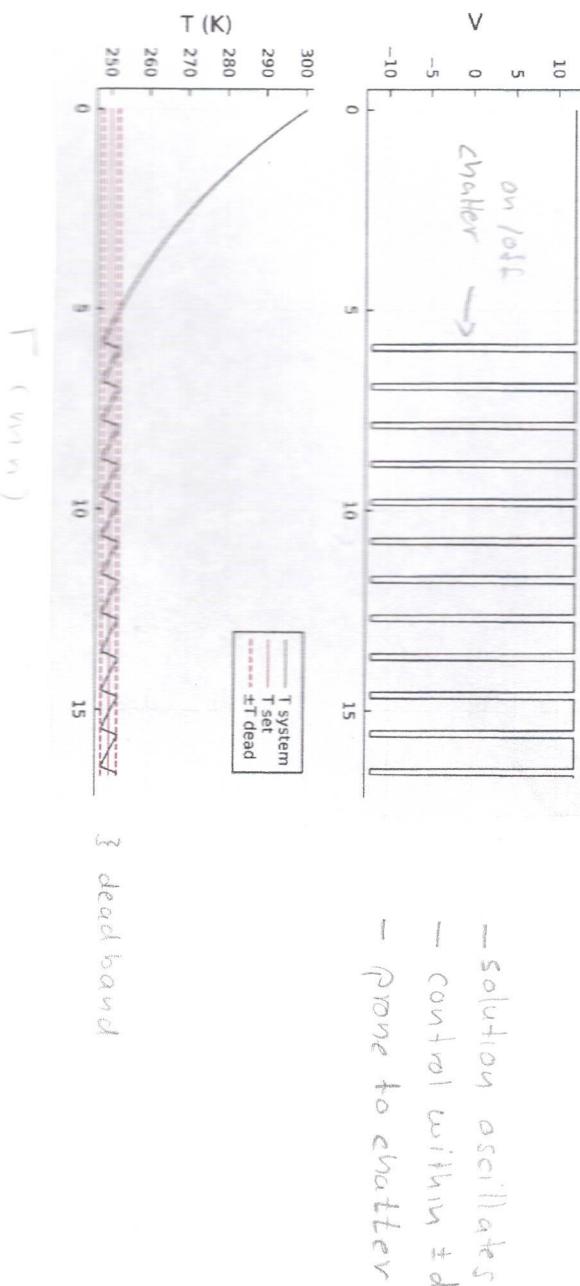
$$e(t) = r - y(t)$$

$$f(t) = \begin{cases} +1 & e(t) + d \leq 0 \wedge e(t + \Delta t) + d > 0 \\ -1 & e(t) - d > 0 \wedge e(t + \Delta t) - d \leq 0 \\ \text{no change} & \text{else} \end{cases}$$

$\Delta t$  past error

t	y(t)	e(t)	e(t)+d	e(t)-d	f(t)
0	303	-3	-1	-5	+1 cooling
1	295	+5	+7	+3	-1 heating
2	299	+1	+3	-1	-1 (no change) in dead band
3	304	-4	-2	-6	+1 cooling
4	-	-	-	-	-

# Example Cold Stage



## Proportional Control

error

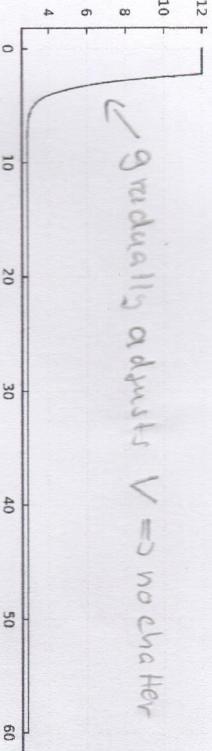
$$g(C+) = \frac{K_p \underbrace{(r - y_{C+})}_{\text{error}}}{S} + P_0$$

$$f(C+) = \begin{cases} -1 & g(C+) < -1 \\ 1 & g(C+) > 1 \\ g(C+) & \text{else} \end{cases}$$

$$T_{set} \quad 260$$

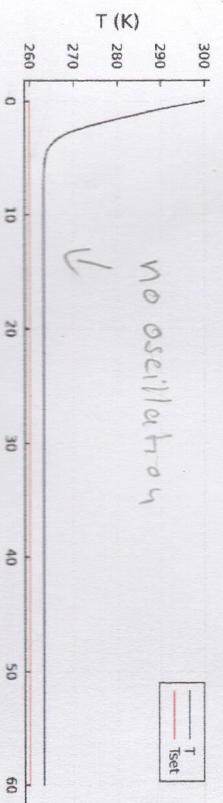
$$T_{span} \quad 20 K_p \quad 1.5 p_0 \quad 0.0$$

$\checkmark$  gradually adjusts  $\checkmark \Rightarrow$  no chattering



$\boxed{T_{set}}$

$\checkmark$  no oscillation



$\checkmark$  offset error eliminated

} offset error can not be

Span: 100% output  
if not within  $\pm S$

# Proportional Integral Control (PI)

Integral over past error

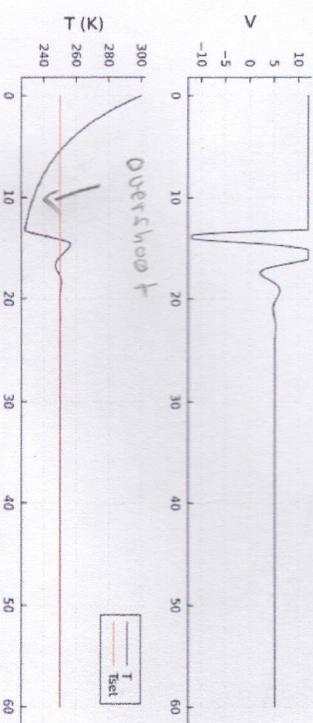
$$e(t) = \frac{r - y(t)}{g(t)}$$

$$g(t) = k_p e(t) + k_i \int_0^t e(\tau) d\tau$$

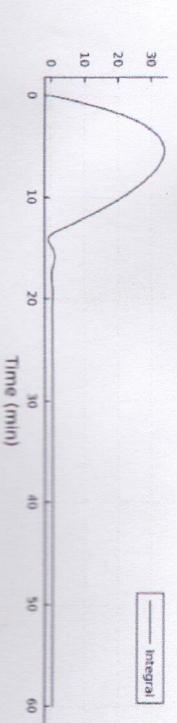
$$f(t) = \begin{cases} -1 & g(t) < -1 \\ 1 & g(t) > 1 \\ 0 & \text{else} \end{cases}$$



Solution is sensitive to integral gain



- Oscillating solution that generally but not always converges
- No offset error



Integral

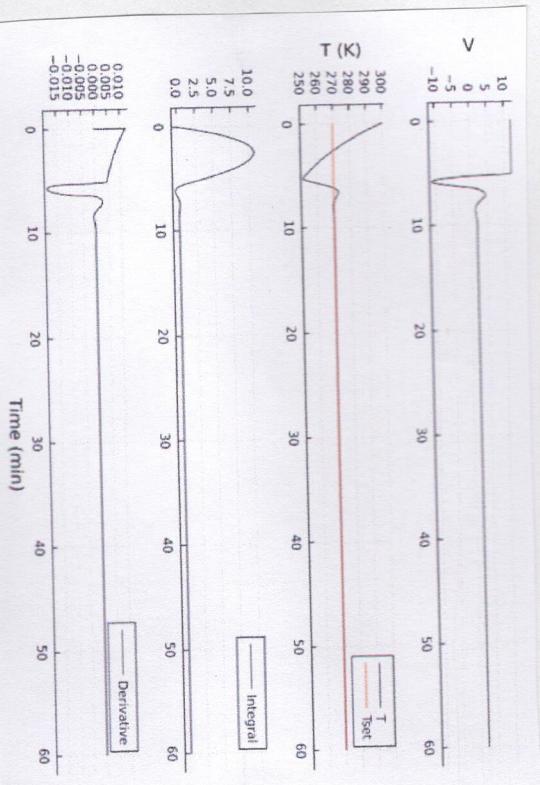
Time (min)

Proportional - Integral - Derivative (PID) control

Same as PI except

$$g(C+) = K_p e(C+) + K_i \int_0^t e(C) dt + K_d \frac{de(C+)}{dt}$$

derivative gain



D-term can

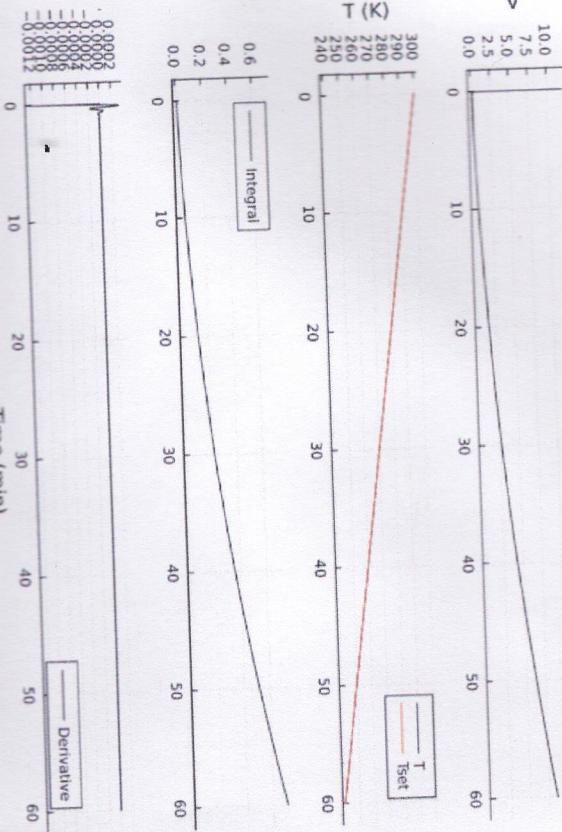
- increase convergence rates
  - be susceptible to noise
- amplification

# Trajectory Control

Setpoint  $r$  is treated as  $r(t)$



- avoids oscillations
- avoids overshoot
- controlled way to steer system



## Hardware Controllers (special domain)

### PID Temperature Controller



- Programmable  $T_{set}$ ,  $T_{span}$ ,  $K_p$ ,  $K_i$ ,  $K_d$  and span
- Reads temp (thermistor)
- modulates input V to ± output
- ✓ for thermocouples

# Formal Reasoning about Control Systems

forcing

output



$$f(s) = a_n y^{(n)} + \dots + a_1 y' + a_0 y$$

(zero initial conditions)

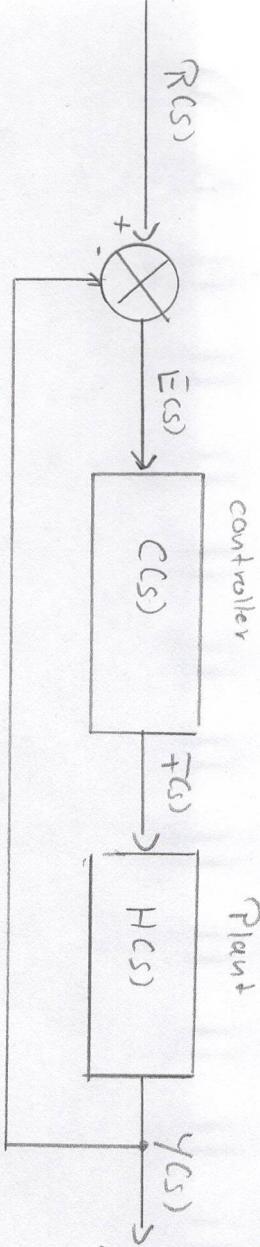
$$\mathcal{L} \bar{f}(s) = (a_n s^n + \dots + a_1 s + a_0) Y(s)$$

$$H(s) = \frac{Y(s)}{\bar{f}(s)} = \frac{1}{a_n s^n + \dots + a_0}$$

transfer function

$$\begin{aligned} Lf'' &= s^2 \bar{f} + sf(0) - f'(0) \\ Lf' &= s\bar{f} - sf(0) \\ Laf &= a\bar{f} \end{aligned}$$

# Closed Loop Control System



Controller

$$f(c+) = K_p e(c+) + K_i \int_0^t e(c-) d\tau + K_d \frac{de(c+)}{dt}$$

$$C(s) = K_p + \frac{K_i}{s} + K_d s$$

$$\frac{Y(s)}{R(s)} = \frac{C(s)H(s)}{1 + C(s)H(s)}$$

use this for  
Stability analysis

System  
transfer  
function

- roots in numerator  $\Rightarrow$  zeros  $\Rightarrow$  stabilize system
- roots in denominator  $\Rightarrow$  poles  $\Rightarrow$  destabilize system
- can add noise filters to CCS

$\mathcal{L}^{-1}(Y(s)) = y(t)$  : analytical analog to  
Simulation

## Controllability

$$\dot{q}' = Aq' + Bf$$

→ Reach any state  $\dot{q}'$  using control system

$\dot{q}'$  is  $n \times 1$

$$R = [B \quad AB \quad A^2B \quad \dots \quad A^{n-1}B]$$

↑  
controllability matrix

→ System is controllable if R is full row rank i.e.  $\text{rank}(R) = n$

↳ derive through  $L\dot{q}'$