

A Design Study Approach to Classical Control

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Homework D.16

For the mass spring, use the `bode` command (from Python or Matlab) to create a graph that simultaneously displays the Bode plots for (1) the plant, and (2) the plant under PID control, using the control gains calculated in Homework [D.10](#).

- (a) What is the tracking error to a unit ramp under PID control?
- (b) If the frequency content of the input disturbance $d_{in}(t)$ is below $\omega_{d_{in}} = 0.1$ radians per second, what percentage of the input disturbance shows up in the output z under PID control?
- (c) If all of the frequency content of the noise $n(t)$ is greater than $\omega_{no} = 100$ radians per second, what percentage of the noise shows up in the output signal z ?

Solution

The Bode plot of the plant $P(s)$, and the loop gain with PD control $P(s)C_{PD}(s)$, and the loop gain with PID control $P(s)C_{PID}(s)$ are shown in Figure [1](#).

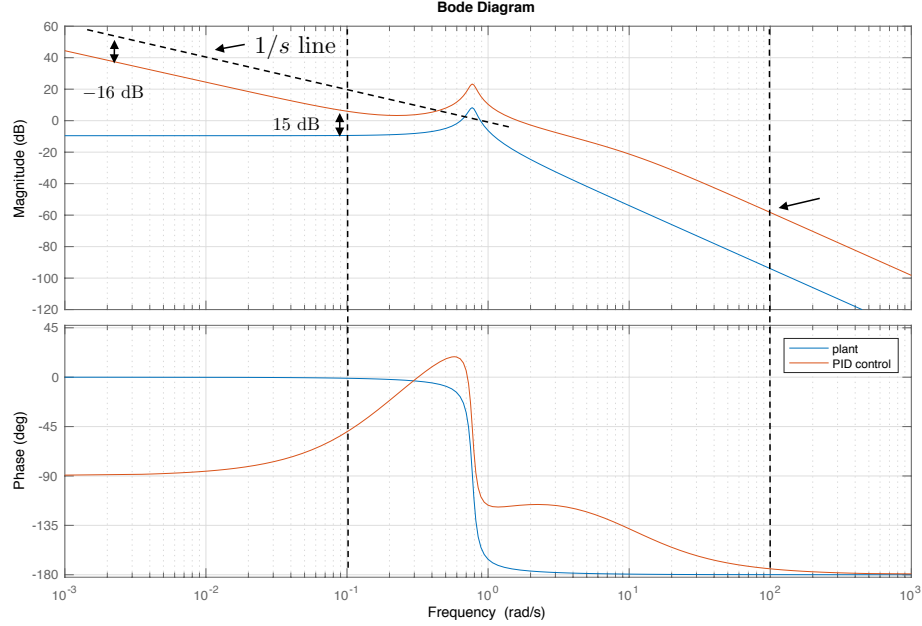


Figure 1: Bode plot for mass spring damper, plant only, and under PID control.

(a) From Figure 1 we see that as $\omega \rightarrow 0$, the Bode magnitude for PID control satisfies $20 \log_{10} |P(j\omega)C(j\omega)| - 20 \log_{10} \left| \frac{1}{j\omega} \right| \rightarrow B_1 = -16$ dB. Therefore, the steady state tracking error is

$$\lim_{t \rightarrow \infty} |e(t)| = \frac{1}{M_v},$$

where $M_v = 10^{B_1/20} = 0.1585$, which implies that the steady state error is 6.3096.

(b) For $\omega \leq \omega_{din} = 0.1$ rad/s, we have

$$20 \log_{10} |P(j\omega)C_{PD}(j\omega)| - 20 \log_{10} |P(j\omega)| \rightarrow B_{din} = 15 \text{ dB}.$$

Therefore,

$$\gamma_{din} = 10^{-15/20} = 0.1778,$$

implying that 18% of the input disturbance will show up in the output.

(c) For $\omega \geq \omega_n = 100$ rad/sec, we see from Figure 1 that $B_{no} = -58$ dB. Therefore, $\gamma_n = 10^{-58/20} = 0.0013$ which implies that 0.13% of the noise will show up in the output signal.