

# 2

# KINEMATICS OF PARTICLES

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## 2/1 INTRODUCTION

Kinematics is the branch of dynamics which describes the motion of bodies without reference to the forces which either cause the motion or are generated as a result of the motion. Kinematics is often described as the “geometry of motion.” Some engineering applications of kinematics include the design of cams, gears, linkages, and other machine elements to control or produce certain desired motions, and the calculation of flight trajectories for aircraft, rockets, and spacecraft. A thorough working knowledge of kinematics is a prerequisite to kinetics, which is the study of the relationships between motion and the corresponding forces which cause or accompany the motion.

### Particle Motion

We begin our study of kinematics by first discussing in this chapter the motions of points or particles. A particle is a body whose physical dimensions are so small compared with the radius of curvature of its path that we may treat the motion of the particle as that of a point. For example, the wingspan of a jet transport flying between Los Angeles and New York is of no consequence compared with the radius of curvature of

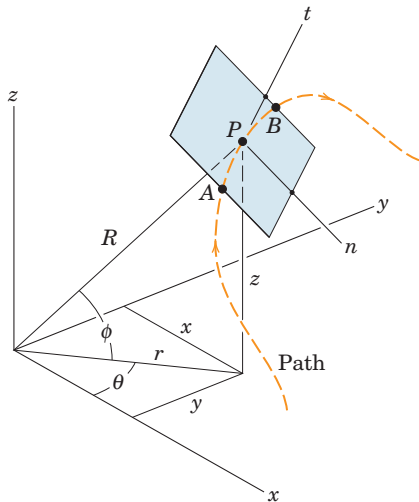


Figure 2/1

its flight path, and thus the treatment of the airplane as a particle or point is an acceptable approximation.

We can describe the motion of a particle in a number of ways, and the choice of the most convenient or appropriate way depends a great deal on experience and on how the data are given. Let us obtain an overview of the several methods developed in this chapter by referring to Fig. 2/1, which shows a particle  $P$  moving along some general path in space. If the particle is confined to a specified path, as with a bead sliding along a fixed wire, its motion is said to be *constrained*. If there are no physical guides, the motion is said to be *unconstrained*. A small rock tied to the end of a string and whirled in a circle undergoes constrained motion until the string breaks, after which instant its motion is unconstrained.

### Choice of Coordinates

The position of particle  $P$  at any time  $t$  can be described by specifying its rectangular coordinates\*  $x, y, z$ , its cylindrical coordinates  $r, \theta, z$ , or its spherical coordinates  $R, \theta, \phi$ . The motion of  $P$  can also be described by measurements along the tangent  $t$  and normal  $n$  to the curve. The direction of  $n$  lies in the local plane of the curve.<sup>†</sup> These last two measurements are called *path variables*.

The motion of particles (or rigid bodies) can be described by using coordinates measured from fixed reference axes (*absolute-motion* analysis) or by using coordinates measured from moving reference axes (*relative-motion* analysis). Both descriptions will be developed and applied in the articles which follow.

With this conceptual picture of the description of particle motion in mind, we restrict our attention in the first part of this chapter to the case of *plane motion* where all movement occurs in or can be represented as occurring in a single plane. A large proportion of the motions of machines and structures in engineering can be represented as plane motion. Later, in Chapter 7, an introduction to three-dimensional motion is presented. We begin our discussion of plane motion with *rectilinear motion*, which is motion along a straight line, and follow it with a description of motion along a plane curve.

## 2/2 RECTILINEAR MOTION

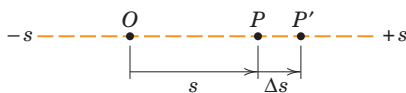


Figure 2/2

Consider a particle  $P$  moving along a straight line, Fig. 2/2. The position of  $P$  at any instant of time  $t$  can be specified by its distance  $s$  measured from some convenient reference point  $O$  fixed on the line. At time  $t + \Delta t$  the particle has moved to  $P'$  and its coordinate becomes  $s + \Delta s$ . The change in the position coordinate during the interval  $\Delta t$  is called the *displacement*  $\Delta s$  of the particle. The displacement would be negative if the particle moved in the negative  $s$ -direction.

\*Often called *Cartesian* coordinates, named after René Descartes (1596–1650), a French mathematician who was one of the inventors of analytic geometry.

<sup>†</sup>This plane is called the *osculating* plane, which comes from the Latin word *osculari* meaning “to kiss.” The plane which contains  $P$  and the two points  $A$  and  $B$ , one on either side of  $P$ , becomes the osculating plane as the distances between the points approach zero.

## Velocity and Acceleration

The average velocity of the particle during the interval  $\Delta t$  is the displacement divided by the time interval or  $v_{av} = \Delta s / \Delta t$ . As  $\Delta t$  becomes smaller and approaches zero in the limit, the average velocity approaches the *instantaneous velocity* of the particle, which is  $v = \lim_{\Delta t \rightarrow 0} \frac{\Delta s}{\Delta t}$  or

$$v = \frac{ds}{dt} = \dot{s} \quad (2/1)$$

Thus, the velocity is the time rate of change of the position coordinate  $s$ . The velocity is positive or negative depending on whether the corresponding displacement is positive or negative.

The average acceleration of the particle during the interval  $\Delta t$  is the change in its velocity divided by the time interval or  $a_{av} = \Delta v / \Delta t$ . As  $\Delta t$  becomes smaller and approaches zero in the limit, the average acceleration approaches the *instantaneous acceleration* of the particle, which is

$$a = \lim_{\Delta t \rightarrow 0} \frac{\Delta v}{\Delta t} \text{ or}$$

$$a = \frac{dv}{dt} = \dot{v} \quad \text{or} \quad a = \frac{d^2s}{dt^2} = \ddot{s} \quad (2/2)$$

The acceleration is positive or negative depending on whether the velocity is increasing or decreasing. Note that the acceleration would be positive if the particle had a negative velocity which was becoming less negative. If the particle is slowing down, the particle is said to be *decelerating*.

Velocity and acceleration are actually vector quantities, as we will see for curvilinear motion beginning with Art. 2/3. For rectilinear motion in the present article, where the direction of the motion is that of the given straight-line path, the sense of the vector along the path is described by a plus or minus sign. In our treatment of curvilinear motion, we will account for the changes in direction of the velocity and acceleration vectors as well as their changes in magnitude.

By eliminating the time  $dt$  between Eq. 2/1 and the first of Eqs. 2/2, we obtain a differential equation relating displacement, velocity, and acceleration.\* This equation is

$$v \, dv = a \, ds \quad \text{or} \quad \dot{s} \, d\dot{s} = \ddot{s} \, ds \quad (2/3)$$

Equations 2/1, 2/2, and 2/3 are the differential equations for the rectilinear motion of a particle. Problems in rectilinear motion involving finite changes in the motion variables are solved by integration of these basic differential relations. The position coordinate  $s$ , the velocity  $v$ , and the acceleration  $a$  are algebraic quantities, so that their signs, positive or negative, must be carefully observed. Note that the positive directions for  $v$  and  $a$  are the same as the positive direction for  $s$ .

\*Differential quantities can be multiplied and divided in exactly the same way as other algebraic quantities.



This sprinter will undergo rectilinear acceleration until he reaches his terminal speed.

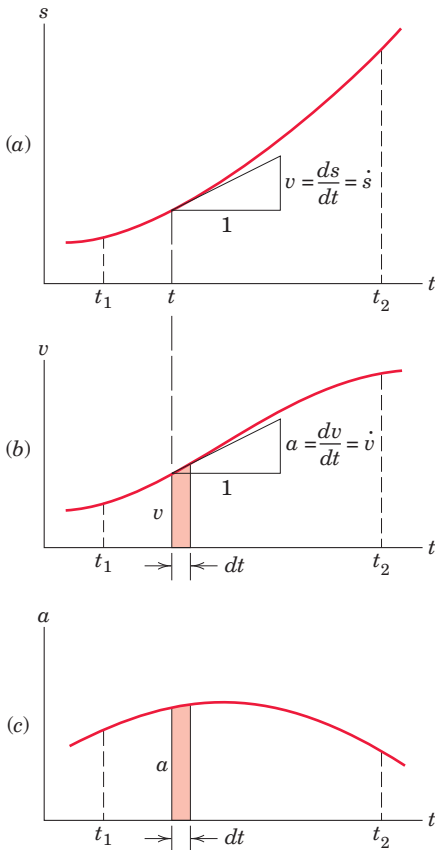


Figure 2/3

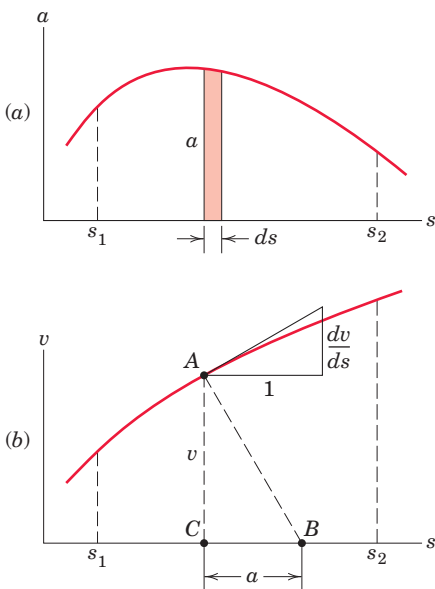


Figure 2/4

### Graphical Interpretations

Interpretation of the differential equations governing rectilinear motion is considerably clarified by representing the relationships among  $s$ ,  $v$ ,  $a$ , and  $t$  graphically. Figure 2/3a is a schematic plot of the variation of  $s$  with  $t$  from time  $t_1$  to time  $t_2$  for some given rectilinear motion. By constructing the tangent to the curve at any time  $t$ , we obtain the slope, which is the velocity  $v = ds/dt$ . Thus, the velocity can be determined at all points on the curve and plotted against the corresponding time as shown in Fig. 2/3b. Similarly, the slope  $dv/dt$  of the  $v$ - $t$  curve at any instant gives the acceleration at that instant, and the  $a$ - $t$  curve can therefore be plotted as in Fig. 2/3c.

We now see from Fig. 2/3b that the area under the  $v$ - $t$  curve during time  $dt$  is  $v dt$ , which from Eq. 2/1 is the displacement  $ds$ . Consequently, the net displacement of the particle during the interval from  $t_1$  to  $t_2$  is the corresponding area under the curve, which is

$$\int_{s_1}^{s_2} ds = \int_{t_1}^{t_2} v dt \quad \text{or} \quad s_2 - s_1 = (\text{area under } v\text{-}t \text{ curve})$$

Similarly, from Fig. 2/3c we see that the area under the  $a$ - $t$  curve during time  $dt$  is  $a dt$ , which, from the first of Eqs. 2/2, is  $dv$ . Thus, the net change in velocity between  $t_1$  and  $t_2$  is the corresponding area under the curve, which is

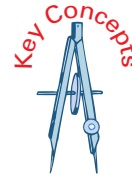
$$\int_{v_1}^{v_2} dv = \int_{t_1}^{t_2} a dt \quad \text{or} \quad v_2 - v_1 = (\text{area under } a\text{-}t \text{ curve})$$

Note two additional graphical relations. When the acceleration  $a$  is plotted as a function of the position coordinate  $s$ , Fig. 2/4a, the area under the curve during a displacement  $ds$  is  $a ds$ , which, from Eq. 2/3, is  $v dv = d(v^2/2)$ . Thus, the net area under the curve between position coordinates  $s_1$  and  $s_2$  is

$$\int_{v_1}^{v_2} v dv = \int_{s_1}^{s_2} a ds \quad \text{or} \quad \frac{1}{2}(v_2^2 - v_1^2) = (\text{area under } a\text{-}s \text{ curve})$$

When the velocity  $v$  is plotted as a function of the position coordinate  $s$ , Fig. 2/4b, the slope of the curve at any point  $A$  is  $dv/ds$ . By constructing the normal  $AB$  to the curve at this point, we see from the similar triangles that  $CB/v = dv/ds$ . Thus, from Eq. 2/3,  $CB = v(dv/ds) = a$ , the acceleration. It is necessary that the velocity and position coordinate axes have the same numerical scales so that the acceleration read on the position coordinate scale in meters (or feet), say, will represent the actual acceleration in meters (or feet) per second squared.

The graphical representations described are useful not only in visualizing the relationships among the several motion quantities but also in obtaining approximate results by graphical integration or differentiation. The latter case occurs when a lack of knowledge of the mathematical relationship prevents its expression as an explicit mathematical function which can be integrated or differentiated. Experimental data and motions which involve discontinuous relationships between the variables are frequently analyzed graphically.



## ANALYTICAL INTEGRATION

If the position coordinate  $s$  is known for all values of the time  $t$ , then successive mathematical or graphical differentiation with respect to  $t$  gives the velocity  $v$  and acceleration  $a$ . In many problems, however, the functional relationship between position coordinate and time is unknown, and we must determine it by successive integration from the acceleration. Acceleration is determined by the forces which act on moving bodies and is computed from the equations of kinetics discussed in subsequent chapters. Depending on the nature of the forces, the acceleration may be specified as a function of time, velocity, or position coordinate, or as a combined function of these quantities. The procedure for integrating the differential equation in each case is indicated as follows.

**(a) Constant Acceleration.** When  $a$  is constant, the first of Eqs. 2/2 and 2/3 can be integrated directly. For simplicity with  $s = s_0$ ,  $v = v_0$ , and  $t = 0$  designated at the beginning of the interval, then for a time interval  $t$  the integrated equations become

$$\int_{v_0}^v dv = a \int_0^t dt \quad \text{or} \quad v = v_0 + at$$

$$\int_{v_0}^v v dv = a \int_{s_0}^s ds \quad \text{or} \quad v^2 = v_0^2 + 2a(s - s_0)$$

Substitution of the integrated expression for  $v$  into Eq. 2/1 and integration with respect to  $t$  give

$$\int_{s_0}^s ds = \int_0^t (v_0 + at) dt \quad \text{or} \quad s = s_0 + v_0 t + \frac{1}{2}at^2$$

These relations are necessarily restricted to the special case where the acceleration is constant. The integration limits depend on the initial and final conditions, which for a given problem may be different from those used here. It may be more convenient, for instance, to begin the integration at some specified time  $t_1$  rather than at time  $t = 0$ .

**Caution:** The foregoing equations have been integrated for constant acceleration only. A common mistake is to use these equations for problems involving variable acceleration, where they do not apply.

**(b) Acceleration Given as a Function of Time,  $a = f(t)$ .** Substitution of the function into the first of Eqs. 2/2 gives  $f(t) = dv/dt$ . Multiplying by  $dt$  separates the variables and permits integration. Thus,

$$\int_{v_0}^v dv = \int_0^t f(t) dt \quad \text{or} \quad v = v_0 + \int_0^t f(t) dt$$

From this integrated expression for  $v$  as a function of  $t$ , the position coordinate  $s$  is obtained by integrating Eq. 2/1, which, in form, would be

$$\int_{s_0}^s ds = \int_0^t v dt \quad \text{or} \quad s = s_0 + \int_0^t v dt$$

If the indefinite integral is employed, the end conditions are used to establish the constants of integration. The results are identical with those obtained by using the definite integral.

If desired, the displacement  $s$  can be obtained by a direct solution of the second-order differential equation  $\ddot{s} = f(t)$  obtained by substitution of  $f(t)$  into the second of Eqs. 2/2.

**(c) Acceleration Given as a Function of Velocity,  $a = f(v)$ .** Substitution of the function into the first of Eqs. 2/2 gives  $f(v) = dv/dt$ , which permits separating the variables and integrating. Thus,

$$t = \int_0^t dt = \int_{v_0}^v \frac{dv}{f(v)}$$

This result gives  $t$  as a function of  $v$ . Then it would be necessary to solve for  $v$  as a function of  $t$  so that Eq. 2/1 can be integrated to obtain the position coordinate  $s$  as a function of  $t$ .

Another approach is to substitute the function  $a = f(v)$  into the first of Eqs. 2/3, giving  $v dv = f(v) ds$ . The variables can now be separated and the equation integrated in the form

$$\int_{v_0}^v \frac{v dv}{f(v)} = \int_{s_0}^s ds \quad \text{or} \quad s = s_0 + \int_{v_0}^v \frac{v dv}{f(v)}$$

Note that this equation gives  $s$  in terms of  $v$  without explicit reference to  $t$ .

**(d) Acceleration Given as a Function of Displacement,  $a = f(s)$ .** Substituting the function into Eq. 2/3 and integrating give the form

$$\int_{v_0}^v v dv = \int_{s_0}^s f(s) ds \quad \text{or} \quad v^2 = v_0^2 + 2 \int_{s_0}^s f(s) ds$$

Next we solve for  $v$  to give  $v = g(s)$ , a function of  $s$ . Now we can substitute  $ds/dt$  for  $v$ , separate variables, and integrate in the form

$$\int_{s_0}^s \frac{ds}{g(s)} = \int_0^t dt \quad \text{or} \quad t = \int_{s_0}^s \frac{ds}{g(s)}$$

which gives  $t$  as a function of  $s$ . Finally, we can rearrange to obtain  $s$  as a function of  $t$ .

In each of the foregoing cases when the acceleration varies according to some functional relationship, the possibility of solving the equations by direct mathematical integration will depend on the form of the function. In cases where the integration is excessively awkward or difficult, integration by graphical, numerical, or computer methods can be utilized.