Virtual Differential

Development and verification

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# Introduction

The virtual differential is a subsystem of the complete all-wheel-drive electric vehicle under development. Its aim is to produce a reference wheel speed, for each of the four wheels, which will be tracked by the electric motor driving the wheel, with the eventual addition of correction factors for enhanced vehicle stability. The introduced differential model is called “virtual” as there not exist a mechanical component transferring power from a mechanical engine to the wheels; rather, the virtual differential will be implemented by an electronic control unit actively controlling actuators which will drive the wheels’ rotation.

The objective of this dissertation is to introduce the virtual differential model, and how it has been developed and verified.

In the following there will be introduced:

* the requirements of the virtual differential subsystem which drove the design;
* the development process;
* use and interface of the developed subsystem;
* the verification and validation processes

## System requirements

In order to interface with the model, the following software modules are required:

* MATLAB R2019b and Simulink

No additional libraries are required.

# Design requirements

The virtual differential model must provide the reference speeds for the four wheels of the all-wheel-drive electric vehicle.

A list of the requirements which drove the design is reported:

* The virtual differential must provide the reference speed of all four wheels of the electric vehicle.
* The speeds must be compatible with those provided by a mechanical differential on a standard vehicle; in particular, they should comply with the speeds provided by the Ackerman model, assuming a front-steering vehicle: when turning, outer wheels must rotate faster than inner wheels.
* The reference wheels’ speeds must conform with the desired vehicle speed, set by the driver through the throttle pedal; in case of advanced driving (ADAS) systems, the speed may be set by a properly functioning ADAS feature.
* The reference wheels’ speeds must be positive (and grater than 0) when vehicle is driving forward, negative (and smaller than 0) when vehicle is reverse moving, and null (equal to 0) when vehicle is (required to be) stopped.
* The virtual differential is allowed to take advantage of some of the vehicle’s signals generated by in-vehicle sensors or by other electronic control units, such as throttle pedal position and steering angle.

# Model-based design

The design of the virtual differential subsystem has been based on the Ackerman steering geometry assuming ideal vehicle and road conditions.

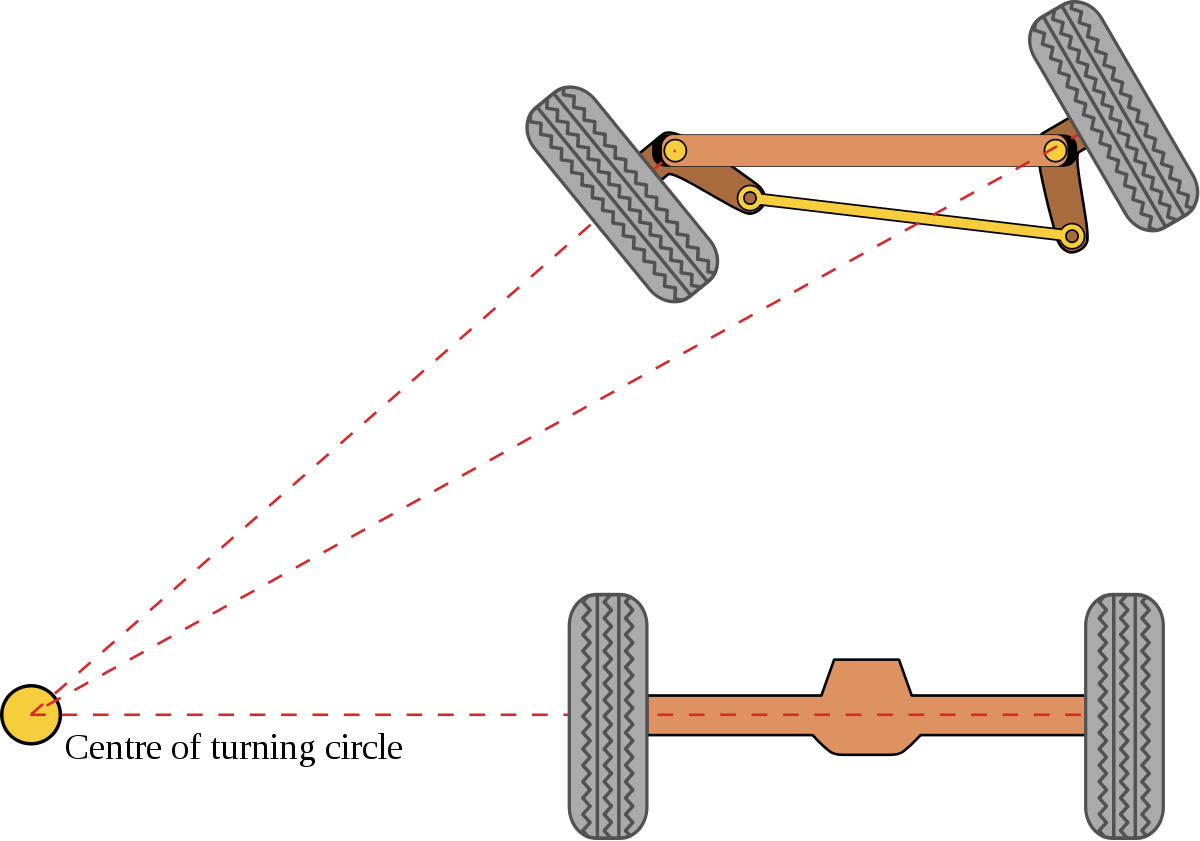
The assumptions for the design are:

* Ideal vehicle and road conditions, leading to the non-slip condition; road is dry asphalt, and vehicle moves on the road without any lateral skidding; eventual skidding will be addressed by other systems.
* Front and rear axles have equal length, and wheels have the same dimension (radius); these parameters are known, as well as the distance between front and rear axles.
* Throttle pedal position reflects the reference vehicle speed, and no other signal influence the requested vehicle speed; action of brake pedal will be addressed by other systems.
* There are sensors providing the instantaneous angular position of the two front wheels.

These assumptions conform with the other developed systems for the all-wheel-drive electric vehicle. The integration of the virtual differential subsystem with other vehicle’s systems introduce further complexity and eliminates some of the idealized assumptions.

The Ackerman steering model describes the reference steering geometry on which most of the vehicles rely on, as it introduces a situation in which none of the wheels of the vehicles is subject to longitudinal slip under ideal driving conditions. Variations from the model allow to reach different levels of performance. It is assumed that the all-wheel-drive electric vehicle under development implements a steering mechanism which conforms with the Ackerman steering model.

According to Ackerman steering geometry, at any steering angle the four vehicle’s wheels rotates around the same center of turning. It means that, during any turn, the lines perpendicular to the wheels’ axis meet at a single point, corresponding to the center of turning around which the vehicle is rotating. Assuming a front-steering vehicle, the center of turning lies on the line connecting the two rear wheels, corresponding, inside the vehicle, with the rear axle.



Once the geometry of the vehicle is known, the speeds of the four wheels can be computed from the knowledge of the steering angle and the (reference) speed of the vehicle.

Calling and the angles from the longitudinal axis of the vehicle to the longitudinal axes of the front wheels respectively, the distance between each front wheel to the center of turning is:

WB being the wheelbase, that is the distance between front and rear axles. The two angles of turning of the two front wheels must be known and they must comply with the Ackerman steering model, meaning that front wheels must be linked by a mechanism imposing Ackerman steering geometry.

The distance between each rear wheel to the center of turning is:

where wheel 3 is the wheel located behind front wheel 1 and wheel 4 is the wheel located behind front wheel 2. The relations do not change switching front wheels 1 and 2.

The angular speed at which the vehicle rotates around the center of turning can be computed as where is the (reference) vehicle speed and is the distance from the center of the vehicle to the center of turning, inferred from geometrical relations.

Finally, the angular speed of the four wheels is computed with the knowledge of the radius of the wheels as:

The angular speeds of the wheels comply with the Ackerman steering geometry and relates to steering angle and vehicle (reference) speed.

# Usage

The virtual differential model may be used whenever the ideal speeds of the four wheels of a vehicle have to be determined. The speeds are ideal as they assume no-slip condition and a vehicle implementing a steering mechanism compliant with Ackerman steering geometry.

In order to use the virtual differential model, some variable parameters, usually provided by in-vehicle sensors or electronic control units, and some fixed parameters, known from car geometry, must be available. These are described in more details below.

## System interfaces

The virtual differential model requires 2 input signals:

* **WhlAng** – 2x1 or 1x2 vector (rad)
  + Vector composed of two real scalar elements.
  + First vector element is the wheel angle of left wheel, second vector element is the wheel angle of the right wheel. The two elements represent the wheel angles of the two front wheels. A wheel angle is the angle between the longitudinal axis of the wheel, perpendicular to the axis of rotation, and the longitudinal axis of the vehicle. The two wheel angles must satisfy Ackerman geometry for the vehicle in use.
  + Values are in radians, ranging from to .
* **RefSpd** – scalar (m/s)
  + Real scalar number.
  + Reference speed at which vehicle is supposed to go. In case of constant vehicle speed, reference speed equals current speed. Reference speed may be provided by reading of throttle pedal position or by an ADAS feature requesting it, such as Adaptive Cruise Control or Traction Control features.
  + Values are in meters per second, ranging from to . Negative values refer to vehicle moving backward.

The virtual differential model provides 1 output signal:

* **WhlSpd** – 4x1 vector (rad/s)
  + Vector composed of four real scalar elements.
  + Each element reports the wheel angular speed of a wheel of the vehicle. Wheels’ speeds are, in order: front-left, front-right, rear-left, rear-right.
  + Values are in radians per second, ranging to . Negative values refer to wheels rotating in reverse direction, that is when vehicle is backward moving.

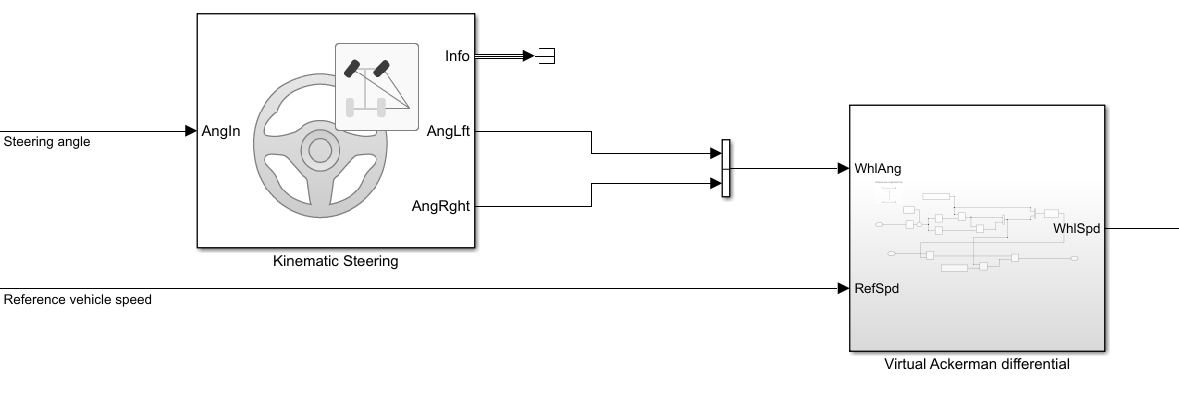
## System parameters

The virtual differential model requires two vehicle parameters:

* **wheel\_base** – scalar (m)
  + Real positive scalar number.
  + It represents the distance between front and rear axles of the vehicle.
  + Values are in meters, ranging from to .
* **wheel\_radius** – scalar (m)
  + Real positive scalar number.
  + It represents the radius of the wheels, assumed to be equal for all the four wheels. It is the distance between wheel’s center of rotation and a planar ground neglecting elastic effects of wheel.
  + Values are in meters, ranging from to .

## How to use

To use the virtual differential model, input signals and parameters must be provided as explained above. Here an example of use is shown.



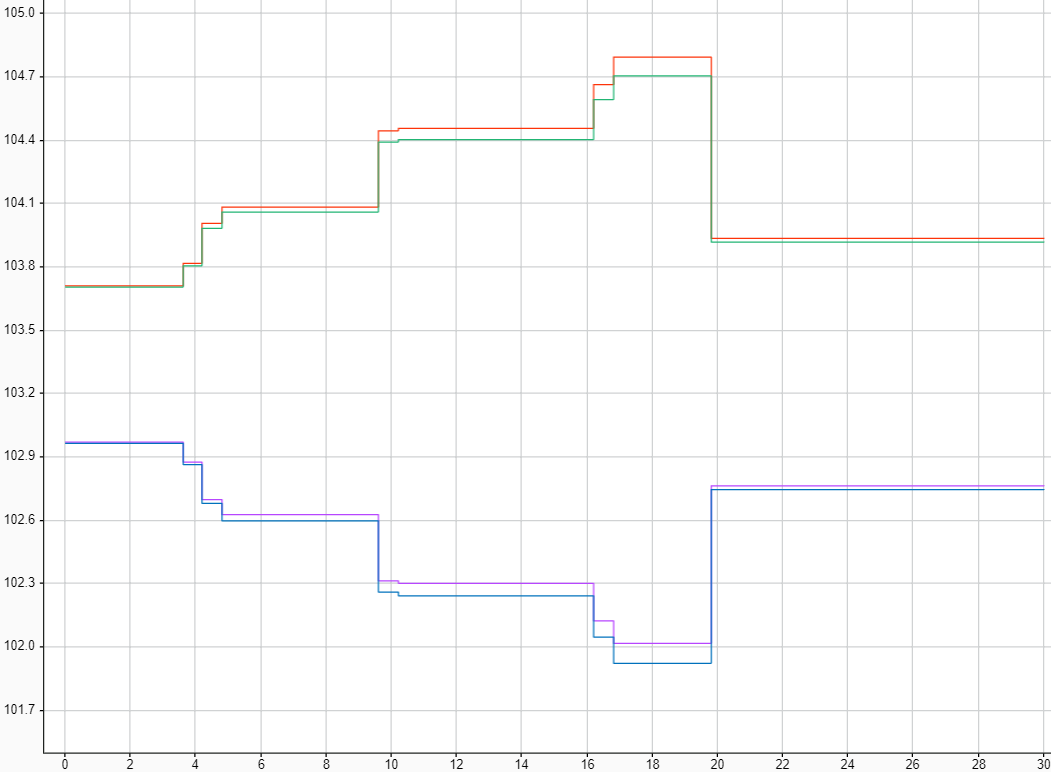
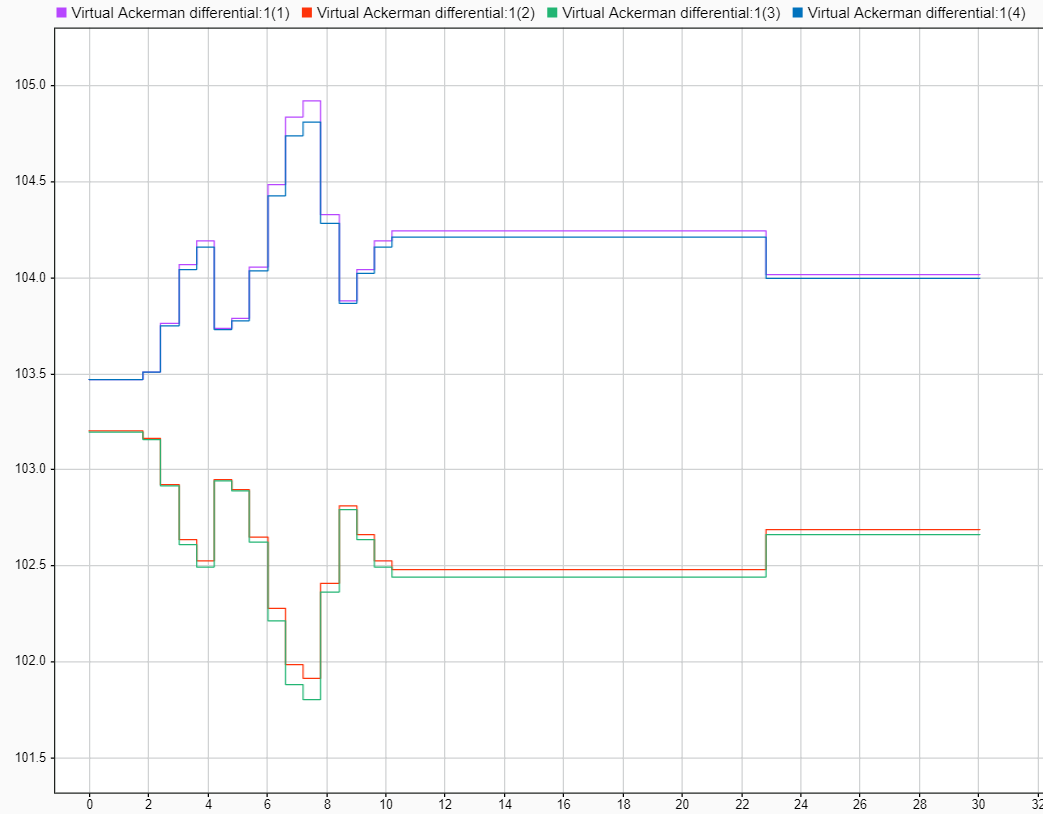
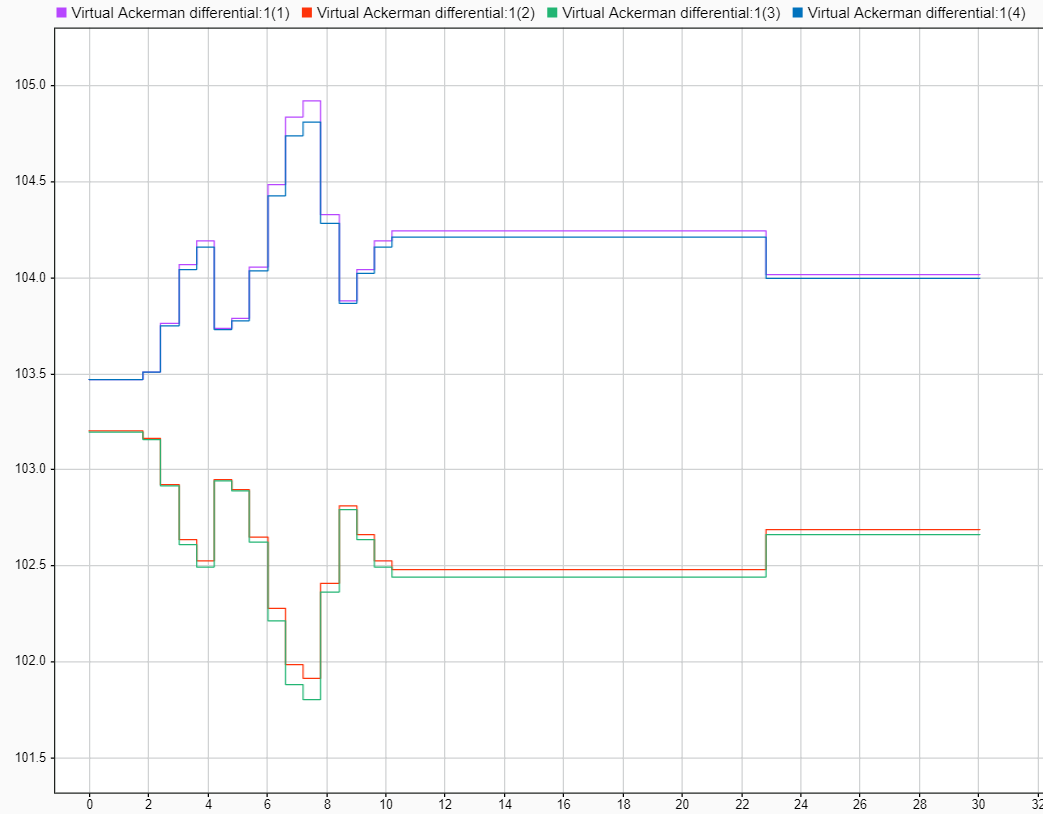
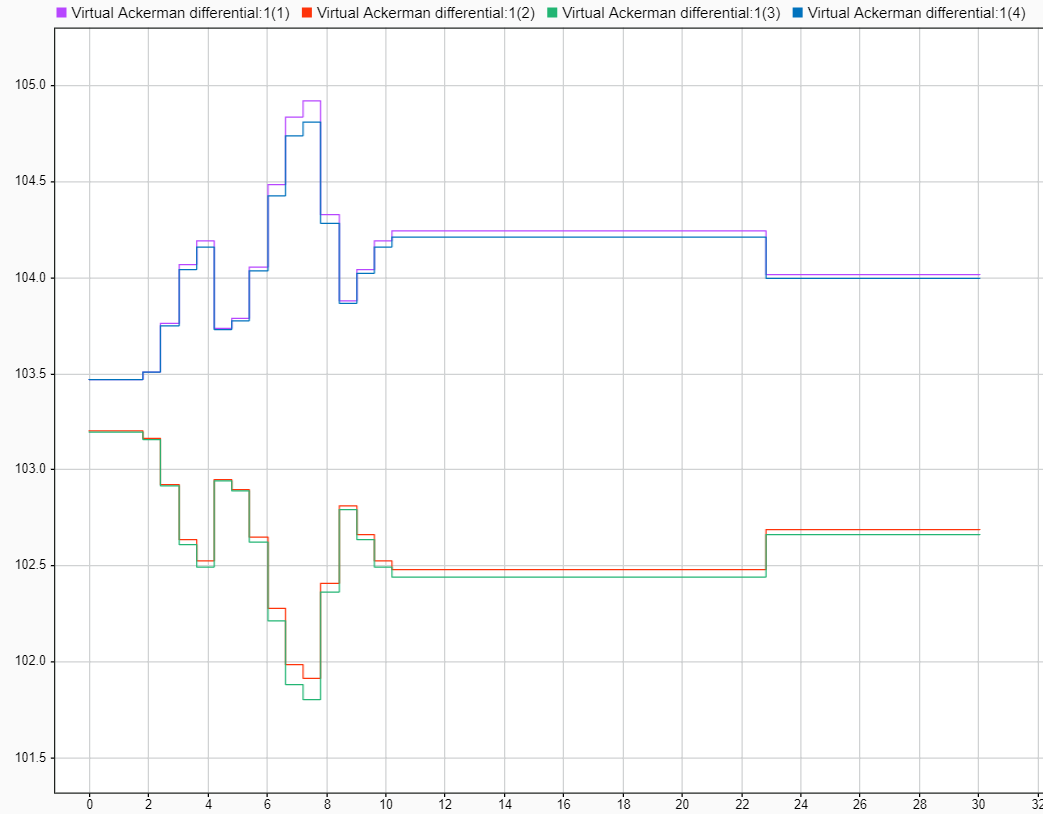
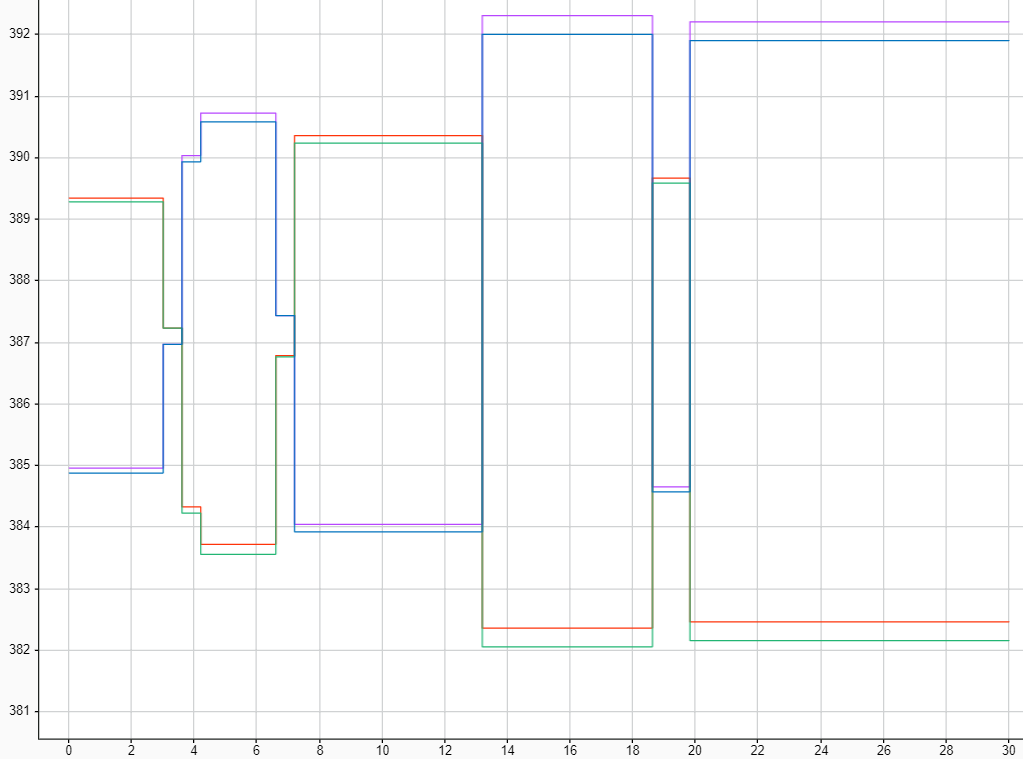
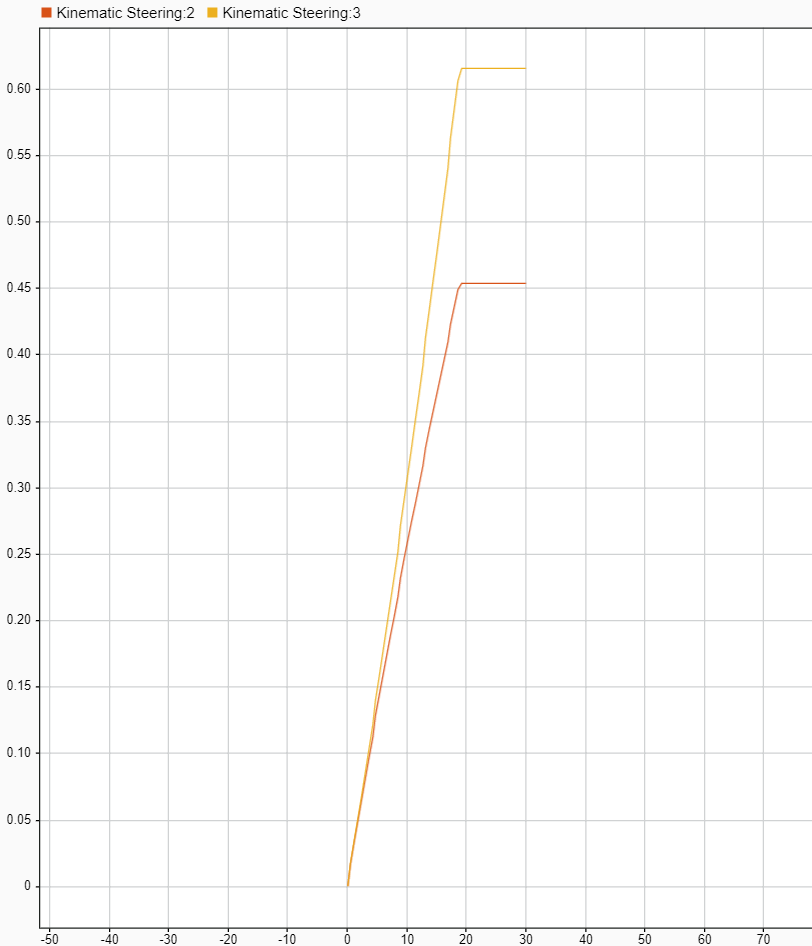
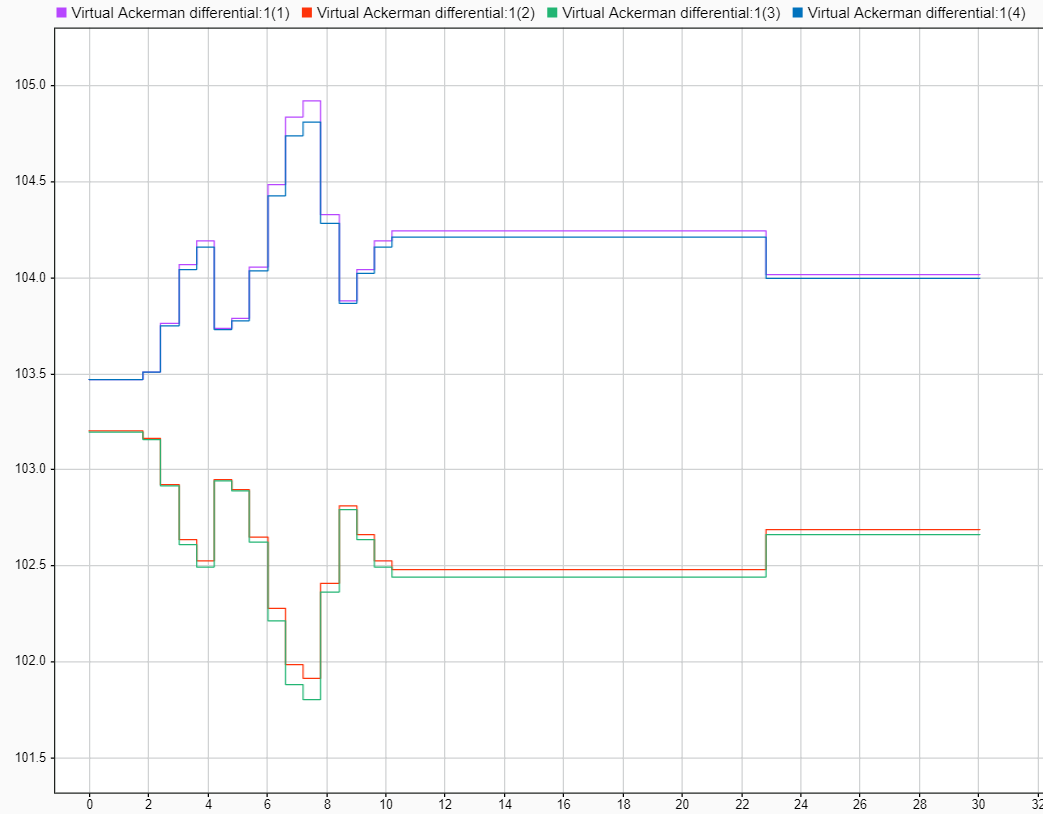
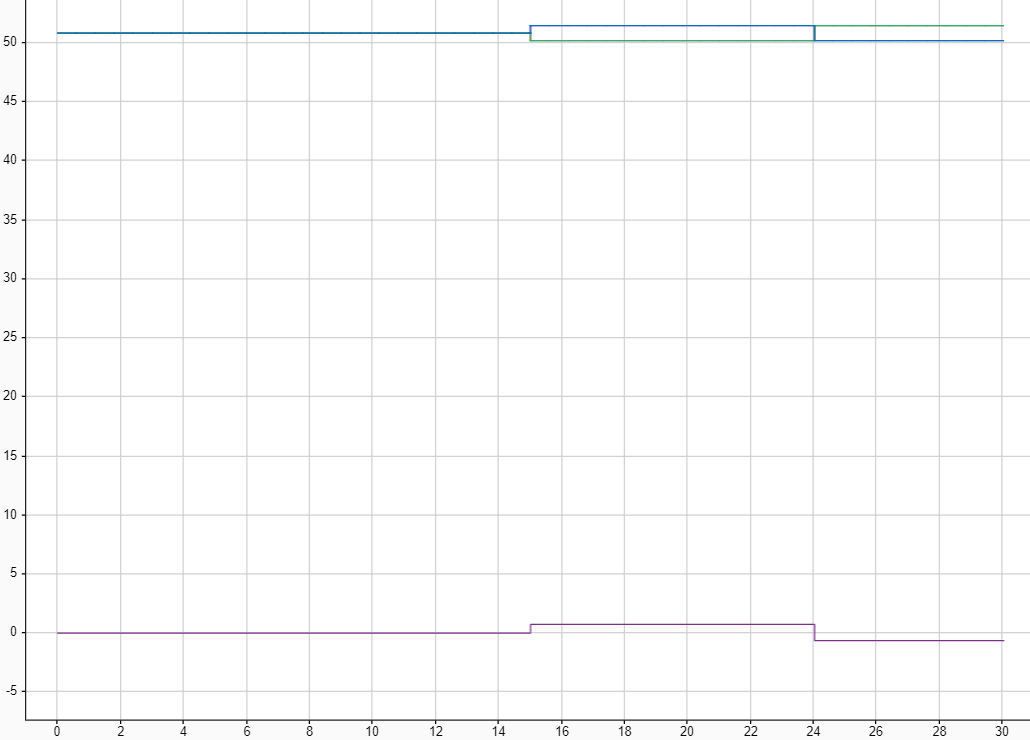
The two wheel angles must be provided as a vector at first input port. They must comply with the Ackerman steering geometry, and one way to ensure it is to use a Kinematic Steering block with steering type set to “Ackerman”. In this way, the steering angle, from steering wheel, is only required to provide the two wheel angles, which must be concatenated in a single vector.

On the other hand, the reference speed may come from reading of the throttle pedal position, and it must represent the speed at which vehicle is required to move. Note that no braking force is considered at this stage, any braking action must be added consequently.

The wheels’ speeds provided by the virtual differential model are provided as a vectorial signal, and they may be used as reference speeds in on-wheel motors, to achieve the requested vehicle speed while maintaining the benefits of a traditional (i.e. mechanical) differential system.

# Validation

In order to have a complete test it has been check the normal behavior, and boundary condition

* In normal condition if the vehicle is steering to right (left image) the left wheel have a speed that is greater than the right, and vice versa (right image). It is possible to notice that the rear wheels goes slowly than the front wheels due to the lowest steering radius  
  
* Ref speed goes + ∞:   
  If the Ref speed goes at +∞ the system limit the max speed at 387 km/h   
    
  
* Steering angle above 75 degree.  
  using a ramp as input it is possible to notice that after 75 degree the system saturate and the output remain constant  
  
* Steering angle change as a step.   
  if the steering angle have a step the control block follow this step, due to the absence of a filter  
  the lowest signal represent the two step that has been applied in order to evaluate this test  
    
  
* Using a steering angle that is not an Ackermann angle the virtual differential keep to elaborate output even if this is a wrong value, without checking.