AEROACOUSTICS

Numerical analysis of the impact of porous media on trailing-edge noise

13 November 2018 | Seong-Ryong Koh | SimLab Fluids & Solids Engineering, JSC



OUTLINE

- Introduction
 - Sound generation in fluid mechanics
 - Computational aeroacoustics (CAA)
- 2 Motivation
- 3 Objectives
- 4 Numerical method
 - Volume-averaging approach: parameters of porous media
 - Flow configuration
 - Porous structure
 - LES/CAA domain
- 5 Results
 - Acoustic fields determined by a variable porosity
 - Acoustic fields at a finite angle-of-attack
- 6 Summary



Aerodynamic sound

- Wave and Vibration in a dynamic system
 - Involve a balance between a restoring force and intertia of a system
 - ► External restoring force, e.g., gravity, surface tension, tube elasticity, magnetic force, Coriolis force



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 - Exception, e.g., water surface waves in 2D horizontal propagation.



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- Wave motions with external restoring force are anisopropic!
 - Dispersion relation, e.g., wave speed as a function of wavenumber
 - Exception, e.g., water surface waves in 2D horizontal propagation.
- Sound generated aerodynamically
 - Restoring force balancing the fluid's intertia provided by its own compressibility (mass conservation)
 - Compressibility properties are same in all directions, i.e., isotropic sound propagation
 - Compressibility implies, the density of a fluid changes!



Classical acoustic theories

have considered

- linear wave propagation,
- single frequency (or harmonics),
- 3 constant flow properties (no spatial gradient),
- 4 and simple geometry (or just in free space).
- Acoustic waves in aerodynamics were understood via linear acoustic theories. So called, 'aeroacoustics' indicates the noise generation by aerodynamic phenomena.
- <u>Linear</u> wave propagation in subsonic flows

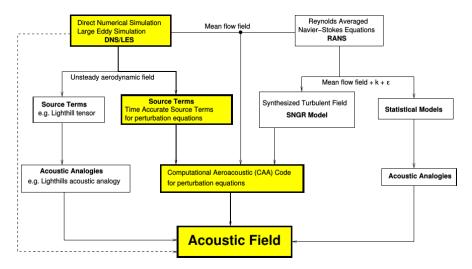


What is CAA?

- Employment of Computational Fluid Dynamics (CFD) techniques in the direct calculation of all physical properties in <u>sound generation</u> and wave propagation
- Numerical solution of fundamental differential or integral equations of the fluid motion
 - Acoustic sources
 - Wave propagation
 - Speed of sound



CAA classification





Challenges to CAA

Small size of quantities of interest:

 $p'_{\text{acoustics}} \ll p'_{\text{hydrodynamic}}$

- High-frequencies of quantities of interest: f_{max} based on the spatial resolution
- Long time solutions necessary: to analyze low-frequency components (large λ_{min})
- Damping and dispersion of oscillations undesirable: low-pass filter to remove unresolved high-frequencies
- Non-reflecting boundary conditions required



Numerical schemes for CAA

CAA requires equal attentions be paid to both *time and space derivatives* which are not usually considered in CFD.

Usually requires accurate statistics in the frequency domain

Extremely large storage required to resolve wave propagation



Numerical schemes for CAA

CAA requires equal attentions be paid to both *time and space derivatives* which are not usually considered in CFD.

- Usually requires accurate statistics in the frequency domain
 - High-order time integration method
 - 2 Numerical dispersion/dissipation
 - 3 Long-time computation for high-resolution analysis
- Extremely large storage required to resolve wave propagation
 - 1 High-order spatial discretization scheme
 - 2 Numerical dispersion/dissipation
 - 3 Huge computational domain for high-resolution analysis



Resource usage for an LES/CAA simulation

jet model jet Mach number	year 2008 isolated jet ${ m Ma}=0.9$	year 2016 real scale nozzle $\mathrm{Ma} = 0.1$
# of cells (LES) # of cells (CAA) resolved freq. range	$\begin{array}{c} 20\times10^6\\ 12\times10^6\\ \mathrm{St}<2 \end{array}$	$\begin{array}{c} 330 \times 10^6 \\ 100 \times 10^6 \\ \mathrm{St} < 10 \end{array}$
MPI processors LES runtime post proc. & CAA runtime disk usage	64 cores 1 week 1-2 weeks $\mathcal{O}(10^2)$ Gb	6000 cores <1 week 1 week $\mathcal{O}(10)$ Tb

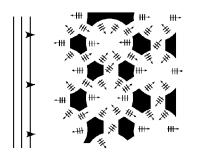
Koh et al., J. Sound Vib., 329, 2010. Cetin et al., Flow Turbul. Combust., 98(2), 2017. Cetin et al., Comput. Rendus Mecanique, 346, 2018

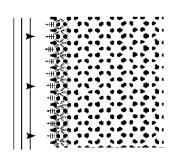


NOISE REDUCTION VIA POROUS MEDIA

Porous medium and noise attenuation

Acoustic wave is assumed isentropic and frictionless in gas. However, in reality the viscous effect changes the acoustic energy being dissipated into heat. e.g., inside porous media.





Schematic of acoustic energy dissipation via porous media as changing the pore-size.

Ref.: Kuczmarski, M., Johnston, J. C., NASA/TM-2011-216995, 2011.



MOTIVATION

- Noise reduction via porous media
 - Acoustic energy dissipation
 - Turbulent flows and the associated acoustic sources modified by porous media
- Parameters influencing acoustic attenuation performance
 - Structure size (d_p)
 - Flow resistance (R_s)
 - Porosity (ψ)
 - Tortuosity (fluid networks), thickness, installation, etc.
- Relationship between the porous parameters and the turbulent flows, e.g., non-dimensional variables in inner scales.
 - Roughness Reynolds number (Red)
 - Permeability Reynolds number (Re_K)



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Acoustic field impacted by

- Porous parameters by Rs and optimization
- Roughness
- Aerodynamic load
- Permeabilit



WS.

OBJECTIVES

- Porous media configuration
 - Find a scaling law between porous parameters and surface impedance
 - Find non-dimensional parameters supporting the scaling law
- Variable porosity
 - Impact of the optimized parameters on the acoustic field
- Sensitivity of noise reduction
 - Impact of porous surfaces on the tone and the broadband noise
 - Impact of aerodynamic load on the noise generation: a zero and a finite angle-of-attack



Flow governing equations: Volume-averaging approach*

Porous parameters

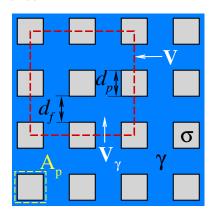
Homogeneous isotropic porous media

• Porosity: $\psi = V_{\gamma}/V$

■ Mean structure size: dp

■ Mean capillary size: d_f

Surface filter area: Ap



^{*} Whitaker, Ind. End. Chem., 61, 1969; Whitaker, Transport Porous Med., 1, 1986; Breugem et al., J. Fluid Mech., 562, 2006; Mößner & Radespiel, Comput. Fluids, 108, 2015



Flow governing equations: Volume-averaging approach

Volume-averaged transport equations

$$\begin{split} \frac{\partial \langle \rho \rangle^{\gamma}}{\partial t} + \nabla \cdot \left(\langle \rho \rangle^{\gamma} \langle \mathbf{u} \rangle \right) &= 0 \\ \frac{\partial}{\partial t} \left(\langle \rho \rangle^{\gamma} \langle \mathbf{u} \rangle \right) + \nabla \cdot \left(\langle \rho \rangle^{\gamma} \langle \mathbf{u} \rangle \langle \mathbf{u} \rangle \right) &= -\nabla \langle \rho \rangle^{\gamma} + \nabla \cdot \tau \\ &+ \underbrace{\frac{1}{V} \int_{A_{\rho}} \left(-\mathbf{I} \rho + \tau \right) \mathbf{n} dA}_{\text{surface filter}} \\ \frac{\partial \langle E \rangle^{\gamma}}{\partial t} + \nabla \cdot \left[\left(\langle E \rangle^{\gamma} + \langle \rho \rangle^{\gamma} \right) \langle \mathbf{u} \rangle \right] &= \nabla \cdot \left(\tau \, \mathbf{u} + k \nabla \langle T \rangle^{\gamma} \right) \\ &+ \underbrace{\nabla \cdot \left[\left(\mathbf{K}_{C} + \mathbf{K}_{D} \right) \cdot \nabla \langle T \rangle^{\gamma} \right]}_{\text{surface filter}} \end{split}$$

Flow governing equations: Volume-averaging approach

Density weighted averaging

$$\left\langle \rho \right\rangle^V = \frac{1}{V} \int_{V_{\gamma}} \rho \, \mathrm{d}V = \psi \langle \rho \rangle^{\gamma} \; , \qquad \left\langle \mathbf{u} \right\rangle = \frac{\left\langle \rho \mathbf{u} \right\rangle^V}{\left\langle \rho \right\rangle^V} = \frac{\left\langle \rho \mathbf{u} \right\rangle^{\gamma}}{\left\langle \rho \right\rangle^{\gamma}}$$

Permeability and Forchheimer tensors

$$\frac{1}{V}\int_{\textit{Ap}}\left(-\textit{Ip}+\tau\right)\textit{ndA} = -\frac{\langle\mu\rangle^{\gamma}}{\textit{K}}\psi\langle\textit{u}\rangle - \frac{\langle\mu\rangle^{\gamma}}{\textit{K}}\psi\textit{F}\langle\textit{u}\rangle$$

$$\mathbf{K} = \frac{\textit{d}_{\textit{p}}^2 \, \psi^3}{\textit{C}_{\textit{K}} (1 - \psi)^2} \, \mathbf{I} \, , \qquad \mathbf{F} = \frac{\textit{d}_{\textit{p}} \, \psi}{\textit{C}_{\textit{F}} (1 - \psi)} \, \frac{\langle \rho \rangle^{\gamma}}{\langle \mu \rangle^{\gamma}} \, |\langle \mathbf{u} \rangle| \, \mathbf{I}$$

Thermal conductivity and thermal dispersion tensor

$$\nabla \cdot \left[(\mathbf{K}_{\mathcal{C}} + \mathbf{K}_{\mathcal{D}}) \cdot \nabla \langle T \rangle^{\gamma} \right] = -\frac{1}{K_{\mathcal{C}}} (\langle T \rangle^{\gamma} - T_{\mathcal{D}})$$



Acoustic governing equations

Acoustic perturbation equations

 \star The complete system of acoustic perturbation equations (APE) for the perturbation variables $(p', \mathbf{u}^a)^T$ reads

$$\begin{split} &\frac{\partial \boldsymbol{p}'}{\partial t} + \overline{\boldsymbol{c}}^2 \nabla \cdot \left(\overline{\boldsymbol{\rho}} \mathbf{u}^a + \overline{\mathbf{u}} \frac{\boldsymbol{p}'}{\overline{\boldsymbol{c}}^2} \right) = \overline{\boldsymbol{c}}^2 q_c \\ &\frac{\partial \mathbf{u}^a}{\partial t} + \nabla (\overline{\mathbf{u}} \cdot \mathbf{u}^a) + \nabla \left(\frac{\boldsymbol{p}'}{\overline{\boldsymbol{\rho}}} \right) = \mathbf{q}_m \;, \end{split}$$

where the acoustic sources in the right-hand side are

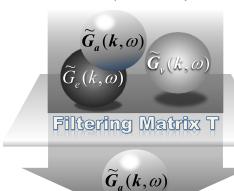
$$egin{aligned} q_c &= -
abla
ho\cdot\mathbf{u}^{m{v}} + rac{\overline{
ho}}{c_p}rac{\overline{D}s'}{Dt} \ \mathbf{q}_m &=
abla \Phi_P +
abla q_{\overline{m}} + T'
abla \overline{s} - s'
abla \overline{T}. \end{aligned}$$

Ewert & Schröder, J. Comput. Phys., 188:365–398, 2003.



Acoustic source filtering

Source vector G represented by the Fourier-Laplace transform



 $\tilde{G}_a(k,\omega)$: acoustic mode

 $\tilde{G}_{\nu}(k,\omega)$: vorticity mode

 $\tilde{G}_e(k,\omega)$: entropy mode

Source excites only acoustic eigenmode $\tilde{G}_a(k,\omega)$.

Ewert & Schröder, *J. Comput. Phys.*, 188:365–398, 2003.



Large-eddy simulation (LES)

- Volume-averaged transport equations for compressible flows
- MILES approach (Fureby and Grinstein, AIAA J., 1999)
- 2nd order accurate modified low-dissipation AUSM scheme (Meinke et al., Comput. Fluids, 2002)
- Explicit 5-stage Runge-Kutta method

Acoustic simulation

- Acoustic perturbation equations (Ewert and Schröder, J. Comput. Phys., 2003)
- 6th order DRP scheme (Tam and Webb, *J. Comput. Phys.*, 1993)
- Alternating 5-6 stage low-dissipation low-dispersion Runge-Kutta method (Hu et al., *J. Comput. Phys.*, 1996)
- High order explicit low-pass filtering (Bogey and Bailly, J. Comput. Phys., 2004)
- Radiation boundary condition (Tam and Webb, J. Comput. Phys., 1993)

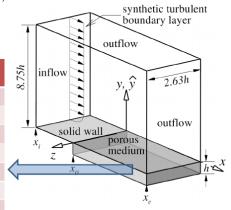


POROUS PARAMETERS

Correlation with flow variables

- $Re_{\delta*} = 1280$ at the inflow boundary
- Streamwise extent: $L_x = 37.5h \ (150\delta^*)$
- $\Delta x^+ = 15$, $\Delta y^+ = 0.8$, $\Delta z^+ = 7.5$
- Porous medium thickness: $h = 4\delta^*$

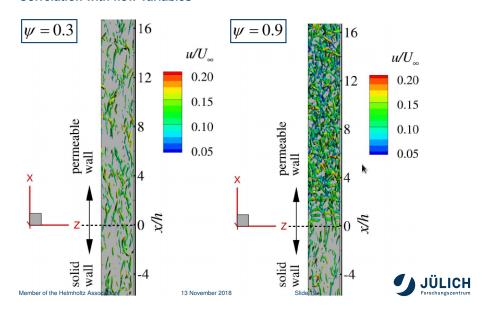
ψ (porosity)	d+ _p (micro- structure size)	
0.3	3.5	
0.5	3.5	
0.5	8.75	
0.5	17.5	
0.5	35	
0.7	3.5	
0.9	3.5	





POROUS PARAMETERS

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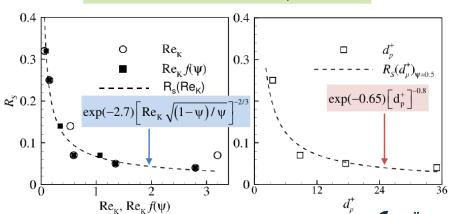


POROUS PARAMETERS

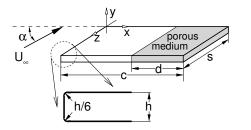
Correlation with flow variables

Koh et al., J. Sound Vib., 421:348-376, 2018.

$${\rm Re}_{\rm K} = \sqrt{{\rm K}} u_{\tau,p} / \nu, \qquad {\rm R}_{\rm s} = \frac{1}{\rho_{\infty} a_{\infty}} \sqrt{\frac{(p_{\rm a}')^2 - (p_{\rm b}')^2}{(v_{\rm a}')^2}}$$

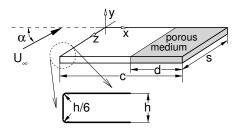


Trailing-edge setup

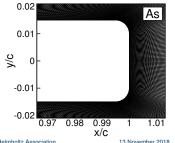


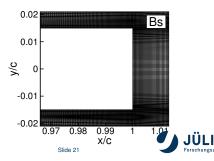
- Flat plate with a finite thickness $Re_c = 135,000, Ma = 0.06$ U_{∞} : freestream velocity (20m/s)
- h: plate thickness (= 0.03c)
 d: porous medium length (= 0.12c)
 s: span size (= 3c)

Trailing-edge setup



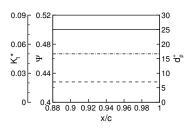
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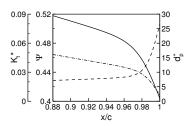
Porous structure setup

constant porosity denoted by 'p'



TBL (Koh et al., J. Sound Vib., 2018)

variable porosity denoted by 'v'

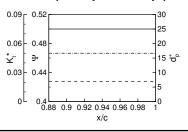


optimization (Koh et al., Comput. Fluids, 2018)

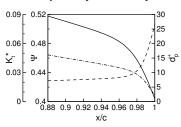


Porous structure setup

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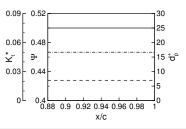


configuration	AoA α (deg)	porosity ψ	mean particle diameter $d_{ ho}^+$	thermal permeability K_t^*
As	0	-	-	-
Ap	0	0.5	7	0.05
Bs	0	-	-	-
Вр	0	0.5	7	0.05
Β̈́ν	0	0.4-0.52	7-26.5	0.006-0.048

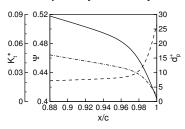


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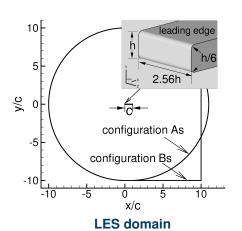


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Bv	0	0.4-0.52	7-26.5	0.006-0.048
As2	2	-	-	-
Ap2	2	0.5	7	0.05
Av2	2	0.4-0.52	7-26.5	0.006-0.04 <u>8</u>

Computational domain

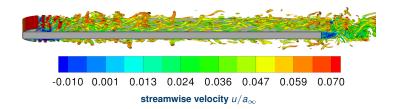


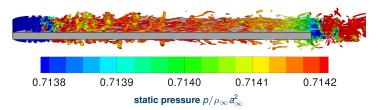
LES 15c domain source 10c domain 110c 0.48c17c 1.56c acoustic domain

CAA domain



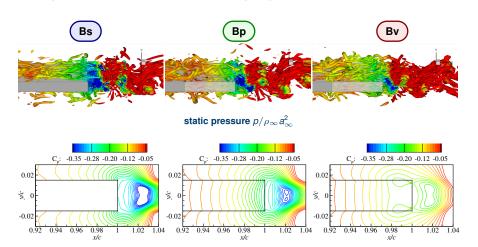
Second invariant of velocity gradient tensor at $Q = 1.0a_{\infty}^2/c^2$







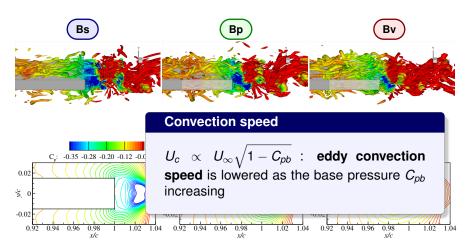
Static pressure contours near the trailing edge



mean static pressure coefficient (C_p)



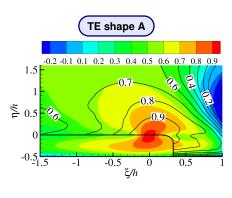
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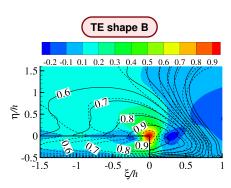


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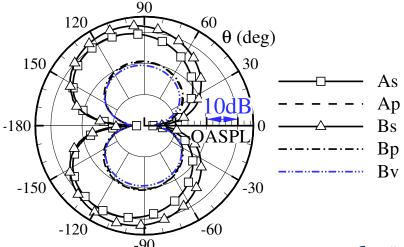
Two-point correlation of pressure fluctuations near the trailing edge $(\xi\eta$ -plane, streamwise/wall normal distance)



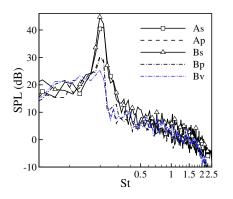


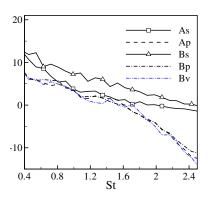


Overall sound pressure level



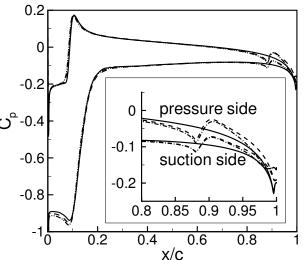
Sound spectra at the coordinates x/c = 1 and y/c = ± 8 ($\theta = \pm 90^{\circ}$)







Surface pressure coefficient (C_p)

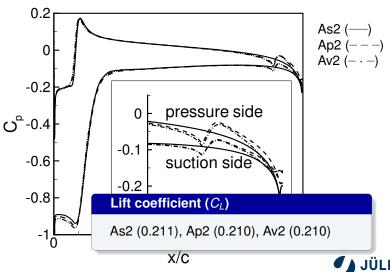




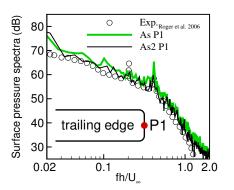


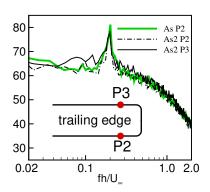
Slide 29

Surface pressure coefficient (C_p)



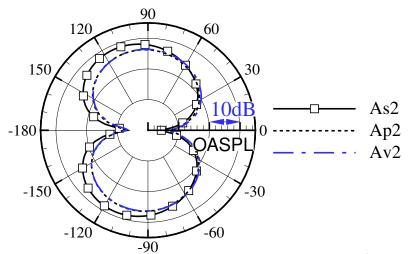
Power spectral density of surface pressure fluctuations





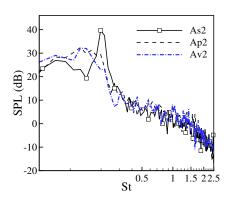


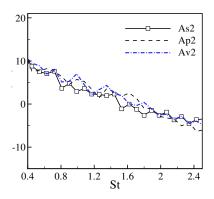
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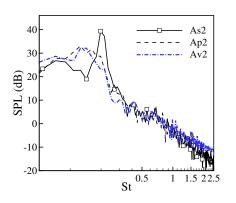
Sound spectra at the coordinates x/c=1 and y/c=8 ($\theta = +90^{\circ}$)

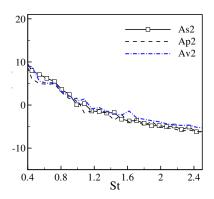






Sound spectra at the coordinates x/c=1 and y/c=-8 ($\theta=-90^{\circ}$)

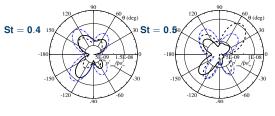


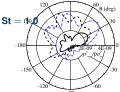




Directivity









SUMMARY

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 - Baseline porous parameters (ψ , Re_d , Re_K) minimizing the flow resistance (R_s).



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- Impact of the variable porosity on noise generation at zero AOA
 - Small correlation length and high base pressure
 - Reduction of the tonal and the broadband noise: max. 11dB OASPL
 - Optimized distribution obtain an additional tonal noise reduction.



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 - Small correlation length and high base pressure
 - Reduction of the tonal and the broadband noise: max. 11dB OASPL
 - Optimized distribution obtain an additional tonal noise reduction.
- Variable porosity at a finite angle-of-attack
 - No additional acoustic reduction via the variable porosity distribution.
 - In future optimization approaches the angle-of-attack should be considered an optimization parameter.

