

## ① Finding the flow properties :

•  $\mu$ ,  $K$ ,  $C_p$  were obtained from Perry's handbook. They were obtained at the "average flow Temperature" =  $\frac{T_{in} + T_{out}}{2}$

•  $\rho$  was obtained at the same Temperature from Engineering toolbox.com, a site suggested in CH2014 (Heat & Mass Transfer)

Thermophysical Properties

Benzene :  $MW = 78 \text{ g mol}^{-1}$  ;  $T_{avg} = 31.3 \text{ K}$

i)  $C_p$  :  $C_1 = 129440$   $C_2 = -169.5$   $C_3 = 0.6471$

$$C_p = C_1 + C_2 T + C_3 T^2$$

$$= 139872 \cdot 41 \text{ J/mol K}$$

$$= 1.792 \text{ kJ/kg K (MW = 92)}$$

ii)  $\mu$  :  $C_1 = 7.5117$   $C_2 = 294.68$   $C_3 = -2.294$

$$\mu = \exp \left( C_1 + \frac{C_2}{T} + C_3 \ln T + C_4 T^5 \right)$$

$$= 4.996 \times 10^{-4} \text{ Pa.s.}$$

iii)  $k$  :  $C_1 = 0.2344$   $C_2 = -0.0003057$

$$k = C_1 + C_2 T = 0.139 \text{ W/m K}$$

iv)  $\rho$  :  $882.2 \text{ kg/m}^3$  (Engineering toolbox.)

References: table 2-138, table 2-147,

Perry's handbook

Toluene :

$$MW = 92 \text{ g/mol}$$

$$T_{\text{avg}} = 333 \text{ K}$$

Using the same references & engine we get.

$$i) C_p = 1.8096 \text{ kJ/kg K}$$

$$ii) \mu = 3.869 \times 10^{-4} \text{ Pa-s}$$

$$iii) k = 0.124 \text{ W/m K}$$

$$iv) \rho = 824.7 \text{ kg/m}^3$$

Flow Properties

Problem i) Pipes :

$$\text{Inner pipe COND 1-1/4" : } \begin{cases} ID = 1.38 \text{ in.} = 0.035 \text{ m} \\ OD = 1.66 \text{ in.} = 0.0422 \text{ m} \end{cases}$$

$$\text{Outer pipe COND 2" : } \begin{cases} ID = 2.067 \text{ in.} = 0.0525 \text{ m} \\ OD = 2.375 \text{ in.} = 0.0603 \text{ m} \end{cases}$$

ii) Reynolds : (flows in inner pipe)

$$Re = \frac{\rho v d}{\mu} = \frac{\dot{m}}{A} \times \frac{4A}{P} \times \frac{1}{\mu}$$

$$= \frac{1.3 \times 4}{\pi (d_{\text{inner}}) \times \mu} = 94518.8048$$

$$Pr = \frac{\mu C_p}{k} = 6.4506$$

From JP Holman table-6.8

$0.6 < Pr < 100$   $Re$  is turbulent regime  
 $\Rightarrow$  Dittus Boelter correlation (with no wall effects)

$$Nu = 0.023 Re^{0.8} Pr^{0.4} \quad \text{0.4 for heating}$$

$$= 116.211 \times 463.43$$

$$h = \frac{Nu k}{d} \quad \text{0.0025 m}$$

$$\Rightarrow h_{\text{benzene}} = 1833.725 \text{ W/m}^2\text{K}$$

Energy Balance:

$$\dot{m}_{\text{benzene}} C_{p,B} \Delta T_{1,B} = \dot{m}_T C_{p,T} \Delta T_{\text{solution}}$$

$$\Rightarrow \dot{m}_{\text{solution}} = 0.044 \text{ kg/s}$$

$$Re_{\text{solution}} = \frac{0.044 \times 4}{\pi (0.0422 - 0.0525) \times 3.89 \times 10^{-4}}$$

$$= 22258.526$$

$$Pr = \frac{3.889 \times 10^{-4} \times 1.8096 \times 10^3}{0.124}$$

$$= 5.6462$$

Similar conditions

$$\therefore Nu = 0.023 Re^{0.8} Pr^{0.3} \quad \text{0.3 since cooling}$$

$$= 116.211$$

$$d = \frac{(0.0525)^2 - (0.0422)^2}{0.0422}$$

for heat transfer  
 $\approx 0.028 \text{ m}$   
 $\rightarrow$  inner perimeter of inner tube only

$$\Rightarrow h = 626.528 \text{ W/m}^2\text{K}$$

## Heat Transfer Calculations

Useful  $Q_{\text{real}} \text{ LMTD} = Q$

$$\text{LMTD} = \frac{\Delta T_1 - \Delta T_2}{\ln\left(\frac{\Delta T_1}{\Delta T_2}\right)}$$

Benzene :  $30 \rightarrow 50^\circ\text{C}$  (counter and  
 $\downarrow \Delta T_2$   $\uparrow \Delta T_1$   
 Toluene :  $40^\circ\text{C} \leftarrow 80^\circ\text{C}$

$$\Rightarrow \text{LMTD} = \frac{30 - 10}{\ln 3} = 18.21^\circ\text{C}$$

$$Q = \dot{m}_{\text{Benzene}} C_{p, \text{Benzene}} \Delta T_{\text{Benzene}} = 46.605 \text{ kJ/s}$$

$$\therefore \frac{1}{\text{Useful } \dot{Q}_{\text{real}}} = 3.9062 \times 10^{-4} \text{ K W/K W/K}$$

~~But  $U_{\text{real}} =$~~

$$\Rightarrow R_{\text{real}} = 3.9062 \times 10^{-4} \text{ K W/K W/K}$$

But  $R_{\text{real}} = \frac{1}{h_i \pi d_i l} + \frac{1}{h_o \pi d_o l} + \frac{U_{\text{foul}}}{l}$   
 $\downarrow$   $\downarrow$   $\downarrow$   
 ID of inner pipe Outer dia of inner pipe fouling factor

$$= \frac{6.9595 \times 10^{-3} + 0.012 + 2 \times 10^{-3}}{l} \quad \text{--- ②}$$

Equating ① & ② we get,

$$d = 47.28 \text{ m.}$$

Assumptions:

- ① ~~Wall~~ <sup>of pipe</sup> resistance has been neglected  
(material of wall not given & wall is thin pipe is thin)
- ② Properties ~~as~~ (thermophysical) calculated at  $T = \frac{T_{\text{inlet}} + T_{\text{outlet}}}{2}$  are assumed to be representative of the whole length of fluid
- ③ Wall effects ignored. ( $T_{\text{fluid}}$  is not uniform radially ~~too~~, but this is ignored)
- ④ Heat losses are assumed to be negligible