

Use a weather forecast website, and utilize the psychrometric chart and the formula we went through in the class to determine the absolute humidity, the wet-bulb temperature and the mass of water vapor in the air in Classroom A of Piacenza campus in the moment that you are solving this exercise Humidity: Relative humidity, Press one atmospheric: Air total pressure (1 hPa: 0.1 kPa), Temperature effective: temperature to be utilized. 16 December.

Relative humidity: 95 % Total air pressure: 102.1 kpa Effective Temperature: 6 C Absolute humidity: 0.005 Kg vapor / Kg dry air Wet bulb temperature: 5

$M_v = \frac{P_v V}{R_v T}$   $P_{\text{Total}} = P_a + P_v$   $\phi = \frac{0.622 P_v}{P - P_v}$   $0.005 = \frac{0.622 P_v}{102.1 - P_v}$   $P_v = 0.814 \text{ Kpa}$   
 Classroom A =  $6 \times 10 \times 6 = 360 \text{ m}^2$   $R_v = 0.4615$   $T = 279 \text{ K}$   $M_v = \frac{(0.814 \times 360)}{(0.4615 \times 279)}$

Utilize the same methodology we went through in the class and determine the sensible and latent load corresponding to internal gains, the ventilation, and the infiltration in a house with a good construction quality and with the same geometry as that of the example which is located in Brandis, Italy

# BRINDISI, Italy

WMO#: 163200

Lat: 40.65N Long: 17.95E Elev: 10 StdP: 101.2 Time Zone: 1.00 (EUW) Period: 86-10 WBAN: 99999

## Annual Heating and Humidification Design Conditions

Coldest Month	Heating DB		Humidification DP/MCDB and HR						Coldest month WS/MCDB				MCWS/PCWD to 99.6% DB	
	99.6%		99.6%			99%			0.4%		1%			
	DP	HR	MCDB	DP	HR	MCDB	WS	MCDB	WS	MCDB	MCWS	PCWD		
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)
(1) 2	2.9	4.1	-5.1	2.5	7.2	-3.0	3.0	7.4	13.4	10.2	12.4	10.6	3.4	250

## Annual Cooling, Dehumidification, and Enthalpy Design Conditions

Hottest Month	Hottest Month DB Range	Cooling DB/MCWB						Evaporation WB/MCDB						MCWS/PCWD to 0.4% DB	
		0.4%		1%		2%		0.4%		1%		2%			
		DB	MCWB	DB	MCWB	DB	MCWB	WB	MCDB	WB	MCDB	WB	MCDB	MCWS	PCWD
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
(2) 8	7.1	32.8	23.6	31.1	24.3	29.9	24.3	27.2	29.7	26.3	29.0	25.6	28.3	4.2	180

Dehumidification DP/MCDB and HR									Enthalpy/MCDB						Hours
0.4%			1%			2%			0.4%		1%		2%		8 to 4 &
DP	HR	MCDB	DP	HR	MCDB	DP	HR	MCDB	Enth	MCDB	Enth	MCDB	Enth	MCDB	12.8/20.6
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
26.3	21.8	29.2	25.4	20.7	28.5	24.7	19.7	27.9	86.0	30.1	82.2	29.1	78.5	28.3	1236

## Extreme Annual Design Conditions

Extreme Annual WS			Extreme Max WB	Extreme Annual DB				n-Year Return Period Values of Extreme DB							
1%	2.5%	5%		Mean		Standard deviation		n=5 years		n=10 years		n=20 years		n=50 years	
				Min	Max	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
11.3	9.9	8.7	31.4	0.4	37.3	1.4	3.0	-0.6	39.4	-1.4	41.1	-2.2	42.8	-3.2	44.9

$Q_{ig}, s = 136 + 2.2A_{cf} + 22N_{oc} = 136 + 2.2 \cdot 200 + 22 \cdot 2 = 620 \text{ W}$   
 $Q_{ig}, l = 20 + 0.22A_{cf} + 12N_{oc} = 20 + 0.22 \cdot 200 + 12 \cdot 2 = 88 \text{ W}$

$AL = A_{es} \cdot A_{ul} = 344 \cdot 1.4 = 481.6 \text{ cm}^2$

$T_{cooling} = 24 \text{ }^\circ\text{C}$ , and heating temperature  $T_{heating} = 20 \text{ }^\circ\text{C}$

$\Delta T_{cooling} = 31.1 \text{ }^\circ\text{C} - 24 \text{ }^\circ\text{C} = 7.1 \text{ }^\circ\text{C} = 7.1 \text{ K}$

$\Delta T_{heating} = 20 \text{ }^\circ\text{C} - (-4.1 \text{ }^\circ\text{C}) = 24.1 \text{ }^\circ\text{C} = 24.1 \text{ K}$

$DR = 7.1 \text{ }^\circ\text{C} = 7.1 \text{ K}$

$IDF_{heating} = 0.073 \text{ L/s} \cdot \text{cm}^2$ ,

$IDF_{cooling} = 0.033 \text{ L/s} \cdot \text{cm}^2$ ,

$Q_{i, heating} = AL \cdot IDF_{heating} = 481.6 \cdot 0.073 \approx 35.157 \text{ L/s}$

$Q_{i, cooling} = AL \cdot IDF_{cooling} = 481.6 \cdot 0.033 \approx 15.893 \text{ L/s}$

$Q_v = 0.05A_{cf} + 3.5(N_{br} + 1) = 0.05 \cdot 200 + 3.5 \cdot (1 + 1) = 17 \text{ L/s}$

$Q_{i-v, heating} = Q_{i, heating} + Q_v \approx 35.157 + 17 = 52.157 \text{ L/s}$

$Q_{i-v, cooling} = Q_{i, cooling} + Q_v \approx 15.893 + 17 = 32.893 \text{ L/s}$

$C_{sensible} = 1.23$ ,  $C_{latent} = 3010$ ,  $\Delta\omega_{cooling} = 0.0039$

$Q_{inf-ventilation\ cooling\ sensible} = C_{sensible} Q_{i-v, cooling} \Delta T_{cooling} \approx 1.23 * 32.893 * 7.1 \approx 287.25\ W$

$Q_{inf-ventilation\ cooling\ latent} = C_{latent} Q_{i-v, cooling} \Delta \omega_{cooling} \approx 3010 * 32.893 * 0.0039 \approx 386.13\ W$

$Q_{inf-ventilation\ heating\ sensible} = C_{sensible} Q_{i-v, heating} \Delta T_{heating} \approx 1.23 * 52.157 * 24.1 \approx 1546.09\ W$