### Lecture 7: Maximum Normal & Shear stresses

Logistics: - HW3 du Thusday

Last time: - Orthogonal lensors @ @ = I det (@) = ± 1
rotation or reflection

- Change in basis: {e;} {e;}

 $\triangle + A_{ij} = i =$ 

→ A rotation

[x] = [4] [x] [3] [4]

- Invariance of tr[3] & det[3]

- Eigen problem: | ≤ x = λ y

≦ is gyn. pas, def.

=> v's ar orthogonal

- Spectral decomposition

- Principle in variants

Toolay: Max Normal & Shear stresses

### Normal and Shear Stress

the second shows = 
$$\frac{1}{2}$$
 =  $\frac{1}{2}$  =  $\frac{1}{2}$ 

# Extremed stress values > important for failure We all know this is related to eigenvalues But Why?

## I, Extremal normal stresses Given any & at x what are the nuit normal n corresponding to extrema in on?

Constrained optimization problem

find extreme of  $\varepsilon_n = \varepsilon_n(\underline{n}) = \underline{n} \cdot \underline{s}\underline{n}$ with constraint that  $|\underline{n}| = 1$   $\underline{n} \cdot \underline{n} \cdot 1 = g(\underline{u})$ 

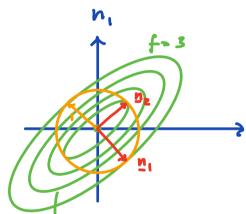
$$\mathcal{L}(n; y) = \overline{n} \cdot \overline{s} \cdot \overline{n} - y (n; n; -1)$$

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function | constraint toophinise Lagrange multiplier

Funchion f(4) = 4. =4 isquadrahic

if >;>0 -> level sets are ellipses



$$\overline{n} = \binom{N^s}{n!}$$

$$f(u) = \underline{G} \cdot \underline{\underline{G}} = 0$$

-> NS & (N) = EN(N) = N . = N .

un constrained minimum

$$\underline{n} = \underline{0} \qquad f(\underline{0}) = 0$$

Hethool of Lagrange multipliers

>extremal values are the stationary points L

\(\lambda(n;\lambda) = n\tilde{c};\lambda;\lambda;\lambda \rangle(n;n;-1)

$$\frac{\partial \mathcal{L}}{\partial \lambda} = n_i n_i - 1 = 0$$

$$\frac{\partial \mathcal{L}}{\partial x} = \sigma_{ij} \left( \frac{\partial n_i}{\partial n_k} n_j + n_i \frac{\partial n_j}{\partial n_k} \right) - \lambda \left( 2n_i \frac{\partial n_i}{\partial n_k} \right) = 0$$

$$common notation  $\frac{\partial n_i}{\partial n_k} = n_{i,k}$ 

$$n = \binom{n_i}{n_s} \frac{\partial n_j}{\partial n_j} = 1 \quad \frac{\partial n_j}{\partial n_s} = 0$$

$$\Rightarrow n_{i,k} = \delta_{i,k} \quad n_{j,k} = \delta_{j,k}$$

$$n_{j,k} = \delta_{j,k}$$$$

$$\frac{\partial \mathcal{Z}}{\partial n_{k}} = \frac{\partial \mathcal{Z}}{\partial n_{k}} \left( \frac{\partial \mathcal{Z}}{\partial n_{k}} \right) - \frac{\partial \mathcal{Z}}{\partial n_{k}} \right) - \frac{\partial \mathcal{Z}}{\partial n_{k}} = 0$$

$$= \frac{\partial \mathcal{Z}}{\partial n_{k}} - \frac{\partial \mathcal{Z}}{\partial n_{k}} - \frac{\partial \mathcal{Z}}{\partial n_{k}} - \frac{\partial \mathcal{Z}}{\partial n_{k}} = 0$$

$$= \frac{\partial \mathcal{Z}}{\partial n_{k}} - \frac{\partial \mathcal{Z}}{\partial n_{k}} = 0$$

$$= \frac{\partial \mathcal{Z}}{\partial n_{k}} - \frac{\partial \mathcal{Z}}{\partial n_{k}} - \frac{\partial \mathcal{Z}}{\partial n_{k}} - \frac{\partial \mathcal{Z}}{\partial n_{k}} = 0$$
switch to equipolic wotahin
$$= \frac{\partial \mathcal{Z}}{\partial n_{k}} - \frac{\partial \mathcal{Z}}{\partial n_{k}} = 0$$

Constrained optinization -> eigenvalur probler.

⇒ eigen vectors of \ are the directions of max & un'u normal stress.

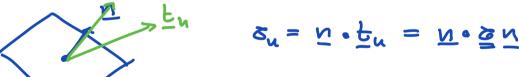
To see that is are associated normal stænes you dot eigen problem with n.

$$\overline{N} \cdot \left(\overline{\mathbb{Q}} - y \,\overline{\mathbb{I}}\right) \,\overline{N} = 0$$

$$\vec{\lambda} \cdot \vec{Q} \vec{\mu} - \gamma \vec{\lambda} \cdot \vec{\lambda} =$$

$$\delta_{N} = \lambda$$

tu = on



λ; s are principal strenes => λ; = =;

& = wax nosmal stery ٥ > ٥ > ٥ = un un un une stren

his am the pricipal directions

What are trachious ou prinipal planes (planes associaded ni's)

$$\bar{\Gamma}^{\bar{n}!} = \bar{\bar{\alpha}} \bar{n}! = (\bar{\sum}_{\bar{3}} s^{2} \bar{n}^{2} \bar{n}^{2}) \bar{n}! = \bar{\sum}_{\bar{3}} s^{2} (\bar{n}^{2} \cdot \bar{n}^{2}) \bar{n}!$$

Fn: = & M:

traction is equal to normal stress & II n; => no shear strerses ou principal planes. tn: = 0

## II) Extremal shear strong

Given the principal dir. [mi] at x what is the direction/unit vector s=[s, se s,] that gives the wax. & viv. values of T?

trachiou in der 
$$\underline{s}$$
 $\underline{t}_s = \underline{\delta} \underline{s} = (\underline{2}\underline{\delta}, \underline{n}; \underline{\delta}, \underline{n};)$ 
 $\underline{t}_s = \underline{\delta} \underline{s} = (\underline{2}\underline{\delta}, \underline{n}; \underline{\delta}, \underline{n};)$ 
 $\underline{s}$ 
 $\underline{s}$ 

$$= \sum_{j=1}^{n} s_{j} s_$$

traction on s

ts = 6, 5, 4, + 6, 5, 4, + 6, 5, 4, + 6, 5, 1,3

need expression for shear shess T

$$|\underline{t}_{s}|^{2} = \varepsilon_{u}^{2} + \tau^{2} \Rightarrow \tau^{2} = |\underline{t}_{s}|^{2} - \varepsilon_{n}^{2}$$

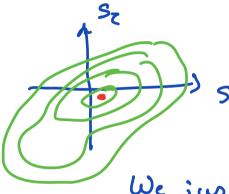
normal stres: & = s.t. = 6, 5, + 2, 5, + 5, 5,

shew shers in principal france:

Looking for extreme of  $t^2(t)$  under the combraint that |s|=1  $(s_1^2+s_2^2+s_3^2=1)$   $s_3^2=1-s_1^2-s_2^2$ 

Solu constrained optimization problem by direct elimination:

eliminale  $s_3^2$  by substituting  $s_3^2 = 1 - s_1^2 - s_2^2$  $\Rightarrow T^2 = T^2(s_1, s_2)$ 



because we are substated ), complicating

=> solut for max min directly

De just weed to find

3t2 = 3t2 = 0

 $\frac{\partial \overline{U}^{2}}{\partial S_{1}^{2}} = 2 \frac{S_{1}}{S_{1}} \left( \frac{S_{1}}{S_{1}} - \frac{S_{3}}{S_{3}} \right) \left\{ \frac{S_{1}}{S_{1}} - \frac{S_{3}}{S_{3}} - 2 \left[ \left( \frac{S_{1}}{S_{1}} - \frac{S_{3}}{S_{3}} \right) \frac{S_{1}^{2}}{S_{1}^{2}} + \left( \frac{S_{2}}{S_{2}} - \frac{S_{3}}{S_{3}} \right) \frac{S_{2}^{2}}{S_{2}^{2}} \right] = 0$ 

First solution (trivial):  $s_1 = s_2 = 0 \implies s_3 = 1$   $\Rightarrow \underline{s} = \pm \underline{u}_3$   $\overline{t} = 0$ 

=> minimu in shed stress

## ou principal places

Second solution: 5, =0

$$\underline{s} = \pm \frac{1}{\sqrt{2}} \left( \underline{n}_2 + \underline{n}_3 \right)$$

We have two solutions:

min: 
$$C = 0$$
 for  $S = \pm N^2$   
max:  $C = 0$  for  $S = \frac{1}{5}(\pm N^2 \pm N^2)$ 

two additional parts of solus by eliminating s, or so in total we have:

Min. shear stresses:

#### Max. shear sherres

$$T_{23} = \frac{1}{2} (\delta_2 - \delta_3) \quad \text{ou} \quad S_{23} = \frac{1}{12} (\pm N_2 \pm N_3)$$

$$T_{13} = \frac{1}{2} (\delta_1 - \delta_3) \quad \text{ou} \quad S_{13} = \frac{1}{\sqrt{2}} (\pm N_1 \pm N_2)$$

$$T_{12} = \frac{1}{2} (\delta_1 - \delta_2) \quad \text{ou} \quad S_{12} = \frac{1}{\sqrt{2}} (\pm N_1 \pm N_2)$$

