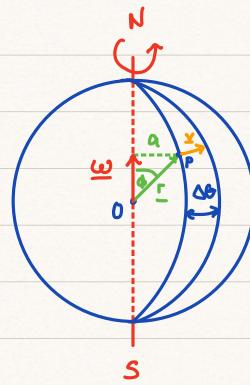
Angular Homentum and Torque

Rotational motion:



Angular velocticy: w=1w1 êw

 $|w| = \frac{d\theta}{dt}$

Position vector: r= Irlêr

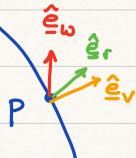
Irl = OP

Velocity: V = IVIêr

| | | = | w | a = | w | | [| sin ()

=> v = IWIIIsin & ev

local coord



ê, I plane of êw and êr

êw xêr = sinbêv

substitute

٧ = الاا الا في × قر

٧ = (ايا في) × (اياف،)

 $\bar{\Lambda} = \overline{\infty} \times \bar{L}$

Example: Your current velocity in ATX (lat. ~30°N)

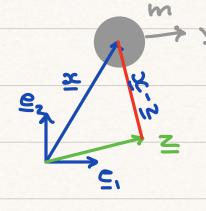
|T1 = 6.37 · 10 m φ ≈ 60° = \frac{\pi}{3} ≈ 1.05

 $|\omega| = 7.3 \cdot 10^{-6} \frac{\text{rad}}{\text{sac}}$ |V| = | w| | r | sin 0 & 403 W

Angular moment um & torque

Linear momentum: <u>L=my</u>

angular momentum:
$$j = (x - Z) \times L$$



ez relative to Ξ force \Rightarrow linear momentum

torque \Rightarrow augular momentum

torque:
$$\underline{T} = \frac{d\dot{f}}{dt} = \frac{d}{dt} \left[(\underline{x} - \underline{z}) \times m\underline{v} \right]$$

$$= m \frac{d}{dt} \left(\underline{x} \times \underline{v} - \underline{z} \times \underline{v} \right)$$

$$= m \left(\underline{\dot{x}} \times \underline{v} + \underline{x} \times \underline{\dot{v}} - \underline{\dot{z}} \times \underline{v} - \underline{z} \times \underline{\dot{v}} \right)$$

$$\underline{\dot{x}} = \underline{v} \qquad = m \left(\underline{v} \times \underline{v} + \underline{x} \times \underline{v} - \underline{z} \times \underline{v} \right)$$

$$\underline{T} = (\underline{x} - \underline{z}) \times \underline{f}$$

$$\underline{T} = (\underline{x} - \underline{z}) \times \underline{f}$$

torque = moment of force or moment

Two basic relations:

Resultant torque due to
1) body force:
$$\underline{\underline{\Gamma}}_b = \int_{\mathcal{Q}} (\underline{x} - \underline{z}) \times \underline{b} \, dV$$

2) surface force:
$$\underline{T}_s = \int_{\Gamma} (\underline{x} - \underline{z}) \times \underline{t}_n dA$$

In rotation we have two important geometric locations in a body:

$$\underline{x}_{m} = \frac{1}{V_{B}} \int_{B} \underline{x} \, dV$$

$$\underline{x}_{m} = \frac{1}{M_{B}} \int_{B} \rho \underline{x} \, dV$$

Note: p = const

$$\underline{x}_{m} = \frac{1}{m^{D}} \int_{D} \underline{x} dV = \frac{\rho}{\rho} V_{D} \int_{X} \underline{x} dV = \underline{x}_{V}$$

Gravitational body force

$$T_b = \int_{\mathcal{B}} (x - x_m) \times pg \, dV$$

$$= \int_{\mathcal{B}} x \times pg - x_m \times pg \, dV$$

$$g = cowst.$$

$$= m \times m \times g - \times m \times mg$$

$$= m (\times m \times g - \times m \times g) = 0$$

controid

Triangle can be suspended

(2D)

From its controid in

any orientation without

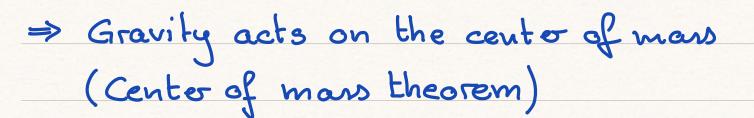
inducing rotation.

torque du to gravity around origin z=0

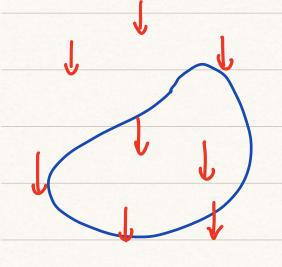
Simplify "moment of gravity"

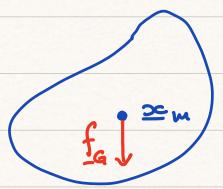
$$T_G = \int_{\mathcal{B}} \times \operatorname{RgdV} = \int_{\mathcal{B}} (\times - \times_{m} + \times_{m}) \times \operatorname{RgdV}$$

IG = Zmx m,g Homent of Gravity



Continuum





force field can be represented as acting on a point where it does not induce a torque.

Hydrostatic surface force

Moment of buoyancy Torque due to hydrostatic surface force around origin

$$\Xi_{B} = \oint_{\partial B} \times \times (-p\hat{n}) dA$$

Show that buoyancy acts on center of mans of displaced fluid == = \frac{1}{V} \speces \rightarrow \frac{1}{B} \rightarrow \fra $\Xi_{B} = \oint \simeq_{V} \times (-p\hat{n}) dA = 0$

(requires some vector calculus => lato) Use to simplify the moment of buoyancy

$$\Rightarrow \quad \Xi_{B} = - \times_{V} \times (mfg)$$

arsures body is fully submeged and pf = const.

Center of volume => center of buoyang

$$\Sigma_{\text{A}} = \Sigma_{\text{B}}$$

Hydrostatic moment

force balance:
$$f_H = f_a + f_B = (m_f - m_b) ge_3$$

similarly

Torque balance:
$$\underline{T}_{H} = \underline{T}_{G} + \underline{t}_{B}$$

$$= \underline{x}_{G} \times m_{b} g - \underline{x}_{B} \times m_{f} g$$

Itydiostatic moment:
$$\underline{T}_{H} = m(\underline{x}_{G} - \underline{x}_{B}) \times g$$