Lecture 21: Balance laws

Last time: Rates of deformation

• Velocity gradient:
$$\nabla_x V = \dot{\underline{\mathbf{f}}}$$

$$\nabla_z \underline{v} = \dot{\underline{\mathbf{f}}} \underline{v}^{-1} = \underline{\mathbf{f}}$$

· Symmetric - Skew de composition:

$$\mathcal{L} = \mathcal{Q} + \mathcal{L}$$

$$\mathcal{Q} = \frac{1}{2} (\nabla_{\underline{V}} + \nabla_{\underline{V}}^{\dagger}) \qquad \text{rate of strain tensor}$$

$$\underline{W} = \frac{1}{2} (\nabla_{\underline{V}} - \nabla_{\underline{V}}^{\dagger}) \qquad \text{spin tensor}$$

Reynolds Transport Theorem

d Sp dVz = S 30 dVz + & \$ 2 - n dAz

dt Qt Qt Qt 32

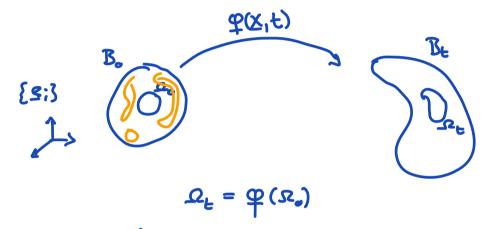
Today: - Balance laws

Local Eulerian Balance Laws

local = no integrals

Eulerieu = = spahal duscriphou

(Lagrangian: X material description)



so is arbiliary

in absence of realions or relativistic effects

was is conserved

using
$$dV_{\infty} = J(\bar{x},t) dV_{\infty} = \int_{P_{\infty}(\bar{x},t),t)} J(\bar{x},t) dV_{\infty}$$

H[D] = $\int_{P_{\infty}(\bar{x},t)} dV_{\infty} = \int_{P_{\infty}(\bar{x},t),t)} J(\bar{x},t) dV_{\infty}$

$$M[\Omega_{\bullet}] = \int_{\Omega_{\bullet}} \rho_{m}(X,t) J(X,t) dV_{X}$$

$$\Rightarrow \int_{\Omega_0} p_{\omega}(\underline{x},t) \Im(\underline{x},t) - p_{\omega}(\underline{x}) dV_{\chi} = 0$$

by arbitraryness of $\mathcal{L}_o \rightarrow \text{integrand must be zero}$ $p_{m}(\underline{X},t)J(\underline{X},t) = p_{o}(\underline{X})$

Lagrangian statement of mars conservation
$$\frac{dV_{\infty}}{dV_{X}} = \frac{\rho_{0}}{\rho_{m}}$$

Convert to Eulerian:
$$\frac{3}{2}$$

$$\frac{3}{2}\left(\rho_{m}(x,t) J(x,t)\right) = \frac{3}{24}\rho_{0}(x) = 0$$

$$\left(\frac{3}{2}\rho_{m}(x,t)\right) J(x,t) + \rho_{m} \frac{3}{2}J(x,t) = 0$$

$$\int_{m}(x,t) J(x,t) + \rho_{m} J(\nabla_{x}\cdot\underline{v})_{m} = 0$$
switch to spatial description $\rho_{m}(x,t) = \rho(x,t)$

$$\sum_{x=0}^{4}(x,t)$$

$$\rho(x,t) + \rho(x,t) \nabla_{x}\cdot\underline{v}(x,t) = 0$$

=> p + p
$$\nabla \cdot \underline{v} = 0$$
 local Eulesieu descripion

$$\frac{\Delta \cdot (b\bar{n})}{\Delta b + b \Delta \cdot n} = 0$$

=>
$$\frac{\partial p}{\partial t} + \nabla \cdot (pz) = 0$$
 conservative local Eulerian form

Time derivative of integrals with respect to mens

$$\frac{d}{dt} \int_{0}^{\infty} \phi(x^{t}) \, b(x^{t}) \, d\Lambda^{x} = \int_{0}^{\infty} \dot{\phi}(x^{t}) \, b(x^{t}) \, d\Lambda^{x}$$

$$\int_{\Omega_{k}} \phi(\underline{x},t) \, b(\underline{x},t) \, d\Lambda^{x} = \int_{\Omega_{k}} \phi^{m}(\underline{x},t) \, b^{m}(\underline{x},t) \, J(\underline{x}t) \, d\Lambda^{x}$$

$$= \int_{\Omega_{k}} \phi^{m}(\underline{x},t) \, b^{m}(\underline{x},t) \, J(\underline{x}t) \, d\Lambda^{x}$$

$$= \int_{\Sigma} \phi_{w}(x,t) \rho_{w}(x) dV_{x} = \int_{\Sigma} \phi_{w}(x,t) \rho_{w}(x,t) dV_{x}$$

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Laws of inestice (Galileo & Newbor)

linear momentum: $[LQ_t] = \int_{\Omega_t} p(z,t) \underline{v}(z,t) dV_z$ angular momentum: $[LQ_t] = \int_{\Omega_t} (z-z) \times p(z,t) \underline{v}(z,t) dV_z$

lu fixed frame of reference the change in momentum is equal to resultant force/torque.

$$\frac{d}{dt} \left[\left[\Omega_{t} \right] = \int_{\mathbb{R}^{2}} \sum_{k} \left(\sum_{i} t \right) dt \right] = \int_{\mathbb{R}^{2}} \sum_{k} \left(\sum_{i} t \right) dt + \int_{\mathbb{R}^{2}} \sum_{i} \left(\sum_{i} t \right) dt$$

$$\frac{d}{dt} \left[\left[\Omega_{t} \right] = \int_{\mathbb{R}^{2}} \sum_{k} \left(\sum_{i} t \right) dt \right] + \int_{\mathbb{R}^{2}} \left(\sum_{i} t \right) dt$$

$$\frac{d}{dt} \left[\left[\Omega_{t} \right] = \int_{\mathbb{R}^{2}} \sum_{k} \left(\sum_{i} t \right) dt \right] = \int_{\mathbb{R}^{2}} \left(\sum_{i} t \right) dt$$

$$\frac{d}{dt} \left[\left[\Omega_{t} \right] = \int_{\mathbb{R}^{2}} \sum_{i} \left(\sum_{i} t \right) dt \right] = \int_{\mathbb{R}^{2}} \sum_{i} \left(\sum_{i} t \right) dt$$

$$\frac{d}{dt} \left[\left[\Omega_{t} \right] = \int_{\mathbb{R}^{2}} \sum_{i} \left(\sum_{i} t \right) dt \right] = \int_{\mathbb{R}^{2}} \sum_{i} \left(\sum_{i} t \right) dt$$

Balance of linear montentum

Couchy stress:
$$\underline{t} = \underline{\underline{s}} \underline{\underline{u}}$$
 $\frac{d}{dt} \int_{\Omega_b} \underline{p} \underline{u} dV_{x} = \int_{\Omega_b} \underline{\underline{u}} dA_{x} + \int_{\Omega_b} \underline{\underline{b}} dV_{x}$

usind derivative with respect to mans
$$\int \rho \dot{v} - \nabla \cdot \underline{\underline{e}} - \rho \underline{\underline{b}} \ dV_{\infty} = 0$$
Re

by arbitraryours et -> bealize

=> pv -
$$\nabla \cdot \underline{\underline{z}} = \underline{\rho}\underline{\underline{b}}$$
 local Eulesian form of lin. mom. balance.

Cauchy's first equation of mation

conservative local Eulerion form

v=0 -> - \(\begin{array}{c} = \

Balance of angular momentum

lhs:

$$= \int_{\mathbb{R}} b(\overline{x} \times \overline{n}) dV^{2}$$

$$= \int_{\mathbb{R}} b(\overline{x} \times \overline{n} + \overline{x} \times \overline{n}) dV^{2}$$

$$= \int_{\mathbb{R}} b(\overline{x} \times \overline{n} + \overline{x} \times \overline{n}) dV^{2}$$

$$= \int_{\mathbb{R}} b(\overline{x} \times \overline{n}) dV^{2}$$

substitute (auchy stress $t = \underline{\epsilon} \underline{n}$ into r.h.s. $\int p(\underline{x} \times \underline{v}) dV_{x} = \int_{\underline{x}} \times \underline{\epsilon} \underline{n} dA_{x} + \int p(\underline{x} \times \underline{b}) dV_{x}$ Re

Re

$$\Rightarrow \int_{\mathbb{R}^{2}} \times \sqrt{x} \cdot \leq dV_{\infty} = \int_{\mathbb{R}^{2}} \times \leq \tilde{u} dA_{\infty}$$

This is exactly statement for the static mechanical egbh => Lecture 14

=> [===T] extends to transient cases?