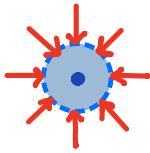


Simple states of stress

I) Hydrostatic stress

$$\underline{\underline{\sigma}} = -p \underline{\underline{I}} \Rightarrow \underline{t}_n = \underline{\underline{\sigma}} \underline{n} = -p \underline{n} \quad \text{for all } \underline{n}$$

$$\underline{t}_n'' = \underline{P}_n'' \underline{t} = (\underline{n} \otimes \underline{n})(-p \underline{n}) = -p (\underline{n} \cdot \underline{n}) \underline{n} = -p \underline{n}$$



$$\Rightarrow \underline{t}_n = \underline{t}_n'' \quad \underline{t}_n^\perp = 0$$

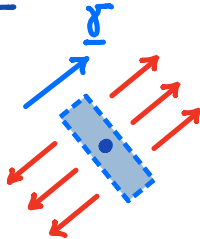
all normal stresses are $\sigma_1 = \sigma_2 = \sigma_3 = -p$

no shear stresses on any plane

II) Uniaxial stress

$$\underline{\underline{\sigma}} = \sigma \underline{\underline{\gamma}} \otimes \underline{\underline{\gamma}} \Rightarrow \underline{t}_n = \underline{\underline{\sigma}} \underline{n} = \sigma (\underline{\gamma} \cdot \underline{n}) \underline{\gamma}$$

($\underline{\gamma}$ is unit vector) Traction is always parallel to $\underline{\gamma}$



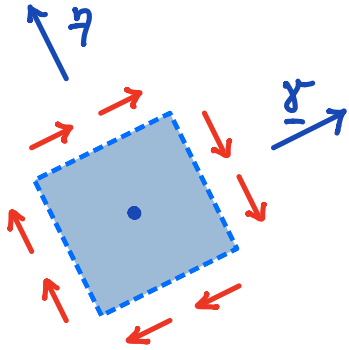
and vanished on surfaces with normal perpendicular to $\underline{\gamma}$.

$\sigma > 0$: pure tension

$\sigma < 0$: pure compression

III, Pure shear stress

$$\underline{\underline{\sigma}} = \tau (\underline{\underline{x}} \otimes \underline{\underline{\eta}} + \underline{\underline{\eta}} \otimes \underline{\underline{x}}) \Rightarrow \underline{\underline{\epsilon}}_n = \underline{\underline{\sigma}} \underline{\underline{n}} = \tau (\underline{\underline{\eta}} \cdot \underline{\underline{n}}) \underline{\underline{x}} + \tau (\underline{\underline{x}} \cdot \underline{\underline{n}}) \underline{\underline{\eta}}$$



$$\underline{\underline{n}} = \underline{\underline{\eta}}: \underline{\underline{\epsilon}}_n = \tau \underline{\underline{x}}$$

$$\underline{\underline{n}} = \underline{\underline{x}}: \underline{\underline{\epsilon}}_n = \tau \underline{\underline{\eta}}$$

IV, Plane stress

If there exists a pair of orthogonal vectors $\underline{\underline{x}}$ and $\underline{\underline{\eta}}$ such that the matrix representation of $\underline{\underline{\sigma}}$ in the frame $\{\underline{\underline{x}}, \underline{\underline{\eta}}, \underline{\underline{x}} \times \underline{\underline{\eta}}\}$ is of the form

$$[\underline{\underline{\sigma}}] = \begin{pmatrix} \sigma_{11} & \sigma_{12} & 0 \\ \sigma_{21} & \sigma_{22} & 0 \\ 0 & 0 & 0 \end{pmatrix}$$

then a state of plane stress exists.

Spherical and deviatoric stress tensors

The Cauchy stress tensor can be decomposed

as $\underline{\underline{\sigma}} = \underline{\underline{\sigma}}_S + \underline{\underline{\sigma}}_D$

spherical stress tensor: $\underline{\underline{\sigma}}_S = -p \underline{\underline{I}} \quad p = -\frac{1}{3} \text{tr}(\underline{\underline{\sigma}})$

deviatoric stress tensor: $\underline{\underline{\sigma}}_D = \underline{\underline{\sigma}} + p \underline{\underline{I}}$

The pressure $p = -\frac{1}{3} \text{tr}(\underline{\underline{\sigma}}) = -\frac{1}{3} (\sigma_1 + \sigma_2 + \sigma_3)$

can be interpreted as the mean normal

traction. The spherical stress is the part

of $\underline{\underline{\sigma}}$ that changes the volume of the body.

Note that $p > 0$ corresponds to compression.

The deviatoric stress is the part of $\underline{\underline{\sigma}}$ that changes the shape of a body without changing its volume. By definition $\text{tr} \underline{\underline{\sigma}}_D = 0$.

Principal invariants of $\underline{\underline{\sigma}}_D$:

$$I_1(\underline{\underline{\sigma}}_D) = \text{tr } \underline{\underline{\sigma}}_D = 0$$

$$J_2(\underline{\underline{\sigma}}) = -I_2(\underline{\underline{\sigma}}_D) = \frac{1}{2} \underline{\underline{\sigma}}_D : \underline{\underline{\sigma}}_D$$

$$J_3(\underline{\underline{\sigma}}) = I_3(\underline{\underline{\sigma}}_D) = \det \underline{\underline{\sigma}}_D$$

The invariants J_2 and J_3 of the deviatoric stress $\underline{\underline{\sigma}}_D$ are used to formulate yield functions in theory of plasticity.