IBONG TIRIRIT (MDSP 3)

MACHINE DESIGN/SHOP PRACTICE (Solved Problems and Answered Elements)

MULTIPLE CHOICE QUESTIONS

Select the best answer from each of the following questions. On the answer sheet provided, shade the box that corresponds to your choice. Strictly no erasures allowed

900 kg of load is to be raised using a double threaded ACME screw driven by a motor at 400rpm. The load is to be raised at a speed of 10 m/min. The screw has a pitch diameter of 36 mm; the coefficient of friction on threads is 0.15. The friction torque on the thrust bearing of the motor is taken as 20 % of the total input. Determine the lead angle.

a) 12.465°

Solution: For the lead,
$$\,L=\frac{V}{n}=\frac{10}{400}=0.025\;m=25\;mm\,$$

For the lead angle,
$$\lambda = tan^{-l} \left(\frac{L}{\pi D_m}\right) = tan^{-l} \left(\frac{25}{36\pi}\right) = 12.465^o$$

A 500 mm brake drum has a simple brake that is to absorb a torque of 250 N-m. If the coefficient of friction and the contact angle between the band and drum are 0.2 and 180°, respectively, and that the length of the brake arm is twice the drum diameter, determine the minimum braking force applied at the free end of the brake arm.

- c) 275.1 N
- d) 572.1 N

$$\text{Solution: } \frac{F_1}{F_2} = e^{f\theta} = 1.874: \quad \rightarrow F_{bmin} = \frac{F_2(a)}{L}: a = D: \quad \rightarrow L = 2D \quad \rightarrow F_{bmin} = \frac{F_2}{2} = \frac{\frac{2T}{D}}{2(1.874-1)} = 572.1 \text{N}$$

In a simply-supported shaft of 2-ft span of 2-in. diameter, the allowable flexural stress is 6,000 psi. Determine the maximum permissible concentrated bending load that it may be carried.

a) 854.7 lb

- c) 587.4 lb
- d) 785.4 lb

Solution: Bending moment is maximum if F is applied at mid span.

$$\sigma = \frac{Mc}{L} = \frac{32M}{\pi D^3} \rightarrow M = \frac{FL}{4} \rightarrow \frac{32(F)(24)}{4\pi(2^3)} = 6000 \rightarrow F = 785.4 \text{lb.}$$

A tensile load of 8 tons is transmitted to a bar of rectangular section is made of AISI C1020 steel (Su = 65,000 psi). If the width of the bar is 1.5 times the thickness, find the bar dimensions if the factor of utilization is 0.25 based upon the ultimate strength.

a) 0.5" x 0.75"

Solution: Factor of Utilization = $\frac{1}{\text{Factor of Safety}}$

$$s = \frac{F}{1.5b^2} = \frac{S_u}{N} = (U.F \times S_u): \frac{8 \times 2000}{1.5b^2} = 0.25 \times 65000: \rightarrow b = 0.81 \text{ in.}$$

and b x h =
$$0.81$$
" x 1.22 "

A connecting rod is to form from a low carbon steel sheet 3/8 in. thick, lined with aluminum 1/8 in. thick. Steel has a modulus of elasticity of 30 x 106 psi, while aluminum of 10.3 x 10⁶ psi. If the aluminum has a tensile stress of 10,000 psi, what is the corresponding stress of the steel sheet? c) 29,126 psi d) 26.191 psi

Solution: σ = E ϵ : With constant strain ϵ , $\sigma_{AL}/E_{AL} = \sigma_{ST}/E_{ST}$.

Thus,
$$\sigma_{ST} = (30/10.3) \times (10,000) = 29,126 \text{ psi.}$$

A 1.5 m long, 9 cm square bar is to be held firmly at one end and to support a load of 2000 kg at the other end. The bar is to be made from steel with yield strength of 500 MPa. What factor of safety is used in the design if the stress concentration factor is 1.3? b) 3.45 c) 2.67

Where,
$$M = FL = 29430000 \text{ N} \cdot \text{mm}$$

Solution:
$$\frac{S_Y}{N} = \frac{K_\tau Mc}{I}$$
 Where, $M = FL = 29\,430\,000\,\text{ N} \cdot \text{mm}$ $I = \frac{a^4}{12} = 5\,467\,500\,\text{mm}^4$ $c = a/2 = 45\,\text{mm}$

then
$$N = \frac{S_y I}{K.Mc}$$
 thus: $N = 1.59$

thus:
$$N = 1.59$$

7. 222.449 kN of tensile load is applied to a round bar made from SAE 1025 low carbon steel. Using a factor of safety of 4 and assuming that the stress concentration factor is 1.45, what is the required outside diameter in mm of the member if the ratio of the inside and outside diameter is 0.75? For the material, the yield strength is 276 MPa.

a) 283.34 b) **116.64** c) 98.76 d) 156.54

8. An abrupt brake locked that the car begins to skid when it is traveling at 96.6 kph. If it takes 2 seconds to slow down the car to 4.83 kph, how far in meters will it have traveled before it car comes to a stop?

a) **53.64** b) 276 c) 137 d) 876

Solution: Velocities, in m/s,
$$V_i = 96.6 \text{ kph} = 26.83 \text{ m/s}$$
 $V_f = 48.3 \text{ kph} = 13.42 \text{ m/s}$

Acceleration,
$$a = \frac{V_f - V_i}{t} = \frac{13.42 - 26.83}{2} = -6.71 \text{ m/s}^2$$
 Distance, $s = \frac{{V_f}^2 - {V_i}^2}{2a} = \frac{0 - (26.83)^2}{2(-6.7)} = 53.64 \text{ m}$

9. What is the weight of a 2 steel plates sized ¾" x 3" x 20 ft?
a) 1836 lb b) 3667 lb c) 1698 lb d) 1848 lb

Solution: Density or specific weight of a steel plate = 0.283 lb/ft³

$$W = \gamma V = (0.283 \text{ lb/in}^3)(0.75 \text{ in.})(3)(12 \text{in})(20)(12 \text{ in})(2) = 3667.68 \text{ lb}$$

10. Determine the load required, in kN, to be applied to a 25 mm round steel bar 2.5 m long (E = 207 GPa) to stretch the bar 1.3 mm. a) 52.84 b) 60 c) 53.048 d) 42.562

Solution:
$$F = \frac{AE}{L}\delta = \frac{\left(\frac{\pi}{4}D^2\right)E}{L}\delta = \frac{\left(\frac{\pi}{4}\right)(25)^2(207\,000)(1.3)}{2.500} = 52\,837.66\ N = 52.84\ kN$$

11. A bus is travelling at 90 kph when the brakes are applied. If it travels a distance of 60 meters before coming to a stop, what is its acceleration m/s²?

a) - 7.08

b) 6.36

c) - 5.21

d) 5.76

Solution:
$$V_1 = (90 \text{ kph})(1000 \text{ m/km}) \left(\frac{1 \text{ hr}}{3600 \text{ s}}\right) = 25 \text{ m/s}$$
 $a = \frac{V_2^2 - V_1^2}{2S} = \frac{0 - (25)^2}{2(60)} = -5.20 \text{ m/s}^2$

12. Find the torsional deflection, in degrees per foot length, of a steel shaft 1.75 inches in diameter which transmits 40 Hp at 1800 rpm. Assume a modulus of rigidity of 12 x 10⁶ psi.

a) 0.0871 b) 0.00871 c) 0.871 d) 0.00871

Solution:
$$\frac{\theta}{L} = \frac{T}{JG} = \frac{32T}{\pi D^4 G} = \frac{32 \left(\frac{63\,000 \text{ Hp}}{n}\right)}{\pi D^4 G} = \frac{32 \left[\frac{63000 (40)}{1800}\right]}{\pi (1.75)^4 (12 \times 10^6)} = 0.00013 \text{ radian} = 0.0871^\circ$$

13. A spur pinion rotates at 1600 rpm and transmits 50 kW to a mating gear. The pitch diameter of the pinion is 100 mm, and the pressure angle is 20°. Determine the tangential load, in N.

a) 5 968 b) 5790 c) 5900 d) 5098

$$\text{Solution: } F_t = \frac{2T}{D} = \frac{2 \bigg(\frac{30P}{\pi n} \bigg)}{D} = \frac{2 \bigg[\frac{30(50)}{\pi \big(1600 \big)} \bigg]}{(0.10)} = 5.968 \text{ kN} = 5 968 \text{ N}$$

14. How much energy, in N-m, does a 450 kg flywheel loss from 180 rpm to 168 rpm? The flywheel has a radius of 375mm.
a) 1472.33 b) 1541.33 c) 1451.33 d) 1433.51

$$\text{Solution:} \ \ V_1 = 2\pi R N_1 \ / \ 60 = 2\pi \left(0.375\right) \!\! \left(\frac{180}{60}\right) = 7.069 \ \ \text{m/s} \\ \qquad V_2 = 2\pi R N_2 \ / \ 60 = 2\pi \left(0.375\right) \!\! \left(\frac{168}{60}\right) = 6.597 \ \ \text{m/s} \\ \qquad V_3 = 2\pi R N_2 \ / \ 60 = 2\pi \left(0.375\right) \!\! \left(\frac{168}{60}\right) = 6.597 \ \ \text{m/s} \\ \qquad V_4 = 2\pi R N_2 \ / \ 60 = 2\pi \left(0.375\right) \!\! \left(\frac{168}{60}\right) = 6.597 \ \ \text{m/s} \\ \qquad V_5 = 2\pi R N_2 \ / \ 60 = 2\pi \left(0.375\right) \!\! \left(\frac{168}{60}\right) = 6.597 \ \ \text{m/s} \\ \qquad V_7 = 2\pi R N_2 \ / \ 60 = 2\pi \left(0.375\right) \!\! \left(\frac{168}{60}\right) = 6.597 \ \ \text{m/s} \\ \qquad V_8 = 2\pi R N_2 \ / \ 60 = 2\pi \left(0.375\right) \!\! \left(\frac{168}{60}\right) = 6.597 \ \ \text{m/s} \\ \qquad V_8 = 2\pi R N_2 \ / \ 60 = 2\pi \left(0.375\right) \!\! \left(\frac{168}{60}\right) = 6.597 \ \ \text{m/s} \\ \qquad V_8 = 2\pi R N_2 \ / \ 60 = 2\pi \left(0.375\right) \!\! \left(\frac{168}{60}\right) = 6.597 \ \ \text{m/s} \\ \qquad V_8 = 2\pi R N_2 \ / \ 60 = 2\pi \left(0.375\right) \!\! \left(\frac{168}{60}\right) = 6.597 \ \ \text{m/s} \\ \qquad V_9 = 2\pi R N_2 \ / \ 60 = 2\pi \left(0.375\right) \!\! \left(\frac{168}{60}\right) = 6.597 \ \ \text{m/s}$$

$$\Delta KE = \frac{m(V_1^2 - V_2^2)}{2} = \frac{450[(7.069)^2 - (6.597)^2]}{2} = 1451.33 \text{ N} \cdot \text{m}$$

15. A ¾ inch thick steel plate is to punch for a 1-in diameter hole every 10 sec. The actual punching takes 1 sec. The ultimate shear strength of the plate is 60,000 psi. The flywheel of the punch press has a mass moment of inertia of 500 in-lb-sec2 and rotates at a mean speed of 150 rpm. What is the speed fluctuation, in rpm?

a) 65.4 rpm

c) 6.749 rpm

d) 67.49 rpm

$$\text{Solution: } \Delta KE = \left(\frac{1}{2}\right) I\left(\omega_1^2 - \omega_1^2\right) = \left(\frac{1}{2}\right) I\left(\omega_1 - \omega_2\right) \left(\omega_1 + \omega_2\right) = I\left(\omega_1 - \omega_2\right) \left(\frac{\omega_1 + \omega_2}{2}\right) = I\left(\omega_1 - \omega_2\right) \omega_1 + \omega_2 = I\left(\omega_1 - \omega_2\right) \omega_2 + \omega_2 + \omega_2 = I\left(\omega_1 - \omega_2\right) \omega_2 + \omega_2 +$$

Mean angular velocity,
$$\omega = \frac{2\pi N}{60} = \frac{2\pi (150)}{60} = 15.71 \,\text{rad/s}$$

Punching Force,
$$F = s_u (\pi dt) = (60\,000)(\pi)(1)(0.75) = 141371.67$$
 lb

Kinetic Energy,
$$\Delta KE = \frac{1}{2}(F)t = (\frac{1}{2})(141\ 371.67)(\frac{0.75}{12}) = 4417.86\ \mathrm{ft}\cdot\mathrm{lb}$$

Speed Fluctuation,
$$(\omega_1 - \omega_2) = \frac{\Delta KE}{I \, \omega} = \frac{\left(4417.86 \, \text{ft} \cdot \text{lb}\right)\!\left(12\right)}{\left(500 \, \text{in} \cdot \text{lb} \cdot \text{s}^2\right)\!\left(15.71\right)} = 6.749 \, \text{rad/s} = 64.5 \, \text{rpm}$$

16. An elevator is used to assist the construction of a building. It rises 366 meters with an operating speed of 5 meters per second and reaches full speed in 10.68 meters. When loaded with construction materials, the elevator weighs 2000 kg. Determine the acceleration of the elevator in m/sec².

Solution:
$$a = \frac{V_2^2 - V_1^2}{2S} = \frac{(5)^2 - (0)^2}{2(10.68)} = 1.17 \text{ m/s}^2$$

17. A disc clutch is made of a material with coefficient of friction of 0.4. The shaft speed is 1200 rpm and axial force is 1500 N. The clutch has 6 pairs of contacting friction surfaces with an outside diameter of 200 mm and an inside diameter of 100 mm. Determine the Hp that can be transmitted by the clutch assuming uniform pressure.

a) 35.2 Hp

b) 23.5 Hp

c) 47.2 Hp

d) 27.4 Hp

Solution: Friction Radius (Uniform pressure), $r_f = \frac{2}{3} \left(\frac{r_o^3 - r_i^3}{r^2 - r_i^2} \right) = \left(\frac{2}{3} \right) \left[\frac{(100)^3 - (50)^3}{(100)^2 - (50)^2} \right] = 77.78 \text{ mm}$

Friction Power or Transmitted Power or Power Capacity,

$$P = \frac{\pi n T_f}{30} = \frac{\pi n}{30} (F_f r_f n_f) = \frac{\pi n}{30} (f F_a r_f n_f) = \frac{\pi (1200)}{30} (0.4)(1.5)(0.07778)(6)$$

Where,

P = power capacity, kW

Fa = axial load, kN

 $\begin{aligned} F_f &= \text{friction force, kN} & & & & & & & & \\ F_f &= &\text{friction radius or mean radius, m} & & & & & \\ & & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & \\ & & & \\$

In a butt weld between two plates 0.5 inch thick and 5 inches long, what load may be applied if the allowable working stress for tensile loading is 13000 psi?

a) 32 500 lb

b) 25 000 lb

c) 44 000 lb

d) 50 000 lb

Solution: For butt weld, $P = s_t t L = (13000 psi)(0.5 inch)(5 inches) = 32500 lb$

19. 150 N-load is tangentially applied to a 140 mm diameter wheel. If the load is constant, determine the work done, in J, in 12 revolutions of the wheel. a) 729 J b) 792 J c) 972 J d) 927 J

Solution: $W = F \cdot s = (150 \text{ N})(\pi)(0.14 \text{ m})(12) = 791.68 \text{ J}$

20. Determine the force applied tangentially to a bar of screw-jack at a radius of 800 mm if the torque required is 600 N-m.

a) 705 N

- b) 507 N
- c) 750 N
- d) 570 N

Solution:
$$F = \frac{T}{R} = \frac{600}{0.80} = 750 \text{ N}$$

21. A 2.54 mm thin hollow sphere is subjected to an internal pressure of 789 kPa. It has a radius of 254 mm. Determine the maximum normal stress on an element of the sphere.

- c) 34.95 kPa
- d) 34.95 MPa

Solution:
$$s_L = \frac{pD}{4t} = \frac{(789 \text{ kPa})(2)(0.254 \text{ m})}{4(0.00254)} = 39450 \text{ kPa} = 39.45 \text{ MPa}$$

22. Determine the hoop stress in a 1.52 m diameter steel pipe that carries water at a pressure head of 168 m. Thickness of the pipe is 9.52 mm. a) 131.5 MPa c) 153.1 MPa b) 135.1 MPa

Solution: $s_t = \frac{pD}{2t} = \frac{\rho ghD}{2t} = \frac{(1000)(9.8066)(168)(1.52)}{2(0.00952)(1000)} = 131523.81 \text{ kPa} = 131.524 \text{ MPa}$

When loaded with a tensile force, a wire of length 2.5 m has a percentage strain of 0.012 %. Determine the extension of the wire. b) 0.20 mm c) 0.30 mm

Solution: $\delta = \text{Strain x } L = \left(\frac{0.012}{100}\right)(2.5 \text{ m})(1000) = 0.3 \text{ mm}$

24. A 20° involute type spur gear has a diametral pitch of 6. If it has a circular pitch of 0.1309 inch, determine the minimum whole depth of the gear. a) 0.5593 inch b) 0.5395 inch c) 0.3595 inch d) 0.3955 inch

Solution: From Vallance, page 262: $h = \frac{2.157}{p_d} = \frac{2.157}{6} = 0.3595$ inch

25. A tempered steel spring is used in a gas engine valve with a mean diameter of 3.81 cm and a wire diameter of 0.635 cm. The maximum load it will have to sustain is 45.15 kg with corresponding deflection of 1.27 cm. Determine the no. of coils to be used. Use modulus of rigidity equal to 80

a) 7.4

d) 14.4

$$\delta = \frac{8FC^3n}{Gd}$$

$$\text{Solution: } C = \frac{D_m}{d} = \frac{3.81}{0.635} = 6 \\ \delta = \frac{8FC^3n}{Gd} \\ n = \frac{\delta\,G\,d}{8FC^3} = \frac{0.0127 \big(80\,\times\,10^9\,\big)\!\big(0.00635\big)}{8\big(45.15\big)\!\big(9.8066\big)\!\big(6\big)^3} = 8.43$$

Actual Number of coils = n + 28.43 + 2 = 10.43

Determine the developed acceleration of a roller coaster in m/s² when the velocity is 20 m/s at a radius of curvature of 6500 cm. b) 6.15 m/s² a) 6.15 m/s² c) 6.51 m/s² d) 5.61 m/s²

Solution: Weight = Centrifugal Force

$$ma = \frac{mV^2}{r}$$

$$a = \frac{V^2}{r} = \frac{(20)^2}{65} = 6.15 \text{ m/s}^2$$

27. A solid shaft is to be used to transmit 75 kW at 550 rpm. If the shaft design stress will not exceed 26 MPa, what is the diameter of the shaft? b) 42.63 mm c) 36.42 mm

Solution: $T = \frac{30P}{\pi n} = \frac{30(75)}{\pi (550)} = 1.3022 \text{ kN} \cdot \text{m}$ $D = \left(\frac{16T}{\pi s}\right)^{\frac{1}{3}} = 63.42 \text{ mm}$

Determine the kinetic energy of a 48-inch diameter spoked steel flywheel ($\gamma = 0.28 \text{ lb/in}^3$) having a 12-in wide x 10-in deep rim rotates at 200 rpm. Neglect the weight of its spokes and hub.

a) 84,448 ft-lb

- b) 68.488 ft-lb
- c) 42,248 ft-lb

Solution: $D_{\rm m} = D_{\rm o} - t = 48 - 10 = 38$ inches

$$V = \pi D_m \left(\frac{n}{60}\right) = \pi \left(\frac{38}{12}\right) \left(\frac{200}{60}\right) = 33.16 \text{ fps}$$

$$W_f = \gamma V = 0.28(\pi) \left(38\right) \left(12\right) \left(10\right) = 4011.21b \qquad \text{KE} = \frac{1}{2} \frac{W_f}{g} V^2 = E_{\text{punching}} = \left(\frac{1}{2}\right) \left(\frac{4011.2}{32.2}\right) \left(33.16\right)^2 = 68488 \text{ ft} \cdot \text{lb}$$

29. A square bar is held so that it cannot expand while raising its temperature by 100 °F. Size of the bar is 2 inches on each side. What stress will be induced in it? $k = 6.8 \times 10^{-6} / {}^{\circ}F \& E = 30 \times 10^{6} \text{ psi}.$

a. 17,400 psi

b. 18,400 psi

c. 19,400 psi

d. 20,400 psi

Solution: Induced Stress, $s = k E(t_2 - t_1) = (6.8 \times 10^{-6})(30 \times 10^6)(100) = 20400 \text{ psi}$

Find the moment of inertia, in inches⁴, of the rod ¼ inch in diameter and 14 inches long. 30.

d) 3.83 x 10⁻⁶

Solution: $I = \frac{\pi D^4}{64} = \frac{\pi (0.25)^4}{64} = 1.917 \times 10^{-4} \text{ in}^4$

31. A disc clutch having an outside diameter of 32 cm and an inside diameter of 12.7 mm is connected to an engine that turns at 750 rpm. The coefficient of friction is 0.6 while the pressure between the friction surfaces is 2 kg/cm². Determine the force on the clutch petal necessary to disengage the clutch disc from the engine.

a) 2621.7 kg

c) 2524.2 kg

d) 1355.1 kg

Solution: $F = pA = p[\pi(R^2 - r^2)] = 1355.14 \text{ kg}$

32. A helical gear of 10 inches pitch diameter has a helix angle of 30°, and there are 30 teeth. Find the value of the normal circular pitch. a) 0.9069 inch/tooth b) 0.6909 inch/tooth c) 0.6096 inch/tooth d) 0.9906 inch/tooth

Solution: $P_c = \frac{\pi D}{T} = \frac{\pi (10)}{30} = 1.0472$ inches/tooth $P_{cn} = P_c \cos \psi = 1.0472 (\cos 30^\circ) = 0.9069$ inch/tooth

- 33. A parallel helical gear set was a 17-tooth pinion driving a 34-tooth gear. The pinion has a right-hand helix angle of 30°, a normal pressure angle of 20°, and a normal diametral pitch of 5 teeth/in. Find the axial circular pitches.

- b) 1.6625 inches/tooth
- c) 1.6526 inches/tooth

Solution: $P_c = \frac{P_{cn}}{\cos W} = \frac{0.62832}{\cos 30^\circ} = 0.72552$ inch/tooth

$$P_a = \frac{P_c}{\tan \psi} = \frac{0.72552}{\tan 30^\circ} = 1.2566 \text{ inches/tooth}$$

34. A deep-groove ball bearing is to carry a radial load 800 lb and a thrust load 700 lb at 1800 rpm. The service is 8 hr/day, but it is not continuous; design for 18 250 hr. The operation is smooth with little vibration; the outer ring rotates. Determine the design life in million revolution (mr) with no more than 10 % failure.

a) 1791 mr

Solution: $B_{10} = (Hrs)(60 \text{ min s / hr})(rpm) = \frac{(18250)(60)(1800)}{10^6} = 1971 \text{ mr}$

35. A shaft that transmits 1000 Hp at 1000 rpm is to be design with a square key. If the allowable shear and compressive stresses in the key are 15 ksi and 30 ksi, respectively, what length of the key is required?

a) 2.1 inches

- b) 2.8 inches
- d) 4.2 inches

Solution: Transmitted torque, $T = \frac{63000 \text{ Hp}}{n} = \frac{63000(1000)}{1000} = 63000 \text{ in } \cdot \text{lb}$

Key width, $b \approx \frac{D}{4}$, for good proportion

Key length based on shear, $L = \frac{2T}{s_a bD} = \frac{2(63000)}{15000(1)(4)} = 2.1$ inches

Key length based on compression, $L = \frac{4T}{s_c t D} = \frac{4(63000)}{30000(1)(4)} = 2.1$

Therefore, use L = 2.1 inches

Two shafts, both 2.5 inches in diameter, is to be connected by a flange coupling. It is to transmit 15,000 in-lb of torque. How many ½" diameter bolts in a 6" diameter bolt circle are required if the shear stress in each bolt is limited to 3000 psi?

a) 9 bolts

- d) 8 bolts

Solution: No. of bolts,
$$n_b = \frac{8T}{\pi s_{sb} d^2 D_b} = \frac{8(15000)}{\pi (3000)(0.5)^2 (6)} = 8.5 \text{ bolts}$$

Use, $n_b = 9$ bolts

- 37. A wheel has an outside and inside diameter of 1 meter and 0.75 meter, respectively. If its arms and hub is neglected, determine the radius of gyration of the wheel.
 - a) 0.5125 m
- b) 0.6125 m
- c) **0.3125 m**
- d) 0.4125 m

$$\text{Solution:} \quad k = \sqrt{\frac{I}{A}} = \sqrt{\frac{\frac{\pi}{64} \left(D_o^4 - D_i^4\right)}{\frac{\pi}{4} \left(D_o^2 - D_i^2\right)}} = \sqrt{\frac{1}{16} \left[\frac{\left(D_o^2 + D_i^2\right) \left(D_o^2 - D_i^2\right)}{D_o^2 - D_i^2}\right]} = \frac{1}{4} \sqrt{D_o^2 + D_i^2} = 0.3125 \text{in.}$$

- 38. Two sleeve bearings support a shaft. The total load on the bearings is 2800 lb. Diameter of the shaft is 1.75 inch. Find the friction power loss, in Hp, if the coefficient of friction between shaft and bearing is 0.10 and the shaft rotates 200 rpm.
 - a) 0.88 Hp

- b) 0.78 Hp
- c) 0.98 Hp

d) 0.68 Hp

Solution:

$$fHp = \frac{T_f n}{63\,000} = \frac{f Fr n}{63\,000} = \frac{0.10(2\,800)\left(\frac{1.75}{2}\right)(200)}{(63\,000)} = 0.78$$

Where,

 T_f = frictional torque, in-lb f = coefficient of friction

F = load. lb

n = rpm

- r = bearing diameter, inches
- 39. A collar has an outside diameter of 4 inches and an inside diameter of 2 inches. If it is loaded with a 2000 lb and rotates at 50 revolution per minute, determine its Hp lost. Coefficient of friction is 0.15.
 - a) 0.7314 Hp
- b) 0.3714 Hp
- c) 0.4713 Hp

d) 0.4371 Hp

Solution:

$$fHp = \frac{T_f n}{63000} = \frac{f W r_f n}{63000} = \frac{0.15(2000 lb)(1.56 inches)(50 rpm)}{63000} = 0.3714$$

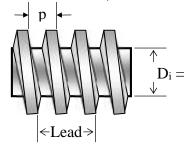
Where,
$$r_f = \frac{2}{3} \left(\frac{r_o^3 - r_i^3}{r_o^2 - r_i^2} \right) = \frac{2}{3} \left[\frac{(2)^3 - (1)^3}{(3)^2 - (1)^2} \right] = 1.56 \text{ inches}$$

- 40. A double screw thread has a pitch of 0.2 inch. Its root diameter is 0.55 inch. Find the outside diameter and the number of thread per inch.
 - a) 0.2 threads/inch
- b) 10 threads/inch
- c) 5 threads/inch
- d) 2.5 threads/inch

Solution:

Solving for the outside or major diameter, $D_o = D_i + p = 0.55 + 0.2 = 0.75$ inch

Solving for the number of threads per inch, $TPI = \frac{1}{p} = \frac{1}{0.2} = 5$ threads / inch



- 41. A driving pulley, 6 inches in diameter, carries a 6-inch wide and 1/3-inch flat belt that transmit 15Hp. The driving pulley rotates at 2000 rpm such that the loose side of the belt is on top. The driven pulley is 18 inches in diameter. The center distance is 8 ft. The belt material is 0.035 lb/in³ and the coefficient of friction is 0.30. Determine the belt net tension.
 - a) 175.5 lb

b) 157.5 lb

c) 155.7 lb

d) 165.7 lb

Solution:

$$F = F_1 - F_2 = \frac{2T}{D} = \frac{2}{D} \left(\frac{63000 \text{ Hp}}{n} \right) = \left(\frac{2}{6} \right) \left[\frac{63000(15)}{2000} \right] = 157.5 \text{ lb}$$

Other Solution: $V_m = \pi D n = \pi \left(\frac{6}{12}\right) (2000) = 3141.59 \text{ fpm}$

$$F = F_1 - F_2 = \frac{33\,000\,\text{Hp}}{V_m} = \frac{33\,000(15)}{3141.59} = 157.56\,\text{ lb}$$

- 42. A disc clutch is made of a material with coefficient of friction of 0.4. The shaft speed is 1200 rpm and axial force is 1500 N. The clutch has 6 pairs of contacting friction surfaces with an outside diameter of 200 mm and an inside diameter of 100 mm. Determine the Hp that can be transmitted by the clutch assuming uniform pressure.
 - a) 35.2 Hp

b) 23.5 Hp

c) 47.2 Hp

d) 27.4 Hp

Solution:

Friction Radius (Uniform pressure), $r_f = \frac{2}{3} \left[\frac{r_o^3 - r_i^3}{r_o^2 - r_i^2} \right] = \left(\frac{2}{3} \right) \left[\frac{(100)^3 - (50)^3}{(100)^2 - (50)^2} \right] = 77.78 \text{ mm}$

Friction Power or Transmitted Power or Power Capacity,

$$P = \frac{\pi n T_f}{30} = \frac{\pi n}{30} (F_f r_f n_f) = \frac{\pi n}{30} (f F_a r_f n_f) = \frac{\pi (1200)}{30} (0.4)(1.5)(0.07778)(6)$$

$$P = 35.19 \text{ kW} = 47.2 \text{ Hp}$$

P = power capacity, kW F_a = axial load, kN

 F_f = friction force, kN

R_f = friction radius or mean radius, m

T_f = torque capacity, kN-m

n_f = no. of pairs of contacting friction surfaces

43. A flywheel is made up of cast iron with specific weight of 0.26 lb per cubic inch. It has a width of 8 inches, mean diameter of 4 ft and mean operating speed is 300 rpm and the coefficient of fluctuation is to be 0.05. It is required to handle 2250 ft-lb of kinetic energy. Find the weight of rim, assuming that the arms and hub are equivalent to 10 % of the total rim weight.

a) 333.7 lb

d) 733.3 lb

Solution:
$$\Delta KE = \frac{1.10 \text{ W}}{g} C_f V^2$$

$$W = \frac{g(\Delta KE)}{1.10 C_f V^2} = \frac{32.2(2250)}{1.10(0.05)(62.83)^2} = 333.7 \text{ lb}$$

Where, V = mean velocity = $\frac{\pi Dn}{60} = \frac{\pi (4)(300)}{60} = 62.83$ fps

C_f = coefficient of fluctuation

g = acceleration due to gravity = 32.2 fps²

ΔKE = Kinetic Energy

44. The axial pitch of a right-handed single-thread worm is 25 mm, normal pressure angle is 14.5°, and the pitch diameter of the worm is 100 mm. It is to be made from hardened steel which has a catalog rating of 2.25 kW at 650 rpm when meshed with a 48-tooth cast-steel gear. The coefficient of friction is 0.085. Determine the shafts center distance.

a) 241 mm

b) 142 mm

c) 412 mm

d) 124 mm

Solution:

$$Speed \ Ratio, \ SR = \frac{\omega_w}{\omega_g} = \frac{n_w}{n_g} = \frac{T_g}{T_w} = \frac{D_g \cos \lambda}{D_w \sin \lambda} = \frac{D_g}{D_w \tan \lambda}$$

$$Tan\lambda = \frac{L}{\pi D_w} = \frac{p}{\pi D_w} = \frac{25}{\pi (100)} = 0.07958$$
 $\rightarrow \lambda = 4.55^{\circ}$

$$\text{Pitch diameter of the gear, } D_g = \left(\frac{T_g}{T_w}\right) D_w \ \tan \lambda = \left(\frac{T_g}{T_w}\right) D_w \left(\frac{p}{\pi D_w}\right) = \left(\frac{T_g}{T_w}\right) \left(\frac{p}{\pi}\right) = \left(\frac{48}{1}\right) \left(\frac{25}{\pi}\right) = 381.97 \ \text{mm}$$

Center Distance,
$$C = \frac{D_w + D_g}{2} = \frac{100 + 381.97}{2} = 241 \text{ mm}$$

45. For a bolted connection, specification suggests that a high grade material of 13 mm bolt be tightened to an initial tension of 55 000 N. What is the appropriate tightening torque?

a) 134 N-m

b) 143 N-m

- c) 341 N-m
- d) 431 N-m

Solution: $T = CDF_i = (0.20)(0.013 \text{ m})(55\,000 \text{ N}) = 143 \text{ N} \cdot \text{m}$

Note: Initial Tension and tightening Torque, T = CDF

Where, T = tightening torque, N-m, kN-m, in-lb

D = bold nominal diameter, inch, mm

F_i = initial tension, N, kN, lb C = torque coefficient

C = 0.20 (for as received bolts)

C = 0.15 (for lubricated bolts)

46. A 74.6 kW engine of a hoist is capable of lifting 44 500 N of load at a height of 6.10 m in 10 seconds. What is the efficiency of the machine? d) 93.36 % a) 36.39 % c) 63.39 %

Solution:
$$W = F \cdot V = (44\,500 \text{ N}) \left(\frac{6.10 \text{ m}}{10 \text{ sec onds}} \right) = 27\,145 \text{ W} = 27.145 \text{ kW}$$

$$\eta = \frac{27.145}{74.6} (100 \%) = 36.39 \%$$

47. Calculate the peripheral speed, in fpm, of a short shaft held with 16 inches pulley and runs at 450 rpm.

Solution:
$$V_m = \pi D n = (\pi) \left(\frac{16}{12}\right) (450) = 1884.96 \text{ fpm}$$

48 An 18-inch high spring deflects 6 inches when strike by 100lb load. Find the average force acting on the spring. b) 600 lb

Solution:
$$W(h + \delta) = \left(\frac{1}{2}\right) F \delta$$

Solution:
$$W(h+\delta) = \left(\frac{1}{2}\right)F\delta$$
 $F = \frac{2W(h+\delta)}{\delta} = \frac{2(100)(18+6)}{6} = 800 \text{ lb}$

- A carbon steel UNC bolt has a stress area of 0.606 in², determine its permissible working stress.
- b) 4505.49 psi

c) 4550.49 psi

d) 4455.09 psi

Solution:
$$s_w = C(A_r)^{0.418} = (5000)(0.606)^{0.418} = 4055.49 \text{ psi}$$

Note: From Vallance, p138. Bolt permissible stress, $\,s_{\,w}^{}=C\left(A_{\,r}^{}\right)^{\!0.418}$

Applied load,
$$F_a = s_w A_r = C(A_r)^{1.418}$$

Where, C = 5000 for carbon steel bolt

C = 15 000 for alloy-steel bolts

C = 1 000 for bronze bolts

The allowable working stress of a 1.5 inches bolt which is screwed up tightly in packed joint is 13000psi, determine the working strength. a) 11 212.5 lb b) 10 212.5 lb c) 12 212.5 lb

Solution: From Machinery's Handbook,
$$W = s_t \left(0.55 d^2 - 0.25 d\right) = \left(13\,000\right) \left[0.55(1.5)^2 - 0.25(1.5)\right] = 11\,212.5 \ lb$$

Where, W = working strength, lb

st = allowable working stress, psi

d = nominal diameter, inch

51. A shaft is made of a material that weighs 0.28 lb/in³ and a modulus of elasticity of 30 x 10⁶ psi. Bearings, 90 inches apart, support the 2-inch shaft that carries no rotating disc. The shaft has a uniform cross section. Find the second critical speed, in rpm. d) 4712.28 rpm a) 4217.28 rpm b) 4172.28 rpm c) 4228.17 rpm

Solution: Lowest critical speed,
$$N = 4270\,000 \left(\frac{D}{L^2}\right) = \left(4\,270\,000\right) \left[\frac{2}{(90)^2}\right] = 1054.32 \text{ rpm}$$

Second Critical Speed,
$$N_2 = 4(1054.32) = 4217.28 \text{ rpm}$$

Note: The next critical speeds from the lowest critical speed can be found by multiplying by 4, 9, 16, 25, and so on.

52. A 75 lb disc is mounted midway a 1-inch diameter shaft supported by bearings 20 inches apart. Find the lowest critical speed in rpm. Neglect the weight of the shaft. Assume that the modulus of elasticity is 30 000 000 psi.

a) 5763.43 rad/s

b) 301.77 rad/s

- c) 1207.09 rad/s
- d) 401.77 rad/s

Solution: For the disc mounted midway between bearings (neglecting shaft weight),

$$\omega_{\rm cr} = \sqrt{\frac{576\,E\,I\,g}{W\,L^3}} = \sqrt{\frac{576\,\big(30\,000\,000\big)\big(0.0982\big)\big(32.2\big)}{\big(75\big)\big(20\big)^3}} = 301.77\,\,\,{\rm rad}\,/\,{\rm s}$$

Where,
$$I = \frac{\pi}{32} D^4 = \left(\frac{\pi}{32}\right) (1)^4 = 0.0982 \text{ in}^4$$

53. A 4-inch wide and 0.25-inch thick leather belt turns at 3601 fpm. Compute for its centrifugal force.

a) 47 lb

b) 74 lb

d) 75 lb

Solution:
$$F_c = \frac{12 \gamma b t V_s^2}{g} = \frac{12 (0.035)(4)(0.25) \left(\frac{3601}{60}\right)^2}{32.2} = 46.98 \text{ lb}$$

Note: Specific weight of leather belt = 0.035 lb/in³ Specific

Specific weight of rubber belt = 0.045 lb/in³

54. A 2-foot pulley rotating at 600 rpm carries a 1/4-inch thick belt that transmits 30 Hp of power. The allowable belt stress is 500 psi; the angle of contact is 150°. Find the belt width required, assuming the coefficient of friction is 0.4 and the specific weight of the belt material is 0.035 lb/in³.

a) 2.61 inches

b) 3.61 inches

c) 4.61 inches

d) 5.61 inches

Solution:
$$F_t = \left(\frac{2}{D}\right)\!\!\left(\frac{63\,000\,\text{Hp}}{n}\right) = \frac{2\,(63\,000)(30)}{(2)(12)(600)} = 262.5\,\text{ lb} \qquad V_m = \pi\,D\,n = (\pi)(2)(600) = 3769.91\,\text{ fpm}$$

$$F_t = \frac{33\,000\,\text{Hp}}{V_m} = \frac{(33\,000)(30)}{3769.91} = 262..61\,\text{ lb}$$

$$\begin{split} F_t &= F_1 - F_2 = bt \Bigg(s - \frac{12\,\gamma\,V_2^2}{g}\Bigg) \!\! \left(\frac{e^{f\theta} - 1}{e^{f\theta}}\right) & b = \frac{F_t}{t \Bigg(s - \frac{12\,\gamma\,V_s^2}{g}\Bigg) \!\! \left(\frac{e^{f\theta} - 1}{e^{f\theta}}\right)} \\ b &= \frac{262.61}{0.25 \Bigg[500 - \frac{12\,\big(0.035\big) \!\! \left(\frac{3769.91}{60}\right)^2}{32.2}\Bigg] \!\! \left(\frac{e^{0.40\big(150\big) \!\! \left(\pi/180^\circ\right)} - 1}{e^{0.4\big(150\big) \!\! \left(\pi/180^\circ\right)}}\right)} = \frac{262.61}{112.13 \!\! \left(\frac{1.8497}{2.8497}\right)} = 3.608 \; inches \end{split}$$

55. Find the total torque necessary to raise the load of 3000 lb using a triple-thread square power screw. The power screw has a root diameter of 2 inches and 2 threads per inch. It is used in conjunction with a collar with an outer diameter of 4 inches and an inner diameter of 2.5 inches. The coefficient of friction is 0.2 for both threads and collar.

a) 2445 lb

d) 4245 lb

Solution: Pitch, $P_a = \frac{1}{TPI} = \frac{1}{2} = 0.5$ inch / thread

Major diameter,
$$D_o = D_i + 2h = D_i + 2\left(\frac{P_a}{2}\right) = D_i + P_a = 2 + 0.5 = 2.5$$
 inches

Mean diameter,
$$D_{m} = \frac{D_{i} + D_{o}}{2} = \frac{2 + 2.5}{2} = 2.25$$
 inches

$$\text{Collar mean radius or friction radius, } \ r_c = \left(\frac{2}{3}\right) \!\! \left(\frac{r_o^3 - r_i^3}{r_o^2 - r_i^2}\right) = \!\! \left(\frac{2}{3}\right) \!\! \left[\frac{(2)^3 - (1.25)^3}{(2)^2 - (1.25)^2}\right] = 1.654 \ \text{inches}$$

Lead Angle,
$$Tan\lambda = \frac{L}{\pi D_m} = \frac{N_t P_a}{\pi D_m} = \frac{(3)(0.5)}{\pi (2.25)} = 0.21221$$
 $\lambda = 11.98^\circ$

$$\text{Total Torque, } T = T_c \, + T_s = f_c \, \, W \, r_c \, + \frac{W \, D_m}{2} \bigg(\frac{Tan\lambda + f}{1 - f \, tan \, \lambda} \bigg)$$

$$T = (0.2)(3000)(1.654) + \left[\frac{3000(2.25)}{2}\right] \left[\frac{0.21221 + 0.2}{1 - 0.2(0.21221)}\right] = 2445.27 \text{ in } \cdot \text{lb}$$

56. Find the efficiency of a triple-thread square power screw raising a load of 3000 lb. The power screw has a root diameter of 2 inches and 2 threads per inch. It is used in conjunction with a collar with an outer diameter of 4 inches and an inner diameter of 2.5 inches. The coefficient of friction is 0.2 for both threads and collar.

a) 25.25 %

b) 39.29 %

c) 29.29 %

d) 24.29 %

$$\text{Solution: } e = \frac{\tan\lambda \left(1 - f \tan\lambda\right) \left(100 \text{ \%}\right)}{\tan\lambda + f + \left(\frac{f_c D_c}{D_m}\right) \left(1 - f \tan\lambda\right)} = \frac{\left(0.21221\right) \left[1 - \left(0.2\right) \left(0.21221\right)\right] \left(100 \text{ \%}\right)}{0.21221 + 0.2 + \left[\frac{0.2 \left(3.308\right)}{2.25}\right] \left[1 - \left(0.2\right) \left(0.21221\right)\right]} = 29.29 \text{ \%}$$

57. The depth of water in a pond is 5 m. What is its depth in fathom?

- a) 2.73 fathoms
- b) 3.72 fathoms
- d) 7.13 fathoms

Solution: depth =
$$\left(5 \text{ m}\right) \left(3.28 \frac{\text{ft}}{\text{m}}\right) \left(\frac{1 \text{ fat hom}}{6 \text{ feet}}\right) = 2.73 \text{ fat hom}$$

58. A container has a capacity of 30 barrels. What is its equivalent volume in m³?

a) 4.77 m³

b) 7.74 m³

- c) 3.77 m³
- d) 5.77 m³

Solution:
$$V = (30 \text{ barrels}) \left(\frac{42 \text{ gallons}}{1 \text{ barrel}} \right) \left(\frac{3.7854 \text{ liters}}{1 \text{ gallon}} \right) \left(\frac{1 \text{ m}^3}{1000 \text{ liters}} \right) = 4.77 \text{ m}^3$$

59. A bridge has a span of 5 furlongs. What is its equivalent length in yards?

- b) 1150 yards

c) 1100 yards

d) 1150 yards

Solution:
$$L = (5 \text{ furlong}) \left(\frac{220 \text{ yards}}{1 \text{ Furlong}} \right) = 1100 \text{ yards}$$

60. Determine the kinetic energy needed in punching of 350 kg flywheel if the speed varies from 200 rpm to 180 rpm with 1 m mean diameter.

a) 3.65 kJ

Solution:
$$V_1 = \frac{\pi D n_1}{60} = \frac{\pi (1 \text{ m})(200)}{60} = 10.47 \text{ m/s}$$

$$V_2 = \frac{\pi Dn_2}{60} = \frac{\pi (1 \text{ m})(180)}{60} = 9.42 \text{ m/s}$$

$$\Delta KE = \frac{m}{2} \left(V_1^2 - V_2^2 \right) = \left(\frac{350 \text{ kg}}{2} \right) \left[(10.47)^2 - (9.42)^2 \right] = 3654.8 \text{ J} = 3.655 \text{ kJ}$$

61. A punch machine uses 10 kJ of energy to punch a square in 20 mm thick plate. If the ultimate strength of the plate is 420 MPa, determine the maximum sides of square that can be punched.

a) 45 mm

c) 60 mm

d) 55 mm

$$\text{Solution: } \Delta KE = \frac{1}{2} \, F \cdot t_p = \frac{1}{2} \Big(4b t_p \Big) t_p s_u = \frac{1}{2} \Big(4b \Big) t_p^2 s_u = 2b \Big(t_p \Big)^2 s_u \\ b = \frac{\Delta KE}{2t_p^2 s_u} = \frac{10\,000\,000\,N \cdot mm}{2 \Big(20\,mm \Big)^2 \Big(420\,N \,/\,mm^2 \Big)} = 29.76\,mm$$

A steel rod of 6 inches long is to resist an impact load of 500 lb dropped through a distance of 2 inches. If the maximum computed stress is to be 20 ksi, determine the required diameter of the rod.

- a) 5.65 inches
- b) 6.55 inches

c) 4.65 inches

d) 6.45 inches

Solution: Maximum deflection due to the maximum stress, $\delta = \frac{s\,L}{E} = \frac{\left(20\,000\,\text{ psi}\right)\left(6\,\text{inches}\right)}{30\,\text{ x }10^6\,\text{ psi}} = 0.004\,\text{inch}$

Solving for cross sectional area,
$$s = \frac{2 \ W}{A} \left(\frac{h}{\delta} + 1\right) \\ A = \frac{2 \ W}{s} \left(\frac{h}{\delta} + 1\right) = \left[\frac{2 \left(500 \ lb\right)}{20 \ 000 \ lb / in^2}\right] \left(\frac{2}{0.004} + 1\right) = 25.05 \ in^2$$

Solving for the Diameter, $D = \sqrt{\frac{4 \text{ A}}{\pi}} = \sqrt{\frac{4(25.05)}{\pi}} = 5.65$ inches

63. A punch machine uses 10 kJ of energy to punch a square in 20 mm thick plate. If the ultimate strength of the plate is 420 MPa, determine the maximum sides of square that can be punched if a factor of safety of 4 is to be applied.

a) 25 mm

b) 35 mm
c) 30 mm
d) 20 mm

Solution: Punching Force,
$$F = \frac{2 \left(\Delta KE \right)}{t_p} = \frac{2 \left(10\,000\ N \cdot m \right)}{0.02\,m} = 1\,000\,000\ N = 1\,000\ kN$$

$$b = \frac{F}{4s_u t_p} = \frac{1000\ 000\ N}{4(420\ N/mm^2)(20\ mm)} = 29.76\ mm \approx 30\ mm$$

64. How many turns a flywheel makes before it stops from a speed of 220 rpm? The flywheel stops in 15 seconds.

a) 27.5 turns
b) 172.8 turns
c) 25.7 turns
d) 52.7 turns

Solution: Acceleration,
$$\alpha = \frac{\omega_f - \omega_o}{t} = \frac{0 - 23.04}{15} = -1.536 \text{ rad/s}^2$$
 $\omega_o = \frac{2 \pi n}{60} = \frac{2(\pi)(220)}{60} = 23.04 \text{ rad/s}$ Angular displacement, $\theta = \omega_o t + \frac{\alpha}{2} t^2 = (23.04)(15) + \left(\frac{-1.536}{2}\right)(15)^2 = 172.8 \text{ radians} = 27.5 \text{ revolutions}$

- 65. Determine the cutting speed, in fpm, of a workpiece with 2 inches diameter and running at 100 rpm. a) 56.23 fpm b) 52.36 fpm c) 36.52 fpm d) 23.56 fpm Solution: $V = \pi D n = (\pi) \left(\frac{2}{12}\right) (100) = 52.36 \text{ fpm}$
- 66. A 40 kg load falls through a height of 0.20 m before striking the free end of a cantilever beam. The beam is 60 mm wide and 100 mm deep. Compute the maximum bending stress and deflection caused by the impact. Neglect the mass of the beam and assume that the 40 kg mass remains in contact with the beam.

 a) 162 MPa

 b) 261 MPa

 c) 126 MPa

 d) 621 MPa

Solution: Static stress,

$$s_{st} = \frac{Mc}{I} = \frac{WLc}{I} = \frac{mgL\left(\frac{h}{2}\right)}{\frac{bh^{3}}{12}} = \frac{6 mgL}{bh^{2}} = \frac{6 (40 kg)(9.8066 m/s^{2})(2 m)}{(0.06 m)(0.10 m)} = 7.845280 N/m^{2} = 7.85 MPa$$

Moment of Inertia,
$$I = \frac{b h^3}{12} = \frac{(0.06)(0.10)^3}{12} = 5 \times 10^{-6} \text{ m}^4$$

$$\text{Deflection of Cantilever, } \delta = \frac{W \ L^3}{3 \ E \ I} = \frac{\left(40\right)\!\left(9.806\right)\!\left(0.10\right)^3}{3\left(20\,000\,000\,000\,N\,/\,m^2\right)\!\left(5\ x\ 10^{-6}\ m^4\right)} = 1.046\ x\ 10^{-3}\ m = 1.046\ mm^2 \right)$$

$$s_{\text{max}} = 7.85 + 7.85 \left[1 + \frac{2(0.20)}{1.046 \text{ x } 10^{-3}} \right]^{0.5} = 161.56 \text{ MPa} \approx 162 \text{ MPa}$$

67. A double thread worm gear has a pitch of 1.125 inches and a pitch diameter of 3 inches. It has a coefficient of friction of 0.20 and normal pressure angle of 14.5°. The worm is supplied by 12 Hp at 1200 rpm motor. Find the tangential force on the gear. The worm is a left hand thread.

a) 420 lb

b) 897 lb

c) 798 lb

b) 879 lb

$$\text{Solution: } \lambda = tan^{-l} \Biggl(\frac{L}{\pi \, D_w} \Biggr) = tan^{-l} \Biggl(\frac{N_t \, P_a}{\pi \, D_w} \Biggr) = tan^{-l} \Biggl[\frac{2 \left(1.125 \right)}{\pi \left(3 \right)} \Biggr] = 13.427^{\circ}$$

Tangential load of worm,
$$F_{tw} = \left(\frac{2}{D_w}\right) \left(\frac{63\,000\,Hp}{n_w}\right) = \left(\frac{2}{3}\right) \left[\frac{63\,000\,(12)}{1200}\right] = 420\,lb$$

$$\text{Tangential load on gear, } F_{tg} = F_{tw} \Biggl(\frac{\cos \phi_n \, \cos \lambda - f \, \sin \lambda}{\cos \phi_n \, \sin \lambda + f \, \cos \lambda} \Biggr) = \Bigl(420 \Biggl) \Biggl(\frac{\cos 14.5^{\circ} \, \cos 13.427^{\circ} - 0.20 \sin 13.427^{\circ}}{\cos 14.5^{\circ} \, \sin 13.427^{\circ} + 0.20 \cos 13.427^{\circ}} \Biggr) = 896.65 \, \, \text{lb}$$

68. A double thread worm has a lead angle of 7.25° and pitch radius of 2.5 inches. Determine the pitch of the worm. a) 1.5 inch d) 0.75 inch

Solution: $L = \pi D_w \tan \lambda = 2\pi R \tan \lambda = 2(\pi)(2.5) \tan 7.25^\circ = 1.9983$ inches

 $P_a = \frac{L}{N_*} = \frac{1.9983}{2} = 0.9992 \text{ inch } \approx 1 \text{ inch}$

6 cm and 12 cm diameter pulleys are used to transmit power. Center distance of the pulleys is 72 cm. If an open belt arrangement is used, calculate the required belt length.

c) 172.4 cm

d) 147.2 cm

$$\text{Solution: } L = \left(\frac{\pi}{2}\right)\!\!\left(D_1 + D_2\right) + 2C + \frac{\left(D_2 - D_1\right)^2}{4C} = \left(\frac{\pi}{4}\right)\!\!\left(6 + 12\right) + 2\left(72\right) + \frac{\left(12 - 6\right)^2}{4\left(72\right)} = 172.4 \text{ cm}$$

70. Determine the depth of the ACME thread if the pitch is 1.25 inches.

b) 0.635 inch

c) 1.25 inches

d) 0.750 inch

Solution: $h = 0.5 P_a + 0.01 = (0.5)(1.25) + 0.01 = 0.635$ inch

71. A chain and sprocket has 18 teeth with chain pitch of 0.5 inch. Find the pitch diameter of the sprocket.

a) 2.88 inches

b) 2.66 inches

c) 2.55 inches

d) 2.77 inches

Solution:
$$D = \frac{P_a}{\sin\left(\frac{180^{\circ}}{T}\right)} = \frac{0.50}{\sin\left(\frac{180^{\circ}}{18}\right)} = 2.879$$
 inches

72. The ratio in band tension in a brake drum is 3. The difference between the tight side and slack side is 5 kN. Find the tension in the tight side. a) 2.75 kN b) 3.75 kN c) 4.75 kN d) 1.75 kN

Solution: $F_1 = 3 F_2$ $F_1 = 5 - F_2 = 5 - \frac{F_1}{2}$ $F_1 = 3.75 \text{ kN}$

73. What will be the resulting critical speed of a shaft if you double its diameter? The original critical speed of the shaft is 800 rpm. a) 1600 rpm

b) 3600 rpm

d) 900 rpm

Solution: $n_{cr} = 4\,270\,000 \left(\frac{D}{L^2}\right)$ $\frac{D}{L^2} = \frac{n}{4\,270\,000} = \frac{800}{4\,270\,000}$

$$\frac{D}{L^2} = \frac{n}{4.270000} = \frac{800}{4.270000}$$

$$n_{\rm cr} = (4270\,000)(2)\left(\frac{800}{4270000}\right) = 1600\,{\rm rpm}$$

74. A 112.5 mm shaft is keyed to a pulley. The pulley is made of a cast iron material and has a diameter of 1200 mm. The key is 28.13 mm square key and 175 mm long. The key and shaft have a shearing stress of 14 000 psi. Determine the force acting at the pulley that will shear the key. a) 4541.83 kg b) 4252.83 kg c) 5441.83 kg d) 4145.83 kg

$$\text{Solution: } F_p = \frac{2T}{D_p} = \frac{2 \left(\frac{s_s b L D}{2}\right)}{D_p} = s_s b L \left(\frac{D}{D_p}\right) = \left(14000 \left(\frac{28.13}{25.4}\right) \left(\frac{175}{25.4}\right) \left(\frac{112.5}{1200}\right) = 10014.74 \text{ lb}$$

A flanged coupling is used in a 75-mm shaft that transmits 300 kW of power at 600 rpm. The coupling has 6 bolts, each 18 mm in diameter. Find the required diameter of the bolts circle based on an average shearing stress of 27.5 MPa.

Solution:
$$D_B = \frac{8T}{\pi s_s d^2 n_B} = \frac{8(30)P}{\pi^2 d^2 s_s n n_B} = \frac{8(30)(300)}{\pi^2 (0.018)^2 (27500)(600)(6)} = 0.2274 \text{ m} = 227.4 \text{ mm}$$

A bolt is screwed up tightly in a packed joint. The allowable working stress is 12,000 psi and size of the bolt is 2 inches. Determine its working strength.

a) 20 120 lb

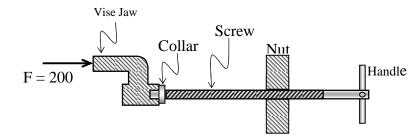
b) 20 100 lb

c) 20 400 lb

d) 20 200 lb

Solution: From Machinery's handbook: $W = s_+(0.55d^2 - 0.25d) = (12\,000)[0.55(2)^2 - 0.25(2)] = 20\,400$ lb

77. A 1-inch single square thread, with 4 threads per inch, is used in a vise mechanism. The frictional radius of the collar is 0.5 inch. The coefficient of friction for both the collar and threads is 0.20. How much external torque must be applied to produce a force of 200 lb against the jaws of the vise? a) 39.73 in-lb b) 33.97 in-lb c) 37.93 in-lb d) 39.37 in-lb



Solution:

For the thread pitch,

$$p = \frac{1}{\text{Number of Threads per Inch}} = \frac{1}{4} = 0.25 \text{ inch}$$

For the minor diameter, $D_i = D_o - p = 1 - 0.25 = 0.75$ inch

For the mean or pitch diameter of the screw, $D_m = \frac{D_o + D_i}{2} = \frac{1 + 0.75}{2} = 0.875$ inch

For the lead angle,
$$\lambda = tan^{-1} \left(\frac{L}{\pi D_m} \right) = tan^{-1} \left(\frac{p}{\pi D_m} \right) = tan^{-1} \left(\frac{0.25}{0.875\pi} \right) = tan^{-1} 0.09095 = 5.197^\circ$$

Solving for the torque required to overcome the collar friction, $T_c = \frac{fFD_c}{2} = \frac{(0.2)(200)(0.5)}{2} = 20 \text{ in} \cdot \text{lb}$

Solving for the torque required to overcome the thread friction,

$$T_{s} = \frac{FD_{m}}{2} \left(\frac{\tan \lambda + f}{1 - f \tan \lambda} \right) = \frac{200(0.875)}{2} \left[\frac{0.09095 + 0.2}{1 - 0.2(0.9095)} \right] = 25.93 \text{ in} \cdot \text{lb}$$

Solving for the total torque, $T = T_s + T_c = 25.93 + 20 = 45.93$ inches

ans

78. A hollow shaft with inside diameter of 1 inch is to replace a 1.5-inch diameter solid shaft in a power transmission. Considering equal torsional strengths, determine the required outside diameter of the hollow shaft. c) 1.75 in. a) 2.25 in. d) 2.0 in.

Solution:
$$Ss_{solid} = \frac{16T}{\pi D^3}$$

$$Ss_{hollow} = \frac{16TD_o}{\pi (D_o^4 - D_1^4)}$$

Thus, for equal strength, $D_o^4 - 1 = D_o(1.5)^3$: $D_o = 1.589$ in. Use the next higher standard size, $D_o = 1.75$ in.

79. A square key is used in a 4-inch diameter shaft transmitting 1000 hp at 1000 rpm. If the allowable shear and compressive stresses in the key are 15 ksi and 30 ksi, respectively, what length of key is required?

a) 53.34 mm

- b) 54.33 mm
- c) 50.34 mm
- d) 55.34 mm

Solution: Transmitted torque,
$$T = \frac{63000 \, Hp}{n} = \frac{63000(1000)}{1000} = 63000 \, in \cdot lb$$

Key width, $b \approx \frac{D}{4}$, for good proportion

Key length based on shear,
$$L = \frac{2T}{s_s bD} = \frac{2(63000)}{15000(1)(4)} = 2.1$$
 inches

Key length based on compression,
$$L = \frac{4T}{s_c t D} = \frac{4(63000)}{30000(1)(4)} = 2.1$$

Therefore, use L = 2.1 inches

- 80. A steel shaft, with a modulus of rigidity of 12 x 10⁶ psi, transmits 40 Hp at 1400 rpm. It is 1.4375 inches in diameter. Find the torsional deflection of the shaft in degrees per foot length.
 - a) 0.246

- b) 0.624
- c) 0.426

d) 1.246

Solution:
$$\frac{\theta}{L} = \frac{T}{JG} = \frac{32 \, T}{\pi \, D^4 G} = \frac{32 \left(63\,000 \, Hp\right)}{\pi \, n \, D^4 \, G} = \frac{32 \left(63\,000\right) \left(40\right) \left(12\right)}{\pi \left(1400\right) \left(1.4375\right)^4 \left(12 \, x \, 10^6\right)} \left(\frac{180^\circ}{\pi}\right) = 0.246 \text{ radian}$$

- 81. What torque is needed by a flywheel to produce an angular acceleration of 100 revolutions per minute, per second? The flywheel is solid disk and has a diameter of 1.5 m and a mass of 800 kg.
 - a) 3265 N-m
- b) 5236 N-m
- c) **2356 N-m**
- d) 6325 N-m

Solution: Torque = Mass Moment of Inertia x Angular Acceleration = $I\alpha$

$$\label{eq:Where, I} \text{Where, I} = \frac{1}{2}mr^2, \text{for cylinders;} \qquad \qquad \text{I} = \frac{1}{2}(800\text{kg})(1.5/2)^2\,\text{m}^2 = 225\text{kg}\cdot\text{m}^2$$
 Then, $T = (225)\text{kg} - \text{m}^2\bigg[\frac{100(2\pi)}{60}\bigg]\frac{\text{rad}}{\text{s}^2} = 2356.2\,\,\text{N}\cdot\text{m}$

- 82. A brass plate with thickness of ¼ inches is to punch a 2 inches diameter hole. Find the pressure required.
 - a) 40 tons

b) 32.5 tons

- c) 20.5 ton
- d) 35 tons

Solution:
$$F = 65 dt = (65)(2)(0.25) = 32.5 tons$$

[For steel plate use 80 as constant] (From Machinery's Hand Book, 26th Ed.)

- 83. A bolt is screwed up tightly in a packed joint. The allowable working stress is 12,000 psi and size of the bolt is 1.5 inches. Determine its working strength
 - a) 12 121.5 lb
- b) 11 212.5 lb

- c) 21 211.5 lb
- d) 12 211.5 lb

Solution:
$$W = S_t (0.55 D^2 - 0.25D) = (13000)[(0.55)(1.5)^2 - (0.25)(1.5)] = 11212.5 \text{ lb}$$

[From Machinery's Hand Book, 26th Ed., p1489]

84. A collar has an outside diameter of 100 mm and an inside diameter of 40 mm. The collar rotates at 1000 rpm and the coefficient of friction between the collar and the pivot surface is 0.15. Determine the frictional Power acting on a collar loaded with 100 kg weight.

a) 572 W

b) 527 W

c) 752 W

d) 275 W

Solution:
$$r_f = \left(\frac{2}{3}\right) \left(\frac{r_o^3 - r_i^3}{r_c^3 - r_i^2}\right) = \left(\frac{2}{3}\right) \left[\frac{(50)^3 - (20)^3}{(50)^2 - (20)^2}\right] = 37.14 \text{ mm} = 0.03714 \text{ m}$$

$$T_f = f W r_f = (0.15)(100 kg)(9.8066 N/kg)(0.03714 m) = 5.464 N \cdot m$$

$$P_f = \frac{\pi n T_f}{30} = \frac{\pi (1000)(5.464)}{30} = 572.2 \text{ W} = 0.572 \text{ kW}$$

85.	Mechanical power at a rate of 37 kW running at 1760 rpm is to transmit using a solid shaft 48.2 cm long. The torsional stress is 8.13 MPa. What is
	the diameter of the shaft?

a) 45 mm

b) 50 mm

- c) 55 mm
- d) 30 mm

Solution:
$$T = \frac{30 P}{\pi n} = \frac{(30)(37)}{\pi (1760)} = 0.20075 \text{ kN} \cdot \text{m} = 200.75 \text{ N} \cdot \text{m}$$

$$D = \left(\frac{16 \text{ T}}{\pi \text{ S}_{\text{s}}}\right)^{\frac{1}{3}} = \left[\frac{16 \left(200750 \text{ N} \cdot \text{mm}\right)}{\pi \left(8.13 \text{ N}/\text{mm}^2\right)}\right]^{\frac{1}{3}} = 50.10 \text{ mm}$$

Solution:
$$D_2 = 2C + D_1 = 2(457.2) + 127 = 1041.4 \text{ mm}$$

- b) 3/8 inch

- d) 1/4 inch

$$\text{Solution: } t = \frac{pD}{2\,S_t} = \frac{\left(120\ psi\right)\!\left(30\ inches\right)}{2\left(8000\ psi\right)} = 0.225\ inch \approx 1/4\ inch$$

- 88. A 36-tooth gear drives a 50.8-mm solid shaft that transmits power at 120 rpm. If the allowable shearing stress is 83 MPa, determine the power transmitted, in Hp.
 - a) 36 Hp

b) 28 Hp

- c) 42 Hp
- d) 38 Hp

Solution:
$$T = \frac{\pi D^3 s_s}{16} = \frac{\pi \left(50.8 \text{ mm}\right)^3 \left(83 \text{ N} / \text{mm}^2\right)}{16} = 2136461.415 \text{ N} \cdot \text{mm}$$

$$P = \frac{\pi n T}{30} = \frac{\pi (120)(2.14)}{30} = 26.89 \text{ kW} = 36.05 \text{ Hp}$$

- 89. A helical spring has an outside diameter of 10.42 cm and a wire diameter of 0.625 cm. The spring has squared and ground ends and a total of 18 coils. Its material has modulus of elasticity in shear of 78.91 GPa. Determine the deflection in the spring due to a load of 50 kg.
 - a) 490 mm

b) 409 mm

- c) 390 mm

Solution:

$$\delta = \frac{8F(D_{w})^{3} N_{a}}{G(d_{w})^{4}} = \frac{8FC^{3} N_{a}}{Gd_{w}}$$

Where, for S & G ends:
$$N_a = N - 2 = 18 - 2 = 16$$
 coils

$$C = \frac{D_{m}}{d_{w}} = \frac{D_{o} - d_{w}}{d_{w}} = \frac{10.42 - 0.625}{0.625} = 15.675$$

$$C = \frac{D_{m}}{d_{w}} = \frac{D_{o} - d_{w}}{d_{w}} = \frac{10.42 - 0.625}{0.625} = 15.672$$

$$\delta = \frac{8 (50 \text{ kg}) (9.8066 \text{ N/kg}) (15.672)^{3} (16)}{(78910 \text{ N/mm}^{2}) (6.25 \text{ mm})} = 489.85 \text{ mm}$$

- c) 2 minutes
- d) 2.5 minutes

Solution:
$$t = \frac{Length}{Rate \ of \ Welding} = \frac{\left(3 \ ft\right)\left(12 \ inches / ft\right)}{18 \ in / min} = 2 \ min \ utes$$

- a) 10 seconds
- b) 8 seconds

- c) 12 seconds
- d) 15 seconds

Solution:
$$t = \frac{\text{Length}}{\left(\text{No. of Teeth / rev}\right)\left(\text{Feed, inch / tooth}\right)\left(\text{rpm}\right)} = \frac{2 \text{ inches}}{\left(24 \text{ teeth / rev}\right)\left(0.005 \text{ in / tooth}\right)\left(100 \text{ rpm}\right)} = 0.1667 \text{ min}$$

- 92. A 14½° spur gear has a pitch of 4. Calculate its tooth thickness.
 - a) 0.3927 inch
- b) 9.975 mm

- c) 0.9975 cm
- d) Any of these

Solution:
$$t = \frac{1.5708}{P_d} = \frac{1.5708}{4} = 0.3927$$
 inch = 9.975 mm = 0.9975 cm

A pump uses a journal bearing with diameter of 76.2 mm is subjected to a load of 4.9 kN while rotating at 200 rpm. If its coefficient of friction is 0.02 and L/D = 2.5, find its projected area, in mm²

a) 41 516.1 mm²

- b) 14 516.1 mm²
- c) 15 416.1 mm²
- d) 16 416.1 mm²

Solution: L = 2.5 D = (2.5)(76.2) = 190.5 mm

Projected Area,
$$A = D(L) = (76.2 \text{ mm})(190.2 \text{ mm}) = 14516.1 \text{ mm}^2$$

Determine the diameter of a line shaft transmitting 12 Hp at 180 rpm with torsional deflection of 0.08 degree per foot length. a) 2.75 inches b) 2.34 inches c) 2.25 inches d) 3.34 inches

Solution:
$$D = 4.6 \left(\frac{P}{N}\right)^{0.25} = \left(4.6\right) \left(\frac{12}{180}\right)^{0.25} = 2.34$$
 inches

Where, P = power transmitted, Hp

N = n = rpm of the shaft

Note: This equation is for torsional deflection of 0.08 degree per ft length and using G = 12 x 10⁶ psi.

Derived from the equations:
$$\frac{\theta}{L} = \frac{32 \, T}{\pi D^3 \, G}$$
 and $Hp = \frac{T \, n}{63 \, 000}$

Calculate the maximum unit shear in a 3-inch diameter steel shaft transmitting 2400 in-lb of torque at 180 rpm.

- b) 543 psi
- c) 435 psi

Solution:
$$s_s = \frac{16 \text{ T}}{\pi D^3} = \frac{(16)(2400)}{\pi (3)^3} = 452.71 \text{ psi}$$

A 1.5-inch diameter shaft has a driver with variable torque. Shaft has a yield strength of 60000 psi. If torque varies from 2000 to 6000 in-lb, determine the shaft mean or average stress.

a) 6 036.10 psi

- b) 6306.10 psi
- c) 3 018.05 psi

d) 9 054.15 psi

Solution:
$$s_{min} = \frac{16T_{min}}{\pi D^3} = \frac{16(2000)}{\pi (1.5)^3} = 3018.05 \text{ psi}$$

$$s_{\text{max}} = \frac{16T_{\text{max}}}{\pi D^3} = \frac{16(6000)}{\pi (1.5)^3} = 9054.15 \text{ psi}$$

$$s_{\rm m} = \frac{s_{\rm min} + s_{\rm max}}{2} = \frac{3018.05 + 9054.15}{2} = 6036.10 \text{ psi}$$

97. In an open belt drive, the driving pulley is 450 mm in diameter and the driven pulley is 1000 mm in diameter. The belt is 300 mm wide and 10 mm thick. The coefficient of friction of the belt drive is 0.30 and the mass of the belt is 2.8 kg/m of belt length. Other data are as follows: center distance between shafts, 4 m; maximum allowable tensile stress on the belt, 1500 kPa; and speed of driving pulley, 900 rpm. Calculate the maximum power that can be transmitted, in kW.

a) 40.82 kW

b) 42.80 kW

- c) 48.20 kW
- d) 28.40 kW

$$\text{Solution: } P = \frac{\pi\,n_1\,T_1}{30} = \left(\frac{\pi\,n}{30}\right)\!\left(F_t\!\left(\frac{D_1}{2}\right)\right) \qquad \qquad \frac{F_1-F_c}{F_2-F_c} = e^{f\theta}$$

$$\frac{F_1 - F_c}{F_2 - F_c} = e^{f\theta}$$

$$F_{t} = F_{1} - F_{2} = (F_{1} - F_{c}) \left(\frac{e^{f \theta} - 1}{e^{f \theta}} \right)$$

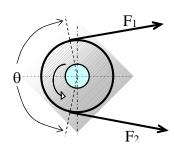
Belt speed,
$$V = \frac{\pi D_1 n_1}{60} = \frac{\pi (0.450)(900)}{60} = 21.21 \text{ m/s}$$

Wrap angle,
$$\theta = \theta_1 = \pi - \left(\frac{D_2 - D_1}{C}\right) = \pi - \left(\frac{1000 - 450}{4000}\right) = 3.0041 \text{ radians}$$

Belt tension ratio,
$$\frac{F_1 - F_c}{F_2 - F_c} = e^{f\theta} = e^{\left(0.30\right)\left(3.0041\right)} = 2.463$$

Centrifugal Force,
$$F_c = \rho \, bt \, V^2 = \left(\frac{2.8 \, kg \, / \, m}{b \, t}\right) \! \left(b \, t\right) \! \left(21.21 \, m \, / \, s\right)^2 = 1259.62 \, N$$

Maximum belt tension, $F_1 = s_t A = s_t b t = (1500000 \text{ N/m}^2)(0.30 \text{ m})(0.01 \text{ m}) = 4500 \text{ N}$



Belt Net Tension,
$$F_t = \left(F_1 - F_c\right) \left(\frac{e^{f\theta} - 1}{e^{f\theta}}\right) = \left(4500 - 1259.62\right) \left(\frac{1.463}{2.463}\right) = 1924.76 \ N$$
 Power to be transmitted,
$$P = \left(\frac{\pi \, n_1}{30}\right) F_t \left(\frac{D_1}{2}\right) = \left(\frac{\pi \left(900\right)}{30}\right) \left(1924.76\right) \left(\frac{0.45}{2}\right) = 40815.98 \ W = 40.82 \ kW$$

- There are three parallel shafts A, B, and C. Shaft A has 24 teeth gear of P_d = 4 meshing with a larger gear on shaft B having 70 teeth. A smaller gear, 20 teeth and pitch of 3, on shaft B meshes with 50 teeth gear on shaft C. Find the torque on shaft C if the input Hp to shaft A is 50 Hp turning at 1200 rpm, and the efficiency of each gear combination is 98 %.
 - a) 1532 ft-lb
- c) 1253 ft-lb

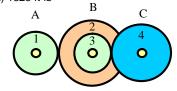
d) 1325 ft-lb

Solution: $T_1 = 24$ teeth $T_2 = 70$ teeth

h
$$T_3 = 20$$
 teeth

$$T_4 = 50$$
 teeth

$$\frac{n_1}{n_4} = \frac{T_2 \times T_4}{T_1 \times T_3}$$
 $n_4 = n_1 \left(\frac{T_1}{T_2}\right) \left(\frac{T_3}{T_4}\right) = (1200) \left(\frac{24}{70}\right) \left(\frac{20}{50}\right) = 164.57 \text{ rpm}$



$$Hp_o = Hp(e) = (50)(0.98)(0.98) = 48.02$$

$$Hp_o = Hp(e) = (50)(0.98)(0.98) = 48.02 T_C = \frac{63000Hp_o}{n_C} = \frac{(63000)(48.02)}{164.57} = 18382.82 \text{ in} \cdot \text{lb} = 1531.90 \text{ ft} \cdot \text{lb}$$

99. If a 450 kg flywheel rotates from 180 rpm to 168 rpm, how much energy, in N-m, it loss? Radius of the flywheel is 375 mm. b) 1541.33 c) 1451.33

Solution:
$$V_1 = 2\pi R N_1 / 60 = 2\pi (0.375) \left(\frac{180}{60}\right) = 7.069 \text{ m/s}$$
 $V_2 = 2\pi R N_2 / 60 = 2\pi (0.375) \left(\frac{168}{60}\right) = 6.597 \text{ m/s}$
$$\Delta KE = \frac{m \left(V_1^2 - V_2^2\right)}{2} = \frac{450 \left[(7.069)^2 - (6.597)^2 \right]}{2} = 1451.33 \text{ N} \cdot \text{m}$$

- 100. A 3/4 inch thick steel plate is to punch for a 1-in diameter hole every 10 sec. The actual punching takes 1 sec. The ultimate shear strength of the plate is 60,000 psi. The flywheel of the punch press has a mass moment of inertia of 500 in-lb-sec2 and rotates at a mean speed of 150 rpm. What is the speed fluctuation, in rpm?
 - a) 65.4 rpm

- b) 64.5 rpm
- c) 6.749 rpm
- d) 67.49 rpm

$$\text{Solution: } \Delta KE = \left(\frac{1}{2}\right) I\left(\omega_1^2 - \omega_1^2\right) = \left(\frac{1}{2}\right) I\left(\omega_1 - \omega_2\right) \left(\omega_1 + \omega_2\right) = I\left(\omega_1 - \omega_2\right) \left(\frac{\omega_1 + \omega_2}{2}\right) = I\left(\omega_1 - \omega\right) \omega = I\left(\omega_1 - \omega_2\right) \left(\frac{\omega_1 + \omega_2}{2}\right) = I\left(\omega_1 - \omega\right) \omega = I\left(\omega_1 - \omega_2\right) \left(\frac{\omega_1 + \omega_2}{2}\right) = I\left(\omega_1 - \omega_2\right) \left(\frac{\omega_1 + \omega$$

Mean angular velocity,
$$\omega = \frac{2\pi N}{60} = \frac{2\pi \left(150\right)}{60} = 15.71 \, rad \, / \, s$$

Punching Force,
$$F = s_u (\pi dt) = (60\,000)(\pi)(1)(0.75) = 141\,371.67$$
 lb

Kinetic Energy,
$$\Delta KE = \frac{1}{2}(F)t = (\frac{1}{2})(141\ 371.67)(\frac{0.75}{12}) = 4417.86\ ft \cdot lb$$

Speed Fluctuation,
$$(\omega_1 - \omega_2) = \frac{\Delta KE}{I \omega} = \frac{(4417.86 \text{ ft} \cdot \text{lb})(12)}{(500 \text{ in} \cdot \text{lb} \cdot \text{s}^2)(15.71)} = 6.749 \text{ rad/s} = 64.5 \text{ rpm}$$

- 101. A shaft, 1.5 inches in diameter, was loaded with a torque of 8,000 in-lb and a bending moment of 12,000 in-lb, determine the maximum shear stress induced on the shaft.
 - a) 21 763.5 psi

- b) 27 163.5 psi
- c) 23 761.5 psi

d) 26 173.5 psi

Solution:

$$s_{s max} = \frac{16}{\pi D^3} \sqrt{M^2 + T^2} = \frac{16}{\pi (1.5)^3} \sqrt{(12000)^2 + (8000)^2} = 21763.5 \text{ psi}$$

- 102. A solid coupling is used in a 75 mm shaft that transmits 300 kW power at 600 rpm. The coupling has 6 bolts and each 18 mm in diameter. Find the required bolt circle diameter if the bolt shearing stress is 27.5 MPa.
 - a) 227.4 mm
- b) 233.6 mm
- c) 254.5 mm

d) 272.6 mm

Solution:
$$T = \frac{30 \, P}{\pi \, n} = \frac{(30)(300)}{\pi (600)} = 4.775 \, kN \cdot m$$

$$D_{B} = \frac{8T}{\pi d^{2} s_{s} n_{B}} = \frac{8(4775000 \text{ N} \cdot \text{mm})}{\pi (18 \text{ mm})^{2} (27.5 \text{ N} / \text{mm}^{2})(6)} = 227.4 \text{ mm}$$

103. A machinist is to cut a steel material with the following choices: both 1.5 inches diameter; high speed steel with recommended cutting speed of 90 fpm; carbide tipped with cutting speed of 300 fpm. Compute the rpm of the cutter.

d) 746

Solution:
$$n = \frac{V}{\pi D} = \frac{300 \text{ fpm}}{\pi \left(\frac{1.5}{12} \text{ ft}\right)} = 763.94 \text{ rpm}$$

104. A centrifugal pump discharges 3000 liters per minute at a head of 10 meters. The pump is driven by a motor and pump efficiency is 68%. The pump rotates at 550 rpm. Find the torsional stress of shaft if shaft diameter is 35mm.

A. 13.85 MPa

D. 14.87 MPa

$$P = \frac{Q \cdot \gamma \cdot TDH}{e} = \frac{\left(\frac{3000}{60 \times 1000}\right) m^3 / s \times 9.81 kN / m^3 \times 10 m}{0.68} = 7.213 kW = 7.213 \times 10^6 N \cdot mm / s$$

$$S_{s} = \frac{16 \cdot T}{\pi \cdot D^{3}} = \frac{16 \times \left(\frac{30 \cdot P}{\pi \cdot N}\right)}{\pi \cdot D^{3}} = \frac{16 \times \left(\frac{30 \times \left(7.213 \times 10^{6}\right)}{\pi \times 550}\right)}{\pi \times 35^{3}} = 14.87 MPa$$

105. A motor with gear ratio of 1.8 is used to drive a 800 mm diameter circular saw blade. The motor rotates at 1800 rpm. Compute for the peripheral speed of the blade.

A. 137.39 ft/sec

- B. 140.65 ft/sec
- C. 132.43 ft/sec
- D. 135.21 ft/sec

Solution

$$SR = \frac{N_{motor}}{N_{sawblade}} \quad ; N_{sawblade} = \frac{1800}{1.8} = 1000 rpm$$

$$V = \pi \cdot D \cdot N = \pi \times \frac{800}{1000} m \times \frac{1000}{60 sec} \times 3.28 ft/m = 137.39 ft/sec$$

106. A mechanical engineer wants a quick calculation on the shaft he will use. For a shaft length of 10 feet, find the diameter of the shaft that could safely deliver.

A. 1.18 in

- B. 7.55 in
- C. 2.34 in
- D. 1.64

Solution:
$$L = 8.95 \cdot \sqrt[3]{D^2}$$

$$D = \sqrt{\left(\frac{10}{8.95}\right)^3} = 1.18in$$

107. Two shafts, made of the same materials, are circular in construction. One is hollow and the other is solid. The hollow shaft inside diameter is onehalf of the external diameter. The external diameter is equal to the diameter of the solid shaft. What is the ratio of the twisting moment of the hollow shaft to that of the solid shaft?

A.1/4

B. 1/3

- C. 9/16
- D. 15/16

$$\text{Solution: } \theta_S \cdot D_S^4 = \theta_H \cdot \left(D_o^4 - D_i^4 \right) = \frac{32 \cdot T \cdot L}{\pi \cdot G}$$

$$\theta_{S} \times D_{o}^{4} = \theta_{H} \times \left[D_{o}^{4} - \left(\frac{D_{o}}{2} \right)^{4} \right]$$

$$\theta_{S} \times D_{o}^{4} = \theta_{H} \times \left(D_{o}^{4} - \frac{D_{o}^{4}}{16} \right)$$

$$\theta_{S} \times D_{o}^{4} = \theta_{H} \times \frac{15D_{o}^{4}}{16}$$

But:
$$D_i = \frac{D_o}{2}$$
 and $D_s = D_o$

$$D_o \times D_o^4 = \theta_W \times \left(D_o^4 - \frac{D_o^4}{D_o^4}\right)$$

$$\theta_{\rm S} \times D_{\rm o}^4 = \theta_{\rm H} \times \frac{15D_{\rm o}^4}{16}$$

$$\theta_{\rm S} = \frac{15}{16} \theta_{\rm H}$$

108. A hollow shaft with an outside diameter of 100 mm is subjected to a maximum torque of 5,403.58 N.m. Determine its thickness if it not to exceed a shearing stress of 60 MPa or a twist of 0.5 degree per meter length of shaft. G=83,000 MPa.

A. 15 mm

- B. 86 mm
- C. 16.8 mm
- D. 14.2 mm

Solution:
$$\frac{\theta}{L} = \frac{32 \cdot T}{\pi \cdot \left(D_o^4 - D_i^4\right) \cdot G} \qquad \frac{0.5^\circ \times \frac{\pi}{180^\circ}}{1 \times 1000} = \frac{32 \times \left(5403.58 \times 10^3\right)}{\pi \times \left(100^4 - D_i^4\right) \times 83000} \qquad D_i = 70 \text{mm}$$
 then;
$$t = \frac{D_o - D_i}{2} = \frac{100 - 70}{2} = 15 \text{mm}$$

109. 90 kW of power is to be transmitted using a hollow shaft. It has an inner diameter of 30 mm and outside diameter of 42 mm. Determine the frequency of rotation of the shaft so that the shear stress cannot exceed 50 MPa.

A 26.6 Hz

B 206Hz

D 66.5 Hz

Solution:
$$T = \frac{P}{2\pi \cdot N} = \frac{90 \times 10^6}{2 \cdot \pi \cdot N} = \frac{1432.3944 \times 10^4}{N}$$
 Then,
$$\frac{1432.3944 \times 10^4}{N} = \frac{50 \times \pi \times \left(42^4 - 30^4\right)}{16 \times 42}$$

Also,
$$T = \frac{S \cdot \pi \cdot \left(D_o^4 - D_i^4\right)}{16D_0}$$

$$N = 26.6 \frac{\text{rev}}{\text{s}} = 26.6 \frac{\text{cycle}}{\text{sec}} = 26.6 \text{Hz}$$

110. A hollow shaft is to substitute a solid shaft with the same material and same torsional strength. The hollow shaft should be half the weight of the solid one. If the solid shaft diameter is 3.5 inches, find the outside diameter and the inside diameter of the shaft in millimeters.

A. 107.31 mm; 86.97 mm

Solution: When solid shaft is to be replaced by hollow shaft having equal strength but 1/2 weight

$$D_s = 3.5in = 88.9mm$$

$$D_o = \left(\frac{1+\sqrt{2}}{2}\right) \cdot D_s = \left(\frac{1+\sqrt{2}}{2}\right) \times 88.9 = 107.31 \text{mm}$$

$$D_i = \sqrt{D_o^2 - \frac{D_s^2}{2}} = \sqrt{107.31^2 - \frac{88.9^2}{2}} = 86.97 mm$$

111. A flange bolt coupling connects a turbine to a generator with an output of 40 MW and rotates at 3600 rpm. Generator efficiency is 90%. The coupling has a bolt circle diameter of 500 mm. If there are 16 bolts, determine the force acting on each bolt.

A. 26.41 kN

B. 29.47 kN

$$\begin{aligned} \text{Solution:} \ \ P_{input} &= \frac{P_{output}}{e} = \frac{40}{0.90} = 44.44 MW \\ F_{b} &= \frac{2T}{n_{b}D_{b}} = \frac{2 \times \left(117.88 \times 10^{3}\right)}{16 \times 500} = 29.47 kN \end{aligned}$$

112. A solid coupling is used in a 75 mm shaft that transmits 300 kW power at 600 rpm. The coupling has 6 bolts and each 18 mm in diameter. Find the required bolt circle diameter if the bolt shearing stress is 27.5 MPa.

A. 227.4 mm

Solution:
$$S_{sb} = \frac{8T}{\pi \cdot n_b \cdot d^2 \cdot D_b}$$
 and $T = \frac{30P}{\pi N}$

$$T = \frac{30P}{\pi N}$$

Then,
$$D_b = \frac{8\left(\frac{30P}{\pi N}\right)}{\pi \cdot n_b \cdot d^2 \cdot S_{sh}} = \frac{8 \times \left[\frac{30 \times \left(300 \times 10^6\right)}{\pi \times 600}\right]}{\pi \times 6 \times 18^2 \times 27.5} = 227.4 \text{mm}$$

113. A 75 mm diameter shaft that transmits 150 kW at 400 rpm is designed with a flat key. If allowable shearing stress is 20 MPa and key width is 15 mm, determine the length of the key.

A. 30.65 mm

B. 31.83 mm

C. 33.75 mm

D. 32.85 mm

$$\text{Solution: } L = \frac{2T}{S_c \cdot b \cdot D} = \frac{2 \left(\frac{30P}{\pi N} \right)}{S_c \cdot b \cdot D} = \frac{2 \times \left[\frac{30 \times \left(150 \times 10^6 \right)}{\pi \times 4000} \right]}{20 \times 75 \times 15} = 31.83 mm$$

114. A square key is to be used in a 40 mm flat key and that will develop a 2 kN.m torque. If bearing stress of key is 448 MPa, determine the cross sectional dimension of flat key to be used using key length of 21.12 mm.

A. 21.12 mm

B. 25.61 mm

C. 28.21 mm

D. 18.61 mm

$$t = \frac{4T}{S_0 \cdot b \cdot D} = \frac{4 \times (2 \times 10^6)}{448 \times 21.12 \times 40} = 21.13 \text{mm}$$

- 115. A line shaft, transmits 7.46 kW at a speed of 1200 rpm, is designed with a rectangular key. If the shearing stress for the shaft and key are 30 N/mm² and 40 N/mm², respectively. What is the diameter of the shaft?
 - A. 18.7 mm
- B. 21.7 mm
- C. 25.8 mm
- D. 30.2 mm

$$S = \frac{16T}{\pi \cdot D^3} = \frac{16\left(\frac{30P}{\pi \cdot N}\right)}{\pi \cdot D^3}$$

hence;
$$D = \sqrt[3]{\frac{16\left(\frac{30P}{\pi \cdot N}\right)}{\pi \cdot S}} = \sqrt[3]{\frac{16\left[\frac{30 \times \left(7.46 \times 10^6\right)}{\pi \times 1200}\right]}{\pi \times 30}} = 21.6mm$$

- 116. A load of 5000 kg is supported by a bearing with 150 mm diameter and 300 mm long. If coefficient of friction is 0.18, find the torque required to rotate the shaft.
 - A. 331 N-m

- C. 873 N-m
- D. 1020 N-m

Solution:
$$F_f = f \cdot W = 0.18 \times (5000 \times 9.8066) = 8825.94N$$

$$T_f = F_f \cdot \frac{D}{2} = 8825.94 \times \frac{0.150}{2} = 662 \text{N.m}$$

- 117. A shaft, rotates at 500 rpm, is supported by a bearing with frictional loss of 15 kW. The bearing load is 30 kN and friction of 0.14. Find the bearing
 - A. 136.42 mm
- B. 146.42 mm
- C. 156.42 mm
- D. 166.42 mm

Solution:
$$F_f = f \cdot W = 0.14 \times 30 = 4.2 kN$$

Then,
$$D = \frac{P_f}{\pi \cdot F_f \cdot N} = \frac{15}{\pi \times 4.2 \times \frac{500}{60}} \times 1000 = 136.42 mm$$

- 118. A shaft, rotates at 1740 rpm, is supported by a bearing with a length of 105 mm and diameter of 64 mm. If the load is light and SAE Oil No. 20 (μ = 2.4 x 10⁻⁶ reyns) is used and diametral clearance is 0.136 mm, find the power loss due to friction.
 - A. 164 watts
- **B. 174 watts**

Solution: $\mu = 2.4 \times 10^{-6} \text{ reyns} = 0.0165 \text{ Pa-s}$

$$h = C_d / 2 = 0.136/2 = 0.068 \text{ mm}$$

Then,
$$T_f = \frac{4 \cdot \pi^2 \cdot r^3 \cdot \mu \cdot L \cdot N_s}{h} = \frac{4 \times \pi^2 \times \left(\frac{0.064}{2}\right)^3 \times 0.0165 \times 0.105 \times \frac{1740}{60}}{0.068 \times 10^{-3}} = 0.9558 N.m$$

$$P_f = 2 \cdot \pi \cdot T_f \cdot N_s = 2 \times \pi \times 0.9558 \times \frac{1740}{60} = 174 watts$$

- 119. A bearing operates satisfactorily with a diametral clearance of 0.0028 inches supports a journal running at 1200 rpm. The bearing is 2.085 inches in diameter and 1.762 in long. It supports a total radial load of 1400 lbs. at 160° F operating temperature of the oil film, the bearing modulus Zn/P was found to be 16.48. Determine the bearing stress.

- B. 381 psi
- C. 481 psi
- D. 581 psi

Solution:
$$S_b = \frac{F}{A} = \frac{F}{D \times L} = \frac{1400}{2.085 \times 1.762} = 381 psi$$

- 120. Find the tooth thickness of a 14° involute gear having a diametral pitch of 6.
 - A. 5.33 mm
- B. 6.65 mm
- C. 8.45 mm
- D. 12.36 mm

Solution:

tooth thickness =
$$\frac{1.5708}{P_d} = \frac{1.5708}{6} \times 25.4 = 6.65 \text{mm}$$

- 121. Compute for the speed mounted in a 52.5 mm diameter shaft receiving power from a prime motor with 250 Hp.
 - A. 2182 rpm
- B. 2081 rpm
- C. 2265 rpm
- D. 2341 rpm

Solution:

$$P = \frac{D^3 \times N}{80}$$

$$P = \frac{D^3 \times N}{80} \qquad \text{hence;} \qquad N = \frac{80P}{D^3} = \frac{80 \times 250}{\left(\frac{52.5}{25.4}\right)^3} = 2265 \text{rpm}$$

122. A spur pinion rotates at 1800 rpm and transmits to mating gear 30 Hp. If the pitch diameter is 8 inches and the pressure angle is 14.5°, determine the total loads in lbs.

A. 123.45 lbs

B. 653.15 lbs

C. 271.14 lbs

D. 327.43 lbs

Solution:

$$T = \frac{63,000 \text{Hp}}{N_{m}} = \frac{63,000 \times 30}{1800} = 1050 \text{in.lb}$$

$$F_t = \frac{2T}{D} = \frac{2 \times 1050}{8} = 262.5$$
lbs

$$F = \frac{F_t}{\cos \phi} = \frac{262.5}{\cos 14.5} = 271.14 \text{lbs}$$

123. A precision cut gear in an intermittent service transmits 25 Hp at a pitch line velocity of 6000 ft/min. Compute for the dynamic load.

Solution:

$$F_t = \frac{Hp \times 33000}{V_m} = \frac{25 \times 33000}{6000} = 137.5 lbs$$

$$\text{For precision cut;} \hspace{0.5cm} F_d = \left\lceil \frac{78 + \sqrt{V_m}}{78} \right\rceil \times F_t = \left\lceil \frac{78 + \sqrt{6000}}{78} \right\rceil \times 137.5 = 274 lbs$$

124. A helical gear having 28 teeth and pitch diameter of 7 has a helix angle of 22 degrees. Find the circular pitch in a plane normal to the pitch.

Solution:

$$P_{cn} = \left(\frac{\pi D}{T}\right) \times \cos \psi = \left(\frac{\pi \times 7}{28}\right) \times \cos 22 = 0.728$$

125. A 20-tooth helical gear has a pitch diameter of 5 inches. Find the normal diametral pitch if helix angle is 22 degrees.

A. 3.31 in

- B. 4.31 in
- C. 5.31 in
- D. 6.31 in

$$P_{dn} = \frac{T}{D \times \cos w} = \frac{20}{5 \times \cos 22} = 4.31 \text{ in}$$

126. A helical gear with a tangential load of 200 lbs has an axial load f 73 lbs. Find the helix angle.

- B. 16 degrees
- D. 20 degrees

$$\psi = \tan^{-1} \left[\frac{F_a}{F_t} \right] = \tan^{-1} \left[\frac{73}{200} \right] = 20^\circ$$

127. In a pair of gear that connects a pair of shaft at 90°, the velocity ratio is 3 to 1. If the gear is straight bevel, what is the cone pitch angle of smaller

A. 71.57 deg

- B. 65.34 deg
- D. 12.34 deg

$$SR = \frac{T_g}{T_p} = \frac{3}{1}$$

$$\tan \alpha = \frac{\sin \theta}{\frac{T_g}{T_p} + \cos \theta}$$

$$SR = \frac{T_g}{T_p} = \frac{3}{1} \qquad \tan\alpha = \frac{\sin\theta}{\frac{T_g}{T} + \cos\theta} \qquad \qquad \text{For} \quad = 90^\circ, \qquad \qquad \alpha = \tan^{-1}\!\!\left[\frac{T_p}{T_g}\right] = \tan^{-1}\!\!\left[\frac{1}{3}\right] = 18.43 \text{ deg}$$

128. A left hand spiral bevel pinion rotates clockwise and transmits power to a mating gear with speed ration of 2 to 1. Calculate the pitch angle of the

A. 16.56 deg

- B. 20.56 deg
- C. 26.56 deg
- D. 32.56 deg

D. 2.14

$$\alpha = \tan^{-1} \left[\frac{1}{SR} \right] = \tan^{-1} \left[\frac{1}{2} \right] = 26.56 \text{ deg}$$

129. A gear has a cone angle of 65° while the pinion has 25°. Find the speed ratio.

Solution:

- $SR = \frac{\sin \beta}{\sin \alpha} = \frac{\sin 65}{\sin 25} = 2.14$
- 130. A work at 1150 rpm drives a worm gear. The velocity is 15 to 1. A 10 hp motor is used to supply the worm with pitch gear diameter of 3 in. Find the force on the worm.

A. 365.37 lbs

- B. 465.37 lbs.
- C. 565.37 lbs
- D. 665.37 lbs

Solution:
$$F_t = \frac{2T}{D} = \frac{2}{D} \left(\frac{63,000 \text{Hp}}{N_m} \right) = \frac{2}{3} \left(\frac{63,000 \times 10}{1150} \right) = 365.21 \text{lbs}$$

131. A double thread worm has normal pressure angle of 14.5, a pitch diameter of 3.5 in and a lead of 1.5 in. Find the pressure angle of the worm.

A. 11.68°

B. 12.68°

C. 13.68°

D. 14.68°

Solution:
$$\lambda = \tan^{-1}\!\!\left(\frac{L}{\pi D_w}\right) = \tan^{-1}\!\!\left(\frac{1.5}{\pi \times 3.5}\right) = 7.768^\circ \\ \text{Then; } \phi = \tan^{-1}\!\!\left(\frac{\tan \phi_n}{\cos \lambda}\right) = \tan^{-1}\!\!\left(\frac{\tan 14.5}{\cos 7.768}\right) = 14.628^\circ$$

- 132. A belt drive uses pulleys with center distance of 72 inches. If the pulley diameters are 6 in. and 12 in., respectively, find the angle of contact on the small pulley.
 - A. 180.60 deg
- B. 243.40 deg
- C. 203.61 deg
- D. 175.22 deg

Solution:
$$\theta = \pi - \frac{D_2 - D_1}{C} = \pi - \frac{12 - 6}{72} = 3.0582 \text{rad} \times \frac{180^{\circ}}{\pi \text{rad}} = 175.22^{\circ}$$

133. An open belt drive uses pulleys with center distance of 72 cm. If the pulley diameters are 6 cm and 12 cm, respectively, determine the belt length.

A. 172.39 cm

D. 190.39 cm

Solution:
$$L = \frac{\pi}{2} \left(D_1 + D_2 \right) + 2C + \frac{\left(D_2 - D_1 \right)^2}{4C} = \frac{\pi}{2} \left(6 + 12 \right) + 2 \left(72 \right) + \frac{\left(12 - 6 \right)^2}{4 \left(72 \right)} = 172.39 cm^2$$

134. A belt, with specific weight of 0.035 lb/in³, is 3/8-inch thick and 12-inch wide. It is used on 24 inches diameter pulley rotating at 600 rpm. If the angle of contact is 150 degrees, the coefficient of friction is 0.3 and stress is 300 psi, how much power can it deliver?

A. 65.4 Hp

B. 69.5 Hp

C. 60.5 Hp

D. 63.5 Hp

$$\begin{split} \text{Solution:} \qquad & F_1 = S \cdot b \cdot t = 300 \times \frac{3}{8} \times 12 = 1350 \text{lbs} \qquad ; \qquad & V_s = \pi \cdot D \cdot N = \pi \times \frac{24}{12} \times \frac{600}{60} = 62.83 \text{fps} \\ & F_c = \frac{12 \cdot \rho \cdot b \cdot t \cdot V_s^2}{g} = \frac{12 \times 0.035 \times \frac{3}{8} \times 12 \times 62.83^2}{32.2} = 231.707 \text{lbs} \\ & F = \left(F_1 - F_c \left(\frac{e^{f\theta} - 1}{e^{f\theta}}\right) = \left(1350 - 231.707\right) \left(\frac{e^{\left(0.3 \times \frac{150 \times \pi}{180}\right)} - 1}{e^{\left(0.3 \times \frac{150 \times \pi}{180}\right)}}\right) = 608.42 \text{lbs} \\ & Hp = \frac{TN_m}{63,000} = \frac{N_m}{63,000} \left(\frac{F \cdot D}{2}\right) = \frac{600}{63,000} \left(\frac{608.42 \times 24}{2}\right) = 69.5 \text{Hp} \end{split}$$

- 135. A 350 mm wide belt has an angle of contact of 160°. The working stress of belt is 2 MPa and coefficient of friction is 0.32. If the pulley has an effective belt pull of 3 kN, determine the thickness of the belt.
 - A. 6.42 mm
- B. 7.24 mr
- C. 8.68 mm
- D. 9.47 mm

$$\begin{array}{ll} \mbox{Solution:} & \frac{F_l}{F_2} = e^{f\theta} = e^{0.32 \times \left(\frac{160 \times \pi}{180}\right)} = 2.444 & \mbox{therefore } : F_2 = \frac{F_l}{2.444} \\ \\ F = F_l - F_2 & \mbox{substituting} & 3 = F_l - \frac{F_l}{2.444} & \mbox{therefore } ; \; F_l = 5.07 \\ \end{array}$$

Then;
$$t = \frac{F_l}{S \cdot b} = \frac{5.07 \times 10^3}{2 \times 350} = 7.24 \text{mm}$$

136. Determine the speed of the belt used in a pulley with belt pull of 2.5 kN and is driven by a 20 Hp motor.

A. 19.58 ft/sec

B. 5.97 ft/sec

C. 7.42 ft/sec

D. 10.86 ft/sec

Solution: F = 2.5 kN 562.12 lbs
$$V_s = \frac{550 Hp}{F} = \frac{550 \times 20}{562.12} = 19.568 ft/sec$$

137	A chain and	snrocket has	18 teeth with chain	nitch of 1/2 in Find th	e pitch diameter of sprocket.
101.	A GHAIH AHA	sprounct mas	10 tootii witii onaiii	pitori or /2 iii. I iiiu tii	c pitch diameter of spiceket.

D. 3.879 in

A. 0.879 in B. 1.879 in Solution:
$$D = \frac{P}{\sin(\frac{180}{T})} = \frac{0.5}{\sin(\frac{180}{18})} = 2.879 in$$

138. A chain and sprocket has 24 teeth with chain pitch of ½ in. If the sprocket turns at 600 rpm, find the speed of chain.

A. 601.72 fpm

B. 621.72 fpm

C. 641.72 fpm

D. 661.752 fpm

$$V = \pi \cdot D \cdot N = \pi \times \frac{P}{\sin\left(\frac{180}{T}\right)} \times N = \pi \times \frac{0.5}{\sin\left(\frac{180}{24}\right) \times 12} \times 600 = 601.72 fpm$$

139. A chain and sprocket has a pitch diameter of 28.654 in and there are 90 teeth available. Find the pitch of the chain.

C. 1 in

$$P = D \times \sin\left(\frac{180}{T}\right) = 28.654 \times \sin\left(\frac{180}{90}\right) = \lim_{T \to \infty} \frac{180}{90} = \lim_{T \to \infty} \frac{18$$

140. A spring deflects 8.5 inches if a load of 50 lb is applied. What load deflects the spring by 2.5 in?

Solution:

$$k = \frac{F_{8.5}}{\delta} = \frac{50}{8.5} = 5.88 lb/in$$
 then; $F_{2.5} = k \cdot \delta = 5.88 \times 2.5 = 14.70 lb$

$$F_{2.5} = k \cdot \delta = 5.88 \times 2.5 = 14.7010$$

141. A 360 kg load is supported by four compression coil spring in parallel. Each spring has a gradient of 0.717 kg/mm. Calculate the deflection.

A. 125.52 mm

B. 132.52 mm

C. 138.52 mm

D. 145.52 mm

Solution:
$$\delta = \frac{F}{4k} = \frac{360}{4(0.717)} = 125.52 \text{mm}$$

142. An UNC bolt has a stress area of 0.606 in2. If it is to be made from a carbon steel material, what is the permissible working stress of the bolt?

A. 4055.5 psi

B. 5244.5 psi

C. 4675.5 psi

D. 4186.5 psi

Solution:

from Vallance;

$$S_w = C \times A_s^{0.418}$$

where: C for carbon steel = 5000 and C for alloy steel = 15000

$$S_w = 5000 \times 0.606^{0.418} = 4055.5$$
psi

143. The allowable working stress of a 1.5 inches bolt which is screwed up tightly in packed joint is 13000psi, determine the working strength.

A. 11.465.5 lbs

B. 13.860.5 lbs

C. 11.212.5 lbs

D. 11.854.5 lbs

Solution:

from Machinery Handbook, $F_w = S_t \left(0.55 d^2 - 0.25 d\right)$

$$F_w = 13,000 \left[0.55 \left(1.5^2 \right) - 0.25 \left(1.5 \right) \right] = 11,212.5 lbs$$

144. The power screw's linear speed is 7 ft/min and lead of 8 mm. If the total torque required to turn the power screw is 50 N.m, find the horsepower input of the power screw.

A. 2.86 Hp

B. 1.84 Hp

C. 2.14 Hp

Solution:
$$N_m = \frac{V_m}{L} = \frac{(8 \times 12 \times 25.4)}{8} = 304.8 rpm$$

$$Hp = \frac{T \cdot N_m}{63000} = \frac{\left(50 \times \frac{39.36 \times 2.205}{9.8066}\right) \times 304.8}{63000} = 2.14Hp$$

145. A double square thread power screw has a root diameter of 0.55 inches. If the screw has a pitch of 0.2 in., determine the major diameter. A. 0.524 in B. 0.750 in C. 0.842 in D. 0.961 in

Solution: For square thread:

$$h = \frac{p}{2} = \frac{0.2}{2} = 0.1$$

$$h = \frac{p}{2} = \frac{0.2}{2} = 0.1$$
 then; $D_o = 2h + D_i = 2(0.1) + 0.55 = 0.750in$

A. 12.6°

B. 14.3°

C. 16.5°

D. 18.3°

$$0.70 = \frac{\tan \lambda (1 - 0.1 \tan \lambda)}{\tan \lambda + 0.1} \times 100\%$$

Therefore: $= 14.3^{\circ}$

147. A block weighing 350 lbs is lifted by an SAE 1040 eyebolt. SAE 1040 material has a S_u = 67 ksi and S_v = 55 ksi. Calculate the stress area (in inches square) of the bolt if it is under the unified coarse series thread.

B. 0.1134

Solution: From Faires,
$$F_w = \frac{S_y}{6} (A_s)^{3/2}$$

Then,
$$A_s = \sqrt[3]{\left(\frac{6 \cdot F_w}{S_y}\right)^2} = \sqrt[3]{\left(\frac{6 \times 350}{55,000}\right)^2} = 0.1134 \text{ in }^2$$

148. The tension on the tight side of the brake is thrice as the slack side. If coefficient of friction is 0.25, find the angle of contact on the band.

A. 240.61 deg

B. 251.78 deg

C. 286.75 deg

D. 275.65 dea

Solution:

$$\frac{F_1}{F_2} = e^{f \cdot \theta} = 3$$

then;
$$\theta = \frac{\ln 3}{0.25} \times \frac{180}{\pi} = 251.78 \text{ deg}$$

149. The maximum tensile stress of a steel band is 55 MPa. The band has a thickness of 4 mm and has a tension of 6 kN in the tight side. Determine the width of band that should be used.

C. 28.28 mm

D. 29.29 mm

Solution:
$$t = \frac{F_l}{S_t \cdot b} = \frac{6 \times 10^3}{55 \times 4} = 27.27 \text{mm}$$

150. In a cone clutch, the angle of the conical elements is 10 degrees. The mean diameter of conical sections is 200 mm and an axial force of 600 N is applied. If a coefficient of friction of 0.45 is assumed, determine the torque that the cone clutch can transmits.

B. 155.49 N.m

C. 175.49 N.m

Solution:

$$T_f = f \cdot \frac{Fa}{\sin \theta} \cdot \frac{D_m}{2} = 0.45 \times \frac{600}{\sin 10} \times \frac{0.200}{2} = 155.49 \text{N} \cdot \text{m}$$

151. In a small LPG station, a spherical tank with an internal pressure of 5MPa has a thickness of 15 mm. The joint efficiency is 96% and stress is limited to 46875 kPa. Find the inner diameter of the tank.

A. 150 mm

B. 510 mm

C. 450 mm

D. 540 mm

Solution:

$$Di = \frac{4 \cdot S_t \cdot t \cdot \eta}{Pi} = \frac{4 \times 46.875 \times 15 \times 0.96}{5} = 540 \text{mm}$$

152. A 675 kg body moves at 24 kph. What is the kinetic energy of the body?

A. 13.000 N.m.

B. 14.000 N.m.

C. 15,000 N.m

D. 16,000 N.m

KE =
$$\frac{1}{2}$$
 m · V² = $\frac{1}{2}$ × 675 × $\left(\frac{24 \times 1000}{3600}\right)^2$ = 15,000 N · m

153. If he force needed to move a mass weighing 56 lb along a horizontal surface is 20 lb, determine the coefficient of friction.
A. 0.01
B. 0.112
C. 0.36
D. 0.28

Solution:

$$W = F_n$$
 and $f \cdot F_n = F$

 $W = F_n$ and $f \cdot F_n = F$ therefore; $f = \frac{F}{F_n} = \frac{F}{W} = \frac{20}{56} = 0.36$

is the process of shaving off small amounts of metal using hand-held tools. This process is most often done to obtain precision alignment of machine parts; it is also done to provide shallow pockets that will hold lubricants on flat surfaces, such as machine ways.

b) Scraping

c) Reaming

d) Millina

____ means putting the parts of something together.

b) Assembling

c) Installing

d) Fabricating

are tools used to turn, or drive, screws; made in may sizes and several shapes; the size is measured by the length of the blade, which is made of tool steel that is hardened and tempered. The screwdriver point for driving slotted screws should be correctly shaped; it must be fit the

a) Wrenches

b) Hammers

c) Screw drivers

d) Scrapers

157. A place in which metal parts are cut to the size required and put together to form mechanical units of machines, the machine so made to be used directly or indirectly in the production of the necessities and luxuries of civilization.

a. Welding shop

b. Junk shop

c. Machine shop

c) Flower shop

158.	A metal-turning machine tool in Wi (feed) in a direction more or less p a. Grinding machine			feed), or in a direction of right		the axis of	
159.	This screw driver has a bent handl a) Offset screw driver	e; it is used where a straight so b) Phillips screw driver		will not reach. nical screw driver	d) Mill sci	rew driver	
160.	A machine tool used mainly for proa. Reaming machine	oducing holes in metal. b. Drill press	c. Weldin	g machine	d. Boring	machine	
161.	${a)$ Screw drivers	ting small wire and for holding, b) Wrenches	twisting, tu	ırning, pulling, and pushing.	d) Hamm	ers	
162.	A machine which is ordinarily used a. Shaper machine	I for finishing flat or partly curve b. Planer machine	ed surfaces c. Milling			usually ove ng machine	
163.	A machine tool in which an abras	sive wheel is used as a cutting	tool to obta	ain a very high degree of accu	uracy and	a smooth	finish on metal parts,
	including soft and hardened steel. a. Drilling machine	b. Milling machine	c. Grindii	ng machine	d. Weldin	g machine	
164.	A file which is thick, rectangular-sh a. Hand file	naped file with parallel edges. b. Pillar file	c. Mill file		d. Half-m	oon file	
165.	A file which is tapered in thickness a. Pillar file	and of uniform width. This is u b. Hand file	sed for filin c. Mill file	ng slots and keyways.	d. Half-m	oon file	
166.	a) Slip-joint pliers	tion pliers, are used for gripping b) Side cutting pliers		cut small-size wire; the slip-joi nose pliers		it possible lose pliers	to grip large parts.
167.	Which of the following is not belong a. flat	g to standard shapes of file? b. half-round	c. three-s	quare	d. none d	of these	
168.	One of the three basic ways of usin a. Lathe filing	ng a file which is done by push b. Draw filing	ing the file c. Straigh		htly diago d. Back fi		vorkpiece.
169.	One of the three basic ways of usin a. Lathe filing	ng a file which consists of strok b. Straight filing	king the file c. Draw fil		ce held in t d. Mill filir		
170.	One of the three basic ways of usin a. Draw filing	ng a file which is performed by b. Lathe filing	grasping th			cross the v	
171.	A hand tool which is commonly us	sed for cutting or twisting wire	and chippi	ng small parts and not to be u	ısed for tiç	ghtening o	loosening bolts and
	nuts. a. Screwdriver	b. Wrench	c. Pliers		d. None o	of the abov	е
172.	A type of pliers which is used for ca. Long-nose pliers	utting wires and holding flat rou b. Side-cutting pliers		int combination pliers or med	chanical p	oliers	d. Square pliers
173.	A type of pliers which is used in ele a. Side-cutting pliers	ectrical works for cutting heavy b. Long-nose pliers	or light wir	es or for cutting small nails. c. Mechanical pliers		d. None o	f the above
	A type of pliers which is used for coa. Long-nose pliers	b. Slide-cutting pliers	is is also u	seful in reaching tight places. c. Diagonal pliers		d. None o	f the above
175.	Which of the following is a primary a) Casting	forming process? b) Turning		c) Fitting		d) Buffing	
176.	Which of the following is not a made a) Turning	chining process? b) Milling		c) Casting		d) Grindin	g
177.	The joints produced by this method a) Soldering	d are as strong as the parent m b) Brazing	netal.	c) Riveting		d) Weldir	ng
178.	Which one of the following is a finis a) Rolling	shing process? b) Forging		c) Sintering		d) Engrav	ing
179.	Which of the following process affe a) Shaping	ects the changes in properties of b) Hardening	of materials	s? c) Sintering		d) Anodiz	ing
	The system used for mass product a) Simplification	b) Standardization	ecified limits	s of sizes for selection of comp c) Interchangeable system	onents at	random is d) Automa	
101.	Alloys are extensively used in indu a) Sufficiently soft	b) Have good strength		c) Malleable		d) Ductile	

182. Which of the following is the most abundantly used material in industry?

	a) Cast iron	b) Mild steel		c) Medium carbon steel		d) High carbon steel
183.	Wrought iron contains a maximum a) 0.01 % carbon	of which of the following? b) 0.3 % carbon		c) 0.6 % carbon		d) 2.0 % carbon
184.	Pig iron is produced by reduction of a) Cupola	of iron ore in a: b) Bessemer converter		c) Open hearth furnace		d) Blast furnace
185.	Which of the following are the mair a) Nickel and manganese	n alloying elements present in s b) Nickel and tungsten	tainless st	eel? c) Nickel and chromium		d) Nickel and cobalt
186.	Along with chromium and vanadiur a) Manganese	m the main constituent of high s b) Nickel	speed stee	l is which of the following? c) Tungsten		d) Vanadium
187.	Grey cast iron gives grey fracture of a) Ferrite	due to the presence of which of b) Pearlite	the follow	ing? c) Graphite		d) Cementite
188.	Copper is the parent metal of brass a) Lead	s. Which of the following is the ob) Tin	other meta	l present in brass? c) Aluminum		d) Zinc
189.	Gun metal contains 5 to 10 percen a) Zinc	t of which of the following mate b) Lead	rials?	c) Tin		d) Aluminum
190.	Y-alloy is an alloy of aluminum and a) Manganese	which of the following metals? b) Iron		c) Tin		d) Copper
191.	Inconel contains 80 % nickel and 1 a) Iron	4 % chromium. The other meta b) Vanadium	al present i	n inconel is: c) Silver		d) Tin
192.	Monel contains mainly with traces a) Aluminum	of iron and manganese. The ma	ain alloyin	g element of monel is: c) Zinc		d) Lead
193.	Bearing materials should posses was a) High tensile strength strength	hich of the following? b) High hardness		c) Hard core		d) High compressive
	Which of the following is the trade a) Alkane The following instrument can be us a) Vernier caliper	b) Polyprex	I to the ed	c) Styrene ges of a part: c) Hermaphordite calipe		d) Durite d) Combination set
196.	Which of the following is the use of a) Laying out the work accuracy			c) Checking surface finish		d) Leveling the machine tools
197.	The thickness of a thin sheet can ba) Ordinary scale	be accurately checked with the b b) Micrometer	help of:	c) Combination set		d) Caliper
198.	V-block is used in mechanical engia) Straightness of the job	ineering workshop to check whi b) Taper on a job	ch of the f	ollowing? c) Height of a job	d) Cylind	drical jobs
199.	is a strong tool used for	general work in the shop. Its ja	ws are poi	nted at such an angle that	it can be used	d in close corners and unhandy
	places. a) Adjustable-end wrench	b) Adjustable S-wrench		c) Monkey wrench	d) Vise-g	rip wrench
200.	A sine bar is specified by: a) Its total length rollers	b) Diameter of the rollers		c) Its weight d) Ce	nter to cente	r distance between the
201.	is a handy tool. It pipe wrench, open-end wrench, or		strong stee	el jaws lock to the work and	will not slip; i	t acts like a vise, clamp, plier,
	a) Adjustable-end wrench	b) Adjustable S-wrench		c) Monkey wrench	d) Vise-g	grip wrench
202.	A ratchet screw is provided on a m a) Lock the measured reading c) Prevent wearing of screw the	3	•	nin constant measuring poero adjustments	ressure for a	ll readings
203.	a) Arbor press	essing parts of machinery toget b) Drill press	ther or ford c) Milling			t in or out of a pulley or gear g machine
204.	Adjusting nut is provided on a micr a) Allow zero adjustment c) Eliminate backlash	ometer screw gauge to:		ensate for wear and tear I in constant measuring pres		

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205. External taper of a job can be measured accurately with help of:

	a) Sine bar and slip gaugesc) Dial indicator and slip gaug			r and dial indicator uges and vernier caliper		
206.	Internal dovetail taper can be mea a) Sine bar and slip gauges c) Balls of standard size and		b) Balls of	standard size and dial indica f standard size and slip ga		
207.	Combination set is used to: a) Measure long distances	b) Measure cylindrical sizes of	f a length	c) Measure angle	s	d) Measure corner radius
208.	a) Toggle clamps	sizes and types are widely used b) C-clamps	d in specia c) Spring		d) Spring	clamps
209.	It is a typical of material used for c a) Forged steel c) Cast iron and coated with c			b) Drop-forged, heat-treated d) Cast iron	steel\	
210.	Which of the following is a materia a) Forged steel c) Cast iron and coated with	,		b) Drop-forged, heat-treated d) Cast iron	steel\	
211.	Which of the following is the typica a) Forged steel	ll material used for an engine cr b) Heat-treated steel		c) Cast iron		d) Wrought iron
212.	What is the function of a governor a) Store energy and give up whate c) Increase variation of speed			b) Regulate the speed during d) Adjust variation of spee	g one cycle o	of a prime mover g the input to the engine
213.	What is a machine used in testing a) Izod test	steel generally strike the specir b) Charpy test		nergy from 220 to 265 ft-lb? c) Rockwell test		d) Test blocked
214.	Almost always the soldering mater a) Lead and tin	rials are a combination of which b) Tin and antimony		owing alloys? c) Lead and antimony		d) Both a, b & c
215.	To ensure an adequate factor of sa a) One half of the shaft diameter c) One eighth of the shaft diameter			keyway, the key width shoul b) One fourth of the shaft c d) One third of the shaft dian	liameter	
	This is defined as the cutting time a) Wear duration Machining operations with the propa) Higher cutting speeds	b) Cycle time	generally h	c) Tool life		d) Life cycle d) Higher cutting accuracy
218.	A material of construction (only oranging corrosion resistance, low ora) Tungsten			s concurrently with zirconiu	m) offers th	e unique combination of wide d) Molybdenum
219.	Which of the following is the lighter a) Aluminum	st of all structural metals? b) Copper		c) Magnesium		d) Manganese
220.	This is a type of welding whereby particles are projected in a form of a) Metal spray welding		d air or gas			or plasma jet and the molten d) plasma arc welding
	Which of the following does not be a) Core-making machine	b) Forging machine		c) Shake-out machine		d) Molding machine
222.	Which of the following is a suitable a) Bench target and scriber	b) Bench top and tachometer	•	tt? c) Bench side and calliper	d) Bench	centers and dial indicator
223.	This is a work supporting device w away of the work piece from cutting a) Tool post		t?	achine. It travels with the cut c) Follower rest	ting tool, and	d used to prevent the springing d) Rubber-flex collets
	Which of the following raw materia a) Silica sand, linseed oil, & fl c) Silica sand, lime, flour	ls are used in the foundry mold lour	ing operati			le
	Which of the following is not part o a) Head screw	b) Thimble		c) Spindle		d) Anvil
	The flux that should be avoided in a) Acid fluxes It is a lathe machine operation us	b) Salt ammoniac		c) Zinc chloride		d) Sodium chloride
	produce grooves for o-ring seats. a) Boring	b) Knurling		c) Facing		d) Parting

228.	The most important operations in manufacturing is hole making. This can be a) Drilling machine b) Sawing machine	e done by which of the following? c) Boring mills	d) Milling machine			
229.	It is a machine operation used to produce regularly shaped, roughness on c					
	a) Milling b) Sawing	c) Drilling	d) Knurling			
230.	Which of the following material properties is adversely affected by grain refin		B 111 ()			
231	a) Creep resistance b) Tensile strength Hard solder is made of which of the following?	c) Ductility	d) yield strength			
231.	a) Tin and zinc b) tin and copper	c) Copper and zinc	d) Tin and lead			
232.	Which of the following is the unit of mass moment of inertia?	,,	,			
	a) kg-m² b) kg/m²	c) kg	d) N-m			
233.	Which of the following is the unit of Young's modulus of elasticity? a) Dimensionless b) Pascal	a) Matter	d) Nouton			
234	When a lathe tool bit burns, it means that the:	c) Metter	d) Newton			
	a) Speed is too low b) Material is too hard	c) Speed is too fast	d) Material can't be cut			
235.	The area of the machine shop whose metal is being melted to form a new s					
226	 a) Foundry area b) Tool and die A corrosion occurs within or adjacent to a crevice formed by contact with an 	c) Welding area	d) Mass production area			
230.	a) Pitting b) Galvanic	c) Erosion	d) Crevice			
237.	Which of the following is not part of a headstock?	,	,			
	a) Anvil b) Spindle	c) Motor	d) Back gear			
238.	It refers to any layer or deposit of extraneous material on a heat-transfer sur		d) Cooling			
239	 a) Low line b) Pressure drop Have the same thermal and minimum film thickness limitation as sleeve bea 	c) Fouling	d) Scaling			
200.	a) Ball bearing b) Roller bearing	c) Thrust bearing	d) Oil whip			
240.	An oil storage tank roof formed to approximately a spherical surface, suppo		,			
	a) Self-supporting umbrella roof	b) Self-supporting cone roof				
241	c) Self-supporting What is a unit deformation?	d) Supported cone roof				
271.	a) Torsion b) Strain	c) Stress	d) Shear			
242.	Accident prevention is:	,	,			
	a) An association of employers, organization & individuals	b) A job of a safety director				
2/12	c) The Foreman's responsibility just as much as production Alloy steel known for its resistance to corrosion, abrasion and wear that	d) The responsibility of top management				
243.	application. It is usually combined with molybdenum to increase the depth h		e in cement and concentrator			
	a) Manganese chromium steel b) Chromium-moly steel	c) Chrome-nickel-moly steel	d) Manganese-moly steel			
244.	Cutting of internal threads of a work piece can be produced by which of the	•				
245	 a) Milling b) Tapping Which of the following milling operations is used to produce curved profiles, 	c) Boring	d) Reaming			
243.	a) Form b) Straddle	c) End	d) Slab			
246.	In screw threads, what is a pitch?	-, <u>-</u>	-, -:			
	a) The distance from a point on one thread to a corresponding point on the next thread measured parallel to the axis.					
	b) The distance of the full length of the threadc) The distance from the bottom of a head on a bolt to the first thread.					
	d) The distance from the bottom of a head on a bott to the first timead.	read				
247.	Tapered shanks are used on large drill presses so that:					
	a) The drill can be centered more easily	b) The drill can be easily forced out o	f the sleeve with a drift			
	c) The shank will not turn when cutting	d) The shank can be ground when wrn				
248	It is a type of chuck wherein a work piece with standard diameter or siz	e could be attached fast and quick spe	cially small size work of mass			
	production. How do you call this type of chuck?	o could be allacined fact and quient ope	cian, cinan cizo nem ci mace			
	a) Collet attachment b) Lathe drive plate	c) Clamp toe dog	d) Steady center rest			
249.	A type of projection where the object is drawn as viewed in a glass box. The					
250	a) Fourth angle projection b) Third angle projection Which of the following is often called as the backbone of lathe machine?	c) Second angle projection	d) First angle projection			
200.	a) Bed b) Headstock	c) Tailstock	d) Carriage			
251.	Which of the following properties of metal is insensitive to the microstructure					
252	a) Hardness b) Ductility Which of the following electic modulus is applied to liquide?	c) Tensile strength	d) Modulus of elasticity			
252.	Which of the following elastic modulus is applied to liquids? a) Young's Modulus b) Bulk Modulus	c) Shear modulus	d) Modulus of rigidity			
253.	Polymers which, at room temperature, can undergo large deformations un	,				
	called as:					
054	a) Meso-polymers b) Thermoplastics	c) Thermosetting polymers	d) Elastomers			
254.	When using a drill press, the work should be held with which of the following a) The hand b) The glove hand	j ? c) Pliers	d) A vise and clamp			
255.	Which of the following classes of materials exhibits a decreasing electrical of	,	a, at the and signify			
	a) Intrinsic semiconductor b) P-type semiconductor	c) Metals	d) Pure ionic materials			
256.	Which of the following is another name of hydrochloric acid?	a) Municipacid	al\ Nitaria a ai al			
257	a) Sulfuric acid b) Acetic acid Fatigue failure occurs when a part is subjected to which of the following?	c) Muriatic acid	d) Nitric acid			
201.		a) Tanaila atraca	d) Compressive stress			
258	a) Fluctuating stress b) Torsion	c) Tensile stress	a) Compressive stress			
200.	A body having identical properties all over is called as:	c) Tensile stress	u) Compressive stress			
	A body having identical properties all over is called as: a) Elastic b) Homogeneous	c) Ductile	d) Isentropic			
	A body having identical properties all over is called as:	c) Ductile				

260.	a) Strain aging	in strength is increased and ductility is d b) Screw dislocation	c) Clustering	emperature atter cold working. d) Twinning
261.	A ductile fracture is characterized b		o, cractering	a) I willing
	a) Fragmentation into more than tw	o pieces	b) Negligible deformation	
	c) Appreciable plastic deformation		d) rapid rate for crank propagation	
262.			lastically and to return it when unloaded?	
263	 a) Resilience The machinability of steel can be in 	b) Hardness	c) Fatigue strength	d) Creep
200.		b) Phosphorous, lead, & sulphur	c) Sulphur, graphite and aluminium	d) Silicon and Sulphur
264.	Which of the following is steel with		c, ca.p., g. ap.,	5, 55 aa. 5 a.p a.
	a) Solidus	b) hyper-eutectoid	c) Eutectoid	d) Austenite
265.		steel depends on which of the following?		DAIL : I
266	 a) Chemical composition Hastalloy contains which of the following 	b) Carbon content	c) Grain size	d) Alloying element present
200.		b) Nickel and copper	c) Copper and aluminium	d) nickel and aluminum
267.	Trimming is a process associated v	,	o, coppor and alammam	a) motor and aranmam
	a) Machining of metals	b) Press work	c) Electroplating	d) Forging
268.		minium results in which of the following?		
	a) Improving machinability		b) One of the best known age and pred	
269	c) Improvement of corrosion re Which of the following does not per		d) Improvement of casting characteristic	JS
200.	a) Welding	b) Soldering	c) Casting	d) Brazing
270.		rtional to the ultimate strength but not wi		-, - · · · · · · · · · · · · · · · · · ·
	a) Yield strength	b) Design stress	c) Shear stress	d) b and c
271.		e all the time during which motion takes		
272	a) Deceleration	b) Uniform motion	c) Acceleration	d) None of these
212.	as:	e is often defined as the angle between	the line of action and the line of tanger	it to the pitch choie. It is termed
	a) Helix angle	b) Angle of recess	c) Angle of obliquity	d) Arc of action
273.	The function of the clutch in the ma		, 3	,
	 a) Lowering the drive speed 		b) To disconnect or connect the shafe	
074	c) Alignment of drive shaft	100-100-01-00-00-0	d) To insure that two shafts line up at hi	gh speed
2/4.	Which of the following is not a heat a) Sintering	b) Annealing	c) Hardening	d) Tempering
275.	An instrument used to analyse the	,	c) Hardering	d) Tempering
	a) Spectrometer	b) Radiometer	c) Profilometer	d) Ultimate analyser
276.	Creep is the term used in referring			
	a) Elongation before yield point	b) Maximum stress of brittle n		del eddended to store
277	c) Fatigue limit of ductile materials		he strain, or deformation, of any mater hat of ferrous metal has a maximum strer	
211.	a) 400 °F	b) 450 °F	c) 1200 °F	d) 350 °F
278.	Which of the following is not a kind	,	3, 1233 1	4, 555
	a) Malleable iron	b) Head iron	c) Gray iron	d) White iron
279.	•	•	idle which is also static while the work is	•
200	a) Dead center	b) Live center	c) Focal center	d) Work center
200.	Which process does not belong to tall a) Resistance welding	b) Soldering	c) Hardening	d) Brazing
281.	It is the maximum stress achieved i	,	o, naraoning	a) Diazing
	a) Yield Strength	b) Ultimate strength	c) Elastic strength	d) Shear strength
282.	A support where one end is pinned			
202	a) Simply supported beam	b) Column	c) Compound supported beam	d) Simple beam
283.		b) Machine element functions	discontinuities, such as concentrated load c) Finite element functions d) Statist	ical Element functions
284.	, ,	s exist while all other tensor elements ar	•	ical Element functions
-	a) Symmetrical tensor	b) Bolt tensor	c) Unsymmetrical tensor	d) Beam tensor
285.	The stress at time of fracture or rup			
	a) Ultimate stress	b) Fracture stress	c) Yield stress	d) Fatigue stress
286.	It is the linear interpolation between a) Rule of thumb	•	c) Rule of mixtures	d) Slide rule
287	,	b) Miner's rule ements forming long-chain molecules.	c) Rule of Illixtures	d) Slide fale
2011		b) Ceramics	c) Glasses	d) Isotropic
288.		o longer linearly proportional to strain.	,	,
	a) Proportional limit	b) Elastic limit	c) Ultimate limit	d) Yield limit
289.			igh temperature gradient that induces a s	tress gradient. Some metal
		c as a result. How do you call this pheno		d) Quenching
290	a) Thermal-shock failure It is the condition of a machine el	b) Thermal fatigue ement when it is completely inoperable	c) Honing e, cannot perform its intended function	d) Quenching adequately, or is unreliable for
_55.	continued safe use. What do you ca		-, por one interior remotion	quartery, or to difficultion to
	a) Fail-safe condition	b) Failure condition	c) Critical condition	d) Salvage condition
291.		plume divided by the original volume.		
	a. linearly elastic	b. camber	c. dilatation	d. partially elastic
292	Material that can sustain elongation	greater than 5% before fracture is calle	ed:	
	a) Brittle	b. ductile	c. deformable	d. rigid

293. 294.	The name given to a load whose v a) Static load	alue is constant over long time b. Sustained load	is:	c. Average load		d. Axial load
295.	It is the peak of the stress-strain cua) Yield strength It is a synergistic collection of macindividual parts. How do you call the	b) Ultimate tensile strength hine elements. It is synergistic		c) Endurance strength		d) Elastic strength ept greater than the sum of the
	a) Synthesis	b) Analysis		c) Mechanical system		d) Concurrent engineering
	How do you call the ratio of stress a) Modulus of rigidity What is the philosophy of involv development?	b) Modulus of resilience	e beginnir	d) Modulus of elasticity ng of a design effort and ke	eping ther	d) Modulus of stiffness n involved throughout produc
	a) Synthesis	b) Design		c) Mechanical system		d) Concurrent engineering
299.	It is one of the design stages that product is available, it often become How do you call this?					
300.	a) Detail stage That portion of the stress-strain dia a) Yield strength	b) Revision stage agram where there is a large in b) Ultimate tensile strength	crease in s	c) Preliminary stage strain with little or no increase i c) Endurance strength	n stress is	d) Feasibility stage called as: d) Elastic strength
301.	This is the first stage of design pro- ideas generated during this stage descriptions of hardware. How do y	e of the design process gene				
302.	a) Detail stage At this point in the stress-strain correlease of the load.	b) Revision stage urve, the material experiences	some am	c) Preliminary stage nount of plastic strain and thus	s will not re	d) Feasibility stage eturn to its original shape after
303.	a) Elastic limit This type of load is the one that is a		ved. How			d) Yield point
304.	a) Static loadAt the highest point of the stress-sta) Apparent stress	b) Fluctuating loadtrain curve during the test, whab) Ultimate stress	t highest s	c) Random load stress on a test bar of the mate c) Elastic stress	rial is meas	d) Fluctuating load sured? d) Yield stress
306.	This is a stress when a given elem compressive stress; the stress cyc a) Repeated and Reversed Stres These steels contain 30 to 50 poir good ductility and moderate hardnea) High-carbon steels It is the process of increasing the ha) Strain hardening	le is repeated many thousands s b) Static stress nts of carbon (0.30%-0.50%). ess requirements come from th b) Medium-carbon steels	of times. Most mach nis group.	How do you call this type of str c) Minimum stress nine elements having moderat c) Low-carbon steels as it is plastically performed. H	ess? e to high s	d) Maximum stress trength requirements with fairly d) Alloy steels call this process?
308.	Which of the following the average		-) 0 040	u. <i>r</i> 3	-1) 0 0741	1. 73
309.	a) 0.035 lb/in ³ As a flat belt drive designer, what r	b) 0.253 lb/in ³ must be the belt speed to make b) 2000 to 3000 fpm			d) 0.074 l	
240	a) 4000 to 4500 fpm	,	,	0 4000 ipin	u) 4000 ii	o 5000 fpm
310.	As suggested by Taylor, what is the a) 75 lb/in of width	b) 71 lb/in of width	c) 73 lb/ir	n of width	d) 80 lb/ir	n of width
311.	Not adaptable to welding due to lova) copper parts	w tensile strength and poor due b) aluminum parts	•	on materials	d) bronze	parts
312.	Bearing surface that completely su a) offset bearing	rrounds the journal is also calle b) centrally loaded bearing	ed: c) full be	aring	d) babbit	bearing
313.	The purpose of lubrication is as me a) to lighten the load	entioned except: b) prevent adhesion	c) prever	nt corrosion/oxidation	d) cool th	e moving elements
	Tools usually used in wood pattern a) saws and chisel This group of materials characteriz the alloy must have a chromium co a) Stainless steels	b) knives and drills tes the high level of corrosion r		offered by alloys in this group.	d) drill ma To be clas	
	a) Toughness These group of stainless steels fal treatable, and their final properties and full hard. These alloys are non	b) Rigidity I into the AISI 200 and 300 ses are determined by the amoust magnetic and are typically use	nt of work	ing, with the resulting temper processing equipment.	referred to	as 1/4 hard, ½ hard, 3/4 hard
	a) Austenitic stainless steels	b) Ferritic stainless steels		c) Martensitic stainless steels	i	d) None of these

318.	temperatures, from 1300°F to 1900	0°F (700°C-1040°C), depending on the	409, 430, 446, and so on. They are magne alloy. They are not heat-treatable, but the eum refining equipment, automotive trim	can be cold-worked to improve	
	a) Austenitic stainless steels	b) Ferritic stainless steels	c) Martensitic stainless steels	d) None of these	
319.	heat-treated, and have higher stre		403, 410, 414, 416, 420, 431, and 440 ty ille retaining good toughness. Typical us- tings, and marine hardware. c) Martensitic stainless steels		
320.	ASTM A36, which has a minimum	yield point of 36 000 psi (248 MPa) ar	y the American Society for Tesfing and Mond is very ductile. It is basically a low-carl American Standard beams, channels, an c) Cast steels	bon, hot-rolled steel available in	
321.	Compounds of metallic elements, ra) Plastic	most frequently oxides, nitrides, and car b) Polymers	bides. c) Ceramics	d) Alloy	
322.	varieties of tool steel materials h manufacturing engineering, they a required. Also, some of these st	ave been classified into seven general are also pertinent to machine design v	bunches, dies, shearing blades, chisels, and types. Whereas most uses of these swhere the ability to maintain a keen edgoe which may be desirable in machine nos.	teels are related to the field of ge under abrasive conditions is	
	a) Structural steels	b) Tool steels	c) Cast steels	d) Stainless steels	
323.	A material having different properti a) Isotropic material	ies in all directions at point in solid. b) Anisotropic material	c) Orthotropic material	d) Ceramic material	
324.	24. Parts subjected to repeated applications of loads or to stress conditions that vary with time over several thousands or millions of cycles fail because of the phenomenon <i>of fatigue</i> . Materials are tested under controlled cyclic loading to determine their ability to resist such repeated loads. The resulting data are reported as which of the following?				
005	a) Endurance strength	b) Ultimate strength	c) Yield strength	d) Elastic strength	
325.	Which of the following is the other a) Fatigue strength	b) Ultimate strength	c) Yield strength	d) Elastic strength	
326.	A material having different propert material symmetry.	ties in three mutually perpendicular dire	ctions at point in solid and having three r	nutually perpendicular planes of	
	a) Orthotropic material	b) Isotropic material	c) Anisotropic material	d) Thermoplastic material	
327.	The onset of plastic deformation is		•	•	
	a) Necking	b. yielding	c. creeping	d. buckling	