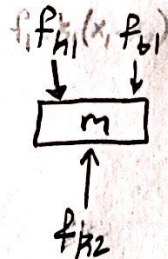
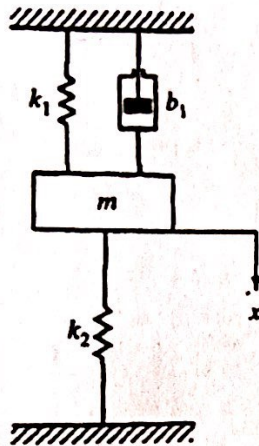


ME 421 Homework #2

Due via Canvas: 11:59pm, Friday, 26th, 2021

1. (20pts) Develop the model of the vertical spring-mass-damper system, assuming the static equilibrium point is the datum point for displacement $x(t)$.



Equations assume initial position is zero and zero gravity

$$f_{k1} = -k_1(x(t))$$

$$f_{b1} = -b_1(x'(t))$$

$$f_{k2} = k_2(x(t))$$

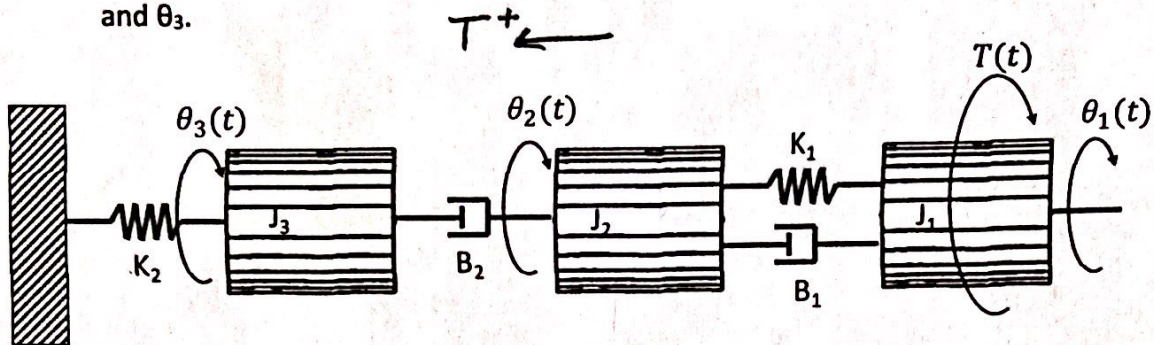
$$m \cdot x''(t) = f_{k1} + f_{b1} - f_{k2}$$

$$m \cdot x''(t) = -k_1 \cdot x(t) - b_1 \cdot x'(t) - k_2 \cdot x(t)$$

or

$$m \ddot{x} = -k_1 x - b_1 \dot{x} - k_2 x$$

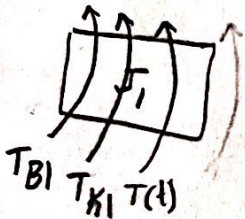
2. (20pts) Obtain a mathematical model of the rotational motion system shown below. The moment of the inertia for the three rotors are J_1 , J_2 , and J_3 . A torque $T(t)$ is applied to J_1 . The angular displacements (rotational angles) of the three rotors are marked as θ_1 , θ_2 , and θ_3 .



$$T_{k2} = k_2(-\theta_3(t)) \quad T_{B2} = B_2(\dot{\theta}_3(t) - \dot{\theta}_2(t))$$

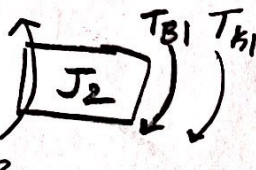
$$T_{k1} = k_1(\theta_2(t) - \theta_1(t)) \quad T_{B1} = B_1(\dot{\theta}_2(t) - \dot{\theta}_1(t))$$

Solving Cylinders ↓



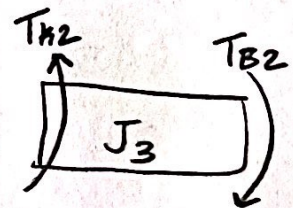
$$J_1 \ddot{\theta}_1 = T_{B1} + T_{k1} + T(t)$$

$$J_1 \ddot{\theta}_1 = B_1(\dot{\theta}_2 - \dot{\theta}_1) + k_1(\theta_2 - \theta_1) + T$$



$$J_2 \ddot{\theta}_2 = T_{B2} - T_{k1} - T_{B1}$$

$$= B_2(\dot{\theta}_3 - \dot{\theta}_2) + k_1(\theta_1 - \theta_2) + B_1(\dot{\theta}_1 - \dot{\theta}_2)$$



$$J_3 \ddot{\theta}_3 = T_{k2} - T_{B2}$$

$$= k_2(-\theta_3) + B_2(\dot{\theta}_2 - \dot{\theta}_3)$$

Math Model:

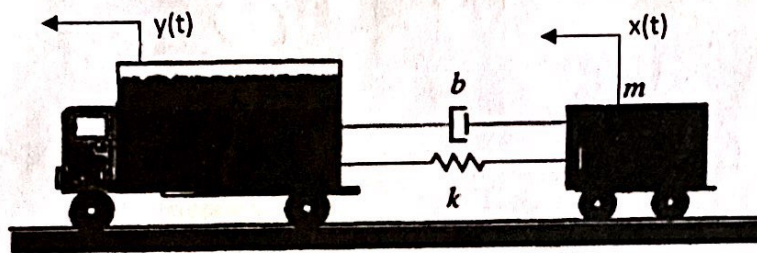
$$① \quad J_1 \ddot{\theta}_1 = B_1(\dot{\theta}_2 - \dot{\theta}_1) + k_1(\theta_2 - \theta_1) + T$$

$$② \quad J_2 \ddot{\theta}_2 = B_2(\dot{\theta}_3 - \dot{\theta}_2) + k_1(\theta_1 - \theta_2) + B_1(\dot{\theta}_1 - \dot{\theta}_2)$$

$$③ \quad J_3 \ddot{\theta}_3 = B_2(\dot{\theta}_2 - \dot{\theta}_3) - k_2(\theta_3)$$

3. (20pts) A cart of mass m is attached to a truck of mass M using a spring of stiffness k and a damper of constant b , as shown in the figure below. The traction force applied to the truck is $f(t)$. Both the truck and the cart begin to move from a parking position. Assume the road friction can be ignored.

- (a) (10pts) Develop the transfer function of the system with $f(t)$ as input and $x(t)$ as output.
 (b) (10pts) Develop the transfer function of the system with $f(t)$ as input and $y(t)$ as output.



$$f(t) \leftarrow \boxed{M} \begin{matrix} \rightarrow f_b \\ \rightarrow f_k \end{matrix} \begin{matrix} f_b \leftarrow \\ f_k \leftarrow \end{matrix} \boxed{m} \quad \begin{matrix} f_b = B(\dot{y} - \dot{x}) \\ f_k = k(y - x) \end{matrix}$$

$$\textcircled{1} \quad M\ddot{y} = f - f_b - f_k = f - B(\dot{y} - \dot{x}) - k(y - x)$$

$$\textcircled{2} \quad m\ddot{x} = f_b + f_k = B(\dot{y} - \dot{x}) + k(y - x)$$

$$\mathcal{L}\{\textcircled{1}\} = Ms^2 Y(s) + Bs Y(s) + k Y(s) = F(s) + Bs X(s) + k X(s) \quad \textcircled{3}$$

$$\mathcal{L}\{\textcircled{2}\} = ms^2 X(s) + Bs X(s) + k X(s) = Bs Y(s) + k Y(s) \quad \textcircled{4}$$

$$\textcircled{4} \Rightarrow Y(s) = \frac{(ms^2 + Bs + k) X(s)}{(Bs + k)} \xrightarrow{\text{put in } \textcircled{3}} \frac{(Ms^2 + Bs + k)(ms^2 + Bs + k) X(s)}{(Bs + k)}$$

$$F(s) = \left(\frac{(Ms^2 + Bs + k)(ms^2 + Bs + k) - (Bs + k)^2}{(Bs + k)} \right) X(s) = F(s) + \frac{(Bs + k)}{X(s)}$$

$$a.) \quad \frac{X(s)}{F(s)} = \frac{(Bs + k)}{(Ms^2 + Bs + k)(ms^2 + Bs + k) - (Bs + k)^2} = G(s)$$

$$b.) \quad (4) \rightarrow X(s) = \frac{(Bs+k) Y(s)}{ms^2+Bs+k}$$

$$(3) \rightarrow (Ms^2+Bs+k) Y(s) = F(s) + (Bs+k) X(s) \quad \text{substitute into}$$

$$\downarrow$$

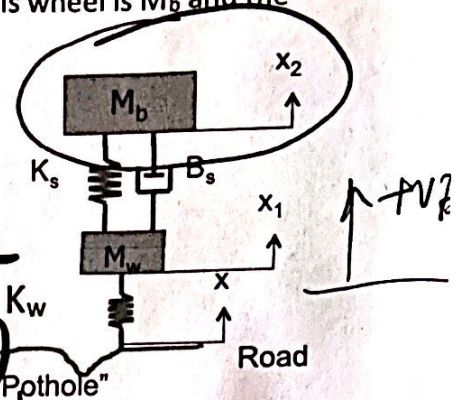
$$(Ms^2+Bs+k) Y(s) = F(s) + \frac{(Bs+k)^2 Y(s)}{ms^2+Bs+k}$$

$$\left((Ms^2+Bs+k) - \frac{(Bs+k)^2}{ms^2+Bs+k} \right) Y(s) = F(s)$$

$$\frac{(Ms^2+Bs+k)(ms^2+Bs+k) - (Bs+k)^2}{ms^2+Bs+k} = \frac{F(s)}{Y(s)}$$

$$G(s) = \frac{Y(s)}{F(s)} = \frac{ms^2+Bs+k}{(Ms^2+Bs+k)(ms^2+Bs+k) - (Bs+k)^2}$$

4. (40pts) The equivalent model for one wheel of a pickup truck is illustrated in the following figure. The mass of the vehicle that is distributed on this wheel is M_b and the mass of the wheel is M_w . The suspension spring has a spring constant K_s and the tire has a spring constant of K_w . The damping constant of the shock absorber is B_s .



- (a) (15pts) Derive the transfer function of the system with $x(t)$ as input and $x_2(t)$ as output.

$$G(s) = \frac{X_2(s)}{X(s)}$$

- (b) (15pts) Using MATLAB plot the displacement of the truck $x_2(t)$ for $0 < t < 20$ s when the truck runs through a pothole of depth of 0.1m for each set of model parameters given below. Include the MATLAB CODE and the plot in your homework submission. Label the x-axis and y-axis of your plot.

Assume

- Set 1. $M_w = 50\text{kg}$, $M_b = 400\text{kg}$, $K_w = 200,000\text{ N/m}$, $K_s = 20,000\text{ N/m}$ and $B_s = 1,000\text{ N-s/m}$.
 Set 2. $M_w = 50\text{kg}$, $M_b = 600\text{kg}$, $K_w = 200,000\text{ N/m}$, $K_s = 20,000\text{ N/m}$ and $B_s = 1,000\text{ N-s/m}$.
 Set 3. $M_w = 50\text{kg}$, $M_b = 600\text{kg}$, $K_w = 200,000\text{ N/m}$, $K_s = 20,000\text{ N/m}$ and $B_s = 10,000\text{ N-s/m}$.

Hint: use the impulse response function in MATLAB. You can use the MATLAB code "ExampleMatlabCodeSimulatingSystemResponses" on Canvas.

- (c) (10pts) Based on the plots in (a) comment on how weight of the car body M_b AND the damping constant of the shock absorber B_s affect the amplitude and duration of the vertical vibration of the car.

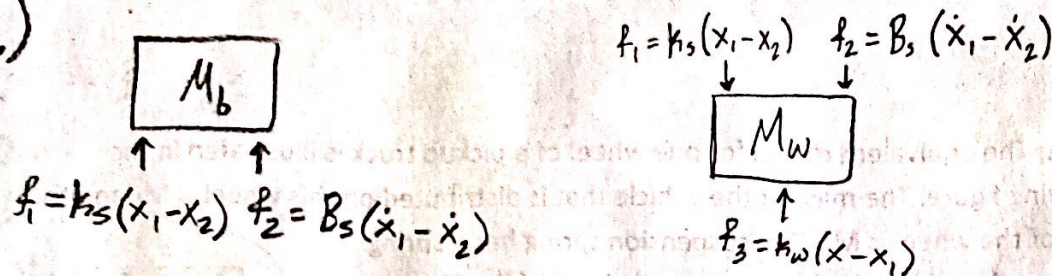
$$x_1(t) = \delta(t)$$



$$x_1(t) = -0.1 \cdot \delta(t) \Rightarrow x_2(t) = ?$$



4.) a.)



$$M_b \ddot{x}_2 = f_1 + f_2 = k_s(x_1 - x_2) + B_s(\dot{x}_1 - \dot{x}_2) \leftarrow \textcircled{1}$$

$$M_w \ddot{x}_1 = f_3 - f_1 - f_2 = k_w(x - x_1) - k_s(x_1 - x_2) - B_s(\dot{x}_1 - \dot{x}_2) \leftarrow \textcircled{2}$$

$$\mathcal{L}\{\textcircled{1}\} = M_b s^2 X_2(s) + k_s X_2(s) + B_s X_2(s) = k_s X_1(s) + B_s X_1(s)$$

$$X_1(s) = \frac{(M_b s^2 + k_s + B_s) X_2(s)}{(k_s + B_s + s)}$$

$$\mathcal{L}\{\textcircled{2}\} = M_w s^2 X_1(s) + k_s X_1(s) + (B_s + s) X_1(s) + k_w X_1(s) = k_w X(s) + k_s X_2(s) + (B_s + s) X_2(s)$$

$$X_1(s) (M_w s^2 + k_s + B_s + s + k_w) = (k_w X(s) + (k_s + B_s + s) X_2(s)) \leftarrow \textcircled{4}$$

③ into ④

$$\left(\frac{(M_b s^2 + k_s + B_s + s)(M_w s^2 + k_s + B_s + s + k_w)}{(k_s + B_s + s)(k_w)} \right) X_2(s) = \left(\frac{k_w}{k_w} \right) X(s) + \frac{(k_s + B_s + s)}{k_w} X_2(s)$$

$$\left(\frac{(M_b s^2 + k_s + B_s + s)(M_w s^2 + k_s + B_s + s + k_w)}{(k_s k_w + B_s k_w + s)} \right) X_2(s) = \left(\frac{(k_s + B_s + s)(k_s + B_s + s)}{k_w (k_s + B_s + s)} \right) X(s) + X_2(s)$$

$$G(s) = \frac{X_2(s)}{X(s)} = \frac{(k_s + B_s + s) k_w}{(M_b s^2 + k_s + B_s + s)(M_w s^2 + k_s + B_s + s + k_w) - (k_s + B_s + s)^2}$$