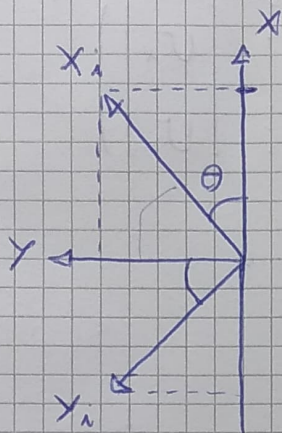


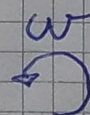
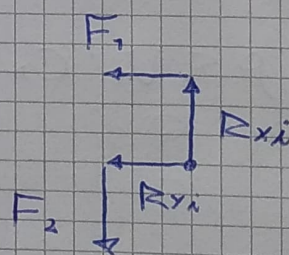
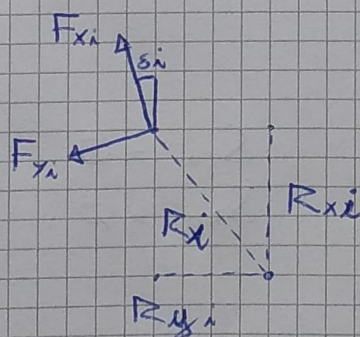
$$\begin{bmatrix} m \dot{U}_x \\ m \dot{U}_y \\ J \dot{\omega} \end{bmatrix} = \begin{bmatrix} F_x + m \omega U_y - \frac{1}{2} \rho S U^2 C_D \\ F_y - m \omega U_x \\ \tau \end{bmatrix}$$



$$\begin{cases} X \rightarrow +X_i \cos \theta \\ Y \rightarrow +X_i \sin \theta \\ X \rightarrow -Y_i \sin \theta \\ Y \rightarrow Y_i \cos \theta \end{cases}$$

$$\begin{bmatrix} F_x \\ F_y \end{bmatrix} = \sum_{i=1}^4 \begin{bmatrix} \cos \theta_{wi} & -\sin \theta_{wi} \\ \sin \theta_{wi} & \cos \theta_{wi} \end{bmatrix} \begin{bmatrix} F_{xi} \\ F_{yi} \end{bmatrix}$$

from WHEEL to BODY

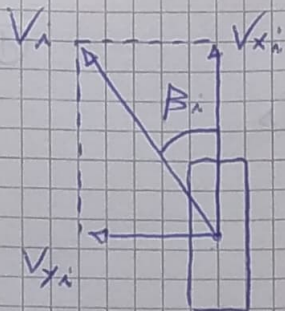


$$\begin{cases} F_1 = F_{xi} \cos \delta + F_{yi} \sin \delta \\ F_2 = F_{xi} \sin \delta - F_{yi} \cos \delta \end{cases}$$

$$T = F_1 R_{xi} + F_2 R_{yi}$$

$$\begin{cases} F_{xi} = N_i M(\lambda_i, \theta_i) \\ F_{yi} = -N_i M(\beta_i, \theta_{\beta i}) \end{cases}$$

$$M(s, \theta_s) = \theta_{1s} (1 - e^{-s \theta_{2s}}) - s \theta_{3s}$$

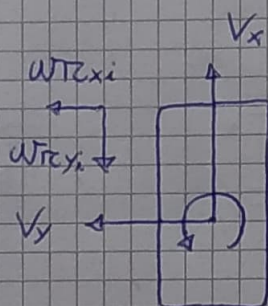


$$B_i = TG^{-1} \begin{pmatrix} U_{yi} \\ U_{xi} \end{pmatrix}$$

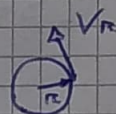
$$\begin{bmatrix} V_{xi} \\ V_{yi} \end{bmatrix} = \begin{bmatrix} C \delta_{wi} & S \delta_{wi} \\ -S \delta_{wi} & C \delta_{wi} \end{bmatrix} \left[W \begin{bmatrix} -r_{xi} \\ r_{xi} \end{bmatrix} + \begin{bmatrix} U_x \\ U_y \end{bmatrix} \right]$$

rotation from BODY
to WHEEL

$$\begin{cases} U_x - W r_{xi} \\ U_y + W r_{xi} \end{cases}$$



become $V_R = rW$



EQUILIBRIUM Trajectory

$$\begin{bmatrix} F_{x0} \\ F_{y0} \\ \tau_0 \end{bmatrix} = \begin{bmatrix} \frac{1}{2} \rho S U_0^2 C_D \\ 0 \\ 0 \end{bmatrix}$$

$$\begin{aligned} \delta_{F0} &= 0 & \lambda_3 &= 0 \\ \delta_{\tau0} &= 0 & \lambda_4 &= 0 \end{aligned}$$

$$F_{x0} = \sum_{i=1}^2 N_i m (\lambda_i, \theta_{\lambda i})$$

FWD

$$\tau_0 = \sum_{i=1}^2 -N_i m (\lambda_i, \theta_{\lambda i}) r_{xi}$$

$$\begin{bmatrix} N_1 \\ N_2 \\ N_3 \\ N_4 \end{bmatrix} = \begin{bmatrix} R_{xR} \\ R_{xR} \\ R_{xF} \\ R_{xF} \end{bmatrix} \frac{m \delta}{2(R_{xF} + R_{xR})} + \begin{bmatrix} -1 \\ -1 \\ 1 \\ 1 \end{bmatrix} \frac{C_D R_z - C_m (R_{xF} + R_{xR})}{2(R_{xF} + R_{xR})} \frac{1}{2} \rho S U_0^2$$

$$R_{x1} = R_{x2} = R_{xF}$$

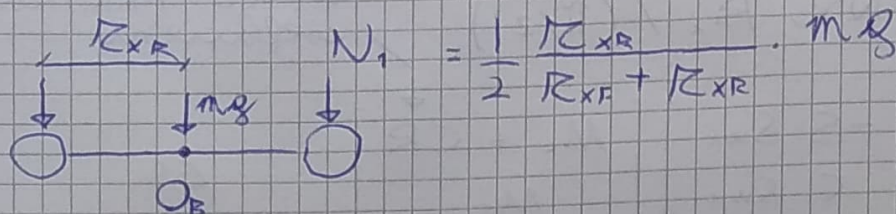
$$R_{y1} = R_{y4} = R_y$$

$$R_{x3} = R_{x4} = -R_{xR}$$

$$R_{y2} = R_{y3} = -R_y$$

• Vehicle MASS:

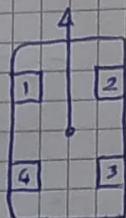
③



$$R_{xF}, R_{xR}, R_y > 0$$

IMPORTANT!

$$\begin{cases} R_{x1}, R_{x2} > 0 \\ R_{x3}, R_{x4} < 0 \\ R_{y1}, R_{y4} > 0 \\ R_{y2}, R_{y3} < 0 \end{cases}$$



• Pitch MOMENTUM:

$$N_x = \left[* \right] \cdot \frac{1}{2} \cdot \frac{1}{2} \cdot \rho S U_o^2$$

$$* \rightarrow \text{FRONT} = - \frac{C_D R_z - C_M (R_{xF} + R_{xR})}{(R_{xF} + R_{xR})}$$

$$\rightarrow \text{REAR} = \frac{C_D R_z - C_M (R_{xF} + R_{xR})}{(R_{xF} + R_{xR})}$$

C_m = aerodynamic PITCH MOMENTUM coefficient

MATLAB CODE

ESP_NLSIM_V1

$$\underline{\underline{F_{AB}}} = \frac{1}{2} \rho S U^2 \begin{bmatrix} C_{x0} \cos(\beta) \\ C_{y0} \sin(\beta) \end{bmatrix} \quad 22$$

$$V_{wi} = \begin{bmatrix} C(\delta_{wi}) & S(\delta_{wi}) \\ -S(\delta_{wi}) & C(\delta_{wi}) \end{bmatrix} \left(W \begin{bmatrix} -R_{yi} \\ R_{xi} \end{bmatrix} + U \begin{bmatrix} \cos(\beta) \\ \sin(\beta) \end{bmatrix} \right) \quad 46$$

$$\beta_i = \text{ATAN} \left(\frac{V_{yi}}{V_{xi}} \right) \quad 50$$

$$M_L = M_{\text{LONG}}(\text{KIND}, L(i))$$

$$M_S = M_{\text{LONG}}(\text{KIND}, B_i / (\pi/2))$$

WHEEL FRICTION coefficient

$$M_B = \begin{bmatrix} C \delta_{wi} & -S \delta_{wi} \\ S \delta_{wi} & C \delta_{wi} \end{bmatrix} \begin{bmatrix} M_L + CR \\ M_S \end{bmatrix} \quad 61$$

$$\underline{\underline{\tau_{B,i}}} = \begin{bmatrix} (S \delta_{wi} R_{xi} - C \delta_{wi} R_{yi}) & (C \delta_{wi} R_{xi} + S \delta_{wi} R_{yi}) \end{bmatrix} M_B$$

Reference the M COMPONENTS in the BODY reference frame 66 SLOPE

$$\underline{\underline{F_z W}} = H^+ \left(\frac{R_z}{2} \rho S U^2 \begin{bmatrix} 0 \\ C_{yB} \sin \beta \\ -C_{x0} \cos \beta \end{bmatrix} + \begin{bmatrix} m g \cos \theta \\ -R_z m U W \cos \beta \end{bmatrix} \right) \quad (*)$$

$$(*) - m g \sin \theta - R_z m U W \sin \beta$$

$$\begin{cases} F_B = \sum_1^4 i M_{B(i)} F_z W(i) + 100 \begin{bmatrix} V_0 - U \\ 0 \end{bmatrix} + F_{aB} \\ \tau_B = \sum_1^4 \tau_{B,i}(i) F_z W(i) + \tau_{eq} \end{cases} \quad 82$$

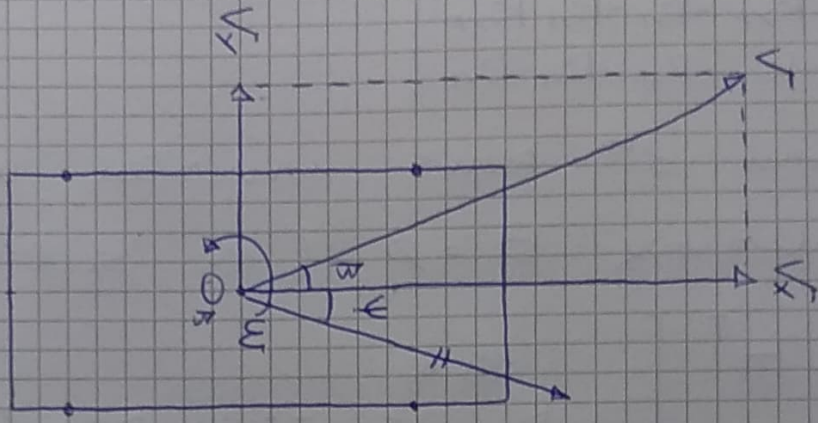
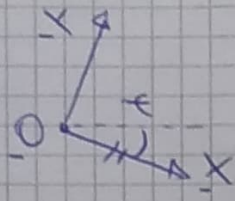
(!) CONTROLS ↑

PLANT

$$dV = \begin{bmatrix} m^{-1} & 0 & 0 \\ 0 & (mU)^{-1} & 0 \\ 0 & 0 & I^{-1} \end{bmatrix} \begin{bmatrix} CB & SB & 0 \\ -SB & CB & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} F_B \\ \tau_B \end{bmatrix} - \begin{bmatrix} 0 \\ mU \\ 0 \end{bmatrix} \quad 80$$

$$dP = \begin{bmatrix} U \cos(\psi + \beta) \\ U \sin(\psi + \beta) \end{bmatrix} \quad d\psi = W$$

$$dx = \begin{bmatrix} dP \\ dV * [0; 1; 1] \\ d\psi \\ \text{TRACK_ERROR} \\ \text{DOT_HAT_X} \end{bmatrix}$$

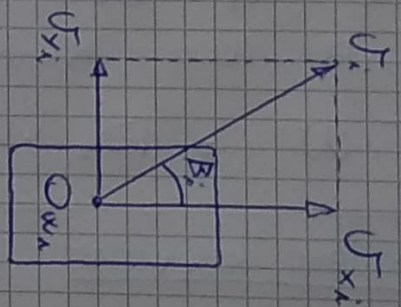


F_{RB} is ALONG the U DIRECTION

C_{x0} = DRAG coefficient along X

C_{y0} = DRAG coefficient along Y

V_{wi} } is CONSISTENT between NOTES and MATLAB
 β_i } ☺



$$F_{RB} = \begin{bmatrix} R_{xi} & R_{yi} \end{bmatrix} \begin{bmatrix} S \delta_{wi} & C \delta_{wi} \\ -C \delta_{wi} & S \delta_{wi} \end{bmatrix} \begin{bmatrix} C \delta_{wi} & -S \delta_{wi} \\ S \delta_{wi} & C \delta_{wi} \end{bmatrix} \begin{bmatrix} M_L + C_R \\ M_S \end{bmatrix}$$

$$\begin{bmatrix} 2SC & -S^2 + C^2 \\ -C^2 + S^2 & 2SC \end{bmatrix}$$

Not coherent with the NOTES (2 ROTATIONS instead of 1)

⇒ suppose NOTES CORRECT

5th MATLAB

$$R_{xF} > 0 \quad R_{xR} < 0 \quad R_{yF} > 0$$

$$H = \begin{bmatrix} 1 & 1 & 1 & 1 \\ R_{xF} & -R_{yF} & -R_{yF} & R_{xF} \\ -R_{xF} & -R_{yF} & -R_{yF} & -R_{xR} \end{bmatrix}$$

2ND and 3RD ROWS inverted with NOTES

Considering $\Theta = 0$ (no slope)

$$F_z W = H^T \begin{bmatrix} m \cdot g \\ \frac{1}{2} R_2 \rho S V^2 \sin(\beta) x_B - R_2 m V W \cos \beta \\ -\frac{1}{2} R_2 \rho S V^2 \cos(\beta) x_0 - R_2 m V W \sin \beta \end{bmatrix}$$

What changes:

- ON NOTES, the AIR RESISTANCE is only along X (no Y component $\rightarrow \beta = 0$)
- ON NOTES is also present the C_m component
- ON MATLAB we have the action of:

$$R_2 m V W = R_2 \cdot \text{ANGULAR MOMENTUM} \quad \text{WHY?}$$

EQUILIBRIUM SOLVER

$$\text{LAMBDA0S} = \text{FSOLVE} (@ \text{LONG_EQ}, 0);$$

$$\text{LAMBDA0} = [\text{LAMBDA0S}; \text{LAMBDA0S}; 0; 0]$$

$$\text{LONG_EQ}(x) = \frac{1}{2} \rho S V_0^2 C_{x0} + F_z W \cdot \begin{bmatrix} M_L(K_{IND}, x) + CR \\ M_L(K_{IND}, x) + CR \\ CR \\ CR \end{bmatrix}$$

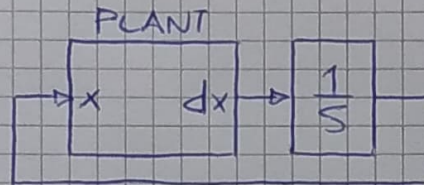
Back to the PLANT:

⑦

$$dV = [F_{Bx} \cos \beta + F_{By} \sin \beta] \frac{1}{m}$$

$$\underline{\underline{d\beta}} = [-F_{Bx} \sin \beta + F_{By} \cos \beta] \frac{1}{mV} - \omega \quad *$$

Instead of having v_x and v_y , here we have V and β

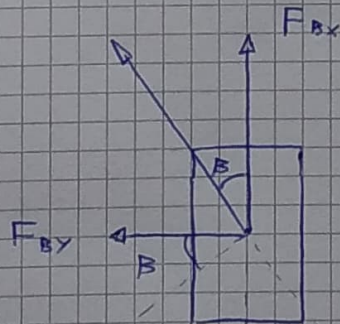


$$X = [P_x \ P_y \ V \ \beta \ \omega \ \psi \ r \ \hat{x}]$$

$$dx = [dP_x \ dP_y \ dV \ d\beta \ d\omega \ \omega \ \text{TRACK ERR} \ \dot{\hat{x}}]$$

multiplied by ϕ WHY?

*



$$\frac{-F_{Bx} \sin \beta + F_{By} \cos \beta}{m}$$

$dV_{\text{TANGENTIAL}}$

$$\Rightarrow d\beta = \frac{dV_{\text{TAN}}}{V} - \omega$$

Strange, the measurement unit is not consistent